

MANUFACTURER'S DATA BOOK

MISSISSIPPI POWER

DEKALB, MISSISSIPPI

KEMPER COUNTY 1 – IGCC PROJECT

P.O. NO: 17496-0001

REQ. NO: MC202

ITEM NO: HX0052

**WASTEWATER AMMONIA PURIFIER
REBOILER**

1-15" x 168" NEN EXCHANGER

ENERGY JOB NO: X-7926

KBR	
ACCEPTANCE FOR ENGINEERING USE	
THIS DOCUMENT IS:	
ACCEPTED	(Code 1) _____
ACCEPTED WITH COMMENTS	(Code 2) _____
NOT ACCEPTED	(Code 3) _____
NOT REVIEWED	(Code 4) <u> X </u>
ACCEPTANCE DOES NOT RELIEVE SUPPLIER FROM FURNISHING MATERIAL IN CONFORMANCE WITH ORDER. REFER TO SDR-1 FOR FULL DEFINITION OF ACCEPTANCE CONDITIONS.	
DISCIPLINE	EXCH
DATE	BY MW 24SEP12



Energy Exchanger Company

Energy Industrial Park
1844 North Garnett Road/Tulsa, OK 74116-1612
(918) 437-3000/FAX (918) 437-7144

Southern Company Generation Kemper County
MM209778 0 Unit 1

ENERGY EXCHANGER COMPANY PO: MPC17496-0001
X-7926-MDB Rev: 0
IGCC - GASIFIER - MULTIPAGE - SHELL AND TUBE HEAT EXCHANGE - WAST

Mississippi Power
Kemper County 1 – IGCC Project
P.O. No: 17496-0001
Req. No: MC202
Item No: HX0052
Energy Job No: X-7926

MANUFACTURER'S DATA BOOK

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4	Heat Number Location Map Material Heat Number Index Mill Test Reports
5	Heat Treat Charts NDE/Weld Map NDE/Weld Index Radiographic Reader Sheets Certificate of Magnetic Particle Examination Brinell Hardness Report
6	Certificate of Weld/Support Pad Air Test Hydrostatic Test Chart Certificate of Hydrostatic Test Certificate of Nitrogen Purge Paint Check List
7	Spare Parts List Installation, Operation & Maintenance Manual

SECTION 1

117 Cornerstone Ct.
Sapulpa, OK 74066Richard Duke –Owner
Office: 918-224-7472
Cell: 918-640-7911
Email: rmduke@cox.net**I.Q.S. Work Order 556
Inspection Shipping Release Notice 11-BB**

Client: SOUTHERN CO. Purchase Order #: MPC-17645; MPC17496 & MPC- 17497

EWO: MPC9117.T61401 Supplier: ENERGY EXCHANGERS

Shop Order # SO # 7926 Location: TULSA, OK

Project: KEMPER COUNTY IGCC UNIT Supplier Ref. #:

Item #	Description	Qty Ordered	Qty Released	Balance
	<u>15-168 NEN exchanger</u> TAG # HX0052	1	1	0
	THE UNIT & FINAL DOCUMENTATION ARE COMPLETE			

This does not relieve the Vendor of any responsibility to ensure that the material or equipment meets all contractual requirements. This notice is issued according to IQS' best knowledge, experience, codes and specifications available to us during this inspection. Neither IQS nor any member of its inspection force is to be held liable for the acceptance of or payment for released items, or the default or negligence on the part of the vendors' obligation to supply quality equipment and materials or breach the conditions of his contract with the purchaser. Actual acceptance of equipment/material is predicated on the purchaser's site inspection, operational integrity and the fulfillment of all contractual warranties connected thereto.

Date: 8/4/12

Bill Bronaugh
IQS

SECTION 2

Energy Exchanger Company



HEAT EXCHANGER SPECIFICATION SHEET

Ref. No.: 17496-0001 (REQ. No.: MC202)

REV. 0 AS PURCHASED

REV	LN #		
2	1		
0	2		
5	3	Customer	SOUTHERN COMPANY SERVICES
5	4	Address	
5	5	Plant Location	KEMPER COUNTY, MS
6	6	Service of Unit	WASTEWATER AMMONIA PURIFIER REBOILER
1	7	Size	15-168
5	8	Surf./Unit	☐ Gross / ☒ Eff 501 ft²
9	9	PERFORMANCE OF ONE UNIT	
10	10	Fluid Allocation	
11	11	Fluid Name	IP STEAM
12	12	Fluid Quantity, Total	9547
13	13	Vapor (In/Out)	
14	14	Liquid	
15	15	Steam	9547
16	16	Water	9547
17	17	Noncondensable	
18	18	Temperature (In/Out)	°F 475. 419.4
19	19	Density (Vapor/Liquid)	Lb/Ft³ 38.33 0.7730 40.57
20	20	Viscosity	cP 0.144 0.0170 0.1303
21	21	Molecular Weight	
22	22	Molecular Weight, Noncondensable	
23	23	Specific Heat	Btu/lb-F 1.1110 0.4710 1.1608
24	24	Thermal Conductivity	Btu/hr-ft-F 0.2199 0.0213 0.2489
25	25	Latent Heat	Btu/lb
26	26	Inlet Pressure	psia 310.
27	27	Velocity	ft/sec 15.35 6.05
28	28	Pressure Drop, Allow./Calc.	psi 5.00 / 0.89
29	29	Fouling Resistance (Min.)	Hr-Ft²-F/Btu 0.0005 0.002
30	30	Heat Exchanged	7963000 Btu/hr; M.T.D. (Corrected) 102.64 °F
31	31	Transfer Rate, Service	154.85 Btu/hr-ft²-F; Clean 450.92 Btu/hr-ft²-F
32	32	CONSTRUCTION OF ONE SHELL	
33	33	Design/Test Pressure	psig 455 / FV / 595 (4) 320 / 420 (4)
34	34	Design Temperature (Max/Min)	°F 515/10 385/10
35	35	No. Passes Per Shell	1 1
36	36	Corrosion Allowance	inch 0.125 0.125
37	37	Connections	In 1 - 4" 300RF 1 - 3" 300RF
38	38	Size & Rating	Out 1 - 2" 300RF 1 - 6" 300RF
39	39	Balance Line	1 - 3/4" 300RF
40	40		
41	41	Tube No.	185
42	42	Tube Type	OD 0.75 In; Thk. ☒ Min ☐ Avg 0.083 In; Length 14. ft; Pitch 0.9375 In 30°
43	43	Shell	SA-106B ☒ ID ☐ OD 15. Material SA-179
44	44	Channel or Bonnet	SA-333-6 (PWHT) Channel Cover SA-266-4N ☐ Integ. ☐ Remov.
45	45	Tubesheet-Stationary	SA-266-4N Tubesheet - Floating
46	46	Floating Head Cover	N/A Impingement Protection PLATE ON BUNDLE (SA-36)
47	47	Baffles-Cross	11 - SA-36 Type H SEG % Cut (☒ Dia ☐ Area) 31. Spacing c/c 13.75 Inlet In
48	48	Baffles-Long	N/A Seal Type
49	49	Supports - Tube	N/A U-Bend
50	50	Bypass Seal Arrangement	N/A Tube-Tubesheet Joint TWO GROOVES & EXPANDED
51	51	Expansion Joint	NONE Type
52	52	py² - Inlet Nozzle	Bundle Entrance Bundle Exit
53	53	Gaskets - Shell Side	N/A Tube Side 321 SS DJ FLEXIBLE GRAPHITE FILLED
54	54	- Floating Head	N/A
55	55	Code Requirements	ASME SECTION VIII, DIV. 1 (STAMP: YES) TEMA Class R
56	56	Weight/Shell	4,300# Filled With Water 5,500# Bundle N/A
57	57	REMARKS	
58	58	(1) DELETED.	
59	59	(2) UNIT IS VERTICAL THERMOSYPHON REBOILER. INLET PIPING EQUIVALENT LENGTH IS 60 FT. OUTLET PIPING EQUIVALENT LENGTH IS 30 FT. LIQUID STATIC HEAD IS 15 FT.	
60	60	(3) APPLY NACE FOR WET H2S SERVICE ON TUBE SIDE.	
61	61	(4) TUBE SIDE IS SUBJECT TO STEAM OUT AT 250°F AND 15 PSIG. STEAM OUT ON THE TUBE SIDE ALONE IS NOT ALLOWED. STEAM OUT SHELL SIDE FIRST WHILE SHELL IS STILL HOT, STEAM OUT TUBE SIDE.	
62	62		
63	63		
64	64		
65	65		

X:\ev-7926.xls

17'-0" OAL. ±

1'-4" 6 1/2" 1'-8" 4'-6" 10' 1'-2" 5 1/2" 7"

1 1/2" DIA. HOLE FOR LIFTING (ONE FOR EACH SUPPORT)

3/4" 16" DB

5/8" DIA. (11 THUD) HOLE FOR GROUNDING (THIS SUPPORT ONLY)

P.D.S. ELEV. 491.9'

IMP. PL

THIS END UP

SEE VIEW A-A AND SEC. B-B FOR TRUE ORIENTATION

SUPPORT LOADING:
FULL WIND: 456 LB
FULL SEISMIC: 387 LB

ALL LOADS AT EXCHANGER CL.

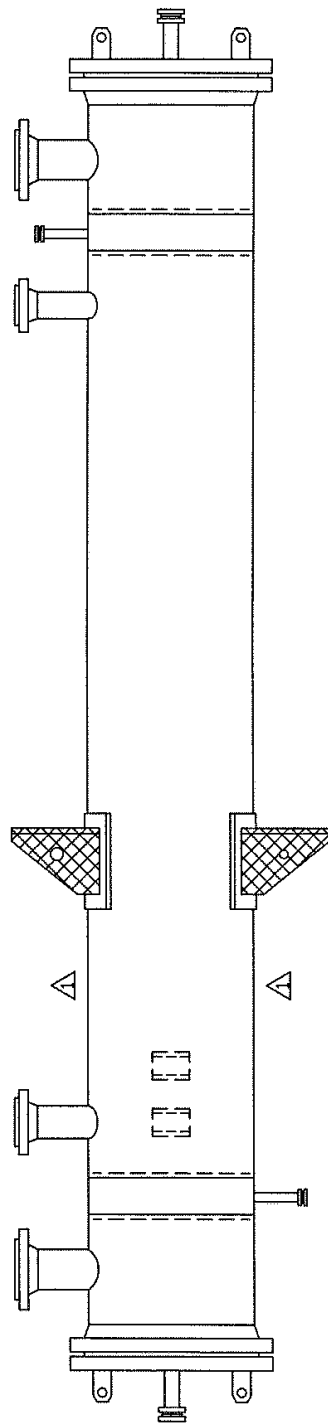
NOZZLE SCHEDULE

HK	IDT	DESCRIPTION	PROCESS	AUXILIARIES
S1	1 1/4"	300#RF VN	INLET	
S2	1 1/2"	300#RF LWN	OUTLET	
S3	1 3/4"	300#RF LWN	BAL. LINE	W/ BLIND
D2	1 3/4"	300#RF LWN	DRAIN	W/ BLIND
V2	1 3/4"	300#RF LWN	VENT	W/ BLIND
T1	1 3/4"	300#RF VN	INLET	
T2	1 1/2"	300#RF VN	OUTLET	
D1	1 1/2"	300#RF LWN	DRAIN	W/ BLIND
V1	1 1/2"	300#RF LWN	VENT	W/ BLIND

CERTIFIED CORRECT FOR FABRICATION
BY: *[Signature]* DATE: 8/7/12

VIEW A-A

SECTION B-B



COATING DATA SHEET - SYSTEM 5 APPLIES.
ABRASIVE BLAST PER SSPC-SP10 WITH AN ANCHOR PROFILE OF 1 TO 2 MILS.
PAINT WITH AMERON , PPG AMERCOTE, DINETCOTE 9 or 9HS (3.0-4.0 MILS DFT)

COATING DATA SHEET - SYSTEM 3 APPLIES.
ABRASIVE BLAST PER SSPC-SP10 WITH AN ANCHOR PROFILE OF 1 TO 2 MILS.
PAINT WITH AMERON, PPG AMERCOATE, DIMETCOATE 9 or 9HS (2.0-3.0 MILS DFT)
FINISH WITH AMERON, PPG AMERCOATE, PSX 892HS (1.0-2.0 MILS DFT) (COLOR: ALUMINUM)

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.O. NO: 17496-0001 REQ NO: MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBOILER
ITEM NO: HX0052

1. A DAILY INSPECTION REPORT SHALL BE MAINTAINED.

2. PAINT SPECIFICATION V42-ITS AND APPLICABLE COATING DATA SHEET SHALL BE FURNISHED TO THE PAINTER.

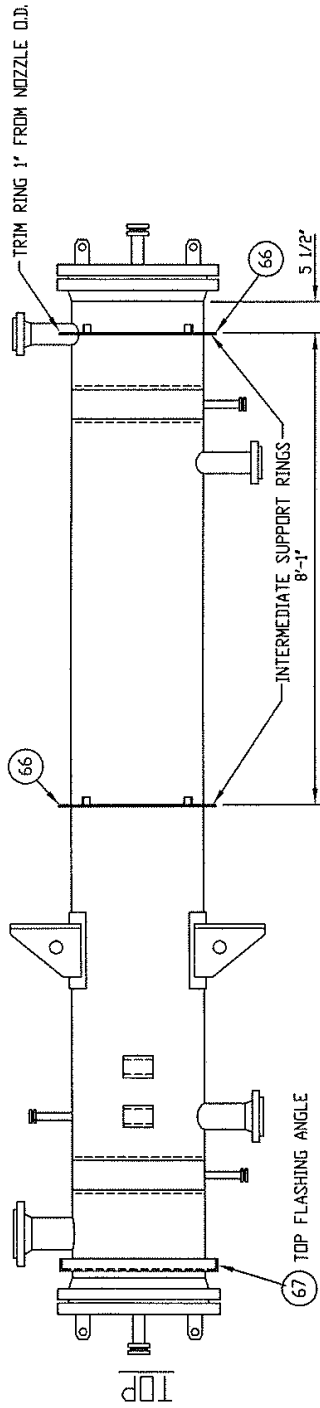
PAINT MAP

OWN, BY
S. MIANO
DATE 4-21-11

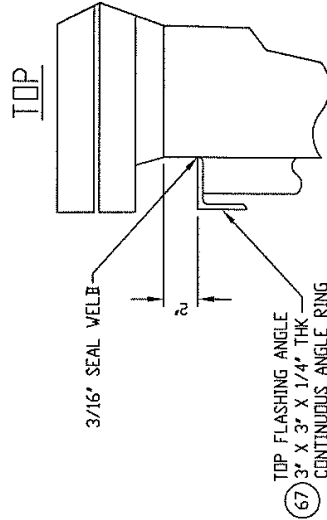
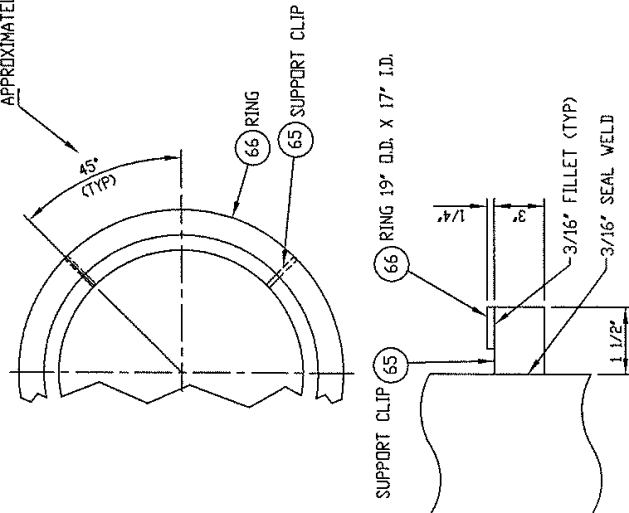
DWG. NO.

X-7926-AA

CUST.
REVISIONS



APPROXIMATELY 6 5/16" O.D. WRAP



TOP FLASHING ANGLE

CERTIFIED AS-BUILT

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
 P.O. NO: 17496-0001 REQ NO: MC202
 SERVICE: WASTEWATER AMMONIA PURIFIER RENDLER
 ITEM NO: HX0052

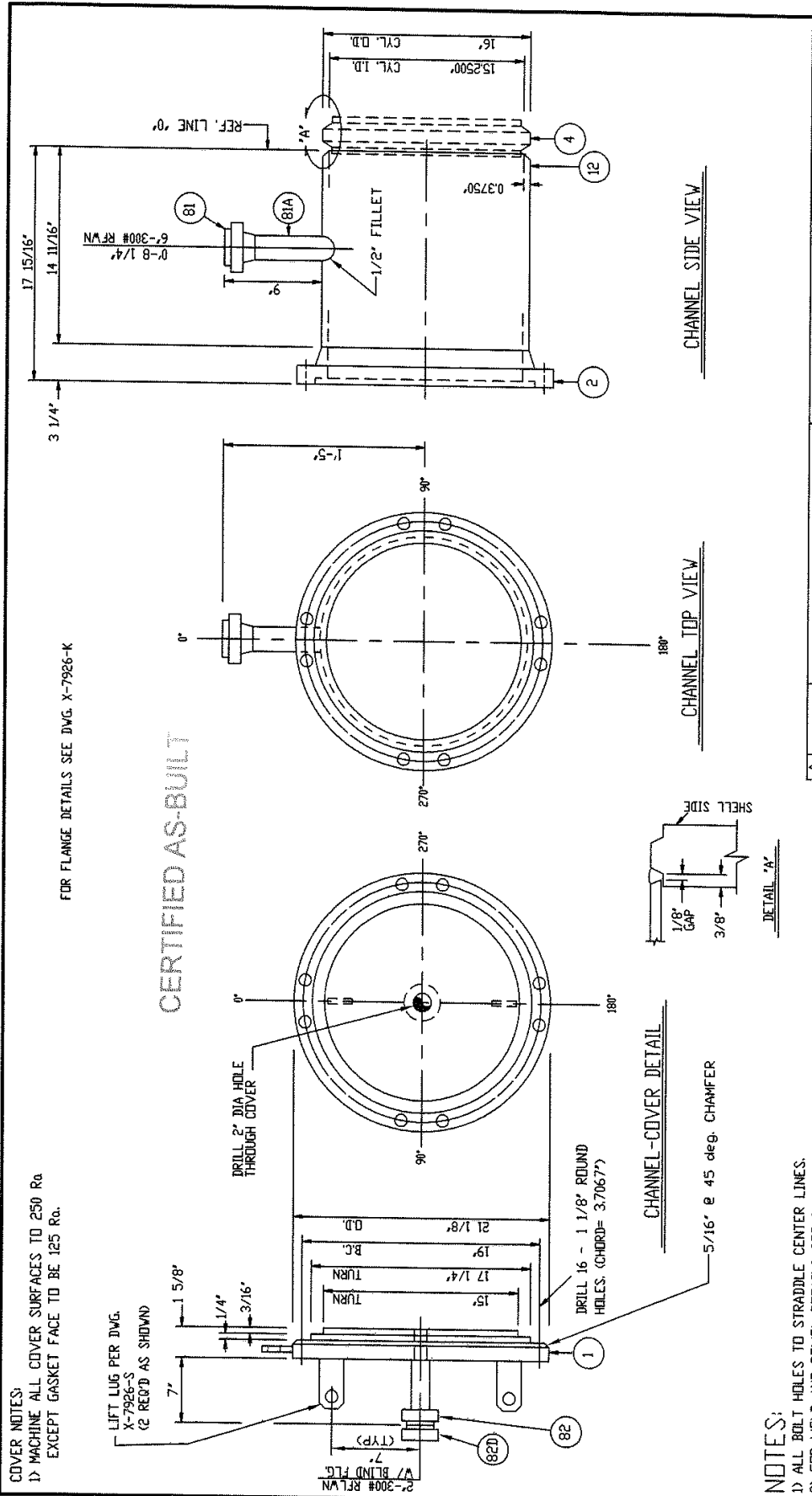
INSULATION SUPPORT DETAILS

DWN. BY: DMORRISON
 DATE: 8-17-11
 CKD. BY: MJD



Energy Exchanger Co.
 3841 W. Barrett St. Tulsa, Okla. 74126

NO.	DATE	REVISIONS	DWG. NO.	REV
			X-7926-AB	0



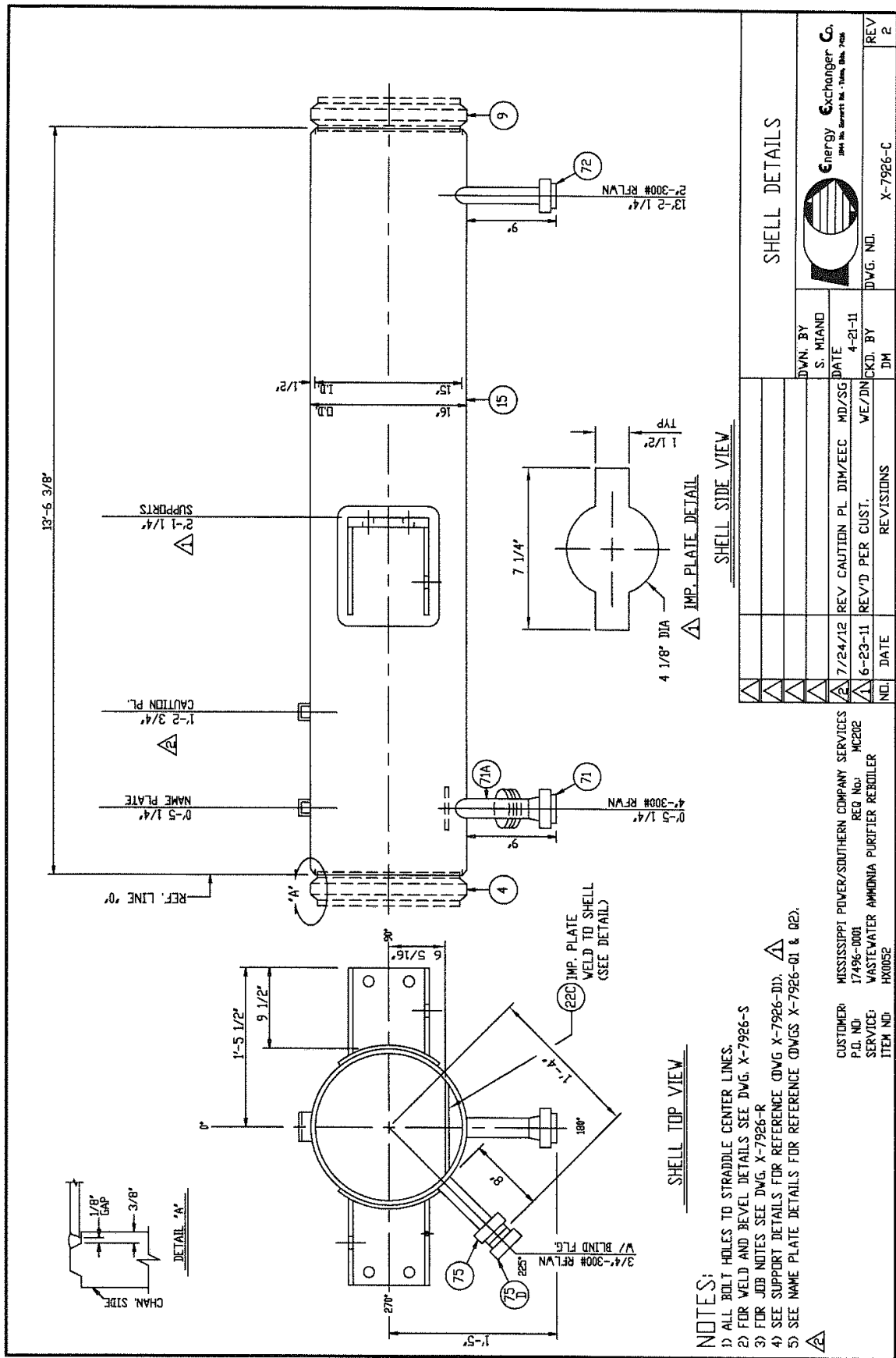
NOTES:

- ALL BOLT HOLES TO STRADDLE CENTER LINES.
- FOR WELD AND BEVEL DETAILS SEE DWG. X-7926-S
- FOR JOB NOTES SEE DWG. X-7926-R
- NOZZLE SWING DIMENSION = 36 1/8"

TOP CHANNEL DETAILS			
NO.	DATE	REVISIONS	
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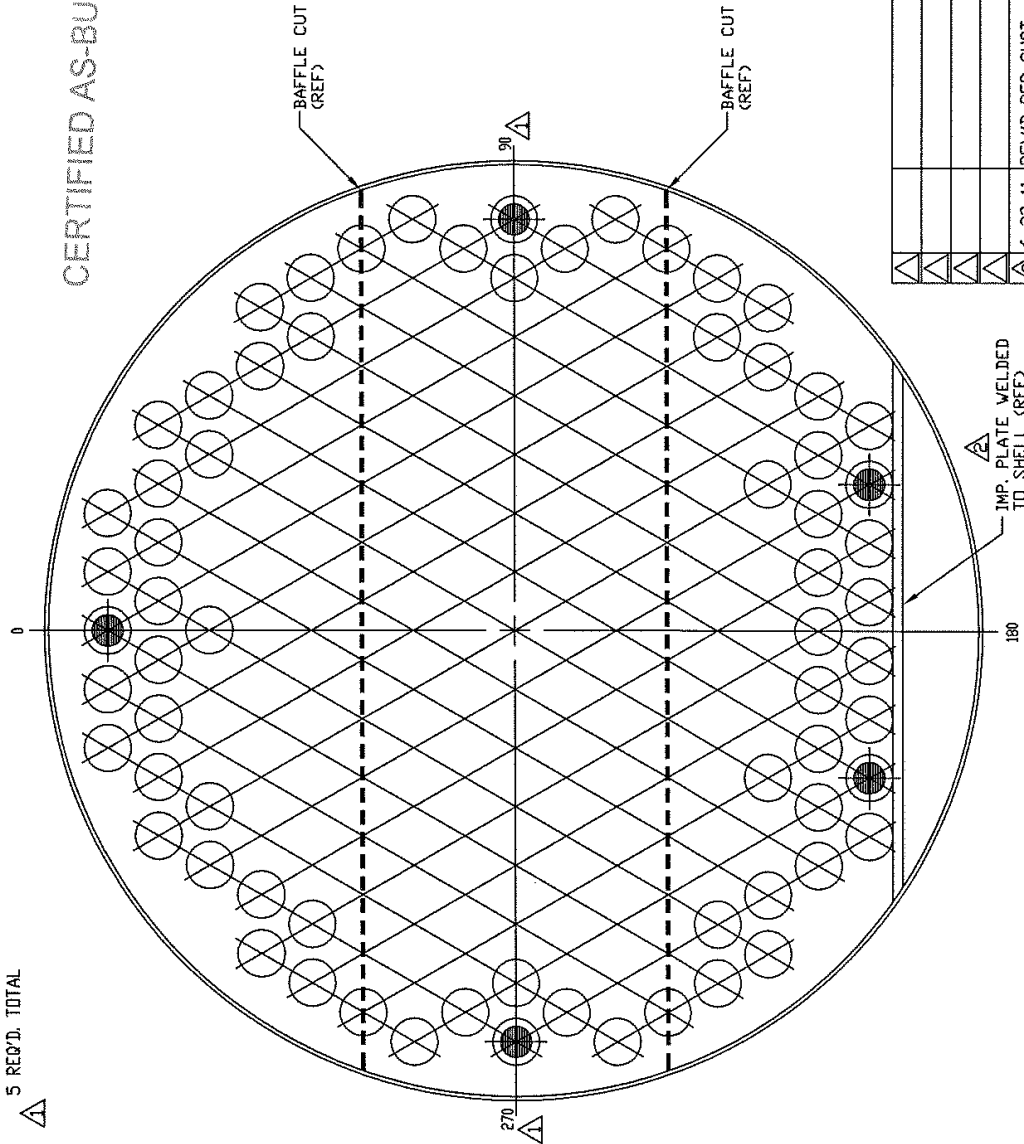
Energy Exchanger Co. 204 No. Liberty St. - Tulsa, Okla. 74103			
DWN. BY S. MIAND	DATE 4-21-11	DWG. NO. X-7926-B	REV 0
CHKD. BY [Signature]			

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
 P.D. NO: 17496-0001
 SERVICE: WASTEWATER AMMONIA PURIFIER REBOLTER
 ITEM NO: HA0052



TIE RODS.
 1/2" I.D.
 5 REQ'D. TOTAL

CERTIFIED AS-BUILT



NOTES:

- 1) DRILL AND REAM TUBE HOLES IN TUBESHEETS TO 0.7600" +0.002" -0.004" DIA. FOR 185 - 3/4" I.D. TUBES.
- 2) TUBE PITCH = 15/16" TRIANGLE.
- 3) NUMBER OF PASSES = 1
- 4) SEE DRAWING X-7926-F FOR TUBE GROOVING DETAILS.
- 5) BAFFLE I.D. = 14 7/8"
- 6) O.T.L. = 14 5/16"

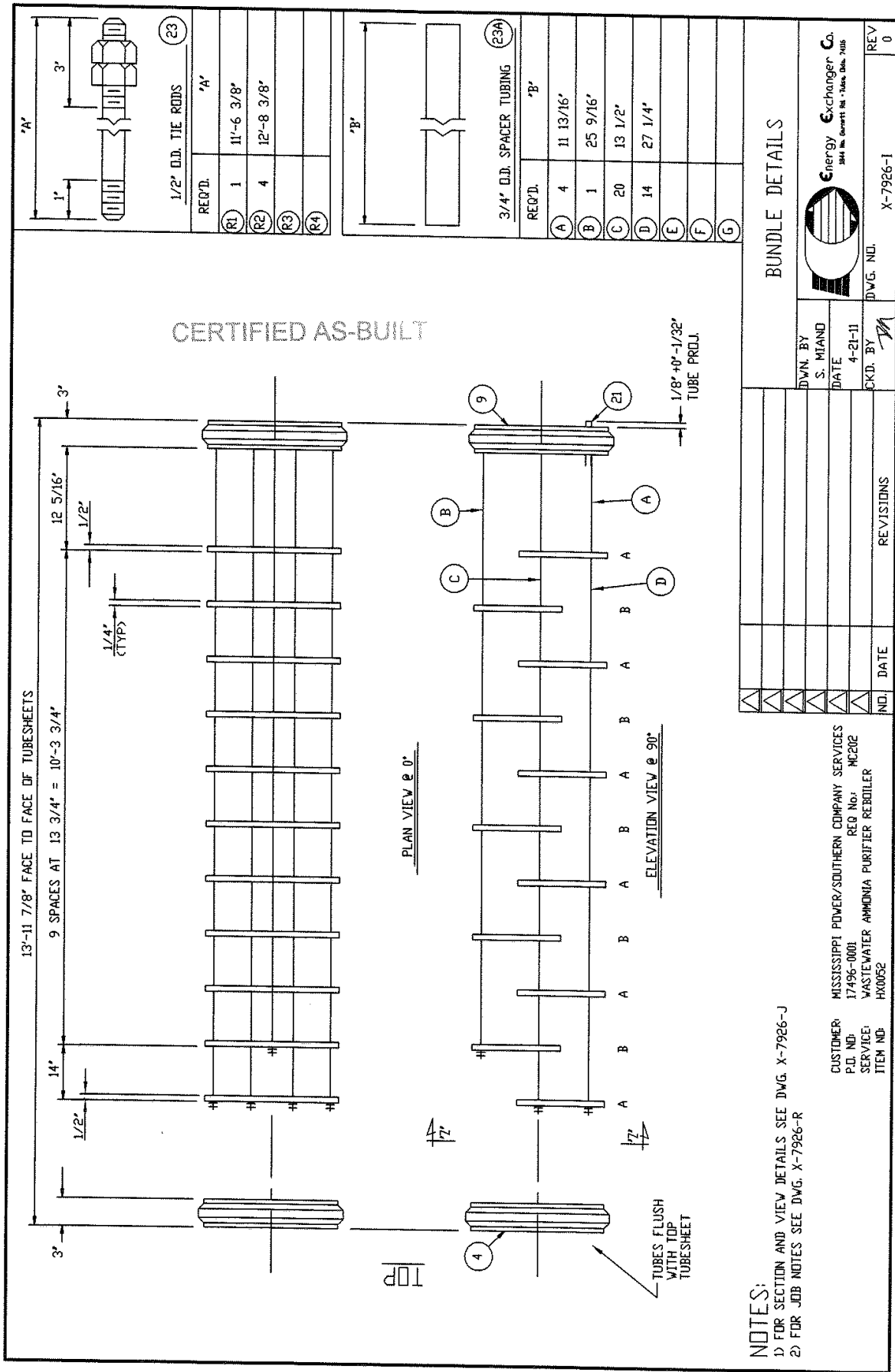
CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
 P.I. NO: 17496-0001 REQ NO: MC202
 SERVICE: WASTEWATER AMMONIA PURIFIER REBOLLER
 ITEM NO: HK0052

TUBE LAYOUT

NO.	DATE	REVISIONS	CKD. BY	DWG. NO.	REV
1	6-23-11	REV'D PER CUST.	MD	X-7926-H	2
2	4-11-11	REV'D PER CUST.	MD/DN		
3					
4					
5					
6					



Energy Exchanger Co.
 2044 W. Garrett Rd. Tulsa, Okla. 74128

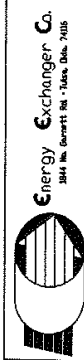


NOTES:

- 1) FOR SECTION AND VIEW DETAILS SEE DWG. X-7926-J
- 2) FOR JOB NOTES SEE DWG. X-7926-R

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
 P.O. NO: 17496-0001
 SERVICE: WASTEWATER AMMONIA PURIFIER REBUILDER
 ITEM NO: H00052

DOWN BY: S. MIAND
 DATE: 4-21-11
 CKD BY: [Signature]



BUNDLE DETAILS

NO.	DATE	REVISIONS	DWG. NO.	REV
			X-7926-1	0

CERTIFIED AS-BUILT

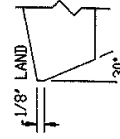
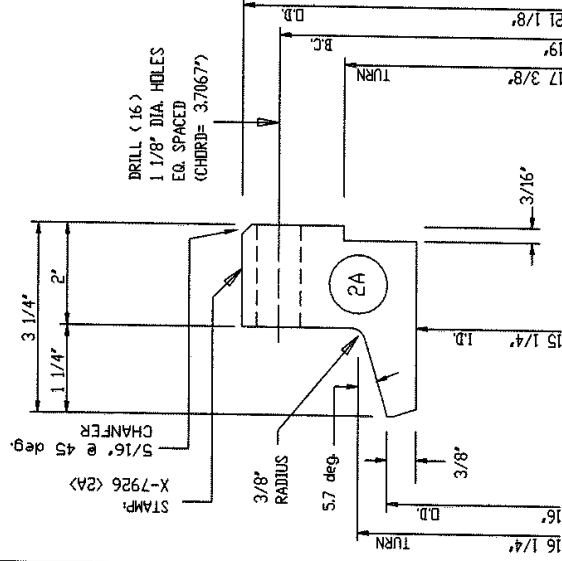
CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.I. NO: 17496-0001 REQ NO: MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBOILER
ITEM NO: HX0052

GIRTH FLANGES

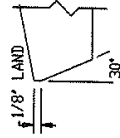
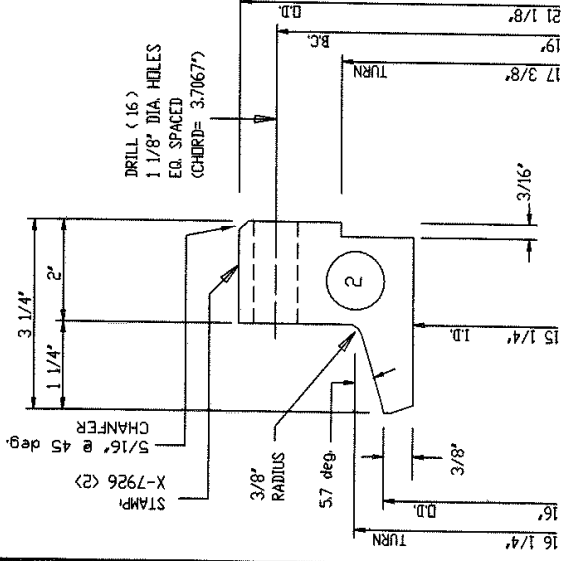
DOWN BY: S. MIAND
DATE: 4-21-11
CHKD BY: *[Signature]*
DWG. NO: X-7926-K
REV: 0



#2A GIRTH FLANGE (ONE) REQ'D.



#2 GIRTH FLANGE (ONE) REQ'D.

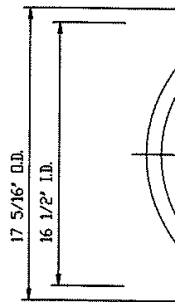


NOTES:

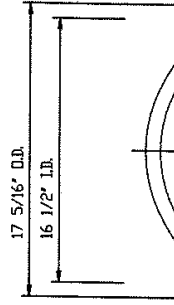
- 1) SEE DWG. X-7926-R FOR JOB NOTES.
- 2) MACHINE ALL SURFACES TO 250 Ra EXCEPT GASKET SURFACES MACHINE TO 125-250 Ra.

REVISIONS

NO. DATE



(1)-REQUIRED.
TOP CHANNEL COVER GASKET
1/8" THK. 321 STN. STL. D.J.
FLEXIBLE GRAPHITE FILLED



(1)-REQUIRED.
BOTTOM CHANNEL COVER GASKET
1/8" THK. 321 STN. STL. D.J.
FLEXIBLE GRAPHITE FILLED

CERTIFIED AS-BUILT

NOTES:

- 1) QUANTITIES SHOWN FOR ONE SET
- (2)-SETS REQUIRED TOTAL.
- ONE SET OF GASKETS TO BE USED IN EXCH. AND
- (1) SET TO BE BOXED, MARKED WITH
- P.I.D. & ITEM NO. & SHIPPED WITH EXCHANGER.
- 2) ALL JACKETED GASKETS TO HAVE
- LAP OF GASKET NEAR SIDE.

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.I.D. NO: 17496-0001 REQ NO: MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBOLER
ITEM NO: HX0052

GASKET DETAILS

NO.	DATE	REVISIONS	CKD. BY <i>MM</i>	DWG. NO. X-7926-D	REV 0
NO.	DATE	REVISIONS	CKD. BY <i>MM</i>	DWG. NO. X-7926-D	REV 0
NO.	DATE	REVISIONS	CKD. BY <i>MM</i>	DWG. NO. X-7926-D	REV 0
NO.	DATE	REVISIONS	CKD. BY <i>MM</i>	DWG. NO. X-7926-D	REV 0



SHOP TO ASSIGN AT COMPLETION OF FABRICATION

60

61

CLIP CORNERS & SEAL WELD TO NAME PLATE BRACKET

NAME PLATE BRACKET DETAIL

ONE - REQ'D.

CERTIFIED AS-BUILT

NAME PLATE DETAILS

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.D. NO. 17496-0001 REQ. NO. MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBUILDER
ITEM NO. HX0052

DOWN BY S. MIANO
DATE 4-21-11
CKD. BY DM
REV. NO. X-7926-Q1

6-23-11 REV'D PER CUST. VE/MA
NO. DATE REVISIONS

NAME PLATE DETAIL

ONE - REQ'D.

CERTIFIED BY
Energy Exchanger Co.
TULSA, OKLAHOMA

MAWP SHELL 455 P.S.I. 515 *F 595 TEST P.S.I.
MAWP TUBE 320 P.S.I. 385 *F 420 TEST P.S.I.
MAWP 15 P.S.I. 515 *F
MAWP 10 P.S.I. 320 P.S.I.
MIN. DESIGN METAL TEMP. 10 *F 455 P.S.I.
SERIAL NO. X-7926 YEAR BUILT 17496-0001 AREA: 210
SERVICE WASTEWATER AMMONIA PURIFIER REBUILDER
ITEM HX0052 SIZE 15-168 TYPE NEN
SHOP TEST: SHELLSIDE: 600 P.S.I.; TUBESIDE: 425 P.S.I.

CAUTION

PRESSURES AND TEMPERATURES SHOWN ON THE NAMEPLATE OF THIS HEAT EXCHANGER RELATE ONLY TO PRESSURE RETAINING CAPABILITY. CERTAIN COMPONENTS HAVE BEEN DESIGNED FOR SPECIFIC OPERATING CONDITIONS. DEVIATION FROM THESE CONDITIONS OR A CHANGE IN SERVICE REQUIRE A MECHANICAL DESIGN CHECK BE PERFORMED TO AVOID POSSIBLE DAMAGE TO THE HEAT EXCHANGER OR DANGER TO PERSONNEL.

CAUTION PLATE DETAIL

ONE - REQ'D.

WARNING

FIXED TUBESHEET HEAT EXCHANGER
INTRODUCE SHELL SIDE FIRST BEFORE INTRODUCING TUBE SIDE

STEAMOUT TUBE SIDE ALONE IS NOT ALLOWED FOR THIS UNIT.
STEAMOUT SHELL SIDE FIRST, WHILE SHELL SIDE IS HOT
STEAMOUT TUBE SIDE.

WARNING PLATE DETAIL

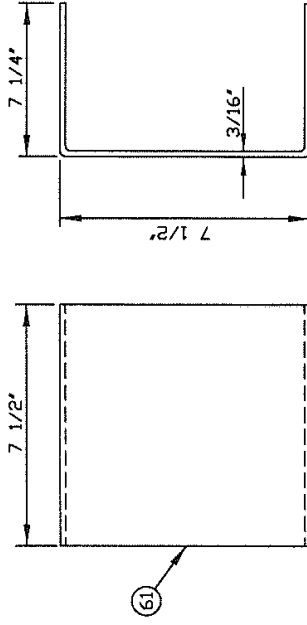
ONE - REQ'D.

CERTIFIED AS-BUILT

CLIP CORNERS & SEAL WELD
TO NAME PLATE BRACKET

(60)

USE BACK SIDE OF EEC NAME PLATE.
SPLIT IN HALF TO MAKE BOTH NAME PLATES
(FIT BOTH ON SAME BRACKET)



CAUTION/WARNING PLATE BRACKET DETAIL

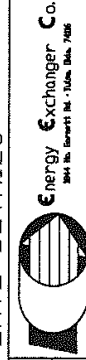
ONE - REQ'D.

CLIP CORNERS & SEAL WELD
TO NAME PLATE BRACKET

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.O. NO. 17496-0001 REQ. NO. MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBOTTLE
ITEM NO. H00052

CAUTION AND WARNING PLATE DETAILS

NO.	DATE	REV PER EEC	DM/ MS	DM	REV
1	7-30-12				1
REVISIONS					
DOWN BY					
S. MIAND					
DATE					
4-21-11					
CKD. BY					
DM					
DWG. NO.					
X-7926-02					



FABRICATION REQUIREMENTS

1. ALL BOLT HEADS TO STRADDLE NATURAL CENTER LINES.
2. NOZZLES SHALL NOT PROTRUDE BEYOND THE INSIDE SURFACE OF THE CHANNEL OR SHELL.
3. CHECK SLOTT FOR 10% ROUNDNESS. USE TWO DISCS EQUAL TO THE BOLT HEAD.
4. FLANGE FLATNESS TOLERANCE AND SURFACE FINISH SHALL BE MEASURED AFTER THE FLANGE HAS BEEN WELDED & PMT.
5. CHANNEL COVER, AND BODY FLANGE PERIPHERAL GASKET SURFACES SHALL HAVE A FLATNESS TOLERANCE OF 1/32" AND MEASURED WITH A DIAL GAUGE.

ALL FLAME-CUT BEVEL WELDS SHALL BE GROUND TO BRIGHT

7. METAL PRIOR TO WELDING.
8. MATCH MARKS REQUIRED ON CHANNEL COVER AND ALL TEMA FLS. WITH A SCORED LINE 1/8" DEEP AT THE 90° AND 270° POSITIONS. THE FOLLOWING PARTS SHALL BE STAMPED WITH MANUFACTURER'S SERIAL NUMBER & ITEM NUMBER: CHANNEL FLANGE, CHANNEL COVER.
9. TURBOSHEET AND TUBE HOLES SHALL BE CLEANED AND DECREASED DIMENSIONS PRIOR TO ASSEMBLY WITH A CHLORIDE FREE SOLVENT.
10. PRIOR TO ASSEMBLY, ALL INTERNALS SHALL BE CLEAN AND FREE OF LOOSE SLAG SCALE OR OTHER DEBRIS.
11. ANY DEFECTS SHALL HAVE THE DEFECT REMOVED AND CONFIRMED IF ORIGINALLY RT, THEN RT REPAIR AND ALSO MT.
12. WELD JOINT SURFACES AND A MINIMUM OF 1" OF THE INTERNAL AND EXTERNAL SURFACES OF THE ADJACENT BRASS METAL SHALL BE CLEAN AND FREE FROM PAINT, OIL, DIRT, SCALE, DOUBLES, AND OTHER FOREIGN MATERIAL DETRIMENTAL TO THE INTEGRITY OF THE WELD.
13. ALL DIMENSIONS TO BE HELD IN TEMA TOLERANCE.

MT. REQUIREMENTS

1. HT ALL CHANNEL WELDS AFTER PWHT.
2. HT ALL LIFTING LUG WELDS
4. TEMPORARY ATTACHMENT WELDS SHALL BE REMOVED, THE AREAS DRESSED, AND EXAMINED.

HEAT TREAT REQUIREMENTS

- TIP CHANNEL, BOTTOM CHANNEL AND TUBESHEETS

ALL MACHINED CONTACT SURFACES AND THREADED CONNECTIONS SHALL BE SUITABLY PROTECTED TO PREVENT SCALING OR LOSS OF FINISH DURING PWHT.

R.T. REQUIREMENTS

1. TUBESIDE SPOT RADIOGRAPHED
- AS A MINIMUM, SPOT RADIOGRAPH SHALL BE AS FOLLOWS:
 - A) ONE SPOT AT EACH CATERGORY A AND B WELDED JOINT, (GASLIZ EVIDENCE EXEMPT)
 - B) AT EACH START AND STOP OF WELDS MADE BY AUTOMATIC SUBMERSED-ARC PROCESS.
 - C) SPOTS SHALL BE AT LEAST 10" LG. OR FULL LENGTH IF WELD IS LESS THAN 10".
 - D) WELD POROSITY LIMITS FOR SPOT RADIOGRAPHS SHALL BE SAME AS LIMITS FOR FULL RADIOGRAPHED JOINTS.
 - E) AT EACH WELD PROCEDURE, WELDER/OPERATOR, & SHIFT.
 - F) SHELLSIDE AND RADIOGRAPH
 - G) IF FILM SHALL BE FINE GRAIN, HD, HIGH CONTRAST FILM (KODAK TYPE AA OR EQUIV) FILM DENSITY OF 2.0-3.5 DETERMINED BY SPECIMEN OR DENSITOMETER.

HARDNESS REQUIREMENTS

1. VELD HARDNESS TESTING, SHALL BE IN ACCORDANCE WITH THE FOLLOWING:
 - A) THE VELD METAL & HAZ OF PRESSURE-RETAINING VELS IN COMPONENTS
 - B) BORELL HARDNESS LIMITS SHALL BE 200 HAZ
 - C) HARDNESS SHALL BE DETERMINED USING A JOMI DIA BALL UNLESS OTHERWISE SPECIFIED.
 - D) ONE READING IN THE VELD METAL OF EACH CIRC. SEAM AND ONE READING IN EACH HEAT AFFECTED ZONE.
 - E) ONE READING IN THE VELD METAL OF NOZZLE TO EXCHANGER VELS AND ONE READING IN EACH HEAT AFFECTED ZONE IN SICK OF THE NOZZLES.
 - F) IF MORE THAN ONE WELDING PROCEDURE IS USED TO FABRICATE LONG OR CIRC VELS, HARDNESS READINGS SHALL BE MADE BY VELS DEPOSITED BY EACH PROCEDURE.
 - G) TESTING SHALL BE DONE ON ACCESSIBLE INSIDE SURFACES.

HYDROTESTING REQUIREMENTS

1. HOLD FOR AT LEAST ONE HOUR. A HYDROTEST CHART IS REQUIRED.
2. MINIMUM TEST WATER TEMP. = 40 deg F.
3. PAINT OR OTHER COATINGS SHALL NOT BE APPLIED OVER WELDS BEFORE FINAL HYDROTEST.
4. LEAK TEST WELD JOINTS WITH 15 PSI AIR AND SOAP SOLUTION, OBSERVING WEIN FLAMES FROM THE INSIDE AND OUTSIDE.
5. LEAKAGE AT MAIN FLANGES REQUIRES CUSTOMER NOTIFICATION IN WRITING. ALL CORRECTIVE ACTION SHALL BE SUSPENDED UNTIL APPROVAL IS RECEIVED BY THE CUSTOMER. THE REASON FOR THE LEAK SHALL BE DETERMINED, DOCUMENTED BY AN NCR AND REPAIRED TO THE SATISFACTION OF THE CUSTOMER.
6. IMMEDIATELY AFTER HYDRO, DRAIN AND DRY BY BLOWING WARM AIR NUT TO EXCEED 120°F.

SHIPPING REQUIREMENTS

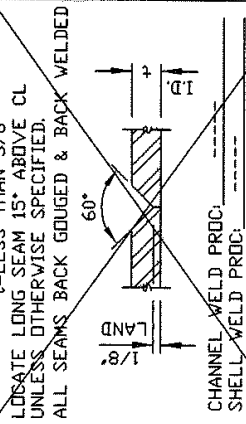
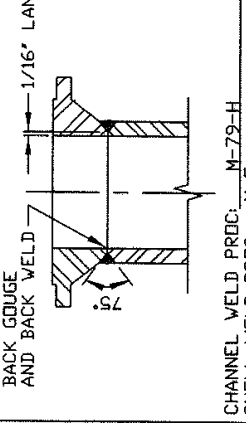
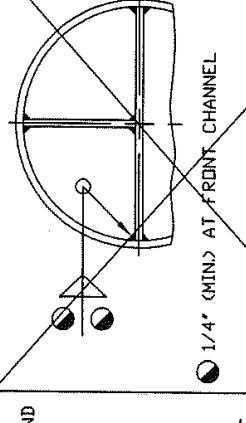
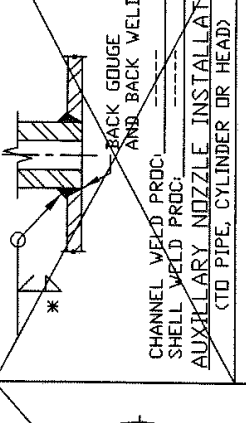
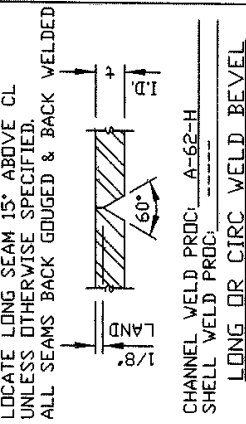
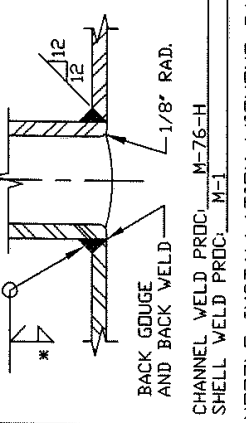
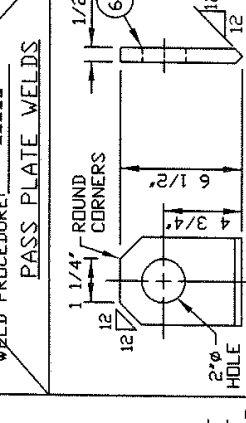
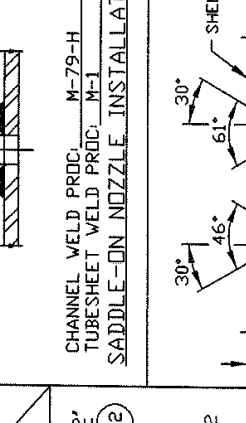
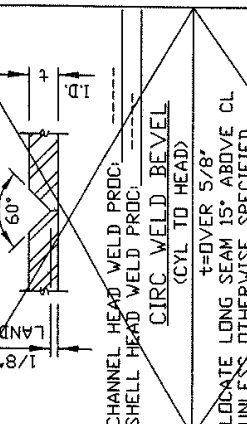
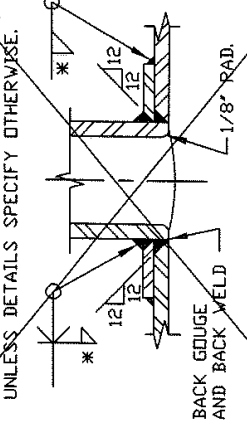
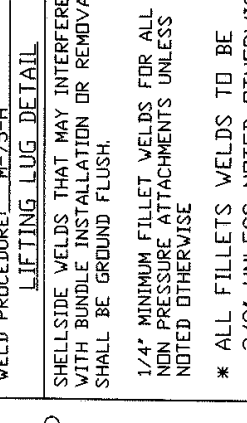
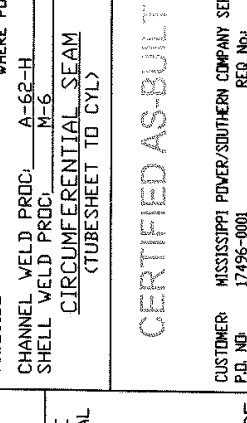
1. PAINT ON THE SIDE OF THE EXCHANGER AND EACH CRATE, ITEM NO, PU NO, AND GROSS WEIGHT. IDENTIFY SPARE PARTS AS "SPARES" AND ITEM NUMBER.
2. CRATES SHALL HAVE A MINIMUM OF 2 STEEL STRAPS ON APPROX. 24" CENTERS. A PACKING LIST SHALL BE PLACED INSIDE AND ATTACHED TO THE OUTSIDE.
3. ALL FLANGED GASKET SURFACES SHALL BE COATED WITH RUST VETO 342 RUST PREVENTIVE AND SHALL BE PROTECTED BY 1/2" STEEL COVERS, A MINIMUM OF (4) BOLTS, PROVIDE A 1/4" NPT DRAIN HOLE WITH PLUG IN BOTTOM COVERS.
4. EXPOSED THREADS OF BOLTS SHALL BE PROTECTED WITH RUST VETO 342 RUST PREVENTIVE TO PREVENT CORROSION DURING TESTING, SHIPPING, AND STORAGE. TAPPED HOLES SHALL BE PLUGGED WITH GREASE.
5. ATTACH A LARGE WEATHERPROOF TAG "CAUTION THIS EQUIPMENT HAS BEEN TREATED WITH A RUST PREVENTIVE AND NO PARTS SHOULD BE DISTURBED UNTIL READY TO BE PLACED IN SERVICE. IF PROTECTIVE FILM IS BROKEN, IT SHALL BE RESTORED FOR CONTINUED STORAGE"
6. MARK C-D-6 ORGY WITH A 3" CIRCLE ON HORIZ. & VERTICLE CENTERLINES.
7. MARK ON TWO SIDES OF EXCHANGER 180" APART IN 3" LETTERS "NO WELDING OR BURNING PERMITTED ON THIS EXCHANGER"
8. THE NORTH ORIENTATION SHALL BE CENTER PUNCHED AND MARKED IN A CONTRASTING COLOR ON THE OUTSIDE OF EXCHANGER, LOCATED APPROX. 12 INCHES ABOVE SUPPORT BASE PLATE. THE MARK SHALL BE A 6 INCH LINE AND NOTED AS "N".
9. SHIPPING SUPPORTS SHALL MATCH CYLINDER CURVATURE.
10. ON BOTH SIDES OF EXCHANGER REQUIRED FOR SHIPPING AND STORAGE. PAINT "WARNING-NITROGEN PURGED, DANGER OF SUFFOCATION UPON OPENING"

JOB NOTES

[illegible]

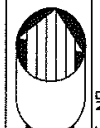
CUSTOMER:	MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.I. NO:	17496-0001
SERVICE:	WASTEWATER AMMONIA PURIFIER REBUILDER
ITEM NO:	HX0052
	REQ No.: MC202

Energy Exchanger Co.
3844 No. Everett Rd. • Tulsa, Okla. 74135

$t \leq \text{LESS THAN } 3/8"$ LOCATE LONG SEAM 15" ABOVE CL UNLESS OTHERWISE SPECIFIED. ALL SEAMS BACK GOUGED & BACK WELDED  CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ LONG OR CIRC WELD BEVEL (CYL TO FLG, OR CYL)	BACK GOUGE AND BACK WELD  CHANNEL WELD PROC: M-79-H SHELL WELD PROC: M-5 WELD-NECK FLANGE INSTALLATION	 1/4" (MIN) AT FRONT CHANNEL PASS PLATE WELDS TO BE FULL PENETRATION FOR 2" (MIN) FROM FACE OF GASKET SURFACE. WELD PROCEDURE: _____ PASS PLATE WELDS	 CHANNEL WELD PROC: M-79-H TUBESHEET WELD PROC: M-1 SADDLE-ON NOZZLE INSTALLATION
$t = 3/8"$ TO 5/8" LOCATE LONG SEAM 15" ABOVE CL UNLESS OTHERWISE SPECIFIED. ALL SEAMS BACK GOUGED & BACK WELDED  CHANNEL WELD PROC: A-62-H SHELL WELD PROC: _____ LONG OR CIRC WELD BEVEL (CYL TO FLG, OR CYL)	BACK GOUGE AND BACK WELD  CHANNEL WELD PROC: M-76-H SHELL WELD PROC: M-1 NOZZLE INSTALLATION WITHOUT PAD	 1 1/4" ROUND CORNERS 1/2" HOLE FULL PENETRATION WELD V/1/4" FILLET WELD WELD PROCEDURE: M-75-H LIFTING LUG DETAIL	 CHANNEL WELD PROC: A-62-H SHELL WELD PROC: M-6 CIRCUMFERENTIAL SEAM (TUBESHEET TO CYL)
$t = 7/16"$ TO 5/8" ALL SEAMS BACK GOUGED & BACK WELDED  CHANNEL HEAD WELD PROC: _____ SHELL HEAD WELD PROC: _____ CIRC WELD BEVEL (CYL TO HEAD)	BACK GOUGE AND BACK WELD  CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ NOZZLE INSTALLATION WITH PAD	1/4" NPT IN EACH WELD PAD OR SEC. OF WELD PAD. PLUG W/ STIFF GREASE, UNLESS DETAILS SPECIFY OTHERWISE.  CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ NOZZLE INSTALLATION WITH PAD	 CHANNEL WELD PROC: M-79-H SHELL WELD PROC: M-1 NOZZLE INSTALLATION WITH PAD

CERTIFIED AS-BUILT

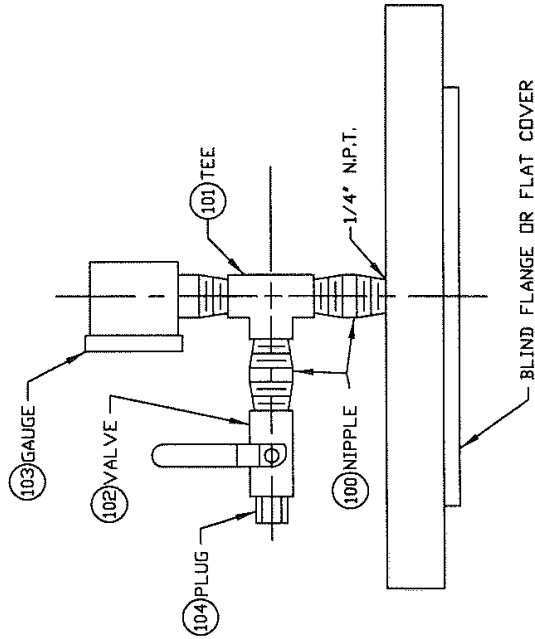
CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES P.D. NO: 17496-0001 SERVICE: WASTEWATER AMMONIA PURIFIER REBUILDER ITEM NO: HX0052	WELD & BEVEL DETAILS	DIVN. BY: S. MIAND DATE: 4-21-11 CKD. BY: <i>SM</i> DWG. NO: X-7926-S REV: 0
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Energy Exchange Co.
1844 No. Barnett Rd. • Tulsa, Okla 74116

NOTES:

- 1) BOTH SIDES SHALL BE THOROUGHLY DRAINED AND DRIED AFTER HYDROTEST. REMOVE MOISTURE BY BLOWING HOT AIR (150°F, MAXIMUM).
- 2) A) SEAL ALL EXTERNAL OPENINGS WITH A 1/8" THK. RUBBER-GASKETED BLIND FLANGE OR FLAT COVER.
B) INSTALL A CONNECTION WITH A VALVE AND PRESSURE GAUGE.
C) PRESSURE TO 15 PSIG AND DEPRESSURIZE TWICE, REPRESSURE TO 15 PSIG WITH DRY NITROGEN.
- 3) STENCIL BY EACH NOZZLE, "CONTENTS UNDER N2 PRESSURE, DO NOT OPEN WITHOUT PURCHASER'S APPROVAL".
- 4) TACKWELD A PIECE OF PIPE TO EACH COVER EQUIPPED WITH THE FITTING AND GAUGE ASSEMBLY, FOR PROTECTION DURING SHIPMENT.

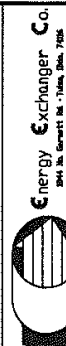


ONLY ONE BLIND FLANGE OR COVER SHALL BE TAPPED FOR CHANNEL SIDE, AND ONE FOR SHELL SIDE. PROVIDE 1/4" N.P.T. WITH PLUG ON BOTTOM COVERS FOR DRAINING PURPOSES.

CERTIFIED AS-BUILT

CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
P.O. NO: 17496-0001 REQ. NO: MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBUILDER
ITEM NO: HX0052

NITROGEN PURGE DETAILS			
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Energy Exchanger Co.
204 W. Garrett St. Tulsa, Ok. 74106
DWG. NO. X-7926-X
REV 0

[illegible]

* DENOTES PRELIMINARY ORDER

CERTIFIED AS-BUILT

ITEM: HX0052

DWG. NO.

REV	
0	

BILL OF MATERIAL

PART NO	ONE REQ'D., DESCRIPTION OF ONE	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
ASME FLANGES								
71	ASME 4'-300# RFWN W/ (Sch. 120 BORE)	SA-105	1	1				*
72	ASME 2'-300# RFLWN X 10' LG. (2.0000" BORE)	SA-105	1	1				*
81	ASME 6'-300# RFWN W/ (Sch. 120 BORE)	SA-105N	1	1		2		*
87	ASME 3'-300# RFWN W/ (Sch. 160 BORE)	SA-105N	1	1		2		*
73	ASME 3/4'-300# RFLWN X 9' LG. (0.7500" BORE)	SA-105	1	1				*
74	ASME 3/4'-300# RFLWN X 9' LG. (0.7500" BORE)	SA-105	1	1				*
75	ASME 3/4'-300# RFLWN X 9' LG. (0.7500" BORE)	SA-105	1	1				*
82	ASME 2'-300# RFLWN X 9' LG. (2.000" BORE)	SA-105N	1	1		2		*
89	ASME 2'-300# RFLWN X 9' LG. (2.000" BORE)	SA-105N	1	1		2		*
71A	SMLS. PIPE 4 1/2" O.D. X (0.4380" Sch. 120) X 7' LG.	SA-106B	1	1				*
81A	SMLS. PIPE 6 5/8" O.D. X (0.5620" Sch. 120) X 6 3/4' LG.	SA-333-6	1	1		1		*
87A	SMLS. PIPE 3 1/2" O.D. X (0.4380" Sch. 160) X 7' LG.	SA-333-6	1	1		1		*
73D	ASME 3/4'-300# RF-BLIND FLANGE	SA-105	1	1				*
74D	ASME 3/4'-300# RF-BLIND FLANGE	SA-105	1	1				*
75D	ASME 3/4'-300# RF-BLIND FLANGE	SA-105	1	1				*
82D	ASME 2'-300# RF-BLIND FLANGE	SA-105N	1	1		2		*
89D	ASME 2'-300# RF-BLIND FLANGE	SA-105N	1	1		2		*
COUPLINGS								

SEE DWG. X-7926-1BM FOR MATERIAL NOTES.

* DENOTES PRELIMINARY ORDER

CERTIFIED AS-BUILT

REQ No:	MC202	SERVICE:	WASTEWATER AMMONIA PURIFIER REBOILER
		Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla. 74116	
CUST: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES			
P.O. 17496-0001		ITEM: HX0052	
PB/M	DR: MD	CK: DN	DATE: 3/8/11
FB/M	DR: SM	CK: ☒	DATE: 4-21-11
DWG. NO.	X-7926-2BM		REV
0			0

BILL OF MATERIAL									
PART NO.	ONE REQ'D., DESCRIPTION OF ONE	MATERIAL SPEC.	QTY ONE	TOT QTY	R EV	NOTE	PO NO.	PROMISED	
BOLTING & GASKETS									
40	LOT OF GASKETS PER DWG. X-7926-D		2	2					
41	1' ROUND X 8-THREAD STUDS X 6' LG. 1&2	SA-193-B7	16	16					
42	1' ROUND X 8-THREAD STUDS X 6' LG. 1A&2A	SA-193-B7	16	16					
43	1' HEAVY HEX NUTS 8-THREAD	SA-194-2H	64	64					
44	5/8" DIA. (11 THD) STUDS X 3 1/8" LG. (3/4"-300# RF)	SA-193-B7	12	12					
44A	5/8" (11 THD) HEAVY HEX NUTS (3/4"-300# RF)	SA-194-2H	24	24					
45	5/8" DIA. (11 THD) STUDS X 3 5/8" LG. (2"-300# RF)	SA-193-B7	16	16					
45A	5/8" (11 THD) HEAVY HEX NUTS (2"-300# RF)	SA-194-2H	32	32					
	.175" THK 321 STN. STL. SPRL. WND. FLEXIBLE GRAPHITE FILLED GSK'T. WITH 1/8" THK. C-STL. OUTER RING FOR:				3				
44B	(3/4"-300# RF)		3	3					
45B	(2"-300# RF)		2	2					
MISC. MATERIAL									
60	STD. NAME PLATE	STN. STL.	2	2					
61	STD. NAME PLATE BRKT.	SA-516-70	2	2					
62	LIFTING LUGS 3 1/2" X 1/2" X 6 1/2" LG	SA-516-70	4	4					
23	FT. 1/2" D.D. TIE RODS	C-STL.	63	63					
23A	FT. 3/4" D.D. SPACER TUBING	SA-214	63	63					
23B	1/2" TIE ROD HEX NUTS	C-STL.	10	10					
22C	4 1/8" X 1/4" THK. X 7 1/4" IMP. PLATE	SA-36	1	1	1				
63	DELETE				1				
64	DELETE				1				
INSULATION SUPPORT MATERIAL									
65	PLATE 1 1/2" X 1/4" THK X 3' LG (CLIPS)	SA-36	8	8	2				
66	PLATE RING 19" O.D. X 17" I.D. X 1/4" THK	SA-36	2	2	2				
67	ANGLE 3" X 3" X 1/4" THK X 70 LG. (ROLLED LEG IN TO 16 1/8" I.D.)	SA-36	1	1	2				
70	NOZZLE SHIPPING COVERS PER SKETCH								
SUPPORT MATERIAL									
120	PLATE 11" X 3/8" THK X 10 1/4" LG	SA-516-70	2	2					
121	PLATE 6" X 3/4" THK X 9 11/16" LG	SA-516-70	2	2					
122	PLATE 6 1/4" X 1/2" THK X 9 11/16" LG	SA-516-70	4	4					
125	GROUNDING BOLT 5/8" DIA (11 THD) X 2" LG HEX HD BOLT	C-STL.	1	1					
NITROGEN PURGE MATERIAL									
100	1/4" PIPE NIPPLE X 2' LG. (THRD. OUTSIDE EACH END)	C. STL.	4	4	4				
101	1/4" STRAIGHT TEE (THRD. INSIDE EACH END)	C. STL.	2	2	4				
102	1/4" PIPE VALVE (INSIDE THRD.) (#5X713)	BRASS	2	2	4				
103	1/4" PRESSURE GAUGE (#5X939)(P570K-60LM)(0-60 PSIG)		2	2	4				
104	1/4" PIPE PLUG	C. STL.	2	2	4				
* DENOTES PRELIMINARY ORDER									
CERTIFIED AS-BUILT									
REQ No: MC202		SERVICE: WASTEWATER AMMONIA PURIFIER REBOILER							
		Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla. 74116							
△	8/6/12	ADD NITROGEN PURGE	DM/ WJ	CUSTOMER: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES P.O.# 17496-0001 ITEM HX0052 DWG. NO. X-7926-3BM REV 4					
△	5/31/12	REV MATL P/N 45A	MD/SG						
△	8-17-11	REV PER CUSTOMER COMMENTS	DM/MD						
△	6-23-11	REV'D PER CUST.	WE/SM						
NO.	DATE	DESCRIPTION	FB/M	DR. SM	CK. DM	DATE: 4-21-11			

X-7926 PACKING LIST

LINE NO	SPARE PARTS LIST	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
SP1	SPARE GASKETS PER DWG. X-7926-0		SET(S)	SET(S)				
SP2								
SP3								
SP4								
SP5								
SP6								
SP7								
SP8								
SP9								
SP10								

SPARE PARTS TO BE BOXED & TAGGED WITH P.O. NO, ITEM NO, SHIPPING WT. & "SPARE GASKETS AND PARTS"

LINE NO	STACKING MATERIAL	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
S1								
S2								
S3								
S4								
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* DENOTES PRELIMINARY ORDER

DO NOT ORDER
FROM PACKING LIST

CERTIFIED AS-BUILT

REQ No: MC202
SERVICE: WASTEWATER AMMONIA PURIFIER REBOILER

△			 Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla. 74116	
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CUST: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES P.O. 17496-0001 ITEM: HX0052				
NO.	DATE	DESCRIPTION	PB/M DR: ---- CK: ---- DATE: ---- DWG. NO. FB/M DR: SM CK: <i>PA</i> DATE: 4-21-11 X-7926-PL	REV 0

Energy Exchanger Company



Miles D. Howell 7/24/12
CERTIFIED AS-BUILT

DATE: 3/8/2011
PAGE: 1 of 37

MECHANICAL CALCULATIONS

CUSTOMER: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES
DESTINATION: KEMPER COUNTY, MISSISSIPPI
P.O. No.: 17496-0001
ITEM No.: HX0052
E.E.C. SERIAL No.: X-7926
SERVICE: WASTEWATER AMMONIA PURIFIER REBOILER
REQ. NO.: MC202

DESIGN DATA		
	SHELL SIDE	TUBE SIDE
DESIGN PRESSURE	455 PSIG * / FV	320 PSIG */**
DESIGN TEMP.	515°F	385°F **
M.D.M.T.	10 °F	10 °F
CORR. ALLOW.	1/8"	1/8"
NO. PASSES	ONE	ONE
M.A.W.P. (H&C)	455 PSIG	320 PSIG
LIMITED BY	FIXED TUBESHEET	FIXED TUBESHEET

NOTE: ALL LOADING CONDITIONS REFERENCED IN
PARA. UG-22 HAVE BEEN ADDRESSED

- DESIGNED TO ASME SEC VIII, DIV 1, 2007 ED, '09 AD.
- MATERIAL ARE IMPACT TEST EXEMPT PER UG-20(f)
- * CALCULATIONS ADD 8 PSIG TO ACCOUNT FOR STATIC HEAD WHEN ERECTED
- ** DESIGN ALSO VALIDATED FOR STEAM OUT OF 15 PSIG @ 250°F.

REVISION LOG

DATE	PAGE	REV.	DESCRIPTION
4/8/2011	28,35-36	1	REV PER CUSTOMER COMMENT & EEC
7/25/2011	AS NOTED	2	FV TO STEAM OUT & MODIFY LUG PER CUSTOMER

W.O. No.: X-7926

SHELL SIDE

```
=====
NOM.
SIZE          PRESS.      TEMP.      C.A.      RT      H.T.      TEMA
 16.000"      463.0psig *    515deg.F  0.1250"   NONE    0deg.F    R
=====
ASME CODE SEC.VIII DIV.I UG-27                      [ CYLINDER ]
=====DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON O.D. FORMULA=====
SHELL CYL
MATERIAL: SA-106B          PRESS = 463.0psig * TEMP = 515deg.F
O.D.          NOM. THK.    RW=0/PIPE=1  CORR.ALLOW  EFFICIENCY
 16.0000"      0.5000"      1           0.1250"    0.85
ATM.STRS      DES.STRS    STRS/STRS    MAWP N-C    MAWP CORR.
17100.0psi    17100.0psi    1.000      812.660psig 576.786psig

P*(RO)
----- = 0.25163" + CORR. ALLOW. = 0.37663" + MTOL. = 0.43043"
S*E+ (.4*P)
*****
*****
```

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

----- External Pressure Calculations -----

```
Ext Press Chart....= 1 CS-2
Actual Chart Used..= 1 CS-2
Material.....= SA-106B
OD.....= 16.00000
T.....= 0.31250
L.....= 168.00000
Temp.....= 515.00000
Stress from MT.....= 17100.00000
Yield from MT.....= 28244.99805
2 x Stress.....= 34200.00000
B1 from Chart@Right= 0.00000
Yield= 2 x B1 x 0.9= 0.00000
S.....= 0.00000
DO/T.....= 51.20000
L/DO.....= 10.50000
A.....= 0.00044
B.....= 5755.72559
PA1.....= 0.00000
PA2.....= 0.00000
PA.....= 149.88869
```

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TOP & BOTTOM CHANNELS

```
=====
NOM.
SIZE          PRESS.      TEMP.      C.A.      RT      H.T.      TEMA
16.000"       328.0psig *    385deg.F  0.1250"   SPOT    1150deg.F  R
=====
ASME CODE SEC.VIII DIV.I UG-27                      [ CYLINDER ]
=====DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON O.D. FORMULA=====
CHANNEL CYLINDERS
MATERIAL: SA-333-6          PRESS = 328.0psig * TEMP = 385deg.F
O.D.          NOM. THK.    RW=0/PIPE=1  CORR.ALLOW  EFFICIENCY
16.0000"      0.3750"      1           0.1250"    0.85
ATM.STRS      DES.STRS    STRS/STRS    MAWP N-C    MAWP CORR.
17100.0psi    17100.0psi  1.000       606.106psig 372.839psig

P*(RO)
----- = 0.17891" + CORR. ALLOW. = 0.30391" + MTOL. = 0.34733"
S*E+(.4*P)
*****
```

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

```
----- External Pressure Calculations -----
Ext Press Chart....= 1 CS-2
Actual Chart Used..= 1 CS-2
Material.....= SA-333-6
OD.....= 16.00000
T.....= 0.20313
L.....= 20.00000
Temp.....= 250.00000
Stress from MT....= 20000.00000
Yield from MT....= 34200.00000
2 x Stress.....= 40000.00000
B1 from Chart@Right= 0.00000
Yield= 2 x B1 x 0.9= 0.00000
S.....= 0.00000
DO/T.....= 78.76729
L/DO.....= 1.25000
A.....= 0.00152
B.....= 13868.42188
PA1.....= 0.00000
PA2.....= 0.00000
PA.....= 234.75772
```



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NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO= S1 4.00" 300 LB RF WN

[INLET] SHELL SIDE

PRESSURE = 463.0psig * TEMP = 515.0deg.F

-----CYLINDER-----NOZZLE-----

SA-106B		SA-106B	
OPER STRS=	17100.0 psi	OPER STRS=	17100.0 psi
ATM STRS =	17100.0 psi	ATM STRS =	17100.0 psi
C.A. =	0.12500"	C.A. =	0.12500"
I.D. =	15.12500"	I.D. =	3.62400"
O.D. =	16.00000"	O.D. =	4.50000"
THK. =	0.43750"	THK.sc120=	0.43800"
T =	0.31250"	TN =	0.31300"
TR =	0.21429"	TRN =	0.06027"
ET =	0.09821"	ETN =	0.25273"

MIN STD. WT. PIPE+C.A.= 0.33237" TR+C.A.= 0.33929"

LIMITS= 7.74800"

fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.3094" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD.	A =	0.83015in ²
EXCESS IN CYL.	A1=	0.38048in ²
EXCESS IN NOZ. (OUTSIDE)	A2=	0.39489in ² H = 0.78125"
AREA OF FILLETS	=	0.14062in ²
AREA AVAILABLE	=	0.91599in ²

NOZ. NECK WELD= 0.37500" AREA= 0.14062in²< AREA REQD-AREA AVAILABLE > = -0.08584in²

MAWP N-C= 742.0psig LIMITED BY AR

MAWP COR= 485.0psig LIMITED BY AR

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

EXTERNAL PRESSURE VERIFICATION:

```

NOZTYP[ 0.0] X-7926          4.00"- 300# RF WN  PRESS= -15 TEMP= 515  INLET
<<<<< CYLINDER >>>>> <<<<< NOZZLE >>>>>
      SA-106B                SA-106B
OPER STRS=17100.0000 OPER STRS=17100.0000 Area reqd.    A= 0.2729
ATM STRS =17100.0000 ATM STRS =17100.0000 Excess in cyl A1= 0.6648
C.A. = 0.12500 C.A. = 0.12500 Excess in noz A2= 0.4562 H= 0.7812
I.D. = 15.12500 I.D. = 3.62400 Excess in noz A3= 0.0000 H= 0.0000
O.D. = 16.00000 O.D. = 4.50000 Area fillets = 0.1406
THK. = 0.43750 THK.sc120= 0.43800 Area avail. = 1.2617
T = 0.31250 TN = 0.31300 Noz neck weld = 0.3750 A= 0.1406
TR = 0.14090 TRN = 0.02100 Pad weld = 0.0000 A= 0.0000
ET = 0.17160 ETN = 0.29200 Internal weld = 0.0000 A= 0.0000
PIPE F=1.0000 E1=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)
MIN STD. WT. PIPE+C.A.=0.33237 TR[using abs(P)]+C.A.=0.13202 CHORD= 0.0000
LIMITS= 7.74800 fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000
Min weld sizes: NS= 0.3094 NP= 0.0000 PS= 0.0000 IF= 0.0000
Area reqd-Area avail =<< -0.98873>>

```

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NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO= T1 3.00" 300 LB RF WN

[INLET] REAR CHANNEL or COVER

PRESSURE = 328.0psig * TEMP = 385.0deg.F

-----CYLINDER-----NOZZLE-----

SA-333-6		SA-333-6	
OPER STRS=	17100.0 psi	OPER STRS=	17100.0 psi
ATM STRS =	17100.0 psi	ATM STRS =	17100.0 psi
C.A. =	0.12500"	C.A. =	0.12500"
I.D. =	15.34375"	I.D. =	2.62400"
O.D. =	16.00000"	O.D. =	3.50000"
THK. =	0.32812"	THK.sc160=	0.43800"
T =	0.20312"	TN =	0.31300"
TR =	0.15228"	TRN =	0.03331"
ET =	0.05084"	ETN =	0.27969"

MIN STD. WT. PIPE+C.A.= 0.31400" TR+C.A.= 0.27728"

LIMITS= 5.74800"

fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.2011" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD.	A =	0.43766in ²
EXCESS IN CYL.	A1=	0.14612in ²
EXCESS IN NOZ. (OUTSIDE)	A2=	0.28406in ² H = 0.50781"
AREA OF FILLETS	=	0.14062in ²
AREA AVAILABLE	=	0.57081in ²

NOZ. NECK WELD= 0.37500" AREA= 0.14062in²< AREA REQD-AREA AVAILABLE > = -0.13315in²

MAWP N-C= 666.0psig LIMITED BY AR

MAWP COR= 376.0psig LIMITED BY AR

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

External pressure for steam out:

NOZTYP[0.0] X-7926 3.00"- 300# RF WN PRESS= -15 TEMP= 250 INLET
 <<<<< CYLINDER >>>>> <<<<< NOZZLE >>>>>
 SA-333-6 SA-333-6
 OPER STRS=17100.0000 OPER STRS=17100.0000 Area reqd. A= 0.0814
 ATM STRS =17100.0000 ATM STRS =17100.0000 Excess in cyl A1= 0.4210
 C.A. = 0.12500 C.A. = 0.12500 Excess in noz A2= 0.2987 H= 0.5078
 I.D. = 15.34375 I.D. = 2.62400 Excess in noz A3= 0.0000 H= 0.0000
 O.D. = 16.00000 O.D. = 3.50000 Area fillets = 0.1406
 THK. = 0.32812 THK.sc160= 0.43800 Area avail. = 0.8604
 T = 0.20312 TN = 0.31300 Noz neck weld = 0.3750 A= 0.1406
 TR = 0.05662 TRN = 0.01890 Pad weld = 0.0000 A= 0.0000
 ET = 0.14650 ETN = 0.29410 Internal weld = 0.0000 A= 0.0000
 PIPE F=1.0000 E1=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)
 MIN STD. WT. PIPE+C.A.=0.31400 TR[using abs(P)]+C.A.=0.13202 CHORD= 0.0000
 LIMITS= 5.74800 fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000
 Min weld sizes: NS= 0.2011 NP= 0.0000 PS= 0.0000 IF= 0.0000
 Area reqd-Area avail = << -0.77899>>



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NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO= T2 6.00" 300 LB RF WN

[OUTLET] FRONT CHANNEL SIDE

PRESSURE = 328.0psig * TEMP = 385.0deg.F

-----CYLINDER-----NOZZLE-----

SA-333-6		SA-333-6	
OPER STRS=	17100.0 psi	OPER STRS=	17100.0 psi
ATM STRS =	17100.0 psi	ATM STRS =	17100.0 psi
C.A. =	0.12500"	C.A. =	0.12500"
I.D. =	15.34375"	I.D. =	5.50100"
O.D. =	16.00000"	O.D. =	6.62500"
THK. =	0.32812"	THK.sc120=	0.56200"
T =	0.20312"	TN =	0.43700"
TR =	0.15228"	TRN =	0.06305"
ET =	0.05084"	ETN =	0.37395"

MIN STD. WT. PIPE+C.A.= 0.37000" TR+C.A.= 0.27728"

LIMITS= 11.50200"

fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.2011" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD.	A =	0.87577in ²	
EXCESS IN CYL.	A1=	0.29240in ²	
EXCESS IN NOZ. (OUTSIDE)	A2=	0.37979in ²	H = 0.50781"
AREA OF FILLETS	=	0.25000in ²	
AREA AVAILABLE	=	0.92219in ²	

NOZ. NECK WELD= 0.50000" AREA= 0.25000in²< AREA REQD-AREA AVAILABLE > = -0.04641in²

MAWP N-C= 551.0psig LIMITED BY AR

MAWP COR= 336.0psig LIMITED BY AR

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

External pressure for steam out:

NOZTYP[0.0] X-7926 6.00"- 300# RF WN PRESS= -15 TEMP= 250 OUTLET

<<<<< CYLINDER >>>>> <<<<< NOZZLE >>>>>

SA-333-6	SA-333-6	
OPER STRS=17100.0000	OPER STRS=17100.0000	Area reqd. A= 0.1628
ATM STRS =17100.0000	ATM STRS =17100.0000	Excess in cyl A1= 0.8425
C.A. = 0.12500	C.A. = 0.12500	Excess in noz A2= 0.4160 H= 0.5078
I.D. = 15.34375	I.D. = 5.50100	Excess in noz A3= 0.0000 H= 0.0000
O.D. = 16.00000	O.D. = 6.62500	Area fillets = 0.1914
THK. = 0.32812	THK.sc120= 0.56200	Area avail. = 1.4499
T = 0.20312	TN = 0.43700	Noz neck weld = 0.4375 A= 0.1914
TR = 0.05662	TRN = 0.02740	Pad weld = 0.0000 A= 0.0000
ET = 0.14650	ETN = 0.40960	Internal weld = 0.0000 A= 0.0000

PIPE F=1.0000 E1=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)

MIN STD. WT. PIPE+C.A.=0.37000 TR[using abs(P)]+C.A.=0.13202 CHORD= 0.0000

LIMITS= 11.50200 fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000

Min weld sizes: NS= 0.2011 NP= 0.0000 PS= 0.0000 IF= 0.0000

Area reqd-Area avail = << -1.28710>>



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ASME SEC.VIII DIV.1 APPENDIX 2				Front Channel to cover flange			
DESIGN CONDITIONS				GASKET and BOLTING CALCULATIONS			
TABLE							
Design Pres=	328.0	psig *		Eff.Gsk OD=	17.2500"	No.Passes=	1
Neg. Pres=	None			Eff.Gsk ID=	16.5000"	N=	0.375"
Design Temp=	385	deg.F		THK=	0.1250"	col.=	2
Flg Matl=	SA-266-4N			Gsk Matl=	S.S. DJNAF	y=	9000psi
Bolt Matl=	SA-193B7			Gsk Face=	FLAT FACE	w=	0.0000"
						m=	3.750
Corr Allow=	0.1250"			Wm2=	89461 #	Am =	3.9125in ²
Flange Desn Sfo	20000.0	psi		Hp =	24452 #	Ab =	8.8160in ²
Atm. Sfa	20000.0	psi		H =	73358 #	W =	159105 #
Bolting Desn Sb	25000.0	psi		Wml=	97811 #	Wml=	97811 #
Atm. Sa	25000.0	psi		Gasket Width Check		Nmin =	0.2310"
CONDITION	LOAD	x		LEVER ARM	=	MOMENT	
	HD =	61890 #		hD=	1.56250"	MD=	96704 in#
Operating	HG =	24452 #		hG=	1.06250"	MG=	25981 in#
	HT =	11467 #		hT=	1.40625"	MT=	16126 in#
						Mo=	138812 in#
Gasket							
Seating	HG =	159105 #		hG=	1.06250"	mo=	169049 in#
Allow.Stress-STRESS CALCULATIONS-Operating				SHAPE CONSTANTS			
1.5 Sfo&2.5 Sno Long Hub,SH	21395.1	psi		K	=	1.3629	h/ho= 0.6350
Sfo Radial Flg,SR	1849.6	psi		T	=	1.7703	F = 0.8305
Sfo Tang Flg,ST	11366.6	psi		Z	=	3.3323	V = 0.2891
Sfo .5(SH+SR)or.5(SH+ST)	16380.8	psi		Y	=	6.4305	f = 1.0000
J(APP.2-14)=	0.6860			U	=	7.0664	e = 0.4219in ⁻¹
Allow.Stress-STRESS CALCULATIONS-Gsk.Seating				OTHER STRESS FORMULA FACTORS			
1.5 Sfa&2.5 Sna Long Hub,SH	26055.6	psi		gl/go=	1.5000	d =	3.008In ³
Sfa Radial Flg,SR	2252.5	psi		ho	=	1.9685"	
Sfa Tang Flg,ST	13842.6	psi		t	=	1.8125"	
Sfa .5(SH+SR)or.5(SH+ST)	19949.1	psi		Alpha	=	1.7647	
J(APP.2-14)=	0.7942			Beta	=	2.0195	
				Gamma	=	0.9968	
				Delta	=	1.9798	
				Lambda	=	2.9766	
				M	=	8955 #	
				m	=	10906 #	
O.D.	=	21.1250"		THK.	=	2.0000"	
I.D.	=	15.2500"		T-Adder	=	0.0000"	
GO	=	0.3750"		G1	=	0.5000"	
HUB O.D.	=	16.2500"		HUB LEN	=	1.2500"	
HUB ANG	=	5.7106deg		RIB LENGTH	=	0.0000"	
G(MEAN)	=	16.8750"		G(CALC)	=	16.8750"	
G(MIN.)	=	15.8750"		G(MAX.)	=	17.0000"	
B.C.	=	19.0000"		B.S.C.F.	=	1.0000	
No. BOLTS	=	16		BOLT DIAMETER	=	1.0000"	
R	=	1.3750"		E	=	1.0625"	
BOLT SPAC	=	3.7067"		TORQUE	=	146ft#	
MIN. SPAC	=	2.2500"		TEMA MAX. SPAC	=	4.5588"	
FLG TURN	=	0.1875"		FACING	=	RECESS	
BPRIME	=	0.0000"		BG1	=	0.0000"	
FLG RWT	=	154Lbs		FLG FWT	=	97Lbs	

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

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///// FRONT COVER CALCULATIONS /////

```

EXCHANGER TYPE....= N E N          MATERIAL.....= SA-266-4N
PRESSURE.....= 328.0 psig          CORROSION ALLOWANCE= 0.1250"
TURN LOW.....= 0.1875"            TURN HIGH.....= 0.2500"
S COVER STRESS...= 20000.0 psi      E.....= 1.0
Sb STRESS BOLTS..= 25000.0 psi      Ab.....= 8.8160 in^2
d.....= 16.8750"                  C.....= 0.3000
W.....= 159105 #                   Wm1.....= 97811 #
hG (ASME).....= 1.0625"            hG (TEMA).....= 0.0000"
G (TEMA).....= 16.8750"            E (MOD.) x (10)-6..= 27.8 psi
COVER THK SET HOLD= [ NO]

```

[PER ASME SECT.VIII UG-34 USING Wm1 (OPERATING)]

$$THK = d \sqrt{\frac{C P}{S E} + \frac{1.9 W m1 hG}{S E d^3}} = 1.4093"$$

[PER ASME SECT.VIII UG-34 USING W (GASKET SEATING)]

$$THK = d \sqrt{\frac{1.9 W hG}{S E d^3}} = 0.9755"$$

```

[ COV THK Min ] CODE Operating      = 1.4093" + 0.1250" = 1.5343"
[ COV THK Min ] CODE Gasket Seating = 0.9755" + 0.1250" = 1.1005"
ACTUAL COVER THICKNESS USED (SET HOLD[ NO])= 1.6250"

```

```

MAIN FLG IS FLANGE MARK # 2
COMP FLG IS FLANGE MARK # 0

```

/// INDIVIDUAL COMPONENTS ///

```

MAWP CORR. MAIN FLG      = 330.0 Limited by: Flange stress.
MAWP CORR. COVER         = 330.0

```

```

MAWP N-C MAIN FLG       = 462.0 Limited by: Flange stress.
MAWP N-C COVER          = 436.0

```

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ASME SEC.VIII DIV.1 UHX-13

```

(FTS) Configuration a
No or EPC=Elastic-Plastic Calcs SSC=Simply Supported Calcs      =SSC
MT Tubes                  =SA-179      MT Tubesheets            =SA-266-4N
MT Channel                =SA-333-6    MT Shell                 =SA-106B
Diameter (Do)             = 14.2708"   Pass Lane Area (AL)       = 0.000in^2
Perimeter of layout (CP)  = 0.0000"    Area enclosed by CP (AP) = 0.000in^2
Tubesheet thickness (h)   = 3.0000"    Tubesheet thickness HOLD  = NO
Tube Pitch (p)            = 0.9375"    Pitch type = Triangle     (60 Deg)
Diameter of tube (dt)     = 0.7500"    Tube thickness (tt)       = 0.0830"
Exp length of tube (ltx)  = 0.0000"    Exp depth ratio (Rho)    = 0.9500
Groove channel side (hg)  = 0.0000"    Groove shell side        = 0.0000"
Channel I.D. (Dc)         = 15.2500"   Channel thickness (Tc)   = 0.3750"
Shell I.D. (Ds)           = 15.0000"   Shell thickness (Ts)     = 0.5000"
TS outside diameter (A)   = 16.0000"   Bolt circle (C)         = 0.0000"
G Channel side (Gc or G1) = 0.0000"    G Shell side (Gs or G1)  = 0.0000"
Bolt load Chan side (Wc)  = 0.0#       Bolt load Sh side (Ws)   = 0.0#
1.8*sqrt(Dc*Tc)           = 0.0000"    1.8*sqrt(Ds*(Ts or Ts1)) = 0.0000"
Thickness at Top TS (tr)  = 3.0000"    Outer Tube Circle        = 14.2708"
hr Ch Op= 0.0000" Ch Atm= 0.0000" Sh Op= 0.0000" Sh Atm= 0.0000"
Tubes welded Backside TS =NO          Hole Size in Tubesheet   = 0.0000"
Radial Differential Thermal Expansion Adjacent to Tubesheet = NO

Design Press Chan (Pt)    = 328.0psig Design Press Shell (Ps) = 463.0psig
Design Temp Chan         = 385.0deg Design Temp Shell         = 515.0deg
Corr Chan                 = 0.12500" Corr Shell                = 0.12500"
Corr Chan side TS (ct)   = 0.12500" Corr Shell side TS (cs) = 0.12500"

Number Tubes (Nt)        = 185.00     F-F Tubesheets (Lt)     = 167.8750"
Un-supported Tube Span (l)= 27.50000" Support Constant (k)    = 1.00000
Expansion Joint           = None        Exp Joint I.D.          = 0.00000"
Spring Rate Corr (Kj)     = 0.0#       Spring Rate Un-Corr (Kjn)= 0.0#
MT Thickened Shl          =            Thk Thickened Shl (Ts1) = 0.00000"
Len Thickened Shl (lll)   = 0.00000" Len Thickened Shl (lll') = 0.00000"

```

Calculate/Conservative For Ps* and Pc* = Calculated values for Ps* and Pc*

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

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ASME SEC.VIII DIV.1 UHX-13 Case 1

(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			
Press Chan (Pt)	=	328.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps =	58260psi
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 27.1800psi	Vt = .300		
Temp CH Tc = 385deg.F	Sc = 17100psi	Syc = 30065psi	Ec = 27.7600psi	Vc = .300	Spssc =	60130psi
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss =	56489psi

Mu = 0.2000	d* = 0.6406"	P* = 0.9375"	Mu* = 0.3167	hg' = 0.0000"	hgs' = 0.0000"
W = 0.0#	wj = 0.0"	ao = 7.135405064"			
Rho s = 1.068614841	Rho c = 1.086133122	Chi s = 0.489028096			
Chi t = 0.690186620	Ks = 3065028.500Lb/In	Kt = 29157.69727Lb/In			
Ks, t = 0.568210900	j = 1.000000000	Beta s = 0.750982881in ⁻¹			
Kappa s = 195800.8281#	Lambda s = 4477665.500psi	Beta c = 0.916105211in ⁻¹			
Kappa c = 72776.67188#	Lambda c = 2178102.500psi	h/p = 2.933333397			
E* = 8183711.000psi	v* = 0.343179107	Xa = 1.815992355			
Zd = 0.234626889	Zv = 0.167576507	Zm = 0.688905299			
K = 1.121169686	F = 0.781989098	Phi = 1.050351381			
Q1 = -0.062311340	Qz1 = 1.219081044	Qz2 = 0.677825630			
U = 1.355651259	Gamma = 0.0"	Omega s = 0.0in ²			
Omega c = 0.0in ²	Omega s* = -2.227341890in ²	Omega c* = -1.231768727in ²			
Delta s = 0.000004880In ³ /Lb	Delta c = 0.000007360In ³ /Lb	Gamma b = 0.0			
Ps' = 0.0psi	Pt' = 864.6029663psi	P Gamma = 0.0psi			
PW = 0.0psi	Prim = -10.75755882psi	Es, w = 0.850000024			
Pe = -289.3275757psi	Q2 = 234.4058228#	Q3 = -0.094136558			
Fm = 0.140259594	Es, w1 = 0.0	lt = 27.50000000"			
Rt = 0.237638906"	Ft = 115.7217941	Ct = 159.8189392			
Fs = 2.000000000	Stb = 6700.172363psi	Ft, min = 0.761786759			
Sigmat, 1 = -29.70440865psi	Sigmat, 2 = 586.3126831psi	Ft, max = 1.190080166			
Sigma s, m = 336.0374146psi	Sigma s, b = -5881.111816psi	Sigma s = 6217.149414psi			
Sigma c, m = 5003.301758psi	Sigma c, b = 20745.13477psi	Sigma c = 25748.43750psi			

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas = 0	Betac, Kappac, Lambdac, Deltac, Omegac = 0		
F = 0.247800201	Phi = 0.332840055	Q1 = 0.010443920	
Qz1 = 1.285379529	Qz2 = 0.950378060	U = 1.900756121	
Omega s* = 0.123963267in ²	Omega c* = 0.642959952in ²	Prim = 7.873120308psi	
Pe = -275.4316101psi	Q2 = -171.5542755#	Q3 = 0.034910910	
		Sigma = -6799 psi	
Sigma	-5175 psi	<= 1.5S	29130 psi [OK Per Simply Sup]
Tau	-1876 psi	<= .8S	15536 psi [OK]
Sigmat, max	586 psi	< St	13384 psi [OK]
Sigmat, min	-29 psi	<= Stb	6700 psi [OK]
Sigma s	6217 psi	<= Spss	56489 psi [OK Per Simply Sup]
Sigma c	25748 psi	<= Spssc	60130 psi [OK Per Simply Sup]
Sigmat, m	336 psi	<= SsEs, w	14535 psi [OK] Step 10

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Job No.: X-7926

By: MD/DN

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(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	463.0psi
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps = 58260psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 27.1800psi	Vt=.300		
Temp CH Tc= 385deg.F	Sc = 17100psi	Syc= 30065psi	Ec = 27.7600psi	Vc=.300	Spss= 60130psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	

Mu= 0.2000	d*= 0.6406"	P*= 0.9375"	Mu*= 0.3167	hg'= 0.0000"	hgs'= 0.0000"
W = 0.0#	wj = 0.0"	ao = 7.135405064"			
Rho s = 1.068614841	Rho c = 1.086133122	Chi s = 0.489028096			
Chi t = 0.690186620	Ks = 3065028.500Lb/In	Kt = 29157.69727Lb/In			
Ks,t = 0.568210900	j = 1.000000000	Beta s = 0.750982881in ⁻¹			
Kappa s = 195800.8281#	Lambda s = 4477665.500psi	Beta c = 0.916105211in ⁻¹			
Kappa c = 72776.67188#	Lambda c = 2178102.500psi	h/p = 2.933333397			
E* = 8183711.000psi	v* = 0.343179107	Xa = 1.815992355			
Zd = 0.234626889	Zv = 0.167576507	Zm = 0.688905299			
K = 1.121169686	F = 0.781989098	Phi = 1.050351381			
Q1 = -0.062311340	Qz1 = 1.219081044	Qz2 = 0.677825630			
U = 1.355651259	Gamma = 0.0"	Omega s = 0.0in ²			
Omega c = 0.0in ²	Omega s* = -2.227341890in ²	Omega c* = -1.231768727in ²			
Delta s = 0.000004880In ³ /Lb	Delta c = 0.000007360In ³ /Lb	Gamma b = 0.0			
Ps' = 811.0084839psi	Pt' = 0.0psi	P Gamma = 0.0psi			
PW = 0.0psi	Prim = 27.45861244psi	Es,w = 0.850000024			
Pe = 277.1334229psi	Q2 = -598.3196411#	Q3 = -0.147119388			
Fm = 0.118377589	Es,w1 = 0.0	Lt = 27.50000000"			
Rt = 0.237638906"	Ft = 115.7217941	Ct = 159.8189392			
Fs = 2.000000000	Stb = 6700.172363psi	Ft,min = 0.808290243			
Sigmat,1 = 12.00926876psi	Sigmat,2 = -447.4617615psi	Ft,max = 1.141799331			
Sigma s,m = 2979.139404psi	Sigma s,b = 16118.24707psi	Sigma s = 19097.38672psi			
Sigma c,m = 0.0psi	Sigma c,b = -3273.823486psi	Sigma c = 3273.823486psi			

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas,Kappas,Lambdas,Deltas,Omeas=0	Betac,Kappac,Lambdac,Deltac,Omeac=0		
F = 0.247800201	Phi = 0.332840055	Q1 = 0.010443920	
Qz1 = 1.285379529	Qz2 = 0.950378060	U = 1.900756121	
Omega s* = 0.123963267in ²	Omega c* = 0.642959952in ²	Prim = -2.142708778psi	
Pe = 260.0437012psi	Q2 = 46.68935013#	Q3 = 0.017496770	
	Sigma = 6180 psi		
Sigma	4183 psi	<= 1.5S	29130 psi [OK Per Simply Sup]
Tau	1797 psi	<= .8S	15536 psi [OK]
Sigmat,max	447 psi	< St	13384 psi [OK]
Sigmat,min	-447 psi	<= Stb	6700 psi [OK]
Sigma s	19097 psi	<= Spss	56489 psi [OK Per Simply Sup]
Sigma c	3273 psi	<= Spss	60130 psi [OK Per Simply Sup]
Sigmatas,m	2979 psi	<= SsEs,w	14535 psi [OK] Step 10

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(FTS) Configuration a
 Tubesheet Thickness (h) = 2.7500"
 F-F Tubesheets (Lt) = 167.6250" F-F Inside Tubesheets (L) = 162.1250"
 Exp length of tube (ltx) = 2.6125" Exp depth ratio (Rho) = 0.9500"
 Channel I.D. (Dc) = 15.5000" Channel thickness (Tc) = 0.2500"
 Shell I.D. (Ds) = 15.2500" Shell thickness (Ts) = 0.3750"
 Press Chan (Pt) = 328.0psi Press Shell (Ps) = 463.0psi
 Corr Chan = 0.12500" Corr Shell = 0.12500"
 Corr Chan side TS (ct) = 0.12500" Corr Shell side TS (cs) = 0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps = 58260psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 27.1800psi	Vt = .300		
Temp CH Tc = 385deg.F	Sc = 17100psi	Syc = 30065psi	Ec = 27.7600psi	Vc = .300	Spac = 60130psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	

Mu = 0.2000 d* = 0.6406" P* = 0.9375" Mu* = 0.3167 hg' = 0.0000" hgs' = 0.0000"
 W = 0.0# wj = 0.0" ao = 7.135405064"
 Rho s = 1.068614841 Rho c = 1.086133122 Chi s = 0.489028096
 Chi t = 0.690186620 Ks = 3065028.500Lb/In Kt = 29157.69727Lb/In
 Ks,t = 0.568210900 j = 1.000000000 Beta s = 0.750982881in⁻¹
 Kappa s = 195800.8281# Lambda s = 4477665.500psi Beta c = 0.916105211in⁻¹
 Kappa c = 72776.67188# Lambda c = 2178102.500psi h/p = 2.933333397
 E* = 8183711.000psi v* = 0.343179107 Xa = 1.815992355
 Zd = 0.234626889 Zv = 0.167576507 Zm = 0.688905299
 K = 1.121169686 F = 0.781989098 Phi = 1.050351381
 Q1 = -0.062311340 Qz1 = 1.219081044 Qz2 = 0.677825630
 U = 1.355651259 Gamma = 0.0" Omega s = 0.0in²
 Omega c = 0.0in² Omega s* = -2.227341890in² Omega c* = -1.231768727in²
 Delta s = 0.000004880In³/Lb Delta c = 0.000007360In³/Lb Gamma b = 0.0
 Ps' = 811.0084839psi Pt' = 864.6029663psi P Gamma = 0.0psi
 PW = 0.0psi Prim = 16.70105362psi Es,w = 0.850000024
 Pe = -12.19415951psi Q2 = -363.9138184# Q3 = 1.109990358
 Fm = 0.644170761 Es,w1 = 0.0 lt = 27.50000000"
 Rt = 0.237638906" Ft = 115.7217941 Ct = 159.8189392
 Fs = 2.000000000 Stb = 6700.172363psi Ft,min = -0.295084894
 Sigmat,1 = -17.69507408psi Sigmat,2 = 138.8509216psi Ft,max = 2.287344933
 Sigma s,m = 3315.177002psi Sigma s,b = 10237.13477psi Sigma s = 13552.31152psi
 Sigma c,m = 5003.301758psi Sigma c,b = 17471.31250psi Sigma c = 22474.61328psi
 Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas,Kappas,Lambdas,Deltas,Omeas=0 Betac,Kappac,Lambdac,Deltac,Omeac=0
 F = 0.247800201 Phi = 0.332840055 Q1 = 0.010443920
 Qz1 = 1.285379529 Qz2 = 0.950378060 U = 1.900756121
 Omega s* = 0.123963267in² Omega c* = 0.642959952in² Prim = 5.730411530psi
 Pe = -15.38790512psi Q2 = -124.8649229# Q3 = 0.329196513
 Sigma = | -618 psi |
 Sigma | -1001 psi | <= 1.5S 29130 psi [OK Per Simply Sup]
 Tau | -79 psi | <= .8S 15536 psi [OK]
 Sigmat,max | 138 psi | < St 13384 psi [OK]
 Sigmat,min | -17 psi | <= Stb 6700 psi [OK]
 Sigma s | 13552 psi | <= Spss 56489 psi [OK Per Simply Sup]
 Sigma c | 22474 psi | <= Spsc 60130 psi [OK Per Simply Sup]
 Sigmas,m | 3315 psi | <= SsEs,w 14535 psi [OK] Step 10

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Job No.: X-7926

By: MD/DN

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(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+ Temp TS T = 515deg.F S = 19420psi Sy = 29045psi E =26.9950psi V =.300 Sps = 58260psi						
Temp TBS Tt= 515deg.F St = 13384psi Syt= 21005psi Et =27.1800psi Vt=.300						
Temp CH Tc= 385deg.F Sc = 17100psi Syc= 30065psi Ec =27.7600psi Vc=.300 Spsc= 60130psi						
Temp SH Ts= 515deg.F Ss = 17100psi Sys= 28244psi Es =26.9950psi Vs=.300 Spss= 56489psi						
Tt,m = 390deg.F At,m= 7.0800In/In						
Ts,m = 422deg.F As,m= 7.1440In/In						
Mu= 0.2000 d*= 0.6406" P*= 0.9375" Mu*= 0.3167 hg'= 0.0000" hgs'= 0.0000"						
W = 0.0# Wj = 0.0" ao = 7.135405064"						
Rho s = 1.068614841 Rho c = 1.086133122 Chi s = 0.489028096						
Chi t = 0.690186620 Ks = 3065028.500Lb/In Kt = 29157.69727Lb/In						
Ks,t = 0.568210900 j = 1.000000000 Beta s = 0.750982881in ⁻¹						
Kappa s = 195800.8281# Lambda s = 4477665.500psi Beta c = 0.916105211in ⁻¹						
Kappa c = 72776.67188# Lambda c = 2178102.500psi h/p = 2.933333397						
E* = 8183711.000psi v* = 0.343179107 Xa = 1.815992355						
Zd = 0.234626889 Zv = 0.167576507 Zm = 0.688905299						
K = 1.121169686 F = 0.781989098 Phi = 1.050351381						
Q1 = -0.062311340 QZ1 = 1.219081044 QZ2 = 0.677825630						
U = 1.355651259 Gamma = -0.040383380" Omega s = 2.351305246in ²						
Omega c = 1.874728680in ² Omega s* = -2.227341890in ² Omega c* = -1.231768727in ²						
Delta s = 0.000004880In ³ /Lb Delta c = 0.000007360In ³ /Lb Gamma b = 0.0						
Ps' = 0.0psi Pt' = 0.0psi P Gamma = -1361.885132psi						
PW = 0.0psi Prim = 0.0psi Es,w = 0.850000024						
Pe = -450.1355896psi Q2 = 0.0# Q3 = -0.062311340						
Fm = 0.153403476 Es,w1 = 0.0 lt = 0.0"						
Rt = 0.0" Ft = 0.0 Ct = 0.0						
Fs = 0.0 Stb = 0.0psi Ft,min = 0.733853519						
Sigmat,1 = 1642.155518psi Sigmat,2 = 2727.956543psi Ft,max = 1.219080925						
Sigma s,m = -3911.373779psi Sigma s,b = -8685.994141psi Sigma s = 12597.36816psi						
Sigma c,m = 0.0psi Sigma c,b = 10008.71875psi Sigma c = 10008.71875psi						
Thermal Expansion Acting on this Case (Yes)						
Sigma -8806 psi <= Sps 58260 psi [OK]						
Tau -2919 psi <= .8S 15536 psi [OK]						
Sigmat,max 2727 psi <= 2St 26769 psi [OK]						
Sigma s 12597 psi <= Spss 56489 psi [OK]						
Sigma c 10008 psi <= Spsc 60130 psi [OK]						
Sigmas,m -3911 psi <= 2Ss 34200 psi [OK] Step 10						
Sigmas,m -3911 psi <= Ss,b 13402 psi [OK] Step 10						

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(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			
Press Chan (Pt)	=	328.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+ Temp TS T = 515deg.F S = 19420psi Sy = 29045psi E = 26.9950psi V = .300 Sps = 58260psi						
Temp TBS Tt= 515deg.F St = 13384psi Syt= 21005psi Et = 27.1800psi Vt=.300						
Temp CH Tc= 385deg.F Sc = 17100psi Syc= 30065psi Ec = 27.7600psi Vc=.300 Spssc= 60130psi						
Temp SH Ts= 515deg.F Ss = 17100psi Sys= 28244psi Es = 26.9950psi Vs=.300 Spss= 56489psi						
Tt,m = 390deg.F At,m= 7.0800In/In						
Ts,m = 422deg.F As,m= 7.1440In/In						
Mu= 0.2000 d*= 0.6406" P*= 0.9375" Mu*= 0.3167 hg'= 0.0000" hgs'= 0.0000"						
W = 0.0# wj = 0.0" ao = 7.135405064"						
Rho s = 1.068614841 Rho c = 1.086133122 Chi s = 0.489028096						
Chi t = 0.690186620 Ks = 3065028.500Lb/In Kt = 29157.69727Lb/In						
Ks,t = 0.568210900 j = 1.000000000 Beta s = 0.750982881in^-1						
Kappa s = 195800.8281# Lambda s = 4477665.500psi Beta c = 0.916105211in^-1						
Kappa c = 72776.67188# Lambda c = 2178102.500psi h/p = 2.933333397						
E* = 8183711.000psi v* = 0.343179107 Xa = 1.815992355						
Zd = 0.234626889 Zv = 0.167576507 Zm = 0.688905299						
K = 1.121169686 F = 0.781989098 Phi = 1.050351381						
Q1 = -0.062311340 Qz1 = 1.219081044 Qz2 = 0.677825630						
U = 1.355651259 Gamma = -0.040383380" Omega s = 2.351305246in^2						
Omega c = 1.874728680in^2 Omega s* = -2.227341890in^2 Omega c* = -1.231768727in^2						
Delta s = 0.000004880In^3/Lb Delta c = 0.000007360In^3/Lb Gamma b = 0.0						
Ps' = 0.0psi Pt' = 864.6029663psi P Gamma = -1361.885132psi						
PW = 0.0psi Prim = -10.75755882psi Es,w = 0.850000024						
Pe = -739.4631348psi Q2 = 234.4058228# Q3 = -0.074763499						
Fm = 0.148260713 Es,w1 = 0.0 lt = 0.0"						
Rt = 0.0" Ft = 0.0 Ct = 0.0						
Fs = 0.0 Stb = 0.0psi Ft,min = 0.744782865						
Sigmat,1 = 1612.451050psi Sigmat,2 = 3314.269043psi Ft,max = 1.207733870						
Sigma s,m = -3575.336182psi Sigma s,b = -14567.10547psi Sigma s = 18142.44141psi						
Sigma c,m = 5003.301758psi Sigma c,b = 30753.85352psi Sigma c = 35757.15625psi						
Thermal Expansion Acting on this Case (Yes)						
Sigma -13982 psi <= Sps 58260 psi [OK]						
Tau -4796 psi <= .8S 15536 psi [OK]						
Sigmat,max 3314 psi <= 2St 26769 psi [OK]						
Sigma s 18142 psi <= Spss 56489 psi [OK]						
Sigma c 35757 psi <= Spssc 60130 psi [OK]						
Sigmas,m -3575 psi <= 2Ss 34200 psi [OK] Step 10						
Sigmas,m -3575 psi <= Ss,b 13402 psi [OK] Step 10						

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ASME SEC.VIII DIV.1 UHX-13 Case 6

(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	463.0psi
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps = 58260psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 27.1800psi	Vt=.300		
Temp CH Tc= 385deg.F	Sc = 17100psi	Syc= 30065psi	Ec = 27.7600psi	Vc=.300	Spss= 60130psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300		
Tt,m = 390deg.F			At,m= 7.0800In/In			
Ts,m = 422deg.F			As,m= 7.1440In/In			
Mu= 0.2000	d*= 0.6406"	P*= 0.9375"	Mu*= 0.3167	hg'= 0.0000"	hgs'= 0.0000"	
W = 0.0#	wj = 0.0"	ao = 7.135405064"				
Rho s = 1.068614841	Rho c = 1.086133122	Chi s = 0.489028096				
Chi t = 0.690186620	Ks = 3065028.500Lb/In	Kt = 29157.69727Lb/In				
Ks,t = 0.568210900	j = 1.000000000	Beta s = 0.750982881in ⁻¹				
Kappa s = 195800.8281#	Lambda s = 4477665.500psi	Beta c = 0.916105211in ⁻¹				
Kappa c = 72776.67188#	Lambda c = 2178102.500psi	h/p = 2.933333397				
E* = 8183711.000psi	v* = 0.343179107	Xa = 1.815992355				
Zd = 0.234626889	Zv = 0.167576507	Zm = 0.688905299				
K = 1.121169686	F = 0.781989098	Phi = 1.050351381				
Q1 = -0.062311340	Qz1 = 1.219081044	Qz2 = 0.677825630				
U = 1.355651259	Gamma = -0.040383380"	Omega s = 2.351305246in ²				
Omega c = 1.874728680in ²	Omega s* = -2.227341890in ²	Omega c* = -1.231768727in ²				
Delta s = 0.000004880In ³ /Lb	Delta c = 0.000007360In ³ /Lb	Gamma b = 0.0				
Ps' = 811.0084839psi	Pt' = 0.0psi	P Gamma = -1361.885132psi				
PW = 0.0psi	Prim = 27.45861244psi	Es,w = 0.850000024				
Pe = -173.0021667psi	Q2 = -598.3196411#	Q3 = 0.073543288				
Fm = 0.209511697	ES,w1 = 0.0	lt = 0.0"				
Rt = 0.0"	Ft = 0.0	Ct = 0.0				
Fs = 0.0	Stb = 0.0psi	Ft,min = 0.614612877				
Sigmat,1 = 1654.164917psi	Sigmat,2 = 2280.494873psi	Ft,max = 1.342878938				
Sigma s,m = -932.2342529psi	Sigma s,b = 7432.252930psi	Sigma s = 8364.487305psi				
Sigma c,m = 0.0psi	Sigma c,b = 6734.895508psi	Sigma c = 6734.895508psi				
Thermal Expansion Acting on this Case (Yes)						
Sigma	-4622 psi	<= Sps	58260 psi	[OK		
Tau	-1122 psi	<= .8S	15536 psi	[OK		
Sigmat,max	2280 psi	<= 2St	26769 psi	[OK		
Sigma s	8364 psi	<= Spss	56489 psi	[OK		
Sigma c	6734 psi	<= Spsc	60130 psi	[OK		
Sigmatas,m	-932 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmatas,m	-932 psi	<= Ss,b	13402 psi	[OK		Step 10

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ASME SEC.VIII DIV.1 UHX-13 Case 7

(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			
Press Chan (Pt)	=	328.0psi	Press Shell (Ps)	=	463.0psi
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+						
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps =	58260psi
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 27.1800psi	Vt=.300		
Temp CH Tc= 385deg.F	Sc = 17100psi	Syc= 30065psi	Ec = 27.7600psi	Vc=.300	Spssc=	60130psi
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss=	56489psi
Tt,m = 390deg.F				At,m= 7.0800In/In		
Ts,m = 422deg.F				As,m= 7.1440In/In		
Mu= 0.2000 d*= 0.6406"	P*= 0.9375"	Mu*= 0.3167	hg'= 0.0000"	hgs'= 0.0000"		
W = 0.0#	wj = 0.0"	ao = 7.135405064"				
Rho s = 1.068614841	Rho c = 1.086133122	Chi s = 0.489028096				
Chi t = 0.690186620	Ks = 3065028.500Lb/In	Kt = 29157.69727Lb/In				
Ks,t = 0.568210900	j = 1.000000000	Beta s = 0.750982881in ⁻¹				
Kappa s = 195800.8281#	Lambda s = 4477665.500psi	Beta c = 0.916105211in ⁻¹				
Kappa c = 72776.67188#	Lambda c = 2178102.500psi	h/p = 2.933333397				
E* = 8183711.000psi	v* = 0.343179107	Xa = 1.815992355				
Zd = 0.234626889	Zv = 0.167576507	Zm = 0.688905299				
K = 1.121169686	F = 0.781989098	Phi = 1.050351381				
Q1 = -0.062311340	Qz1 = 1.219081044	Qz2 = 0.677825630				
U = 1.355651259	Gamma = -0.040383380"	Omega s = 2.351305246in ²				
Omega c = 1.874728680in ²	Omega s* = -2.227341890in ²	Omega c* = -1.231768727in ²				
Delta s = 0.000004880In ³ /Lb	Delta c = 0.000007360In ³ /Lb	Gamma b = 0.0				
Ps' = 811.0084839psi	Pt' = 864.6029663psi	P Gamma = -1361.885132psi				
PW = 0.0psi	Prim = 16.70105362psi	Es,w = 0.850000024				
Pe = -462.3297424psi	Q2 = -363.9138184#	Q3 = -0.031391341				
Fm = 0.166173488	Es,w1 = 0.0	lt = 0.0"				
Rt = 0.0"	Ft = 0.0	Ct = 0.0				
Fs = 0.0	Stb = 0.0psi	Ft,min = 0.706714809				
Sigmat,1 = 1624.460449psi	Sigmat,2 = 2866.807617psi	Ft,max = 1.247256994				
Sigma s,m = -596.1968384psi	Sigma s,b = 1551.140869psi	Sigma s = 2147.337646psi				
Sigma c,m = 5003.301758psi	Sigma c,b = 27480.03125psi	Sigma c = 32483.33203psi				
Thermal Expansion Acting on this Case (Yes)						
Sigma	-9798 psi	<= Sps	58260 psi	[OK		
Tau	-2999 psi	<= .8S	15536 psi	[OK		
Sigmat,max	2866 psi	<= 2St	26769 psi	[OK		
Sigma s	2147 psi	<= Spss	56489 psi	[OK		
Sigma c	32483 psi	<= Spssc	60130 psi	[OK		
Sigmas,m	-596 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmas,m	-596 psi	<= Ss,b	13402 psi	[OK		Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0000"	F-F Inside Tubesheets (L)	=	161.8750"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.8500"	Channel thickness (Tc)	=	0.3750"
Channel I.D. (Dc)	=	15.2500"	Shell thickness (Ts)	=	0.5000"
Shell I.D. (Ds)	=	15.0000"			
Press Chan (Pt)	=	328.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps =	58260psi
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 27.1800psi	Vt = .300		
Temp CH Tc = 385deg.F	Sc = 17100psi	Syc = 30065psi	Ec = 27.7600psi	Vc = .300	Spssc =	60130psi
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spsss =	56489psi

Mu = 0.2000	d* = 0.6406"	P* = 0.9375"	Mu* = 0.3167	hg' = 0.0000"	hgs' = 0.0000"
W =	0.0#	wj =	0.0"	ao =	7.135405064"
Rho s =	1.051096559	Rho c =	1.068614841	Chi s =	0.489028096
Chi t =	0.690186620	Ks =	4060272.000Lb/In	Kt =	29202.72852Lb/In
Ks, t =	0.751553595	j =	1.000000000	Beta s =	0.652987421in ⁻¹
Kappa s =	403557.5938#	Lambda s =	6561481.500psi	Beta c =	0.750982881in ⁻¹
Kappa c =	201349.5469#	Lambda c =	3951385.000psi	h/p =	3.200000048
E* =	8183711.000psi	v* =	0.343179107	Xa =	1.701923966
Zd =	0.290400684	Zv =	0.171428442	Zm =	0.700460076
K =	1.121169686	F =	1.091558099	Phi =	1.466158032
Q1 =	-0.098789386	Qz1 =	1.147183657	Qz2 =	0.428855240
U =	0.857710540	Gamma =	0.0"	Omega s =	0.0in ²
Omega c =	0.0in ²	Omega s* =	-2.835061073in ²	Omega c* =	-1.925397873in ²
Delta s =	0.000003540In ³ /Lb	Delta c =	0.000004750In ³ /Lb	Gamma b =	0.0
Ps' =	0.0psi	Pt' =	723.7817993psi	P Gamma =	0.0psi
PW =	0.0psi	Prim =	-10.63892651psi	Es, w =	0.850000024
Pe =	-293.8066406psi	Q2 =	311.5614929#	Q3 =	-0.140445143
Fm =	0.124783158	Es, w1 =	0.0	lt =	0.0"
Rt =	0.0"	Ft =	0.0	Ct =	0.0
Fs =	0.0	Stb =	0.0psi	Ft, min =	0.843329072
Sigmat, 1 =	106.3562698psi	Sigmat, 2 =	504.9070129psi	Ft, max =	1.116201997
Sigma s, m =	224.6351318psi	Sigma s, b =	-3699.683838psi	Sigma s =	3924.318848psi
Sigma c, m =	3254.634766psi	Sigma c, b =	13458.42090psi	Sigma c =	16713.05469psi

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas=0	Betac, Kappac, Lambdac, Deltac, Omegac=0
F = 0.247800201	Phi = 0.332840055
Qz1 = 1.214750290	Qz2 = 0.704934061
Omega s* = 0.068162687in ²	Omega c* = 0.569926202in ²
Pe = -278.3838509psi	Q2 = -151.5931854#
	Q3 = 0.016556310
Sigma = -3928 psi	Stress = -5741 psi
Tau = -1747 psi	
Sigmat, max = 504 psi	
Sigma s = 3924 psi	
Sigma c = 16713 psi	
Sigmat, m = 224 psi	

[OK Per Simply Sup] [OK] [OK] [OK Per Simply Sup] [OK Per Simply Sup] [OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0000"		
F-F Tubesheets (Lt)	=	167.8750"	F-F Inside Tubesheets (L)	= 161.8750"
Exp length of tube (ltx)	=	2.8500"	Exp depth ratio (Rho)	= 0.9500"
Channel I.D. (Dc)	=	15.2500"	Channel thickness (Tc)	= 0.3750"
Shell I.D. (Ds)	=	15.0000"	Shell thickness (Ts)	= 0.5000"
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	= 463.0psi
Corr Chan	=	0.00000"	Corr Shell	= 0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	= 0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps = 58260psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 27.1800psi	Vt = .300		
Temp CH Tc = 385deg.F	Sc = 17100psi	Syc = 30065psi	Ec = 27.7600psi	Vc = .300	Spss = 60130psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	

Mu = 0.2000	d* = 0.6406"	P* = 0.9375"	Mu* = 0.3167	hg' = 0.0000"	hgs' = 0.0000"
W =	0.0#	wj =	0.0"	ao =	7.135405064"
Rho s =	1.051096559	Rho c =	1.068614841	Chi s =	0.489028096
Chi t =	0.690186620	Ks =	4060272.000Lb/In	Kt =	29202.72852Lb/In
Ks,t =	0.751553595	j =	1.000000000	Beta s =	0.652987421in ⁻¹
Kappa s =	403557.5938#	Lambda s =	6561481.500psi	Beta c =	0.750982881in ⁻¹
Kappa c =	201349.5469#	Lambda c =	3951385.000psi	h/p =	3.200000048
E* =	8183711.000psi	v* =	0.343179107	Xa =	1.701923966
Zd =	0.290400684	Zv =	0.171428442	Zm =	0.700460076
K =	1.121169686	F =	1.091558099	Phi =	1.466158032
Q1 =	-0.098789386	Qz1 =	1.147183657	Qz2 =	0.428855240
U =	0.857710540	Gamma =	0.0"	Omega s =	0.0in ²
Omega c =	0.0in ²	Omega s* =	-2.835061073in ²	Omega c* =	-1.925397873in ²
Delta s =	0.000003540in ³ /Lb	Delta c =	0.000004750in ³ /Lb	Gamma b =	0.0
Ps' =	712.1762695psi	Pt' =	0.0psi	P Gamma =	0.0psi
PW =	0.0psi	Prim =	22.11296082psi	Es,w =	0.850000024
Pe =	293.7540283psi	Q2 =	-647.5791016#	Q3 =	-0.185386166
Fm =	0.105413839	Es,w1 =	0.0	lt =	27.50000000"
Rt =	0.237638906"	Ft =	115.7217941	Ct =	159.8189392
Fs =	2.000000000	Stb =	6700.172363psi	Ft,min =	0.874724805
Sigmat,1 =	-151.7903748psi	Sigmat,2 =	-457.8683167psi	Ft,max =	1.084322572
Sigma s,m =	2248.613770psi	Sigma s,b =	11225.03125psi	Sigma s =	13473.64453psi
Sigma c,m =	0.0psi	Sigma c,b =	-1941.632080psi	Sigma c =	1941.632080psi

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas,Kappas,Lambdas,Deltas,Omeegas=0		Betac,Kappac,Lambdac,Deltac,Omegac=0			
F	= 0.247800201	Phi	= 0.332840055	Q1	=-0.004834560
Qz1	= 1.214750290	Qz2	= 0.704934061	U	= 1.409868121
Omega s*	= 0.068162687in^2	Omega c*	= 0.569926202in^2	Prim	=-0.873914421psi
Pe	= 275.5547180psi	Q2	= 25.59262848#	Q3	=-0.001186180

Sigma	3318 psi	<= 1.5S	29130 psi	[OK Per Simply Sup]
Tau	1746 psi	<= .8S	15536 psi	[OK]
Sigmat,max	457 psi	< St	13384 psi	[OK]
Sigmat,min	-457 psi	<= Stb	6700 psi	[OK]
Sigma s	13473 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	1941 psi	<= Spss	60130 psi	[OK Per Simply Sup]
Sigmatas,m	2248 psi	<= SsEs,w	14535 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0000"	F-F Inside Tubesheets (L)	=	161.8750"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.8500"	Channel thickness (Tc)	=	0.3750"
Channel I.D. (Dc)	=	15.2500"	Shell thickness (Ts)	=	0.5000"
Shell I.D. (Ds)	=	15.0000"			
Press Chan (Pt)	=	328.0psi	Press Shell (Ps)	=	463.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps = 58260psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 27.1800psi	Vt= .300		
Temp CH Tc= 385deg.F	Sc = 17100psi	Syc= 30065psi	Ec = 27.7600psi	Vc=.300	Spss= 60130psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	

Mu= 0.2000	d*= 0.6406"	P*= 0.9375"	Mu*= 0.3167	hg'= 0.0000"	hgs'= 0.0000"
W = 0.0#	wj = 0.0"	ao = 7.135405064"			
Rho s = 1.051096559	Rho c = 1.068614841	Chi s = 0.489028096			
Chi t = 0.690186620	Ks = 4060272.000Lb/In	Kt = 29202.72852Lb/In			
Ks,t = 0.751553595	j = 1.000000000	Beta s = 0.652987421in ⁻¹			
Kappa s = 403557.5938#	Lambda s = 6561481.500psi	Beta c = 0.750982881in ⁻¹			
Kappa c = 201349.5469#	Lambda c = 3951385.000psi	h/p = 3.200000048			
E* = 8183711.000psi	v* = 0.343179107	Xa = 1.701923966			
Zd = 0.290400684	Zv = 0.171428442	Zm = 0.700460076			
K = 1.121169686	F = 1.091558099	Phi = 1.466158032			
Q1 = -0.098789386	Qz1 = 1.147183657	Qz2 = 0.428855240			
U = 0.857710540	Gamma = 0.0"	Omega s = 0.0in ²			
Omega c = 0.0in ²	Omega s* = -2.835061073in ²	Omega c* = -1.925397873in ²			
Delta s = 0.000003540In ³ /Lb	Delta c = 0.000004750In ³ /Lb	Gamma b = 0.0			
Ps' = 712.1762695psi	Pt' = 723.7817993psi	P Gamma = 0.0psi			
PW = 0.0psi	Prim = 11.47403431psi	Es,w = 0.850000024			
Pe = -0.052605059psi	Q2 = -336.0176392#	Q3 = 250.8166199			
Fm = 125.4954987	Es,w1 = 0.0	lt = 27.50000000"			
Rt = 0.237638906"	Ft = 115.7217941	Ct = 159.8189392			
Fs = 1.250000000	Stb = 10720.27539psi	Ft,min = -174.4750366			
Sigmat,1 = -45.43417740psi	Sigmat,2 = 47.03876877psi	Ft,max = 179.1358643			
Sigma s,m = 2473.249023psi	Sigma s,b = 7525.347168psi	Sigma s = 9998.595703psi			
Sigma c,m = 3254.634766psi	Sigma c,b = 11516.78906psi	Sigma c = 14771.42383psi			

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas=0		Betac, Kappac, Lambdac, Deltac, Omegac=0			
F	= 0.247800201	Phi	= 0.332840055	Q1	= -0.004834560
Qz1	= 1.214750290	Qz2	= 0.704934061	U	= 1.409868121
Omega s*	= 0.068162687in^2	Omega c*	= 0.569926202in^2	Prim	= 4.302555561psi
Pe	= -2.829133034psi	Q2	= -126.0005569#	Q3	= 1.744656920

Sigma	-707 psi	<= 1.5S	29130 psi	[OK Per Simply Sup]
Tau	0 psi	<= .8S	15536 psi	[OK]
Sigmat,max	47 psi	< St	13384 psi	[OK]
Sigmat,min	-45 psi	<= Stb	10720 psi	[OK]
Sigma s	9998 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	14771 psi	<= Spss	60130 psi	[OK Per Simply Sup]
Sigmatas,m	2473 psi	<= SsEs,w	14535 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0000"	F-F Inside Tubesheets (L)	=	161.8750"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.8500"	Channel thickness (Tc)	=	0.3750"
Channel I.D. (Dc)	=	15.2500"	Shell thickness (Ts)	=	0.5000"
Shell I.D. (Ds)	=	15.0000"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+ Temp TS T = 515deg.F S = 19420psi Sy = 29045psi E = 26.9950psi V = .300 Sps = 58260psi						
Temp TBS Tt= 515deg.F St = 13384psi Syt= 21005psi Et = 27.1800psi Vt=.300						
Temp CH Tc= 385deg.F Sc = 17100psi Syc= 30065psi Ec = 27.7600psi Vc=.300 Spsc= 60130psi						
Temp SH Ts= 515deg.F Ss = 17100psi Sys= 28244psi Es = 26.9950psi Vs=.300 Spss= 56489psi						
Tt,m = 390deg.F At,m= 7.0800In/In						
Ts,m = 422deg.F As,m= 7.1440In/In						
Mu= 0.2000 d*= 0.6406" P*= 0.9375" Mu*= 0.3167 hg'= 0.0000" hgs'= 0.0000"						
W = 0.0# Wj = 0.0" ao = 7.135405064"						
Rho s = 1.051096559 Rho c = 1.068614841 Chi s = 0.489028096						
Chi t = 0.690186620 Ks = 4060272.000Lb/In Kt = 29202.72852Lb/In						
Ks,t = 0.751553595 j = 1.000000000 Beta s = 0.652987421in^-1						
Kappa s = 403557.5938# Lambda s = 6561481.500psi Beta c = 0.750982881in^-1						
Kappa c = 201349.5469# Lambda c = 3951385.000psi h/p = 3.200000048						
E* = 8183711.000psi v* = 0.343179107 Xa = 1.701923966						
Zd = 0.290400684 Zv = 0.171428442 Zm = 0.700460076						
K = 1.121169686 F = 1.091558099 Phi = 1.466158032						
Q1 = -0.098789386 Qz1 = 1.147183657 Qz2 = 0.428855240						
U = 0.857710540 Gamma = -0.040321112" Omega s = 2.903223753in^2						
Omega c = 2.495324135in^2 Omega s* = -2.835061073in^2 Omega c* = -1.925397873in^2						
Delta s = 0.000003540In^3/Lb Delta c = 0.000004750In^3/Lb Gamma b = 0.0						
Ps' = 0.0psi Pt' = 0.0psi P Gamma = -1361.885132psi						
PW = 0.0psi Prim = 0.0psi Es,w = 0.850000024						
Pe = -544.8251343psi Q2 = 0.0# Q3 = -0.098789386						
Fm = 0.142736554 Es,w1 = 0.0 lt = 0.0						
Rt = 0.0 Ft = 0.0 Ct = 0.0						
Fs = 0.0 Stb = 0.0psi Ft,min = 0.814228356						
Sigmat,1 = 2205.285889psi Sigmat,2 = 3103.194336psi Ft,max = 1.145751119						
Sigma s,m = -3579.255371psi Sigma s,b = -7406.755371psi Sigma s = 10986.01074psi						
Sigma c,m = 0.0psi Sigma c,b = 8855.452148psi Sigma c = 8855.452148psi						
Thermal Expansion Acting on this Case (Yes)						
Sigma	-8333 psi	<= Sps	58260 psi	[OK		
Tau	-3239 psi	<= .8S	15536 psi	[OK		
Sigmat,max	3103 psi	<= 2St	26769 psi	[OK		
Sigma s	10986 psi	<= Spss	56489 psi	[OK		
Sigma c	8855 psi	<= Spsc	60130 psi	[OK		
Sigmas,m	-3579 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmas,m	-3579 psi	<= Ss,b	13949 psi	[OK		Step 10

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Job No.: X-7926

By: MD/DN

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0000"		
F-F Tubesheets (Lt)	=	167.8750"	F-F Inside Tubesheets (L)	= 161.8750"
Exp length of tube (ltx)	=	2.8500"	Exp depth ratio (Rho)	= 0.9500"
Channel I.D. (Dc)	=	15.2500"	Channel thickness (Tc)	= 0.3750"
Shell I.D. (Ds)	=	15.0000"	Shell thickness (Ts)	= 0.5000"
Press Chan (Pt)	=	328.0psi	Press Shell (Ps)	= 0.0psi
Corr Chan	=	0.00000"	Corr Shell	= 0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	= 0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+						
Temp TS T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps =	58260psi
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 27.1800psi	Vt=.300		
Temp CH Tc= 385deg.F	Sc = 17100psi	Syc= 30065psi	Ec = 27.7600psi	Vc=.300	Spssc=	60130psi
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss=	56489psi
Tt,m = 390deg.F				At,m= 7.0800In/In		
Ts,m = 422deg.F				As,m= 7.1440In/In		
Mu= 0.2000 d*= 0.6406"	P*= 0.9375"	Mu*= 0.3167 hg'= 0.0000"	hgs'= 0.0000"			
W = 0.0#	wj = 0.0"	ao = 7.135405064"				
Rho s = 1.051096559	Rho c = 1.068614841	Chi s = 0.489028096				
Chi t = 0.690186620	Ks = 4060272.000Lb/In	Kt = 29202.72852Lb/In				
Ks,t = 0.751553595	j = 1.000000000	Beta s = 0.652987422lin ⁻¹				
Kappa s = 403557.5938#	Lambda s = 6561481.500psi	Beta c = 0.750982881lin ⁻¹				
Kappa c = 201349.5469#	Lambda c = 3951385.000psi	h/p = 3.200000048				
E* = 8183711.000psi	v* = 0.343179107	Xa = 1.701923966				
Zd = 0.290400684	Zv = 0.171428442	Zm = 0.700460076				
K = 1.121169686	F = 1.091558099	Phi = 1.466158032				
Q1 = -0.098789386	Qz1 = 1.147183657	Qz2 = 0.428855240				
U = 0.857710540	Gamma = -0.040321112"	Omega s = 2.903223753in ²				
Omega c = 2.495324135in ²	Omega s* = -2.835061073in ²	Omega c* = -1.925397873in ²				
Delta s = 0.000003540In ³ /Lb	Delta c = 0.000004750In ³ /Lb	Gamma b = 0.0				
Ps' = 0.0psi	Pt' = 723.7817993psi	P Gamma = -1361.885132psi				
PW = 0.0psi	Prim = -10.63892651psi	Es,w = 0.850000024				
Pe = -838.6317749psi	Q2 = 311.5614929#	Q3 = -0.113383077				
Fm = 0.136446759	Es,w1 = 0.0	lt = 0.0"				
Rt = 0.0"	Ft = 0.0	Ct = 0.0				
Fs = 0.0	Stb = 0.0psi	Ft,min = 0.824423552				
Sigmat,1 = 2311.642334psi	Sigmat,2 = 3608.101318psi	Ft,max = 1.135398865				
Sigma s,m = -3354.620361psi	Sigma s,b = -11106.43945psi	Sigma s = 14461.05957psi				
Sigma c,m = 3254.634766psi	Sigma c,b = 22313.87305psi	Sigma c = 25568.50781psi				
Thermal Expansion Acting on this Case (Yes)						
Sigma	-12262 psi	<= Sps	58260 psi	[OK		
Tau	-4986 psi	<= .8S	15536 psi	[OK		
Sigmat,max	3608 psi	<= 2St	26769 psi	[OK		
Sigma s	14461 psi	<= Spss	56489 psi	[OK		
Sigma c	25568 psi	<= Spssc	60130 psi	[OK		
Sigmas,m	-3354 psi	<= 2Ss	34200 psi	[OK		] Step 10
Sigmas,m	-3354 psi	<= Ss,b	13949 psi	[OK		] Step 10

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Job No.: X-7926

By: MD/DN

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0000"	F-F Inside Tubesheets (L)=	161.8750"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	= 0.9500"
Exp length of tube (ltx)	=	2.8500"	Channel thickness (Tc)	= 0.3750"
Channel I.D. (Dc)	=	15.2500"	Shell thickness (Ts)	= 0.5000"
Shell I.D. (Ds)	=	15.0000"		
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	= 463.0psi
Corr Chan	=	0.00000"	Corr Shell	= 0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	= 0.00000"

	Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+							
Temp TS	T = 515deg.F	S = 19420psi	Sy = 29045psi	E = 26.9950psi	V = .300	Sps =	58260psi
Temp TBS	Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 27.1800psi	Vt=.300		
Temp CH	Tc= 385deg.F	Sc = 17100psi	Syc= 30065psi	Ec = 27.7600psi	Vc=.300	Spssc=	60130psi
Temp SH	Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss=	56489psi
Tt,m	= 390deg.F		At,m= 7.0800In/In				
Ts,m	= 422deg.F		As,m= 7.1440In/In				
Mu=	0.2000	d*= 0.6406"	P*= 0.9375"	Mu*= 0.3167	hg'= 0.0000"	hgs'= 0.0000"	
W	=	0.0#	wj	=	0.0"	ao	= 7.135405064"
Rho s	=	1.051096559	Rho c	=	1.068614841	Chi s	= 0.489028096
Chi t	=	0.690186620	Ks	=	4060272.000Lb/In	Kt	= 29202.72852Lb/In
Ks,t	=	0.751553595	j	=	1.000000000	Beta s	= 0.652987421in ⁻¹
Kappa s	=	403557.5938#	Lambda s	=	6561481.500psi	Beta c	= 0.750982881in ⁻¹
Kappa c	=	201349.5469#	Lambda c	=	3951385.000psi	h/p	= 3.200000048
E*	=	8183711.000psi	v*	=	0.343179107	Xa	= 1.701923966
Zd	=	0.290400684	Zv	=	0.171428442	Zm	= 0.700460076
K	=	1.121169686	F	=	1.091558099	Phi	= 1.466158032
Q1	=	-0.098789386	Qz1	=	1.147183657	Qz2	= 0.428855240
U	=	0.857710540	Gamma	=	-0.040321112"	Omega s	= 2.903223753in ²
Omega c	=	2.495324135in ²	Omega s*	=	-2.835061073in ²	Omega c**	=-1.925397873in ²
Delta s	=	0.000003540In ³ /Lb	Delta c	=	0.000004750In ³ /Lb	Gamma b	= 0.0
Ps'	=	712.1762695psi	Pt'	=	0.0psi	P Gamma	=-1361.885132psi
PW	=	0.0psi	Prim	=	22.11296082psi	Es,w	= 0.850000024
Pe	=	-251.0711212psi	Q2	=	-647.5791016#	Q3	= 0.002529120
Fm	=	0.186404258	Es,w1	=	0.0	lt	= 0.0"
Rt	=	0.0"	Ft	=	0.0	Ct	= 0.0
Fs	=	0.0	Stb	=	0.0psi	Ft,min	= 0.743447363
Sigmat,1	=	2053.495850psi	Sigmat,2	=	2645.326172psi	Ft,max	= 1.217622638
Sigma s,m	=	-1330.641724psi	Sigma s,b	=	3818.275391psi	Sigma s	= 5148.916992psi
Sigma c,m	=	0.0psi	Sigma c,b	=	6913.820801psi	Sigma c	= 6913.820801psi
Thermal Expansion Acting on this Case (Yes)							
Sigma	-5015 psi	<= Sps	58260 psi	[OK	]		
Tau	-1492 psi	<= .8S	15536 psi	[OK	]		
Sigmat,max	2645 psi	<= 2St	26769 psi	[OK	]		
Sigma s	5148 psi	<= Spss	56489 psi	[OK	]		
Sigma c	6913 psi	<= Spssc	60130 psi	[OK	]		
Sigmas,m	-1330 psi	<= 2Ss	34200 psi	[OK	] Step 10		
Sigmas,m	-1330 psi	<= Ss,b	13949 psi	[OK	] Step 10		

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Job No.: X-7926

By: MD/DN

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(FTS) Configuration a
 Tubesheet Thickness (h) = 3.0000"
 F-F Tubesheets (Lt) = 167.8750" F-F Inside Tubesheets (L) = 161.8750"
 Exp length of tube (ltx) = 2.8500" Exp depth ratio (Rho) = 0.9500"
 Channel I.D. (Dc) = 15.2500" Channel thickness (Tc) = 0.3750"
 Shell I.D. (Ds) = 15.0000" Shell thickness (Ts) = 0.5000"
 Press Chan (Pt) = 328.0psi Press Shell (Ps) = 463.0psi
 Corr Chan = 0.00000" Corr Shell = 0.00000"
 Corr Chan side TS (ct) = 0.00000" Corr Shell side TS (cs) = 0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+ Temp TS T = 515deg.F S = 19420psi Sy = 29045psi E = 26.9950psi V = .300 Sps = 58260psi						
Temp TBS Tt= 515deg.F St = 13384psi Syt= 21005psi Et = 27.1800psi Vt=.300						
Temp CH Tc= 385deg.F Sc = 17100psi Syc= 30065psi Ec = 27.7600psi Vc=.300 Spssc= 60130psi						
Temp SH Ts= 515deg.F Ss = 17100psi Sys= 28244psi Es = 26.9950psi Vs=.300 Spss= 56489psi						
Tt,m = 390deg.F At,m= 7.0800In/In						
Ts,m = 422deg.F As,m= 7.1440In/In						
Mu= 0.2000 d*= 0.6406" P*= 0.9375" Mu*= 0.3167 hg'= 0.0000" hgs' = 0.0000"						
W = 0.0# wj = 0.0" ao = 7.135405064"						
Rho s = 1.051096559 Rho c = 1.068614841 Chi s = 0.489028096						
Chi t = 0.690186620 Ks = 4060272.000Lb/In Kt = 29202.72852Lb/In						
Ks,t = 0.751553595 j = 1.000000000 Beta s = 0.652987421in^-1						
Kappa s = 403557.5938# Lambda s = 6561481.500psi Beta c = 0.750982881in^-1						
Kappa c = 201349.5469# Lambda c = 3951385.000psi h/p = 3.200000048						
E* = 8183711.000psi v* = 0.343179107 Xa = 1.701923966						
Zd = 0.290400684 Zv = 0.171428442 Zm = 0.700460076						
K = 1.121169686 F = 1.091558099 Phi = 1.466158032						
Q1 = -0.098789386 Qz1 = 1.147183657 Qz2 = 0.428855240						
U = 0.857710540 Gamma = -0.040321112" Omega s = 2.903223753in^2						
Omega c = 2.495324135in^2 Omega s* = -2.835061073in^2 Omega c* = -1.925397873in^2						
Delta s = 0.000003540In^3/Lb Delta c = 0.000004750In^3/Lb Gamma b = 0.0						
Ps' = 712.1762695psi Pt' = 723.7817993psi P Gamma = -1361.885132psi						
PW = 0.0psi Prim = 11.47403431psi Es,w = 0.850000024						
Pe = -544.8777466psi Q2 = -336.0176392# Q3 = -0.074564837						
Fm = 0.153177202 Es,w1 = 0.0 lt = 0.0"						
Rt = 0.0" Ft = 0.0 Ct = 0.0						
Fs = 0.0 Stb = 0.0psi Ft,min = 0.797305107						
Sigmat,1 = 2159.851807psi Sigmat,2 = 3150.232910psi Ft,max = 1.162935019						
Sigma s,m = -1106.006470psi Sigma s,b = 118.5915146psi Sigma s = 1224.598022psi						
Sigma c,m = 3254.634766psi Sigma c,b = 20372.24219psi Sigma c = 23626.87695psi						
Thermal Expansion Acting on this Case (Yes)						
Sigma -8944 psi <= Sps 58260 psi [OK]						
Tau -3239 psi <= .8S 15536 psi [OK]						
Sigmat,max 3150 psi <= 2St 26769 psi [OK]						
Sigma s 1224 psi <= Spss 56489 psi [OK]						
Sigma c 23626 psi <= Spssc 60130 psi [OK]						
Sigmas,m -1106 psi <= 2Ss 34200 psi [OK] Step 10						
Sigmas,m -1106 psi <= Ss,b 13949 psi [OK] Step 10						

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By: MD/DN

3/9/2011

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(FTS) Configuration a
 Tubesheet Thickness (h) = 2.7500"
 F-F Tubesheets (Lt) = 167.6250" F-F Inside Tubesheets (L) = 162.1250"
 Exp length of tube (ltx) = 2.6125" Exp depth ratio (Rho) = 0.9500"
 Channel I.D. (Dc) = 15.5000" Channel thickness (Tc) = 0.2500"
 Shell I.D. (Ds) = 15.2500" Shell thickness (Ts) = 0.3750"
 Press Chan (Pt) = -15.0psi Press Shell (Ps) = 15.0psi
 Corr Chan = 0.12500" Corr Shell = 0.12500"
 Corr Chan side TS (ct) = 0.12500" Corr Shell side TS (cs) = 0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 250deg.F	S = 20000psi	Sy = 32400psi	E = 28.3500psi	V = .300	Sps = 64800psi	
Temp TBS Tt = 250deg.F	St = 13400psi	Syt = 23400psi	Et = 28.5500psi	Vt = .300		
Temp CH Tc = 70deg.F	Sc = 17100psi	Syc = 35000psi	Ec = 29.2000psi	Vc = .300	Spcc = 70000psi	
Temp SH Ts = 250deg.F	Ss = 17100psi	Sys = 31500psi	Es = 28.3500psi	Vs = .300	Spss = 63000psi	
Tt,m = 250deg.F				At,m = 6.8000In/In		
Ts,m = 250deg.F				As,m = 6.8000In/In		

Mu = 0.2000 d* = 0.6436" P* = 0.9375" Mu* = 0.3135 hg' = 0.0000" hgs' = 0.0000"
 W = 0.0# wj = 0.0" ao = 7.135405064"
 Rho s = 1.068614841 Rho c = 1.086133122 Chi s = 0.489028096
 Chi t = 0.690186620 Ks = 3218876.000Lb/In Kt = 30627.38477Lb/In
 Ks,t = 0.568097115 j = 1.000000000 Beta s = 0.750982881in⁻¹
 Kappa s = 205628.9531# Lambda s = 4702419.500psi Beta c = 0.916105211in⁻¹
 Kappa c = 76551.83594# Lambda c = 2291088.000psi h/p = 2.933333397
 E* = 8473583.000psi v* = 0.345110953 Xa = 1.821840048
 Zd = 0.232152075 Zv = 0.167142197 Zm = 0.687355042
 K = 1.121169686 F = 0.791096687 Phi = 1.064112782
 Q1 = -0.063094512 Qz1 = 1.220655560 Qz2 = 0.681773245
 U = 1.363546491 Gamma = 0.0" Omega s = 2.351305246in²
 Omega c = 1.874728799in² Omega s* = -2.227341890in² Omega c* = -1.231768847in²
 Delta s = 0.000004650In³/Lb Delta c = 0.000006990In³/Lb Gamma b = 0.0
 Ps' = 26.27744675psi Pt' = -39.54505539psi P Gamma = 0.0psi
 PW = 0.0psi Prim = 1.389595509psi Es,w = 0.850000024
 Pe = 22.19907761psi Q2 = -29.96763420# Q3 = -0.116123118
 Fm = 0.131072953 Es,w1 = 0.0 lt = 27.50000000"
 Rt = 0.237638906 Ft = 115.7217941 Ct = 155.1886749
 Fs = 2.000000000 Stb = 7337.745605psi Ft,min = 0.778746426
 Sigmat,1 = 1.992301345psi Sigmat,2 = -41.38737488psi Ft,max = 1.171834469
 Sigma s,m = 81.05524445psi Sigma s,b = 792.9991455psi Sigma s = 874.0543823psi
 Sigma c,m = -228.8095245psi Sigma c,b = -1057.013428psi Sigma c = 1285.822998psi
 Thermal Expansion Acting on this Case (Yes)

Sigma	374 psi	<= Sps	64800 psi	[OK]	
Tau	143 psi	<= .8S	16000 psi	[OK]	
Sigmat,max	41 psi	<= 2St	26800 psi	[OK]	
Sigmat,min	-41 psi	<= Stb	7337 psi	[OK]	
Sigma s	874 psi	<= Spss	63000 psi	[OK]	
Sigma c	1285 psi	<= Spcc	70000 psi	[OK]	
Sigmatas,m	81 psi	<= 2Ss	34200 psi	[OK]	] Step 10

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REV 2
 By: MD/DN 6/23/2011

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(FTS) Configuration a

Tubesheet Thickness (h)	=	2.7500"	F-F Inside Tubesheets (L)	=	162.1250"
F-F Tubesheets (Lt)	=	167.6250"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.6125"	Channel thickness (Tc)	=	0.2500"
Channel I.D. (Dc)	=	15.5000"	Shell thickness (Ts)	=	0.3750"
Shell I.D. (Ds)	=	15.2500"			

Press Chan (Pt)	=	15.0psi	Press Shell (Ps)	=	-15.0psi	
Corr Chan	=	0.12500"	Corr Shell	=	0.12500"	
Corr Chan side TS (ct)	=	0.12500"	Corr Shell side TS (cs)	=	0.12500"	

	Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+							
Temp TS	T = 250deg.F	S = 20000psi	Sy = 32400psi	E = 28.3500psi	V = .300	Sps =	64800psi
Temp TBS	Tt= 250deg.F	St = 13400psi	Syt= 23400psi	Et = 28.5500psi	Vt= .300		
Temp CH	Tc= 250deg.F	Sc = 17100psi	Syc= 31500psi	Ec = 28.3500psi	Vc= .300	Spssc=	63000psi
Temp SH	Ts= 250deg.F	Ss = 17100psi	Sys= 31500psi	Es = 28.3500psi	Vs= .300	Spsss=	63000psi
Tt,m	= 250deg.F				At,m=	6.8000In/In	
Ts,m	= 250deg.F				As,m=	6.8000In/In	
Mu=	0.2000	d*= 0.6436"	P*= 0.9375"	Mu*= 0.3135	hg'= 0.0000"	hgs'= 0.0000"	
W	=	0.0#	wj	=	0.0"	ao	= 7.135405064"
Rho s	=	1.068614841	Rho c	=	1.086133122	Chi s	= 0.489028096
Chi t	=	0.690186620	Ks	=	3218876.000Lb/In	Kt	= 30627.38477Lb/In
Ks,t	=	0.568097115	j	=	1.000000000	Beta s	= 0.750982881in ⁻¹
Kappa s	=	205628.9531#	Lambda s	=	4702419.500psi	Beta c	= 0.916105211in ⁻¹
Kappa c	=	74323.43750#	Lambda c	=	2224395.000psi	h/p	= 2.933333397
E*	=	8473583.000psi	v*	=	0.345110953	Xa	= 1.821840048
Zd	=	0.232152075	Zv	=	0.167142197	Zm	= 0.687355042
K	=	1.121169686	F	=	0.785942256	Phi	= 1.057179570
Q1	=	-0.062597513	Qz1	=	1.221113086	Qz2	= 0.683654904
U	=	1.367309809	Gamma	=	0.0"	Omega s	= 2.351305246in ²
Omega c	=	1.874728680in ²	Omega s*	=	-2.227341890in ²	Omega c*	= -1.231768727in ²
Delta s	=	0.000004650in ³ /Lb	Delta c	=	0.000007200in ³ /Lb	Gamma b	= 0.0
Ps'	=	-26.27744675psi	Pt'	=	39.54505539psi	P Gamma	= 0.0psi
PW	=	0.0psi	Prim	=	-1.393430710psi	Es,w	= 0.850000024
Pe	=	-22.19604301psi	Q2	=	30.05034447#	Q3	= -0.115779750
Fm	=	0.131214440	Es,w1	=	0.0	lt	= 27.500000000"
Rt	=	0.237638906"	Ft	=	115.7217941	Ct	= 155.1886749
Fs	=	2.000000000	Stb	=	7337.745605psi	Ft,min	= 0.778442085
Sigmat,1	=	-2.037630558psi	Sigmat,2	=	41.40456772psi	Ft,max	= 1.172150493
Sigma s,m	=	-81.02886963psi	Sigma s,b	=	-793.9796143psi	Sigma s	= 875.0084839psi
Sigma c,m	=	228.8095245psi	Sigma c,b	=	1047.676636psi	Sigma c	= 1276.486206psi
Thermal Expansion Acting on this Case (Yes)							
Sigma	-375 psi	<= Sps	64800 psi	[OK			
Tau	-143 psi	<= .8S	16000 psi	[OK			
Sigmat,max	41 psi	<= 2St	26800 psi	[OK			
Sigmat,min	-2 psi	<= Stb	7337 psi	[OK			
Sigma s	875 psi	<= Spss	63000 psi	[OK			
Sigma c	1276 psi	<= Spssc	63000 psi	[OK			
Sigmas,m	-81 psi	<= 2Ss	34200 psi	[OK			] Step 10
Sigmas,m	-81 psi	<= Ss,b	17100 psi	[OK			] Step 10

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***** TUBE CALCULATIONS *****

Tube size.....=	0.75000"	Straight or U-Tube.=	Straight Tube	
Internal Press...=	343.00000psig*	External press.....=	478.00000psig*	
Temperature used.=	515.00000deg.	Tube Matl MT# 34...=	SA-179	
Stress Oper.....=	13384.99902psi	Stress Atm.....=	13400.00000psi	
Yield.....=	21005.00195psi	Inside Radius.....=	0.29200"	
Joint Eff.....=	1.00000	Mill tol...= 0.00% =	0.00000"	
Wall Thk. t.....=	0.08300"	Tube gauge.....=		
Mean Rad., R.....=	0.00000"	Tube Length.....=	168.00000"	
Corrosion ID.....=	0.00000"	Corrosion OD.....=	0.00000"	
Thinning factor..=	1.00000			
tr = PR/SE - .6P =	0.00760"	(tr*TF)+MT+CA.....=	0.00760"	
MAWP Internal H/C=	3250.30688psig	MAWP Internal N/C..=	3253.94971psig	
MAWP External H/C=	1928.78552psig	MAWP External N/C..=	2159.90234psig	

* DESIGN PRESSURE HAS BEEN INCREASED BY 8 PSI DUE TO STATIC HEAD

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Customer : SOUTHERN / MISS. POWER CO.		Item No.	HX 0052	W.O. : X-7926
Subject : Seismic and Wind Loading Factors		Date	3/8/2011	By : MD/DN
Wind Loading		Seismic Loading		
Given Data: From Customer Spec		Given Data: From Customer Spec		
Exposure Cat C		Occupancy Cat: III		
Occupancy Cat III		Design Cat: B		
Importance Fact 1.150		Reliability 1		
Kzt 1.000		Importance Factor 1.25		
V 100.000 mph		Site Class C		
qz = 0.00256KzKztKdV^2I				
Height of vessel 15.000 ft or less				
Alpha 9.5 Table 6-2		R (Table 15.4.2) 3		
Zg 900 Table 6-2				
Kz 0.849 Table 6-3				
Kd 0.950 (Table 6-4)				
qz = 23.742				
G 0.850				
Cf = 0.800 (Fig 6-21)		Sds 0.1630 g		
Wind Pressure = qz*G*Cf		Sd1 0.0990 g		
16.144 psf				
		Cs = Sds/(R/I) 0.0679		
		Min Cs = 0.0100		
		SNOW LOADS		
		Pg 5 psf		
		CAT C		
		Ct 1.2		
		I 1.1		
		Pf = 0.7*Ct*I 6.6 psf		
		SNOW LOAD AREA 28 sf		
		ADDITIONAL WEIGHT 184.8 lbs		

Input Echo, Leg & Lug Item 1, Description: X-7926 SUPPORTS

Design Internal Pressure		463.00	psig	← INCLUDES 8 PSI FOR STATIC HEAD
Design Temperature for Attachment	TEMP	515.00	F	
Vessel Outside Diameter	OD	16.000	in.	
Vessel Wall Thickness	Ts	0.4375	in.	
Vessel Corrosion Allowance	Cas	0.1250	in.	
Vessel Material		SA-106 B		
Vessel Material UNS Number		K03006		
Vessel Allowable Stress at Design	S	17100.00	psi	
Analysis Type:		Support Lug		
Tangent to Tangent Length of Vessel	TANTAN	13.431	ft.	
Height of Bottom Tangent Above Grade		1.677	ft.	

Empty Weight of Vessel	Wemp	4484.80	lb.	← WEIGHTS INCLUDE 316.8 LB
Operating Weight of Vessel (vertical load)	W	5684.80	lb.	FOR SNOW LOADING
Force Coefficient (Shape factor)	Cf	0.800		
Additional Area		0.00	in ²	
User defined Wind Pressure		16.144	psf	
Design Wind Code	ASCE 7-05			
Type of Hill		None		

User Entered Seismic Zone Factor	Cs	0.068		
Additional Horizontal Force on Vessel	FF	0.00	lb.	
Location of Horizontal Force above Base Point	FH	0.00	ft.	

Support Lug with Full Reinforcing Rings?		No		
Number of Support Lugs	NLUG	2		
Location of Support Lugs above Base Point	LLUG	11.7500	ft.	
Distance from Vessel OD to Lug Contact Point	DLUG	7.6250	in.	
Lug Support Force Bearing Width	WFB	4.0000	in.	
Support Lug Material		SA-516 70		
Support Lug Material UNS Number		K02700		
Support Lug Yield Stress	Ylug	30715.00	psi	
Radial Width of bottom Support Lug Plate	Wpl	9.1250	in.	
Effective Force Bearing Length	Lpl	6.0000	in.	
Thickness of bottom Support Lug Plate	Tpl	0.7500	in.	
Distance between Gussets	Dgp	5.0000	in.	
Mean Width of Gusset Plate	Wgp	5.2500	in.	
Height of Gusset Plate	Hgp	6.2500	in.	
Thickness of Gusset Plate	Tgp	0.5000	in.	

Perform WRC-107 (Loal Stress) Analysis on Lug		Yes		
Lug Pad Width (Circumferential direction)	wpad	10.000	in.	
Lug Pad Length (Longitudinal direction)	lpad	11.000	in.	
Lug Pad Thickness	tpad	0.375	in.	

Occasional Load Factor (AISC A5.2)	Occfac	1.00		
------------------------------------	--------	------	--	--

Results for support lugs, Description : X-7926 SUPPORTS

Vessel Wind Area, Af		: 4060.200	in ²
Vessel Wind load = P * Af * Cf		: 455.194	lb.
Location of Centroid above Base Point		: 8.393	ft.

Overturning Moment at Support Lug	Mlug	1528.	ft.lb.
Weight Load at the top of one Lug		2842.	lb.
Shear at top of one Lug		0.	lb.

Distance from vessel center to Lug centroid [Rlug]:

$$= (OD / 2.0 + DLUG) / 12.0$$

$$= (16.0000 / 2.0 + 7.6250) / 12.0$$

$$= 1.30 \text{ ft.}$$

Force on one Lug (compression side) [Flug]:

$$= (W/Nlug + Mlug / (Rlug * 2))$$

$$= (5684 / 2 + 1528 / (1.30 * 2))$$

$$= 3429.23 \text{ lb.}$$

BOLTING:
(4) ¾" DIAM.
A325 BOLTS
TRANSV. LOAD
2843 LB.
CALC. STRESS =
1177 PSI
BOLTING IS
ACCEPTABLE

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Bearing Area [Ba]:

$$= L_{pl} * W_{fb}$$

$$= 6.000 * 4.000$$

$$= 24.00 \text{ in}^2$$

Bending Str. in bottom Plate (Unif. Load) Per Bednar p.156 [Spl2]:

$$= \text{Beta1} * \text{Flug}/\text{Ba} * \text{Wfb}^2 / \text{Tpl}^2 \text{ per Roark \& Young 5th Ed.}$$

$$= 0.903 * 3429.2 / 24.000 * 4.000^2 / 0.750^2$$

$$= 3671.65 \text{ psi}$$

Allowable Stress for Support Plate [Spa]:

$$= \text{Min}(0.66 * \text{Ylug} * \text{Occfac}, 0.75 * \text{Ylug})$$

$$= \text{Min}(0.66 * 30715 * 1.00 , 23036.2)$$

$$= 20271.90 \text{ psi}$$

Stress in Gusset Plate (Force / Gusset Plate Area) [Sgp]:

$$= (\text{Flug}/2) / (\text{Wgp} * \text{Tgp})$$

$$= (3429 / 2) / (5.250 * 0.5000)$$

$$= 653.19 \text{ psi}$$

Gusset Plate Allowable Stress [Sga]:

$$= (1 - (\text{Klr})^2 / (2 * \text{Cc}^2)) * \text{Fy} /$$

$$(5/3 + 3 * (\text{Klr}) / (8 * \text{Cc}) - (\text{Klr}^3) / (8 * \text{Cc}^3))$$

$$= (1 - (50.00)^2 / (2 * 136.52^2)) * 30715 /$$

$$(5/3 + 3 * (50.00) / (8 * 136.52) - (50.00^3) / (8 * 136.52^3))$$

$$= 15938.26 \text{ psi}$$

Maximum Compressive Stress in the Gussets per Bednar [SgpB]:

$$= \text{Flug} * (3 * \text{Dlug} - \text{Wpl}) / (\text{Tgp} * \text{Wpl}^2 * (\text{Sin}(\text{Alph}_G))^2)$$

$$= 3429 * (3 * 7.625 - 9.125) / (0.5000 * 9.125^2 * (\text{Sin}(38.88))^2)$$

$$= 2874.00 \text{ psi}$$

Gusset Plate Allowable Compressive Stress per Bednar [SgaB]:

$$= 18000. / (1 + (1/18000) * (\text{Hgp} / \text{Sin}(\text{Alph}_G) / (0.289 * \text{Tgp}))^2)$$

$$= 18000 / (1 + (1/18000) * (6.250 / \text{Sin}(38.88)) / (0.289 * 0.5000))^2)$$

$$= 14243.44 \text{ psi}$$

Summary Of Results

Stress (psi)	Actual	Allowable	P/F
Bottom Plt. Str. (Unif. Load-Bednar) :	3671.65	20271.90	Ok
Gusset Plt. Str. (Force/Gusset Area) :	653.19	15938.26	Ok
Gusset Plt. Max Comp. Str. (Bednar) :	2874.00	14243.44	Ok

Result of WRC-107 analysis on the Vessel-Support Lug Junction:

(Worst load case + Int. Pressure, ref. Pressure Vessel Design Manual, D. Moss)

Sustained (Wt) Forces/Moments for WRC107 Analysis :

Distance from Lug bottom to Lug centroid, b :	2.635	in.
Force on lug in Sustained Condition, Flug_Sus :	2842.40	lb.
Longitudinal Shear = Flug_Sus :	2842.40	lb.
Long. Moment = Flug_Sus * DLUG + FF / Nlug * b :	1806.11	ft.lb.

Occasional (Wind/Seismic) Forces/Moments for WRC107 Analysis :

Horizontal Live load max(Wind, Seismic), V :	455.194	lb.
Force on lug in Occasional Condition, Flug_Occ :	586.83	lb.
Longitudinal Shear = Flug_Occ :	586.83	lb.
Long. Moment = Flug_Occ * DLUG + V / Nlug * b :	422.86	ft.lb.

Parameter C11	C11	3.00	in.
Parameter C22	C22	5.00	in.
Pad Parameter C11P	C11P	10.00	in.
Pad Parameter C22P	C22P	11.00	in.
Thickness of Reinforcing Pad	tpad	0.38	in.

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WRC 107 Stress Calculation for SUSTained loads:

Radial Load	P	0.0	lb.
Circumferential Shear	VC	0.0	lb.
Longitudinal Shear	VL	2842.4	lb.
Circumferential Moment	MC	0.0	ft.lb.
Longitudinal Moment	ML	1806.1	ft.lb.
Torsional Moment	MT	0.0	ft.lb.

Dimensionless Parameters used : Gamma = 11.68

Dimensionless Loads for Cylindrical Shells at Attachment Junction:

Curves read for 1979	Beta	Figure	Value	Location
N(PHI) / (P/Rm)	0.303	4C	1.763	(A,B)
N(PHI) / (P/Rm)	0.303	3C	1.310	(C,D)
M(PHI) / (P)	0.226	2C1	0.067	(A,B)
M(PHI) / (P)	0.226	1C	0.095	(C,D)
N(PHI) / (MC/(Rm**2 * Beta))	0.221	3A !	0.403	(A,B,C,D)
M(PHI) / (MC/(Rm * Beta))	0.243	1A	0.091	(A,B,C,D)
N(PHI) / (ML/(Rm**2 * Beta))	0.263	3B !	1.370	(A,B,C,D)
M(PHI) / (ML/(Rm * Beta))	0.255	1B	0.036	(A,B,C,D)
N(x) / (P/Rm)	0.267	3C	1.436	(A,B)
N(x) / (P/Rm)	0.267	4C	1.844	(C,D)
M(x) / (P)	0.273	1C1	0.081	(A,B)
M(x) / (P)	0.273	2C	0.052	(C,D)
N(x) / (MC/(Rm**2 * Beta))	0.221	4A !	0.668	(A,B,C,D)
M(x) / (MC/(Rm * Beta))	0.295	2A	0.046	(A,B,C,D)
N(x) / (ML/(Rm**2 * Beta))	0.263	4B !	0.439	(A,B,C,D)
M(x) / (ML/(Rm * Beta))	0.289	2B	0.054	(A,B,C,D)

Note - The ! mark next to the figure name denotes curve value exceeded.

Stress Concentration Factors Kn = 1.00, Kb = 1.00

Stresses in the Vessel at the Attachment Junction

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb.	P	0	0	0	0	0	0	0	0
Circ. Bend.	P	0	0	0	0	0	0	0	0
Circ. Memb.	MC	0	0	0	0	0	0	0	0
Circ. Bend.	MC	0	0	0	0	0	0	0	0
Circ. Memb.	ML	-2236	-2236	2236	2236	0	0	0	0
Circ. Bend.	ML	-4862	4862	4862	-4862	0	0	0	0
Tot. Circ. Str.		-7098	2625	7098	-2625	0	0	0	0
Long. Memb.	P	0	0	0	0	0	0	0	0
Long. Bend.	P	0	0	0	0	0	0	0	0
Long. Memb.	MC	0	0	0	0	0	0	0	0
Long. Bend.	MC	0	0	0	0	0	0	0	0
Long. Memb.	ML	-964	-964	964	964	0	0	0	0
Long. Bend.	ML	-6390	6390	6390	-6390	0	0	0	0
Tot. Long. Str.		-7354	5425	7354	-5425	0	0	0	0
Shear	VC	0	0	0	0	0	0	0	0
Shear	VL	0	0	0	0	-413	-413	413	413
Shear	MT	0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-413	-413	413	413
Str. Int.		7354	5425	7354	5425	826	826	826	826

Dimensionless Parameters used : Gamma = 25.10

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Dimensionless Loads for Cylindrical Shells at Pad edge:

Curves read for 1979	Beta	Figure	Value	Location
N(PHI) / (P/Rm)	0.707	4C !	2.131	(A,B)
N(PHI) / (P/Rm)	0.707	3C !	0.933	(C,D)
M(PHI) / (P)	0.659	2C1 !	0.008	(A,B)
M(PHI) / (P)	0.659	1C !	0.059	(C,D)
N(PHI) / (MC/(Rm**2 * Beta))	0.658	3A !	0.727	(A,B,C,D)
M(PHI) / (MC/(Rm * Beta))	0.734	1A !	0.063	(A,B,C,D)
N(PHI) / (ML/(Rm**2 * Beta))	0.679	3B !	1.200	(A,B,C,D)
M(PHI) / (ML/(Rm * Beta))	0.694	1B !	0.009	(A,B,C,D)
N(x) / (P/Rm)	0.685	3C !	0.933	(A,B)
N(x) / (P/Rm)	0.685	4C !	2.131	(C,D)
M(x) / (P)	0.689	1C1 !	0.020	(A,B)
M(x) / (P)	0.689	2C !	0.030	(C,D)
N(x) / (MC/(Rm**2 * Beta))	0.658	4A !	2.548	(A,B,C,D)
M(x) / (MC/(Rm * Beta))	0.818	2A !	0.027	(A,B,C,D)
N(x) / (ML/(Rm**2 * Beta))	0.679	4B !	0.655	(A,B,C,D)
M(x) / (ML/(Rm * Beta))	0.725	2B !	0.014	(A,B,C,D)

Note - The ! mark next to the figure name denotes curve value exceeded.

Stress Concentration Factors $K_n = 1.00$, $K_b = 1.00$

Stresses in the Vessel at the Edge of Reinforcing Pad

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P	P	0	0	0	0	0	0	0	0
Circ. Bend. P	P	0	0	0	0	0	0	0	0
Circ. Memb. MC	MC	0	0	0	0	0	0	0	0
Circ. Bend. MC	MC	0	0	0	0	0	0	0	0
Circ. Memb. ML	ML	-1774	-1774	1774	1774	0	0	0	0
Circ. Bend. ML	ML	-2143	2143	2143	-2143	0	0	0	0
Tot. Circ. Str.		-3918	368	3918	-368	0	0	0	0
Long. Memb. P	P	0	0	0	0	0	0	0	0
Long. Bend. P	P	0	0	0	0	0	0	0	0
Long. Memb. MC	MC	0	0	0	0	0	0	0	0
Long. Bend. MC	MC	0	0	0	0	0	0	0	0
Long. Memb. ML	ML	-1047	-1047	1047	1047	0	0	0	0
Long. Bend. ML	ML	-3375	3375	3375	-3375	0	0	0	0
Tot. Long. Str.		-4423	2328	4423	-2328	0	0	0	0
Shear VC	VC	0	0	0	0	0	0	0	0
Shear VL	VL	0	0	0	0	-413	-413	413	413
Shear MT	MT	0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-413	-413	413	413
Str. Int.		4423	2328	4423	2328	826	826	826	826

WRC 107 Stress Calculation for OCCasional loads:

Radial Load	P	0.0	lb.
Circumferential Shear	VC	0.0	lb.
Longitudinal Shear	VL	586.8	lb.
Circumferential Moment	MC	0.0	ft.lb.
Longitudinal Moment	ML	422.9	ft.lb.
Torsional Moment	MT	0.0	ft.lb.

Dimensionless Parameters used : $\Gamma = 11.68$

Stress Concentration Factors $K_n = 1.00$, $K_b = 1.00$

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Stresses in the Vessel at the Attachment Junction

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P		0	0	0	0	0	0	0	0
Circ. Bend. P		0	0	0	0	0	0	0	0
Circ. Memb. MC		0	0	0	0	0	0	0	0
Circ. Bend. MC		0	0	0	0	0	0	0	0
Circ. Memb. ML		-523	-523	523	523	0	0	0	0
Circ. Bend. ML		-1138	1138	1138	-1138	0	0	0	0
Tot. Circ. Str.		-1662	614	1662	-614	0	0	0	0
Long. Memb. P		0	0	0	0	0	0	0	0
Long. Bend. P		0	0	0	0	0	0	0	0
Long. Memb. MC		0	0	0	0	0	0	0	0
Long. Bend. MC		0	0	0	0	0	0	0	0
Long. Memb. ML		-225	-225	225	225	0	0	0	0
Long. Bend. ML		-1496	1496	1496	-1496	0	0	0	0
Tot. Long. Str.		-1721	1270	1721	-1270	0	0	0	0
Shear VC		0	0	0	0	0	0	0	0
Shear VL		0	0	0	0	-85	-85	85	85
Shear MT		0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-85	-85	85	85
Str. Int.		1721	1270	1721	1270	170	170	170	170

Dimensionless Parameters used : Gamma = 25.10

Stress Concentration Factors Kn = 1.00, Kb = 1.00

Stresses in the Vessel at the Edge of Reinforcing Pad

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P		0	0	0	0	0	0	0	0
Circ. Bend. P		0	0	0	0	0	0	0	0
Circ. Memb. MC		0	0	0	0	0	0	0	0
Circ. Bend. MC		0	0	0	0	0	0	0	0
Circ. Memb. ML		-415	-415	415	415	0	0	0	0
Circ. Bend. ML		-501	501	501	-501	0	0	0	0
Tot. Circ. Str.		-917	86	917	-86	0	0	0	0
Long. Memb. P		0	0	0	0	0	0	0	0
Long. Bend. P		0	0	0	0	0	0	0	0
Long. Memb. MC		0	0	0	0	0	0	0	0
Long. Bend. MC		0	0	0	0	0	0	0	0
Long. Memb. ML		-245	-245	245	245	0	0	0	0
Long. Bend. ML		-790	790	790	-790	0	0	0	0
Tot. Long. Str.		-1035	545	1035	-545	0	0	0	0
Shear VC		0	0	0	0	0	0	0	0
Shear VL		0	0	0	0	-85	-85	85	85
Shear MT		0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-85	-85	85	85
Str. Int.		1035	545	1035	545	170	170	170	170

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WRC 107 Stress Summations:

Vessel Stress Summation at Attachment Junction

Type of Stress Int.	Stress Values at (psi)							
Location	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Pm (SUS)	4955	5418	4955	5418	4955	5418	4955	5418
Circ. Pm (OCC)	0	0	0	0	0	0	0	0
Circ. Pm (TOTAL)	4955	5418	4955	5418	4955	5418	4955	5418
Circ. Pl (SUS)	-2236	-2236	2236	2236	0	0	0	0
Circ. Pl (OCC)	-523	-523	523	523	0	0	0	0
Circ. Pl (TOTAL)	-2760	-2760	2760	2760	0	0	0	0
Circ. Q (SUS)	-4862	4862	4862	-4862	0	0	0	0
Circ. Q (OCC)	-1138	1138	1138	-1138	0	0	0	0
Circ. Q (TOTAL)	-6000	6000	6000	-6000	0	0	0	0
Long. Pm (SUS)	2477	2477	2477	2477	2477	2477	2477	2477
Long. Pm (OCC)	0	0	0	0	0	0	0	0
Long. Pm (TOTAL)	2477	2477	2477	2477	2477	2477	2477	2477
Long. Pl (SUS)	-964	-964	964	964	0	0	0	0
Long. Pl (OCC)	-225	-225	225	225	0	0	0	0
Long. Pl (TOTAL)	-1190	-1190	1190	1190	0	0	0	0
Long. Q (SUS)	-6390	6390	6390	-6390	0	0	0	0
Long. Q (OCC)	-1496	1496	1496	-1496	0	0	0	0
Long. Q (TOTAL)	-7886	7886	7886	-7886	0	0	0	0
Shear Pm (SUS)	0	0	0	0	0	0	0	0
Shear Pm (OCC)	0	0	0	0	0	0	0	0
Shear Pm (TOTAL)	0	0	0	0	0	0	0	0
Shear Pl (SUS)	0	0	0	0	-413	-413	413	413
Shear Pl (OCC)	0	0	0	0	-85	-85	85	85
Shear Pl (TOTAL)	0	0	0	0	-498	-498	498	498
Shear Q (SUS)	0	0	0	0	0	0	0	0
Shear Q (OCC)	0	0	0	0	0	0	0	0
Shear Q (TOTAL)	0	0	0	0	0	0	0	0
Pm (SUS)	4955	5418	4955	5418	4955	5418	4955	5418
Pm (SUS+OCC)	4955	5418	4955	5418	4955	5418	4955	5418
Pm+Pl (SUS)	2719	3182	7192	7655	5022	5475	5022	5475
Pm+Pl (SUS+OCC)	2195	2658	7715	8178	5052	5500	5052	5500
Pm+Pl+Q (Total)	6599	9173	13716	6395	5052	5500	5052	5500
Type of Stress Int.	Max. S.I.		S.I. Allowable		Result			
	psi							
Pm (SUS)	5418		17100		Passed			
Pm (SUS+OCC)	5418		20520		Passed			
Pm+Pl (SUS)	7655		25650		Passed			
Pm+Pl (SUS+OCC)	8178		30780		Passed			
Pm+Pl+Q (TOTAL)	13716		51300		Passed			

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WRC 107 Stress Summations:**Vessel Stress Summation at Reinforcing Pad Edge**

Type of Stress Int.	Stress Values at (psi)							
Location	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Pm (SUS)	11162	11625	11162	11625	11162	11625	11162	11625
Circ. Pm (OCC)	0	0	0	0	0	0	0	0
Circ. Pm (TOTAL)	11162	11625	11162	11625	11162	11625	11162	11625
Circ. Pl (SUS)	-1774	-1774	1774	1774	0	0	0	0
Circ. Pl (OCC)	-415	-415	415	415	0	0	0	0
Circ. Pl (TOTAL)	-2190	-2190	2190	2190	0	0	0	0
Circ. Q (SUS)	-2143	2143	2143	-2143	0	0	0	0
Circ. Q (OCC)	-501	501	501	-501	0	0	0	0
Circ. Q (TOTAL)	-2645	2645	2645	-2645	0	0	0	0
Long. Pm (SUS)	5581	5581	5581	5581	5581	5581	5581	5581
Long. Pm (OCC)	0	0	0	0	0	0	0	0
Long. Pm (TOTAL)	5581	5581	5581	5581	5581	5581	5581	5581
Long. Pl (SUS)	-1047	-1047	1047	1047	0	0	0	0
Long. Pl (OCC)	-245	-245	245	245	0	0	0	0
Long. Pl (TOTAL)	-1292	-1292	1292	1292	0	0	0	0
Long. Q (SUS)	-3375	3375	3375	-3375	0	0	0	0
Long. Q (OCC)	-790	790	790	-790	0	0	0	0
Long. Q (TOTAL)	-4166	4166	4166	-4166	0	0	0	0
Shear Pm (SUS)	0	0	0	0	0	0	0	0
Shear Pm (OCC)	0	0	0	0	0	0	0	0
Shear Pm (TOTAL)	0	0	0	0	0	0	0	0
Shear Pl (SUS)	0	0	0	0	-413	-413	413	413
Shear Pl (OCC)	0	0	0	0	-85	-85	85	85
Shear Pl (TOTAL)	0	0	0	0	-498	-498	498	498
Shear Q (SUS)	0	0	0	0	0	0	0	0
Shear Q (OCC)	0	0	0	0	0	0	0	0
Shear Q (TOTAL)	0	0	0	0	0	0	0	0
Pm (SUS)	11162	11625	11162	11625	11162	11625	11162	11625
Pm (SUS+OCC)	11162	11625	11162	11625	11162	11625	11162	11625
Pm+Pl (SUS)	9388	9851	12937	13400	11193	11654	11193	11654
Pm+Pl (SUS+OCC)	8972	9435	13353	13816	11207	11666	11207	11666
Pm+Pl+Q (Total)	6326	12081	15998	11170	11207	11666	11207	11666
Type of Stress Int.	Max. S.I. psi				S.I. Allowable		Result	
Pm (SUS)	11625				17100		Passed	
Pm (SUS+OCC)	11625				20520		Passed	
Pm+Pl (SUS)	13400				25650		Passed	
Pm+Pl (SUS+OCC)	13816				30780		Passed	
Pm+Pl+Q (TOTAL)	15998				51300		Passed	

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The support gusset shall have a 1.500" diameter hole that may be used as a lifting lug.
Only the edge of the support in contact with the cylinder/pad used in analysis.

(Welds to support pad are not utilized and results are very conservative)

Design Internal Pressure		0.00	psig
Design Temperature for Attachment	TEMP	100.00	F
Vessel Outside Diameter	OD	16.000	in.
Vessel Wall Thickness	Ts	0.4375	in.
Vessel Corrosion Allowance	Cas	0.1250	in.
Vessel Material		SA-106 B	
Vessel Material UNS Number		K03006	
Vessel Allowable Stress at Design	S	17100.00	psi
Analysis Type:		Lifting Lug	

Empty Weight of Vessel	Wemp	4300.00	lb.
Operating Weight of Vessel (vertical load)	W	26250.00	lb.

Force Coefficient (Shape factor)	Cf	0.800	
Additional Area		0.00	in²
User defined Wind Pressure		16.144	psf
Design Wind Code	ASCE	7-95	
Type of Hill		None	

User Entered Seismic Zone Factor	Cs	0.068	
----------------------------------	----	-------	--

Lifting Lug Material		SA-516 70	
Lifting Lug Material UNS Number		K02700	
Lifting Lug Yield Stress	YIELD	38000.00	psi

Lifting Lug Orientation to Vessel	Perpendicular		
Total Height of Lifting Lug	w	8.0000	in.
Thickness of Lifting Lug	t	0.5000	in.
Diameter of Hole in Lifting Lug	dh	1.5000	in.
Radius of Semi-Circular Arc of Lifting Lug	r	2.3750	in.
Height of Lug from bottom to Center of Hole	h	3.0000	in.
Offset from Vessel OD to Center of Hole	off	4.0000	in.
Minimum thickness of Fillet Weld around Lug	tw	0.3750	in.
Length of weld along side of Lifting Lug	wl	6.2500	in.
Length of Weld along Bottom of Lifting Lug	wb	0.5000	in.
Lift Orientation		Vertical	
Force Along Vessel Axis	Fax	4300.00	lb.
Force Normal to Vessel	Fn	4300.00	lb.
Force Tangential to Vessel	Ft	0.00	lb.
Impact Factor	Impfac	1.50	

Occasional Load Factor (AISC A5.2)	Occfac	1.12	
------------------------------------	--------	------	--

Results for lifting lugs, Description : X-7926 FLL

Vessel Wind Area, Af	: 5088.000	in²
Vessel Wind load = P * Af * Cf	: 570.421	lb.
Location of Centroid above Base Point	: 8.948	ft.

Weld Group Inertia about the Circumferential Axis	ILC	17.239	in**4
Weld Group Centroid distance in the Long. Direction YLL		3.373	in.
Dist. of Weld Group Centroid from Lug bottom YLL_B		2.998	in.
Weld Group Inertia about the Longitudinal Axis	ILL	0.077	in**4
Weld Group Centroid Distance in the Circ. Direction YLC		0.250	in.

Applying the Impact factor to the loads:

Fax = 4300.00 * 1.50 = 6450.00	lb.
Fn = 4300.00 * 1.50 = 6450.00	lb.

Primary Shear Stress in the Welds due to Shear Loads [Ssll]:

= Sqrt(Fax² + Ft² + Fn²) / ((2*wl + wb) * tw)	
= Sqrt(6450² + 0² + 6450²) / ((2*6.2 + 0.5) * 0.3750)	
= 1871.11	psi

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Shear Stress in the Welds due to Bending Loads [Sblf]:

$$\begin{aligned}
 &= (F_n \cdot (h - Y_{LL_B})) \cdot Y_{LL} / I_{LC} + (F_{ax} \cdot off \cdot Y_{LL} / I_{LC}) + (F_t \cdot off \cdot Y_{LC} / I_{LL}) \\
 &= (6450 \cdot (3.000 - 2.998)) \cdot 3.373 / 17.239 + \\
 &\quad (6450 \cdot 4.000 \cdot 3.373 / 17.239) + \\
 &\quad (0 \cdot 4.000 \cdot 0.250 / 0.077) \\
 &= 5050.42 \text{ psi}
 \end{aligned}$$

Total Shear Stress for Combined Loads [St]:

$$\begin{aligned}
 &= S_{s11} + S_{blf} \\
 &= 1871.113 + 5050.416 \\
 &= 6921.53 \text{ psi}
 \end{aligned}$$

Allowable Shear Stress for Combined Loads [Sta]:

$$\begin{aligned}
 &= 0.4 \cdot \text{Yield} \cdot \text{Occfac} \text{ (AISC Shear All.)} \\
 &= 0.4 \cdot 38000 \cdot 1.12 \\
 &= 17100.00 \text{ psi}
 \end{aligned}$$

Shear Stress in Lug above Hole [Shs]:

$$\begin{aligned}
 &= \text{Sqrt}(F_{ax}^2 + F_n^2 + F_t^2) / S_{ha} \\
 &= \text{Sqrt}(6450^2 + 6450^2 + 0^2) / 1.625 \\
 &= 5613.34 \text{ psi}
 \end{aligned}$$

Allowable Shear Stress in Lug above Hole [Sas]:

$$\begin{aligned}
 &= 0.4 \cdot \text{Yield} \cdot \text{Occfac} \\
 &= 0.4 \cdot 38000 \cdot 1.12 \\
 &= 17100.00 \text{ psi}
 \end{aligned}$$

Pin Hole Bearing Stress [Pbs]:

$$\begin{aligned}
 &= \text{Sqrt}(F_{ax}^2 + F_n^2) / (t \cdot d_h) \\
 &= \text{Sqrt}(6450^2 + 6450^2) / (0.500 \cdot 1.500) \\
 &= 12162.24 \text{ psi}
 \end{aligned}$$

Allowable Bearing Stress [Pba]:

$$\begin{aligned}
 &= \text{Min}(0.75 \cdot \text{Yield} \cdot \text{Occfac}, 0.9 \cdot \text{Yield}) \text{ AISC Bearing All.} \\
 &= \text{Min}(0.75 \cdot 38000 \cdot 1.12, 34200.0) \\
 &= 32062.50 \text{ psi}
 \end{aligned}$$

Bending stress at the base of the lug [Fbs]:

$$\begin{aligned}
 &= F_t \cdot off / (w \cdot t^2 / 6) + F_{ax} \cdot off / (w^2 \cdot t / 6) \\
 &= 0 \cdot 4.000 / (8.000 \cdot 0.500^2 / 6) + \\
 &\quad 6450 \cdot 4.000 / (8.000^2 \cdot 0.500 / 6) \\
 &= 4837.50 \text{ psi}
 \end{aligned}$$

Tensile stress at the base of the lug [Fa]:

$$\begin{aligned}
 &= F_n / (w \cdot t) = 6450 / (8.000 \cdot 0.500) \\
 &= 1612.50 \text{ psi}
 \end{aligned}$$

Total Combined Stress at the base of the lug:

$$= F_{bs} + F_a = 6450.0 \text{ psi}$$

Lug Allowable Stress for Bending and Tension:

$$\begin{aligned}
 &= \text{Min}(0.66 \cdot \text{Yield} \cdot \text{Occfac}, 0.75 \cdot \text{Yield}) \\
 &= \text{Min}(0.66 \cdot 38000 \cdot 1.12, 28500.0) = 28215.0 \text{ psi}
 \end{aligned}$$

Note: Check the Shell Stresses using method such as WRC-107.

Summary Of Results

Stress (psi)	Actual	Allowable	P/F
Primary Shear Stress of Weld :	6921.53	17100.00	Ok
Shear Stress above Hole :	5613.34	17100.00	Ok
Pin Hole Bearing Stress :	12162.24	32062.50	Ok
Total Combined Stress at the lug base :	6450.00	28215.00	Ok



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Energy Exchanger Company

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Customer : MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES

Item No HX0052

W.O. : X-7926

Subject : LIFTING LUG CALCULATION

Date 3/9/2011

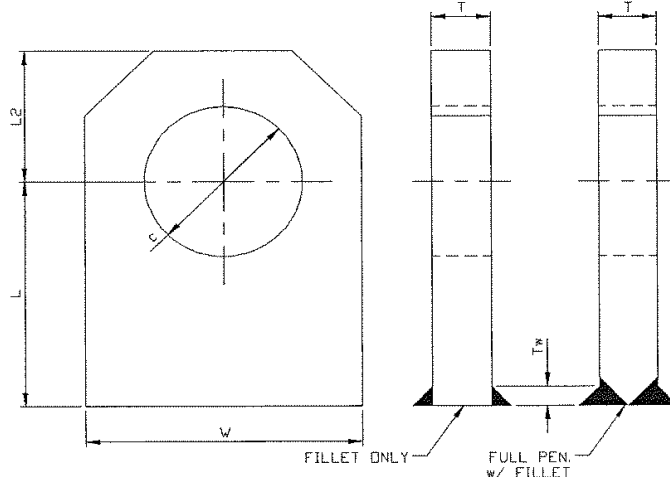
By : MD/DN

Note : Lugs are not designed for side loading. Do not side load lugs.

Material	SA-516-70
Yield Strength	38000 psi
T =	1/2 in
Tw =	1/4 in
W=	3 1/2 in
L =	4 3/4 in
L2=	1 3/4 in
OAL =	6 1/2 in
C=	2 in
Analysis Type	Full Pen w/ Fillet
Impact Factor	1.500
Occasional Load Factor	1.125
Heaviest Lug Load	150 #
(CHANNEL COVER)	
Weld Group Area (Aw)	2.651 in ²
Weld Group Inertia (Iw)	3.067 in ⁴
Shear Area (As)	0.375 in ²
Lug Inertia (I)	1.786 in ⁴

Allowable Loading

Shear Stress	17100 psi
0.4 * Yield * Occ Factor	
Bearing Stress	32062.5 psi
MIN(0.75*Yield*Occ Factor, 0.9*Yield)	
Combined	28215 psi
MIN(.66*Yield*Occ Factor, 0.75*Yield)	



Calculated Loads

Primary Shear in Welds	84.87 psi
Load*Impact/(Aw)	
Bending Shear Stress in Welds	146.71 psi
Load*Impact*(W+2*Tw)/2/(Iw)	
Total Combined Shear	231.57 psi
Shear in Lug Above Hole	400.00 psi
Load*Impact/(As)	
Pin Hole Bearing Stress	225.00 psi
Load*Impact/(T*C)	
Bending Stress @ Lug Base	1046.94 psi
Load*Impact*L/(W^2*T/6)	
Tensile @ Lug Base	128.57 psi
Load*Impact/(W*T)	
Total Combined Stress @ lug Base	1175.51 psi

SECTION 3

FORM U-1 MANUFACTURER'S DATA REPORT FOR PRESSURE VESSELS
As Required by the Provisions of the ASME Code Rules, Section VIII, Division 1

1. Manufactured and certified by ENERGY EXCHANGER COMPANY, 1844 NORTH GARNETT ROAD, TULSA, OK 74116
(Name and address of Manufacturer)

2. Manufactured for SOUTHERN COMPANY SERVICES, INC. 42 INVERNESS CENTER PARKWAY BIRMINGHAM, AL 35242
(Name and address of Purchaser)

3. Location of installation MISSISSIPPI POWER COMPANY DEKALB, MISSISSIPPI
(Name and address)

4. Type: VERT. HEAT EXCHANGER X-7926
(Horiz., vert., or sphere) (Tank, separator, jkt. vessel, heat exch., etc.) (Mfg's serial No.)

N/A X-7926-A REV. 4 6052 2012
(CRN) (Drawing No.) (Nat'l Bd. No.) (Year built)

5. ASME Code, Section VIII, Div. 1 2007 EDITION, '09 ADD. N/A N/A
[Edition and Addenda (date)] (Code Case No.) [Special Service per UG-120(d)]

Items 6-11 incl. to be completed for single wall vessels, jackets of jacketed vessels, shell of heat exchangers, or chamber of multichamber vessels.

6. Shell (a) No. of course(s): 1 (b) Overall length (ft & in.): 13'-6 3/8"

Course(s)			Material	Thickness		Long. Joint (Cat. A)			Circum. Joint (Cat. A, B, & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	16" O.D.	13'-6 3/8"	SA-106-B	1/2"	1/8"	S	NONE	0.85	7	NONE	0.85	N/A	N/A

7. Heads: (a) NA (b) NA
(Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.) (Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)														
(b)														

If removable, bolts used (describe other fastening) NA
(Mat'l Spec. No., Grade, size, No.)

8. Type of jacket NA Jacket closure NA
(Describe as ogee & weld, bar, etc.)

If bar, give dimensions NA If bolted, describe or sketch.

9. MAWP 455 15 psi at max. temp. 515 515 °F Min. design metal temp. 10 °F at 455 psi.
(internal) (external) (internal) (external)

10. Impact test NO, EXEMPT PER UG-20(f) at test temperature of NA °F.
[Indicate yes or no and the component(s) impact tested]

11. Hydro., pneu., or comb.-Test press. 600 P.S.I.G. (2) Proof test NA

Items 12 and 13 to be completed for tube sections.

12. Tubesheet: SA-266-4N 15 1/4" 3" 1/4" Welded
[Stationary (Mat'l Spec. No.)] [Dia., in. (subject to press.)] (Nom. thk., in.) (Corr. Allow., in.) [Attachment (welded or bolted)]

NA NA NA NA NA
[Floating (Mat'l Spec. No.)] [Dia., in.] (Nom. thk., in.) (Corr. Allow., in.) [Attachment]

13. Tubes: SA-179 3/4" 0.083" (MIN.) THK. 185 Straight
(Mat'l Spec. No., Grade or Type) (O.D., in.) (Nom. thk., in. or gauge) (Number) [Type (Straight or U)]

Items 14-18 incl. to be completed for inner chambers of jacketed vessels or channels of heat exchangers.

14. Shell (a) No. of course(s): 1/1 (b) Overall length (ft & in.): 1'-2 11/16"/1'-0 13/16"

Course(s)			Material	Thickness		Long. Joint (Cat. A)			Circum. Joint (Cat. A, B, & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	16" O.D.	1'-2 11/16"	SA-333-6	3/8"	1/8"	S	NONE	0.85	1 (f)	SPOT	0.85	1150	1.00
1	16" O.D.	1'-0 13/16"	SA-333-6	3/8"	1/8"	S	NONE	0.85	1 (f)	SPOT	0.85	1150	1.00

15. Heads: (a) SA-266-4N H.T. @ 1150 deg F. for 1 HR. (b) SA-266-4N H.T. @ 1150 deg F. for 1 HR.
(Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.) (Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	END	1 5/8"	1/8"						16 7/8"					
(b)	END	1 5/8"	1/8"						16 7/8"					

If removable, bolts used (describe other fastening) (a) SA-193-B7, 1" DIA., 16 (b) SA-193-B7, 1" DIA., 16
(Mat'l Spec. No., Grade, size, No.)

FORM U-1 (Back)

16. MAWP 320 NA psi at max. temp. 385 NA °F Min. design metal temp. 10 °F at 320 psi.
(Internal) (external) (Internal) (external)
17. Impact test NO, EXEMPT PER UG-20(f) at test temperature of NA °F.
(Indicate yes or no and the component(s) impact tested)
18. Hydro., pneu., or comb. Test press. 425 P.S.I.G. (2) Proof test NA
19. Nozzles, inspection, and safety valve openings:

Purpose (Inlet, Outlet, Drain, etc.)	No.	Diameter or Size	Flange Type	Material		Nozzle Thickness		Reinforcement Material	How Attached		Location (Insp. Open.)
				Nozzle	Flange	Nom.	Corr.		Nozzle	Flange	
INLET	1	4"-300#	RFWN	SA-106-B	SA-105	0.4380"	1/8"	WELD	WELDED	WELDED	SHELL
OUTLET	1	2"-300#	RFLWN	SA-105	—	0.6600"	1/8"	WELD	WELDED	—	SHELL
BAL. LINE	1	3/4"-300#	RFLWN	SA-105	—	0.5700"	1/8"	WELD	WELDED	—	SHELL
VENT/DRAIN	1/1	3/4"-300#	RFLWN	SA-105	—	0.5700"	1/8"	WELD	WELDED	—	TUBESHEETS
INLET	1	3"-300#	RFWN	SA-333-B	SA-105N	0.4380"	1/8"	WELD	WELDED	WELDED	BOTTOM CHAN.
OUTLET	1	6"-300#	RFWN	SA-333-B	SA-105N	0.5620"	1/8"	WELD	WELDED	WELDED	TOP CHANNEL
VENT/DRAIN	1/1	2"-300#	RFLWN	SA-105N	—	0.6600"	1/8"	WELD	WELDED	—	CHAN. COVERS

20. Supports: Skirt NO Lugs — Legs — Others 2-SADDLES Attached WELDED TO SHELL
(Yes or no) (No.) (No.) (Describe) (Where and how)
21. Manufacturer's Partial Data Reports properly identified and signed by Commissioned Inspectors have been furnished for the following items of the report:
 (List the name of part, item number, mfg's. name and identifying number)

22. Remarks: SERVICE: WASTEWATER AMMONIA PURIFIER REBOILER, P.O. NO: 17496-0001; ITEM NO.: HX0052

SIZE: 15-168, TYPE: NEN, TUBES DESIGNED W/ 0" C.A., SAFETY RELIEF DEVICES BY OTHERS

LINES 14-18 INCLUDES (2) 21 1/8" O.D. X 15 1/4" I.D. X 3 1/4" THK., SA-266-4N, RING

(1) Cylinder to tubesheet circumferential seams are Type 7, with .85 Efficiency; (2) Field Test: Shell Side=595 PSIG; Tube Side= 420 PSIG

CERTIFICATE OF SHOP COMPLIANCE

We certify that the statements in this report are correct and that all details of design, material, construction, and workmanship of this vessel conform to the ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1.

U Certificate Authorization No. 12370 Expires SEPTEMBER 27, 20 13

Date 7-23-12 Name ENERGY EXCHANGER COMPANY Signed [Signature]
(Manufacturer) (Representative)

CERTIFICATE OF SHOP INSPECTION

I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of OKLAHOMA and employed by ONEBEACON AMERICA INSURANCE COMPANY of LYNN, MA have inspected the pressure vessel described in this Manufacturer's Data Report on 7.23.2012, and

state that, to the best of my knowledge and belief, the Manufacturer has constructed this pressure vessel in accordance with ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date 7.23.12 Signed [Signature] Commissions N.B.# 12064 A OKLA.# 7
(Authorized Inspector) (Natl Board incl. endorsements, State, Province, and No.)

CERTIFICATE OF FIELD ASSEMBLY COMPLIANCE

We certify that the statements on this report are correct and that field assembly construction of all parts of this vessel conforms with the requirements of ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. U Certificate of Authorization No.

U Certificate of Authorization No. _____ Expires _____, 20 _____

Date _____ Name _____ Signed _____
(Assembler) (Representative)

CERTIFICATE OF FIELD ASSEMBLY INSPECTION

I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of _____ and employed by _____, have compared the statements in this Manufacturer's Data Report with the described pressure vessel and state that parts referred to as data items _____, not included in the certificate of shop inspection, have been

inspected by me and to the best of my knowledge and belief, the Manufacturer has constructed and assembled this pressure vessel in accordance with ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. The described vessel was inspected and _____ psi.

By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date _____ Signed _____ Commissions _____
(Authorized Inspector) (Natl Board incl. endorsements, State, Province, and No.)

NR 6052



CERTIFIED BY
Energy Exchanger Company
TULSA, OKLAHOMA



RT-3-T
HT-T

MAWP SHELL	455	P.S.I. @	515	OF	595	TEST P.S.I.G.
MAWP	15	P.S.I. @	515	OF		
MAWP TUBE	320	P.S.I. @	385	OF	420	TEST P.S.I.G.
MAWP	---	P.S.I. @	---	OF		
MIN. DESIGN METAL TEMP.		SHELL	10	OF @	455	P.S.I.G.
		TUBE	10	OF @	320	P.S.I.G.
SERIAL NO.	X-7926	YEAR BUILT			2012	

P.O. NO.

17496-0001 AREA: 210

SERVICE

WASTEWATER AMMONIA PURIFIER REBOILER

ITEM

HX0052

SIZE

15-168

TYPE

NEN

SHOP TEST: SHELLSIDE: 600 PSIG TUBESIDE: 425 PSIG

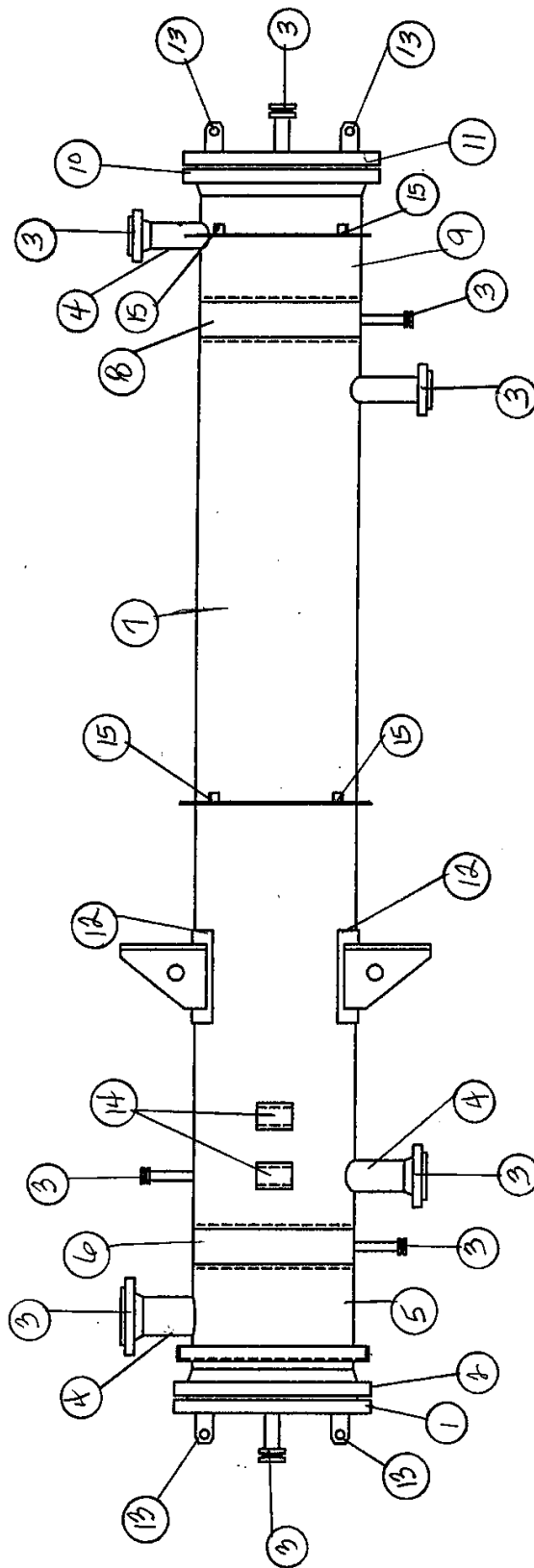
CAUTION

PRESSURES AND TEMPERATURES SHOWN ON THE NAMEPLATE OF THIS HEAT EXCHANGER RELATE ONLY TO PRESSURE RETAINING CAPABILITY, CERTAIN COMPONENTS HAVE BEEN DESIGNED FOR SPECIFIC OPERATING CONDITIONS. DEVIATION FROM THESE CONDITIONS OR A CHANGE IN SERVICE REQUIRE A MECHANICAL DESIGN CHECK BE PERFORMED TO AVOID POSSIBLE DAMAGE TO THE HEAT EXCHANGER OR DANGER TO PERSONNEL.

WARNING

FIXED TUBESHEET HEAT EXCHANGER. INTRODUCE SHELLSIDE FIRST BEFORE INTRODUCING TUBESIDE. STEAMOUT TUBESIDE ALONE IS NOT ALLOWED FOR THIS UNIT. STEAMOUT SHELLSIDE FIRST, WHILE SHELLSIDE IS HOT STEAMOUT TUBESIDE.

SECTION 4



- ⑬ Tubes
- ⑰ Studs
- ⑱ Nuts

ENERGY EXCHANGER COMPANY MATERIAL HEAT NUMBER INDEX

Project; Kemper County 1 - IGCC

CUSTOMER: Southern Co. (for) Mississippi Power Company
 DESTINATION: DeKalb, Mississippi
 P.O. No: 17496-0001
 Req. No: MC202
 ITEM NO: HX0052
 SERVICE: Wastewater Ammonia Purifier Reboiler
 DESCRIPTION: 1-15" x 168" NEN Exchanger
 PAGE: 1 of 3
 DATE: June 4, 2012
 JOB NO: X-7926

PART NO.	HEAT NO.	SLAB NO.	STEEL PRODUCER	GRADE	DESCRIPTION	PCS	SIZE
1	227N568		ArcelorMittal	SA-266-4N	(#1 Channel Cover)	1	21 1/8" OD x 1 5/8" thk.
2	103467		Nucor	SA-266-4N	(#2 Channel Flange)	1	21 1/8" OD x 15 1/4" ID x 3 1/4" thk.
3	C845AN		FVC	SA-105-N	(Nozzle Flange V-1)	1	2" 300# RF LWN x 9" lg.
3	3019785		Galperti	SA-105-N	(Nozzle Flange V-1)	1	2" 300# RF Blind
3	30642/02		Galperti	SA-105-N	(Nozzle Flange T-2)	1	6" 300# RF WN Sch 120 Bore
3	A606AF		FVC	SA-105	(Nozzle Flange V-2)	1	3/4" 300# RF LWN x 9" lg.
3	79209		Galperti	SA-105-N	(Nozzle Flange V-2)	1	3/4" 300# RF Blind
3	32534R/02		Galperti	SA-105-N	(Nozzle Flange S-1)	1	4" 300# RF WN Sch 120 Bore
3	A606AF		FVC	SA-105	(Nozzle Flange S-3)	1	3/4" 300# RF LWN x 9" lg.
3	79209		Galperti	SA-105-N	(Nozzle Flange S-3)	1	3/4" 300# RF Blind
3	B834		FCI	SA-105-N	(Nozzle Flange S-2)	1	2" 300# RF LWN x 10" lg.
3	A606AF		FVC	SA-105	(Nozzle Flange D-2)	1	3/4" 300# RF LWN x 9" lg.
3	79209		Galperti	SA-105-N	(Nozzle Flange D-2)	1	3/4" 300# RF Blind

I hereby certify that the above information is correct to the best of my knowledge and belief.

ENERGY EXCHANGER COMPANY

By:



ENERGY EXCHANGER COMPANY MATERIAL HEAT NUMBER INDEX

Project: Kemper County 1 - IGCC

CUSTOMER: Southern Co. (for) Mississippi Power Company
 DESTINATION: DeKalb, Mississippi
 P.O. No: 17496-0001
 Req. No: MC202
 ITEM NO: HX0052
 SERVICE: Wastewater Ammonia Purifier Reboiler
 DESCRIPTION: 1-15" x 168" NEN Exchanger
 PAGE: 2 of 3
 DATE: June 4, 2012
 JOB NO: X-7926

PART NO	HEAT NO	SLAB NO	STEEL PRODUCER	GRADE	DESCRIPTION	PCS	SIZE
3	3022384		Galperti	SA-105-N	(Nozzle Flange T-1)	1	3" 300# RF WN Sch 160 Bore
3	C845AN		FVC	SA-105-N	(Nozzle Flange D-1)	1	2" 300# RF LWN x 9" lg.
3	3019785		Galperti	SA-105-N	(Nozzle Flange D-1)	1	2" 300# RF Blind
4	U82073		U.S.S.	SA-333-6	(Nozzle Pipe T-2)	1	6" Sch 120 x 6 3/4" lg.
4	574171		Ipsco	SA-106-B	(Nozzle Pipe S-1)	1	4" Sch 120 x 7" lg.
4	320369		V&M	SA-333-6	(Nozzle Pipe T-1)	1	3" Sch 160 x 7" lg.
5	A03724		U.S.S.	SA-333-6	(Top Channel Cylinder)	1	16" OD x 3/8" thk. x 1'-2 11/16" lg.
6	227N568		ArcelorMittal	SA-266-4N	(#4 Tubesheet)	1	16" OD x 3" thk.
7	WA1176		U.S.S.	SA-106-B	(Shell Cylinder)	1	16" OD x 1/2" thk. x 13'-6 3/8" lg.
8	227N568		ArcelorMittal	SA-266-4N	(#9 Tubesheet)	1	16" OD x 3" thk.
9	A03724		U.S.S.	SA-333-6	(Bottom Channel Cylinder)	1	16" OD x 3/8" thk. x 1'-0 13/16" lg.
10	100646		Nucor	SA-266-4N	(#2A Channel Flange)	1	21 1/8" OD x 15 1/4" ID x 3 1/4" thk.
11	227N568		ArcelorMittal	SA-266-4N	(#1A Channel Cover)	1	21 1/8" OD x 1 5/8" thk.

I hereby certify that the above information is correct to the best of my knowledge and belief.

ENERGY EXCHANGER COMPANY

By: *Spencer Patterson*

ENERGY EXCHANGER COMPANY

MATERIAL HEAT NUMBER INDEX

Project; Kemper County 1 - IGCC

CUSTOMER: Southern Co. (for) Mississippi Power Company ITEM NO: HX0052 PAGE: 3 of 3
 DESTINATION: DeKalb, Mississippi SERVICE: Wastewater Ammonia Purifier Reboiler DATE: June 4, 2012
 P.O. No: 17496-0001 DESCRIPTION: 1-15" x 168" NEN Exchanger JOB NO: X-7926
 Req. No: MC202

PART NO.	HEAT NO.	SLAB NO.	STEEL PRODUCER	GRADE	DESCRIPTION	PCS.	SIZE
12	0505315	03	Nucor	SA-516-70	(Support Pads)	2	11" x 3/8" thk. x 10 1/4" lg.
13	B1T7537	03	Nucor	SA-516-70	(Lifting Lugs)	4	3 1/2" x 1/2" thk. x 6 1/2" lg.
14	W0H543	0024	SSAB	SA-516-70	(Name Plate Bracket)	2	7 1/2" x 3/16" thk. x 22" lg.
15	3018892		CMC Steel	SA-36	(Insulation Clips)	8	1 1/2" x 1/4" Thk. x 3" lg.
16	181073		Benteler	SA-179	(Tubes)	185	3/4"OD x .083" (min) x 14'-0" lg.
17	11881890		Vulcan Thrd. Prod.	SA-193-B7	(Studs)	32	1" (8 Thd) x 6" lg.
18	SN392		Unytite	SA-194-2H	(Nuts)	64	1" (8 Thd)

I hereby certify that the above information is correct to the best of my knowledge and belief.

ENERGY EXCHANGER COMPANY

By:





OKLAHOMA FORGE, INC.

Certification of Analysis and Tests

INVOICE NUMBER
111-66965

Date: 6/20/11

Toll Free (866)909-5553
Local(918)446-5553 FAX (918)446-4150

e-Mail ~ sales@oklahomaforge.com
Web Site ~ www.oklahomaforge.com

P.O. Box 701500
Tulsa, Oklahoma 74170-1500

Sold To		Your Purchase Order		Sales Order	
ENERGY EXCHANGER COMPANY 1844 N. GARNETT ROAD TULSA, OK 74116 USA		Number: 1199 By: JIM ELDER Date: 5/17/2011		Order: 111-51903 DELIVERY 6/20/2011	

Your Part Number		Shop Order Number		Shape		Dimensions	
Section		Heat Number		Material Specification		Chemical Analysis	
Physical Properties		Grain Size		Type		Temp	

1	X-7926-4/9	111-51903-01	DISC	16.000"OD x 3.000"TH	2											
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED												
		C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
		.18	1.16	.014	.008	.23	.10	.09	.27	.027	.017	<.001	.021		.42	SN-.011; B-.0003
		Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%in 2In	Red of Area%	Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
		49,600	75,500	34	68	146				FINE	CVN	-50F	114	127	137	80

2	X-7926-1/1A	111-51903-02	DISC	21.125"OD x 1.625"TH	2											
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED												
		C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
		.18	1.16	.014	.008	.23	.10	.09	.27	.027	.017	<.001	.021		.42	SN-.011; B-.0003
		Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%in 2In	Red of Area%	Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
		49,600	75,500	34	68	146				FINE	CVN	-50F	114	127	137	80

3	X-7930-4	111-51903-03	DISC	22.000"OD x 3.000"TH	2											
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED												
		C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
		.18	1.16	.014	.008	.23	.10	.09	.27	.027	.017	<.001	.021		.42	SN-.011; B-.0003
		Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%in 2In	Red of Area%	Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
		49,600	75,500	34	68	146				FINE	CVN	-50F	114	127	137	80

4	X-7930-1	111-51903-04	DISC	27.875"OD x 3.250"TH	2											
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED												
		C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
		.18	1.16	.014	.008	.23	.10	.09	.27	.027	.017	<.001	.021		.42	SN-.011; B-.0003
		Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%in 2In	Red of Area%	Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
		49,600	75,500	34	68	146				FINE	CVN	-50F	114	127	137	80

8	X-7930-2	111-51903-08	RING	27.875"OD x 20.500"ID x 4.188"TH	2											
		NUCOR	110786	SA266-10 GR4 NORMALIZED												
		C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
		.17	1.17	.010	.002	.20	.10	.10	.19	.03	.016	.003	.024		.41	SN-.008; TI-.002
		Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%in 2In	Red of Area%	Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
		48,800	73,500	35	74	143				FINE	CVN	-50F	85	85	104	67

9	X-7929-2/2A	111-51903-09	RING	28.188"OD x 23.000"ID x 3.063"TH	2											
		NUCOR	110786	SA266-10 GR4 NORMALIZED												
		C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
		.17	1.17	.010	.002	.20	.10	.10	.19	.03	.016	.003	.024		.41	SN-.008; TI-.002
		Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%in 2In	Red of Area%	Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
		48,800	73,500	35	74	143				FINE	CVN	-50F	85	85	104	67

Notes:

VERIFIED BY

Page 1

JUN 27 2011

I CERTIFY THAT THERE IS A TRUE COPY OF THE ORIGINAL TEST SHEET NOW ON FILE AT THE OFFICE OF OKLAHOMA FORGE, INC. AND THAT THIS MATERIAL WAS MANUFACTURED IN THE UNITED STATES OF AMERICA.

Anneal by heating to 1450-1550 degrees F for one hour/inch of thinnest section. Slow cool furnace to 700 degrees F then air cool by opening door.
Normalize by heating to 1650 degrees F. Hold at 1650 degrees F for one hour/inch of thinnest section and air cool.
Solution Anneal by heating to 2000-2050 degrees F and Water Quenching.

R. Patterson



OKLAHOMA FORGE, INC.

Certification of Analysis and Tests

111-66822

Date: 6/10/11

Toll Free (866)909-5553
Local(918)446-5553 FAX (918)446-4150

e-Mail ~ sales@oklahomaforge.com
Web Site ~ www.oklahomaforge.com

P.O. Box 701500
Tulsa, Oklahoma 74170-1500

ENERGY EXCHANGER COMPANY
1844 N. GARNETT ROAD
TULSA, OK 74116 USA

1199
JIM ELDER
5/17/2011

111-51903
DELIVERY
6/10/2011

7 A X-7926-2				111-51903-07A				RING		21.125"OD x 15.250"ID x 3.250"TH									
				NUCOR				103467		SA266-10 GR4 NORMALIZED									
C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other					
.17	1.19	.013	.001	.18	.07	.14	.16	.04	.029	.001	.029		.430	SN-.007;TI-.001					
Yield Lb/SqIn		Tensile Strength Lb/SqIn		Elong%in 2In		Red of Area%		Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3	
46,000		71,500		36		75		146				FINE	CVN	-50F	103	92	88	130	

7 B X-7926- 2A				111-51903-07B			RING		21.125"OD x 15.250"ID x 3.250"TH									
				NUCOR			100646		SA266-10 GR4 NORMALIZED									
C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other				
.17	1.18	.015	.0036	.23	.06	.11	.23	.02	.020	.002	.028		.42	SN-.009;TI-.001				
Yield Lb/SqIn		Tensile Strength Lb/SqIn		Elong%in 2In		Red of Area%		Brinell	Rckwl	C	Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
48,800		73,500		36		75		146				FINE	CVN	-50F	99	78	118	101

VERIFIED BY

JUN 13 2011

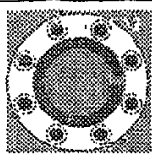
EEC QC DEPT

Notes:

L. Jatterson

Page 1

I CERTIFY THAT THERE IS A TRUE COPY OF THE ORIGINAL TEST SHEET NOW ON FILE AT THE OFFICE OF OKLAHOMA FORGE, INC. AND THAT THIS MATERIAL WAS MANUFACTURED IN THE UNITED STATES OF AMERICA.	
Anneal by heating to 1450-1550 degrees F for one hour/inch of thinnest section. Slow cool furnace to 700 degrees F then air cool by opening door. Normalize by heating to 1650 degrees F, Hold at 1650 degrees F for one hour/inch of thinnest section and air cool. Solution Anneal by heating to 2000-2050 degrees F and Water Quenching.	

**GALPERCI, INC.**

180 Southbelt Industrial Drive
Houston, Texas 77047
PH. 713-433-0700 - FAX 713-433-5580
Email: galperci@galperci-am.com

PROGRESSIVE SUPPLY

P. O. BOX 470748
TULSA OK 74145

This MTR applies to your

PO# 1221

TEST/CERTIFICATE

EN 10204

INTERNAL JOB

3.1B**091351**

M.T. CERTIFICATE N.

013947/B

DATE

06/12/2009

YOUR P.O.

098331

DATE

05/15/2009

ITEM	DESCRIPTION	QUANTITY	BP	MATERIAL	HEAT N°	#
019/019	6" 300 WN RF 120 A105 B1 F	10	B	ASTM A/SA105 03c	30642/02	A
022/022	12" 600 WN RF XS A105 B1 F	2	B	ASTM A/SA105 03c	10354W/02	B
036/036	24" 150 SO RF A105 B1 F	2	B	ASTM A/SA105 03c	10092W/02	C

Noz. T2

#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CB %	B %	FL %
A	0.180	1.130	0.250	0.012	0.010	0.070	0.080	0.010	0.230	0.0001		0.025	0.001	0.002	0.012	0.001		0.40
A C	0.170	1.150	0.230	0.015	0.013	0.060	0.070	0.020	0.220	0.0001		0.036	0.001	0.002	0.012	0.001		0.39
B	0.180	1.160	0.190	0.010	0.008	0.100	0.120	0.020	0.190	0.0140			0.0004	0.001	0.006	0.0004	0.0001	0.42
B C	0.170	1.140	0.160	0.015	0.003	0.120	0.090	0.030	0.160	0.0090			0.0004	0.001	0.006	0.0004	0.0001	0.40
C	0.160	1.170	0.270	0.020	0.001	0.110	0.190	0.030	0.170	0.0200	0.012	0.030	0.0010	0.003	0.010	0.0010		0.41
C C	0.150	1.190	0.290	0.018	0.001	0.120	0.180	0.020	0.180	0.0150	0.012	0.050	0.0010	0.003	0.010	0.0010		0.40

NOTES
FL=Carbon Equivalent Long Formula

C = CHECK ANALYSIS

#		MECHANICAL PROPERTIES											
		TENSILE								CHARPY TEST			
		TYPE	Ø DIAMETER mm.	AREA mmq.	LENGHT mm.	TEMP. F	TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	O	12.50	122.70	50.00	68	78,744	47,971	27.5	66.5				160
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	O	12.50	122.70	50.00	68	78,510	47,333	29.6	69.2				160
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
C	O	12.50	122.70	50.00	68	78,184	47,844	30.0	68.5				153

NOTES

O = OBTAINED
S = STANDARD

LINE 019 COUNTRY OF ORIGIN ITALY

#	HEAT TREATMENTS	GENERAL NOTES
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	ROUGHNESS
A		F STOCK FINISH 125-250 microinch
B	Normalized at 900 dgr.C for 3 h.-Cooling from 900 dgr.C in still air	JUN 16 2011
B		
C	Normalized at 900 dgr.C for 2 h.-Cooling from 900 dgr.C in still air	EEG QC DEPT
C		STANDARD B1 = ASME B16.5

NOTES

Material I.A.W. Nace MR-0175-2002

TRADE MARK / LOGO

Stamped on product



INT. INSPECT DATE

06/12/2009

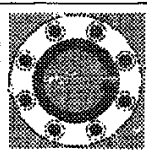
INT. INSPECTOR

D. RUZICKA

INSPECTION AUTHORITY

*Dion Ruzicka***GALPERCI, INC.**

An ISO 9001/2000 certified company.
Certification issued by:
Lloyds register quality assurance
Certificate No. UQA 0110391



Galper, Inc.
160 Southbelt Industrial Drive
Houston, Texas 77047
Tel: 713-433-0700 - FAX 713-433-0880
Email: galpermail@galper-inc.com

PROGRESSIVE SUPPLY

P. O. BOX 470748
TULSA OK 74145

This MTR applies to your

PO# 1221

Part #3

TEST/CERTIFICATE EN 10204 INTERNAL JOB
3.1B **110100**

M.T. CERTIFICATE N. 001307/B DATE 02/14/2011 YOUR P.O. 11-9480 DATE 01/10/2011
X-7426-71

ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#
001/001	12" 150 WN RF STD A105 B1 F	15	B	ASTM A/SA105 03c	3018256	A
005/005	4" 150 WN RF 120 A105 B1 F	25	B	ASTM A/SA105 03c	258121	B
007/007	4" 300 WN RF 120 A105 B1 F	40	A	ASTM A/SA105 03c	32534R/02	C
008/008	4" 300 WN RF 160 A105 B1 F	50	B	ASTM A/SA105 03c	32534R/02	D

Noz. \$1

#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	BN %	AL %	NB %	TI %	N %	CB %	B %	FL %
A	0.190	1.070	0.240	0.010	0.021	0.100	0.090	0.030	0.260	0.003			0.001			0.001		0.41
A C	0.180	1.050	0.260	0.013	0.018	0.110	0.100	0.040	0.250	0.003			0.001			0.001		0.40
B	0.200	0.900	0.210	0.012	0.019	0.140	0.100	0.029	0.280	0.001			0.002			0.002		0.40
B C	0.190	0.920	0.230	0.015	0.022	0.130	0.090	0.039	0.270	0.001			0.002			0.002		0.40
C	0.170	1.160	0.260	0.013	0.009	0.120	0.060	0.010	0.170	0.012		0.022	0.001	0.012		0.001	0.0001	0.40
C C	0.160	1.180	0.280	0.010	0.006	0.130	0.050	0.020	0.160	0.007		0.032	0.001	0.012		0.001	0.0001	0.40

NOTES
FL = Carbon Equivalent Long Formula C = CHECK ANALYSIS

#	TYPE	Ø DIAMETER mm.	AREA mm²	LENGHT mm.	TEMP. F	TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES	HB HARDNESS (AVERAGE)
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	O	12.50	122.70	50.00	68	76,615	49,169	28.7	69.6				154
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	O	12.50	122.70	50.00	68	77,631	46,875	28.0	68.0				152
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
C	O	12.50	122.70	50.00	68	77,810	47,793	30.4	68.8				154
D	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
D	O	12.50	122.70	50.00	68	78,206	46,505	29.6	67.6				150

NOTES
O = OBTAINED
S = STANDARD

#	HEAT TREATMENTS	GENERAL NOTES
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	ROUGHNESS F - STOCK FINISH 125-250 microinch
B	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air	VERIFIED BY
C	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	STANDARD B1 - ASME B16.34
D	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	DATE JUN 16 2011

NOTES
Material I.A.W. Nace MR-0175-2003 COUNTRY OF ORIGIN USA ✓

EEC QC DEPT

RADE MARK / LOGO INT. INSPECT DATE INT. INSPECTOR
Stamped on product 02/14/2011 D.RUZICKA

INSPECTION AUTHORITY
Dan Ruzicka

GALPER, INC
An ISO 9001/2008 certified company.
Certification issued by:
Lloyds register quality assurance
Certificate No. UQA 0110391

X-7926-72, 74, 15

Part #3



A DIVISION OF AMERI-FORGE GROUP INC.

Forged Vessel Connections
2525 De Soto St. (77091)
P.O. Box 38421 (77238-8421)
Houston, Texas

MATERIAL TEST REPORT

MTR # A183731

http://www.fvc.cc

QA@fvc.cc

TEL: (713) 688-9705 FAX: (713) 688-7954

Noz. V2, D2, S3

S.O. No: 8-02434

Date: 10/21/2008

REV #

Purchaser: Mattsco Supply Co.

Distributor:

P.O. No.: 853-S-08

Distributor's Order No:

Page 1

ITEM NO	QTY	DESCRIPTION	SPEC	HEAT #	HEAT TREAT
6	1	LWN, SA105, 300#.75 x 9", RF&D J55433-0 J55433-1	SA105 2007 Edition, 2008a Adden	A606AF	As-Forged

$$(C + (Mn / 5)) + ((Cr + Mo + V) / 5) + ((Ni + Cu) / 15) = .442$$

CHEMICAL ANALYSIS AND MECHANICAL PROPERTIES

Heat #: A606AF

11789240

Chemical values below represent percentage of weight

C (Carbon)	Mn (Manganese)	P (Phosphorus)	S (Sulphur)	Si (Silicon)	Cr (Chromium)	Ni (Nickel)	Mo (Molybdenum)
.19 ✓	1.05 *CAS # 7349-98-5	.018 ✓ *CAS # 7723-14-0	.022 ✓	.26 ✓	.19 *CAS # 7440-47-3	.11 *CAS # 7440-02-0	.05 ✓ *CAS # 1313-27-5
Cu (Copper)	Al (Aluminum)	V (Vanadium)	As (Arsenic)	Sb (Antimony)	Sn (Tin)	Co (Cobalt)	N (Nitrogen)
.32 ✓ *CAS # 7440-50-8	.006	.002 ✓ *CAS # 7440-62-2			.013		
Ti (Titanium)	Ta (Tantalum)	Cb (Columbium)	O (Oxygen)	H (Hydrogen)			
		.013 ✓					

TENSILE (PSI)	YIELD (PSI)	ELONG (IN 2")	R.A. (%)	CHARPYS (FT - LBS)	TEMP ° F
82988	58388	28.0	66.0		
SHEAR %		LATERAL EXPANSION (MLS)		BHN	
				< 187	

COMMENTS

Test Bar: 187-187 BHN ✓

FVC STANDARD MANUFACTURING PRACTICES:

- 1) FLANGE DIMENSIONS & BOLTING COMPLY - ASME B16.5
- 2) SELF-REINFORCING NOZZLES COMPLY WITH SPECIFICATION OF SECT. VIII DIV. 1 OR DIV.2 PER CUSTOMER DESIGN/FVC DESIGN.
- 3) MATERIAL SPECIFICATION COMPLIES WITH ASME SECT. II, PART A.
- 4) ALL FVC MATERIAL MANUFACTURED PER FINE GRAIN PRACTICE.
- 5) ALL FVC MATERIAL CERTIFIED PER EN 10204, 3.1.B UNLESS OTHERWISE SPECIFIED
- 6) CERTIFICATE OF INSURANCE AVAILABLE UPON REQUEST.

CHECKED APR 15 2009

EPCRA Supplier Notification:

* This product may contain one or more toxic chemicals subject to the reporting requirements of Section 313 of the Emergency Planning and Community Right-to-Know Act (Title III of the Superfund Amendments and Reauthorization Act of 1986) and 40 C.F.R. Part 372. Potentially reportable chemicals are indicated with an asterisk (*) under the chemical's name and percent-by-weight value, followed by a Chemical Abstract Service (CAS) registry number for that chemical. It is your responsibility alone to determine whether your facility is required to submit a Toxic Release Inventory Report under EPCRA Section 313.

Quality Assurance

04/08/2009 From: MATTSO SUPPLY CO.

To: ENERGY EXCHANGERS

Our# :31620

Line# :1

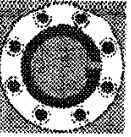
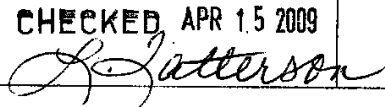
QTY: :6

To: :ENERGY EXCHANGERS

CPO# :4326

X-7986-73D, 74D, 75D

Part #3

 GALPER, INC. 160 Southbelt Industrial Drive Houston, Texas 77047 PH: 713-433-0700 - FAX: 713-433-6660 Email: galper@galper-inc.com		MATTSCO SUPPLY P.O. BOX 2925 TULSA OK 74101															
TEST/CERTIFICATE 3.1B		EN 10204 083305															
M.T. CERTIFICATE N. 011219/B		DATE 10/21/2008															
YOUR P.O. 946-S-09A		DATE 10/13/2008															
ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°												
024/024	3" 600 WN RF 160 A105 B1 F	15	A	ASTM A/SA105 03c	252597	A											
024/024	3" 600 WN RF 160 A105 B1 F	5	B	ASTM A/SA105 03c	3832/02	B											
026/026	6" 150 BL RF A105 B1 F	20	B	ASTM A/SA105 03c	30469/02	C											
027/027	0.75" 300 BL RF A105 B1 F	25	B	ASTM A/SA105 03c	79209	D											
CHEMICAL ANALYSIS																	
#	C %	MIN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CB %	FL %
A	0.200	0.910	0.200	0.010	0.023	0.140	0.090	0.028	0.320	0.0001			0.002			0.002	0.41
A C	0.190	0.930	0.180	0.013	0.020	0.150	0.100	0.018	0.330	0.0001			0.002			0.002	0.40
B	0.175	1.140	0.190	0.009	0.010	0.095	0.210	0.036	0.220	0.0010	0.014	0.020	0.001		0.009	0.001	0.42
B C	0.165	1.160	0.170	0.012	0.013	0.105	0.220	0.046	0.230	0.0010	0.014	0.040	0.001		0.009	0.001	0.41
C	0.180	1.190	0.250	0.011	0.009	0.100	0.060	0.010	0.180	0.0230	0.009	0.025	0.002	0.021		0.002	0.42
C C	0.170	1.210	0.230	0.014	0.012	0.110	0.070	0.020	0.190	0.0180	0.009	0.039	0.002	0.021		0.002	0.41
D	0.185	1.120	0.220	0.013	0.013	0.090	0.060	0.010	0.200	0.0240						0.001	0.41
D C	0.195	1.140	0.240	0.010	0.010	0.100	0.070	0.020	0.190	0.0340						0.001	0.43
NOTES FL-Carbon Equivalent Long Formula																	
C - CHECK ANALYSIS																	
MECHANICAL PROPERTIES																	
#	TENSILE								CHARPY TEST								
	TYPE	Ø DIAMETER mm.	AREA mm²	LENGTH mm.	TEMP. F	TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES		HB HARDNESS (AVERAGE)			
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187			
A	O	12.50	122.70	50.00	68	76,790	50,217	27.6	67.6					151			
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187			
B	O	12.50	122.70	50.00	68	77,987	50,277	31.0	68.5					157			
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187			
C	O	12.50	122.70	50.00	68	76,572	49,075	29.0	66.0					157			
D	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187			
D	O	12.50	122.70	50.00	68	77,220	50,301	29.0	69.8					164			
NOTES O - OBTAINED S - STANDARD																	
#	HEAT TREATMENTS										GENERAL NOTES						
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air										ROUGHNESS F - STOCK FINISH 125-250 microinch STANDARD B1 - ASME B16.5						
B	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air																
C	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air																
D	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air																
NOTES Material I.A.W. Nace MR-0175-2002																	
TRADE MARK		INT. INSPECT DATE		INT. INSPECTOR													
		10/21/2008		D.RUZICKA													
INSPECTION AUTHORITY						CHECKED APR 15 2009 											
																	



Material Test Report

Heat Code: C845AN

ISO 9001:2008 Certified

This MTR applies to your

PO#

An Ameri-Forge Group Company

2525 De Soto Houston, TX 77091

Sales: (713) 868-4421 Fax: (713) 455-8366

Progressive Supply	PO: 11-9503	Sales Order: 1-00542	Line: 4
Attn: Accounts Payable	Item Code: LWN020300090A002A1	Qty Shipped: 10	
P.O. Box 470748	Item Desc: LWN, SA105N, 300#, 2"x9", RF		
Tulsa, OK 74147	Supplier: STEEL DYNAMICS, INC	Supplier Heat: A103136	
Spec: ASME SA105N: 2007 Edition, 2009 Addenda			

Element	(%wt)	Ladle	Product	EPCRA	CAS#	Element	(%wt)	Ladle	Product	EPCRA	CAS#
C	Carbon	0.19 ✓				Sn	tin	0.009			
Mn	Manganese	1.15 ✓		✓	7439-96-5	B	Boron	0.000			
P	Phosphorous	0.006 ✓				Ti	Titanium	0.001		✓	7440-02-0
S	Sulphur	0.004 ✓				Al	Aluminum	0.022			
Si	Silicon	0.27 ✓				Ca	Calcium	0.0017			
Cu	Copper	0.16 ✓		✓	7440-50-8	N	Nitrogen	0.0079			
Ni	Nickel	0.06 ✓		✓	7440-02-0	O					
Cr	Chromium	0.09 ✓		✓	7440-47-3	As	Arsenic	0.007		✓	7440-38-2
Mo	Molybdenum	0.02 ✓				Sb	Antimony	0.006		✓	7440-36-0
V	Vanadium	0.021 ✓				CE	C+Mn/5+(Cr+Mo+V)/5	0.423			
Cb	Columbium	0.002 ✓				H	% wt	2.0			

Mechanical Testing		Other	
HBW	141 ✓	Country of Origin	USA ✓
Elg (%)	35.40 ✓	Manufactured In	USA ✓
RA (%)	71.20 ✓	Production Method	EEF
Tensile (ksi)	74.686 ✓	Reduction Ratio (Mill) x:1	3.5/1
Yield (ksi)	47.669 ✓	Gr.Sz.	5
		Calcium Treated	YES
Heat Treatment			
Heat Treat Normalized ✓			
VERIFIED BY JUN 16 2011 EEC QC DEPT <i>S. Jatterson</i>			

PO# 100483
NACE MR0175 & MR0103 (hardness only), CE=C+(Mn/6)+((Cr+Mo+V)/5)+((Ni+Cu)/15), Elongation in 2" round
FVC STANDARD MANUFACTURING PRACTICES:
1) FLANGE DIMENSIONS & BOLTING COMPLY - ASME B16.5
2) SELF-REINFORCING NOZZLES COMPLY WITH SPECIFICATION OF SECTION VIII DIV. 1 OR DIV. 2 PER CUSTOMER DESIGN/FVC DESIGN
3) MATERIAL SPECIFICATION COMPLIES WITH ASME SECT. II PART A
4) ALL FVC MATERIAL MANUFACTURED PER FINE GRAIN PRACTICE
5) ALL FVC MATERIAL CERTIFIED PER EN 10204, 3.1.B UNLESS OTHERWISE SPECIFIED
6) CERTIFICATE OF INSURANCE AVAILABLE UPON REQUEST

EPCRA Supplier Notification: This product may contain one or more toxic chemicals subject to the reporting requirements of Section 313 of the Emergency Planning and Community Right-to-Know Act (Title III of the Superfund Amendments and Reauthorization Act of 1986) and 40 C.F.R. Part 372. Potentially reportable chemicals are indicated with a checkmark in the "EPCRA" column and a Chemical Abstract Services (CAS) registry number is provided for each such chemical in addition to the percent by weight of the chemical present in this product. It is your responsibility alone to determine whether your facility is required to submit a Toxic Release Inventory Report under EPCRA Section 313.

Certification No.: 1002717

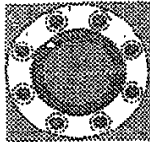
Certification Date: 5/5/2011

Issued By: Stephanie Kobs

This report is issued in compliance with the requirements of EN10204 3.1 / ISO 10474 3.1.b

Estela Cuellar

Estela Cuellar - Quality Manager



GALPERTI, INC
160 Southbelt Industrial Drive
Houston, Texas 77047
PH. 713-433-0700 - FAX 713-433-6680
Email: galperti@galperti-am.com

PROGRESSIVE SUPPLY

P. O. BOX 470748
TULSA OK 74145

This MTR applies to your

PO#

Part #3

1221

TEST/CERTIFICATE

EN 10204

INTERNAL JOB

3.1B

111166

M.T. CERTIFICATE N.

003083/B

DATE

04/07/2011

YOUR P.O.

11-9523

DATE

03/31/2011

X-7926-82D+89D

ITEM	DESCRIPTION	QUANTITY	R/P	MATERIAL	HEAT N°	#
004/004	12" 600 WN RF 80 A105 B1 F	4	B	ASTM A/SA105 03c	102817N/02	A
005/005	8" 150 WN RF XS A105 B1 F	50	B	ASTM A/SA105 03c	100506N/02	B
006/006	2" 300 BL RF A105 B1 F	200	B	ASTM A/SA105 03c	3019785	C

Noz. VI + DI

CHEMICAL ANALYSIS

#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CB %	FL %
A	0.170	1.170	0.190	0.008	0.002	0.080	0.080	0.030	0.150	0.017		0.026	0.002			0.002	0.40
A C	0.160	1.190	0.210	0.005	0.002	0.090	0.070	0.040	0.160	0.012		0.037	0.002			0.002	0.40
B	0.170	1.150	0.260	0.011	0.001	0.110	0.070	0.040	0.200	0.016		0.028	0.003			0.003	0.41
B C	0.160	1.130	0.280	0.014	0.001	0.100	0.060	0.050	0.210	0.011		0.048	0.003			0.003	0.39
C	0.190	1.050	0.230	0.006	0.017	0.050	0.090	0.020	0.190	0.003			0.001			0.001	0.39
C C	0.180	1.070	0.250	0.009	0.020	0.060	0.100	0.010	0.180	0.003			0.001			0.001	0.39

NOTES

FL=Carbon Equivalent Long Formula

C = CHECK ANALYSIS

MECHANICAL PROPERTIES

#	TYPE	TENSILE								CHARPY TEST			
		Ø DIAMETER mm.	AREA mm²	LENGHT mm.	TEMP. F	TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES	HB HARDNESS (AVERAGE)
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	O	12.50	122.70	50.00	68	77,523	50,416	28.1	66.0				162
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	O	12.50	122.70	50.00	68	77,993	49,943	30.0	68.0				152
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
C	O	12.50	122.70	50.00	68	77,474	49,259	27.0	70.0				155

NOTES

O = OBTAINED
S = STANDARD

HEAT TREATMENTS

A	Normalized at 900 dgr.C for 3 h.-Cooling from 900 dgr.C in still air
A	
B	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air
B	
C	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air
C	

GENERAL NOTES

ROUGHNESS
F - STOCK FINISH 125-250 microinch

VERIFIED BY

STANDARD
B1 JUN 16 2011

NOTES

Material I.A.W. Nace MR-0175-2003 COUNTRY OF ORIGIN:USA ✓

EEC QC DEPT

L. J. J. J. J.

TRADE MARK / LOGO



Stamped on product

INT. INSPECT DATE

04/07/2011

INT. INSPECTOR

D. RUZICKA

INSPECTION AUTHORITY

[Signature]

GALPERTI, INC

An ISO 9001/2008 certified company.

Certification issued by:

Lloyds register quality assurance

Certificate No. UQA 0110391

FCI FORGED COMPONENTS INC.

14527 Smith Rd.
Humble, Texas 7739
TEL: (281) 441-4088
FAX: (281) 441-8899

Page 1 of 1
REPORT NUMBER 48290
DATE 5/2/2011
FCI ORDER NUMBER 52051
Processed I.A.W. EN10204 3.1

Part #3

CUSTOMER ORDER NUMBER 119533
SOLD/SHIPPED TO PROGRESSIVE SUPPLY
P.O. BOX 470748

TULSA OK 74147 USA

Item Quantity Description

6 15 2" X 300 X 12" LG RF LWN

X-7926-72 Noz. 5.2

HEAT NUMBER:
B834

MATERIAL TYPE:
SA105N

Material Origin: USA ✓
Manufactured in USA ✓
I.A.W. ASME 2010 ED.

CHEMICAL ANALYSIS

C	.190 ✓
Mn	1.030 ✓
P	.0090 ✓
S	.0200 ✓
Si	.220 ✓
Cr	.090 ✓
Mo	.030 ✓
V	.004 ✓
Cu	.230 ✓
Ni	.080 ✓
Cb	.002 ✓

PHYSICAL PROPERTIES

Yield PSI	57,400 ✓
Tensile PSI	79,200 ✓
Elongation	31 ✓
Reduction of Area	72 ✓
Hardness	152 HBW ✓

Al .023

C E .41

VERIFIED BY

JUN 16 2011

EEC QC DEPT

Heat Treatment NORMALIZED ✓
Temperature 1650°F
Time at Temperature 1/2 HR / IN THK MIN
Cooling Media AIR

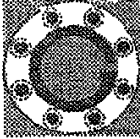
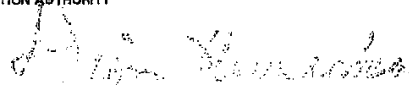
We hereby certify that all test results and process information contained herein are correct and true as contained in the records of the company.

Prepared by

Name: Janie Garcia

Title: QA Representative

Part #3

 GALPER, INC. 180 Southbelt Industrial Drive Houston, Texas 77047 PH. 713-433-0700 - FAX 713-433-5560 Email: galper@galper-inc.com		PROGRESSIVE SUPPLY P. O. BOX 470748 This MTR applies to your TULSA OK 74145 PO# <u>1221</u>															
		TEST/CERTIFICATE EN 10204 INTERNAL JOB 3.1B 110699															
M.T. CERTIFICATE N. DATE YOUR P.O. DATE 003382/B 04/14/2011 119502 X-7926-87 02/23/2011																	
ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#											
012/012	3" 300 WN RF 160 A105 B1 F	25	B	ASTM A/SA105 03c	3022384	A											
Noz. T1																	
CHEMICAL ANALYSIS																	
#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NR %	TI %	N %	CB %	FL %
A	0.200	1.060	0.250	0.011	0.016	0.100	0.100	0.030	0.250	0.003		0.002	0.001			0.001	0.42
A	0.190	1.080	0.270	0.008	0.013	0.090	0.110	0.040	0.240	0.003		0.021	0.001			0.001	0.41
NOTES: FL=Carbon Equivalent Long Formula C = CHECK ANALYSIS																	
MECHANICAL PROPERTIES																	
#	TENSILE									CHARPY TEST							
	TYPE	Ø DIAMETER (IN)	AREA (IN ²)	LENGTH (IN)	TEMP. F	TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES	HB HARDNESS (AVERAGE)				
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187				
A	O	12.50	122.70	50.00	68	77,228	50,270	28.8	69.0				161				
NOTES: O = OBTAINED S = STANDARD																	
#	HEAT TREATMENTS										GENERAL NOTES						
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air										ROUGHNESS STOCK FINISH 125-250 microinch VERIFIED BY JUN 16 2011 STANDARD B1 = ASME B16.5 EEC QC DEPT						
NOTES: Material I.A.W. Nace MR-0175-2003 COUNTRY OF ORIGIN: USA ✓																	
TRADE MARK / LOGO		INT. INSPECT DATE		INT. INSPECTOR		GALPER, INC. An ISO 9001/2008 certified company. Certification issued by: Lloyds register quality assurance Certificate No. UQA 0110391											
Stamped on product 		04/14/2011		D. RUZICKA													
INSPECTION AUTHORITY																	



UNITED STATES STEEL

TUBULAR PRODUCTS

CERTIFIED TEST REPORT

(IN ACCORDANCE WITH ISO 10474/EN10204/DIN50049 "Type 3.1")

DATE: 12/02/09
TIME: 09:34:37
SERIAL NO: F0056226

MILL ORDER ITEM NO DR47538.03	SHIPPER'S NO. 102543-00	VEHICLE ID	VENDOR USS TUBULAR PRODUCTS USS FAIRFIELD WORKS P.O. BOX 599 FAIRFIELD ALABAMA 35064
SOLD TO ADDRESS PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145		MAIL TO ADDRESS This MTR applies to your PO# 1221	

X-7926-81A

Noz. Ta

SPECIFICATION AND GRADE
PRESSURE PIPE CARBON SMLS HF ASTM A333-*05 GRADE 1 ASTM A333-*05 GRADE 6 ASME SA333-*2007 EDITION GRADE 1 ASME SA333-*2007 EDITION GRADE 6 ASME SA106-*08 GRADE B ASME SA106-*2007 EDITION GRADE B DUAL STENCIL BLK REG MILL COAT PE BEV 30 DEG MEETING ALL THE APPLICABLE REQUIREMENTS OF NACE STANDARD MR-01-75 *2003/COR.1:2005 AND MR0103-2007

MATERIAL COND.	QUENCH & TEMPER				Q.T. (mm)				WALL (mm)				L _o (mm)				L _o (mm)			
	PRODUCT IDENTIFICATION	TEST TYPE/ ORIENTATION	TEST COND.	GAUGE WIDTH	YIELD	UTS	ELONG.	YIT	YIT	ELONG.	HARDNESS	MIN HYDRO	UTS	ELONG.	YIT	YIT	ELONG.	HARDNESS	MIN HYDRO	UTS
U82073	STRIP/L/B	HT	HT	1.5	MIN: 35,000 MAX: 73,000	MIN: 50,000 MAX: 94,500	MIN: 30.0 MAX: 46.0	MIN: 0.77 MAX: 0.77	MIN: 0.77 MAX: 0.77	MIN: 30.0 MAX: 46.0	MIN: 99.5 MAX: 86.8	2800	MIN: 30.0 MAX: 46.0	MIN: 0.77 MAX: 0.77	MIN: 0.77 MAX: 0.77	MIN: 30.0 MAX: 46.0	MIN: 99.5 MAX: 86.8	2800	MIN: 30.0 MAX: 46.0	MIN: 99.5 MAX: 86.8
U82073	STRIP/L/B	HT	HT	1.5	MIN: 35,000 MAX: 73,000	MIN: 50,000 MAX: 94,500	MIN: 30.0 MAX: 46.0	MIN: 0.77 MAX: 0.77	MIN: 0.77 MAX: 0.77	MIN: 30.0 MAX: 46.0	MIN: 99.5 MAX: 86.8	2800	MIN: 30.0 MAX: 46.0	MIN: 0.77 MAX: 0.77	MIN: 0.77 MAX: 0.77	MIN: 30.0 MAX: 46.0	MIN: 99.5 MAX: 86.8	2800	MIN: 30.0 MAX: 46.0	MIN: 99.5 MAX: 86.8

LEGEND:	T. TRANSVERSE				Q.T. QUENCH & TEMPER				AR. AS ROLLED				B. BODY				W. WELD			
	PRODUCT IDENTIFICATION	TYPE	HEAT	PROD	UTS	ELONG.	YIT	YIT	UTS	ELONG.	YIT	YIT	UTS	ELONG.	YIT	YIT	UTS	ELONG.	YIT	YIT
U82073	HEAT	HEAT	16	106	008	008	19	04	05	029	008	00	00	00	00	00	00	00	00	00
U82073	PROD	PROD	18	100	008	008	18	04	05	026	008	00	00	00	00	00	00	00	00	00
U82073	PROD	PROD	17	99	007	007	18	04	05	026	007	00	00	00	00	00	00	00	00	00

VERIFIED BY

JUL 11 2011

DECIMAL POSITIONS FOR ELEMENTS ARE INDICATED BY THE LEFT MARGIN, VERTICAL DOTTED LINE OR DECIMAL POINT.

EEC QC DEPT

X. Patterson



UNITED STATES STEEL

TUBULAR PRODUCTS

CERTIFIED TEST REPORT
(IN ACCORDANCE WITH ISO 10474/EN10204/DIN50049 "type 3.1")

DATE: 12/02/09
TIME: 09:34:37
SERIAL NO: F0056226

MILL ORDER ITEM NO DR47538 03		SHIPPER'S NO.		P.O. NUMBER 102543-00		O.D. 6.625 (168.275)		I.D. (mm) 0.562 (14.274)		WALL 0.562 (14.274)		I.D. (mm)		WALL		I.D. (mm)							
MATERIAL COND: QUENCH & TEMPER		MIN COLLAPSE		GRAIN SIZE		FLAT		BEND		DIR		TEST LOK.		TEMP DEG F		SIZE		CHARTER VOUCHER IMPACT TESTING		TUBS		X-SHEAR	
PRODUCT IDENTIFICATION U82073		OK		** END OF DATA THIS SHEET **		L		B		- 60		2/3		HT		127		143		136		135	
LEGEND		L - LONGITUDINAL		T - TRANSVERSE		B - BODY		W - WELD		HAZ - HEAT AFFECTED ZONE													
TEST / INSPECTION		YES																					
FULL LENGTH VISUAL		X																					
FULL LENGTH EMI		X																					
FULL LENGTH MPI		X																					
FULL LENGTH UT		X																					
END AREA INSPECTION (PLAIN END)		X																					
SPECIAL END AREA (SEA) INSP																							
FULL LENGTH DRIFT																							
ADDITIONAL NOTES/COMMENTS																							
ALL MELTING AND MANUFACTURING TOOK PLACE IN THE USA. ✓ MANUFACTURED IN AN ISO 9001 CERTIFIED FACILITY - CERTIFICATE #30727. NO REPAIRS BY WELDING. NO MERCURY OR MERCURY COMPOUNDS ARE ADDED TO THE STEEL AND ALL MERCURY ** END OF DATA **																							

THIS IS TO CERTIFY THAT THE PRODUCT DESCRIBED HEREIN WAS MANUFACTURED,
SAMPLED, TESTED AND INSPECTED IN ACCORDANCE WITH THE SPECIFICATION
AND FULFILLS THE REQUIREMENTS IN SUCH RESPECTS.

PREPARED BY THE OFFICE OF: J. MAJURZAK - MANAGER, Q.A.

DATE 12/02/09

Part #4



STOCK #73

ORDER: 031595	009	HEAT: 574171	P.O.: 6419	PART NO.																																																								
SOLD TO: PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145		SHIP TO: This MTR applies to your PO# 19010																																																										
SPECIFICATION(S): ASME ASTM SA53/A53-10 GR B, SA106/A106-10 GR B/C, API 5L PSL1 GR B/X42 FORTY - FOURTH EDITION CERTIFIED NACE MR -01-75 DATED 10/15/2009																																																												
GRADE: 1023M	SIZE: 4.500 X 0.438	DRL	WPF: 19.02	QUALITY: SEAMLESS HOT FINISH																																																								
CONDITION (SPECIAL): ELECTRIC FURNACE FINE GRAIN MELT PRACTICE TEST METHODS USED: ASTM E415																																																												
TEST RESULTS CONFORM TO THE STANDARDS AND SPECIFICATION LISTED ABOVE.																																																												
<table border="1"><tr><td>C</td><td>Mn</td><td>S</td><td>P</td><td>Si</td><td>Cr</td><td>Ni</td><td>Mo</td><td>Cu</td><td>Al</td><td>V</td><td>Co</td><td>Ti</td><td>B</td></tr><tr><td>.23</td><td>.58</td><td>.005</td><td>.009</td><td>.28</td><td>.14</td><td>.11</td><td>.03</td><td>.30</td><td>.014</td><td>.003</td><td>.001</td><td>.0020</td><td>.0001</td></tr><tr><td>.23</td><td>.57</td><td>.001</td><td>.008</td><td>.26</td><td>.13</td><td>.10</td><td>.03</td><td>.26</td><td>.013</td><td>.002</td><td>.001</td><td>.0020</td><td>.0001</td></tr><tr><td>.23</td><td>.58</td><td>.001</td><td>.009</td><td>.27</td><td>.13</td><td>.10</td><td>.03</td><td>.27</td><td>.012</td><td>.001</td><td>.001</td><td>.0020</td><td>.0001</td></tr></table>					C	Mn	S	P	Si	Cr	Ni	Mo	Cu	Al	V	Co	Ti	B	.23	.58	.005	.009	.28	.14	.11	.03	.30	.014	.003	.001	.0020	.0001	.23	.57	.001	.008	.26	.13	.10	.03	.26	.013	.002	.001	.0020	.0001	.23	.58	.001	.009	.27	.13	.10	.03	.27	.012	.001	.001	.0020	.0001
C	Mn	S	P	Si	Cr	Ni	Mo	Cu	Al	V	Co	Ti	B																																															
.23	.58	.005	.009	.28	.14	.11	.03	.30	.014	.003	.001	.0020	.0001																																															
.23	.57	.001	.008	.26	.13	.10	.03	.26	.013	.002	.001	.0020	.0001																																															
.23	.58	.001	.009	.27	.13	.10	.03	.27	.012	.001	.001	.0020	.0001																																															
MATERIAL 100% MELTED AND MANUFACTURED IN THE USA.																																																												
Specimen ASTM E8	Yield KSI	Tensile KSI	ELNG 2"	R/A	C.E.	Rockwell ASTM E18 Avg	Hydrotest																																																					
1.000" STR	48.9	75.6	41.0		.39	77.6 80.1 75.6 77.8 RB	3000 PSI 5 Sec Hold																																																					
EN 10204 TYPE 3.1b DATED 01/2005 FLAT(S) - SATISFACTORY EMI INSPECTION - SATISFACTORY					VERIFIED BY EEC QC DEPT J. Jatterson mefinda@mbriale 9/28/11																																																							
DATE					QUALITY ASSURANCE/KOPPEL																																																							
MATERIAL WAS NOT EXPOSED TO MERCURY DURING PROCESSING. NO WELDING OR WELD REPAIR PERFORMED ON THIS MATERIAL. MATERIAL IS FREE OF HARMFUL RADIATION.																																																												
THIS CERTIFICATION MAY NOT BE REPRODUCED EXCEPT IN FULL.																																																												

IPSCO KOPPEL WORKS
P.O. BOX 750
BEAVER FALLS, PA 15010

IPSCO AMBRIDGE WORKS
P.O. BOX 410
AMBRIDGE, PA 15003

IPSCO BAYTOWN WORKS
2600 SPUR 55
BAYTOWN, TX 77523

Part #4

V&M FRANCE TUBERIE SAINT SAULVE SAINT SAULVE ZONE INDUSTRIELLE 59680 SAINT SAULVE 	 VALLOUREC & MANNESMANN TUBES Vallourec Group	MATERIAL TEST REPORT - INSPECTION CERTIFICATE 3.1 EN 10204 : 2004 No. : 16328SV07 Page: 1 / 5 Date: 31.10.2007
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X-7926 - 87H

Noz. T1

(A01) V&M FRANCE This MTR applies to your PO# <u>1221</u>	(A02.1) V&M-Order-No. VL0510 (A02.2) Suborder 856/4818 7
(A03.1) Customer PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145	(A07.1) Order-No. 41-046815
(A03.2) Orderer V & M TUBES CORPORATION / 77056 HOUSTON (TEXAS) 77056	(A07.2) Order-No. 41-046815 Date 06.09.2007
(A07.3) Order-No. F3033000	
(B01, B02, B04) Description of the product Hot finished seamless pipes for low-temperature service Ends bevelled, angle 30° (+5 / -0), root face 1.6 mm (± 0.8) Inside without rust protection - Outside dry varnish ASTM A 333 - 04 A ASTM A 999 - 04 A ASME SA 333, Edition 2004 ASME SA 999, Edition 2004, Addenda 2005 ASME Boiler and Pressure Vessel Code, Sect. II, Part A, Edition 2004, Addenda 2006 Nace MR 0175 / ISO 15156-2:2003 / COR.1:2005 / EN ISO 15156-2:2003, Annex A.2.1.2 Spec. Wfus-2, Rev. F, dated 04.2006 Grade 6 Grade 1	
ROCKWELL C HARDNESS ≈ 22 GUARANTEED ✓ MERCHANDISE IS FREE OF MERCURY CONTAMINATION AND NO WELD REPAIR WAS PERFORMED	

(A13) V&M Item	(A09) Cust. Item	(B14) Item text	(B09) Dimensions	(B10) Single length
4			NPS 3 SCH 160 ✓ OD-Tolerance + 0.031 in - 0.031 in WT-Tolerance + 15 % - 12.5 %	Random length from 36 to 42 Ft

(A13) V&M Item	(A09) Cust. Item	(B07.1) Heat	(B06) Quantity	(B11) Total length ft	(B13) Weight lb
4		320369	52	1.990,29	28.720

(C71)
HEAT CHEMICAL ANALYSIS

VERIFIED BY

JUL 11 2011

EEC QC DEPT

K. Patterson

Part #4

V&M FRANCE TUBERIE SAINT SAULVE SAINT SAULVE ZONE INDUSTRIELLE 59880 SAINT SAULVE 	 VALLOUREC & MANNESMANN TUBES Vallourec Group	MATERIAL TEST REPORT - INSPECTION CERTIFICATE 3.1 EN 10204 : 2004 No. : 16328Sv07 Page: 2 / 5 Date: 31.10.2007
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(C71)
HEAT CHEMICAL ANALYSIS

For each reduction of 0,01% carbon below 0.30 %, an increase of 0,05% manganese above 1.05 % would be permitted to a maximum of 1,35% manganese.

(B07.1) Heat	(B15) Process	C %	Si %	Mn %	P %	S %	Cr %	Mo %	Ni %	Cu %	V %
min	-	-	0,10	0,40	-	-	-	-	-	-	-
max	-	0,22	-	1,05	0,025	0,025	-	-	-	-	-
320369	Oxygen (BOF)	0,13	0,17	1,02	0,011	0,002	0,110	0,010	0,030	0,020	0,004

(B07.1) Heat	1003 %										
min	-										
max	0,40										
320369	0,33										

1003 $CE = C + \frac{Mn}{6} + \frac{(Cr + Mo + V)}{5} + \frac{(Ni + Cu)}{15}$

Heats fully killed

(B04)
HEAT TREATMENT

NORMALIZED CONDITION MIN 920°C (1688 F) COOLING AIR

TENSILE TEST RESULTS

Type (C10.1)	Tube strip specimen
Test temperature (C03)	Room temperature
Direction (C03)	longitudinal

Part #4

V&M FRANCE TUBERIE SAINT SAULVE SAINT SAULVE ZONE INDUSTRIELLE 59880 SAINT SAULVE 	 VALLOUREC & MANNESMANN TUBES Vallourec Group	MATERIAL TEST REPORT - INSPECTION CERTIFICATE 3.1 EN 10204 : 2004 No. : 163285v07 Page: 3 / 5 Date: 31.10.2007
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TENSILE TEST RESULTS

(307.1) Heat	(300.1) Test Piece	(C10.2) Dimension	(C11) YS ↓ R _{p0.2} psi	(C12) TS ↓ R _m psi	(C13.2) Elong. ↓ 2" %						
		ln / sq ln									
min		-	34984	60063	35						
max		-	-	-	-						
320369	01MM697	0,74x0,42 0,31	41763	65666	41						
320369	02MM697	0,74x0,43 0,32	42073	66592	41						
320369	03MM697	0,74x0,47 0,35	42073	66011	42						

(C10.2) Dimension	Test piece dimensions Test piece area
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IMPACT TEST RESULTS

Type (C40)	Charpy V
Test temperature (C05)	-80 °F
Direction (C02)	longitudinal

(307.1) Heat	(300.1) Test Piece	(C41) Dimension	(C42.1) Impact1	(C42.1) Impact2	(C42.1) Impact3	(C43.1) Mean	(C42.2) Shear fr1	(C42.2) Shear fr2	(C42.2) Shear fr3		
		ln / sq ln	ft-lbf	ft-lbf	ft-lbf	ft-lbf	%	%	%		
min		-	10,3	10,3	10,3	13,3	-	-	-		
max		-	-	-	-	-	-	-	-		
320369	01MM697	0,39x0,39 0,12	208,4	222,5	244,1	225,0	100	100	100		

Part #4

V&M FRANCE TUBERIE SAINT SAULVE SAINT SAULVE ZONE INDUSTRIELLE 59680 SAINT SAULVE 	 VALLOUREC & MANNESMANN TUBES Vallourec Group	MATERIAL TEST REPORT - INSPECTION CERTIFICATE 3.1 EN 10204 : 2004 No. : 183285v07 Page: 4 / 5 Date: 31.10.2007
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IMPACT TEST RESULTS

(907.1) Heat	(000.1) Test Piece	(C41) Dimension	(C42.3) L.Exp1	(C42.3) L.Exp2	(C42.3) L.Exp3						
		in / sq in	mm	mm	mm						
min		-	-	-	-						
max		-	-	-	-						
320369	01MM697	0,39x0,39 0,12	1,3	1,5	1,7						

(C41) Dimension Test piece dimensions Test piece area

(C86)

TECHNOLOGICAL AND OTHER TESTS ON SPECIMENS

Test	Conditions	Test rate	Result
Flattening test			Satisfactory ✓

(094)

OTHER TESTS ON PIPE

Test	Conditions	Test rate	Result
Hydrostatic test	2500 PSI 5 SEC	100% each lot	Satisfactory ✓
Appearance & Dimensions		100% each lot	Satisfactory

(A04, B06)

MARKING, IDENTIFICATION

Paint stenciled on one side	 V & M FRANCE A/SA333 1+6 HF SCH 160 S LT 60 F 3 0.438 HT HEAT NUMBER 41-046615 F3033000
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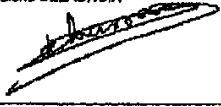
(201)

The supplied products are in compliance with the requirements of the order

Part #4

V&M FRANCE TUBERIE SAINT SAULVE SAINT SAULVE ZONE INDUSTRIELLE 59880 SAINT SAULVE 	 VALLOUREC & MANNESMANN TUBES Vallourec Group	MATERIAL TEST REPORT - INSPECTION CERTIFICATE 3.1 EN 10204 : 2004 No. : 16328SV07 Page: 5 / 5 Date: 31.10.2007
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(ACS, 202, 203)

Date	31.10.2007
Validated by	Inspection Representative
	Valérie DELACROIX 
☎	+ (33) 3 27 23 14 56
✉	+ (33) 3 27 23 15 25
@	valerie.delacroix@vmtubes.fr
Stamp	

This testimonial and certification respectively may neither be modified nor used for other products. Offences are regarded as falsification of documents and will be subject to criminal prosecution.

1221

Part #549

DATE: 12/26/05
TIME: 10:24:16
SERIAL NO:

TUBULAR PRODUCTS
CERTIFIED TEST REPORT

(TYPE B - IN ACCORDANCE WITH ISO 10474EN10204DIN50049 3.1.b)

UNITED STATES STEEL



MILL ORDER/ITEM NO. DR33040 04		SHIPPER NO. 3255		PD NUMBER		VEHICLE ID		MAIL TO ADDRESS		VENDOR	
SOLD TO ADDRESS		PROGRESSIVE SUPPLY, INC. P.O. Box 470748 Tulsa, OK 74147		X-7926-124 17				USS TUBULAR PRODUCTS 2199 EAST 28TH ST. LORAIN, OH 44055			
SPECIFICATION AND GRADE											
PRESSURE PIPE CARBON SMLS HF ASTM A333-X04A GRADE 1 ASTM A333-X04A GRADE 6 ASME SA333-X2004 EDITION GRADE 1 ASME SA333-X2004 EDITION GRADE 6 ASME SA106-X2004 EDITION GRADE B/C DUAL STENCIL BLK REG MILL COAT PE BEV 30 DEG MEETING ALL THE APPLICABLE REQUIREMENTS OF NACE STANDARD MR-01-75 X2002											
MATERIAL COND:		FULL BODY NORMALIZED		OD: 16.000 (406.400)		ID: 0.375 (9.525)		in (mm)		in (mm)	
PRODUCT IDENTIFICATION		TENSILE		YIELD		TENSILE		Y/T		HARDNESS	
TEST TYPE/ ORIENTATION		GAUGE WIDTH		EXT %		MIN: MAX:		MIN: MAX:		MIN: MAX:	
A03724 G112NA STRIP/L/B		1.500		.50		70000		0.69		99.0	
A03724 G112NA STRIP/L/B		1.500		.50		77500		0.69		1100	
		END OF DATA THIS SHEET									
LEGEND:		T - TRANSVERSE N - NORMALIZED		QT - QUENCHED & TEMPERED SR - STRESS RELIEVED		AR - AS ROLLED		B - BODY		W - WELD	
PRODUCT IDENTIFICATION		TYPE		C V		P S		V SI		CR MO	
A03724		HEAT		20		112		007		22	
A03724 G112NA PROD		20		112		007		008		22	
A03724 G112NA PROD		20		112		007		008		22	
VERIFIED BY		JUL 11 2011									
*C.E. IS BASED ON THE FOLLOWING EQUATION(S): CE = C + (MN/6) + (CR+MO+V)/5 + (NI+CU)/15											

DECIMAL POSITIONS FOR ELEMENTS ARE INDICATED BY THE LEFT MARGIN, VERTICAL DOTTED LINE OR DECIMAL POINT.

J. Patterson

DATE: 12/20/05
TIME: 10:24:16
SERIAL NO:

TUBULAR PRODUCTS
CERTIFIED TEST REPORT
(TYPE B - IN ACCORDANCE WITH ISO 10474/EN10204/DIN5049 3.1.b)

UNITED STATES STEEL



MILL ORDER/ITEM NO. DR33040 04		SHIPPER'S NO. 3255		P.O. NUMBER		OD: 16.000 (406.400)		ID: 0.375 (9.525)		W (mm)		D (mm)			
MATERIAL SPECIFICATION FULL BODY-NORMALIZED															
PRODUCT IDENTIFICATION A03724 G112NA	FLAT	BEND	GRAIN SIZE	MIN COLLAPSE	DIR	TEST LOC.	TEMP	SIZE	TEST COND.	CHARPY V-NOTCH IMPACT TESTING			% SHEAR		
	OK				L B	THIS SHEET	- 60	2/3	NM	1	2	3	1	2	3
LEGEND: L - LONGITUDINAL T - TRANSVERSE B - BODY W - WELD HAZ - HEAT AFFECTED ZONE															
TESTING / INSPECTION INFORMATION															
TEST / INSPECTION															
FULL LENGTH VISUAL															
FULL LENGTH EMI															
FULL LENGTH MPI															
END AREA INSPECTION (PLAIN END)															
SPECIAL END AREA (SEA) INSP.															
FULL LENGTH DRIFT															
RESULTS / COMMENTS															
OD X OD/ID L L X 10.0% NOTCH															
OD OD/ID L L															
MPI X UT															
MPI UT															
DRIFT MANDREL SIZE:															
ADDITIONAL NOTES/COMMENTS															
MELTED AND MANUFACTURED IN THE USA. NO REPAIRS BY WELDING. NO MERCURY OR MERCURY COMPOUNDS ARE ADDED TO THE STEEL AND ALL MERCURY BEARING EQUIPMENT IS PROTECTED BY A DOUBLE BOUNDARY OF CONTAINMENT.															

THIS IS TO CERTIFY THAT THE PRODUCT DESCRIBED HEREIN WAS MANUFACTURED, SAMPLED, TESTED AND/OR INSPECTED IN ACCORDANCE WITH THE SPECIFICATION AND FULFILLS THE REQUIREMENTS IN SUCH RESPECTS.

PREPARED BY THE OFFICE OF: J. MAJKRZAK - MANAGER, Q.A.

DATE: 12/26/05

Part #549

DATE: 02/09/11
TIME: 04:52:32
SERIAL NO: L0034839

TUBULAR PRODUCTS
CERTIFIED TEST REPORT
(IN ACCORDANCE WITH ISO 10474/EN10204/DIN50049 "Type 3.1")

UNITED STATES STEEL
MILL ORDER ITEM NO
DR00278 04

SHIPPER'S NO. R83324	PO NUMBER D1706000	VEHICLE ID S0063174
SOLD TO ADDRESS MAIL TO ADDRESS		
VEHICLE		
USS TUBULAR PRODUCTS 2199 EAST 28TH ST. LORAIN, OH 44055		

PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145		This MTR applies to your PO# 1221	
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SPECIFICATION AND GRADE PIPE CARBON SMLS STD PIPE API 5L-44TH ED DTD OCT 2007 AND ISO 3183-2007 MOD PSL-2 GRADE B AND GRADE X42 R N OR Q AND ANNEX C & E ASTM A53-07 ASTM A106-08 GRADE B QUAD STENCIL SPECIAL OD TOLERANCE ASME SA53-2010 EDITION ASME SA106-2010 EDITION GRADE B BLK REG MILL COAT PE BEV 30 DEG MEETING ALL THE APPLICABLE REQUIREMENTS OF NACE STANDARD MR-01-75 *2003/COR.1.2005 AND MR0103-2007	
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MATERIAL COND.	AS ROLLED	O.D.		I.D.		WALL		IN (mm)		IN (mm)		W. WELD	
		16.000 (406.400)		0.500 (12.700)		0.500 (12.700)		0.500 (12.700)		0.500 (12.700)		0.500 (12.700)	
PRODUCT IDENTIFICATION	TENSILE TEST TYPE/ ORIENTATION	TEST COND.	GAUGE WIDTH IN	YIELD MIN: MAX:	TENSILE MIN: MAX:	ELONG % (IN 2") MIN: MAX:	HARDNESS SCALE: HRB MIN: MAX:	MIN HYDRO PSI	DWELL SEC	IN (mm)	IN (mm)	W. WELD	CE- MAX
WALL175 S399AA	STRIP/L/B	AR	1.500	60000 .50	78000 .50	29.0 42.0	99.0 81.8	2240	5				
WALL176 S400AA	STRIP/L/B	AR	1.500	58500 .50	78000 .50	44.0 82.0	82.0	2240	5				
** END OF DATA THIS SHEET **													

VERIFIED BY
JUL 11 2011
DECIMAL POSITIONS FOR ELEMENTS ARE INDICATED BY THE LEFT MARGIN, VERTICAL DOTTED LINE OR DECIMAL POINT.

EEC QC DEPT
J. Patterson

Part #7

UNITED STATES STEEL

**TUBULAR PRODUCTS
CERTIFIED TEST REP
(IN ACCORDANCE WITH ISO 10474/EN10101)**

DATE: 02/09/11
TIME: 04:52:32
SERIAL NO: L0034839

[illegible]

THIS IS TO CERTIFY THAT THE PRODUCT DESCRIBED HEREIN WAS MANUFACTURED,
SAMPLED, TESTED AND/OR INSPECTED IN ACCORDANCE WITH THE SPECIFICATION
AND FULFILLS THE REQUIREMENTS IN SUCH RESPECTS.

PREPARED BY THE OFFICE OF: R. HARRIS - MANAGER, Q.A.

DATE 02/09/11

Part #7

12/28/2010 From: AMERICAN ALLOY STEEL
P.O. #: 0808
Item : 5 (1 PC) 3/8" X 96" X 240"
S.O. #: 380743
To: ENERGY EXCHANGERS
AA PL#: 5103345
TAG: STOCK

NUCOR
PLATE MILL

P.O. Box 279
Winston, NC 27986
(252) 358-3700

Mill Test Report

Page 1

AMERICAN ALLOY STEEL
PLATE **603385**
292
NUCOR
It's our Nature.

Issuing Date : 07/30/2010 BIL No. : 288541
Vehicle No: TTPX 806485
Specification : 0.3750" x 96.000" x 240.000"
ASTM A516 70-06/ASME SA516 70 P/Q 2009 Addenda Normalized
Test Coupons at 1850F Hold 30 Min per Inch of thickness Air Cooled

Load No. : 270193 Our Order No. : 83431/1
Ship To : AMERICAN ALLOY STEEL, INC.
6230 NORTH HOUSTON ROSSLYN RD.
NORTH HOUSTON, TX 77091
Cust. Order No. : 82828-OK
Ship To : AMERICAN ALLOY STEEL, INC.
C/O SAND SPRINGS RAILWAY
COMPANY
TRANS LOAD FACILITY
TULSA, OK 74116

Marking : 2450/28400 82626-OK

X-7926-120

Heat No	C	Mn	P	S	Si	Cu	Ni	Cr	Mo	Al	V	Nb	Ti	N	Ca	B	Sn	CEQ	PCM
0505315	0.19	1.04	0.015	0.002	0.17	0.32	0.10	0.13	0.02	0.024	0.005	0.003	0.002		0.0013	0.0002	0.011	0.42	0.27
Plate Serial No	Tensile Test										Charpy Impacts								
	Pieces	Tons	Dir.	Yield	(psi)	Tensile	Elongation % In 2"	Elongation % In 8"	Dir.	1	(%) shear	2	(%) shear	3	(%) shear	Ave.	Size	Temp	Ave.
0505315-03	12	14.70	T	61,600	77,000		21.6												
			T	50,100	74,100		24.8												

CHECKED JAN - 3 2011

J. J. J. J.

Certified a true copy of the original retained in our file.
AMERICAN ALLOY STEEL
Reviewed by: **D. S. J. J.**

Part # **12**

Test coupons only, normalized 30 minutes per inch of thickness at 1850 F ± 25 F. Hold 30 minutes minimum;

Manufactured to fully killed fine grain practice by Electric Arc Furnace. Welding or weld repair was not performed on this material. We hereby certify that the contents of this report are accurate and correct. All test results and operations performed by the material manufacturer are in compliance with the applicable specifications, including customer specifications.

Yield by 0.5% method unless otherwise specified. CEQ = C+(Mn/8)+(Cr+Mo+V/5)+(Cu+Ni/15)
Pcm = C+(S/30)+(Mn/20)+(Cu/20)+(P/40)+(Cr/20)+(Mo/15)+(V/10)+B
Milled and manufactured in the USA. ISO 9001:2000 certified (#009461) by SRI Quality System Registrar (#0995-09). PED 97/23/EC 7th Annex 1, Para. 4.3 Compliant.
DIN 50049 3.1.B/EN 10204 3.1B(2004). DIN EN 10204 3.1(2005) compliant. For ABS grades only. Quality Assurance certificate 06-MMPQA-383

T. A. DePralla, Metallurgical

08/02/2010 11:50:54 AM

STOCK LIFTING LUGS

Load Number	Tally	Mill Order Number	P.O. Number	Part Number	Certificate Number	Date
389976	00000000417856	N-106993-003	32981		L333877-1	07/23/2011 04:48
Grade						
Order Description: A516 70, 0.5000 IN x 96.000 IN x 480.000 IN						
Quality Plan Description: A/SAS16-70: ASTM A516-70-10/ASME SAS16-70 2009 ADDENDA						

Shipped Item	Heat/Slab Number	Certified By	C	Mn	P	S	Si	Cu	Ni	Cr	Mo	Co	V	Al	Ti	N2	B	Ca	Sn	Cb
1G1767D	B117537-03 ***	B117537	0.23	1.11	0.010	0.007	0.20	0.20	0.07	0.07	0.011	0.000	0.002	0.037	0.001	0.007	0.0001	0.0031	0.007	0.45
1G1767E	B117537-03 ***	B117537	0.23	1.11	0.010	0.007	0.20	0.20	0.07	0.07	0.011	0.000	0.002	0.037	0.001	0.007	0.0001	0.0031	0.007	0.45

Shipped Item	Certified By	Heat Number	Yield ksi	Tensile ksi	Y/T	ELONGATION %		Bend OK?	Hard HB	Charpy Impacts (ft-lbf)				Shear %			Test Temp		
						2"	8"			Size mm	1	2	3	Avg	1	2		3	Avg
1G1767D	SLG1767BIT	B117537 ***	54.5	72.7	75.0	33.7													
1G1767D	SLG1767FTT	B117537 ***	55.5	76.3	72.7	34.2													
1G1767D	SLG1767MTT	B117537 ***	52.5	72.7	72.2	33.4													
1G1767E	SLG1767BIT	B117537 ***	54.5	72.7	75.0	33.7													
1G1767E	SLG1767FTT	B117537 ***	55.5	76.3	72.7	34.2													
1G1767E	SLG1767MTT	B117537 ***	52.5	72.7	72.2	33.4													

X-7926-62 Out to 6'1/2" LG.

Name	PIONEER
PO#	767386
Count	1
Size	1/2 X 96 X 240 A516-70
Heat#	B117537-03

PIONEER STEEL INC.	
DATE	22.12 QTY. 100
PO#	1874
PART#	stock
BY	

Items: 2 PCS; 6 Weight: 39205 LBS

Mercury has not come in contact with this product during the manufacturing process not has any mercury been used by the manufacturing process. Certified in accordance with EN 10204 3.1. No weld repair has been performed on this material. Manufactured to a fully killed fine grain practice. ** Produced from Coil ** ISO 9001:2008 Registered, PED Certified

VERIFIED BY *[Signature]* April 2012 April Platts - QA Engineer

23 2012

REC QC DEPT

[Signature]

SSAB

Test Certificate

Form TCI: Revision 1: Date 31 Oct 2000

13609 Industrial Road, Houston, TX 77015

Customer: METALS USA PLATES & SHAPES SOU P.O. BOX 328 191 EAST ILLINOIS EMID OK 73702		Customer P.O. No.: MUS-231893		Product Description: ASTM A516-76(66)/ASME SA516-76(66A)		MIH Order No.: 41-277746-01		Shipping Manifest: HTB64418					
Size: 0.188 X 96.00 X 480.0 (IN)		Ship Date: 07 Sep 18		Cert No: 031116772		Ship Date: 07 Sep 18		Cert Date: 07 Sep 18					
STOCK NB BRACKETS		X-7926-61											
Tensiles		Charpy Impact Tests											
Heat Id	Place Id	Place Dimensions	Tst Loc	YS (PSI)	UTS (PSI)	RA (PSI)	2in 8in Dir	Average Hardness	Abs. Energy (FTLB)	% Shear	Tst Dir	Tst Temp (F)	BDWTT
W0B789	0051	0.187 X 96.00 (T.L.C)	L58000 C59800 T61000 L55000 C56000 T54000	75000 75000 74000 70000 71000 70000	27 26 25 28 27 31	T T T T T T							
W0H543	0024	0.187 X 96.00 (T.L.C)	L58000 C59800 T61000 L55000 C56000 T54000	75000 75000 74000 70000 71000 70000	27 26 25 28 27 31	T T T T T T							
Heat Id: W0B789, W0H543 Chemical Analysis: C, Mn, P, S, Si, Tot Al, Sol Al, Cu, Ni, Cr, Mo, Nb, V, Ti, CEV W0B789: C .19, Mn 1.11, P .014, S .003, Si .20, Tot Al .017, Sol Al .28, Cu .14, Ni .12, Cr .05, Mo .03, Nb .03, V .012, Ti .44, CEV .012 W0H543: C .19, Mn 1.10, P .009, S .003, Si .21, Tot Al .030, Sol Al .029, Cu .26, Ni .14, Cr .09, Mo .05, Nb .03, V .014, Ti .42, CEV .014													
MERCURY IS NOT A METALLURGICAL COMPONENT OF THE STEEL AND NO MERCURY WAS INTENTIONALLY ADDED DURING THE MANUFACTURE OF THIS PRODUCT. KILLED STEEL, PRODUCED TO A FINE GRAIN PRACTICE CEV (IIW) = C + Mn/6 + (Cr+Mo+V)/5 + (Ni+Cu)/15 MATERIAL MARKED WITH AN ASTERISK IS PRODUCED FROM COIL MTR DIN EN10204:2004 TYPE 3.1 COMPLIANT 100% MELTED AND MANUFACTURED IN THE USA. PRODUCTS SHIPPED: 0051 PCS: 6, WGT: 14741 * W0H543 0024 PCS: 12, WGT: 29482													
ENERGY EXCHANGER		HT#W0H543 SL#0024		ITEM#165PCS		VERIFIED BY		SEP 19 2011		EFC QC DEPT		Jason Thomas	
PO#1462		SO#203277											
Cust Part #:													

Part #14



CMC STEEL TEXAS
1 STEEL MILL DRIVE
SEGUIN TX 78155-7510

CERTIFIED MILL TEST REPORT

For additional copies call
830-372-8771

We hereby certify that the test results presented here

are accurate and conform to the reported grade specification

Daniel J. Schacht
Daniel J. Schacht

Quality Assurance Manager

HEAT NO.: 3018892

SECTION: FLAT 1/4x1-1/2 20'0"

A36/52950

GRADE: ASTM A36-08/A529-05 Gr 50

ROLL DATE: 09/03/2010

MELT DATE: 08/24/2010

S Steel Service Co. - OK

O 24412 Amah Pkwy

L Claremore OK

D US 74019-4304

T 9183416322

O 9183416390

S Steel Service Co. - OK

H 24412 Amah Pkwy

I Claremore OK

P US 74019-4304

T 9183416322

O 9183416390

Delivery#: 80441928

BOL#: 70155381

CUST PO#: 33419

CUST P/N:

DLVRY LBS / HEAT: 9216.000 LB

DLVRY PCS / HEAT: 362 EA

Characteristic Value

Characteristic Value

Characteristic Value

C 0.19% ✓

Mn 0.67% ✓

P 0.013% ✓

S 0.040% ✓

Si 0.17% ✓

Cu 0.30% ✓

Cr 0.12% ✓

Ni 0.09% ✓

Mo 0.023% ✓

V 0.009% ✓

Co 0.001% ✓

Sn 0.013% ✓

Al 0.002% ✓

Carbon Eq A529 0.39% ✓

Yield Strength test 1 52.5ksi ✓

Tensile Strength test 1 72.5ksi ✓

Elongation test 1 26% ✓

Elongation Gage Lgth test 1 8IN

THIS MATERIAL IS FULLY KILLED, 100% MELTED AND MANUFACTURED IN THE USA, WITH NO WELD REPAIR OR MERCURY CONTAMINATION IN THE PROCESS.

REMARKS :

MEETS THE REQUIREMENTS OF ASME SA36-2008A

CHECKED FEB 24 2011

01/28/2011 14:51:32

Page 1 OF 1

R. J. Patterson

Part #15

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49 5254 81-0 Fax: +49 5254 13666

BENTELER
Stahl/Rohr

X-7926

(185 p.s.)

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1
EN 10204:2005-01

Benteler Stahl/Rohr GmbH - Postfach 1340 - 33043 Paderborn - Deutschland
Webco Industries, Inc.
9101 West 21st Street
SAND SPRINGS, OK 74063
USA

Dokument-Nr.:
Document No.: 41-488044/001/P

No. du document:

Kunden-Bestell-Nr.:
Purchase Order No.: BST 08-7130 / Webco 10013786

No. de commande du client:

Benteler Auftrags-Nr.:
Benteler Order No.: 1530657

No. de commande Benteler:

Versandanzeigen-Nr.:
Dispatch Note No.: 4128151

No. d'avis d'expédition:

Produkt: NAHTLOSE STAHLROHRE

Product: SEAMLESS STEEL TUBES

Produit: TUBES D'ACIER SANS SOUDURE

Prüf-Nr.:
Inspection No.:

No. du certificat:

Hersteller:
Manufacturer: Warmrohrwerk/Zieherlei

Producteur:
Manufacturer: ISO/TS 16949 (ISO 9001) DNV CERT-116744-2003
PED 97/23/EC TÜV CERT 07-202-5635 WZ-006607

Herstellerzeichen:
Manufacturer's brand:

Marque du producteur:

Stempel des Abnahmebeauftragten: WA

Stamp of the inspection representative:
Timbre du contrôleur:

Stahlschmelzungsverfahren: ELEKTROSTAHL

Steelmaking process: ELECTRIC FURNACE

Procédé d'élaboration de l'acier: FOUR ELECTRIQUE

Lieferbedingungen: ASTM-A 179-1990a, ASME SA-179, ASME Section II Part A Edition 2007

Terms of delivery:

Conditions de livraison:

Maße - Toleranzen:

Dimensions-tolerances:

Dimensions-tolérances:

Stahlsorte:

Steel grade:

Nuance d'acier:

Lieferzustand:

Delivery condition:

État de livraison:

Produktkennzeichnung:

Product marking:

Marquage du produit:

CHECKED FEB 22 2010

see attachment product marking

Pos.	Stück	Maße	Länge	Gewicht	Schmelzen-Nr.	Prüfdruck	Rohr-Nr.-Gruppe	Vielfachlängen
Item	Number	Dimensions	Length	Weight	Heat No.	Test pressure	Tube number group	Multiple lengths
Posse	Nombre	Dimensions	Longueur	Poids	No. de coulée	Pression d'épreuve	Série de no. des tubes	Longueurs multiples
			feet	lbs				
0001	318	0.750" O.D. * 0.083" MW	29 FT 1"	5851	181070			
0002	385	0.750" O.D. * 0.083" MW	30 FT 1"	7291	181070			
0003	244	0.750" O.D. * 0.083" MW	31 FT 3"	4815	181070			

Part # 16

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49 5254 81-0 Fax: +49 5254 13666

BENTELER
Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1
INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1

Dokument-Nr.: 41-488044/001/P
Document No.:
No. du document:

Prüf-Nr.:
Inspection No.:
No. du certificat:

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Pos.	Stück	Maße	Item	Number	Dimensions	Poste	Nombre	Dimensions	Länge	Gewicht	Schmelzen-Nr.	Prüfdruck	Rohr-Nr.-Gruppe	Vielfachlängen
									Longueur	Poids	Heat No.	Test pressure	Tube number group	Multiple lengths
									feet	lbs	No. de coulée	Pression d'épreuve	Série de no. des tubes	Longueurs multiples
0004	153	0.750" O.D. * 0.083" MW	37 FT 1"	5673,75	3583	181069								
0005	187	0.750" O.D. * 0.083" MW	38 FT 1"	7121,10	4497	181069								
0006	262	0.750" O.D. * 0.083" MW	40 FT 2"	10523,03	6706	181073								
0007	427	0.750" O.D. * 0.083" MW	41 FT 1"	17542,36	11158	181073								
0008	472	0.750" O.D. * 0.083" MW	42 FT 1"	19863,39	12606	181073								
0009	491	0.750" O.D. * 0.083" MW	43 FT 1"	21152,70	13395	548853								
0010	195	0.750" O.D. * 0.083" MW	44 FT 1"	8595,87	5423	548853								
0011	139	0.750" O.D. * 0.083" MW	45 FT 1"	6266,40	4001	548853								
0012	140	0.750" O.D. * 0.083" MW	46 FT 1"	6451,58	4123	548853								
0013	231	0.750" O.D. * 0.083" MW	51 FT 6"	11896,39	7571	549318								

Schmelzenanalyse [%] / Heat analysis [%] / Analyse sur coulée [%]

Pos.	Schmelzen-Nr.	C	Si	Mn	P	S
Item	Heat No.					
Poste	No. de coulée					
0001	181070	0,100	0,160	0,48	0,008	0,003
0002	181070	0,100	0,160	0,48	0,008	0,003
0003	181070	0,100	0,160	0,48	0,008	0,003
0004	181069	0,100	0,160	0,48	0,008	0,003
0005	181069	0,100	0,160	0,48	0,008	0,003
0006	181073	0,100	0,170	0,47	0,010	0,003
0007	181073	0,100	0,170	0,47	0,010	0,003
0008	181073	0,100	0,170	0,47	0,010	0,003
0009	548853	0,100	0,160	0,47	0,007	0,002
0010	548853	0,100	0,160	0,47	0,007	0,002
0011	548853	0,100	0,160	0,47	0,007	0,002
0012	548853	0,100	0,160	0,47	0,007	0,002

last #18

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13866



Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1

Schmelzenanalyse [%] / Heat analysis [%] / Analyse sur coulée [%]

Pos. Schmelzen-Nr.

Item	Heat No.	C	SI	MN	P	S
Poste	No. de coulée					

0013	549318	0,100	0,150	0,46	0,013	0,003
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41-488044/001/P

Prüf-Nr.:
Inspection No.:
No. du certificat:

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Prüfergebnisse / Test results / Résultats des essais

Die Rohre sind auf Dichtheit geprüft durch:

The tubes have been submitted to a leak tightness test by:

Les tubes ont passé un contrôle d'étanchéité par:

Hydrostatic test: eddy current testing instead of hydrostatic test

PASSED ✓

Die Rohre wurden zerstörungsfrei geprüft:

The tubes are non destructive tested:

Les tubes ont passé un essai non destructif:

ET-test: acc. to ASME SA-450, ASTM-A 450; acc. to ASTM-E 309; Imperfections

PASSED ✓

Augensichtkontrolle:

Visual inspection:

Examen visuel:

PASSED

Aufweitversuch:

Drift expanding/flare test:

Essai d'évasement:

PASSED

Maßkontrolle:

Dimensions examination:

Vérification des dimensions:

PASSED

Ringfaltversuch:

Flattening test:

Essai d'aplatissement:

PASSED ✓

Ergebnisse der mechanischen Prüfung / Results of mechanical testing / Résultats des essais mécaniques

Die Probenahme erfolgte an Vielfachlängen.

The sampling was carried out on multiple lengths.

L'échantillonnage était réalisé aux longueurs multiples.

Härteprüfung / Hardness test / Essai de dureté

Pos.	Proben-Nr.	Schmelzen-Nr.	Härte
Item	Specimen No.	Heat No.	Hardness
Poste	No. de l'éprouv.	No. de coulée	Dureté

Anforderungen	HRC	HB	HV	HRB	HBW
---------------	-----	----	----	-----	-----

Requirements

Exigences

MAX 72

0001	000001	181070			
------	--------	--------	--	--	--

064

0001	000002	181070			
------	--------	--------	--	--	--

065

Part #16

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13666

ABNAHMEPRÜFZEUGNIS EN 10204-3.1
INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1

Härteprüfung / Hardness test / Essai de dureté

Dokument-Nr.: 41-48804/001/P
Document No.:
No. du document:

Prüf-Nr.:
Inspection No.:
No. du certificat:

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Pos. Item Poste	Proben-Nr. Specimen No. No. de l'éprouv.	Schmelzen-Nr. Heat No. No. de coulée	Härte Hardness Dureté	HRC	HB	HV	HRB	HBW
Anforderungen Requirements Exigences								
0001	000003	181070					MAX 72	
0001	000004	181070					065	
0002	000001	181070					063	
0002	000002	181070					064	
0002	000003	181070					065	
0002	000004	181070					065	
0003	000001	181070					063	
0003	000002	181070					064	
0003	000003	181070					065	
0003	000004	181070					065	
0004	000001	181069					063	
0004	000002	181069					064	
0005	000001	181069					064	
0005	000002	181069					064	
0006	000001	181073					065	
0006	000002	181073					063	
0006	000003	181073					064	
0006	000004	181073					063	
0007	000001	181073					063	
0007	000002	181073					063	
0007	000003	181073					064	
0007	000004	181073					063	
0008	000001	181073					063	

Part #16

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13686

ABNAHMEPRÜFZEUGNIS EN 10204-3.1
INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1
Härteprüfung / Hardness test / Essai de dureté

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Prüf-Nr.:
Inspection No.:
No. du certificat:

41-488044/001/P

Dokument-Nr.:
Document No.:
No. du document:

Pos. Item Poste	Proben-Nr. Specimen No. No. de l'éprouv.	Schmelzen-Nr. Heat No. No. de coulée	Härte Hardness Dureté	HRC	HB	HV	HRB	HBW
Anforderungen Requirements Exigences								
0008	000002	181073					MAX 72	
0008	000003	181073					064	
0008	000004	181073					064	
0008	000005	181073					065	
0008	000006	181073					063	
0009	000001	548853					063	
0009	000002	548853					064	
0009	000003	548853					065	
0009	000004	548853					064	
0010	000001	548853					064	
0010	000002	548853					065	
0011	000001	548853					064	
0011	000002	548853					065	
0012	000001	548853					064	
0012	000002	548853					065	
0013	000001	549318					065	
0013	000002	549318					064	

Part # 16

Vermerk / Remark

Certificate remarks: Steel and tubes manufactured by Benteler Stahl / Rohr ✓

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13666

BENTELER 

Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1
INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1
Beiblatt Produktkennzeichnung / Attachment product marking / Supplément marquage du produit

Pos.	Produktkennzeichnung	Dokument-Nr.: Document No.: No. du document:	41-488044/001/P	Prüf-Nr.: Inspection No.: No. du certificat:	Blatt: 6 / Page: Page:
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Pos. Produktkennzeichnung

Item Produktkennzeichnung

Poste Marquage du produit

0001, 0002, 0003, 0004, 0005, 0006, 0007, 0008, 0011, 0012

FS: BENTELER A179-SA179 HEAT-NO. DIMENSIONS TUBES-LENGTH FEET 2008 WA MADE IN GERMANY

0009, 0010

FS: BENTELER A179-SA179 HEAT-NO. DIMENSIONS TUBES-LENGTH FEET 2108 WA MADE IN GERMANY

FS: BENTELER A179-SA179 HEAT-NO. DIMENSIONS TUBES-LENGTH FEET 2408 WA MADE IN GERMANY

AEZ = Auditenbeschriftung, Etching ink marking, Gravure à l'encre FK = Farbkennzeichnung, colour marking, marquage par couleur FS = Farbschablonierung, nakit stencilling, marquage par peinture FSD = Farbschablonenrucker, Colour jet printer, Impprimante à jet d'encre de c
LS = Längsschablonierung, longitudinal stencilling, marquage longitudinal par peinture PKE = Ebenenkenzeichnung, tag marking, marquage sur étiquette PS = Prägestempel, die stamp, marquage par poinçonnage TS = Tintenstrahlkennzeichnung, ink jet spray marking, imprimante à jet d'encre

Verkäufer(in) / Salesman/ woman in charge / Personne chargée :

Mr Buerger, Tel.: 05254/81-4266, Fax: 4299

Paderborn Schloß Neuhaus, 24.06.2008, TEL.: 05254.81-1767 FAX: 05254.81-1752

Abnahmebeauftragter

Inspection representative

Contrôleur

ALWIN PANKRATZ / TEM

Es wird bestätigt, daß die gelieferten Erzeugnisse den techn. Lieferbedingungen des Auftrages entsprechen. Dieses Dokument wurde mittels EDV erstellt und ist ohne Unterschrift rechtsgültig.
We certify that the supplied products comply with the order specification. This document was prepared by means of electronic data processing and is valid without signature.
Nous attestons que les produits livrés sont conformes aux stipulations de la commande. Ce document a été établi par traitement électronique de l'information et est valide sans signature.

Part #16

DATE: 11/16/11

W. M. HEITGRAS COMPANY
P.O. BOX 2044
1316 N. OSAGE DRIVE
TULSA, OKLAHOMA 74101
(918) 583-3131

Part #17

CERTIFIED MATERIAL TEST REPORT

ENERGY EXCHANGERS
1844 N. GARNETT RD.
TULSA OK 74116

LOT # 11881890
HEAT # 11881890
STOCK # B7S100C600
1-8 X 6 ASTM A-193 B-7 STUD
CUS PO# 1618
ORDER # 280518

SPECIFICATIONS:

ASTM A193 B7 ALL THREAD ROD

X.7926.41+42

VENDOR: VULCAN THREADED PRODUCTS, INC.

QTY SOLD 32

- CHEMICAL COMPOSITION -

C %	✓	MN %	✓	P %	✓	S %	✓	SI %	✓	NI %	✓	CR %	✓
00.4200		00.8600		00.0110		00.0110		00.3100		00.0600		00.9600	
MO %	✓	CU %		CO %		V %		AL %		FE %		TA %	
00.2040						00.0050		00.0350					
TI %		ZN %		SE %		CB %		W %		N %		SN %	
												00.0060	

- CHARPY IMPACT -

FIRST: 0 SECOND: 0 THIRD: 0 AVERAGE: 0

- MECHANICAL PROPERTIES -

TENSILE STRENGTH: 136,500 Psi ✓
YIELD : 116,600 ✓
ELON : 20.00 ✓
RA : 59.00 ✓

HARDNESS : 28 HRC ✓
PROOF STRESS: 0 Psi

COMMENTS:

MELTED & MFG IN THE USA ✓
TEMPERING TEMP 1420 F
META S1/R1/C1

VERIFIED BY

NOV 17 2011

EEC QC DEPT

L. Jefferson

W.M. HEITGRAS COMPANY

Deary
QUALITY CONTROL MANAGER

INSPECTION CERTIFICATE

STOCK

Customer	Specification	Size	Lot No.	Date
	ASTM A-194 GRADE 2H HEAVY HEX. NUT.	1 - 8 UNC	8N392	Mar. 28, '11

Mechanical properties tested in accordance to ASTM F606/F606M, ASTM A370, ASTM E18



UNYTITE, INC.
One Unytite Drive
Peru, Illinois 61354
815-224-2221 — FAX# 815-224-3434

Chemical Composition (%)												
Mill	Material	Heat No.	Spec.	C	Si	Mn	P	S	Cu	Ni	Cr	Mo
GERDAU	AMER CARBON			MIN 0.40	MAX 0.40	MAX 1.00	MAX 0.040	MAX 0.050				
ISTEEL (NO. 1)	STEEL	M62635		0.44	0.21	0.66	0.008	0.024	0.24	0.07	0.11	0.02

Mechanical Property Inspection				Heat Treatment	
Item	Proof Load	Cone stripping	Hardness	After Heat Treatment Hardness	Absorbed Energy
Spec.	106,000 lbf	-	24 - 38 HRC	NIT 89 HBS 149	T: MIN. 850 P
	"	"	28.2 28.2 28.6 28.8 28.5	103.8	Q: FORGING Q (W.Q.) T: 1211 F/45M. (W.C.) Q: Quenching T: Tempering ST: Solution Treatment
	5		28.5		
Results	GOOD				

OFFICIAL SEAL
JEAN MARCHET
NOTARY PUBLIC - STATE OF ILLINOIS
MY COMMISSION EXPIRES 10/4/13
11-68-50

VERIFIED BY
JUN 1 2011

EEC QC DEPT

" 2H U "

Production Quantity
25,897 pcs.

Remarks:

GOOD

ANSI B1.1
CLASS 2B
GOOD

ANSI B18.2.2
GOOD

Material used for the nut was melted and manufactured in the USA. This nut was manufactured in the USA to the above specification.

We hereby certify that the material described has been manufactured and inspected satisfactorily with the requirement of the above specification.

Chief of Quality Assurance Section

[Signature]

Part #18

SECTION 5

ENERGY EXCHANGER
JOB# 54800
PO#PX9589

BODYCOTE THERMAL PROCESSING FCE#2001

X-7926 Top & Bottom Channel Covers



nk

CALIBRATION DATE 5-10-12
CALIBRATION DUE DATE 8-10-12

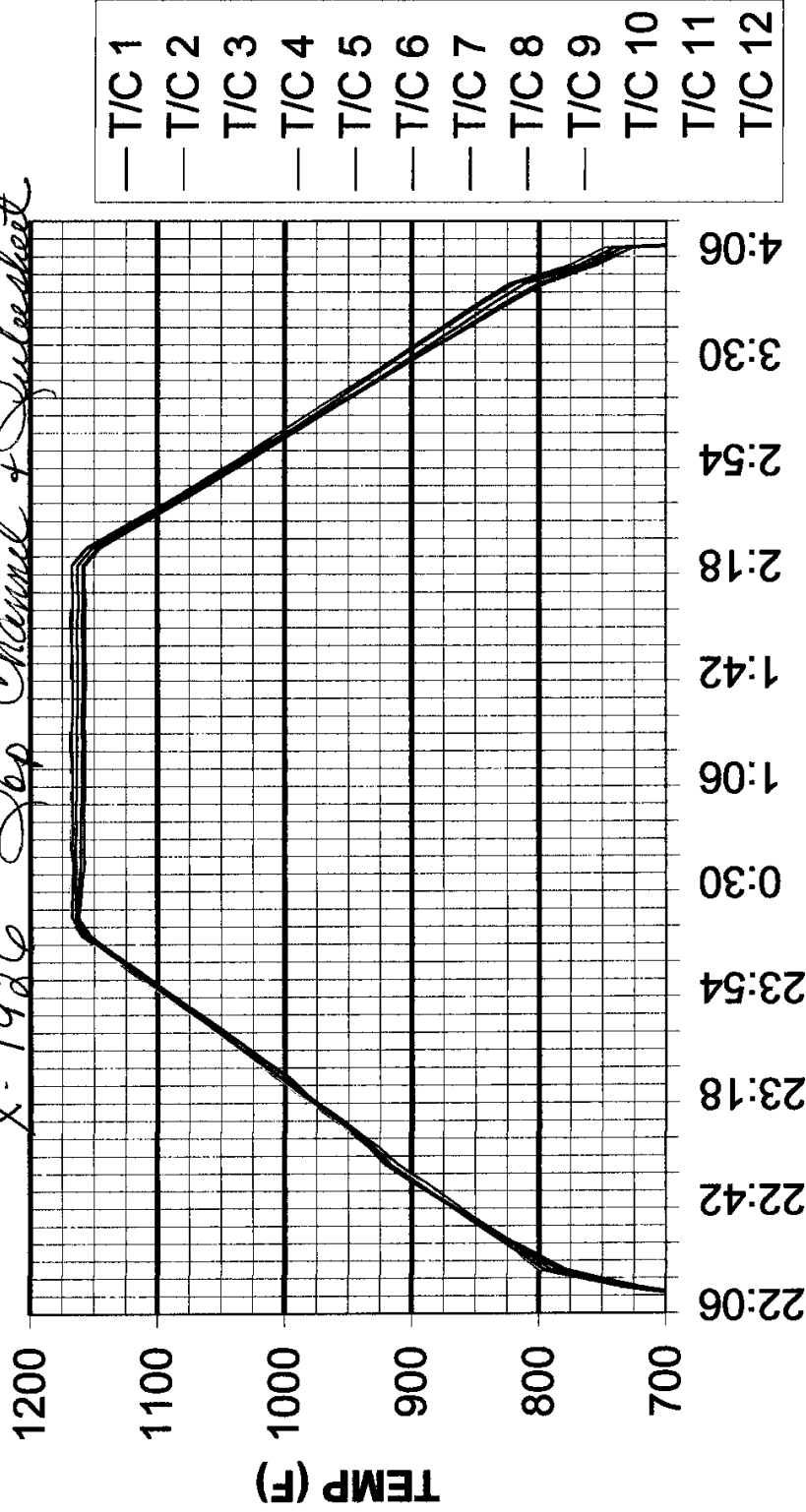
SIGNED BY CECIL HESTER

Cecil Hester

ENERGY EXCHANGER
JOB# 54617
PO# X-7926

BODYCOTE THERMAL PROCESSING FCE#2001

X-7926 Top Channel + Julee sheet



MB

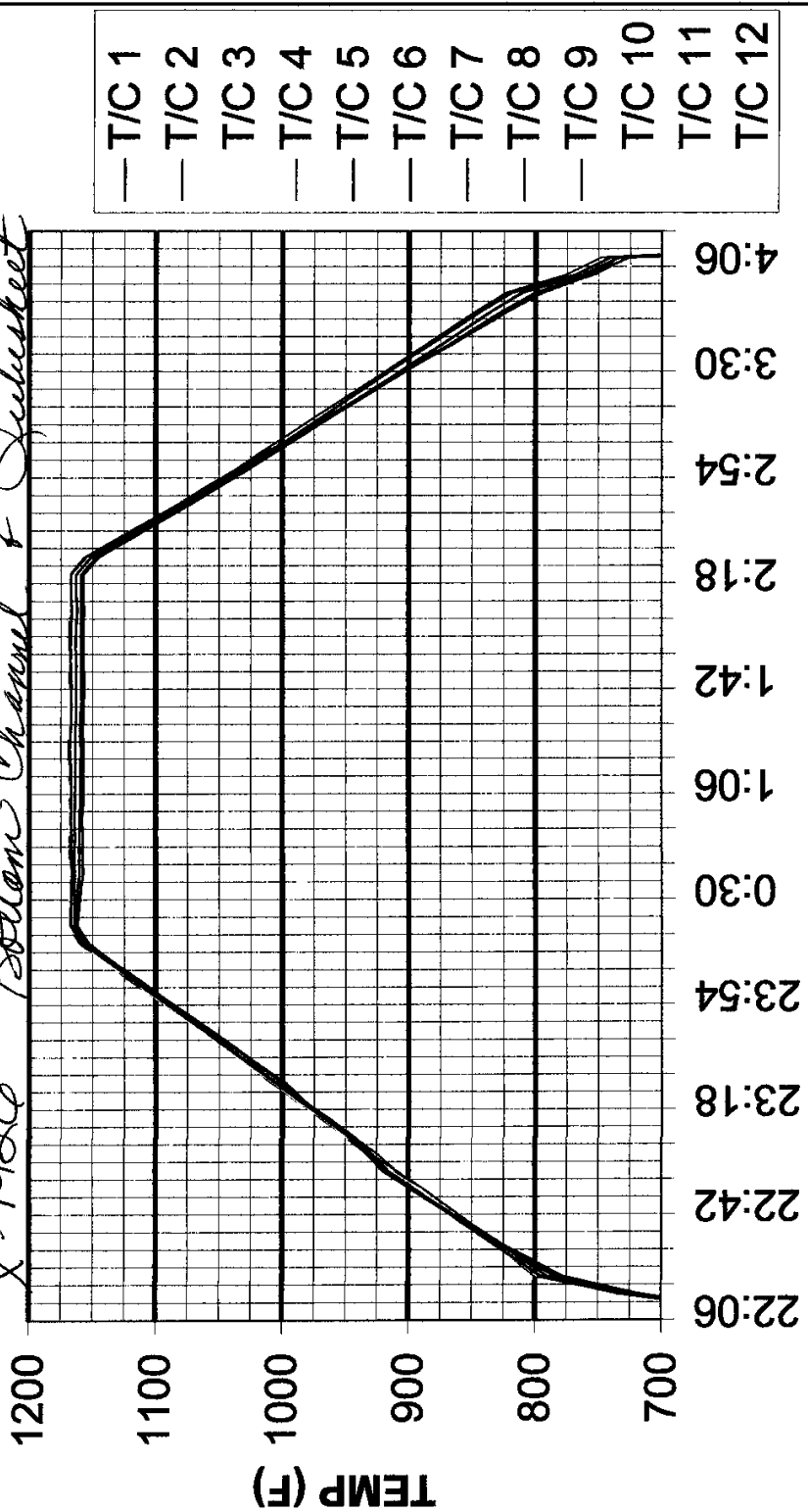
CALIBRATION DATE 5-10-12
CALIBRATION DUE DATE 8-10-12

SIGNED BY CECIL HESTER
Cecil Hester

ENERGY EXCHANGER
JOB# 54619
PO# X-7926

BODYCOTE THERMAL PROCESSING FCE#2001

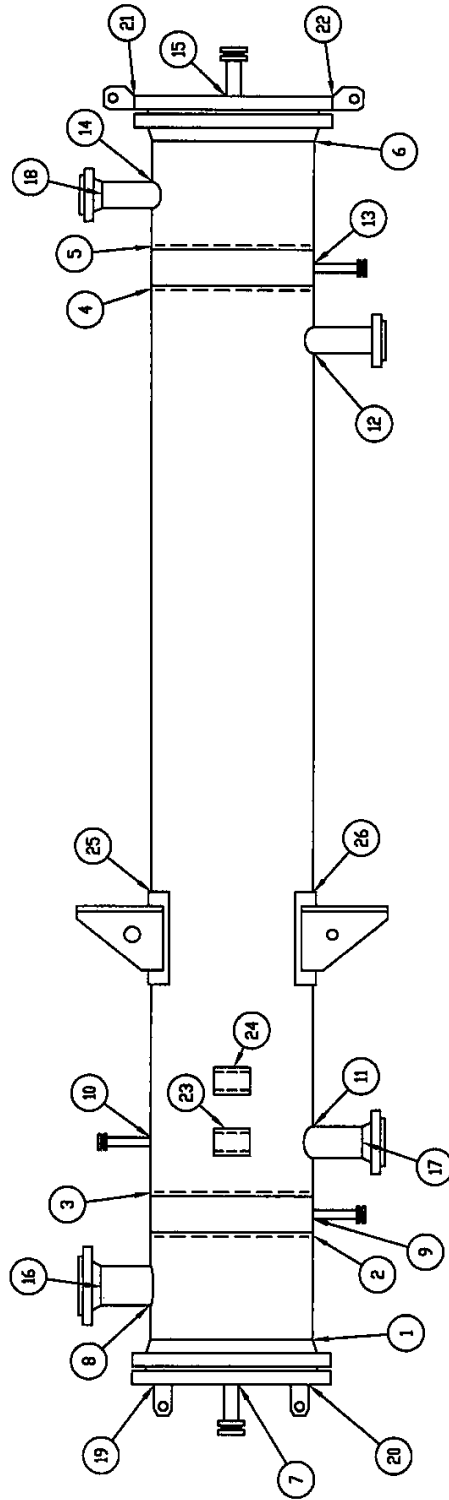
X-7926 Bottom Channel & Inlet Sheet



43

CALIBRATION DATE 5-10-12
CALIBRATION DUE DATE 8-10-12

SIGNED BY CECIL HESTER
Cecil Hester



CUSTOMER: MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES
 P.D. NO. 17496-0001 REG. NO. MC282
 SERVICE: WASTEWATER AMMONIA PURIFIER REBILER
 ITEM NO. H00062

NDE/WELD MAP

 Energy Exchange Co. 2001 W. Kentucky Ave. - York, Pa. 17403	DWN. BY	MD	DATE	7/26/12	DWG. NO.	X-7926-NDE	REV	0
	DATE	7/26/12	DATE					
	DATE		DATE					
	DATE		DATE					
REVISIONS								
NO.	DATE							

ENERGY EXCHANGER COMPANY

CUSTOMER: SOUTHERN COMPANY SERVICES **P.O.#:** 17496-0001
TAG: HX0052 **ENERGY JOB #** X-7926

NDE / WELD INDEX

[illegible]

ENERGY EXCHANGER COMPANY

JOB # X-7926

P.O.# 17496-0001

PART IDENTIFICATION Channels Spot uw-51

TYPE OF MATERIAL SA-333-6 MATERIAL THICKNESS 16" OD x 3/8"

SPECIFICATION NO: Q.C. WELDMENT **CODE/STANDARD:** ASME SEC. VIII, DIV. 1, A- O9

[illegible]

RADIOGRAPHIC DATA

1. RT DATE 6-2-12

2. MATERIAL EXAMINED:

(X) WELDMENT

() **CASTING**

() OTHER

3. METHODS:

(4) SINGLE EXPOSURE

() MULTIPLE EXPOSURE

() DOUBLE WALL TECHNIQUE

() SINGLE WALL TECHNIQUE

☐ **DOUBLE WALL VIEWING**

☐ SINGLE WALL VIEWING

() SINGLE FILM LOADING

() DOUBLE FILM LOADING

4. IQI (PENETRAMETER) DATA:

DESIGNATION G.O 14

SENSITIVITY LEVEL 0.013

LOCATION (FS) (SS)

5. FILM DATA:

FILM TYPE 100 80

FILM DENSITY MAX. 2-4

FILM DENSITY MIN. 2-4

6. SOURCE DATA:

ISOTOPE U22 SIZE 0.166

CURIES 92 E.T. 11

S.O.D. 7/58 S.S.O.F. 2/2

RADIOGRAPHER D. TOLSAK

INTERPRETER K. Carson

DATE REVIEWED 6-3-12

CUSTOMER INSPECTOR**DATE REVIEWED**

DATE: 6/5/12

CERTIFICATION OF EXAMINATION
BY
MAGNETIC PARTICLE (DRY POWDER METHOD)
(Procedure MT-1 Rev. 6)

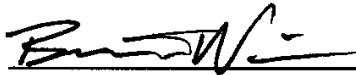
P.O. # 17496-0001TAG # HX0052ENERGY JOB # X-7926

MT LIFTING LUG WELDS,
NOZZLE BACK CUTS, ALL
EXTERNAL WELDS, ARC
STRIKES & TEMPORARY
ATTACHMENTS.

This certification is made to document that all Magnetic Particle Examinations have been performed on X-7926 in accordance with the requirements of ASME Section VIII, Division I, Appendix VI, ASME Section V, Article 7, customer specifications and Energy Exchanger Company drawings.

TEST INSTRUMENT: ESC-Y400 / ESC MS-27DPARTICLE TYPE & COLOR: DRY / GRAY Lot: 17748MATERIAL AND THICKNESS: SHELL: SA-106-B (3/8")
CHANNEL: SA-106-B (1/2")LIGHTING: 100 FT CANDLES MIN. @ SURFACE

ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager
ASNT NDT Level III, #193332
CWI # 10082731



BRINELL HARDNESS REPORT

JOB NO: X-7926

[illegible]

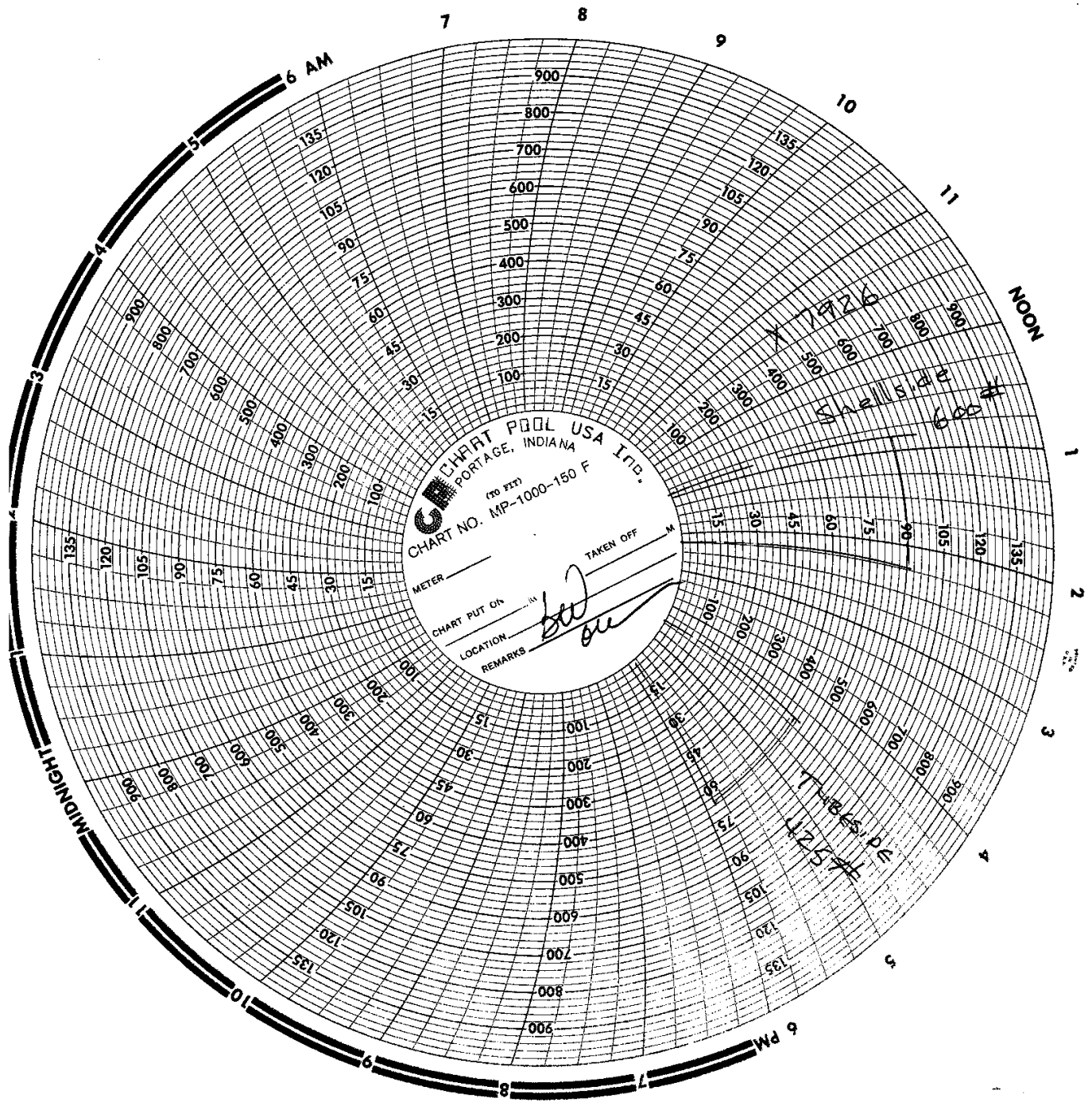
SECTION 6

DATE: 7-9-12**CERTIFICATE OF WELD/SUPPORT PAD AIR TEST**P.O. NO: 17496-0001TAG NO: HX0052ENERGY JOB NO: X-7926

This is to certify all weld/support pads on the above referenced equipment were air tested @ 15 P.S.I. and found to be satisfactory.

ENERGY EXCHANGER COMPANY


Brent Wilson
Quality Control Manager



DATE: 7-23-12

CERTIFICATE OF HYDROSTATIC TEST

P.O.# 17496-0001

TAG# HX0052

ENERGY JOB # X-7926

This is to certify the above referenced equipment was
hydro tested at the following pressures:

600 PSIG SHELL SIDE

425 PSIG TUBE SIDE

ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager

/lt



Date: 8-4-12

CERTIFICATE OF NITROGEN PURGE

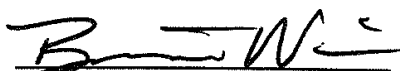
P.O. NO: 17496-0001

TAG NO: HK0052

ENERGY JOB NO: X-7926

This is to certify the above referenced equipment was purged
with 15 PSI dry nitrogen after hydro test.

ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager

Reviewed / Accepted
I.Q.S. # 3

William Brennaugh



ENERGY EXCHANGER COMPANY

PAINT CHECK LIST

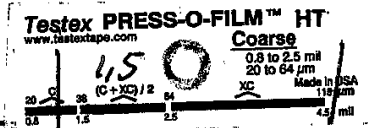
Contractor: EATON INDUSTRIAL COATINGS

Unit Serial No: X-7926 Customer Specification No: _____

Paint Required: 3-4 mils D-9 Primer; _____ mils _____ Intermediate;

(Supplies) → 1-2 mils 892 Finish

Satisfactory
Date



1. Paint Shelf Life

A. Materials in accordance with Project Specifications & Drawings

2. Notify Energy Exchanger Company

3. Weather Conditions:

97 temperature; 52% humidity; 95 substrate, 77 dew point,
EEC verify _____

4. Surface Prep:

(Req'd) SSPC-SP- 10 1-2 mils; (Actual) SSPC-SP- 10 1.5 mils

5. Surface Contamination

6. Surface Imperfections

7. Paint Contractor Re-check above

8. Storage, Mixing, Application

9. Masked Surfaces

10. Check Compressed Air for Contamination

11. Wet Film Thickness:

(Req'd) _____ (Actual) _____

12. Dry Film Thickness:

SHELL (Req'd) 3-4 (Actual) 4
SUPPLIES " 4-6 " 6

13. Tie Down/Shipping Preparation

2nd COAT (SUPPLIES)
TEMP 88
S.TEMP 85
HUMIDITY 54%
DEW POINT 69

me	7/23/12	

me	7/23/12	
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me	7/23/12	

me	7/26/12	
----	---------	--

me	7/26/12	
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SECTION 7

ENERGY EXCHANGER COMPANY

SPARE PARTS LISTS

- * 1) PRICES ARE F.C.A. ENERGY SHOPS, TULSA, OK.
- * 2) SHIPPING TERMS ARE PREPAY AND CHARGE.
- * 3) PRICE ARE VALID FOR TWELVE (12) MONTHS.
- * 4) UNIT PRICES ARE PER SET OF SPARE GASKETS.

**** 1) ENERGY EXCHANGER COMPANY RECOMMENDS ONE SET OF SPARE GASKETS PER EXCHANGER PER YEAR OF OPERATION.**

Energy Exchanger Company



Installation, Operation and Maintenance Manual for **SHELL AND TUBE HEAT EXCHANGERS**

A Member of the
Tubular Exchanger
Manufacturers Association



Energy Industrial Park/1844 North Garnett/Tulsa, Oklahoma 74116-1698
918 437-3000 / FAX 918 437-7144

INSTALLATION, OPERATION & MAINTENANCE MANUAL — Shell & Tube Heat Exchangers

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FOREWORD

Included in this manual are Tables and Figures which are reprinted in their entirety from the Standards of the Tubular Exchanger Manufacturers Association Seventh Edition (1988). Pages containing such material contain a citation to that publication.

Additionally, there are extensive extracts of text reprinted from the 7th Edition TEMA Standards. Pages containing such excerpts contain a footnote to that effect.

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GENERAL INSTRUCTIONS FOR INSTALLATION, OPERATION AND MAINTENANCE OF SHELL AND TUBE HEAT EXCHANGERS

SAFETY CONSIDERATIONS

LIFTING DEVICES

When eyebolts, lugs, clips or other devices are installed by the manufacturer, it is intended that these lifting devices are to lift only the empty components to which they are attached and not the assembled heat exchanger, unless explicitly specified otherwise.

Proper rigging must be used in lifting assembled heat exchangers. Exchangers are usually much heavier than pressure vessels of comparable dimensions; therefore, weights should be carefully evaluated. Where available, determinations should include reference to manufacturers drawings, added components, bill of lading, etc. Special care must be exercised in handling exchangers with expansion joints.

ROUTINE PRECAUTIONARY PROCEDURES

A heat exchanger is a multi-chamber pressure vessel designed at specific limits of pressure, temperature and fluid flow conditions as shown on the exchanger specification sheet and heat exchanger nameplate(s). The process system, which includes the heat exchanger, must be safeguarded so that the heat exchanger design conditions and operating limits are not exceeded. All operating and maintenance personnel should be made aware of specific limitations including pressures and temperatures, flow rates, start-up/shut-down procedures and cleaning procedures.

Do not remove channel covers, shell covers, floating head covers, bonnets or connecting piping until all pressure has been relieved and both shell and tube sides are completely drained. Exceptions are permissible when design permits independent pressure testing of shell or tube sides.

Plugged tubes and double tubesheets, unless vented, may remain pressurized after shell and tube sides are depressurized. Caution may be exercised in loosening of tube plugs or opening of vents to avoid sudden release of pressure or harmful fluids.

Many heat exchangers circulate fluids which are toxic, lethal or flammable and dangerous to the human system. These fluids could cause problems if bolted or threaded joints are not maintained in leak-tight condition under operating or no-flow ambient conditions. Proper precautions, such as effective draining and purging, must be taken in handling and decontamination when exchangers are opened for any reason. It is essential that the user advise maintenance or repair personnel, including outside contractors, when toxic, lethal, or flammable conditions exist.

When heat exchangers are cleaned, it is important that full characteristics of the circulating fluids, fouling material, and the cleaning agent be known and care exercised in handling them. Use eye protection, a respirator, or other appropriate protective devices.

Do not blow out heat exchangers with air when the process fluids or the cleaning fluids being handled are flammable or reactive.

SECTION 1 HEAT EXCHANGER NOMENCLATURE

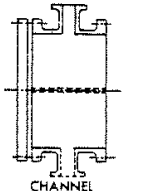
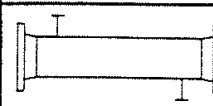
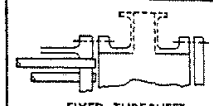
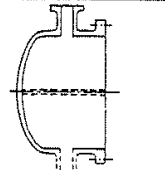
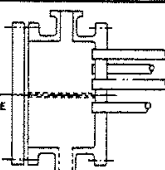
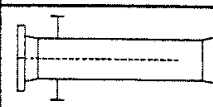
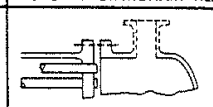
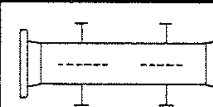
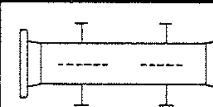
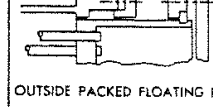
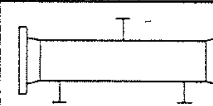
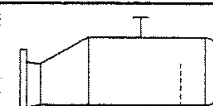
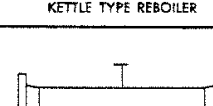
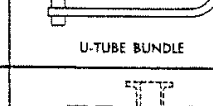
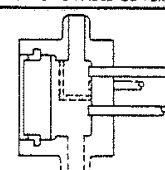
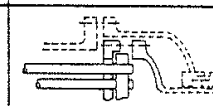
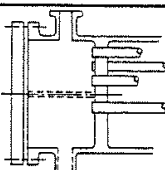
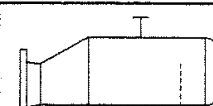
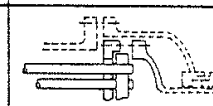
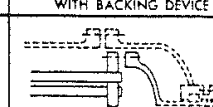
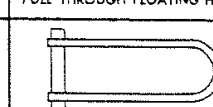
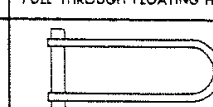
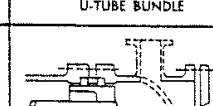
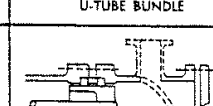
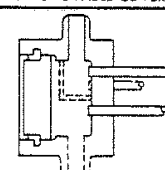
N-1 TYPE DESIGNATION-RECOMMENDED PRACTICE

It is recommended that heat exchanger type be designated by letters as described below.

N-1.1 TYPE

Type designation shall be by letters describing stationary head, shell (omitted for bundles only), and rear head, in that order, as indicated in Figure N-1.1.

Figure N-1.1

FRONT END STATIONARY HEAD TYPES		SHELL TYPES		REAR END HEAD TYPES	
A		E		L	
	CHANNEL AND REMOVABLE COVER				
					
	BONNET (INTEGRAL COVER)				
					
B	REMOVABLE TUBE BUNDLE ONLY	F		M	
	CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER				
					
	SPLIT FLOW				
					
C	DOUBLE SPLIT FLOW	H		N	
	DIVIDED FLOW				
					
	KETTLE TYPE REBOILER				
					
D	CROSS FLOW	X		P	
					
	SPECIAL HIGH PRESSURE CLOSURE				
					
	FLOATING HEAD WITH BACKING DEVICE				
N		K		S	
	CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER				
					
	PULL THROUGH FLOATING HEAD				
					
D	U-TUBE BUNDLE	U		W	
					
	EXTERNALLY SEALED FLOATING TUBESHEET				
					
	SPECIAL HIGH PRESSURE CLOSURE				

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SECTION 1 HEAT EXCHANGER NOMENCLATURE

N-2 NOMENCLATURE OF HEAT EXCHANGER COMPONENTS

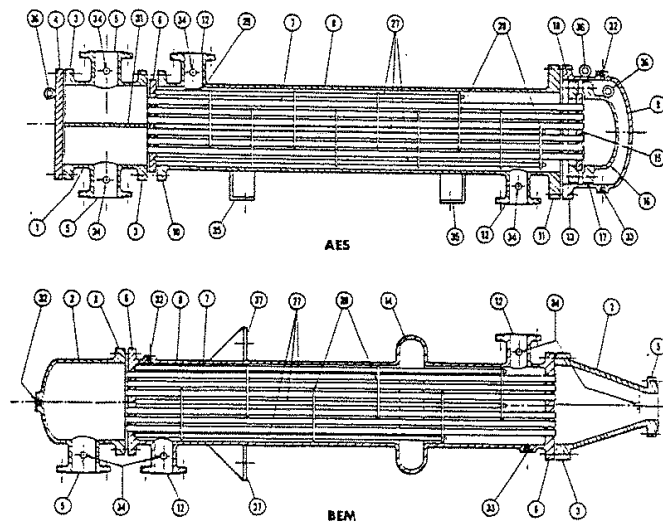
For the purpose of establishing standard terminology, Figure N-2 illustrates various types of heat exchangers. Typical parts and connections, for illustrative purposes only, are numbered for identification in Table N-2. (See Note)

Table N-2

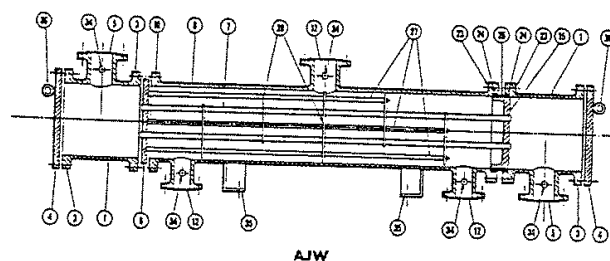
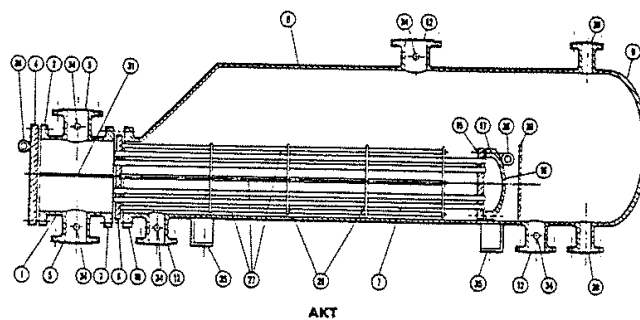
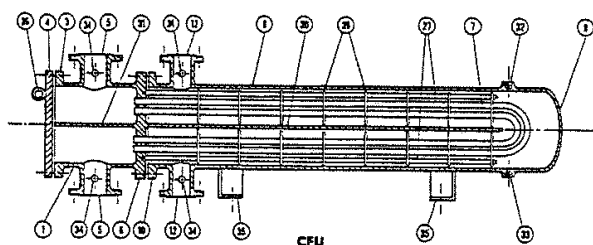
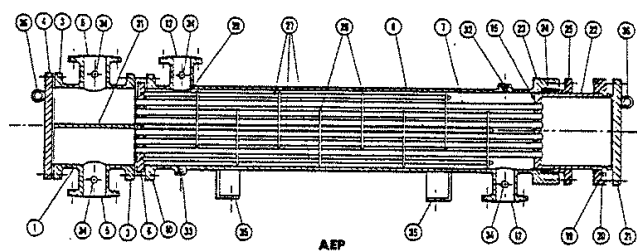
1. Stationary Head — Channel	20. Slip-on Backing Flange
2. Stationary Head — Bonnet	21. Floating Head Cover — External
3. Stationary Head Flange — Channel or Bonnet	22. Floating Tubesheet Skirt
4. Channel Cover	23. Packing Box
5. Stationary Head Nozzle	24. Packing
6. Stationary Tubesheet	25. Packing Gland
7. Tubes	26. Lantern Ring
8. Shell	27. Tierods and Spacers
9. Shell Cover	28. Transverse Baffles or Support Plates
10. Shell Flange — Stationary Head End	29. Impingement Plate
11. Shell Flange — Rear Head End	30. Longitudinal Baffle
12. Shell Nozzle	31. Pass Partition
13. Shell Cover Flange	32. Vent Connection
14. Expansion Joint	33. Drain Connection
15. Floating Tubesheet	34. Instrument Connection
16. Floating Head Cover	35. Support Saddle
17. Floating Head Flange	36. Lifting Lug
18. Floating Head Backing Device	37. Support Bracket
19. Split Shear Ring	38. Weir
	39. Liquid Level Connection

NOTE: Manufacturers may identify corresponding parts using numbers differing from the above.

Figure N-2



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SECTION 2

HEAT EXCHANGER IDENTIFICATION AND DEFINITIONS

D-2.1 EXCHANGER IDENTIFICATION

D-2.11 COMPLETE EXCHANGER

Manufacturers normally assign a serial number which is unique and permanently identifies each exchanger. Inquiries to the manufacturer should reference this number when it is shown on the nameplate and manufacturer's drawings. Additional useful information normally shown includes size, type, and item number.

D-2.12 EXCHANGER PARTS

Exchanger parts should be identified by the exchanger serial number and by the name and part number corresponding to those shown in Section 1, Table N-2, or as shown on the manufacturer's drawings.

D-2.2 DEFINITIONS

D-2.21 EXCHANGER UNIT

One or more exchangers designed for a specific service.

D-2.22 ITEM NUMBER

The purchaser's identification number for an exchanger unit.

D-2.23 NORMAL OPERATING CONDITIONS

The thermal and hydraulic performance requirements generally specified for designing the heat exchanger.

D-2.24 UPSET CONDITIONS

A departure from specified operating conditions.

D-2.25 PULSATING FLUID CONDITIONS

Conditions of flow generally characterized by rapid fluctuations in pressure and flow rate resulting from sources outside of the heat exchanger such as: pumps, compressors, blowers, etc.

D-2.26 START-UP CONDITIONS

The conditions of operation which exist from the time that flow of either or both process streams is initiated to the time that steady-state operating conditions are achieved.

D-2.27 SHUT-DOWN CONDITIONS

The conditions of operation which exist from the time of steady-state operating conditions to the time that flow of both process streams has ceased.

D-2.28 CODE

All references to Code herein mean the ASME Boiler and Pressure Vessel Code, Section VIII, Division 1, Pressure Vessels.

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SECTION 3

INSTALLATION OF HEAT EXCHANGERS

E-3 INSTALLATION

E-3.1 INSTALLATION PLANNING

E-3.11 CLEARANCE FOR DISMANTLING

For straight tube exchangers fitted with removable bundles, provide sufficient clearance at the stationary head end to permit removal of the bundle from the shell and provide adequate space beyond the rear head end to permit removal of the shell cover and/or floating head cover.

For fixed tubesheet exchangers, provide sufficient clearance at one end to permit withdrawal and replacement of the tubes and enough space beyond the head at the opposite end to permit removal of the bonnet or channel cover.

For U-tube heat exchangers, provide sufficient clearance at the stationary head end to permit withdrawal of the tube bundle, or at the opposite end to permit removal of the shell.

E-3.12 FOUNDATIONS

Foundations must be adequate so that exchangers will not settle and impose excessive strains on the exchanger. Foundation bolts should be set to allow for setting inaccuracies. In concrete footings, pipe sleeves at least one size larger than bolt diameter slipped over the bolt and cast in place are best for this purpose, as they allow the bolt center to be adjusted after the foundation has set.

E-3.13 PIPING

E-3.131 THERMAL EXPANSION

It is essential that adequate provisions be made in the connecting piping to allow for thermal expansion without imposing unspecified piping loads on the exchanger. This is particularly important for single pass internal floating head designs, outside packed floating heads, and fixed tubesheet types with shell expansion joints. Unless precautions are taken, leakage or damage may result.

E-3.132 PULSATION AND VIBRATION

In all installations, care must be taken to eliminate or minimize transmission of fluid pulsations and mechanical vibrations to the heat exchangers.

E-3.14 CLEANING

Convenient means should be provided for cleaning the unit as suggested under "Maintenance." (Section 5)

E-3.15 LEVELING

Exchangers must be installed so that pipe connections are made without forcing, disassembling or loosening of flanged assemblies.

E-3.16 STACKED HEAT EXCHANGERS

The manufacturer normally stacks exchangers prior to completion of fabrication to insure proper alignment of interconnecting nozzles. Shims separating the intermediate shell supports are normally used. Therefore, it is important that shims, if used, match the thicknesses and locations used by the manufacturer.

E-3.17 CONNECTION PROTECTORS

Remove shipping covers and plugs immediately prior to installation. Before connecting to piping, inspect all openings in the heat exchanger for foreign material. The entire system should be clean and free of foreign objects before starting operation. Do not expose heat exchanger internals to moisture or harmful contaminants.

E-3.18 PIPE CONNECTIONS

Pipe connections must be in accordance with the manufacturer's dimensional outline drawing.

E-3.19 FOUNDATION BOLTS

Foundation bolts should be loosened at one end of unit to allow free expansion of shells. Slotted holes in supports are provided for this purpose.

E-3.20 If the heat exchanger is equipped with a bellows type expansion joint, the expansion joint must be adequately protected during shipping and setting. Shipping supports must be removed in accordance with instructions.

E-3.3 HEAT EXCHANGER STORAGE

Heat exchangers normally are not protected for storage. If storage is necessary, a covered location at uniform temperature is preferable and provisions must be made to protect the heat exchanger interior from moisture, freezing, or harmful contaminants.

SECTION 4

OPERATION OF HEAT EXCHANGERS

E-4. OPERATION

E-4.1 PERFORMANCE OF HEAT EXCHANGERS

Satisfactory performance and service life can be expected only from heat exchangers which are properly installed, operated in accordance with design conditions, and receive preventive maintenance and cleaning on an appropriate schedule. These requirements are the responsibility of the user.

E-4.11 PERFORMANCE FAILURES

The failure of heat exchanger equipment to perform satisfactorily may be caused by one or more factors, such as:

- (1) Excessive fouling.
- (2) Air or gas binding resulting from improper piping installation or lack of suitable vents.
- (3) Operating conditions differing from design conditions.
- (4) Maldistribution of flow in the unit.
- (5) Excessive clearances between the baffles and shell and/or tubes, due to corrosion.
- (6) Improper thermal design.
- (7) Flooding resulting from inadequate drainage of condensate.

E-4.2 OPERATION OF HEAT EXCHANGERS

E-4.21 OPERATING PROCEDURES — NORMAL

Before placing any exchanger in operation, reference should be made to the exchanger drawings, specification sheet(s), and name plate(s) for any special instructions. Local safety and health regulations must be considered. Improper start-up or shut-down sequences, particularly of fixed tubesheet units, may cause leaking of tube-to-tubesheet and/or bolted flanged joints.

E-4.22 START-UP OPERATION

Most exchangers with removable tube bundles may be placed in service by first establishing circulation of the cold medium, followed by the gradual introduction of the hot medium. During start-up all vent valves should be opened and left open until all passages have been purged of air and are completely filled with fluid. For fixed tubesheet exchangers, fluids must be introduced in a manner to minimize differential expansion between the shell and tubes. (Refer E-4.24)

E-4.23 SHUT-DOWN OPERATION

For exchangers with removable bundles, the units may be shut down by first gradually stopping the flow of the hot medium and then stopping the flow of the cold medium. If it is necessary to stop the flow of cold medium, the circulation of hot medium through the exchanger should also be stopped. For fixed tubesheet exchangers the unit must be shut down in a manner to minimize differential expansion between shell and tubes, (Refer E-4.24). When shutting down the system, all units should be drained completely when there is the possibility of freezing or corrosion damage. To guard against water hammer, condensate should be drained from steam heaters and similar apparatus during start-up or shut-down. To reduce water retention after drainage, the tube side of water cooled exchangers should be blown out with air.

E-4.24 START-UP AND SHUT-DOWN PROCEDURES FOR FIXED TUBESHEET EXCHANGERS

When start-up and shut-down instructions are furnished, they must be followed. When specific instructions are not supplied, the recommended procedure is to start, or stop, the circulation of both shell and tube side fluids gradually and at the same time.

E-4.3 OPERATING LIMITATIONS

E-4.31 NORMAL OPERATING CONDITIONS

Equipment must not be operated at conditions more severe than those specified on the heat exchanger specification sheet, and/or name plates. A change in service or deviation from the specified operating conditions may require that a mechanical design check be made in order to avoid possible damage to the heat exchanger.

E-4.32 SUPPLEMENTAL NAME PLATE DATA

The manufacturer may supply supplemental data where it is pertinent to the operation or testing of the exchanger. This may include information pertaining to differential design and test pressure conditions, restrictions on operating conditions for fixed tubesheet type exchangers, or other restrictive conditions applicable to the design and/or operation of the unit or its components. Such information may be noted on the standard name plate or on a supplemental plate attached to the exchanger at the name plate location.

E-4.33 FLOW RATE

Heat exchangers are not to be operated at flow rates greater than those shown on the heat exchanger specification sheet. Excessive flow rates can cause vibration and/or erosion and severely damage the heat exchanger.

E-4.34 HYDRAULIC HAMMER

Exchangers should not be subjected to pulsating, slugging, or unstable flow conditions which may impair performance or result in destructive hydraulic hammer. Adequate condensate drainage is essential for steam heaters and similar apparatus.

E-4.35 TEMPERATURE SHOCKS

Exchangers normally should not be subjected to abrupt temperature fluctuations. Hot fluid must not be suddenly introduced when the unit is cold, nor cold fluid when the unit is hot.

E-4.36 FLUID TEMPERATURE LIMITS

E-4.361 Heat exchangers are not to be operated at fluid temperatures more severe than those shown on the heat exchanger specification sheet.

E-4.362 For fixed tubesheet exchangers fluid inlet temperatures may be severely limited when there is no fluid on the opposite side. Operation with fluid circulation on one side only should be avoided, unless specifically provided for in the design of the heat exchanger.

E-4.4 BOLTED JOINTS

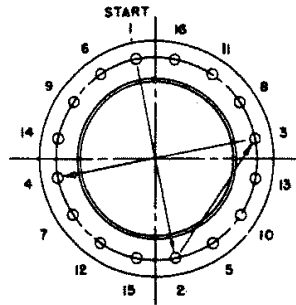
E-4.41 RECOMMENDATIONS FOR RETIGHTENING

Heat exchangers are pressure tested before leaving the manufacturer's shop in accordance with ASME Code requirements. However, normal yielding of gasket joints may occur in the interval between testing in the manufacturer's shop and installation at the job site. Therefore, all external bolted joints may require retightening after installation and, if necessary, after the exchanger has reached operating temperature.

E-4.42 RECOMMENDED BOLT TIGHTENING PROCEDURE

It is important that all bolted joints be tightened uniformly and in a diametrically staggered pattern as illustrated in Figure E-4.42 except for special high pressure closures when the instructions of the manufacturer should be followed.

Figure E-4.42



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SECTION 5 MAINTENANCE OF HEAT EXCHANGERS

E-5. MAINTENANCE

E-5.1 INSPECTION OF UNIT

At regular intervals and as frequently as experience indicates, an examination should be made of the interior and exterior condition of the unit. Neglect in keeping all tubes clean may result in complete stoppage of flow through some tubes which could cause severe thermal strains, leaking tube joints, or structural damage to other components. Sacrificial anodes, when provided, should be inspected to determine whether they should be cleaned or replaced.

E-5.11 INDICATIONS OF FOULING

Exchangers subject to fouling or scaling should be cleaned periodically. A light sludge or scale coating on the tube greatly reduces its efficiency. A marked increase in pressure drop and/or reduction in performance usually indicates cleaning is necessary. The unit should first be checked for air or vapor binding to confirm that this is not the cause for the reduction in performance. Since the difficulty of cleaning increases rapidly as the scale thickness or deposit increases, the intervals between cleanings should not be excessive.

E-5.12 PREVENTIVE MAINTENANCE

Additional important reasons for inspections include:

- (1) Corrosion and erosion may weaken various parts of an exchanger and may eventually cause mechanical failure, or impaired performance.
- (2) Determination of fouling and corrosion rates for prediction of service life and scheduling down-time.
- (3) Implementation of preventive maintenance procedures which reduce rates of fouling and general deterioration, or eliminate leaks.

E-5.13 ACCESS TO TUBES

Before disassembly, the user must assure himself that the unit has been depressurized, vented and drained, neutralized and/or purged of hazardous materials.

To inspect the inside of the tubes and also make them accessible for cleaning, the following procedures should be used:

- (1) Stationary Head End
 - (a) Type A, C, & D, remove cover only.
 - (b) Type B, remove bonnet.
- (2) Rear Head End
 - (a) Type L, N, & P, remove cover only.
 - (b) Type M, remove bonnet.
 - (c) Type S & T, remove shell cover and floating head cover.
 - (d) Type W, remove channel cover or bonnet.

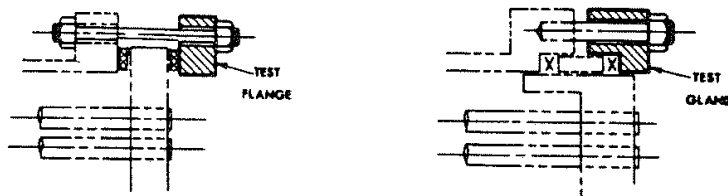
E-5.14 LOCATING LEAKS IN TUBES

The following procedures may be used to locate perforated or split tubes and leaking joints between tubes and tubesheets. In most cases, the entire front face of each tubesheet will be accessible for inspection. The point where water escapes indicates a defective tube or tube-to-tubesheet joint.

- (1) Units with removable channel cover:
Remove channel cover and apply hydraulic pressure in the shell.
- (2) Units with bonnet type head:
 - (a) On fixed tubesheet units where tubesheets are an integral part of the shell, remove bonnet and apply hydraulic pressure in the shell.
 - (b) On fixed tubesheet units where tubesheets are not an integral part of the shell and on units with removable bundles, remove bonnet, re-bolt tubesheet to shell or install test flange or gland, whichever is applicable, and apply hydraulic pressure in the shell. See Figure E-5.14-1 for typical test flange and test gland.

CAUTION: Bolting directly to the edges of the tubesheets may result in overstress unless it was considered in the design of the tubesheets.

Figure E-5.14-1

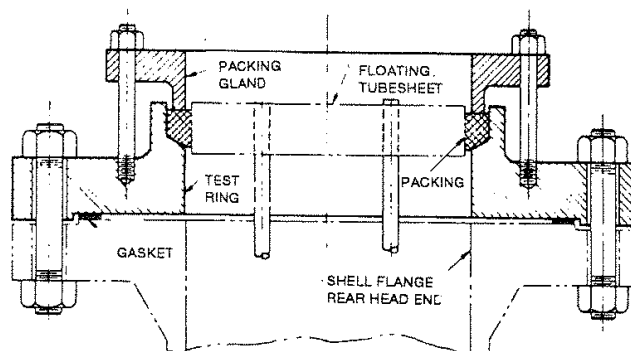


- (3) Units with type S or T floating head:
Remove channel cover or bonnet, shell cover and floating head cover. Install test ring and bolt in place with gasket and packing. Apply hydraulic pressure in the shell. A typical test ring is shown in Figure E-5.14-2. When a test ring is not available it is possible to locate leaks in the floating head end by removing the shell cover and applying hydraulic pressure in the tubes. Leaking tube joints may then be located by sighting through the tube lanes.

NOTE: Care must be exercised when testing partially assembled exchangers to prevent over extension of expansion joints or overloading of tubes and/or tube-to-tubesheet joints.

Hydrostatic test should be performed so that the temperature of the metal is at least 30°F above the minimum design metal temperature, MDMT.

Figure E-5.14-2



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E-5.2 TUBE BUNDLE REMOVAL

Bundles must be withdrawn from the stationary end as noted on the drawing. All parts such as glands, retainer rings, split shear rings and closure packing must be removed before attempting to remove bundle. Threaded eyebolts may be attached to the stationary tubesheet when tapped holes have been provided for this purpose. When eyebolt tappings are not provided, rods may be passed through two or more tubes and attached to a steel bearing plate over the floating tubesheet. Provide a protective spacer between the bearing plate and tubesheet to protect the tube ends.

E-5.21 HANDLING TUBE BUNDLES

Tube bundles should not be handled with hooks or other devices which might damage the tubes. Bundles should be supported on cradles or skids. Horizontal tube bundles should be lifted by means of suitable slings. Baffles can be bent and damaged by dragging a bundle over a rough surface. All gasket and packing contact surfaces should be protected from accidental damage since these areas are generally difficult to repair.

E-5.3 CLEANING TUBE BUNDLES

E-5.31 CLEANING METHODS

The heat transfer surfaces of heat exchangers should be kept reasonably clean to assure satisfactory performance. Convenient means for cleaning should be made available.

Heat exchangers may be cleaned by either chemical or mechanical methods. The method selected must be the choice of the operator of the plant and will depend on the type of deposit and the facilities available in the plant. Following are several cleaning procedures that may be considered:

- (1) Circulating hot wash oil or light distillate through tubes or shell at high velocity may effectively remove sludge or similar soft deposits.
- (2) Some salt deposits may be washed out by circulating hot fresh water.
- (3) Commercial cleaning compounds are available for removing sludge or scale provided hot wash oil or water is not available or does not give satisfactory results.
- (4) High pressure water jet cleaning.
- (5) Scrapers, rotating wire brushes, and other mechanical means for removing hard scale, coke, or other deposits.
- (6) Employ services of a qualified organization that provides cleaning services. These organizations will check the nature of the deposits to be removed, furnish proper solvents and/or acid solutions containing inhibitors, and provide equipment and personnel for a complete cleaning job.

E-5.32 CLEANING PRECAUTIONS

- (1) Tubes should not be cleaned by blowing steam through individual tubes since this heats the tube and may result in severe expansion strain, deformation of the tube, or loosening of the tube-to-tube sheet joint.
- (2) When mechanically cleaning a tube bundle, care should be exercised to avoid damaging the tubes.
- (3) Cleaning compounds must be compatible with the metallurgy of the exchanger.
- (4) Cleaning procedures, especially those which circulate hot fluids through the tubes, can cause significant temperature differentials between the shell and tubes. Such cleaning procedures can cause damage to fixed tubesheet heat exchangers and should be avoided unless specifically provided for in the design of the heat exchanger.

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E-5.4 GASKET REPLACEMENT

Gasket and gasket surfaces should be thoroughly cleaned and should be free of scratches and other defects. Gaskets should be properly positioned before attempting to retighten bolts. It is recommended that when a heat exchanger is dismantled for any cause, it be reassembled with new gaskets. This will tend to prevent future leaks and/or damage to the gasket seating surfaces of the heat exchanger. Composition gaskets become dried out and brittle so that they do not always provide an effective seal when reused. Metal or metal jacketed gaskets, when compressed initially, flow to match their contact surfaces. In so doing they are work hardened and, if reused, may provide an imperfect seal or result in deformation and damage to the gasket contact surfaces of the exchanger.

Bolted joints and flanges are designed for use with the particular type of gasket specified. Substitution of a gasket of different construction or improper dimensions may result in leakage and damage to gasket surfaces. Therefore, any gasket substitutions should be of compatible design.

Any leakage at a gasketed joint should be rectified and not permitted to persist as it may result in damage to the gasket surfaces.

When metal jacketed filled type gaskets are used with a tongue and groove joint without a nubbin, the gasket should be installed so that the tongue bears on the seamless side of the gasket jacket. When a nubbin is used, the nubbin should bear on the seamless side.

E-5.5 SPARE AND REPLACEMENT PARTS

The procurement of spare or replacement parts from the manufacturer will be facilitated if the correct name for the part, as shown in Section 1, Table N-2, of this manual is given, together with the serial number, type, size, and other information from the nameplate. Replacement parts should be purchased from the original manufacturer.

E-5.6 EXPANDED TUBE JOINTS

E-5.61 TUBE HOLE FINISH

Tube hole finish affects the mechanical strength and leak tightness of an expanded tube-to-tubesheet joint. In general:

- (1) A rough tube hole provides more mechanical strength than a smooth tube hole. This is influenced by a complex relationship of modulus of elasticity, yield strength, and hardness of the materials being used.
- (2) A smooth tube hole does not provide the mechanical strength that a rough tube hole does, but it can provide a pressure tight joint at a lower level of tube wall reduction.
- (3) Very light wall tubes require a smoother tube hole finish than heavier wall tubes.
- (4) Significant longitudinal scratches can provide leak paths through an expanded tube-to-tubesheet joint and should therefore be removed.

E-5.62 TUBE WALL REDUCTION

The optimum tube wall reduction for an expanded tube-to-tubesheet joint depends on a number of factors, some of which are:

- (1) Tube hole finish.
- (2) The presence or absence of the tube hole serrations (grooves).
- (3) The tube hole size and tolerance.
- (4) Tubesheet ligament width and its relation to tube diameter and thickness.

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- (5) Tube wall thickness.
- (6) Tube hardness and change in hardness during cold working.
- (7) Tube O.D. tolerance.
- (8) Type of expander used.
- (9) Type of torque control or final tube thickness control.
- (10) Function of tube joint, i.e., strength in resistance to pulling out, minimum cold work for corrosion purposes, freedom from leaks, ease of replacement, etc.
- (11) Length of expanded joint.
- (12) Compatibility of tube and tubesheet materials.

E-5.63 TUBE EXPANDING

A suitable tube expander should be used to tighten a leaking tube joint. Care should be taken to insure that tubes are not over expanded. Do not re-expand tubes which are not leaking. Do not expand tubes beyond 1/8" from the back face of the tubesheet.

E-5.64 TYPICAL COMPUTATION OF TUBE WALL REDUCTION

EXAMPLE OF DETERMINING THE EXPANDED TUBE INSIDE DIAMETER FOR A SPECIFIED AMOUNT OF WALL REDUCTION

GIVEN: TUBE DIAMETER = 0.750
 TUBE HOLE DIAMETER = 0.760
 TUBE WALL THICKNESS = 0.065
 SPECIFIED WALL REDUCTION = 5%

SOLUTION: WALL REDUCTION = $0.065 \times 0.05 = 0.00325$
 FINAL WALL THICKNESS = $0.065 - 0.00325 = 0.06175$
 FINAL EXPANDED TUBE I.D. = $0.760 - 2(0.06175) = 0.6365$

CAUTION: *THE USE OF NOMINAL DIMENSIONS, WITHOUT ATTENTION TO PERMISSIBLE TOLERANCES CAN LEAD TO INCORRECT CONCLUSIONS.*

EXPANDING PROCEDURES SHOULD BE BASED ON MEASURED DIMENSIONS, NOT NOMINAL DIMENSIONS.

E-5.65 PLUGGING TUBES IN TUBE BUNDLES

In U-tube heat exchangers, and other exchangers of special design, it may not be feasible to remove and replace defective tubes. Defective tubes may be plugged using commercially available tapered plugs with ferrules or tapered only plugs which may or may not be seal welded. Excessive tube plugging may result in reduced thermal performance, higher pressure drop, and/or mechanical damage.

E-5.7 WELDED TUBE JOINTS

Welding procedure and testing techniques for either seal welded or strength welded tube joints normally are by agreement between manufacturer and purchaser and therefore should be considered prior to repairs to welded tube joints or tube replacements.

E-5.8 END FLANGES AND BOLTING

The purpose of this section is to alert the users, and provide basic data for consideration in assembly and maintenance of bolted flanged joints. Controlling the torque applied in tightening threaded bolts is the most economical and commonly used method for the control of initial tension. There are several factors which affect the relationship between torque and tension of threaded bolts. A few of the factors are the type of lubricant and/or plating, if any, used on the threads and the materials from which the bolt and nut are manufactured.

In assembling gasketed joints it is essential that the gasket first be seated uniformly by initial tightening of studs using a staggered quarterly pattern. After the gasket is seated, torquing may be completed stepwise to the final desired value, following a staggered or sequential pattern.

The following may be used as references and aids when assembling bolted flanged joints:

1. Recommended assembly and maintenance of flanged joints as shown in the Crane Company Catalog #60, Engineering Data Section.
2. Torque Control, Sixth Edition, Sturtevant Company.
3. ASME Code, Appendix S.

E-5.9 SPECIAL HIGH PRESSURE CLOSURES

The instructions of the heat exchanger manufacturer should be followed when provided.

E-5.10 REPAIRS, ALTERATION, RERATING, AND REPLACEMENT PARTS.

In order to maintain the Code integrity of the vessel, any repairs by welding, subsequent heat treating, alterations, replacement pressure parts, or rerating must be done in accordance with the National Board Inspection Code.

Potentially toxic or harmful contaminants must be neutralized prior to repairs.

Following repair by welding, post weld heat treatment may be a requirement for preservation of resistance to corrosion.