

- REVISIONS -									
ZONE	REV	DESCRIPTION	CHK BY	CHK BY	APP BY	REL BY	DATE		
	A	UPDATED ALL WELD SYMBOLS PER FABRICATOR, UPDATED DRAWING PER CUSTOMER AND FABRICATOR MARKUPS	DPM	DBR	DBR	SDM	10/14/11		
	B	CHANGED MIN LENGTH & EXTERNAL PROJECTION OF N8	DPM	DPM	DPM	SDM	03/02/12		
	C	CHANGED EXTERNAL PROJECTION OF N8	DPM	DPM	DPM	SDM	03/21/12		
	D	UPDATED CUSTOMER PO No	DPM	DBR	DBR	SDM	05/30/12		

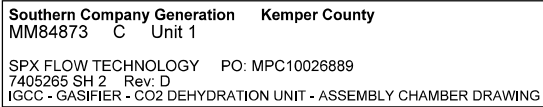
- CUT TABLE -			
ITEM	QTY	CUT	CUT FROM DESCRIPTION
101	1	DETAIL	DETAIL FORGING 24.2500D,19.25ID,SA105
201	1	DETAIL	DETAIL PLATE,PVQ .375,SA516GR70
301	1	40 1/4	BBE PIPE,WELDED 20.000,SCH 20,SA53B-ERW
401	2	DETAIL	DETAIL PL,STOCK .750,SA36
402	1	DETAIL	DETAIL PL,STOCK .750,SA36
403	2	DETAIL	DETAIL PL,STOCK .750,SA36
404	2	DETAIL	DETAIL PL,STOCK .750,SA36
405	6	DETAIL	DETAIL PL,STOCK .750,SA36
406	6	DETAIL	DETAIL PL,STOCK .750,SA36
407	6	DETAIL	DETAIL PL,STOCK .750,SA36
408	1	DETAIL	DETAIL PL,STOCK .750,SA36
501	2	38	DETAIL BAR,EXTRUSN WT6X7,SA36
601	1	DETAIL	SBE BAR,SQUARE SA36, .500X .500,HR,ORIGIN
701	1	DETAIL	SBE BAR,RECT SA36,1.000X1.000,HR
801	2	DETAIL	REF,CTF GRATING,BAR 2X3/16IN,CS,3X20FT
802	2	DETAIL	REF,CTF GRATING,BAR 2X3/16IN,CS,3X20FT
901	2	128 3/4 DIA	REF,CTF SHEET,PERF 20G,304SST,.063X.500,48X120
1001	1	DETAIL	DETAIL PL,STOCK 1.000,SA36
1002	1	DETAIL	DETAIL PL,STOCK 1.000,SA36
1101	1	DETAIL	SBE PIPE,SMLS 3.000,SCH160,SA106B
1201	1	10	SBE PIPE,SMLS 3.000,SCH160,SA106B
1301	4	DETAIL	DETAIL PL,STOCK 1.250,SA36
1401	1	DETAIL	DETAIL PL,STOCK .500-48.00X120.00,SA36
1402	1	DETAIL	DETAIL PL,STOCK .500,SA36
1501	1	DETAIL	SBE BAR,ROUND SA36,1.000,HR
1601	1	DETAIL	DETAIL PL,STOCK .250,SA36
1602	1	DETAIL	DETAIL PL,STOCK .250,SA36
1701	5	DETAIL	SBE BAR,ROUND SA36, .750,HR
1702	3	7 1/2	SBE BAR,ROUND SA36, .750,HR
1801	4	1 3/8	SBE PIPE,SMLS 3.000,SCH 40,SA106B

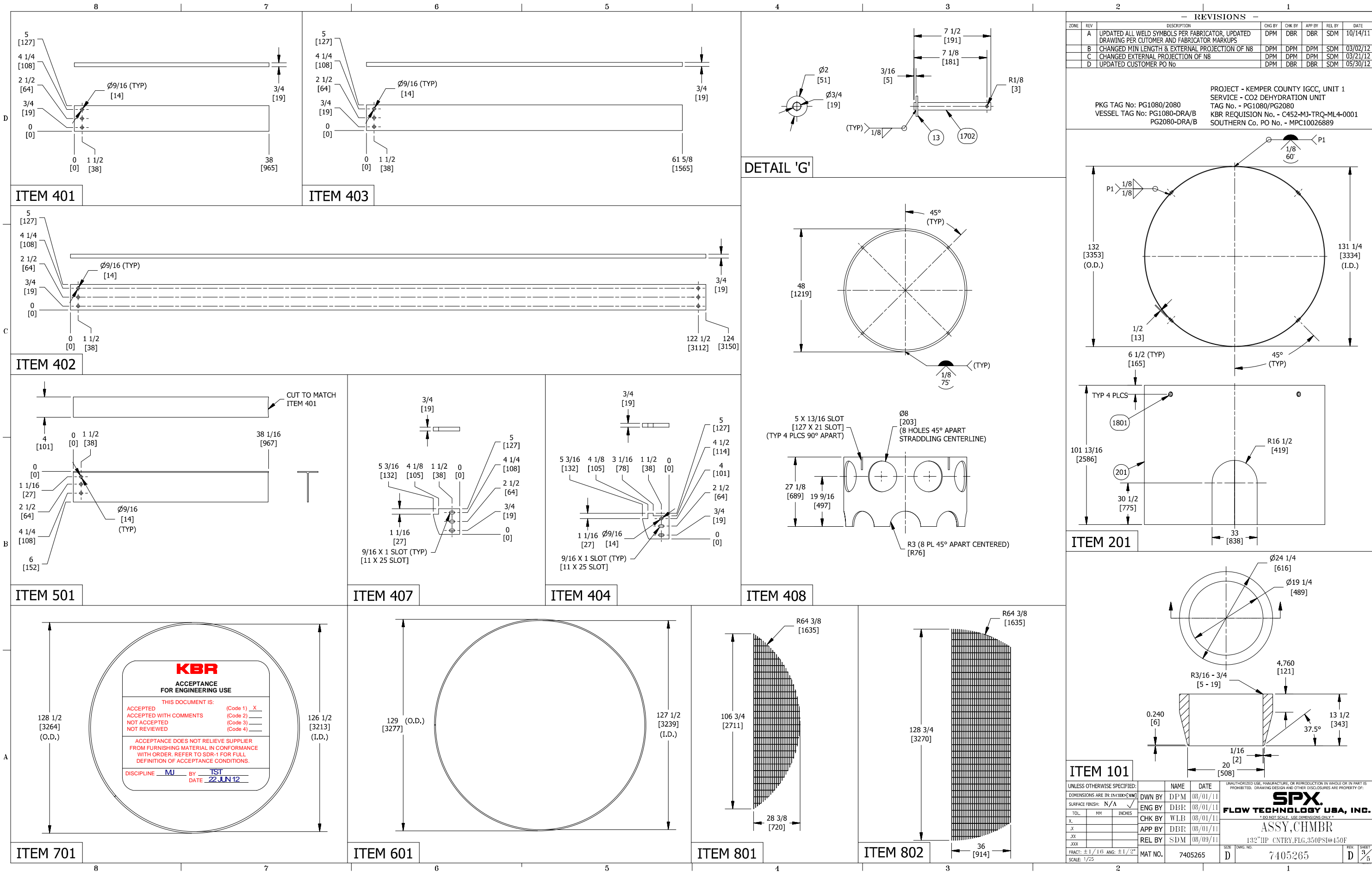
*SEE PAGE 5 NOZZLE TABLE FOR MIN. NOZZLE LENGTHS

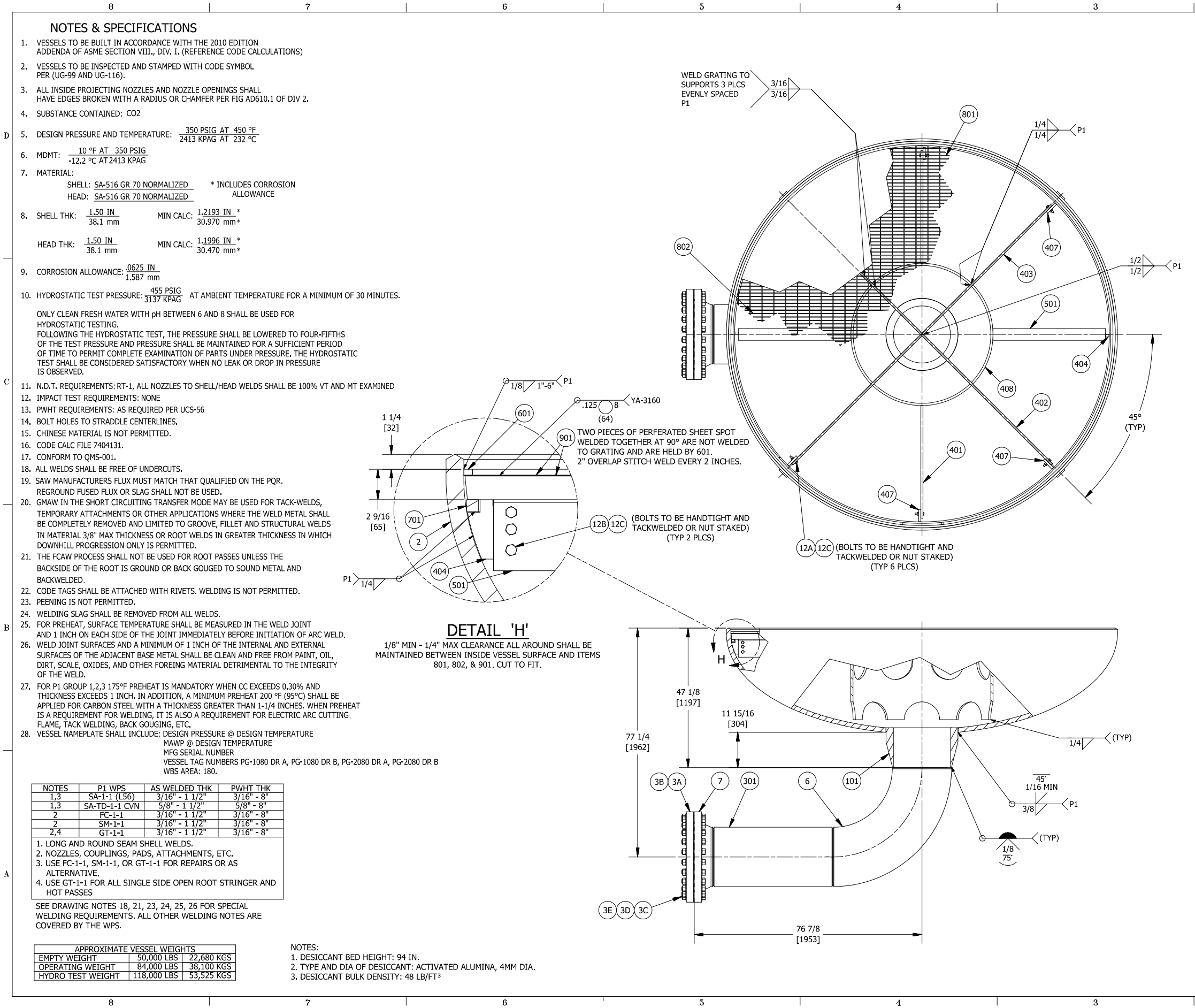
- BILL OF MATERIAL -		
ITEM	DESCRIPTION	QTY
1	PLATE,PVQ SHELL 132.00 DIA, 1.5 THK,SA516GR70N	1
2	HEAD,2:1 ELP 132.000,1.500, 35.75,SA516GR70N	2
3	FLG,RFWN FLG,RFWN 20.000,CLS300,LONGWN,SA105	2
3A	FLG,BLIND 20.000,RF,CLS300,SA105 (2 FOR HYDRO)	3
3B	GSKT,FLANGE 20.00, .175,300#,RF,SPIRAL WOUND 304/GRPH/304	3
3C	STUD,THREAD 1.250- 8X 8.000,SA193B7,GALV A153	72
3D	NUT,HEXAGON 1.250- 8,SA194-2H,GALV A153	144
3E	WSHR,PLAIN 1.250,CS,NARROW,GALVANIZED A153	144
4	FLG,RFWN 8.000,CLS300,LONGWN,SA105	3
4A	FLG,BLIND 8.000,RF,CLS300,SA105	3
4B	GSKT,FLANGE 8.00, .175,300#,RF,SPIRAL WOUND 304/GRPH/304	3
4C	STUD,THREAD .875- 9X 6.000,SA193B7 GALV A153	36
4D	NUT,HEXAGON .875-9,SA194-2H GALVANIZED A153	72
4E	WSHR,PLAIN .875, CS, NARROW, HOT DIP GALV A153	72
5	HANDLE,LIFT 14X6,3/4DIA,SA36	1
6	ELBOW,WELD 16.00, LR,90,SCH 40,SA234GR	1
7	FLG,RFWN 20.000,CLS300,SCH 20,SA105	1
8	FLG,RFWN FLG,RFWN 2.000,CLS300,LONGWN,SA105	1
8A	FLG,BLIND 2.000,RF,CLS300,SA105 (FOR HYDRO)	1
8B	GSKT,FLANGE 2.00, .175,300#,RF,SPIRAL WOUND 304/GRPH/304	1
8C	STUD,THREAD .625- 11X 4.000,SA193B7 GALV A153	8
8D	WSHR,PLAIN .625,CS,NARROW,HOT DIP GALVANIZED A153	16
8E	NUT,HEXAGON .625-11,SA194-2H GALVANIZED A153	16
9	FLG,RFWN FLG,RFWN 1.500,CLS300,LONGWN,SA105	1
9A	FLG,BLIND 1.500,CLS300,SA105 (FOR HYDRO)	1
9B	GSKT,FLANGE 1.50, .175,300#,RF,SPIRAL WOUND 304/GRPH/304	1
9C	STUD,THREAD .750- 10X 4.000,SA193B7 GALV	4
9D	WSHR,PLAIN .750,CS,NARROW,HOT DIP GALVANIZED A153	8
9E	NUT,HEXAGON .750-10,SA194-2H GALVANIZED A153	8
10	BOLT,EYE 1.000- 8X 12.000,CS,GALV A153	1
10A	WSHR,PLAIN 1.000,CS,WIDE,GALV A153	1
10B	NUT,HEXAGON 1.000- 8,SA194-2H,GALV A153	2
11	NPL,STANDOFF SUPPLIER STANDOFF	1
11A	NPL,CODE SUPPLIER ASME NAMEPLATE	1
12A	BOLT,HEX .500-13X 2.750,SA193B8	18
12B	BOLT,HEX .500-13X 2.000,SA193B8	6
12C	NUT,HEXAGON .500-13,SA194B8	24
13	WSHR,PLAIN .750,CS,WIDE,GALVANIZED A153	6
14	WSHR,PLAIN .750,DELIN,WIDE	6
15	PIN COTTER PIN 3/16" X 1-1/4",CS	3

UNLESS OTHERWISE SPECIFIED:		NAME	DATE
DIMENSIONS ARE IN: INCHES (MM)		DPM	08/01/11
SURFACE FINISH: N/A		ENG BY	DBR 08/01/11
TOL. MM INCHES		CHK BY	WLB 08/01/11
X.		APP BY	DBR 08/01/11
.X		REL BY	SDM 08/09/11
.XX			
.XXX			
FRACT: ± 1/16 ANG: ± 1/2°		MAT NO.	7405265
SCALE: 1/2"		SIZE	DWG. NO. 7405265
		REV.	1/5

Southern Company Generation Kemper County	
MM84872 C Unit 1	
SPX FLOW TECHNOLOGY PO: MPC10026889	
7405265 SH 1 Rev: D	
IGCC - GASIFIER - CO2 DEHYDRATION UNIT - ASSEMBLY CHAMBER DRAWING	







2 1

REVISIONS

ZONE	REV	DESCRIPTION	CHG BY	CHK BY	APP BY	REL BY	DATE
	A	UPDATED ALL WELD SYMBOLS PER FABRICATOR, UPDATED DRAWING PER CUTOMER AND FABRICATOR MARKUPS	DPM	DBR	DBR	SDM	10/14/11
	B	CHANGED MIN LENGTH & EXTERNAL PROJECTION OF N8	DPM	DPM	DPM	SDM	03/02/12
	C	CHANGED EXTERNAL PROJECTION OF N8	DPM	DPM	DPM	SDM	03/21/12
	D	UPDATED CUSTOMER PO# AND ADDED DESICCANT NOTES	DPM	DBR	DBR	SDM	05/30/12

KBR

ACCEPTANCE FOR ENGINEERING USE

THIS DOCUMENT IS:

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ACCEPTED WITH COMMENTS (Code 2) _____
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NOT REVIEWED (Code 4) _____

ACCEPTANCE DOES NOT RELIEVE SUPPLIER FROM FURNISHING MATERIAL IN CONFORMANCE WITH ORDER. REFER TO SDR-1 FOR FULL DEFINITION OF ACCEPTANCE CONDITIONS.

DISCIPLINE MJ BY TST
DATE 22 JUN 12

PROJECT - KEMPER COUNTY IGCC, UNIT 1
SERVICE - CO₂ DEHYDRATION UNIT
TAG No. - PG1080/PG2080
KBR REQUISITION No. - C452-MJ-TRQ-ML4-0001
SOUTHERN Co. PO No. - MPC10026889

PKG TAG No: PG1080/2080
VESSEL TAG No: PG1080-DRA/B
PG2080-DRA/B

ITEM 5

ITEM 1402

ITEM 1301

UNLESS OTHERWISE SPECIFIED:

DIMENSIONS ARE IN: INCHES (MM)

SURFACE FINISH: N/A

TOL. MM INCHES

X. .X .XX .XXX

FRACT: ± 1/16 ANG: ± 1/2"

SCALE: 1/25

NAME DATE

DWN BY DPM 08/01/11

ENG BY DLR 08/01/11

CHK BY WLB 08/01/11

APP BY DLR 08/01/11

REL BY SDM 08/09/11

MAT NO. 7405265

UNAUTHORIZED USE, MANUFACTURE, OR REPRODUCTION IN WHOLE OR IN PART IS PROHIBITED. DRAWING DESIGN AND OTHER DISCLOSURES ARE PROPERTY OF:

SPX FLOW TECHNOLOGY USA, INC.

ASSY, CHMBR

132" HP CNTRY, FLG, 350PSI @ 450F

SIZE D DWG. NO. 7405265

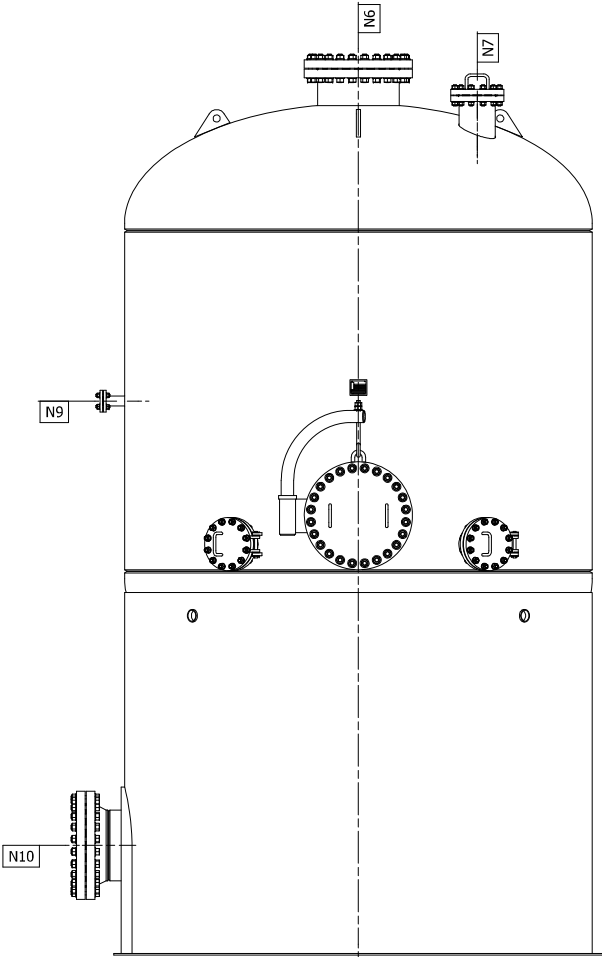
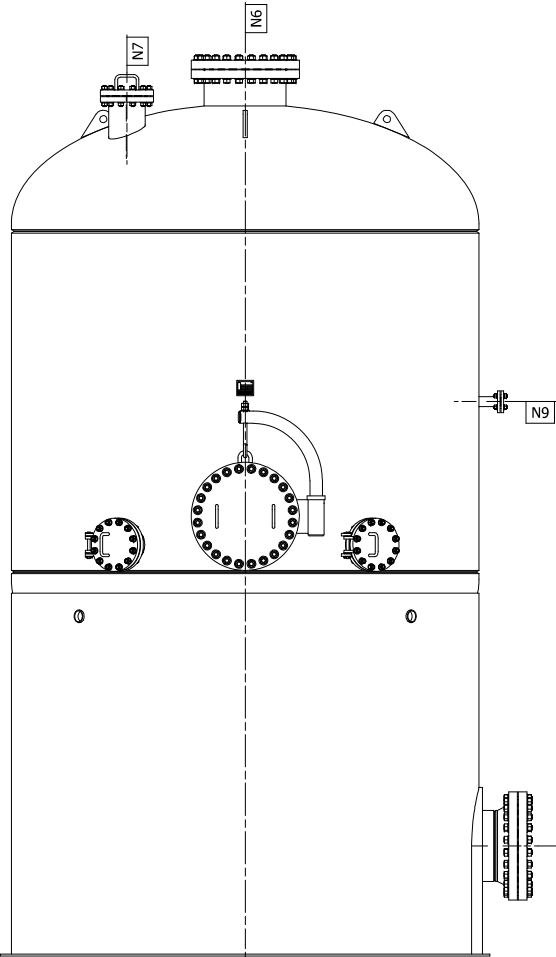
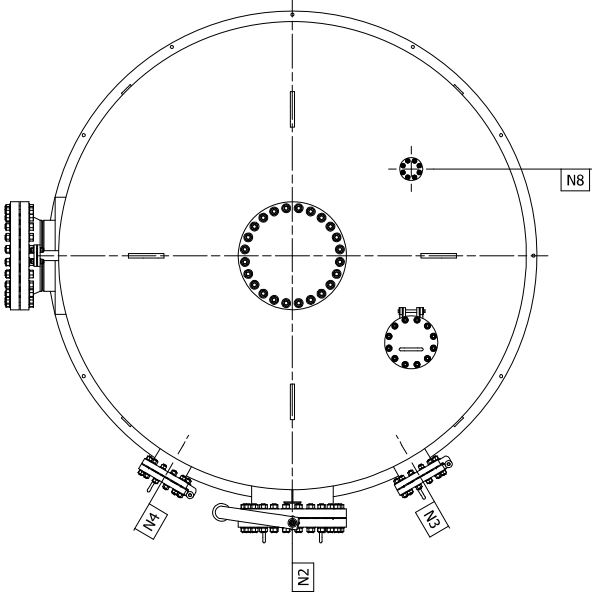
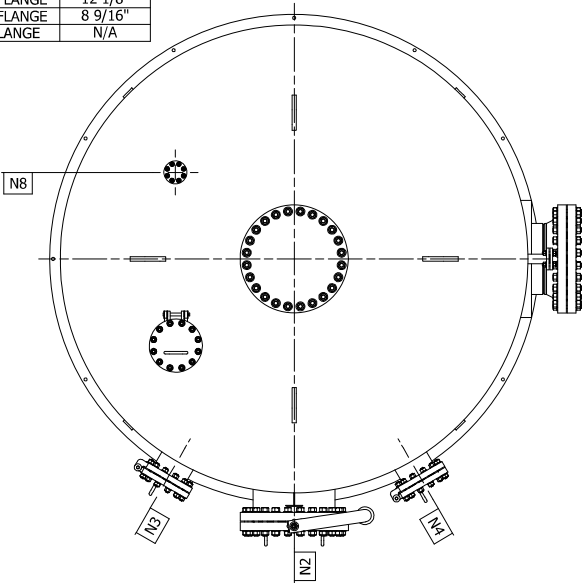
REV. 4

SHEET 5

Southern Company Generation Kemper County
MM84875 C Unit 1

SPX FLOW TECHNOLOGY PO: MPC10026889
7405265 SH 4 Rev: D
IGCC - GASIFIER - CO₂ DEHYDRATION UNIT - ASSEMBLY CHAMBER DRAWING

NOZZLE TABLE				
TAG	DESCRIPTION	SIZE	TYPE	MIN LENGTH
N2	MANWAY IN SHELL	20"	300# RF LWN FLANGE	13 1/2"
N3	DRAIN NOZZLE IN SHELL	8"	300# RF LWN FLANGE	7 13/16"
N4	DRAIN NOZZLE IN SHELL	8"	300# RF LWN FLANGE	7 13/16"
N6	INLET NOZZLE IN TOP HEAD	20"	300# RF LWN FLANGE	14 9/16"
N7	FILL NOZZLE IN TOP HEAD	8"	300# RF LWN FLANGE	16 5/16"
N8	RELIEF VALVE NOZZLE IN TOP HEAD	2"	300# RF LWN FLANGE	12 1/8"
N9	INSTRUMENT NOZZLE IN SHEET	1 1/2"	300# RF LWN FLANGE	8 9/16"
N10	WELD STUB OUTLET NOZZLE IN BOTTOM HEAD	20"	300# RF WN FLANGE	N/A



MAT# 7404131
TAG #'S: PG1080 DRA, PG2080 DRA

MAT# 7405265
TAG #'S: PG1080 DRB, PG2080 DRB

- REVISIONS -									
ZONE	REV	DESCRIPTION	CHG BY	CHK BY	APP BY	REL BY	DATE		
	A	UPDATED ALL WELD SYMBOLS PER FABRICATOR, UPDATED DRAWING PER CUTOMER AND FABRICATOR MARKUPS	DPM	DBR	DBR	SDM	10/14/11		
	B	CHANGED MIN LENGTH & EXTERNAL PROJECTION OF N8	DPM	DPM	DPM	SDM	03/02/12		
	C	CHANGED EXTERNAL PROJECTION OF N8	DPM	DPM	DPM	SDM	03/21/12		
	D	UPDATED CUSTOMER PO# AND BASE ANALYSIS TABLE	DPM	DBR	DBR	SDM	05/30/12		

KBR

**ACCEPTANCE
FOR ENGINEERING USE**

THIS DOCUMENT IS:

ACCEPTED (Code 1) X
ACCEPTED WITH COMMENTS (Code 2) _____
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NOT REVIEWED (Code 4) _____

ACCEPTANCE DOES NOT RELIEVE SUPPLIER
FROM FURNISHING MATERIAL IN CONFORMANCE
WITH ORDER. REFER TO SDR-1 FOR FULL
DEFINITION OF ACCEPTANCE CONDITIONS.

DISCIPLINE MJ BY TST
DATE 22 JUN 12

BASE ANALYSIS AND FOUNDATION LOADING				
LOAD	VESSEL CONDITION	BASE MOMENT (lb-ft)	W (lb)	SHEAR (lb)
WIND	OPERATING, CORRODED	236,165.7	151,412.4	9,464.0
WIND	OPERATING, NEW	236,165.8	155,963.5	9,464.0
WIND	EMPTY, CORRODED	236,165.7	150,488.6	9,464.0
WIND	EMPTY, NEW	236,165.8	155,039.7	9,464.0
SEISMIC	OPERATING, CORRODED	188,165.1	151,412.4	5,759.0
SEISMIC	OPERATING, NEW	193,701.1	155,963.5	5,932.0
SEISMIC	EMPTY, CORRODED	187,098.0	150,488.6	5,724.0
SEISMIC	EMPTY, NEW	192,634.1	155,039.7	5,899.0

PKG TAG No: PG1080/2080
VESSEL TAG No: PG1080-DRA/B
PG2080-DRA/B

PROJECT - KEMPER COUNTY IGCC, UNIT 1
SERVICE - CO2 DEHYDRATION UNIT
TAG No. - PG1080/PG2080
KBR REQUISITION No. - C452-MJ-TRQ-ML4-0001
SOUTHERN Co. PO No. - MPC10026889

UNLESS OTHERWISE SPECIFIED:		NAME	DATE	UNAUTHORIZED USE, MANUFACTURE, OR REPRODUCTION IN WHOLE OR IN PART IS PROHIBITED. DRAWING DESIGN AND OTHER DISCLOSURES ARE PROPERTY OF:	
DIMENSIONS ARE IN: INCHES [MM]		DWN BY	DPM	SPX FLOW TECHNOLOGY USA, INC.	
SURFACE FINISH: N/A		ENG BY	D13R		
TOL.	MM	CHK BY	WLB	ASSY, CHMBR	
.X		APP BY	DBR		
.XX		REL BY	SDM	132" HP CNTRY, FLG, 350PSI @ 450F	
.XXX					
FRACT: ± 1/16 ANG: ± 1/2"		MAT NO.	7405265	SIZE	DWG. NO.
SCALE: 1/25				D	7405265

Southern Company Generation Kemper County
MM92923 A Unit 1

SPX FLOW TECHNOLOGY PO: MPC10026889
7405265 SH 5 Rev: D
IGCC - GASIFIER - CO2 DEHYDRATION UNIT - ASSEMBLY CHAMBER DRAWING

LIFTING LUG MAXIMUM LOAD CALCULATION (VERTICAL MOUNTED LUGS)

S.O.: 1471495.1
LIFT LUG CAT.# Drawing Detail
VESSEL DWG No. 7405265
THICKNESS: 1.250 in 31.75 mm
MATERIAL: SA36

Vessel Tag No.: PG-1080-DR A/B
PG-2080-DR A/B

LUG TO SKID FILLET WELD LEG: 0.500 in 12.7 mm

YIELD STRENGTH VALUE: 30,000.00 psi 206,844.00 kpa
SHEAR STRESS AT YIELD: 15,000.00 psi 103,422.00 kpa
SAFETY FACTOR: 4
YIELD STRENGTH WITH S.F. APPLIED: 7,500.00 psi 51,711.00 kpa
MAX. SHEAR STRESS WITH S.F. APPLIED: 3,750.00 psi 25,855.50 kpa

DEFINITION OF SYMBOLS:	
Aw =>	TOTAL LENGTH OF WELD (INCHES)
F =>	ALLOWABLE LOAD ON WELD (LBS/IN2 OF WELD LEG AREA)
w =>	FILLET WELD LEG DIMENSION (IN)
Sh =>	MATERIAL SHEAR STRENGTH (LBS/IN ²)
d =>	MIN DISTANCE FROM LIFTING LUG SLOT TO OUTSIDE EDGE (IN)
t =>	MATERIAL THICKNESS (IN)
WI =>	MAXIMUM ALLOWABLE WELD LOAD (LBS)
SI =>	MAXIMUM ALLOWABLE SHEAR LOAD (LBS)

Aw: 24.00 in 609.60 mm
F: 3,750 psi 25,855.50 kpa
w: 0.5 in 12.70 mm
Sh: 3,750 psi 25,855.50 kpa

t (Actual): 1.25 in t (Min): 1.1111 in
t (Actual): 31.75 mm t (Min): 28.2222 mm

d: 1.5 in 38.10 mm

MAXIMUM ALLOWABLE WELD LOAD: WI = Aw*w*F 20,411.55 kg
WI = 45,000.00 lbs
MAXIMUM ALLOWABLE SHEAR LOAD: SI = d*t²*Sh 6,378.61 kg
SI = 14,062.50 lbs

MAXIMUM LOAD TO BE THE LESSER OF "WI" OR "SI"
MAX SINGLE LIFT LUG LOAD SHALL BE NO MORE THAN: 14,062.50 lbs 6,378.61 kg
MAXIMUM TOTAL LOAD TO BE LIFTED: 50,000.00 lbs

LIFT LUG QTY: 4
EACH LUG TO LIFT: 12500.00 lbs 5669.88 kg


BASED ON THE CALCULATIONS, THE REQUIRED THICKNESS THIS DESIGN EXCEEDS THE MINIMUM REQUIREMENT
PASSES

THE THICKNESS RATIO IS: 0.89

NOTE: The thickness ratio and reduction in minimum design metal temperature will allow the designer to rate specific materials at lower temperatures without the need for impact testing based on UCS-66, IMPACT TEST EXEMPTION CURVES, Section VIII, Div. 1, page 171

Created By: Daniel W. Brinton
Date: 6/17/2010

Project: Kemper County IGCC, UNIT 1
Service: CO2 Dehydration Package
Tag No.: PG1080 / PG2080
KBR Requisition No.: C452-MJ-TRQ-ML4-000
Southern Co. PO No.: MPC 10026889



**ACCEPTANCE
FOR ENGINEERING USE**

THIS DOCUMENT IS:

ACCEPTED	(Code 1) <u>X</u>
ACCEPTED WITH COMMENTS	(Code 2) <u> </u>
NOT ACCEPTED	(Code 3) <u> </u>
NOT REVIEWED	(Code 4) <u> </u>

ACCEPTANCE DOES NOT RELIEVE SUPPLIER FROM FURNISHING MATERIAL IN CONFORMANCE WITH ORDER. REFER TO SDR-1 FOR FULL DEFINITION OF ACCEPTANCE CONDITIONS.

DISCIPLINE MJ BY TST
DATE 28 DEC 11

Southern Company Generation Kemper County
MM84869 C Unit 1

SPX FLOW TECHNOLOGY PO: MPC10026889
1471495.1-1 Rev: NA
IGCC - GASIFIER - TRANSPORT AIR DRYERS - LIFTING LUG MAXIMUM LOAD

SPX Flow Technology USA, Inc.
4647 SW 40th Ave Ocala, Florida 34474

Date Printed: 5/16/2012

Project Description

Project: Kemper County IGCC, Unit 1
Service: CO2 Dehydration Unit
Tag No.: PG1080 / PG2080
KBR Requisition No.: C452-MJ-TRQ-ML4-0001
Southern Co. PO No.: MPC10026889

Vessel Tag no.: PG1080-DR A/B & PG2080-DR A/B

KBR

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NOT ACCEPTED	(Code 3) <u> </u>
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DISCIPLINE	<u>MJ</u>	BY	<u>TST</u>
		DATE	<u>11 JUN 12</u>

Southern Company Generation Kemper County
MM84871 B Unit 1

SPX FLOW TECHNOLOGY PO: MPC10026889
7404131-PG-1080 Rev: NA
IGCC - GASIFIER - MULTIPAGE - CO2 DEHYDRATION UNIT - ASSEMBLY CHA

SPX Flow Technology USA, Inc.

4647 SW 40th Ave Ocala, Florida 34474

Date Printed: 5/16/2012

CUSTOMER

Southern Company

Project- Kemper County IGCC, Unit 1
Birmingham, AL 35202

VESSEL LOCATION

Mississippi Power Company

Kemper County
, MS 39352

VESSEL DESCRIPTION

Service - CO2 Dehydration Units

Vessel designed per the ASME Boiler & Pressure Vessel Code,
Section VIII, Division 1, 2010 Edition
with DesignCalcs, Version: 2012.4
Vessel is ASME Code Stamped

Wind Analysis performed in accordance with ASCE 7-05
Seismic Analysis performed in accordance with ASCE 7-05

Job No: 7404131

Vessel Number: PG-1080 2080

Desiccant Chamber No.: PG1080-DR A/B & PG2080-DR A/B

NAMEPLATE INFORMATION

Vessel MAWP: 350.00 PSI at 450 °F

MDMT: 10 °F at 350.00 PSI

Serial Number(s): _____

National Board Number(s): _____

Year Built: 2011

Radiography: RT 1

Postweld Heat Treated: NONE

Construction Type: W

Notes

Project: Kemper County IGCC, Unit 1

Service - CO2 Dehydration Unit

Desiccant Chamber No.: PG1080-DRA/DRB & PG2080-DRA/DRB

Nameplates stamped with Vessel Tag Numbers and "WBS AREA 180"

Signatures

Designed by:: _____ Date: ____/____/____

Approved by:: _____ Date: ____/____/____

Table of Contents

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132" OD HEAD, 2:1 ELLIP, BOTTOM	2
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SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL	8
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SPX Flow Technology USA, Inc.

132" OD ROLLED SHELL x 96" LG

Customer: **Southern Company**

Job No: 7404131

Number: 1

Vessel Number: PG-1080 2080

Mark Number: S1

Date Printed: 5/16/2012

Cylindrical Shell Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	3.00 PSI	Long. Joint Efficiency:	100 %
Shell Material:	SA-516 Gr. 70	Factor B Chart:	CS-2
Material Condition:	Normalized	Material Stress (hot):	20000 PSI
Shell Length:	96.0000 in.	Material Stress (cold):	20000 PSI
		Compressive Stress:	13100 PSI
Corrosion Allowance:	0.0625 in.	Actual Circumferential Stress:	16066 PSI
External Corrosion Allowance:	0.0000 in.	Actual Longitudinal Stress:	7857 PSI
Outside Diameter (new):	132.0000 in.	Extreme Fiber Elongation:	1.15 %
Outside Diameter (corroded):	132.0000 in.	Specific Gravity:	1.00
Shell Surface Area:	276.46 Sq. Ft.	Weight of Fluid:	45381.19 lb.
Shell Estimated Volume:	5432.23 Gal.	Total Flooded Shell Weight:	61911.49 lb.
Circ. Joint Efficiency:	100 %	Shell Weight:	16530.31 lb.

Minimum Design Metal Temperature Data

Min. Temperature Curve:	D	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction:	Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction:	No	Computed Minimum Temperature:	-34 °F

Design Thickness Calculations**Longitudinal Stress Calculations per Paragraph UG-27(c)(2)**

$$t = \frac{PR}{2SE + 0.4P} = \frac{353.00 * 64.5625}{2 * 20000 * 1.00 + 0.4 * 353.00} = 0.5678 + 0.0625 \text{ (corrosion)} + 0.0000 \text{ (ext. corrosion)} = \text{minimum of } \mathbf{0.6303} \text{ in.}$$

Circumferential Stress Calculations per Appendix 1-1(a)(1)

$$t = \frac{PR_o}{SE + 0.4P} = \frac{353.00 * 66.0000}{20000 * 1.00 + 0.4 * 353.00} = 1.1568 + 0.0625 \text{ (corrosion)} + 0.0000 \text{ (ext. corrosion)} = \text{minimum of } \mathbf{1.2193} \text{ in.}$$

External loads do not control design.

Nominal Shell Thickness Selected = 1.5000 in.

SPX Flow Technology USA, Inc.

132" OD HEAD, 2:1 ELLIP, BOTTOM

Customer: **Southern Company**

Job No: 7404131

Number: 1

Vessel Number: PG-1080 2080

Mark Number: H1

Date Printed: 5/16/2012

Ellipsoidal Head Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Joint Efficiency:	100 %
Head Material:	SA-516 Gr. 70	Factor B Chart:	CS-2
Material Condition:	Normalized	Material Stress (hot):	20000 PSI
Corrosion Allowance:	0.0625 in.	Material Stress (cold):	20000 PSI
External Corrosion Allowance:	0.0000 in.	Actual Head Stress:	17285 PSI
Head Location:	Bottom	Straight Flange :	2.0000 in.
Outside Diameter :	132.0000 in.	Head Depth (ho) :	33.7500 in.
Thin Out :	0.1250 in.		

$$K = \frac{1}{6} [2 + (D/2h)^2] : 1.00$$

Extreme Fiber Elongation:	5.07 % (Hot Formed Head meets UCS-79(d) criteria)	Specific Gravity:	1.00
Head Surface Area:	134.60 Sq. Ft.	Weight of Fluid:	11090.24 lb.
Head Estimated Volume:	1329.77 Gal.	Total Flooded Head Weight:	19231.07 lb.
Head Weight:	8140.83 lb.		

Minimum Design Metal Temperature Data

Min. Temperature Curve:	D	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction:	Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction:	No	Computed Minimum Temperature:	-31 °F

Design Thickness Calculations**Design Thickness Calculations per Appendix 1-4(c)**

$$t = \frac{PD_oK}{2SE + 2P(K - 0.1)} = \frac{350.00 * 132.0000 * 1.00}{2 * 20000 * 1.00 + 2 * 350.00 * (1.00 - 0.1)}$$

$$= 1.1371 + 0.0625 \text{ (corrosion)} + 0.0000 \text{ (ext. corrosion)} + 0.1250 \text{ (thin out)} = \text{minimum of } \mathbf{1.3246} \text{ in.}$$

Nominal Head Thickness Selected = **1.5000** in.

SPX Flow Technology USA, Inc.

132" OD HEAD, 2:1 ELLIP, TOP

Customer: **Southern Company**

Job No: 7404131

Number: 2

Vessel Number: PG-1080 2080

Mark Number: H2

Date Printed: 5/16/2012

Ellipsoidal Head Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Joint Efficiency:	100 %
Head Material:	SA-516 Gr. 70	Factor B Chart:	CS-2
Material Condition:	Normalized	Material Stress (hot):	20000 PSI
Corrosion Allowance:	0.0625 in.	Material Stress (cold):	20000 PSI
External Corrosion Allowance:	0.0000 in.	Actual Head Stress:	17285 PSI
Head Location:	Top	Straight Flange :	2.0000 in.
Outside Diameter :	132.0000 in.	Head Depth (ho) :	33.7500 in.
Thin Out :	0.1250 in.		

$$K = \frac{1}{6} [2 + (D/2h)^2] : 1.00$$

Extreme Fiber Elongation:	5.07 % (Hot Formed Head meets UCS-79(d) criteria)	Specific Gravity:	1.00
Head Surface Area:	134.60 Sq. Ft.	Weight of Fluid:	11090.24 lb.
Head Estimated Volume:	1329.77 Gal.	Total Flooded Head Weight:	19231.07 lb.
Head Weight:	8140.83 lb.		

Minimum Design Metal Temperature Data

Min. Temperature Curve:	D	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction:	Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction:	No	Computed Minimum Temperature:	-31 °F

Design Thickness Calculations**Design Thickness Calculations per Appendix 1-4(c)**

$$t = \frac{PD_oK}{2SE + 2P(K - 0.1)} = \frac{350.00 * 132.0000 * 1.00}{2 * 20000 * 1.00 + 2 * 350.00 * (1.00 - 0.1)}$$

$$= 1.1371 + 0.0625 \text{ (corrosion)} + 0.0000 \text{ (ext. corrosion)} + 0.1250 \text{ (thin out)} = \text{minimum of } \mathbf{1.3246} \text{ in.}$$

Nominal Head Thickness Selected = **1.5000** in.

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 20" MANWAY IN SHELL

Customer: **Southern Company**
Job No: 7404131
Number: 2
ID Number: 2

Vessel Number: PG-1080 2080
Mark Number: N2

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	3.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	6.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	3.0000 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	20.0000 in.	Nozzle Wall Thickness(new):	1.5600 in.
Nozzle ID (corroded):	20.1250 in.	Nozzle Wall Thickness(corroded):	1.4975 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve: B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction: Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction: No	Computed Minimum Temperature:	-89 °F

Host Component: Shell 1 - 132" OD ROLLED SHELL x 96" LG

Material: SA-516 Gr. 70	Shell wall thickness(new):	1.5000 in.
Material Stress(S _v): 20000 PSI	Shell wall thickness(corroded):	1.4375 in.

Nozzle Detail Information

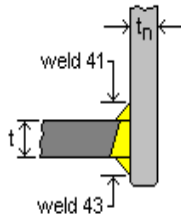


Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 1.5600 in.

Outside Groove Weld Depth: 1.5000 in.

Nozzle passes through the vessel, attached by a groove weld.
Nozzle is adequate for UG-45 requirements.
Opening is adequately reinforced for Internal Pressure.
Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 20" MANWAY IN SHELL

Job No: 7404131
Number: 2
ID Number: 2

Vessel Number: PG-1080 2080
Mark Number: N2

Date Printed: 5/16/2012

Required Shell Thickness per Paragraph UG-37(a)

$$t_r = \frac{P R_o}{S E + 0.4 P} = \frac{353.00 * 66.0000}{20000 * 1 + 0.4 * 353.00} = \mathbf{1.1567 \text{ in.}}$$

Nozzle Required Thickness Calculations

Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)

$$t_{rn} = \frac{P R_n}{S E - 0.6 P} = \frac{353.00 * 10.0625}{19800 * 1 - 0.6 * 353.00} = \mathbf{0.1813 \text{ in.}}$$

Strength Reduction Factors

$$f_{r1} = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900 \quad f_{r2} = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900$$

UG-45 Thickness Calculations

Nozzle Thickness for Pressure Loading (plus corrosion)

$$t_a = \frac{P R_n}{S E - 0.6 P} + C_a + \text{ext. Ca} = \frac{353.00 * 10.0625}{19800 * 1.00 - 0.6 * 353.00} + 0.0625 + 0.0000 = \mathbf{0.2438 \text{ in.}}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P R_o}{S E + 0.4 P} + C_a + \text{ext. Ca} = \frac{353.00 * 66.0000}{20000 * 1 + 0.4 * 353.00} + 0.0625 + 0.0000 = \mathbf{1.2192 \text{ in.}}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + C_a + \text{ext. Ca} = \mathbf{0.3905 \text{ in.}}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = \mathbf{0.3905 \text{ in.}}$$

$$t_{UG-45} = \max(t_a, t_b) = \mathbf{0.3905 \text{ in.}}$$

Wall thickness = $t_n = \mathbf{1.5600}$ is greater than or equal to UG-45 value of $\mathbf{0.3905}$

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 20" MANWAY IN SHELL

Job No: 7404131
 Number: 2
 ID Number: 2

Vessel Number: PG-1080 2080
 Mark Number: N2

Date Printed: 5/16/2012

Limits of Reinforcement (UG-40)

$$\begin{aligned} L_{par} &= \max(d, R_n + t + t_n) = \max(20.1250, 10.0625 + 1.4375 + 1.4975) &= \mathbf{20.1250} \text{ in.} \\ L_{noro} &= \min(2.5 t, 2.5 t_n + t_e) = \min(2.5 * 1.4375, 2.5 * 1.4975 + 0.0000) &= \mathbf{3.5938} \text{ in.} \\ L_{nori} &= \min(2.5 t, 2.5 t_i) = \min(2.5 * 1.4375, 2.5 * 1.4350) &= \mathbf{3.5875} \text{ in.} \end{aligned}$$

Nozzle Reinforcement Calculations (Internal Pressure)

$$A = \max\{C [d t_r F + 2 t_n t_r F (1 - f_{r1})], 0\} = \max\{1.0000 * [20.1250 * 1.1567 * 1.00 + 2 * 1.4975 * 1.1567 * 1.00 * (1 - 0.9900)], 0\} \\ = \mathbf{23.3132} \text{ sq. in.}$$

$$\begin{aligned} A1 &= \max\{2 L_{par} - d\} (E_1 t - F t_r) - 2 t_n (E_1 t - F t_r) (1 - f_{r1}), 0\} = \\ &\max\{2 * 20.1250 - 20.1250\} * (1.0000 * 1.4375 - 1.00 * 1.1567) - 2 * 1.4975 * (1.0000 * 1.4375 - 1.00 * 1.1567) * (1 - 0.9900), 0\} \\ &= \mathbf{5.6427} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A2 &= \max\{2 \min(h_o, L_{noro}) [\min(t_n, L_{par} - 0.5 d) - t_n] f_{r2}, 0\} = \\ &\max\{2 * \min(6.0000, 3.5938) * [\min(1.4975, 20.1250 - 0.5 * 20.1250) - 0.1813] * 0.9900, 0\} \\ &= \mathbf{9.3656} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A3 &= \max\{2 \min(h, L_{nori}) \min(t_i, L_{par} - 0.5 d) f_{r2}, 0\} = \\ &\max\{2 * \min(2.9375, 3.5875) * \min(1.4350, 20.1250 - 0.5 * 20.1250) * 0.9900, 0\} \\ &= \mathbf{8.3463} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A41 &= f_{r2} [L_{41}^2 - (L_{41} - L_{41pareff})^2 - (L_{41} - L_{41noreff})^2] = \\ &0.9900 * [0.3750^2 - (0.3750 - 0.3750)^2 - (0.3750 - 0.3750)^2] \\ &= \mathbf{0.1392} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A42 &= f_{r4} L_{42pareff} L_{42noreff} = \\ &0.0000 * 0.0000 * 0.0000 \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A43 &= f_{r2} [L_{43}^2 - (L_{43} - L_{43pareff})^2 - (L_{43} - L_{43noreff})^2] = \\ &0.9900 * [0.4107^2 - (0.4107 - 0.4107)^2 - (0.4107 - 0.4107)^2] \\ &= \mathbf{0.1670} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A5 &= \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\text{Area Available (Internal Pressure)} = A1 + A2 + A3 + A41 + A42 + A43 + A5 = 23.6608 \text{ sq. in., which is } \geq A (23.3132)$$

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 20" MANWAY IN SHELL

Job No: 7404131
 Number: 2
 ID Number: 2

Vessel Number: PG-1080 2080
 Mark Number: N2

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations

Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 1.4975 = **0.7500 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7}$ = **0.3571 in.**

Weld 41, actual weld leg = **0.3750 in.**

Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 1.4975 = **0.7500 in.**

Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ca}}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.7500 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7}$ = **0.4464 in.**

Weld 43, actual weld leg = **0.5000 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**

Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**

Inner fillet, Weld 43, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 21.6225 * 1.4975 * 13860$ = **704600 lb.**

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 23.1200 * 0.3750 * 9702$ = **132100 lb.**

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 23.1200 * 1.5000 * 14652$ = **797800 lb.**

Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 23.1200 * 0.4107 * 9702$ = **144600 lb.**

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [23.3132 - 5.6427 + 2 * 1.4975 * 0.9900 * (1.00 * 1.4375 - 1.00 * 1.1567)] * 20000 = **370100 lb.**

W1-1 = (A2 + A5 + A41 + A42) * Sv = (9.3656 + 0.0000 + 0.1392 + 0.0000) * 20000 = **190100 lb.**

W2-2 = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (9.3656 + 8.3463 + 0.1392 + 0.1670 + 2 * 1.4975 * 1.4375 * 0.9900) * 20000 = **445600 lb.**

W3-3 = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv = (9.3656 + 8.3463 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 1.4975 * 1.4375 * 0.9900) * 20000 = **445600 lb.**

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = 132100 + 704600 = **836700 lb.**

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = 132100 + 797800 + 144600 = **1074500 lb.**

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 132100 + 144600 + 797800 = **1074500 lb.**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Customer: **Southern Company**

Job No: 7404131

Number: 3

ID Number: 4

Vessel Number: PG-1080 2080

Mark Number: N3

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	3.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	5.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	1.0000 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	8.0000 in.	Nozzle Wall Thickness(new):	1.1250 in.
Nozzle ID (corroded):	8.1250 in.	Nozzle Wall Thickness(corroded):	1.0625 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve: B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction: Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction: No	Computed Minimum Temperature:	-103 °F

Host Component: Shell 1 - 132" OD ROLLED SHELL x 96" LG

Material: SA-516 Gr. 70	Shell wall thickness(new):	1.5000 in.
Material Stress(S _v): 20000 PSI	Shell wall thickness(corroded):	1.4375 in.

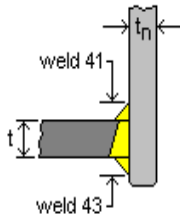
Nozzle Detail Information

Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 1.1250 in.

Outside Groove Weld Depth: 1.5000 in.

Nozzle passes through the vessel, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Job No: 7404131
 Number: 3
 ID Number: 4

Vessel Number: PG-1080 2080
 Mark Number: N3

Date Printed: 5/16/2012

Required Shell Thickness per Paragraph UG-37(a)

$$t_r = \frac{P R_o}{S E + 0.4 P} = \frac{353.00 * 66.0000}{20000 * 1 + 0.4 * 353.00} = 1.1567 \text{ in.}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$t_{rn} = \frac{P R_n}{S E - 0.6 P} = \frac{353.00 * 4.0625}{19800 * 1 - 0.6 * 353.00} = 0.0732 \text{ in.}$$

Strength Reduction Factors

$$f r_1 = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900 \quad f r_2 = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{P R_n}{S E - 0.6 P} + C_a + \text{ext. Ca} = \frac{353.00 * 4.0625}{19800 * 1.00 - 0.6 * 353.00} + 0.0625 + 0.0000 = 0.1357 \text{ in.}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P R_o}{S E + 0.4 P} + C_a + \text{ext. Ca} = \frac{353.00 * 66.0000}{20000 * 1 + 0.4 * 353.00} + 0.0625 + 0.0000 = 1.2192 \text{ in.}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + C_a + \text{ext. Ca} = 0.3815 \text{ in.}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = 0.3815 \text{ in.}$$

$$t_{UG-45} = \max(t_a, t_b) = 0.3815 \text{ in.}$$

Wall thickness = $t_n = 1.1250$ is greater than or equal to UG-45 value of **0.3815**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Job No: 7404131
 Number: 3
 ID Number: 4

Vessel Number: PG-1080 2080
 Mark Number: N3

Date Printed: 5/16/2012

Limits of Reinforcement (UG-40)

$$\begin{aligned}
 L_{par} &= \max(d, R_n + t + t_n) = \max(8.1250, 4.0625 + 1.4375 + 1.0625) &= \mathbf{8.1250} \text{ in.} \\
 L_{noro} &= \min(2.5 t, 2.5 t_n + t_e) = \min(2.5 * 1.4375, 2.5 * 1.0625 + 0.0000) &= \mathbf{2.6563} \text{ in.} \\
 L_{nori} &= \min(2.5 t, 2.5 t_i) = \min(2.5 * 1.4375, 2.5 * 1.0000) &= \mathbf{2.5000} \text{ in.}
 \end{aligned}$$

Nozzle Reinforcement Calculations (Internal Pressure)

$$\mathbf{A} = \max\{C [d t_r F + 2 t_n t_r F (1 - f_{r1})], 0\} = \max\{1.0000 * [8.1250 * 1.1567 * 1.00 + 2 * 1.0625 * 1.1567 * 1.00 * (1 - 0.9900)], 0\} = \mathbf{9.4228} \text{ sq. in.}$$

$$\begin{aligned}
 \mathbf{A1} &= \max\{2 L_{par} - d\} (E_1 t - F t_r) - 2 t_n (E_1 t - F t_r) (1 - f_{r1}), 0\} = \\
 &\max\{(2 * 8.1250 - 8.1250) * (1.0000 * 1.4375 - 1.00 * 1.1567) - 2 * 1.0625 * (1.0000 * 1.4375 - 1.00 * 1.1567) * (1 - 0.9900), 0\} \\
 &= \mathbf{2.2755} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 \mathbf{A2} &= \max\{2 \min(h_o, L_{noro}) [\min(t_n, L_{par} - 0.5 d) - t_n] f_{r2}, 0\} = \\
 &\max\{2 * \min(5.0000, 2.6563) * [\min(1.0625, 8.1250 - 0.5 * 8.1250) - 0.0732] * 0.9900, 0\} \\
 &= \mathbf{5.2031} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 \mathbf{A3} &= \max\{2 \min(h, L_{nori}) \min(t_i, L_{par} - 0.5 d) f_{r2}, 0\} = \\
 &\max\{2 * \min(0.9375, 2.5000) * \min(1.0000, 8.1250 - 0.5 * 8.1250) * 0.9900, 0\} \\
 &= \mathbf{1.8563} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 \mathbf{A41} &= f_{r2} [L_{41}^2 - (L_{41} - L_{41pareff})^2 - (L_{41} - L_{41noreff})^2] = \\
 &0.9900 * [0.3750^2 - (0.3750 - 0.3750)^2 - (0.3750 - 0.3750)^2] \\
 &= \mathbf{0.1392} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 \mathbf{A42} &= f_{r4} L_{42pareff} L_{42noreff} = \\
 &0.0000 * 0.0000 * 0.0000 \\
 &= \mathbf{0.0000} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 \mathbf{A43} &= f_{r2} [L_{43}^2 - (L_{43} - L_{43pareff})^2 - (L_{43} - L_{43noreff})^2] = \\
 &0.9900 * [0.4107^2 - (0.4107 - 0.4107)^2 - (0.4107 - 0.4107)^2] \\
 &= \mathbf{0.1670} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 \mathbf{A5} &= \\
 &= \mathbf{0.0000} \text{ sq. in.}
 \end{aligned}$$

Area Available (Internal Pressure) = A1 + A2 + A3 + A41 + A42 + A43 + A5 = 9.6411 sq. in., which is \geq A (9.4228)

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Job No: 7404131
 Number: 3
 ID Number: 4

Vessel Number: PG-1080 2080
 Mark Number: N3

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations

Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 1.0625 = **0.7500 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7}$ = **0.3571 in.**

Weld 41, actual weld leg = **0.3750 in.**

Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 1.0625 = **0.7500 in.**

Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + c_a}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.7500 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7}$ = **0.4464 in.**

Weld 43, actual weld leg = **0.5000 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**

Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**

Inner fillet, Weld 43, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 9.1875 * 1.0625 * 13860$ = **212400 lb.**

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 10.2500 * 0.3750 * 9702$ = **58500 lb.**

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 10.2500 * 1.5000 * 14652$ = **353700 lb.**

Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 10.2500 * 0.4107 * 9702$ = **64100 lb.**

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [9.4228 - 2.2755 + 2 * 1.0625 * 0.9900 * (1.00 * 1.4375 - 1.00 * 1.1567)] * 20000 = **154800 lb.**

W1-1 = (A2 + A5 + A41 + A42) * Sv = (5.2031 + 0.0000 + 0.1392 + 0.0000) * 20000 = **106800 lb.**

W2-2 = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (5.2031 + 1.8563 + 0.1392 + 0.1670 + 2 * 1.0625 * 1.4375 * 0.9900) * 20000 = **207800 lb.**

W3-3 = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv = (5.2031 + 1.8563 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 1.0625 * 1.4375 * 0.9900) * 20000 = **207800 lb.**

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = 58500 + 212400 = **270900 lb.**

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = 58500 + 353700 + 64100 = **476300 lb.**

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 58500 + 64100 + 353700 = **476300 lb.**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Customer: **Southern Company**

Job No: 7404131

Number: 4

ID Number: 4

Vessel Number: PG-1080 2080

Mark Number: N4

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	3.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	6.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	1.0000 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	8.0000 in.	Nozzle Wall Thickness(new):	1.1300 in.
Nozzle ID (corroded):	8.1250 in.	Nozzle Wall Thickness(corroded):	1.0675 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve: B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction: Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction: No	Computed Minimum Temperature:	-103 °F

Host Component: Shell 1 - 132" OD ROLLED SHELL x 96" LG

Material: SA-516 Gr. 70	Shell wall thickness(new):	1.5000 in.
Material Stress(S _v): 20000 PSI	Shell wall thickness(corroded):	1.4375 in.

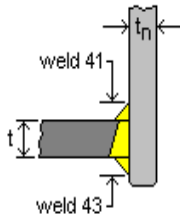
Nozzle Detail Information

Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 1.1300 in.

Outside Groove Weld Depth: 1.5000 in.

Nozzle passes through the vessel, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Job No: 7404131
 Number: 4
 ID Number: 4

Vessel Number: PG-1080 2080
 Mark Number: N4

Date Printed: 5/16/2012

Required Shell Thickness per Paragraph UG-37(a)

$$t_r = \frac{P R_o}{S E + 0.4 P} = \frac{353.00 * 66.0000}{20000 * 1 + 0.4 * 353.00} = 1.1567 \text{ in.}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$t_{rn} = \frac{P R_n}{S E - 0.6 P} = \frac{353.00 * 4.0625}{19800 * 1 - 0.6 * 353.00} = 0.0732 \text{ in.}$$

Strength Reduction Factors

$$f r_1 = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900 \quad f r_2 = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{P R_n}{S E - 0.6 P} + C_a + \text{ext. Ca} = \frac{353.00 * 4.0625}{19800 * 1.00 - 0.6 * 353.00} + 0.0625 + 0.0000 = 0.1357 \text{ in.}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P R_o}{S E + 0.4 P} + C_a + \text{ext. Ca} = \frac{353.00 * 66.0000}{20000 * 1 + 0.4 * 353.00} + 0.0625 + 0.0000 = 1.2192 \text{ in.}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + C_a + \text{ext. Ca} = 0.3815 \text{ in.}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = 0.3815 \text{ in.}$$

$$t_{UG-45} = \max(t_a, t_b) = 0.3815 \text{ in.}$$

Wall thickness = $t_n = 1.1300$ is greater than or equal to UG-45 value of **0.3815**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Job No: 7404131
 Number: 4
 ID Number: 4

Vessel Number: PG-1080 2080
 Mark Number: N4

Date Printed: 5/16/2012

Limits of Reinforcement (UG-40)

$$\begin{aligned}
 L_{par} &= \max(d, R_n + t + t_n) = \max(8.1250, 4.0625 + 1.4375 + 1.0675) &= \mathbf{8.1250} \text{ in.} \\
 L_{noro} &= \min(2.5 t, 2.5 t_n + t_e) = \min(2.5 * 1.4375, 2.5 * 1.0675 + 0.0000) &= \mathbf{2.6688} \text{ in.} \\
 L_{nori} &= \min(2.5 t, 2.5 t_i) = \min(2.5 * 1.4375, 2.5 * 1.0050) &= \mathbf{2.5125} \text{ in.}
 \end{aligned}$$

Nozzle Reinforcement Calculations (Internal Pressure)

$$\begin{aligned}
 A &= \max\{C [d t_r F + 2 t_n t_r F (1 - f_{r1})], 0\} = \max\{1.0000 * [8.1250 * 1.1567 * 1.00 + 2 * 1.0675 * 1.1567 * 1.00 * (1 - 0.9900)], 0\} \\
 &= \mathbf{9.4229} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A1 &= \max\{(2 L_{par} - d) (E_1 t - F t_r) - 2 t_n (E_1 t - F t_r) (1 - f_{r1}), 0\} = \\
 &\max\{(2 * 8.1250 - 8.1250) * (1.0000 * 1.4375 - 1.00 * 1.1567) - 2 * 1.0675 * (1.0000 * 1.4375 - 1.00 * 1.1567) * (1 - 0.9900), 0\} \\
 &= \mathbf{2.2755} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A2 &= \max\{2 \min(h_o, L_{noro}) [\min(t_n, L_{par} - 0.5 d) - t_n] f_{r2}, 0\} = \\
 &\max\{2 * \min(6.0000, 2.6688) * [\min(1.0675, 8.1250 - 0.5 * 8.1250) - 0.0732] * 0.9900, 0\} \\
 &= \mathbf{5.2540} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A3 &= \max\{2 \min(h, L_{nori}) \min(t_i, L_{par} - 0.5 d) f_{r2}, 0\} = \\
 &\max\{2 * \min(0.9375, 2.5125) * \min(1.0050, 8.1250 - 0.5 * 8.1250) * 0.9900, 0\} \\
 &= \mathbf{1.8655} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A41 &= f_{r2} [L_{41}^2 - (L_{41} - L_{41pareff})^2 - (L_{41} - L_{41noreff})^2] = \\
 &0.9900 * [0.3750^2 - (0.3750 - 0.3750)^2 - (0.3750 - 0.3750)^2] \\
 &= \mathbf{0.1392} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A42 &= f_{r4} L_{42pareff} L_{42noreff} = \\
 &0.0000 * 0.0000 * 0.0000 \\
 &= \mathbf{0.0000} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A43 &= f_{r2} [L_{43}^2 - (L_{43} - L_{43pareff})^2 - (L_{43} - L_{43noreff})^2] = \\
 &0.9900 * [0.4107^2 - (0.4107 - 0.4107)^2 - (0.4107 - 0.4107)^2] \\
 &= \mathbf{0.1670} \text{ sq. in.}
 \end{aligned}$$

$$\begin{aligned}
 A5 &= \\
 &= \mathbf{0.0000} \text{ sq. in.}
 \end{aligned}$$

Area Available (Internal Pressure) = A1 + A2 + A3 + A41 + A42 + A43 + A5 = 9.7013 sq. in., which is \geq A (9.4229)

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 8" DES DRAIN NOZZLE IN SHELL

Job No: 7404131
 Number: 4
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Vessel Number: PG-1080 2080
 Mark Number: N4

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations

Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 1.0675 = **0.7500 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7}$ = **0.3571 in.**

Weld 41, actual weld leg = **0.3750 in.**

Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 1.0675 = **0.7500 in.**

Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ca}}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.7500 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7}$ = **0.4464 in.**

Weld 43, actual weld leg = **0.5000 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**

Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**

Inner fillet, Weld 43, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 9.1925 * 1.0675 * 13860$ = **213500 lb.**

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 10.2600 * 0.3750 * 9702$ = **58600 lb.**

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 10.2600 * 1.5000 * 14652$ = **354000 lb.**

Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 10.2600 * 0.4107 * 9702$ = **64200 lb.**

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [9.4229 - 2.2755 + 2 * 1.0675 * 0.9900 * (1.00 * 1.4375 - 1.00 * 1.1567)] * 20000 = **154800 lb.**

W1-1 = (A2 + A5 + A41 + A42) * Sv = (5.2540 + 0.0000 + 0.1392 + 0.0000) * 20000 = **107900 lb.**

W2-2 = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (5.2540 + 1.8655 + 0.1392 + 0.1670 + 2 * 1.0675 * 1.4375 * 0.9900) * 20000 = **209300 lb.**

W3-3 = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv = (5.2540 + 1.8655 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 1.0675 * 1.4375 * 0.9900) * 20000 = **209300 lb.**

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = 58600 + 213500 = **272100 lb.**

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = 58600 + 354000 + 64200 = **476800 lb.**

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 58600 + 64200 + 354000 = **476800 lb.**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 20" INLET NOZZLE IN TOP HEAD

Customer: **Southern Company**

Job No: 7404131

Number: 6

ID Number: 6

Vessel Number: PG-1080 2080

Mark Number: N6

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	6.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	2.7500 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	20.0000 in.	Nozzle Wall Thickness(new):	1.5626 in.
Nozzle ID (corroded):	20.1250 in.	Nozzle Wall Thickness(corroded):	1.5001 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve: B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction: Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction: No	Computed Minimum Temperature:	-89 °F

Host Component: Head 2 - 132" OD HEAD, 2:1 ELLIP, TOP

Material:	SA-516 Gr. 70	Head wall thickness(new):	1.5000 in.
Material Stress(S _v):	20000 PSI	Head wall thickness - thin out (corroded):	1.3125 in.

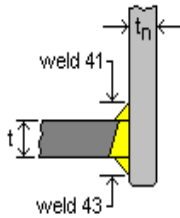
Nozzle Detail Information

Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 1.5626 in.

Outside Groove Weld Depth: 1.5000 in.

Nozzle passes through the vessel, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 20" INLET NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 6
 ID Number: 6

Vessel Number: PG-1080 2080
 Mark Number: N6

Date Printed: 5/16/2012

Required Head Thickness per Paragraph UG-37(a)

$$tr = \frac{P K_1 D_o}{(2SE + 0.8P)} = \frac{350.00 * 0.9000 * 132.0000}{(2 * 20000 * 1 + 0.8 * 350.00)} = 1.0323 \text{ in.}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$trn = \frac{PRn}{SE - 0.6P} = \frac{350.00 * 10.0625}{19800 * 1 - 0.6 * 350.00} = 0.1798 \text{ in.}$$

Strength Reduction Factors

$$fr1 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900 \quad fr2 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{PRn}{SE - 0.6P} + Ca + \text{ext. Ca} = \frac{350.00 * 10.0625}{19800 * 1.00 - 0.6 * 350.00} + 0.0625 + 0.0000 = 0.2423 \text{ in.}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P K D_o}{(2SE + 2P(K - 0.1))} + Ca + \text{ext. Ca} = \frac{350.00 * 1.0000 * 132.0000}{(2 * 20000 * 1 + 2 * 350.00 * (1.0000 - 0.1))} + 0.0625 + 0.0000 = 1.1996 \text{ in.}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + Ca + \text{ext. Ca} = 0.3905 \text{ in.}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = 0.3905 \text{ in.}$$

$$t_{UG-45} = \max(t_a, t_b) = 0.3905 \text{ in.}$$

Wall thickness = $t_n = 1.5626$ is greater than or equal to UG-45 value of **0.3905**

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 20" INLET NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 6
 ID Number: 6

Vessel Number: PG-1080 2080
 Mark Number: N6

Date Printed: 5/16/2012

Limits of Reinforcement (UG-40)

$$\begin{aligned} L_{par} &= \max(d, R_n + t + t_n) = \max(20.1250, 10.0625 + 1.3125 + 1.5001) &= \mathbf{20.1250} \text{ in.} \\ L_{noro} &= \min(2.5 t, 2.5 t_n + t_e) = \min(2.5 * 1.3125, 2.5 * 1.5001 + 0.0000) &= \mathbf{3.2813} \text{ in.} \\ L_{nori} &= \min(2.5 t, 2.5 t_i) = \min(2.5 * 1.3125, 2.5 * 1.4376) &= \mathbf{3.2813} \text{ in.} \end{aligned}$$

Nozzle Reinforcement Calculations (Internal Pressure)

$$A = \max\{C [d t_r F + 2 t_n t_r F (1 - f_{r1})], 0\} = \max\{1.0000 * [20.1250 * 1.0323 * 1.00 + 2 * 1.5001 * 1.0323 * 1.00 * (1 - 0.9900)], 0\} \\ = \mathbf{20.8060} \text{ sq. in.}$$

$$\begin{aligned} A1 &= \max\{(2 L_{par} - d) (E_1 t - F t_r) - 2 t_n (E_1 t - F t_r) (1 - f_{r1}), 0\} = \\ &\max\{(2 * 20.1250 - 20.1250) * (1.0000 * 1.3125 - 1.00 * 1.0323) - 2 * 1.5001 * (1.0000 * 1.3125 - 1.00 * 1.0323) * (1 - 0.9900), 0\} \\ &= \mathbf{5.6306} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A2 &= \max\{2 \min(h_o, L_{noro}) [\min(t_n, L_{par} - 0.5 d) - t_n] f_{r2}, 0\} = \\ &\max\{2 * \min(6.0000, 3.2813) * [\min(1.5001, 20.1250 - 0.5 * 20.1250) - 0.1798] * 0.9900, 0\} \\ &= \mathbf{8.5778} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A3 &= \max\{2 \min(h, L_{nori}) \min(t_i, L_{par} - 0.5 d) f_{r2}, 0\} = \\ &\max\{2 * \min(2.6875, 3.2813) * \min(1.4376, 20.1250 - 0.5 * 20.1250) * 0.9900, 0\} \\ &= \mathbf{7.6498} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A41 &= f_{r2} [L_{41}^2 - (L_{41} - L_{41pareff})^2 - (L_{41} - L_{41noreff})^2] = \\ &0.9900 * [0.3750^2 - (0.3750 - 0.3750)^2 - (0.3750 - 0.3750)^2] \\ &= \mathbf{0.1392} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A42 &= f_{r4} L_{42pareff} L_{42noreff} = \\ &0.0000 * 0.0000 * 0.0000 \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A43 &= f_{r2} [L_{43}^2 - (L_{43} - L_{43pareff})^2 - (L_{43} - L_{43noreff})^2] = \\ &0.9900 * [0.4107^2 - (0.4107 - 0.4107)^2 - (0.4107 - 0.4107)^2] \\ &= \mathbf{0.1670} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A5 &= \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\text{Area Available (Internal Pressure)} = A1 + A2 + A3 + A41 + A42 + A43 + A5 = 22.1645 \text{ sq. in., which is } \geq A (20.8060)$$

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 20" INLET NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 6
 ID Number: 6

Vessel Number: PG-1080 2080
 Mark Number: N6

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations

Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 1.5001 = **0.7500 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7}$ = **0.3571 in.**

Weld 41, actual weld leg = **0.3750 in.**

Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 1.5001 = **0.7500 in.**

Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ca}}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.7500 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7}$ = **0.4464 in.**

Weld 43, actual weld leg = **0.5000 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**

Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**

Inner fillet, Weld 43, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 21.6251 * 1.5001 * 13860$ = **705900 lb.**

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 23.1252 * 0.3750 * 9702$ = **132100 lb.**

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 23.1252 * 1.5000 * 14652$ = **797900 lb.**

Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 23.1252 * 0.4107 * 9702$ = **144700 lb.**

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [20.8060 - 5.6306 + 2 * 1.5001 * 0.9900 * (1.00 * 1.3125 - 1.00 * 1.0323)] * 20000 = **320200 lb.**

W1-1 = (A2 + A5 + A41 + A42) * Sv = (8.5778 + 0.0000 + 0.1392 + 0.0000) * 20000 = **174300 lb.**

W2-2 = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (8.5778 + 7.6498 + 0.1392 + 0.1670 + 2 * 1.5001 * 1.3125 * 0.9900) * 20000 = **408600 lb.**

W3-3 = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv = (8.5778 + 7.6498 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 1.5001 * 1.3125 * 0.9900) * 20000 = **408600 lb.**

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = 132100 + 705900 = **838000 lb.**

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = 132100 + 797900 + 144700 = **1074700 lb.**

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 132100 + 144700 + 797900 = **1074700 lb.**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES FILL NOZZLE IN TOP HEAD

Customer: **Southern Company**

Job No: 7404131

Number: 7

ID Number: 7

Vessel Number: PG-1080 2080

Mark Number: N7

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	6.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	2.5625 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	8.0000 in.	Nozzle Wall Thickness(new):	1.1300 in.
Nozzle ID (corroded):	8.1250 in.	Nozzle Wall Thickness(corroded):	1.0675 in.
Developed Opening:	8.5196 in.	Tangential Dimension L:	38.0000 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve:	B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction:	Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction:	No	Computed Minimum Temperature:	-103 °F

Host Component: Head 2 - 132" OD HEAD, 2:1 ELLIP, TOP

Material:	SA-516 Gr. 70	Head wall thickness(new):	1.5000 in.
Material Stress(S _v):	20000 PSI	Head wall thickness - thin out (corroded):	1.3125 in.

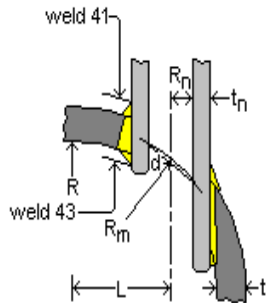
Nozzle Detail Information

Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 1.1300 in.

Outside Groove Weld Depth: 1.5000 in.

tangential to the vessel wall, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 8" DES FILL NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 7
 ID Number: 7

Vessel Number: PG-1080 2080
 Mark Number: N7

Date Printed: 5/16/2012

Required Head Thickness per Paragraph UG-37(a)

$$tr = \frac{P K_1 D_o}{(2SE + 0.8P)} = \frac{350.00 * 0.9000 * 132.0000}{(2 * 20000 * 1 + 0.8 * 350.00)} = \mathbf{1.0323 \text{ in.}}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$trn = \frac{PRn}{SE - 0.6P} = \frac{350.00 * 4.0625}{19800 * 1 - 0.6 * 350.00} = \mathbf{0.0726 \text{ in.}}$$

Strength Reduction Factors

$$fr1 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900 \quad fr2 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{PRn}{SE - 0.6P} + Ca + \text{ext. Ca} = \frac{350.00 * 4.0625}{19800 * 1.00 - 0.6 * 350.00} + 0.0625 + 0.0000 = \mathbf{0.1351 \text{ in.}}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P K D_o}{(2SE + 2P(K - 0.1))} + Ca + \text{ext. Ca} = \frac{350.00 * 1.0000 * 132.0000}{(2 * 20000 * 1 + 2 * 350.00 * (1.0000 - 0.1))} + 0.0625 + 0.0000 = \mathbf{1.1996 \text{ in.}}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + Ca + \text{ext. Ca} = \mathbf{0.3815 \text{ in.}}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = \mathbf{0.3815 \text{ in.}}$$

$$t_{UG-45} = \max(t_a, t_b) = \mathbf{0.3815 \text{ in.}}$$

Wall thickness = $t_n = \mathbf{1.1300}$ is greater than or equal to UG-45 value of $\mathbf{0.3815}$

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 8" DES FILL NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 7
 ID Number: 7

Vessel Number: PG-1080 2080
 Mark Number: N7

Date Printed: 5/16/2012

Limits of Reinforcement (UG-40)

$$\begin{aligned} L_{par} &= \max(d, R_n + t + t_n) = \max(8.6446, 4.3223 + 1.3125 + 1.0675) &= \mathbf{8.6446 \text{ in.}} \\ L_{noro} &= \min(2.5 t, 2.5 t_n + t_e) = \min(2.5 * 1.3125, 2.5 * 1.0675 + 0.0000) &= \mathbf{2.6688 \text{ in.}} \\ L_{nori} &= \min(2.5 t, 2.5 t_i) = \min(2.5 * 1.3125, 2.5 * 1.0050) &= \mathbf{2.5125 \text{ in.}} \end{aligned}$$

Nozzle Reinforcement Calculations (Internal Pressure)

$$A = \max\{C [d t_r F + 2 t_n t_r F (1 - f_{r1})], 0\} = \max\{1.0000 * [8.6446 * 1.0323 * 1.00 + 2 * 1.0675 * 1.0323 * 1.00 * (1 - 0.9900)], 0\} \\ = \mathbf{8.9459 \text{ sq. in.}}$$

$$\begin{aligned} A1 &= \max\{2 L_{par} - d\} (E_1 t - F t_r) - 2 t_n (E_1 t - F t_r) (1 - f_{r1}), 0\} = \\ &\max\{(2 * 8.6446 - 8.6446) * (1.0000 * 1.3125 - 1.00 * 1.0323) - 2 * 1.0675 * (1.0000 * 1.3125 - 1.00 * 1.0323) * (1 - 0.9900), 0\} \\ &= \mathbf{2.4162 \text{ sq. in.}} \end{aligned}$$

$$\begin{aligned} A2 &= \max\{2 \min(h_o, L_{noro}) [\min(t_n, L_{par} - 0.5 d) - t_{rn}] f_{r2}, 0\} = \\ &\max\{2 * \min(6.0000, 2.6688) * [\min(1.0675, 8.6446 - 0.5 * 8.6446) - 0.0726] * 0.9900, 0\} \\ &= \mathbf{5.2572 \text{ sq. in.}} \end{aligned}$$

$$\begin{aligned} A3 &= \max\{2 \min(h, L_{nori}) \min(t_i, L_{par} - 0.5 d) f_{r2}, 0\} = \\ &\max\{2 * \min(2.5000, 2.5125) * \min(1.0050, 8.6446 - 0.5 * 8.6446) * 0.9900, 0\} \\ &= \mathbf{4.9748 \text{ sq. in.}} \end{aligned}$$

$$\begin{aligned} A41 &= f_{r2} [L_{41}^2 - (L_{41} - L_{41pareff})^2 - (L_{41} - L_{41noreff})^2] = \\ &0.9900 * [0.3750^2 - (0.3750 - 0.3750)^2 - (0.3750 - 0.3750)^2] \\ &= \mathbf{0.1392 \text{ sq. in.}} \end{aligned}$$

$$\begin{aligned} A42 &= f_{r4} L_{42pareff} L_{42noreff} = \\ &0.0000 * 0.0000 * 0.0000 \\ &= \mathbf{0.0000 \text{ sq. in.}} \end{aligned}$$

$$\begin{aligned} A43 &= f_{r2} [L_{43}^2 - (L_{43} - L_{43pareff})^2 - (L_{43} - L_{43noreff})^2] = \\ &0.9900 * [0.4107^2 - (0.4107 - 0.4107)^2 - (0.4107 - 0.4107)^2] \\ &= \mathbf{0.1670 \text{ sq. in.}} \end{aligned}$$

$$\begin{aligned} A5 &= \\ &= \mathbf{0.0000 \text{ sq. in.}} \end{aligned}$$

$$\mathbf{Area \text{ Available (Internal Pressure)} = A1 + A2 + A3 + A41 + A42 + A43 + A5 = 12.9544 \text{ sq. in., which is } \geq A (8.9459)}$$

SPX Flow Technology USA, Inc.
SMLS LWN (FVC) - 8" DES FILL NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 7
 ID Number: 7

Vessel Number: PG-1080 2080
 Mark Number: N7

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations

Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 1.0675 = **0.7500 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7} = \mathbf{0.3571 \text{ in.}}$

Weld 41, actual weld leg = **0.3750 in.**

Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 1.0675 = **0.7500 in.**

Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + ca}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.7500 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7} = \mathbf{0.4464 \text{ in.}}$

Weld 43, actual weld leg = **0.5000 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**

Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**

Inner fillet, Weld 43, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 9.1925 * 1.0675 * 13860 = \mathbf{213500 \text{ lb.}}$

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 10.2600 * 0.3750 * 9702 = \mathbf{58600 \text{ lb.}}$

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 10.2600 * 1.5000 * 14652 = \mathbf{354000 \text{ lb.}}$

Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 10.2600 * 0.4107 * 9702 = \mathbf{64200 \text{ lb.}}$

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [8.9459 - 2.4162 + 2 * 1.0675 * 0.9900 * (1.00 * 1.3125 - 1.00 * 1.0323)] * 20000 = **142400 lb.**

W1-1 = (A2 + A5 + A41 + A42) * Sv = (5.2572 + 0.0000 + 0.1392 + 0.0000) * 20000 = **107900 lb.**

W2-2 = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (5.2572 + 4.9748 + 0.1392 + 0.1670 + 2 * 1.0675 * 1.3125 * 0.9900) * 20000 = **266200 lb.**

W3-3 = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv = (5.2572 + 4.9748 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 1.0675 * 1.3125 * 0.9900) * 20000 = **266200 lb.**

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = 58600 + 213500 = **272100 lb.**

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = 58600 + 354000 + 64200 = **476800 lb.**

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 58600 + 64200 + 354000 = **476800 lb.**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 2" RELIEF VALVE NOZZLE IN TOP HEAD

Customer: **Southern Company**

Job No: 7404131

Number: 8

ID Number: 8

Vessel Number: PG-1080 2080

Mark Number: N8

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	6.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	1.0000 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	2.0000 in.	Nozzle Wall Thickness(new):	0.6600 in.
Nozzle ID (corroded):	2.1250 in.	Nozzle Wall Thickness(corroded):	0.5975 in.
Developed Opening:	2.1288 in.	Tangential Dimension L:	38.0000 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve:	B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction:	Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction:	No	Computed Minimum Temperature:	-133 °F

Host Component: Head 2 - 132" OD HEAD, 2:1 ELLIP, TOP

Material:	SA-516 Gr. 70	Head wall thickness(new):	1.5000 in.
Material Stress(S _v):	20000 PSI	Head wall thickness - thin out (corroded):	1.3125 in.

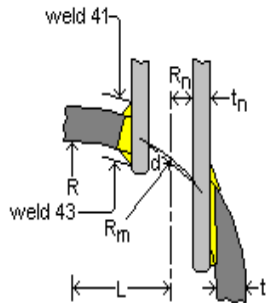
Nozzle Detail Information

Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 0.6600 in.

Outside Groove Weld Depth: 1.5000 in.

tangential to the vessel wall, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Reinforcement calculations are not required per UG-36(c)(3)(a) See Uw-14 for exceptions.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 2" RELIEF VALVE NOZZLE IN TOP HEAD

Job No: 7404131
 Number: 8
 ID Number: 8

Vessel Number: PG-1080 2080
 Mark Number: N8

Date Printed: 5/16/2012

Required Head Thickness per Paragraph UG-37(a)

$$tr = \frac{P K_1 D_o}{(2SE + 0.8P)} = \frac{350.00 * 0.9000 * 132.0000}{(2 * 20000 * 1 + 0.8 * 350.00)} = 1.0323 \text{ in.}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$trn = \frac{PRn}{SE - 0.6P} = \frac{350.00 * 1.0625}{19800 * 1 - 0.6 * 350.00} = 0.0190 \text{ in.}$$

Strength Reduction Factors

$$fr1 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900 \quad fr2 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{PRn}{SE - 0.6P} + Ca + \text{ext. Ca} = \frac{350.00 * 1.0625}{19800 * 1.00 - 0.6 * 350.00} + 0.0625 + 0.0000 = 0.0815 \text{ in.}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P K D_o}{(2SE + 2P(K - 0.1))} + Ca + \text{ext. Ca} = \frac{350.00 * 1.0000 * 132.0000}{(2 * 20000 * 1 + 2 * 350.00 * (1.0000 - 0.1))} + 0.0625 + 0.0000 = 1.1996 \text{ in.}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + Ca + \text{ext. Ca} = 0.2515 \text{ in.}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = 0.2515 \text{ in.}$$

$$t_{UG-45} = \max(t_a, t_b) = 0.2515 \text{ in.}$$

Wall thickness = $t_n = 0.6600$ is greater than or equal to UG-45 value of **0.2515**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 2" RELIEF VALVE NOZZLE IN TOP HEAD

Job No: 7404131
Number: 8
ID Number: 8Vessel Number: PG-1080 2080
Mark Number: N8

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations**Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)**Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 0.5975 = **0.5975 in.**Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7}$ = **0.3571 in.**Weld 41, actual weld leg = **0.3750 in.**Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 0.5975 = **0.5975 in.**Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ca}}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.5975 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7}$ = **0.4464 in.**Weld 43, actual weld leg = **0.5000 in.****Unit Stresses per Paragraphs UG-45(c) and UW-15**Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**Inner fillet, Weld 43, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI****Strength of Connection Elements**Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 2.7225 * 0.5975 * 13860$ = **35400 lb.**Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 3.3200 * 0.3750 * 9702$ = **19000 lb.**Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 3.3200 * 1.5000 * 14652$ = **114600 lb.**Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 3.3200 * 0.4107 * 9702$ = **20800 lb.****Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)****W** = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [2.3389 - 1.0670 + 2 * 0.5975 * 0.9900 * (1.00 * 1.3125 - 1.00 * 1.0323)] * 20000 = **32100 lb.****W1-1** = (A2 + A5 + A41 + A42) * Sv = (1.7110 + 0.0000 + 0.1392 + 0.0000) * 20000 = **37000 lb.****W2-2** = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (1.7110 + 0.9931 + 0.1392 + 0.1670 + 2 * 0.5975 * 1.3125 * 0.9900) * 20000 = **91300 lb.****W3-3** = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv = (1.7110 + 0.9931 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 0.5975 * 1.3125 * 0.9900) * 20000 = **91300 lb.****Check Strength Paths****Path 1-1** = Upper fillet in shear + Nozzle wall in shear = 19000 + 35400 = **54400 lb.****Path 2-2** = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = 19000 + 114600 + 20800 = **154400 lb.****Path 3-3** = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 19000 + 20800 + 114600 = **154400 lb.**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 1-1/2" INSTRUMENT CONN IN SHELL

Customer: **Southern Company**

Job No: 7404131

Number: 9

ID Number: 9

Vessel Number: PG-1080 2080

Mark Number: N9

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	6.0000 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	1.0000 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	1.5000 in.	Nozzle Wall Thickness(new):	0.6250 in.
Nozzle ID (corroded):	1.6250 in.	Nozzle Wall Thickness(corroded):	0.5625 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.5000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve: B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction: Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction: No	Computed Minimum Temperature:	-135 °F

Host Component: Shell 1 - 132" OD ROLLED SHELL x 96" LG

Material: SA-516 Gr. 70	Shell wall thickness(new):	1.5000 in.
Material Stress(S _v): 20000 PSI	Shell wall thickness(corroded):	1.4375 in.

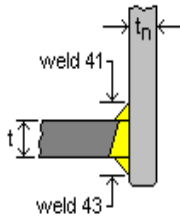
Nozzle Detail Information

Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Internal Weld Leg Size(Weld 43): 0.5000 in.

Nozzle Wall Thickness(t_n): 0.6250 in.

Outside Groove Weld Depth: 1.5000 in.

Nozzle passes through the vessel, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Reinforcement calculations are not required per UG-36(c)(3)(a) See Uw-14 for exceptions.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 1-1/2" INSTRUMENT CONN IN SHELL

Job No: 7404131
 Number: 9
 ID Number: 9

Vessel Number: PG-1080 2080
 Mark Number: N9

Date Printed: 5/16/2012

Required Shell Thickness per Paragraph UG-37(a)

$$t_r = \frac{P R_o}{S E + 0.4 P} = \frac{350.00 * 66.0000}{20000 * 1 + 0.4 * 350.00} = 1.1470 \text{ in.}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$t_{rn} = \frac{P R_n}{S E - 0.6 P} = \frac{350.00 * 0.8125}{19800 * 1 - 0.6 * 350.00} = 0.0145 \text{ in.}$$

Strength Reduction Factors

$$f r_1 = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900 \quad f r_2 = \min \left(\frac{S_n}{S_v}, 1.0000 \right) = \min \left(\frac{19800}{20000}, 1.0000 \right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{P R_n}{S E - 0.6 P} + C_a + \text{ext. Ca} = \frac{350.00 * 0.8125}{19800 * 1.00 - 0.6 * 350.00} + 0.0625 + 0.0000 = 0.0770 \text{ in.}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P R_o}{S E + 0.4 P} + C_a + \text{ext. Ca} = \frac{350.00 * 66.0000}{20000 * 1 + 0.4 * 350.00} + 0.0625 + 0.0000 = 1.2095 \text{ in.}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + C_a + \text{ext. Ca} = 0.2405 \text{ in.}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = 0.2405 \text{ in.}$$

$$t_{UG-45} = \max(t_a, t_b) = 0.2405 \text{ in.}$$

Wall thickness = $t_n = 0.6250$ is greater than or equal to UG-45 value of **0.2405**

SPX Flow Technology USA, Inc.

SMLS LWN (FVC) - 1-1/2" INSTRUMENT CONN IN SHELL

Job No: 7404131
 Number: 9
 ID Number: 9

Vessel Number: PG-1080 2080
 Mark Number: N9

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations**Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)**

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 0.5625 = **0.5625 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7} = \mathbf{0.3571 \text{ in.}}$

Weld 41, actual weld leg = **0.3750 in.**

Weld 43 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.4375, or 0.5625 = **0.5625 in.**

Weld 43 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ca}}{0.7} = \frac{(\text{smaller of } 0.25 \text{ or } (0.5625 * 0.7)) + 0.0625}{0.7} = \frac{0.3125}{0.7} = \mathbf{0.4464 \text{ in.}}$

Weld 43, actual weld leg = **0.5000 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = $0.70 * S_n = 0.70 * 19800 = \mathbf{13860 \text{ PSI}}$

Upper fillet, Weld 41, in shear = $0.49 * \text{Material Stress} = 0.49 * 19800 = \mathbf{9702 \text{ PSI}}$

Vessel groove weld, in tension = $0.74 * \text{Material Stress} = 0.74 * 19800 = \mathbf{14652 \text{ PSI}}$

Inner fillet, Weld 43, in shear = $0.49 * \text{Material Stress} = 0.49 * 19800 = \mathbf{9702 \text{ PSI}}$

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress} = \frac{1}{2} * \pi * 2.1875 * 0.5625 * 13860 = \mathbf{26800 \text{ lb.}}$

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress} = \frac{1}{2} * \pi * 2.7500 * 0.3750 * 9702 = \mathbf{15700 \text{ lb.}}$

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress} = \frac{1}{2} * \pi * 2.7500 * 1.5000 * 14652 = \mathbf{94900 \text{ lb.}}$

Inner fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{inner fillet in shear unit stress} = \frac{1}{2} * \pi * 2.7500 * 0.4107 * 9702 = \mathbf{17200 \text{ lb.}}$

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = $[A - A1 + 2 t_n f r1 (E1t - Ftr)] S_v = [1.8768 - 1.1587 + 2 * 0.5625 * 0.9900 * (1.00 * 1.4375 - 1.00 * 1.1470)] * 20000 = \mathbf{20800 \text{ lb.}}$

W1-1 = $(A2 + A5 + A41 + A42) S_v = (1.5258 + 0.0000 + 0.1392 + 0.0000) * 20000 = \mathbf{33300 \text{ lb.}}$

W2-2 = $(A2 + A3 + A41 + A43 + 2 t_n t f r1) S_v = (1.5258 + 0.9281 + 0.1392 + 0.1670 + 2 * 0.5625 * 1.4375 * 0.9900) * 20000 = \mathbf{87200 \text{ lb.}}$

W3-3 = $(A2 + A3 + A5 + A41 + A42 + A43 + 2 t_n t f r1) S_v = (1.5258 + 0.9281 + 0.0000 + 0.1392 + 0.0000 + 0.1670 + 2 * 0.5625 * 1.4375 * 0.9900) * 20000 = \mathbf{87200 \text{ lb.}}$

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = $15700 + 26800 = \mathbf{42500 \text{ lb.}}$

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear = $15700 + 94900 + 17200 = \mathbf{127800 \text{ lb.}}$

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = $15700 + 17200 + 94900 = \mathbf{127800 \text{ lb.}}$

SPX Flow Technology USA, Inc.

SMLS WELD STUB (FVC) - 20"

Customer: **Southern Company**

Job No: 7404131

Number: 10

ID Number: 10

Vessel Number: PG-1080 2080

Mark Number: N10

Date Printed: 5/16/2012

Nozzle Design Information

Design Pressure:	350.00 PSI	Design Temperature:	450 °F
Static Head:	0.00 PSI	Nozzle Efficiency (E):	100 %
Nozzle Material:	SA-105	Joint Efficiency (E ₁):	1.00
		Factor B Chart:	CS-2
External Projection:	3.2500 in.	Allowable Stress at Design Temperature (S _n):	19800 PSI
Internal Projection:	0.0000 in.	Allowable Stress at Ambient Temperature:	20000 PSI
Inside Corrosion Allowance:	0.0625 in.	Correction Factor (F):	1.00
External Corrosion Allowance:	0.0000 in.	Nozzle Path:	None
Nozzle ID (new):	19.2500 in.	Nozzle Wall Thickness(new):	2.5000 in.
Nozzle ID (corroded):	19.3750 in.	Nozzle Wall Thickness(corroded):	2.4375 in.
		Upper Weld Leg Size(Weld 41):	0.3750 in.
		Internal Weld Leg Size(Weld 43):	0.0000 in.
		Outside Groove Weld Depth:	1.5000 in.

Minimum Design Metal Temperature

Min. Temp. Curve: B	Pressure at MDMT:	350.00 PSI
UCS-66(b) reduction: Yes	Minimum Design Metal Temperature:	10 °F
UCS-68(c) reduction: No	Computed Minimum Temperature:	-89 °F

Host Component: Head 1 - 132" OD HEAD, 2:1 ELLIP, BOTTOM

Material: SA-516 Gr. 70	Head wall thickness(new):	1.5000 in.
Material Stress(S _v): 20000 PSI	Head wall thickness - thin out (corroded):	1.3125 in.

Nozzle Detail Information

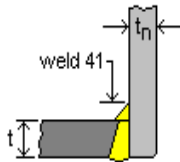


Fig. UW-16.1 (d)

Upper Weld Leg Size(Weld 41): 0.3750 in.

Nozzle Wall Thickness(t_n): 2.5000 in.

Outside Groove Weld Depth: 1.5000 in.

Nozzle passes through the vessel, attached by a groove weld.

Nozzle is adequate for UG-45 requirements.

Opening is adequately reinforced for Internal Pressure.

Weld Strength Paths are adequate.

SPX Flow Technology USA, Inc.

SMLS WELD STUB (FVC) - 20"

Job No: 7404131
 Number: 10
 ID Number: 10

Vessel Number: PG-1080 2080
 Mark Number: N10

Date Printed: 5/16/2012

Required Head Thickness per Paragraph UG-37(a)

$$tr = \frac{P K_1 D_o}{(2SE + 0.8P)} = \frac{350.00 * 0.9000 * 132.0000}{(2 * 20000 * 1 + 0.8 * 350.00)} = \mathbf{1.0323 \text{ in.}}$$

Nozzle Required Thickness Calculations**Required Nozzle Thickness for Internal Pressure per Paragraph UG-37(a)**

$$trn = \frac{PRn}{SE - 0.6P} = \frac{350.00 * 9.6875}{19800 * 1 - 0.6 * 350.00} = \mathbf{0.1731 \text{ in.}}$$

Strength Reduction Factors

$$fr1 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900 \quad fr2 = \min\left(\frac{S_n}{S_v}, 1.0000\right) = \min\left(\frac{19800}{20000}, 1.0000\right) = 0.9900$$

UG-45 Thickness Calculations**Nozzle Thickness for Pressure Loading (plus corrosion)**

$$t_a = \frac{PRn}{SE - 0.6P} + Ca + \text{ext. Ca} = \frac{350.00 * 9.6875}{19800 * 1.00 - 0.6 * 350.00} + 0.0625 + 0.0000 = \mathbf{0.2356 \text{ in.}}$$

Nozzle Thickness for Internal Pressure (plus corrosion) Based on Host

$$t_{b1} = \frac{P K D_o}{(2SE + 2P(K - 0.1))} + Ca + \text{ext. Ca} = \frac{350.00 * 1.0000 * 132.0000}{(2 * 20000 * 1 + 2 * 350.00 * (1.0000 - 0.1))} + 0.0625 + 0.0000 = \mathbf{1.1996 \text{ in.}}$$

Minimum Thickness (plus corrosion) per Table UG-45

$$t_{b3} = \text{minimum thickness (Table UG-45)} + Ca + \text{ext. Ca} = \mathbf{0.3905 \text{ in.}}$$

Nozzle Minimum Thickness Based on Host and Table UG-45

$$t_b = \min[t_{b3}, \max(t_{b1}, t_{b2})] = \mathbf{0.3905 \text{ in.}}$$

$$t_{UG-45} = \max(t_a, t_b) = \mathbf{0.3905 \text{ in.}}$$

Wall thickness = $t_n = \mathbf{2.5000}$ is greater than or equal to UG-45 value of $\mathbf{0.3905}$

SPX Flow Technology USA, Inc.
SMLS WELD STUB (FVC) - 20"

Job No: 7404131
Number: 10
ID Number: 10

Vessel Number: PG-1080 2080
Mark Number: N10

Date Printed: 5/16/2012

Limits of Reinforcement (UG-40)

$$\begin{aligned} L_{par} &= \max(d, R_n + t + t_n) = \max(19.3750, 9.6875 + 1.3125 + 2.4375) &= \mathbf{19.3750} \text{ in.} \\ L_{noro} &= \min(2.5 t, 2.5 t_n + t_e) = \min(2.5 * 1.3125, 2.5 * 2.4375 + 0.0000) &= \mathbf{3.2813} \text{ in.} \\ L_{nori} &= \min(2.5 t, 2.5 t_i) = \min(2.5 * 1.3125, 2.5 * 2.3750) &= \mathbf{3.2813} \text{ in.} \end{aligned}$$

Nozzle Reinforcement Calculations (Internal Pressure)

$$A = \max\{C [d t_r F + 2 t_n t_r F (1 - f_{r1})], 0\} = \max\{1.0000 * [19.3750 * 1.0323 * 1.00 + 2 * 2.4375 * 1.0323 * 1.00 * (1 - 0.9900)], 0\} \\ = \mathbf{20.0511} \text{ sq. in.}$$

$$\begin{aligned} A1 &= \max\{2 L_{par} - d\} (E_1 t - F t_r) - 2 t_n (E_1 t - F t_r) (1 - f_{r1}), 0\} = \\ &\max\{2 * 19.3750 - 19.3750\} * (1.0000 * 1.3125 - 1.00 * 1.0323) - 2 * 2.4375 * (1.0000 * 1.3125 - 1.00 * 1.0323) * (1 - 0.9900), 0\} \\ &= \mathbf{5.4152} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A2 &= \max\{2 \min(h_o, L_{noro}) [\min(t_n, L_{par} - 0.5 d) - t_n] f_{r2}, 0\} = \\ &\max\{2 * \min(3.2500, 3.2813) * [\min(2.4375, 19.3750 - 0.5 * 19.3750) - 0.1731] * 0.9900, 0\} \\ &= \mathbf{14.5714} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A3 &= \max\{2 \min(h, L_{nori}) \min(t_i, L_{par} - 0.5 d) f_{r2}, 0\} = \\ &\max\{2 * \min(0.0000, 3.2813) * \min(2.3750, 19.3750 - 0.5 * 19.3750) * 0.9900, 0\} \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A41 &= f_{r2} [L_{41}^2 - (L_{41} - L_{41pareff})^2 - (L_{41} - L_{41noreff})^2] = \\ &0.9900 * [0.3750^2 - (0.3750 - 0.3750)^2 - (0.3750 - 0.3750)^2] \\ &= \mathbf{0.1392} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A42 &= f_{r4} L_{42pareff} L_{42noreff} = \\ &0.0000 * 0.0000 * 0.0000 \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A43 &= f_{r2} L_{43pareff} L_{43noreff} = \\ &0.9900 * 0.0000 * 0.0000 \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\begin{aligned} A5 & \\ &= \mathbf{0.0000} \text{ sq. in.} \end{aligned}$$

$$\text{Area Available (Internal Pressure)} = A1 + A2 + A3 + A41 + A42 + A43 + A5 = 20.1258 \text{ sq. in., which is } \geq A \text{ (20.0511)}$$

SPX Flow Technology USA, Inc.
SMLS WELD STUB (FVC) - 20"

Job No: 7404131
 Number: 10
 ID Number: 10

Vessel Number: PG-1080 2080
 Mark Number: N10

Date Printed: 5/16/2012

Nozzle Weld Strength Calculations

Attachment Weld Strength per Paragraph UW-16 (Corroded Condition)

Weld 41 tmin = smaller of 0.75, t, or tn = smaller of 0.75, 1.3125, or 2.4375 = **0.7500 in.**

Weld 41 Leg min. = $\frac{(\text{smaller of } 0.25 \text{ or } (t_{\min} * 0.7)) + \text{ext. CA}}{0.7} = \frac{0.2500}{0.7}$ = **0.3571 in.**

Weld 41, actual weld leg = **0.3750 in.**

Unit Stresses per Paragraphs UG-45(c) and UW-15

Nozzle wall in shear = 0.70 * Sn = 0.70 * 19800 = **13860 PSI**

Upper fillet, Weld 41, in shear = 0.49 * Material Stress = 0.49 * 19800 = **9702 PSI**

Vessel groove weld, in tension = 0.74 * Material Stress = 0.74 * 19800 = **14652 PSI**

Strength of Connection Elements

Nozzle wall in shear = $\frac{1}{2} * \pi * \text{mean nozzle diameter} * t_n * \text{Nozzle wall in shear unit stress}$ =
 $\frac{1}{2} * \pi * 21.8125 * 2.4375 * 13860$ = **1156900 lb.**

Upper fillet in shear = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{weld leg} * \text{upper fillet in shear unit stress}$ = $\frac{1}{2} * \pi * 24.2500 * 0.3750 * 9702$ = **138500 lb.**

Groove Weld in Tension = $\frac{1}{2} * \pi * \text{Nozzle OD} * \text{groove depth} * \text{groove weld tension unit stress}$ =
 $\frac{1}{2} * \pi * 24.2500 * 1.3125 * 14652$ = **732200 lb.**

Load to be carried by welds, per UG-41(b)(1) and Fig. UG-41.1 sketch (a)

W = [A - A1 + 2 tn fr1(E1t - Ftr)] Sv = [20.0511 - 5.4152 + 2 * 2.4375 * 0.9900 * (1.00 * 1.3125 - 1.00 * 1.0323)] * 20000 = **319800 lb.**

W1-1 = (A2 + A5 + A41 + A42) * Sv = (14.5714 + 0.0000 + 0.1392 + 0.0000) * 20000 = **294200 lb.**

W2-2 = (A2 + A3 + A41 + A43 + 2 tn t fr1) Sv = (14.5714 + 0.0000 + 0.1392 + 0.0000 + 2 * 2.4375 * 1.3125 * 0.9900) * 20000
 = **420900 lb.**

W3-3 = (A2 + A3 + A5 + A41 + A42 + A43 + 2 tn t fr1) * Sv =
 (14.5714 + 0.0000 + 0.0000 + 0.1392 + 0.0000 + 0.0000 + 2 * 2.4375 * 1.3125 * 0.9900) * 20000 = **420900 lb.**

Check Strength Paths

Path 1-1 = Upper fillet in shear + Nozzle wall in shear = 138500 + 1156900 = **1295400 lb.**

Path 2-2 = Upper fillet in shear + Groove weld in tension + Inner fillet in shear =
 138500 + 732200 + 0 = **870700 lb.**

Path 3-3 = Upper fillet in shear + Inner fillet in shear + Groove weld in tension = 138500 + 0 + 732200 = **870700 lb.**

SPX Flow Technology USA, Inc.

Skirt

Customer: **Southern Company**

Job No: 7404131

Number: 1

Vessel Number: PG-1080 2080

Mark Number: SK1

Date Printed: 5/16/2012

Cylindrical Skirt Design InformationDesign Temperature: 450 °F
Skirt Material: SA-36Skirt Length: 102.0000 in.
Corrosion Allowance: 0.0625 in.
Outside Diameter: 132.0000 in.Surface Area: 293.7389 Sq. Ft.
Long. Factor A: 0.0005919Joint Efficiency: 70 %
Factor B Chart: CS-2
Material Stress(hot): 16600 PSI
Material Stress(cold): 16600 PSI
Yield Strength: 30050 PSI
Modulus of Elasticity: 27.4 10⁶ PSI
Density: 0.2800 lb/in.³
Weight: 4428.71 lb.
Long. Factor B: 8053 PSI**Nominal Skirt Thickness Selected = 0.3750 in.**

SPX Flow Technology USA, Inc.**Base Ring**

Customer: **Southern Company**
 Job No: 7404131
 Mark Number: BP1

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Base Ring Design Information

Design Temperature:	450 °F	Yield Strength:	30050 PSI
Base Ring Material:	SA-36	Density:	0.2800 lb/in. ³
Base Plate Thickness (T _b):	0.5000 in.	Skirt OD at Bottom (D _{sk}):	132.0000 in.
Base Plate OD (OD _b):	138.0000 in.	Base Plate ID (ID _b):	130.0000 in.
Base Plate Width (W _b):	4.0000 in.	Width Outside of Skirt (W _{bo}):	3.0000 in.

Anchor Bolt Information

Anchor Bolt Material:	SA-325	Material Stress(hot):	20200 PSI
Bolt Size:	3/4" in.	Material Stress(cold):	20200 PSI
Bolt Circle (BC):	136.0000 in.	Density:	0.2800 lb/in. ³
Nominal Diameter (D _b):	0.7500 in.	Number of Bolts (N _b):	12
Root Area (A _b):	0.3020 sq. in.	Bolt Hole Diameter (D _{bh}):	0.8750 in.
Threads Per Inch (thd):	10.00	Distance across Flats of Bolting Nut:	1.2500 in.
		Ultimate 28 Day Concrete Strength:	2500 PSI

Base Support Analysis - Operating Pressurized Condition - Occasional Loads - Seismic Case 5

Total Weight of Tower as adjusted by load case combinations

W = 92244 lb.

Maximum total moment at base of tower

M = 519240 in.-lb.**Base Ring Calculations - Operating Pressurized Condition - Occasional Loads - Seismic Case 5**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{519240}{54824} \right) + \left(\frac{92244}{1683.89} \right) \quad f_{c \max} = 64 \text{ PSI}$$

$$\text{Base ring stress, } S_b = W_{bo}^2 * \left(\frac{3 * f_{c \max}}{T_b^2} \right) = 3.0000^2 * \left(\frac{3 * 64}{0.5000^2} \right) \quad S_b = 6939 \text{ PSI}$$

Anchor Bolt and Concrete Stresses - Operating Pressurized Condition - Occasional Loads - Seismic Case 5Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 64 PSI****Base Support Allowable Stresses - Operating Pressurized Condition - Occasional Loads - Seismic Case 5**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI**

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Base Ring

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Base Support Stress Ratios - Operating Pressurized Condition - Occasional Loads - Seismic Case 5Base Ring, R_b $R_b = 0.385$ Bolting, R_{bolt} $R_{bolt} = 0.000$ Concrete, R_c $R_c = 0.086$

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

Base Support Analysis - Operating Pressurized Condition - Occasional Loads - Wind Case 5

Total Weight of Tower as adjusted by load case combinations

 $W = 90186 \text{ lb.}$

Maximum total moment at base of tower

 $M = 396118 \text{ in.-lb.}$ **Base Ring Calculations - Operating Pressurized Condition - Occasional Loads - Wind Case 5**

No uplift, therefore

 $f_s = 0 \text{ PSI}$

$$f_{c \text{ max}} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{396118}{54824} \right) + \left(\frac{90186}{1683.89} \right)$$

 $f_{c \text{ max}} = 61 \text{ PSI}$

$$\text{Base ring stress, } S_b = W_{bo}^2 * \left(\frac{3 * f_{c \text{ max}}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 61}{0.5000^2} \right)$$

 $S_b = 6565 \text{ PSI}$ **Anchor Bolt and Concrete Stresses - Operating Pressurized Condition - Occasional Loads - Wind Case 5**Anchor bolt stress, $S_{bolt} = f_s$ $S_{bolt} = 0 \text{ PSI}$ Concrete stress, $S_c = f_{c \text{ max}}$ $S_c = 61 \text{ PSI}$ **Base Support Allowable Stresses - Operating Pressurized Condition - Occasional Loads - Wind Case 5**Base support material yield strength, S_y $S_y = 30050 \text{ PSI}$ Base ring allowable stress, $S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050$ $S_{ba} = 18030 \text{ PSI}$ Anchor bolt allowable stress, S_{bolta} $S_{bolta} = 20200 \text{ PSI}$ Concrete safe bearing load, $S_{ca} = 0.3 * 28 \text{ day concrete strength} = 0.3 * 2500.00$ $S_{ca} = 750 \text{ PSI}$ **Base Support Stress Ratios - Operating Pressurized Condition - Occasional Loads - Wind Case 5**Base Ring, R_b $R_b = 0.364$ Bolting, R_{bolt} $R_{bolt} = 0.000$ Concrete, R_c $R_c = 0.081$

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

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Base Support Analysis - Operating Pressurized Condition - Sustained Loads

Total Weight of Tower as adjusted by load case combinations

W = 90186 lb.

Maximum total moment at base of tower

M = 0 in.-lb.**Base Ring Calculations - Operating Pressurized Condition - Sustained Loads**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{0}{54824} \right) + \left(\frac{90186}{1683.89} \right)$$

f_{c max} = 54 PSI

$$\text{Base ring stress, } S_b = W_{bo}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 54}{0.5000^2} \right)$$

S_b = 5784 PSI**Anchor Bolt and Concrete Stresses - Operating Pressurized Condition - Sustained Loads**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 54 PSI****Base Support Allowable Stresses - Operating Pressurized Condition - Sustained Loads**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Operating Pressurized Condition - Sustained Loads**Base Ring, R_b**R_b = 0.321**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.071**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

SPX Flow Technology USA, Inc.

Base Ring

Job No: 7404131
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Vessel Number: PG-1080 2080

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Base Support Analysis - Operating Unpressurized Condition - Occasional Loads - Seismic Case 5

Total Weight of Tower as adjusted by load case combinations

W = 92244 lb.

Maximum total moment at base of tower

M = 519240 in.-lb.**Base Ring Calculations - Operating Unpressurized Condition - Occasional Loads - Seismic Case 5**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{519240}{54824} \right) + \left(\frac{92244}{1683.89} \right)$$

f_{c max} = 64 PSI

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 64}{0.5000^2} \right)$$

S_b = 6939 PSI**Anchor Bolt and Concrete Stresses - Operating Unpressurized Condition - Occasional Loads - Seismic Case 5**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 64 PSI****Base Support Allowable Stresses - Operating Unpressurized Condition - Occasional Loads - Seismic Case 5**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Operating Unpressurized Condition - Occasional Loads - Seismic Case 5**Base Ring, R_b**R_b = 0.385**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.086**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

Base Support Analysis - Operating Unpressurized Condition - Occasional Loads - Wind Case 5

Total Weight of Tower as adjusted by load case combinations

W = 90186 lb.

Maximum total moment at base of tower

M = 396118 in.-lb.

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Base Ring

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Base Ring Calculations - Operating Unpressurized Condition - Occasional Loads - Wind Case 5

No uplift, therefore

$$f_s = 0 \text{ PSI}$$

$$f_{c \text{ max}} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{396118}{54824} \right) + \left(\frac{90186}{1683.89} \right) \quad f_{c \text{ max}} = 61 \text{ PSI}$$

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \text{ max}}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 61}{0.5000^2} \right) \quad S_b = 6565 \text{ PSI}$$

Anchor Bolt and Concrete Stresses - Operating Unpressurized Condition - Occasional Loads - Wind Case 5Anchor bolt stress, $S_{bolt} = f_s$

$$S_{bolt} = 0 \text{ PSI}$$

Concrete stress, $S_c = f_{c \text{ max}}$

$$S_c = 61 \text{ PSI}$$

Base Support Allowable Stresses - Operating Unpressurized Condition - Occasional Loads - Wind Case 5Base support material yield strength, S_y

$$S_y = 30050 \text{ PSI}$$

Base ring allowable stress, $S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050$

$$S_{ba} = 18030 \text{ PSI}$$

Anchor bolt allowable stress, S_{bolta}

$$S_{bolta} = 20200 \text{ PSI}$$

Concrete safe bearing load, $S_{ca} = 0.3 * 28 \text{ day concrete strength} = 0.3 * 2500.00$

$$S_{ca} = 750 \text{ PSI}$$

Base Support Stress Ratios - Operating Unpressurized Condition - Occasional Loads - Wind Case 5Base Ring, R_b

$$R_b = 0.364$$

Bolting, R_{bolt}

$$R_{bolt} = 0.000$$

Concrete, R_c

$$R_c = 0.081$$

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

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Base Support Analysis - Operating Unpressurized Condition - Sustained Loads

Total Weight of Tower as adjusted by load case combinations

W = 90186 lb.

Maximum total moment at base of tower

M = 0 in.-lb.**Base Ring Calculations - Operating Unpressurized Condition - Sustained Loads**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{0}{54824} \right) + \left(\frac{90186}{1683.89} \right)$$

f_{c max} = 54 PSI

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 54}{0.5000^2} \right)$$

S_b = 5784 PSI**Anchor Bolt and Concrete Stresses - Operating Unpressurized Condition - Sustained Loads**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 54 PSI****Base Support Allowable Stresses - Operating Unpressurized Condition - Sustained Loads**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Operating Unpressurized Condition - Sustained Loads**Base Ring, R_b**R_b = 0.321**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.071**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

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Base Ring

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Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Base Support Analysis - Empty Pressurized Condition - Occasional Loads - Seismic Case 5

Total Weight of Tower as adjusted by load case combinations

W = 50364 lb.

Maximum total moment at base of tower

M = 270467 in.-lb.**Base Ring Calculations - Empty Pressurized Condition - Occasional Loads - Seismic Case 5**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{270467}{54824} \right) + \left(\frac{50364}{1683.89} \right)$$

f_{c max} = 35 PSI

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 35}{0.5000^2} \right)$$

S_b = 3763 PSI**Anchor Bolt and Concrete Stresses - Empty Pressurized Condition - Occasional Loads - Seismic Case 5**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 35 PSI****Base Support Allowable Stresses - Empty Pressurized Condition - Occasional Loads - Seismic Case 5**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Empty Pressurized Condition - Occasional Loads - Seismic Case 5**Base Ring, R_b**R_b = 0.209**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.046**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

Base Support Analysis - Empty Pressurized Condition - Occasional Loads - Wind Case 5

Total Weight of Tower as adjusted by load case combinations

W = 49241 lb.

Maximum total moment at base of tower

M = 396118 in.-lb.

SPX Flow Technology USA, Inc.

Base Ring

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Base Ring Calculations - Empty Pressurized Condition - Occasional Loads - Wind Case 5

No uplift, therefore

$$f_s = 0 \text{ PSI}$$

$$f_{c \text{ max}} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{396118}{54824} \right) + \left(\frac{49241}{1683.89} \right) \quad f_{c \text{ max}} = 36 \text{ PSI}$$

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \text{ max}}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 36}{0.5000^2} \right) \quad S_b = 3938 \text{ PSI}$$

Anchor Bolt and Concrete Stresses - Empty Pressurized Condition - Occasional Loads - Wind Case 5Anchor bolt stress, $S_{bolt} = f_s$

$$S_{bolt} = 0 \text{ PSI}$$

Concrete stress, $S_c = f_{c \text{ max}}$

$$S_c = 36 \text{ PSI}$$

Base Support Allowable Stresses - Empty Pressurized Condition - Occasional Loads - Wind Case 5Base support material yield strength, S_y

$$S_y = 30050 \text{ PSI}$$

Base ring allowable stress, $S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050$

$$S_{ba} = 18030 \text{ PSI}$$

Anchor bolt allowable stress, S_{bolta}

$$S_{bolta} = 20200 \text{ PSI}$$

Concrete safe bearing load, $S_{ca} = 0.3 * 28 \text{ day concrete strength} = 0.3 * 2500.00$

$$S_{ca} = 750 \text{ PSI}$$

Base Support Stress Ratios - Empty Pressurized Condition - Occasional Loads - Wind Case 5Base Ring, R_b

$$R_b = 0.218$$

Bolting, R_{bolt}

$$R_{bolt} = 0.000$$

Concrete, R_c

$$R_c = 0.049$$

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

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Base Ring

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Mark Number: BP1

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Base Support Analysis - Empty Pressurized Condition - Sustained Loads

Total Weight of Tower as adjusted by load case combinations

W = 49241 lb.

Maximum total moment at base of tower

M = 0 in.-lb.**Base Ring Calculations - Empty Pressurized Condition - Sustained Loads**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{0}{54824} \right) + \left(\frac{49241}{1683.89} \right)$$

f_{c max} = 29 PSI

$$\text{Base ring stress, } S_b = W_{bo}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 29}{0.5000^2} \right)$$

S_b = 3158 PSI**Anchor Bolt and Concrete Stresses - Empty Pressurized Condition - Sustained Loads**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 29 PSI****Base Support Allowable Stresses - Empty Pressurized Condition - Sustained Loads**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Empty Pressurized Condition - Sustained Loads**Base Ring, R_b**R_b = 0.175**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.039**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

SPX Flow Technology USA, Inc.

Base Ring

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Base Support Analysis - Empty Unpressurized Condition - Occasional Loads - Seismic Case 5

Total Weight of Tower as adjusted by load case combinations

W = 50364 lb.

Maximum total moment at base of tower

M = 270467 in.-lb.**Base Ring Calculations - Empty Unpressurized Condition - Occasional Loads - Seismic Case 5**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{270467}{54824} \right) + \left(\frac{50364}{1683.89} \right)$$

f_{c max} = 35 PSI

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 35}{0.5000^2} \right)$$

S_b = 3763 PSI**Anchor Bolt and Concrete Stresses - Empty Unpressurized Condition - Occasional Loads - Seismic Case 5**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 35 PSI****Base Support Allowable Stresses - Empty Unpressurized Condition - Occasional Loads - Seismic Case 5**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Empty Unpressurized Condition - Occasional Loads - Seismic Case 5**Base Ring, R_b**R_b = 0.209**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.046**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

Base Support Analysis - Empty Unpressurized Condition - Occasional Loads - Wind Case 5

Total Weight of Tower as adjusted by load case combinations

W = 49241 lb.

Maximum total moment at base of tower

M = 396118 in.-lb.

SPX Flow Technology USA, Inc.

Base Ring

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Base Ring Calculations - Empty Unpressurized Condition - Occasional Loads - Wind Case 5

No uplift, therefore

$$f_s = 0 \text{ PSI}$$

$$f_{c \text{ max}} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{396118}{54824} \right) + \left(\frac{49241}{1683.89} \right) \quad f_{c \text{ max}} = 36 \text{ PSI}$$

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \text{ max}}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 36}{0.5000^2} \right) \quad S_b = 3938 \text{ PSI}$$

Anchor Bolt and Concrete Stresses - Empty Unpressurized Condition - Occasional Loads - Wind Case 5Anchor bolt stress, $S_{bolt} = f_s$

$$S_{bolt} = 0 \text{ PSI}$$

Concrete stress, $S_c = f_{c \text{ max}}$

$$S_c = 36 \text{ PSI}$$

Base Support Allowable Stresses - Empty Unpressurized Condition - Occasional Loads - Wind Case 5Base support material yield strength, S_y

$$S_y = 30050 \text{ PSI}$$

Base ring allowable stress, $S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050$

$$S_{ba} = 18030 \text{ PSI}$$

Anchor bolt allowable stress, S_{bolta}

$$S_{bolta} = 20200 \text{ PSI}$$

Concrete safe bearing load, $S_{ca} = 0.3 * 28 \text{ day concrete strength} = 0.3 * 2500.00$

$$S_{ca} = 750 \text{ PSI}$$

Base Support Stress Ratios - Empty Unpressurized Condition - Occasional Loads - Wind Case 5Base Ring, R_b

$$R_b = 0.218$$

Bolting, R_{bolt}

$$R_{bolt} = 0.000$$

Concrete, R_c

$$R_c = 0.049$$

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

SPX Flow Technology USA, Inc.

Base Ring

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Base Support Analysis - Empty Unpressurized Condition - Sustained Loads

Total Weight of Tower as adjusted by load case combinations

W = 49241 lb.

Maximum total moment at base of tower

M = 0 in.-lb.**Base Ring Calculations - Empty Unpressurized Condition - Sustained Loads**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{0}{54824} \right) + \left(\frac{49241}{1683.89} \right)$$

f_{c max} = 29 PSI

$$\text{Base ring stress, } S_b = W_{bo}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 29}{0.5000^2} \right)$$

S_b = 3158 PSI**Anchor Bolt and Concrete Stresses - Empty Unpressurized Condition - Sustained Loads**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 29 PSI****Base Support Allowable Stresses - Empty Unpressurized Condition - Sustained Loads**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Empty Unpressurized Condition - Sustained Loads**Base Ring, R_b**R_b = 0.175**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.039**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

SPX Flow Technology USA, Inc.

Base Ring

Job No: 7404131
Mark Number: BP1

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Base Support Analysis - Test Condition

Total Weight of Tower as adjusted by load case combinations

W = 104756 lb.

Maximum total moment at base of tower

M = 0 in.-lb.**Base Ring Calculations - Test Condition**

No uplift, therefore

f_s = 0 PSI

$$f_{c \max} = \left(\frac{M}{Z_c} \right) + \left(\frac{W}{A_c} \right) = \left(\frac{0}{54824} \right) + \left(\frac{104756}{1683.89} \right)$$

f_{c max} = 62 PSI

$$\text{Base ring stress, } S_b = W_{bo}{}^2 * \left(\frac{3 * f_{c \max}}{Tb^2} \right) = 3.0000^2 * \left(\frac{3 * 62}{0.5000^2} \right)$$

S_b = 6719 PSI**Anchor Bolt and Concrete Stresses - Test Condition**Anchor bolt stress, S_{bolt} = f_s**S_{bolt} = 0 PSI**Concrete stress, S_c = f_{c max}**S_c = 62 PSI****Base Support Allowable Stresses - Test Condition**Base support material yield strength, S_y**S_y = 30050 PSI**Base ring allowable stress, S_{ba} = 1.0 * 0.6 * S_y = 1.0 * 0.6 * 30050**S_{ba} = 18030 PSI**Anchor bolt allowable stress, S_{bolta}**S_{bolta} = 20200 PSI**Concrete safe bearing load, S_{ca} = 0.3 * 28 day concrete strength = 0.3 * 2500.00**S_{ca} = 750 PSI****Base Support Stress Ratios - Test Condition**Base Ring, R_b**R_b = 0.373**Bolting, R_{bolt}**R_{bolt} = 0.000**Concrete, R_c**R_c = 0.083**

* NOTE: Stress ratios greater than 1.0 represent overstressed conditions.

SPX Flow Technology USA, Inc.

Service - CO2 Dehydration Units

Customer: **Southern Company**

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

ASME Flange Design Information

Host	Description	Type	Size (in.)	Material	ASME Class	Material Group	MAP (PSI)
SMLS LWN (FVC) - 20" M	Smls LWN (FVC) - 20"	Weld Neck	20		300	1.1	620.00
SMLS LWN (FVC) - 8" DE	Smls LWN (FVC) - 8"	Weld Neck	8		300	1.1	620.00
Smls LWN (FVC) - 16"	Smls LWN (FVC) - 16"	Weld Neck	16		300	1.1	620.00
SMLS LWN (FVC) - 20" I	Smls Long Weld Neck-2	Weld Neck	20		300	1.1	620.00
SMLS LWN (FVC) - 8" DE	Smls LWN (FVC) - 8"	Weld Neck	8		300	1.1	620.00
SMLS LWN (FVC) - 2" RE	Smls LWN (FVC) - 2"	Weld Neck	2		300	1.1	620.00
SMLS LWN (FVC) - 1-1/2	Smls LWN (FVC) - 1-1/2	Weld Neck	1.5		300	1.1	620.00
SMLS WELD STUB (FVC) -	Smls LWN (FVC) - 20"	Weld Neck	20		300	1.1	620.00
SMLS LWN (FVC) - 8" DE	Smls Long Weld Neck-	Weld Neck	8		300	1.1	620.00

SPX Flow Technology USA, Inc.

Lift Lug

Customer: **Southern Company**

Job No: 7404131

Number: 1

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

WRC-107 Loading Information

Design Pressure:	350.00 PSI	Allowable Stress Multiplier:	3.0
Design Temperature:	450 °F	Yield Multiplier:	1.0

Elliptical Vessel Information

Outside Diameter:	130.5000 in.	Design Temperature:	450 °F
Thickness:	1.3750 in.	Corrosion Allowance:	0.0625 in.
Material:	SA-516 Gr. 70		
Material Condition:	Normalized	Allowable Stress at Design Temperature:	20000 PSI
Density:	0.2800 lb/in.^3	Allowable Stress at Ambient Temperature:	20000 PSI
Modulus of Elasticity:	27.4 10^6 PSI	Yield Strength:	31750 PSI

Rigid Square Attachment Information

Square Width:	5.1400 in.	Design Temperature:	450 °F
Material:	SA-36		
Density:	0.2800 lb/in.^3	Allowable Stress at Design Temperature:	16600 PSI
Modulus of Elasticity:	27.4 10^6 PSI	Allowable Stress at Ambient Temperature:	16600 PSI
Factor Kn:	1.00	Yield Strength:	30050 PSI
		Factor Kb:	1.00

Solve For Stresses

Radial Load, P:	= 50000 lb.
External shear load in the 2-2 direction, V1:	= 25000 lb.
External shear load in the 1-1 direction, V2:	= 0 lb.
External overturning moment in the 1-1 direction, M1:	= 0 Ft.-lb.
External overturning moment in the 2-2 direction, M2:	= 0 Ft.-lb.
External torsional moment, Mt:	= 0 Ft.-lb.

Geometric Parameters

$$U = \frac{c_1}{0.875\sqrt{R_m * T}} = \frac{2.5700}{0.875\sqrt{64.5938 * 1.3125}} = 0.3190$$

Allowable Stress:

min[Yield Multiplier * min(Host Yield, Attachment Yield), Stress Multiplier * min(Host Stress, Attachment Stress)] = 30050

The Vessel Combined Stress Intensity is less than or equal to the Allowable Stress

SPX Flow Technology USA, Inc.

Lift Lug

Job No: 7404131
Number: 1

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

From Figure	Read Curves For	Absolute values of stress	A _U	A _L	B _U	B _L	C _U	C _L	D _U	D _L
Pressure			8560	8560	8560	8560	8560	8560	8560	8560
SR-2	$\frac{N_x T}{P} = 0.1829$	$K_n * \frac{N_x T}{P} * \frac{P}{T^2} = 5308$	-5308	-5308	-5308	-5308	-5308	-5308	-5308	-5308
	$\frac{M_x}{P} = 0.1305$	$K_b * \frac{M_x}{P} * \frac{6P}{T^2} = 22723$	-22723	22723	-22723	22723	-22723	22723	-22723	22723
SR-3	$\frac{N_x T \sqrt{R_m T}}{M_1} = 0.1625$	$K_n * \frac{N_x T \sqrt{R_m T}}{M_1} * \frac{M_1}{T^2 \sqrt{R_m T}} = 0$					0	0	0	0
	$\frac{M_x \sqrt{R_m T}}{M_1} = 0.5126$	$K_b * \frac{M_x \sqrt{R_m T}}{M_1} * \frac{6M_1}{T^2 \sqrt{R_m T}} = 0$					0	0	0	0
	$\frac{N_x T \sqrt{R_m T}}{M_2} = 0.1625$	$K_n * \frac{N_x T \sqrt{R_m T}}{M_2} * \frac{M_2}{T^2 \sqrt{R_m T}} = 0$	0	0	0	0				
	$\frac{M_x \sqrt{R_m T}}{M_2} = 0.5126$	$K_b * \frac{M_x \sqrt{R_m T}}{M_2} * \frac{6M_2}{T^2 \sqrt{R_m T}} = 0$	0	0	0	0				
σ_x Stress Totals			-19472	25975	-19472	25975	-19472	25975	-19472	25975
Pressure			8560	8560	8560	8560	8560	8560	8560	8560
SR-2	$\frac{N_y T}{P} = 0.0551$	$K_n * \frac{N_y T}{P} * \frac{P}{T^2} = 1598$	-1598	-1598	-1598	-1598	-1598	-1598	-1598	-1598
	$\frac{M_y}{P} = 0.0393$	$K_b * \frac{M_y}{P} * \frac{6P}{T^2} = 6836$	-6836	6836	-6836	6836	-6836	6836	-6836	6836
SR-3	$\frac{N_y T \sqrt{R_m T}}{M_1} = 0.0505$	$K_n * \frac{N_y T \sqrt{R_m T}}{M_1} * \frac{M_1}{T^2 \sqrt{R_m T}} = 0$					0	0	0	0
	$\frac{M_y \sqrt{R_m T}}{M_1} = 0.1281$	$K_b * \frac{M_y \sqrt{R_m T}}{M_1} * \frac{6M_1}{T^2 \sqrt{R_m T}} = 0$					0	0	0	0
	$\frac{N_y T \sqrt{R_m T}}{M_2} = 0.0505$	$K_n * \frac{N_y T \sqrt{R_m T}}{M_2} * \frac{M_2}{T^2 \sqrt{R_m T}} = 0$	0	0	0	0				
	$\frac{M_y \sqrt{R_m T}}{M_2} = 0.1281$	$K_b * \frac{M_y \sqrt{R_m T}}{M_2} * \frac{6M_2}{T^2 \sqrt{R_m T}} = 0$	0	0	0	0				
σ_y Stress Totals			126	13797	126	13797	126	13797	126	13797
Due to load, V ₁		$\frac{V_1}{\pi r_o T} = 1853$					-1853	-1853	1853	1853
Due to load, V ₂		$\frac{V_2}{\pi r_o T} = 0$	0	0	0	0				
Shear Stress Totals			0	0	0	0	-1853	-1853	1853	1853
Combined Stress Intensities			19598	25975	19598	25975	19945	26251	19945	26251
<p>1) When $\tau \neq 0$, Combined Stress Intensity = largest absolute magnitude of either $\frac{1}{2} [\sigma_x + \sigma_y \pm \sqrt{(\sigma_x - \sigma_y)^2 + 4\tau^2}]$ or $\sqrt{(\sigma_x - \sigma_y)^2 + 4\tau^2}$</p> <p>2) When $\tau = 0$, Combined Stress Intensity = largest absolute magnitude of σ_x, σ_y, or $(\sigma_x - \sigma_y)$</p>										

Allowable Stress:
min[Yield Multiplier * min(Host Yield, Attachment Yield), Stress Multiplier * min(Host Stress, Attachment Stress)] = 30050
The Largest Vessel Combined Stress Intensity = 26251 PSI

SPX Flow Technology USA, Inc.

WRC - Nozzle 6 Top

Customer: **Southern Company**

Job No: 7404131

Number: 2

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

WRC-107 Loading Information

Design Pressure:	350.00	PSI	Allowable Stress Multiplier:	3.0
Design Temperature:	450	°F	Yield Multiplier:	1.0

Elliptical Vessel Information

Outside Diameter:	130.5000	in.	Design Temperature:	450	°F
Thickness:	1.3750	in.	Corrosion Allowance:	0.0625	in.
Material:	SA-516 Gr. 70				
Material Condition:	Normalized				
Density:	0.2800	lb/in.^3	Allowable Stress at Design Temperature:	20000	PSI
Modulus of Elasticity:	27.4	10^6 PSI	Allowable Stress at Ambient Temperature:	20000	PSI
			Yield Strength:	31750	PSI

Hollow Cylinder Attachment Information

Outside Radius, ro:	11.5626	in.	Design Temperature:	450	°F
Thickness, t:	1.5626	in.	Corrosion Allowance:	0.0625	in.
Material:	SA-105				
Density:	0.2800	lb/in.^3	Allowable Stress at Design Temperature:	19800	PSI
Modulus of Elasticity:	27.4	10^6 PSI	Allowable Stress at Ambient Temperature:	20000	PSI
Factor Kn:	1.00		Yield Strength:	30050	PSI
			Factor Kb:	1.00	

Solve For Stresses

Radial Load, P:	= 100000 lb.
External shear load in the 2-2 direction, V1:	= 100000 lb.
External shear load in the 1-1 direction, V2:	= 100000 lb.
External overturning moment in the 1-1 direction, M1:	= 50000 Ft.-lb.
External overturning moment in the 2-2 direction, M2:	= 50000 Ft.-lb.
External torsional moment, Mt:	= 50000 Ft.-lb.

Geometric Parameters

$$U = \frac{r_o}{\sqrt{R_m * T}} = \frac{11.5626}{\sqrt{64.5938 * 1.3125}} = 1.2558$$

$$\tau = \frac{r_m}{t} = \frac{10.8126}{1.5001} = 7.2079$$

$$\rho = \frac{T}{t} = \frac{1.3125}{1.5001} = 0.8749$$

Allowable Stress:

min[Yield Multiplier * min(Host Yield, Attachment Yield), Stress Multiplier * min(Host Stress, Attachment Stress)] = 30050

The Vessel Combined Stress Intensity is less than or equal to the Allowable Stress

SPX Flow Technology USA, Inc.

WRC - Nozzle 6 Top

Job No: 7404131
Number: 2

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

From Figure	Read Curves For	Absolute values of stress	A _U	A _L	B _U	B _L	C _U	C _L	D _U	D _L
Pressure			8560	8560	8560	8560	8560	8560	8560	8560
SP1 to 10	$\frac{N_x T}{P} = 0.0304$	$K_n * \frac{N_x T}{P} * \frac{P}{T^2} = 1765$	-1765	-1765	-1765	-1765	-1765	-1765	-1765	-1765
	$\frac{M_x}{P} = 0.0304$	$K_b * \frac{M_x}{P} * \frac{6P}{T^2} = 10588$	-10588	10588	-10588	10588	-10588	10588	-10588	10588
SM1 to 10	$\frac{N_x T \sqrt{Rm T}}{M_1} = 0.0353$	$K_n * \frac{N_x T \sqrt{Rm T}}{M_1} * \frac{M_1}{T^2 \sqrt{Rm T}} = 1335$					-1335	-1335	1335	1335
	$\frac{M_x \sqrt{Rm T}}{M_1} = 0.0440$	$K_b * \frac{M_x \sqrt{Rm T}}{M_1} * \frac{6M_1}{T^2 \sqrt{Rm T}} = 9986$					-9986	9986	9986	-9986
	$\frac{N_x T \sqrt{Rm T}}{M_2} = 0.0353$	$K_n * \frac{N_x T \sqrt{Rm T}}{M_2} * \frac{M_2}{T^2 \sqrt{Rm T}} = 1335$	-1335	-1335	1335	1335				
	$\frac{M_x \sqrt{Rm T}}{M_2} = 0.0440$	$K_b * \frac{M_x \sqrt{Rm T}}{M_2} * \frac{6M_2}{T^2 \sqrt{Rm T}} = 9986$	-9986	9986	9986	-9986				
σ_x Stress Totals			-15115	26035	7529	8732	-15115	26035	7529	8732
Pressure			8560	8560	8560	8560	8560	8560	8560	8560
SP1 to 10	$\frac{N_y T}{P} = 0.0615$	$K_n * \frac{N_y T}{P} * \frac{P}{T^2} = 3570$	-3570	-3570	-3570	-3570	-3570	-3570	-3570	-3570
	$\frac{M_y}{P} = 0.0095$	$K_b * \frac{M_y}{P} * \frac{6P}{T^2} = 3309$	-3309	3309	-3309	3309	-3309	3309	-3309	3309
SM1 to 10	$\frac{N_y T \sqrt{Rm T}}{M_1} = 0.0609$	$K_n * \frac{N_y T \sqrt{Rm T}}{M_1} * \frac{M_1}{T^2 \sqrt{Rm T}} = 2304$					-2304	-2304	2304	2304
	$\frac{M_y \sqrt{Rm T}}{M_1} = 0.0169$	$K_b * \frac{M_y \sqrt{Rm T}}{M_1} * \frac{6M_1}{T^2 \sqrt{Rm T}} = 3836$					-3836	3836	3836	-3836
	$\frac{N_y T \sqrt{Rm T}}{M_2} = 0.0609$	$K_n * \frac{N_y T \sqrt{Rm T}}{M_2} * \frac{M_2}{T^2 \sqrt{Rm T}} = 2304$	-2304	-2304	2304	2304				
	$\frac{M_y \sqrt{Rm T}}{M_2} = 0.0169$	$K_b * \frac{M_y \sqrt{Rm T}}{M_2} * \frac{6M_2}{T^2 \sqrt{Rm T}} = 3836$	-3836	3836	3836	-3836				
σ_y Stress Totals			-4458	9831	7820	6767	-4458	9831	7820	6767
Due to load, V ₁		$\frac{V_1}{\pi r_o T} = 2097$					-2097	-2097	2097	2097
Due to load, V ₂		$\frac{V_2}{\pi r_o T} = 2097$	2097	2097	-2097	-2097				
Due to Torsion, M _T		$\frac{M_T}{2\pi r_o^2 T} = 544$	544	544	544	544	544	544	544	544
Shear Stress Totals			2642	2642	-1553	-1553	-1553	-1553	2642	2642
Combined Stress Intensities			15734	26455	9235	9588	15337	26182	10320	10568

- 1) When $\tau < 0$, Combined Stress Intensity = largest absolute magnitude of either
 $\frac{1}{2} [\sigma_x + \sigma_y \pm \sqrt{(\sigma_x - \sigma_y)^2 + 4\tau^2}]$
 or
 $\sqrt{(\sigma_x - \sigma_y)^2 + 4\tau^2}$
- 2) When $\tau = 0$, Combined Stress Intensity = largest absolute magnitude of σ_x , σ_y , or $(\sigma_x - \sigma_y)$

Allowable Stress:
 $\min[\text{Yield Multiplier} * \min(\text{Host Yield, Attachment Yield}), \text{Stress Multiplier} * \min(\text{Host Stress, Attachment Stress})] = 30050$
 The Largest Vessel Combined Stress Intensity = 26455 PSI

SPX Flow Technology USA, Inc.

WRC - Nozzle 10 Bottom

Customer: **Southern Company**

Job No: 7404131

Number: 3

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

WRC-107 Loading Information

Design Pressure:	350.00	PSI	Allowable Stress Multiplier:	3.0
Design Temperature:	450	°F	Yield Multiplier:	1.0

Elliptical Vessel Information

Outside Diameter:	130.5000	in.	Design Temperature:	450	°F
Thickness:	1.3750	in.	Corrosion Allowance:	0.0625	in.
Material:	SA-516 Gr. 70				
Material Condition:	Normalized		Allowable Stress at Design Temperature:	20000	PSI
Density:	0.2800	lb/in.^3	Allowable Stress at Ambient Temperature:	20000	PSI
Modulus of Elasticity:	27.4	10^6 PSI	Yield Strength:	31750	PSI

Hollow Cylinder Attachment Information

Outside Radius, ro:	12.1250	in.	Design Temperature:	450	°F
Thickness, t:	2.5000	in.	Corrosion Allowance:	0.0625	in.
Material:	SA-105				
Density:	0.2800	lb/in.^3	Allowable Stress at Design Temperature:	19800	PSI
Modulus of Elasticity:	27.4	10^6 PSI	Allowable Stress at Ambient Temperature:	20000	PSI
Factor Kn:	1.00		Yield Strength:	30050	PSI
			Factor Kb:	1.00	

Solve For Stresses

Radial Load, P:	= 100000 lb.
External shear load in the 2-2 direction, V1:	= 100000 lb.
External shear load in the 1-1 direction, V2:	= 100000 lb.
External overturning moment in the 1-1 direction, M1:	= 50000 Ft.-lb.
External overturning moment in the 2-2 direction, M2:	= 50000 Ft.-lb.
External torsional moment, Mt:	= 50000 Ft.-lb.

Geometric Parameters

$$U = \frac{r_o}{\sqrt{R_m * T}} = \frac{12.1250}{\sqrt{64.5938 * 1.3125}} = 1.3169$$

$$\tau = \frac{r_m}{t} = \frac{10.9063}{2.4375} = 4.4744$$

$$\rho = \frac{T}{t} = \frac{1.3125}{2.4375} = 0.5385$$

Allowable Stress:

min[Yield Multiplier * min(Host Yield, Attachment Yield), Stress Multiplier * min(Host Stress, Attachment Stress)] = 30050

The Vessel Combined Stress Intensity is less than or equal to the Allowable Stress

SPX Flow Technology USA, Inc.

WRC - Nozzle 10 Bottom

Job No: 7404131
Number: 3

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

From Figure	Read Curves For	Absolute values of stress	A _U	A _L	B _U	B _L	C _U	C _L	D _U	D _L
Pressure			8560	8560	8560	8560	8560	8560	8560	8560
SP1 to 10	$\frac{N_x T}{P} = 0.0310$	$K_n * \frac{N_x T}{P} * \frac{P}{T^2} = 1800$	-1800	-1800	-1800	-1800	-1800	-1800	-1800	-1800
	$\frac{M_x}{P} = 0.0299$	$K_b * \frac{M_x}{P} * \frac{6P}{T^2} = 10414$	-10414	10414	-10414	10414	-10414	10414	-10414	10414
SM1 to 10	$\frac{N_x T \sqrt{Rm T}}{M_1} = 0.0332$	$K_n * \frac{N_x T \sqrt{Rm T}}{M_1} * \frac{M_1}{T^2 \sqrt{Rm T}} = 1256$					-1256	-1256	1256	1256
	$\frac{M_x \sqrt{Rm T}}{M_1} = 0.0436$	$K_b * \frac{M_x \sqrt{Rm T}}{M_1} * \frac{6M_1}{T^2 \sqrt{Rm T}} = 9896$					-9896	9896	9896	-9896
	$\frac{N_x T \sqrt{Rm T}}{M_2} = 0.0332$	$K_n * \frac{N_x T \sqrt{Rm T}}{M_2} * \frac{M_2}{T^2 \sqrt{Rm T}} = 1256$	-1256	-1256	1256	1256				
	$\frac{M_x \sqrt{Rm T}}{M_2} = 0.0436$	$K_b * \frac{M_x \sqrt{Rm T}}{M_2} * \frac{6M_2}{T^2 \sqrt{Rm T}} = 9896$	-9896	9896	9896	-9896				
σ_x Stress Totals			-14805	25814	7498	8535	-14805	25814	7498	8535
Pressure			8560	8560	8560	8560	8560	8560	8560	8560
SP1 to 10	$\frac{N_y T}{P} = 0.0473$	$K_n * \frac{N_y T}{P} * \frac{P}{T^2} = 2746$	-2746	-2746	-2746	-2746	-2746	-2746	-2746	-2746
	$\frac{M_y}{P} = 0.0084$	$K_b * \frac{M_y}{P} * \frac{6P}{T^2} = 2926$	-2926	2926	-2926	2926	-2926	2926	-2926	2926
SM1 to 10	$\frac{N_y T \sqrt{Rm T}}{M_1} = 0.0459$	$K_n * \frac{N_y T \sqrt{Rm T}}{M_1} * \frac{M_1}{T^2 \sqrt{Rm T}} = 1736$					-1736	-1736	1736	1736
	$\frac{M_y \sqrt{Rm T}}{M_1} = 0.0142$	$K_b * \frac{M_y \sqrt{Rm T}}{M_1} * \frac{6M_1}{T^2 \sqrt{Rm T}} = 3223$					-3223	3223	3223	-3223
	$\frac{N_y T \sqrt{Rm T}}{M_2} = 0.0459$	$K_n * \frac{N_y T \sqrt{Rm T}}{M_2} * \frac{M_2}{T^2 \sqrt{Rm T}} = 1736$	-1736	-1736	1736	1736				
	$\frac{M_y \sqrt{Rm T}}{M_2} = 0.0142$	$K_b * \frac{M_y \sqrt{Rm T}}{M_2} * \frac{6M_2}{T^2 \sqrt{Rm T}} = 3223$	-3223	3223	3223	-3223				
σ_y Stress Totals			-2071	10227	7848	7253	-2071	10227	7848	7253
Due to load, V ₁		$\frac{V_1}{\pi r_o T} = 2000$					-2000	-2000	2000	2000
Due to load, V ₂		$\frac{V_2}{\pi r_o T} = 2000$	2000	2000	-2000	-2000				
Due to Torsion, M _T		$\frac{M_T}{2\pi r_o^2 T} = 495$	495	495	495	495	495	495	495	495
Shear Stress Totals			2495	2495	-1505	-1505	-1505	-1505	2495	2495
Combined Stress Intensities			15277	26204	9188	9530	14981	25958	10174	10470
<p>1) When $\tau \neq 0$, Combined Stress Intensity = largest absolute magnitude of either $\frac{1}{2} [\sigma_x + \sigma_y \pm \sqrt{(\sigma_x - \sigma_y)^2 + 4\tau^2}]$ or $\sqrt{(\sigma_x - \sigma_y)^2 + 4\tau^2}$</p> <p>2) When $\tau = 0$, Combined Stress Intensity = largest absolute magnitude of σ_x, σ_y, or $(\sigma_x - \sigma_y)$</p>										

Allowable Stress:
min[Yield Multiplier * min(Host Yield, Attachment Yield), Stress Multiplier * min(Host Stress, Attachment Stress)] = 30050
The Largest Vessel Combined Stress Intensity = 26204 PSI

SPX Flow Technology USA, Inc.Customer: **Southern Company**
Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Loading Summary

Type	Starting Elevation (in.)	Ending Elevation (in.)	Density (lb./Ft^3)	Thickness (in.)	Wind Diameter (in.)	Wind Pressure (lb./ft.^2)
Liquid	104.0000	200.0000	46.0000	-	-	100.00

Attachment Summary

Attachment No.	Elevation (in.)	Description	Attachment Weight (lb.)	Horizontal Force (lb.)	Attachment Moment (in.-lb.)
1	140.0000	misc. dssc, bed	12000.00	0	0

SPX Flow Technology USA, Inc.

Customer: **Southern Company**
Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Tower Analysis

Operating Pressurized Condition - Occasional Loads - Seismic Case 5

First Period of Natural Vibration for Operating conditions: 0.0298 seconds/cycle

ASCE 7-05 Seismic Design Information

0.2 s Spectral Response Accel (S_s):	0.163		
1 s Spectral Response Accel. (S_1):	0.099	Response Modification Factor (R):	3.000
Seismic Use Group:	II	Site Class:	C
Total seismic shear force at the base (V):	4410 lb.		

ASCE 7-05 Seismic Analysis

Segment Type	Segment No.	Wi Weight (lb.)	Hi Mid-Elevation (in.)	Wi * Hi (Ft.-lb.)	Segment Seismic Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Seismic Stress (PSI)
Head	1	16317.98	214.8750	292194	792	0	1485	1
Shell	2	15102.87	183.5000	230948	626	792	4407	3
Shell	3	15102.87	154.5000	194449	527	1418	8682	5
Attach	4t	12000.00	140.0000	140000	379			
Shell	4	18748.39	122.0000	190609	517	2324	16687	11
Shell	5	344.38	103.0000	2956	8	2841	17161	11
Head	6t	8140.83	102.0000	69197	188			
Skirt	6	1107.18	89.2500	8235	22	3036	23644	67
Skirt	7	1107.18	63.7500	5882	16	3058	30166	85
Skirt	8	1107.18	38.2500	3529	10	3074	36712	104
Skirt	9	1107.18	12.7500	1176	3	3084	43270	122
Total		90186.03		1139175				

Stress Summary

Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	1	7947	0	14	7934	-15
Shell	2	3	8017	0	23	7997	-26
Shell	3	5	8017	0	31	7991	-37
Shell	4	11	8017	0	63	7964	-74
Shell	5	11	8017	0	64	7964	-74
Skirt	6	67	0	0	687	-620	-754
Skirt	7	85	0	0	696	-611	-781
Skirt	8	104	0	0	705	-601	-808
Skirt	9	122	0	0	714	-591	-836

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Positive values represent tensile stress, and negative values represent compressive.
Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7934	20000	0.3967	-15	-13100	0.0011	-15875	0.0009
Shell	2	7997	20000	0.3998	-26	-13100	0.0020	-15875	0.0016
Shell	3	7991	20000	0.3995	-37	-13100	0.0028	-15875	0.0023
Shell	4	7964	20000	0.3982	-74	-13100	0.0056	-15875	0.0046
Shell	5	7964	20000	0.3982	-74	-13100	0.0057	-15875	0.0047
Skirt	6	-620	-8053	0.0770	-754	-8053	0.0936	-9008	0.0837
Skirt	7	-611	-8053	0.0758	-781	-8053	0.0970	-9008	0.0867
Skirt	8	-601	-8053	0.0746	-808	-8053	0.1004	-9008	0.0898
Skirt	9	-591	-8053	0.0734	-836	-8053	0.1038	-9008	0.0928

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Operating Pressurized Condition - Occasional Loads - Wind Case 5

ASCE 7-05 Wind Design Information

Basic Wind Speed (V):	100 MPH	Wind Exposure type:	C
Importance factor (I):	1.15	Force (shape) coefficient (C _f):	0.7
Total deflection of top of tower is :	0.0006 in.		
Total deflection dist. per 100 ft height is :	0.0033 in.		

First Period of Natural Vibration for Operating conditions: 0.0298 seconds/cycle

ASCE 7-05 Wind Analysis

Segment Type	Segment No.	Segment Height (in.)	Wind Load Diameter (in.)	Projection Area (Sq. Ft.)	(Kz) Exposure Factor	Wind Pressure (lb./ft. ²)	Wind Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Wind Stress (PSI)
Head	1	33.7500	132.0000	30.9375	0.90	16.7	516	0	726	0
Shell	2	29.0000	132.0000	26.5833	0.87	16.1	429	516	2492	2
Shell	3	29.0000	132.0000	26.5833	0.85	15.8	419	945	5284	3
Shell	4	36.0000	132.0000	33.0000	0.85	15.8	520	1365	10158	6
Shell	5	2.0000	132.0000	1.8333	0.85	15.8	29	1885	10475	7
Skirt	6	25.5000	132.0000	23.3750	0.85	15.8	369	1914	14934	42
Skirt	7	25.5000	132.0000	23.3750	0.85	15.8	369	2283	20176	57
Skirt	8	25.5000	132.0000	23.3750	0.85	15.8	369	2651	26201	74
Skirt	9	25.5000	132.0000	23.3750	0.85	15.8	369	3020	33010	93

Stress Summary

Segment Type	Segment No.	S _m Attachments With Wind or Seismic (PSI)	S _p Internal Pressure Stress (PSI)	S _p External Pressure Stress (PSI)	S _w Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	7947	0	14	7934	-14
Shell	2	2	8015	0	22	7995	-24
Shell	3	3	8015	0	31	7988	-34
Shell	4	6	8015	0	62	7960	-68
Shell	5	7	8015	0	62	7960	-69
Skirt	6	42	0	0	672	-630	-714
Skirt	7	57	0	0	680	-623	-737
Skirt	8	74	0	0	689	-615	-763
Skirt	9	93	0	0	698	-604	-791

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons

Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7934	20000	0.3967	-14	-13100	0.0011	-15875	0.0009
Shell	2	7995	20000	0.3997	-24	-13100	0.0018	-15875	0.0015
Shell	3	7988	20000	0.3994	-34	-13100	0.0026	-15875	0.0021
Shell	4	7960	20000	0.3980	-68	-13100	0.0052	-15875	0.0043
Shell	5	7960	20000	0.3980	-69	-13100	0.0053	-15875	0.0043
Skirt	6	-630	-8053	0.0782	-714	-8053	0.0887	-9008	0.0793
Skirt	7	-623	-8053	0.0774	-737	-8053	0.0916	-9008	0.0819
Skirt	8	-615	-8053	0.0764	-763	-8053	0.0948	-9008	0.0847
Skirt	9	-604	-8053	0.0750	-791	-8053	0.0982	-9008	0.0878

Operating Pressurized Condition - Sustained Loads

First Period of Natural Vibration for Operating conditions: 0.0298 seconds/cycle

Moment Analysis							
Segment Type	Segment No.	Weight (lb.)	Elevation (in.)	Segment Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Stress (PSI)
Head	1	16317.98	214.8750	0	0	0	0
Shell	2	15102.87	183.5000	0	0	0	0
Shell	3	15102.87	154.5000	0	0	0	0
Attach	4t	12000.00	140.0000	0			
Shell	4	18748.39	122.0000	0	0	0	0
Shell	5	344.38	103.0000	0	0	0	0
Head	6t	8140.83	102.0000	0			
Skirt	6	1107.18	89.2500	0	0	0	0
Skirt	7	1107.18	63.7500	0	0	0	0
Skirt	8	1107.18	38.2500	0	0	0	0
Skirt	9	1107.18	12.7500	0	0	0	0
Total		90186.03					

Stress Summary							
Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	7947	0	14	7933	-14
Shell	2	0	8015	0	22	7993	-22
Shell	3	0	8015	0	31	7985	-31
Shell	4	0	8015	0	62	7954	-62
Shell	5	0	8015	0	62	7953	-62
Skirt	6	0	0	0	672	-672	-672
Skirt	7	0	0	0	680	-680	-680
Skirt	8	0	0	0	689	-689	-689
Skirt	9	0	0	0	698	-698	-698

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7933	20000	0.3967	-14	-13100	0.0011	-15875	0.0009
Shell	2	7993	20000	0.3997	-22	-13100	0.0017	-15875	0.0014
Shell	3	7985	20000	0.3992	-31	-13100	0.0023	-15875	0.0019
Shell	4	7954	20000	0.3977	-62	-13100	0.0047	-15875	0.0039
Shell	5	7953	20000	0.3977	-62	-13100	0.0047	-15875	0.0039
Skirt	6	-672	-8053	0.0834	-672	-8053	0.0834	-9008	0.0746
Skirt	7	-680	-8053	0.0845	-680	-8053	0.0845	-9008	0.0755
Skirt	8	-689	-8053	0.0856	-689	-8053	0.0856	-9008	0.0765
Skirt	9	-698	-8053	0.0866	-698	-8053	0.0866	-9008	0.0774

Operating Unpressurized Condition - Occasional Loads - Seismic Case 5

First Period of Natural Vibration for Operating Unpressurized conditions: 0.0298 seconds/cycle

ASCE 7-05 Seismic Design Information

0.2 s Spectral Response Accel (S_s):	0.163		
1 s Spectral Response Accel. (S_1):	0.099	Response Modification Factor (R):	3.000
Seismic Use Group:	II	Site Class:	C
Total seismic shear force at the base (V):	4410 lb.		

ASCE 7-05 Seismic Analysis

Segment Type	Segment No.	Wi Weight (lb.)	Hi Mid-Elevation (in.)	Wi * Hi (Ft.-lb.)	Segment Seismic Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Seismic Stress (PSI)
Head	1	16317.98	214.8750	292194	792	0	1485	1
Shell	2	15102.87	183.5000	230948	626	792	4407	3
Shell	3	15102.87	154.5000	194449	527	1418	8682	5
Attach	4t	12000.00	140.0000	140000	379			
Shell	4	18748.39	122.0000	190609	517	2324	16687	11
Shell	5	344.38	103.0000	2956	8	2841	17161	11
Head	6t	8140.83	102.0000	69197	188			
Skirt	6	1107.18	89.2500	8235	22	3036	23644	67
Skirt	7	1107.18	63.7500	5882	16	3058	30166	85
Skirt	8	1107.18	38.2500	3529	10	3074	36712	104
Skirt	9	1107.18	12.7500	1176	3	3084	43270	122
Total		90186.03		1139175				

Stress Summary

Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	1	0	0	14	-13	-15
Shell	2	3	70	0	23	50	-26
Shell	3	5	70	0	31	44	-37
Shell	4	11	70	0	63	17	-74
Shell	5	11	70	0	64	17	-74
Skirt	6	67	0	0	687	-620	-754
Skirt	7	85	0	0	696	-611	-781
Skirt	8	104	0	0	705	-601	-808
Skirt	9	122	0	0	714	-591	-836

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Positive values represent tensile stress, and negative values represent compressive.
Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	-13	-13100	0.0010	-15	-13100	0.0011	-15875	0.0009
Shell	2	50	20000	0.0025	-26	-13100	0.0020	-15875	0.0016
Shell	3	44	20000	0.0022	-37	-13100	0.0028	-15875	0.0023
Shell	4	17	20000	0.0009	-74	-13100	0.0056	-15875	0.0046
Shell	5	17	20000	0.0008	-74	-13100	0.0057	-15875	0.0047
Skirt	6	-620	-8053	0.0770	-754	-8053	0.0936	-9008	0.0837
Skirt	7	-611	-8053	0.0758	-781	-8053	0.0970	-9008	0.0867
Skirt	8	-601	-8053	0.0746	-808	-8053	0.1004	-9008	0.0898
Skirt	9	-591	-8053	0.0734	-836	-8053	0.1038	-9008	0.0928

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Operating Unpressurized Condition - Occasional Loads - Wind Case 5

ASCE 7-05 Wind Design Information

Basic Wind Speed (V):	100 MPH	Wind Exposure type:	C
Importance factor (I):	1.15	Force (shape) coefficient (C _f):	0.7
Total deflection of top of tower is :	0.0006 in.		
Total deflection dist. per 100 ft height is :	0.0033 in.		

First Period of Natural Vibration for Operating Unpressurized conditions: 0.0298 seconds/cycle

ASCE 7-05 Wind Analysis

Segment Type	Segment No.	Segment Height (in.)	Wind Load Diameter (in.)	Projection Area (Sq. Ft.)	(Kz) Exposure Factor	Wind Pressure (lb./ft. ²)	Wind Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Wind Stress (PSI)
Head	1	33.7500	132.0000	30.9375	0.90	16.7	516	0	726	0
Shell	2	29.0000	132.0000	26.5833	0.87	16.1	429	516	2492	2
Shell	3	29.0000	132.0000	26.5833	0.85	15.8	419	945	5284	3
Shell	4	36.0000	132.0000	33.0000	0.85	15.8	520	1365	10158	6
Shell	5	2.0000	132.0000	1.8333	0.85	15.8	29	1885	10475	7
Skirt	6	25.5000	132.0000	23.3750	0.85	15.8	369	1914	14934	42
Skirt	7	25.5000	132.0000	23.3750	0.85	15.8	369	2283	20176	57
Skirt	8	25.5000	132.0000	23.3750	0.85	15.8	369	2651	26201	74
Skirt	9	25.5000	132.0000	23.3750	0.85	15.8	369	3020	33010	93

Stress Summary

Segment Type	Segment No.	S _m Attachments With Wind or Seismic (PSI)	S _p Internal Pressure Stress (PSI)	S _p External Pressure Stress (PSI)	S _w Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	0	0	14	-13	-14
Shell	2	2	68	0	22	47	-24
Shell	3	3	68	0	31	41	-34
Shell	4	6	68	0	62	13	-68
Shell	5	7	68	0	62	13	-69
Skirt	6	42	0	0	672	-630	-714
Skirt	7	57	0	0	680	-623	-737
Skirt	8	74	0	0	689	-615	-763
Skirt	9	93	0	0	698	-604	-791

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons

Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	-13	-13100	0.0010	-14	-13100	0.0011	-15875	0.0009
Shell	2	47	20000	0.0024	-24	-13100	0.0018	-15875	0.0015
Shell	3	41	20000	0.0020	-34	-13100	0.0026	-15875	0.0021
Shell	4	13	20000	0.0006	-68	-13100	0.0052	-15875	0.0043
Shell	5	13	20000	0.0006	-69	-13100	0.0053	-15875	0.0043
Skirt	6	-630	-8053	0.0782	-714	-8053	0.0887	-9008	0.0793
Skirt	7	-623	-8053	0.0774	-737	-8053	0.0916	-9008	0.0819
Skirt	8	-615	-8053	0.0764	-763	-8053	0.0948	-9008	0.0847
Skirt	9	-604	-8053	0.0750	-791	-8053	0.0982	-9008	0.0878

Operating Unpressurized Condition - Sustained Loads

First Period of Natural Vibration for Operating Unpressurized conditions: 0.0298 seconds/cycle

Moment Analysis							
Segment Type	Segment No.	Weight (lb.)	Elevation (in.)	Segment Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Stress (PSI)
Head	1	16317.98	214.8750	0	0	0	0
Shell	2	15102.87	183.5000	0	0	0	0
Shell	3	15102.87	154.5000	0	0	0	0
Attach	4t	12000.00	140.0000	0			
Shell	4	18748.39	122.0000	0	0	0	0
Shell	5	344.38	103.0000	0	0	0	0
Head	6t	8140.83	102.0000	0			
Skirt	6	1107.18	89.2500	0	0	0	0
Skirt	7	1107.18	63.7500	0	0	0	0
Skirt	8	1107.18	38.2500	0	0	0	0
Skirt	9	1107.18	12.7500	0	0	0	0
Total		90186.03					

Stress Summary							
Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	0	0	14	-14	-14
Shell	2	0	68	0	22	46	-22
Shell	3	0	68	0	31	37	-31
Shell	4	0	68	0	62	7	-62
Shell	5	0	68	0	62	6	-62
Skirt	6	0	0	0	672	-672	-672
Skirt	7	0	0	0	680	-680	-680
Skirt	8	0	0	0	689	-689	-689
Skirt	9	0	0	0	698	-698	-698

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	-14	-13100	0.0011	-14	-13100	0.0011	-15875	0.0009
Shell	2	46	20000	0.0023	-22	-13100	0.0017	-15875	0.0014
Shell	3	37	20000	0.0019	-31	-13100	0.0023	-15875	0.0019
Shell	4	7	20000	0.0003	-62	-13100	0.0047	-15875	0.0039
Shell	5	6	20000	0.0003	-62	-13100	0.0047	-15875	0.0039
Skirt	6	-672	-8053	0.0834	-672	-8053	0.0834	-9008	0.0746
Skirt	7	-680	-8053	0.0845	-680	-8053	0.0845	-9008	0.0755
Skirt	8	-689	-8053	0.0856	-689	-8053	0.0856	-9008	0.0765
Skirt	9	-698	-8053	0.0866	-698	-8053	0.0866	-9008	0.0774

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Empty Pressurized Condition - Occasional Loads - Seismic Case 5

First Period of Natural Vibration for Empty Pressurized conditions: 0.0228 seconds/cycle

ASCE 7-05 Seismic Design Information

0.2 s Spectral Response Accel (S_s):	0.163		
1 s Spectral Response Accel. (S_1):	0.099	Response Modification Factor (R):	3.000
Seismic Use Group:	II	Site Class:	C
Total seismic shear force at the base (V):	2408 lb.		

ASCE 7-05 Seismic Analysis

Segment Type	Segment No.	Wi Weight (lb.)	Hi Mid-Elevation (in.)	Wi * Hi (Ft.-lb.)	Segment Seismic Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Seismic Stress (PSI)
Head	1	8140.83	214.8750	145772	423	0	794	1
Shell	2	4993.53	183.5000	76359	222	423	2174	1
Shell	3	4993.53	154.5000	64292	187	645	4034	3
Attach	4t	12000.00	140.0000	140000	407			
Shell	4	6198.86	122.0000	63022	183	1238	8115	5
Shell	5	344.38	103.0000	2956	9	1421	8352	5
Head	6t	8140.83	102.0000	69197	201			
Skirt	6	1107.18	89.2500	8235	24	1631	11852	33
Skirt	7	1107.18	63.7500	5882	17	1655	15392	43
Skirt	8	1107.18	38.2500	3529	10	1672	18960	54
Skirt	9	1107.18	12.7500	1176	3	1682	22539	64
Total		49240.68		580420				

Stress Summary

Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	1	7947	0	14	7934	-15
Shell	2	1	7947	0	23	7926	-24
Shell	3	3	7947	0	31	7918	-34
Shell	4	5	7947	0	63	7889	-68
Shell	5	5	7947	0	64	7889	-69
Skirt	6	33	0	0	363	-330	-397
Skirt	7	43	0	0	372	-329	-416
Skirt	8	54	0	0	381	-327	-434
Skirt	9	64	0	0	390	-326	-453

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Positive values represent tensile stress, and negative values represent compressive.
Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7934	20000	0.3967	-15	-13100	0.0011	-15875	0.0009
Shell	2	7926	20000	0.3963	-24	-13100	0.0018	-15875	0.0015
Shell	3	7918	20000	0.3959	-34	-13100	0.0026	-15875	0.0021
Shell	4	7889	20000	0.3945	-68	-13100	0.0052	-15875	0.0043
Shell	5	7889	20000	0.3944	-69	-13100	0.0053	-15875	0.0043
Skirt	6	-330	-8053	0.0410	-397	-8053	0.0493	-9008	0.0440
Skirt	7	-329	-8053	0.0408	-416	-8053	0.0516	-9008	0.0461
Skirt	8	-327	-8053	0.0406	-434	-8053	0.0539	-9008	0.0482
Skirt	9	-326	-8053	0.0405	-453	-8053	0.0563	-9008	0.0503

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Empty Pressurized Condition - Occasional Loads - Wind Case 5

ASCE 7-05 Wind Design Information

Basic Wind Speed (V):	100 MPH	Wind Exposure type:	C
Importance factor (I):	1.15	Force (shape) coefficient (C _f):	0.7
Total deflection of top of tower is :	0.0006 in.		
Total deflection dist. per 100 ft height is :	0.0033 in.		

First Period of Natural Vibration for Empty Pressurized conditions: 0.0228 seconds/cycle

ASCE 7-05 Wind Analysis

Segment Type	Segment No.	Segment Height (in.)	Wind Load Diameter (in.)	Projection Area (Sq. Ft.)	(Kz) Exposure Factor	Wind Pressure (lb./ft. ²)	Wind Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Wind Stress (PSI)
Head	1	33.7500	132.0000	30.9375	0.90	16.7	516	0	726	0
Shell	2	29.0000	132.0000	26.5833	0.87	16.1	429	516	2492	2
Shell	3	29.0000	132.0000	26.5833	0.85	15.8	419	945	5284	3
Shell	4	36.0000	132.0000	33.0000	0.85	15.8	520	1365	10158	6
Shell	5	2.0000	132.0000	1.8333	0.85	15.8	29	1885	10475	7
Skirt	6	25.5000	132.0000	23.3750	0.85	15.8	369	1914	14934	42
Skirt	7	25.5000	132.0000	23.3750	0.85	15.8	369	2283	20176	57
Skirt	8	25.5000	132.0000	23.3750	0.85	15.8	369	2651	26201	74
Skirt	9	25.5000	132.0000	23.3750	0.85	15.8	369	3020	33010	93

Stress Summary

Segment Type	Segment No.	S _m Attachments With Wind or Seismic (PSI)	S _p Internal Pressure Stress (PSI)	S _p External Pressure Stress (PSI)	S _w Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	7947	0	14	7934	-14
Shell	2	2	7947	0	22	7927	-24
Shell	3	3	7947	0	31	7920	-34
Shell	4	6	7947	0	62	7892	-68
Shell	5	7	7947	0	62	7892	-69
Skirt	6	42	0	0	355	-313	-397
Skirt	7	57	0	0	364	-307	-421
Skirt	8	74	0	0	372	-298	-446
Skirt	9	93	0	0	381	-288	-474

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons

Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7934	20000	0.3967	-14	-13100	0.0011	-15875	0.0009
Shell	2	7927	20000	0.3963	-24	-13100	0.0018	-15875	0.0015
Shell	3	7920	20000	0.3960	-34	-13100	0.0026	-15875	0.0021
Shell	4	7892	20000	0.3946	-68	-13100	0.0052	-15875	0.0043
Shell	5	7892	20000	0.3946	-69	-13100	0.0053	-15875	0.0043
Skirt	6	-313	-8053	0.0389	-397	-8053	0.0493	-9008	0.0441
Skirt	7	-307	-8053	0.0381	-421	-8053	0.0522	-9008	0.0467
Skirt	8	-298	-8053	0.0370	-446	-8053	0.0554	-9008	0.0496
Skirt	9	-288	-8053	0.0357	-474	-8053	0.0589	-9008	0.0526

Empty Pressurized Condition - Sustained Loads

First Period of Natural Vibration for Empty Pressurized conditions: 0.0228 seconds/cycle

Moment Analysis							
Segment Type	Segment No.	Weight (lb.)	Elevation (in.)	Segment Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Stress (PSI)
Head	1	8140.83	214.8750	0	0	0	0
Shell	2	4993.53	183.5000	0	0	0	0
Shell	3	4993.53	154.5000	0	0	0	0
Attach	4t	12000.00	140.0000	0			
Shell	4	6198.86	122.0000	0	0	0	0
Shell	5	344.38	103.0000	0	0	0	0
Head	6t	8140.83	102.0000	0			
Skirt	6	1107.18	89.2500	0	0	0	0
Skirt	7	1107.18	63.7500	0	0	0	0
Skirt	8	1107.18	38.2500	0	0	0	0
Skirt	9	1107.18	12.7500	0	0	0	0
Total		49240.68					

Stress Summary							
Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	7947	0	14	7933	-14
Shell	2	0	7947	0	22	7925	-22
Shell	3	0	7947	0	31	7917	-31
Shell	4	0	7947	0	62	7886	-62
Shell	5	0	7947	0	62	7885	-62
Skirt	6	0	0	0	355	-355	-355
Skirt	7	0	0	0	364	-364	-364
Skirt	8	0	0	0	372	-372	-372
Skirt	9	0	0	0	381	-381	-381

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7933	20000	0.3967	-14	-13100	0.0011	-15875	0.0009
Shell	2	7925	20000	0.3963	-22	-13100	0.0017	-15875	0.0014
Shell	3	7917	20000	0.3958	-31	-13100	0.0023	-15875	0.0019
Shell	4	7886	20000	0.3943	-62	-13100	0.0047	-15875	0.0039
Shell	5	7885	20000	0.3943	-62	-13100	0.0047	-15875	0.0039
Skirt	6	-355	-8053	0.0441	-355	-8053	0.0441	-9008	0.0394
Skirt	7	-364	-8053	0.0452	-364	-8053	0.0452	-9008	0.0404
Skirt	8	-372	-8053	0.0462	-372	-8053	0.0462	-9008	0.0413
Skirt	9	-381	-8053	0.0473	-381	-8053	0.0473	-9008	0.0423

SPX Flow Technology USA, Inc.

Job No: 7404131

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Date Printed: 5/16/2012

Empty Unpressurized Condition - Occasional Loads - Seismic Case 5

First Period of Natural Vibration for Empty Unpressurized conditions: 0.0228 seconds/cycle

ASCE 7-05 Seismic Design Information

0.2 s Spectral Response Accel (S_s):	0.163		
1 s Spectral Response Accel. (S_1):	0.099	Response Modification Factor (R):	3.000
Seismic Use Group:	II	Site Class:	C
Total seismic shear force at the base (V):	2408 lb.		

ASCE 7-05 Seismic Analysis

Segment Type	Segment No.	Wi Weight (lb.)	Hi Mid-Elevation (in.)	Wi * Hi (Ft.-lb.)	Segment Seismic Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Seismic Stress (PSI)
Head	1	8140.83	214.8750	145772	423	0	794	1
Shell	2	4993.53	183.5000	76359	222	423	2174	1
Shell	3	4993.53	154.5000	64292	187	645	4034	3
Attach	4t	12000.00	140.0000	140000	407			
Shell	4	6198.86	122.0000	63022	183	1238	8115	5
Shell	5	344.38	103.0000	2956	9	1421	8352	5
Head	6t	8140.83	102.0000	69197	201			
Skirt	6	1107.18	89.2500	8235	24	1631	11852	33
Skirt	7	1107.18	63.7500	5882	17	1655	15392	43
Skirt	8	1107.18	38.2500	3529	10	1672	18960	54
Skirt	9	1107.18	12.7500	1176	3	1682	22539	64
Total		49240.68		580420				

Stress Summary

Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	1	0	0	14	-14	-15
Shell	2	1	0	0	23	-21	-24
Shell	3	3	0	0	31	-29	-34
Shell	4	5	0	0	63	-58	-68
Shell	5	5	0	0	64	-58	-69
Skirt	6	33	0	0	363	-330	-397
Skirt	7	43	0	0	372	-329	-416
Skirt	8	54	0	0	381	-327	-434
Skirt	9	64	0	0	390	-326	-453

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Positive values represent tensile stress, and negative values represent compressive.
Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	-14	-13100	0.0010	-15	-13100	0.0011	-15875	0.0009
Shell	2	-21	-13100	0.0016	-24	-13100	0.0018	-15875	0.0015
Shell	3	-29	-13100	0.0022	-34	-13100	0.0026	-15875	0.0021
Shell	4	-58	-13100	0.0044	-68	-13100	0.0052	-15875	0.0043
Shell	5	-58	-13100	0.0045	-69	-13100	0.0053	-15875	0.0043
Skirt	6	-330	-8053	0.0410	-397	-8053	0.0493	-9008	0.0440
Skirt	7	-329	-8053	0.0408	-416	-8053	0.0516	-9008	0.0461
Skirt	8	-327	-8053	0.0406	-434	-8053	0.0539	-9008	0.0482
Skirt	9	-326	-8053	0.0405	-453	-8053	0.0563	-9008	0.0503

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Empty Unpressurized Condition - Occasional Loads - Wind Case 5

ASCE 7-05 Wind Design Information

Basic Wind Speed (V):	100 MPH	Wind Exposure type:	C
Importance factor (I):	1.15	Force (shape) coefficient (C _f):	0.7
Total deflection of top of tower is :	0.0006 in.		
Total deflection dist. per 100 ft height is :	0.0033 in.		

First Period of Natural Vibration for Empty Unpressurized conditions: 0.0228 seconds/cycle

ASCE 7-05 Wind Analysis

Segment Type	Segment No.	Segment Height (in.)	Wind Load Diameter (in.)	Projection Area (Sq. Ft.)	(Kz) Exposure Factor	Wind Pressure (lb./ft. ²)	Wind Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Wind Stress (PSI)
Head	1	33.7500	132.0000	30.9375	0.90	16.7	516	0	726	0
Shell	2	29.0000	132.0000	26.5833	0.87	16.1	429	516	2492	2
Shell	3	29.0000	132.0000	26.5833	0.85	15.8	419	945	5284	3
Shell	4	36.0000	132.0000	33.0000	0.85	15.8	520	1365	10158	6
Shell	5	2.0000	132.0000	1.8333	0.85	15.8	29	1885	10475	7
Skirt	6	25.5000	132.0000	23.3750	0.85	15.8	369	1914	14934	42
Skirt	7	25.5000	132.0000	23.3750	0.85	15.8	369	2283	20176	57
Skirt	8	25.5000	132.0000	23.3750	0.85	15.8	369	2651	26201	74
Skirt	9	25.5000	132.0000	23.3750	0.85	15.8	369	3020	33010	93

Stress Summary

Segment Type	Segment No.	S _m Attachments With Wind or Seismic (PSI)	S _p Internal Pressure Stress (PSI)	S _p External Pressure Stress (PSI)	S _w Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	0	0	14	-13	-14
Shell	2	2	0	0	22	-21	-24
Shell	3	3	0	0	31	-27	-34
Shell	4	6	0	0	62	-55	-68
Shell	5	7	0	0	62	-56	-69
Skirt	6	42	0	0	355	-313	-397
Skirt	7	57	0	0	364	-307	-421
Skirt	8	74	0	0	372	-298	-446
Skirt	9	93	0	0	381	-288	-474

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons

Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	-13	-13100	0.0010	-14	-13100	0.0011	-15875	0.0009
Shell	2	-21	-13100	0.0016	-24	-13100	0.0018	-15875	0.0015
Shell	3	-27	-13100	0.0021	-34	-13100	0.0026	-15875	0.0021
Shell	4	-55	-13100	0.0042	-68	-13100	0.0052	-15875	0.0043
Shell	5	-56	-13100	0.0042	-69	-13100	0.0053	-15875	0.0043
Skirt	6	-313	-8053	0.0389	-397	-8053	0.0493	-9008	0.0441
Skirt	7	-307	-8053	0.0381	-421	-8053	0.0522	-9008	0.0467
Skirt	8	-298	-8053	0.0370	-446	-8053	0.0554	-9008	0.0496
Skirt	9	-288	-8053	0.0357	-474	-8053	0.0589	-9008	0.0526

Empty Unpressurized Condition - Sustained Loads

First Period of Natural Vibration for Empty Unpressurized conditions: 0.0228 seconds/cycle

Moment Analysis							
Segment Type	Segment No.	Weight (lb.)	Elevation (in.)	Segment Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Stress (PSI)
Head	1	8140.83	214.8750	0	0	0	0
Shell	2	4993.53	183.5000	0	0	0	0
Shell	3	4993.53	154.5000	0	0	0	0
Attach	4t	12000.00	140.0000	0			
Shell	4	6198.86	122.0000	0	0	0	0
Shell	5	344.38	103.0000	0	0	0	0
Head	6t	8140.83	102.0000	0			
Skirt	6	1107.18	89.2500	0	0	0	0
Skirt	7	1107.18	63.7500	0	0	0	0
Skirt	8	1107.18	38.2500	0	0	0	0
Skirt	9	1107.18	12.7500	0	0	0	0
Total		49240.68					

Stress Summary							
Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	0	0	14	-14	-14
Shell	2	0	0	0	22	-22	-22
Shell	3	0	0	0	31	-31	-31
Shell	4	0	0	0	62	-62	-62
Shell	5	0	0	0	62	-62	-62
Skirt	6	0	0	0	355	-355	-355
Skirt	7	0	0	0	364	-364	-364
Skirt	8	0	0	0	372	-372	-372
Skirt	9	0	0	0	381	-381	-381

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	-14	-13100	0.0011	-14	-13100	0.0011	-15875	0.0009
Shell	2	-22	-13100	0.0017	-22	-13100	0.0017	-15875	0.0014
Shell	3	-31	-13100	0.0023	-31	-13100	0.0023	-15875	0.0019
Shell	4	-62	-13100	0.0047	-62	-13100	0.0047	-15875	0.0039
Shell	5	-62	-13100	0.0047	-62	-13100	0.0047	-15875	0.0039
Skirt	6	-355	-8053	0.0441	-355	-8053	0.0441	-9008	0.0394
Skirt	7	-364	-8053	0.0452	-364	-8053	0.0452	-9008	0.0404
Skirt	8	-372	-8053	0.0462	-372	-8053	0.0462	-9008	0.0413
Skirt	9	-381	-8053	0.0473	-381	-8053	0.0473	-9008	0.0423

SPX Flow Technology USA, Inc.

Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Test Condition

First Period of Natural Vibration for Test Condition conditions: 0.0301 seconds/cycle

Moment Analysis							
Segment Type	Segment No.	Weight (lb.)	Elevation (in.)	Segment Force (lb.)	Shear @ Top (lb.)	Moment @ Bottom (Ft.-lb.)	Stress (PSI)
Head	1	19236.87	214.8750	0	0	0	0
Shell	2	18684.91	183.5000	0	0	0	0
Shell	3	18684.91	154.5000	0	0	0	0
Shell	4	23195.06	122.0000	0	0	0	0
Shell	5	1288.61	103.0000	0	0	0	0
Head	6t	19236.87	102.0000	0			
Skirt	6	1107.18	89.2500	0	0	0	0
Skirt	7	1107.18	63.7500	0	0	0	0
Skirt	8	1107.18	38.2500	0	0	0	0
Skirt	9	1107.18	12.7500	0	0	0	0
Total		104755.96					

Stress Summary							
Segment Type	Segment No.	Sm Attachments With Wind or Seismic (PSI)	Sp Internal Pressure Stress (PSI)	Sp External Pressure Stress (PSI)	Sw Weight Stress (PSI)	Maximum Tensile Stress (PSI)	Maximum Compressive Stress (PSI)
Head	1	0	7613	0	13	7599	-13
Shell	2	0	7613	0	21	7591	-21
Shell	3	0	7613	0	29	7583	-29
Shell	4	0	7613	0	40	7573	-40
Shell	5	0	7613	0	40	7572	-40
Skirt	6	0	0	0	654	-654	-654
Skirt	7	0	0	0	661	-661	-661
Skirt	8	0	0	0	668	-668	-668
Skirt	9	0	0	0	676	-676	-676

Positive values represent tensile stress, and negative values represent compressive.

Stress ratios greater than 1.0 represent overstressed conditions.

Stress Comparisons									
Segment Type	Segment No.	Maximum Tensile Stress (PSI)	Allowable Stress (PSI)	Maximum Tensile Stress Ratio	Maximum Compressive Stress (PSI)	Allowable Stress (PSI)	Maximum Compressive Stress Ratio	Critical Buckling Stress (PSI)	Critical Buckling Ratio
Head	1	7599	20000	0.3800	-13	-13100	0.0010	-15875	0.0008
Shell	2	7591	20000	0.3796	-21	-13100	0.0016	-15875	0.0013
Shell	3	7583	20000	0.3792	-29	-13100	0.0023	-15875	0.0019
Shell	4	7573	20000	0.3786	-40	-13100	0.0030	-15875	0.0025
Shell	5	7572	20000	0.3786	-40	-13100	0.0031	-15875	0.0025
Skirt	6	-654	-8053	0.0812	-654	-8053	0.0812	-11507	0.0568
Skirt	7	-661	-8053	0.0821	-661	-8053	0.0821	-11507	0.0575
Skirt	8	-668	-8053	0.0830	-668	-8053	0.0830	-11507	0.0581
Skirt	9	-676	-8053	0.0839	-676	-8053	0.0839	-11507	0.0587

SPX Flow Technology USA, Inc.Customer: **Southern Company**
Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

MDMT Report by Components

Design MDMT is 10 °F

Component	Material	Curve	Pressure	MDMT
132" OD ROLLED SHELL x 96" LG	SA-516 Gr. 70	D	350.00 PSI	-34 °F
Smls LWN (FVC) - 20"	SA-105	B	350.00 PSI	-89 °F
SMLS LWN (FVC) - 20" MANWAY IN SHELL	SA-105	B	350.00 PSI	-89 °F
SMLS LWN (FVC) - 8" DES DRAIN NOZZLE	SA-105	B	350.00 PSI	-103 °F
SMLS LWN (FVC) - 8" DES DRAIN NOZZLE	SA-105	B	350.00 PSI	-103 °F
SMLS LWN (FVC) - 1-1/2" INSTRUMENT C	SA-105	B	350.00 PSI	-135 °F
132" OD HEAD, 2:1 ELLIP, BOTTOM	SA-516 Gr. 70	D	350.00 PSI	-31 °F
SMLS WELD STUB (FVC) - 20"	SA-105	B	350.00 PSI	-89 °F
132" OD HEAD, 2:1 ELLIP, TOP	SA-516 Gr. 70	D	350.00 PSI	-31 °F
Smls LWN (FVC) - 16"	SA-105	B	350.00 PSI	-89 °F
SMLS LWN (FVC) - 20" INLET NOZZLE IN T	SA-105	B	350.00 PSI	-89 °F
SMLS LWN (FVC) - 8" DES FILL NOZZLE I	SA-105	B	350.00 PSI	-103 °F
SMLS LWN (FVC) - 2" RELIEF VALVE NOZ	SA-105	B	350.00 PSI	-133 °F

Component with highest MDMT: 132" OD HEAD, 2:1 ELLIP, BOTTOM.

Computed MDMT = -31 °F

The required design MDMT of 10 °F has been met or exceeded for the calculated MDMT values.

ASME Flanges Are Not Included in MDMT Calculations.

SPX Flow Technology USA, Inc.Customer: **Southern Company**
Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Summary Information

	<u>Dry Weight</u>	<u>Flooded Weight</u>
Shell	16530.31 lb.	61911.49 lb.
Head	16281.66 lb.	38462.14 lb.
Nozzle	1971.76 lb.	1971.76 lb.
ASME Flange	1667.00 lb.	1667.00 lb.
Skirt	4428.71 lb.	4428.71 lb.
Totals	<hr/> 40879.44 lb.	<hr/> 108441.11 lb.
	<u>Volume</u>	
Shell	5432.23 Gal.	
Head	2659.53 Gal.	
Nozzle	52.35 Gal.	
Totals	<hr/> 8144.11 Gal.	
	<u>Area</u>	
Shell	276.46 Sq. Ft.	
Head	269.21 Sq. Ft.	
Nozzle	48.82 Sq. Ft.	
Skirt	293.74 Sq. Ft.	
Totals	<hr/> 888.23 Sq. Ft.	

SPX Flow Technology USA, Inc.

Customer: **Southern Company**
Job No: 7404131

Vessel Number: PG-1080 2080

Date Printed: 5/16/2012

Hydrostatic Test Information Par. UG-99(b)
Gauge at Top

Component	Const.	x	STest	/	SDesign	x	Pressure	=	Component Hydro Test Pressure
132" OD HEAD, 2:1 ELLI	1.3	x	20000	/	20000	x	350.00	=	455.00
132" OD HEAD, 2:1 ELLI	1.3	x	20000	/	20000	x	350.00	=	455.00
132" OD ROLLED SHELL x	1.3	x	20000	/	20000	x	350.00	=	455.00
SMLS LWN (FVC) - 1-1/2	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS LWN (FVC) - 2" RE	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS LWN (FVC) - 20" I	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS LWN (FVC) - 20" M	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS LWN (FVC) - 8" DE	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS LWN (FVC) - 8" DE	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS LWN (FVC) - 8" DE	1.3	x	20000	/	19800	x	350.00	=	459.60
SMLS WELD STUB (FVC) -	1.3	x	20000	/	19800	x	350.00	=	459.60

Calculated Test Pressure: 455.00 PSI

Special Notes:

This calculation assumes one chamber.

This calculation is limited by the lowest component pressure per chamber.