

MANUFACTURER'S DATA BOOK

**MISSISSIPPI POWER
DEKALB, MISSISSIPPI**

KBR	
ACCEPTANCE FOR ENGINEERING USE	
THIS DOCUMENT IS:	
ACCEPTED	(Code 1) _____
ACCEPTED WITH COMMENTS	(Code 2) _____
NOT ACCEPTED	(Code 3) _____
NOT REVIEWED	(Code 4) <u> X </u>
ACCEPTANCE DOES NOT RELIEVE SUPPLIER FROM FURNISHING MATERIAL IN CONFORMANCE WITH ORDER. REFER TO SDR-1 FOR FULL DEFINITION OF ACCEPTANCE CONDITIONS.	
DISCIPLINE	EXCH BY MW DATE <u>24SEP12</u>

KEMPER COUNTY 1 – IGCC PROJECT

P.O. NO: 17496-0001

REQ. NO: MC202

ITEM NO: HX0042

**HYDROGEN SULFIDE STRIPPER
REBOILER**

1-23" x 168" NEN EXCHANGER

ENERGY JOB NO: X-7929



Energy Exchanger Company

Energy Industrial Park
1844 North Garnett Road/Tulsa, OK 74116-1612
(918) 437-3000/FAX (918) 437-7144

Southern Company Generation Kemper County
MM209937 0 Unit 1

ENERGY EXCHANGER COMPANY PO: MPC17496-0001
X-7929-MDB Rev: 0
IGCC - GASIFIER - MULTIPAGE - SHELL AND TUBE HEAT EXCHANGER - MAN

Mississippi Power
Kemper County 1 – IGCC Project
P.O. No: 17496-0001
Req. No: MC202
Item No: HX0042
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MANUFACTURER'S DATA BOOK

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7	Spare Parts List Installation, Operation & Maintenance Manual

SECTION 1

117 Cornerstone Ct.
Sapulpa, OK 74066Richard Duke –Owner
Office: 918-224-7472
Cell: 918-640-7911
Email: rmduke@cox.net**I.Q.S. Work Order 556
Inspection Shipping Release Notice 15**

Client: SOUTHERN CO. Purchase Order #: MPC-17645; MPC17496 &
MPC- 17497

EWO: MPC9117.T61401 Supplier: ENERGY EXCHANGERS

Shop Order # SO # 7929 Location: TULSA, OK

Project: KEMPER COUNTY IGCC Supplier Ref. #:
UNIT

Item #	Description	Qty Ordered	Qty Released	Balance
	23-168 NEN Exchanger TAG# HX0042--SO# 7929 AND FINAL DOCUMENTATION IS NOW COMPLETE	1	1	0

This does not relieve the Vendor of any responsibility to ensure that the material or equipment meets all contractual requirements. This notice is issued according to IQS' best knowledge, experience, codes and specifications available to us during this inspection. Neither IQS nor any member of its inspection force is to be held liable for the acceptance of or payment for released items, or the default or negligence on the part of the vendors' obligation to supply quality equipment and materials or breach the conditions of his contract with the purchaser. Actual acceptance of equipment/material is predicated on the purchaser's site inspection, operational integrity and the fulfillment of all contractual warranties connected thereto.

Date: 9/8/12

*Bill Bronaugh**IQS*

SECTION 2

Energy Exchanger Company



HEAT EXCHANGER SPECIFICATION SHEET

REQ. NO.: MC-202

P. No.: 17496-0001

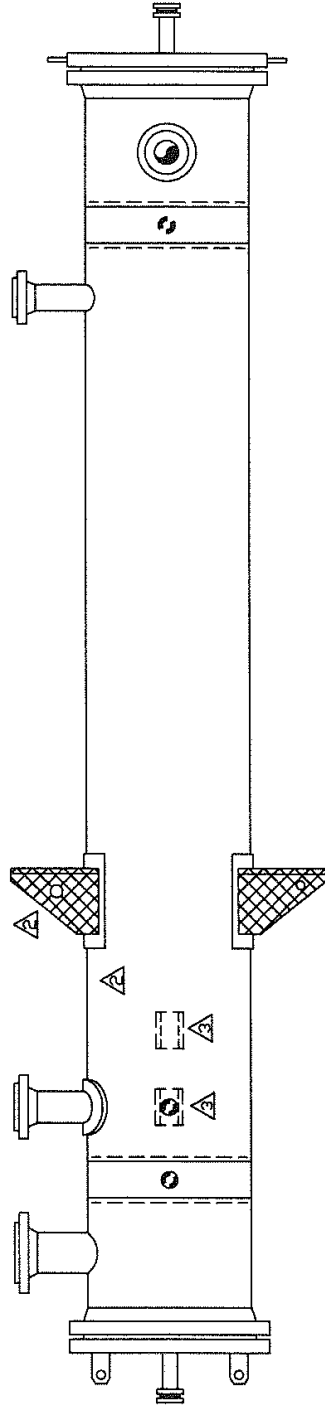
REV. 0 AS PURCHASED

REV	LN #		
1	1	Customer	SOUTHERN COMPANY
2	2	Job No.:	X-7929 "AS BUILT"
3	3	Address	
4	4	Rev.:	3
5	5	Date:	8/15/2012
6	6	Plant Location	KEMPER COUNTY, MS
7	7	File:	10-7309
8	8	Service of Unit	HYDROGEN SULFIDE STRIPPER REBOILER
9	9	Item:	HX0042
10	10	Size	23-168
11	11	Type	NEN
12	12	Vert	
13	13	Connected In	1
14	14	Parallel	1
15	15	Series	
16	16	Surf./Unit	☐ Gross / ☒ Eff
17	17	1241. ft ² /Shell/Unit	1
18	18	Surf./Shell	☐ Gross / ☒ Eff
19	19	1241. ft ²	
20	20	PERFORMANCE OF ONE UNIT	
21	21	Fluid Allocation	Shell Side
22	22	Fluid Name	IP STEAM
23	23	Fluid Quantity, Total	19654 X 1.1
24	24	Tube Side	SOUR WATER
25	25	Fluid Quantity, Total	209000 X 1.1
26	26	Vapor (In/Out)	16929 X 1.1
27	27	Liquid	209000 X 1.1
28	28	Steam	19654 X 1.1
29	29	Water	19654 X 1.1
30	30	Noncondensable	
31	31	Temperature (In/Out)	475. 419.4 332.7 346.
32	32	Density (Vapor/Liquid)	Lb/Ft ³ 45.99 0.4360 47.73
33	33	Viscosity	cP 0.137 0.0160 0.133
34	34	Molecular Weight	
35	35	Molecular Weight, Noncondensable	
36	36	Specific Heat	Btu/lb-F 1.1270 0.5000 1.1210
37	37	Thermal Conductivity	Btu/hr-ft-F 0.3010 0.0210 0.3210
38	38	Latent Heat	Btu/lb
39	39	Inlet Pressure	psia 310. 192.73 @ COLUMN BOTTOM
40	40	Velocity	ft/sec 15.68 7.99
41	41	Pressure Drop, Allow./Calc.	psi 5.00 / 0.77
42	42	Fouling Resistance (Min.)	Hr-Ft ² -F/Btu 0.0005 0.002
43	43	Heat Exchanged	16392000 X 1.1 Btu/hr; M.T.D. (Corrected) 80.79 °F
44	44	Transfer Rate, Service	179.85 Btu/hr-ft ² -F; Clean 461.55 Btu/hr-ft ² -F
45	45	CONSTRUCTION OF ONE SHELL	
46	46	Design/Test Pressure	psig 455 /FV/595 205/270 (4)
47	47	Design Temperature (Max/Min)	°F 515/10 390/10
48	48	No. Passes Per Shell	1 1
49	49	Corrosion Allowance	Inch 0.125 0.1875
50	50	Connections	In 1 - 6" 300RF 1 - 6" 300RF
51	51	Size & Rating	Out 1 - 3" 300RF 1 - 10" 300RF
52	52	Balance Line	1 - 3/4" 300RF
53	53	Tube No.	462
54	54	OD	0.75
55	55	In; Thk.	☒ Min ☐ Avg 0.083
56	56	In; Length	14. ft; Pitch 0.9375 In 30°
57	57	Tube Type	BARE
58	58	Material	SA-179
59	59	Shell	SA-106-B ☒ ID 22.876 In ;Shell Cover N/A
60	60	Channel or Bonnet	SA-333-6 PWHT Channel Cover SA-516-70N PWHT
61	61	Tube-sheet-Stationary	SA-516-70N PWHT Tube-sheet - Floating N/A
62	62	Floating Head Cover	Impingement Protection YES - CS PLATE ON BUNDLE
63	63	Baffles-Cross	9 - SA-36 Type H SEG % Cut (☒ Dia.) 27.9 Spacing c/c 15.625 Inlet In
64	64	Baffles-Long	Seal Type
65	65	Supports - Tube	U-Bend Type
66	66	Bypass Seal Arrangement	Tube-Tubesheet Joint TWO GROOVES & EXPANDED
67	67	Expansion Joint	NONE Type
68	68	pv ² - Inlet Nozzle	Bundle Entrance Bundle Exit
69	69	Gaskets - Shell Side	Tube Side 321 S.S. DJ GRAPH. FILLED
70	70	- Floating Head	
71	71	Code Requirements	ASME SECTION VIII, DIV. 1: STAMP: YES TEMA Class R
72	72	Weight/Shell	8,700# Filled With Water 11,700# Bundle N/A
73	73	REMARKS	
74	74	(1) DELETED.	
75	75	(2) UNIT IS A VERTICAL THERMOSYPHON REBOILER. INLET PIPING EQUIVALENT LENGTH = 78 FT. OUTLET PIPING EQUIVALENT LENGTH = 30 FT. STATIC HEAD IS 14 FT.	
76	76	(3) TUBE SIDE IS IN WET H2S SERVICE. NACE COMPLIANCE REQUIRED.	
77	77	(4) TUBE SIDE SUBJECT TO STEAMOUT AT 250°F. AND 15 PSIG. STEAMOUT ON TUBE SIDE ALONE IS NOT ALLOWED FOR THIS UNIT. STEAM OUT ON SHELL SIDE FIRST, WHILE SHELL IS STILL HOT, STEAM OUT TUBE SIDE.	

X:\eX-7929.xls

Miles Swall 9/13/12






 COATING DATA SHEET - SYSTEM 5 APPLIES.
 ABRASIVE BLAST PER SSPC-SP10 WITH AN ANCHOR PROFILE OF 1 TO 2 MILS.
 PAINT WITH AMERON, PPG AMERCOTE, DIMETCOTE 9 or 9HS (3.0-4.0 MILS DFT)


 COATING DATA SHEET - SYSTEM 3 APPLIES.
 ABRASIVE BLAST PER SSPC-SP10 WITH AN ANCHOR PROFILE OF 1 TO 2 MILS.
 PAINT WITH AMERON, PPG AMERCOTE, DIMETCOTE 9 or 9HS (2.0-3.0 MILS DFT)
 FINISH WITH AMERON, PPG AMERCOTE, PSX 892HS (1.0-2.0 MILS DFT) (COLOR: ALUMINUM)

CERTIFIED AS-BUILT

CUSTOMER: MISSISSIPPI POWER
 / SOUTHERN COMPANY SERVICES
 P.I. NO. 17496-0001 REQ NO. MC202
 SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
 ITEM NO. HX0042

PAINT MAP

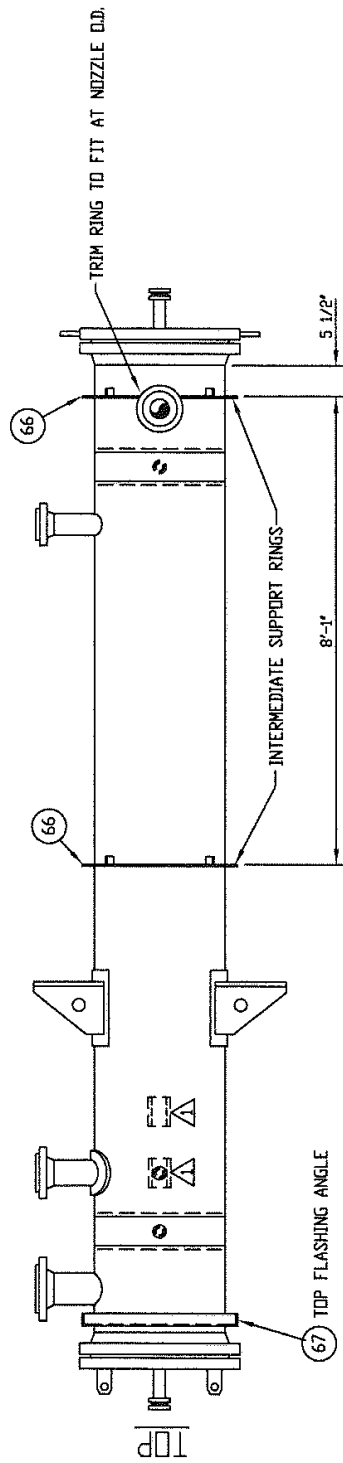
- NOTES:
1. A DAILY INSPECTION REPORT SHALL BE MAINTAINED.
 2. PAINT SPECIFICATION V42-ITS AND APPLICABLE COATING DATA SHEET SHALL BE FURNISHED TO THE PAINTER.

NO.	DATE	REVISIONS	SM	CKD. BY	DATE	DWN. BY	DATE
1	8-24-12	REV'D PER EEC	DM/ID	DMORRISON			
2	8-10-11	REV'D PER CUST.	WE/MD				
3	5/12/11	REVISED PER CUSTOMER	MD/SG		3-14-11		
				REV			
				3			

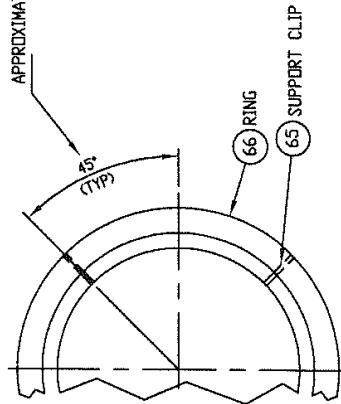


Energy Exchanger Co.
 2041 W. Kentucky Ave. - Tulsa, Okla. 74106

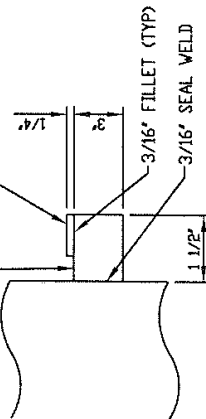
B.W.G. NO. X-7929-AA
 REV 3



APPROXIMATELY 9 7/16" O.D. WRAP

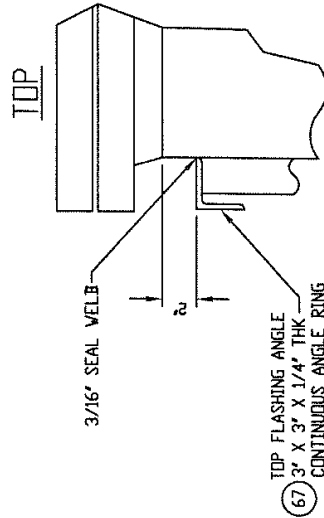


SUPPORT CLIP **65** RING 27" O.D. X 25" I.D.



INTERMEDIATE SUPPORT RINGS

CERTIFIED AS-BUILT

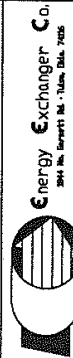


TOP FLASHING ANGLE

CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.O. NO. 17496-0001
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
ITEM NO. HX0042

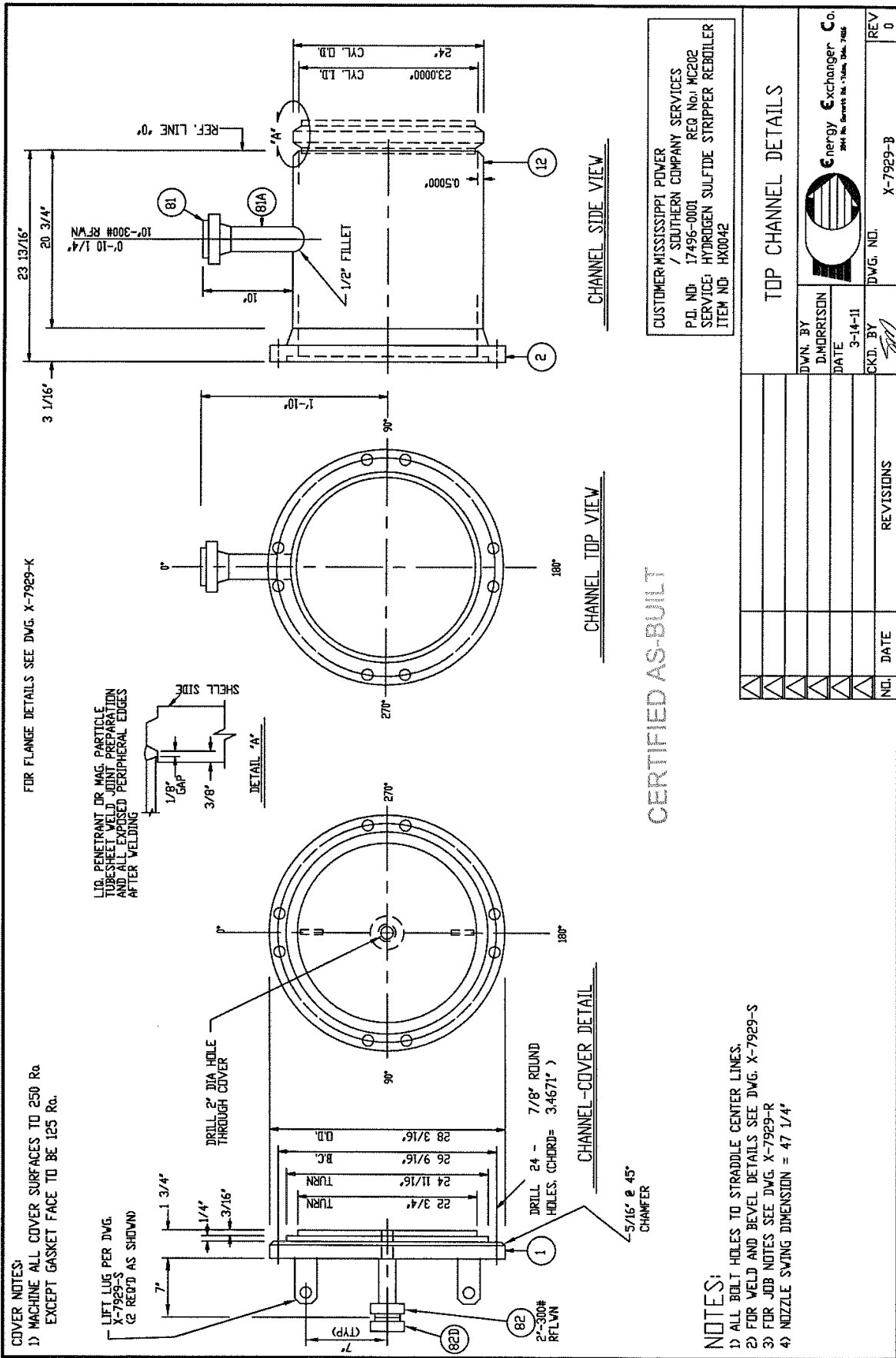
INSULATION SUPPORT DETAILS

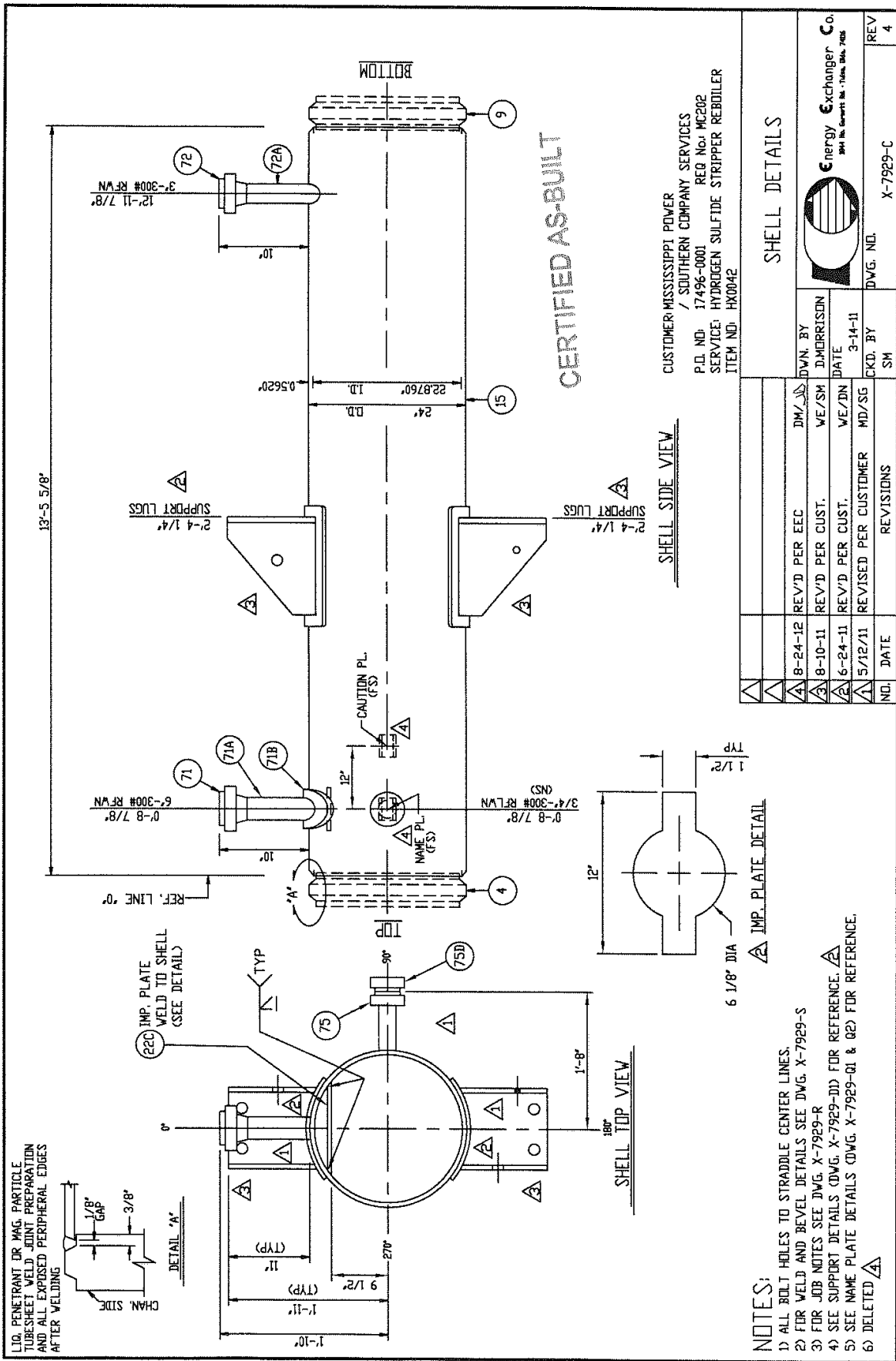
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Energy Exchanger Co.
2004 W. Highway 111 - Tulsa, Okla. 74128

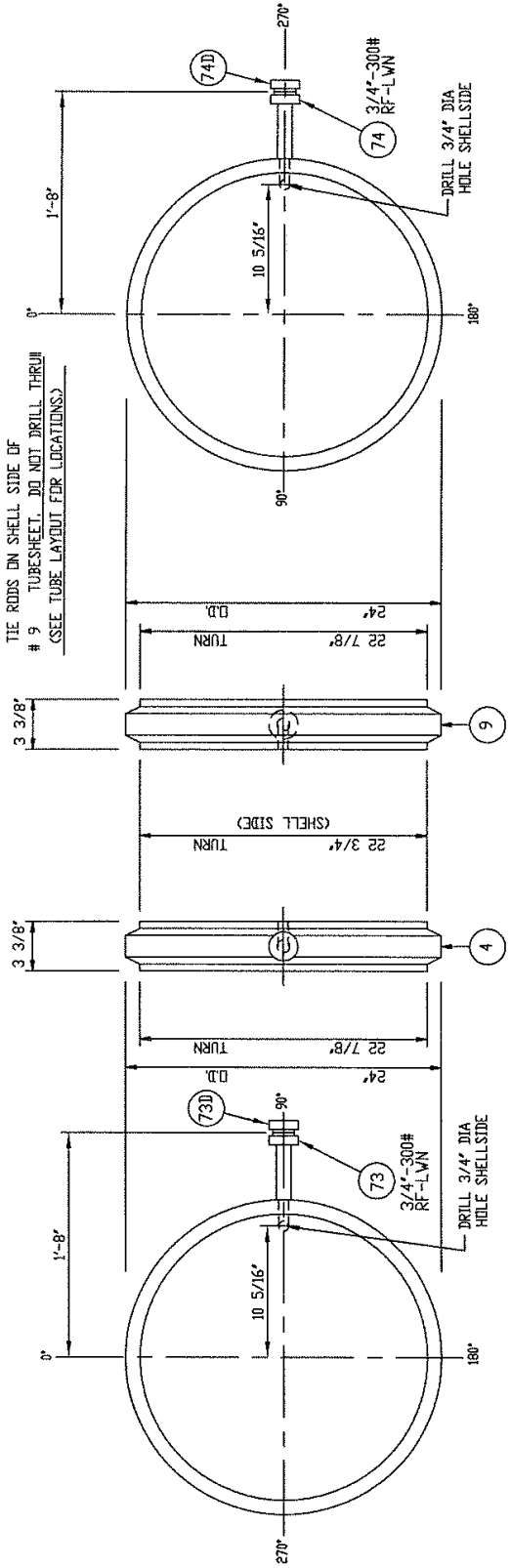
X-7929-AB





Kemper County MM209937

DRILL & TAP 3/4" DEEP FOR 6- 1/2" DIA.
TIE RODS ON SHELL SIDE OF
9 TUBESHEET. DO NOT DRILL THRU!
(SEE TUBE LAYOUT FOR LOCATIONS)



TOP CHANNEL TUBESHEET

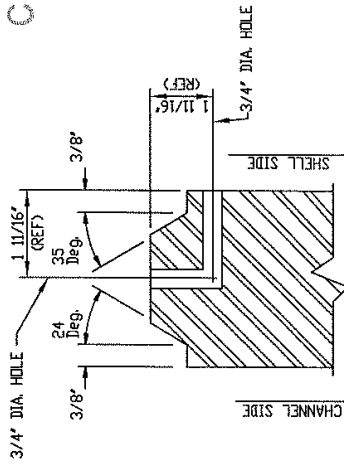
BOTTOM CHANNEL TUBESHEET

CERTIFIED AS-BUILT

NOTES:

- 1) MACHINE ALL SURFACES TO 250 Ra EXCEPT GASKET FACE TO BE 125 Ra.
- 2) REAM & RING-GROOVE TUBE HOLES AFTER POST-WELD HEAT TREAT.
- 3) FOR JOB NOTES SEE DWG. X-7929-R
- 4) SEE DWG. X-7929-S FOR WELD AND BEVEL DETAILS.

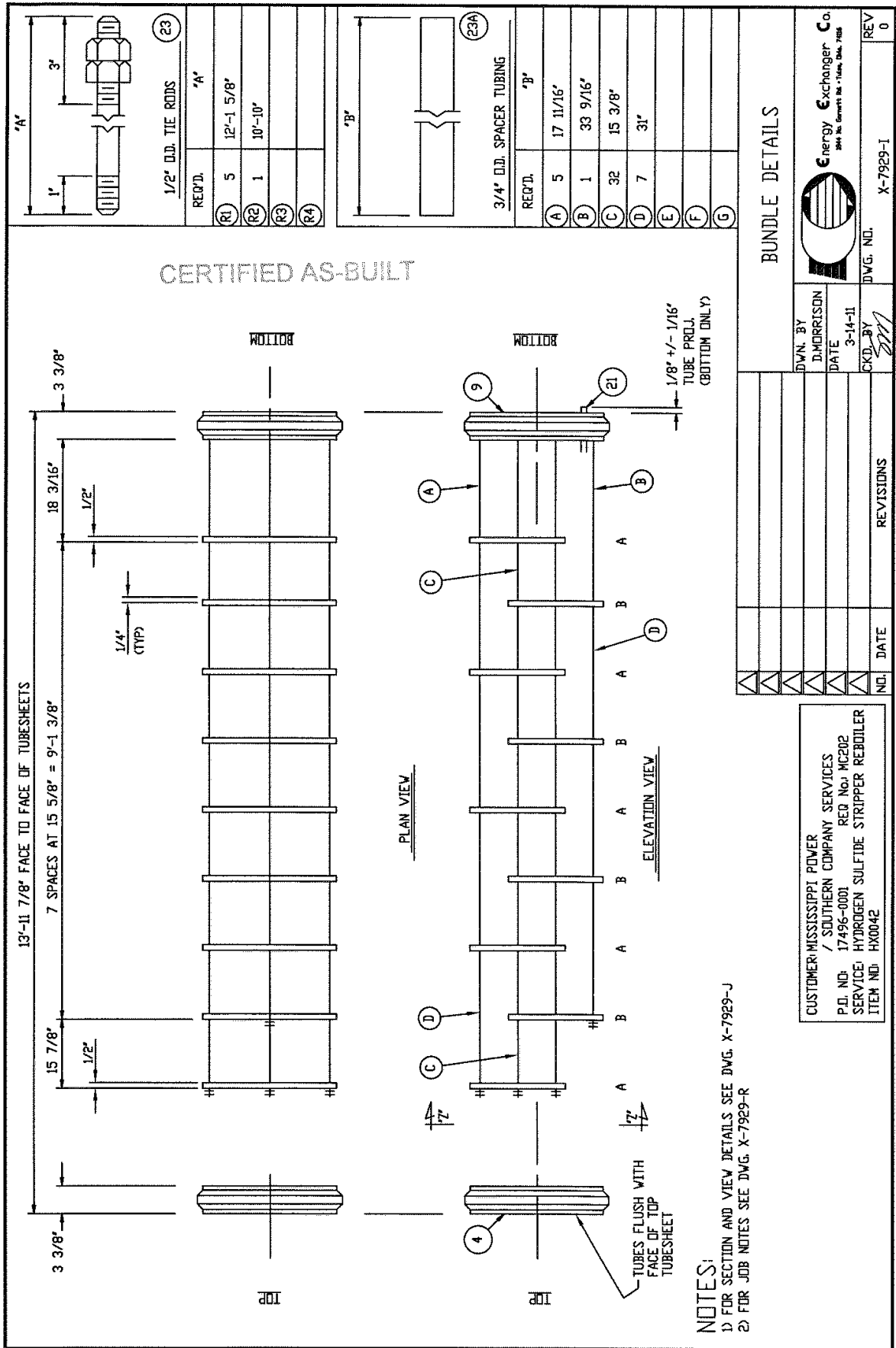
CUSTOMER: MISSISSIPPI POWER / SOUTHERN COMPANY SERVICES
P.O. NO. 17496-0001 REQ NO. MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBUILER
ITEM NO: HX0042



TUBESHEET DETAIL
(TYP. BOTH TUBESHEETS)

TUBESHEET DETAILS

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CERTIFIED AS-BUILT

NOTES:

- 1) FOR SECTION AND VIEW DETAILS SEE DWG. X-7929-J
- 2) FOR JOB NOTES SEE DWG. X-7929-R

BUNDLE DETAILS

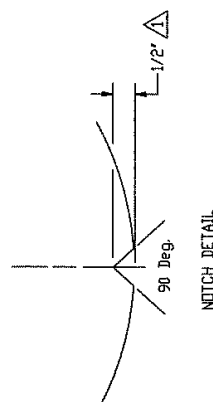
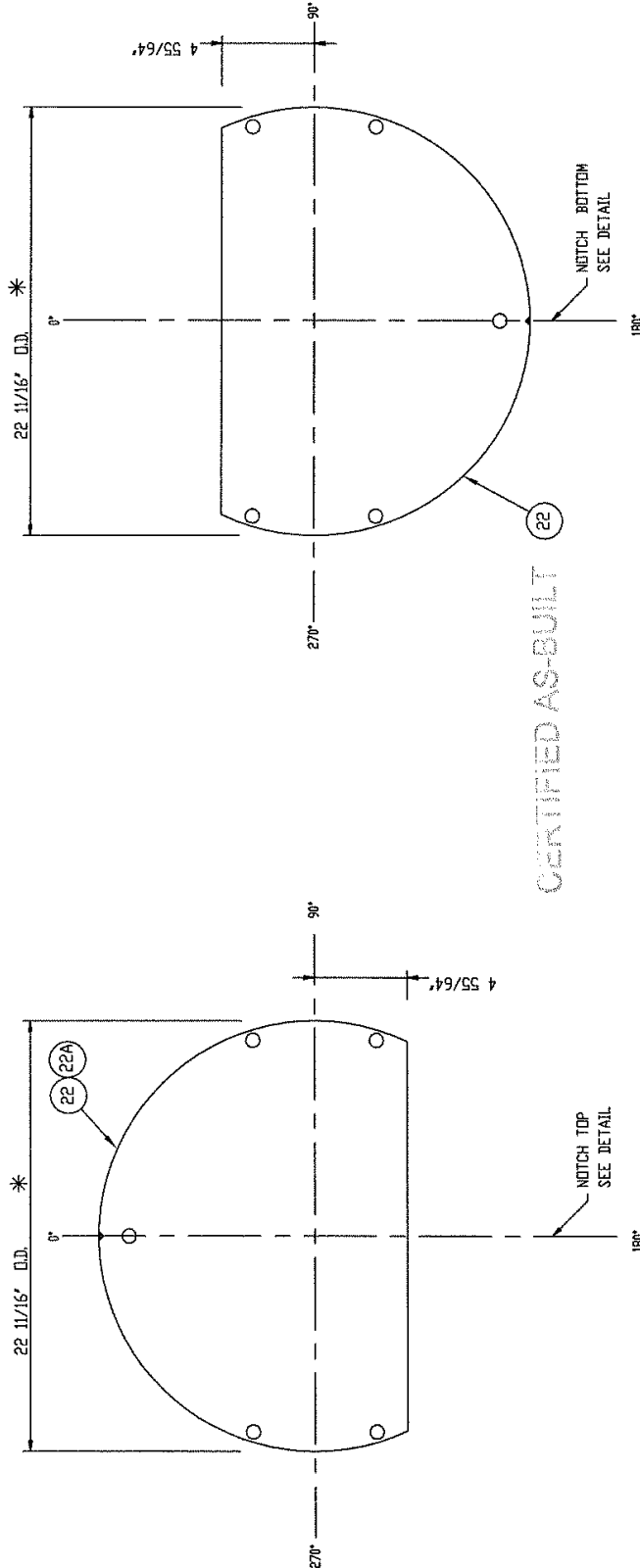
CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.O. NO: 17496-0001 REQ NO: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBUILDER
ITEM NO: HX0042

DWN. BY
D. MORRISON
DATE
3-14-11
CKD. BY
[Signature]

REV
0

DWG. NO.
X-7929-I

Energy Exchange Co.
304 N. Garrett St. • Tulsa, Okla. 74103



CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.O. NO. 17496-0001 REQ NO. MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBUILER
ITEM NO. HX0042

NOTES:

- 1) FOR JOB NOTES SEE DWG. X-7929-R
- 2) DRILL TUBE HOLES IN BAFFLES 25/32" DIA. & DEBURR.
- 3) DRILL TIE ROD HOLES 9/16" DIA.
- SEE TUBE LAYOUT (DWG. X-7929-H) FOR LOCATION.
- 4) RADIUS ALL CORNERS ON BAFFLE CUTS.
- * 5) CHECK SHELL I.D. BEFORE MACHINING BAFFLE O.D.
- 6) MACHINE O.D. OF BAFFLES TO 250 Ra.
- 7) SEE TUBE LAYOUT (DWG. X-7929-H) FOR BAFFLE CUT LOCATIONS.

BAFFLE DETAILS			
	DWN. BY	D. MORRISON	
	DATE	3-14-11	
	CKD. BY	SM	
	REVISED PER CUSTOMER	MD/SG	D.W.G. NO. X-7929-J
	DATE	5/12/11	
	NO.	REVISIONS	REV 1

CERTIFIED AS-BUILT

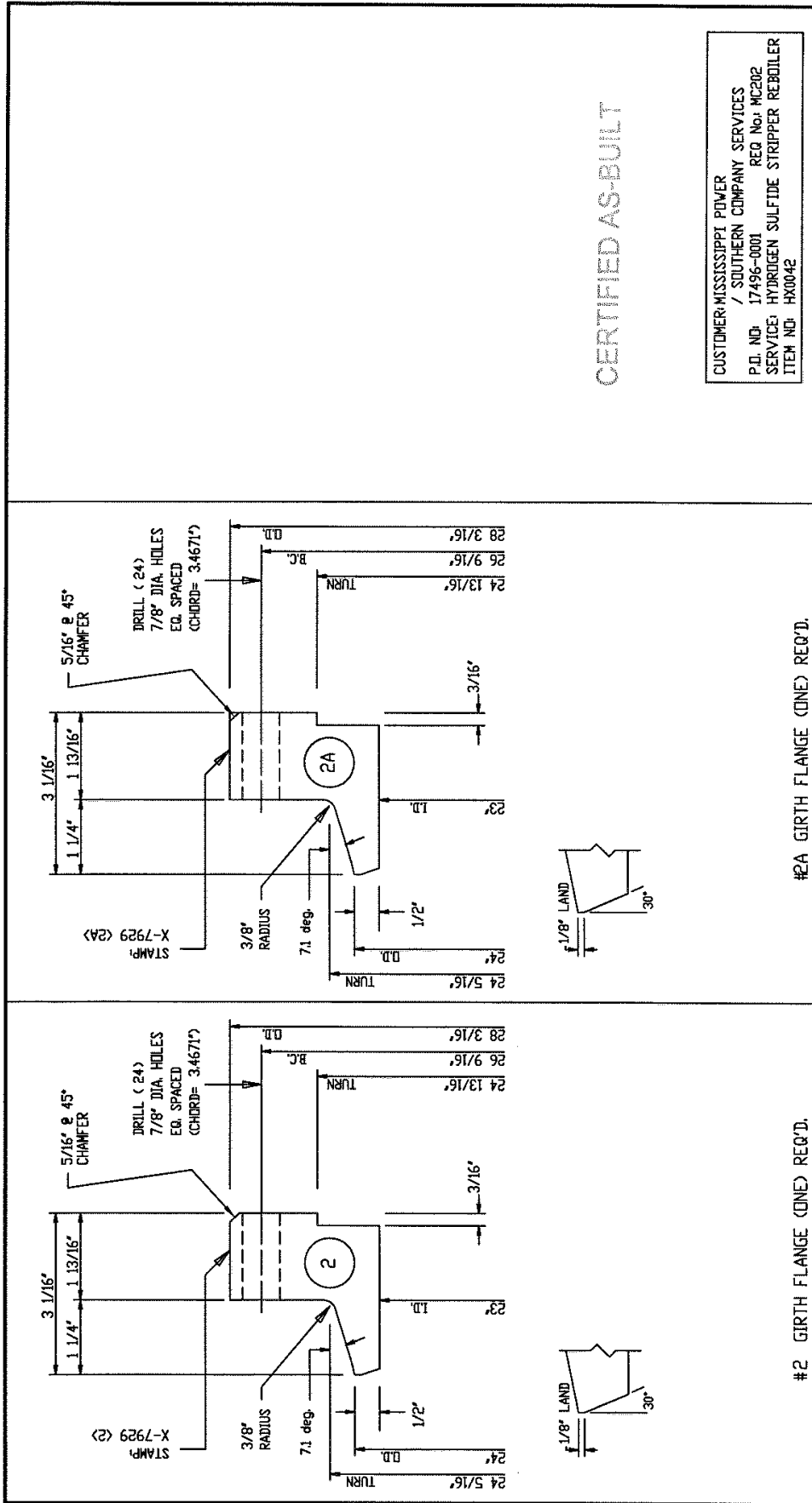
CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.D. NO: 17496-0001 REQ NO: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
ITEM NO: HX0042

GIRTH FLANGES

APPROVED BY: D. MORRISON
DATE: 3-14-11
DWG. NO: X-7929-K
REV: 0

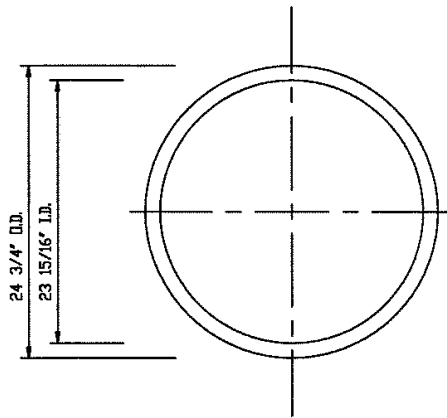


REVISIONS

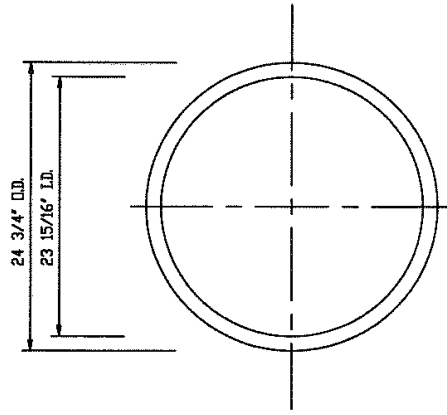


NOTES:

- 1) SEE DWG. X-7929-R FOR JOB NOTES.
- 2) MACHINE ALL SURFACES TO 250 Ra EXCEPT GASKET SURFACES MACHINE TO 125 - 250 Ra.



(1)-REQUIRED.
TOP CHANNEL COVER GASKET
1/8" THK 321 STN STL DJ FLEXIBLE GRAPHITE FILLED



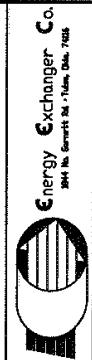
(1)-REQUIRED.
BOTTOM CHANNEL COVER GASKET
1/8" THK 321 STN STL DJ FLEXIBLE GRAPHITE FILLED

CERTIFIED AS-BUILT

- NOTES:
- 1) QUANTITIES SHOWN FOR ONE SET
(2)-SET REQUIRED TOTAL.
ONE SET OF GASKETS TO BE USED IN EXCH. AND
(1) SET TO BE BOXED, MARKED WITH
P.D. & ITEM NO. & SHIPPED WITH EXCHANGER.
 - 2) ALL JACKETED GASKETS TO HAVE
LAP OF GASKET NEAR SIDE.

CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.D. NO: 17496-0001 REQ NO: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
ITEM NO: HX0042

GASKET DETAILS			
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DWG. NO. X-7929-0

NAME PLATE BRACKET DETAIL

ONE - REQ'D.

FBI

CLIP CORNERS & SEAL WELD
TO NAME PLATE BRACKET

DESIGN PRESSURE: SHELL=464 P.S.I., TUBE=214 P.S.I.,

NAME PLATE DETAIL

ONE - REQ'D.

NAME PLATE DETAILS

DWN, BY

D. MORRI:

DATE 3-14-

CKD. BY

1453

REVISIONS

NO.	DATE
-----	------

1

Energy Exchanger Co.
3944 No. Everett Rd. • Tulsa, Okla. 74215

REV	0
Q1	

Kemper County MM209937

CAUTION

PRESSURES AND TEMPERATURES SHOWN ON THE NAMEPLATE OF THIS HEAT EXCHANGER RELATE ONLY TO PRESSURE RETAINING CAPABILITY. CERTAIN COMPONENTS HAVE BEEN DESIGNED FOR SPECIFIC OPERATING CONDITIONS. DEVIATION FROM THESE CONDITIONS OR A CHANGE IN SERVICE REQUIRE A MECHANICAL DESIGN CHECK BE PERFORMED TO AVOID POSSIBLE DAMAGE TO THE HEAT EXCHANGER OR DANGER TO PERSONNEL.

CAUTION PLATE DETAIL

ONE - REQ'D.

WARNING

FIXED TUBESHEET HEAT EXCHANGER
INTRODUCE SHELL SIDE FIRST BEFORE INTRODUCING TUBE SIDE
STEAMOUT TUBE SIDE ALONE IS NOT ALLOWED FOR THIS UNIT.
STEAMOUT SHELL SIDE FIRST, WHILE SHELL SIDE IS HOT
STEAMOUT TUBE SIDE.

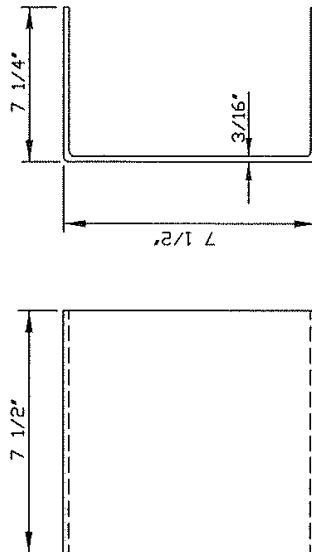
WARNING PLATE DETAIL

ONE - REQ'D.

CLIP CORNERS & SEAL WELD
TO NAME PLATE BRACKET



USE BACK SIDE OF EEC NAME PLATE.
SPLIT IN HALF TO MAKE BOTH NAME PLATES
(FIT BOTH ON SAME BRACKET)



CAUTION PLATE BRACKET DETAIL

ONE - REQ'D.

CERTIFIED AS-BUILT

CLIP CORNERS & SEAL WELD
TO NAME PLATE BRACKET

CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.O. NO: 17496-0001 REG. NO: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
ITEM NO: HX0042

NAME PLATE DETAILS			
NO.	DATE	REV. PER CUSTOMER	MD/SG
1	5/12/11	3-14-11	
DWN. BY		DATE	CKD. BY
D. MORRISON		3-14-11	SM
REV. PER CUSTOMER		MD/SG	REV. NO.
3-14-11			X-7929-Q2
DATE		REV. NO.	REV. 1



Energy Exchange Co.
344 W. Garrett Rd. - Tulsa, Okla. 74106

1. ALL BOLT HOLES TO STRADDLE NATURAL CENTER LINES.
2. NUZZLES SHALL NOT PROTRUDE BEYOND THE INSIDE SURFACE OF THE CHANNEL OR SHELL.

3. CHECK SHELL FOR OUT-OF-ROUNDNESS. USE TWO DISCS EQUAL THE BAFFLE DIA. MOUNTED ON A SHAFT 18" APART.
4. FLANGE FLATNESS TOLERANCE AND SURFACE FINISH SHALL BE MEASURED AFTER THE FLANGE HAS BEEN WELDED & PWHT.

5. CHANNEL COVER AND BODY FLANGE PERIPHERAL GASKET SURFACES SHALL HAVE A FLATNESS TOLERANCE OF $1/32"$ AND MEASURED WITH A DIAL GAUGE.

6, DELETED

7. ALL FLAME-CUT BEVEL WELDS SHALL BE GRIND TO BRIGHT METAL PRIOR TO WELDING.
8. MATCH MARKS REQUIRED ON CHANNEL COVER AND ALL TEMA FLAGS, WITH A SCRIBED LINE 1/8" DEEP AT THE 90° AND 270° POSITIONS.
9. THE FOLLOWING PARTS SHALL BE STAMPED WITH MANUFACTURER'S SERIAL NUMBER & ITEM NUMBER: CHANNEL FLANGE, CHANNEL

10. TUBESHEET AND TUBE HOLES SHALL BE CLEANED AND DEGREASED IMMEDIATELY PRIOR TO ASSEMBLY WITH A CHLORIDE FREE SOLVENT.

11. PRIOR TO ASSEMBLY, ALL INTERNALS SHALL BE CLEAN AND FREE OF LOOSE SLAG, SCALE OR OTHER DEBRIS.

12. ANY WELD DEFECTS SHALL HAVE THE DEFECT REMOVED AND, CONFIRMED IF ORIGINALLY RT, THEN RT REPAIR AND ALSO MT.

13. WELD JOINT SURFACES AND A MINIMUM OF 1" OF THE INTERNAL AND EXTERNAL SURFACES OF THE ADJACENT BASE METAL SHALL BE CLEAN AND FREE FROM PAINT, OIL, DIRT, SCALE, OXIDES, AND OTHER FOREIGN MATERIAL DETRIMENTAL TO THE INTEGRITY OF THE WELD.

1. DELETED
2. AT ALL LIFTING LUG WELDS
3. DELETED
4. TEMPORARY ATTACHMENT WELDS SHALL BE REMOVED, THE AREAS DRESSED, AND EXAMINED.

7. WT ALL CHANNEL SIDE WELDS AFTER PWHT.

TOP CHANNEL & BTM CHANNEL)

A) HEAT FROM 800 deg.F TO 1150 deg.F. AT 400 deg.F PER HOUR.
B) HOLD AT 1150 deg.F ± 25 deg. F. FOR 100 -1 HOUR CHD.
C) COOL FROM 1150 deg.F TO 800 deg.F AT 500 deg.F PER HOUR.
D) FURTHER COOLING MAY BE IN STILL AIR.

ALL MACHINED CONTACT SURFACES AND THREADED CONNECTIONS SHALL BE SUITABLY PROTECTED TO PREVENT SCALING OR LOSS OF FINISH DURING PWHT.

1. TUBESIDE: SPOT RADIOGRAPHED.
AS A MINIMUM, SPOT RADIOGRAPH SHALL BE AS FOLLOWS:
A) ONE SPOT AT EACH CATEGORY A AND B WELDED JOINT.
(NOZZLE WELDS EXEMPT)
B) AT EACH START AND STOP OF WELDS MADE BY AUTOMATIC
SUBMERGED-ARC PROCESS.

- C) SPOTS SHALL BE AT LEAST 10" LG. OR FULL LENGTH IF WELD IS LESS THAN 10".
- D) WELD POSIBILITY LIMITS FOR SPOT RADIOGRAPHS SHALL BE SAME AS LIMITS FOR FULL RADIOGRAPHED JOINTS.
- E) AT EACH WELD PROCEDURE, WELDER/OPERATOR & SHUTTERMAN SHALL BE FINE GRAIN HD. HIGH CONTRAST FILM DETERMINED BY SPECIMEN OR DENSITOMETER.

4. DELETED

0313154 5

1. WELD HARDNESS TESTING SHALL BE IN ACCORDANCE WITH THE FOLLOWING

- A) THE WELD METAL & HAZ OF PRESSURE-RETAINING WELDS IN COMPONENTS
- B) BRINELL HARDNESS LIMITS SHALL BE 200 MAX.
- C) HARDNESS SHALL BE DETERMINED USING A 10mm DIA BALL IN LESS THAN ONE HOUR.

D) DELETED

E) ONE READING IN THE WELD METAL OF EACH CIRC. SEAM AND ONE READING IN EACH HEAT AFFECTED ZONE.

F) ONE READING IN THE WELD METAL OF NOZZLE TO EXCHANGER WELDS AND ONE READING IN EACH HEAT AFFECTED ZONE IN 50% OF THE NOZZLES.

IF MORE THAN ONE WELDING PROCEDURE IS USED TO FABRICATE LONG OR CURC. WELDS, HARDNESS READINGS SHALL BE MADE IF WELDS DEPOSITED BY EACH PROCEDURE. HD TESTING SHALL BE DONE ON ACCESSIBLE INSIDE SURFACES.

AD
COURTESY OF THE NATIONAL ARCHIVES

CUSTOMER: MISSISSIPPI POWER

CUSTOMER PROSPECTUS / SOUTHERN COMPANY SERVICES

P.П. НП: 17496-0001 RFD No: MC202

SERVICE! HYDROGEN SULFIDE STRIPPER REBUILER

1. HOLD FOR AT LEAST ONE HOUR. A HYDROTEST CHART IS REQ'D.
2. MINIMUM TEST WATER TEMP. = 50 deg F.
3. PAINT OR OTHER COATINGS SHALL NOT BE APPLIED OVER WELDS BEFORE FINAL HYDROTEST.
4. LEAK TEST WELDS PAINT WITH 15 PSI AIR AND SOAP SOLUTION. INSPECTING WELDS WITH THE INSIDE AND OUTSIDE

5. LEAKAGE AT MAIN FLANGES REQUIRES CUSTOMER NOTIFICATION IN WRITING. ALL CORRECTIVE ACTION SHALL BE SUSPENDED UNTIL APPROVAL IS RECEIVED BY THE CUSTOMER. THE REASON FOR THE LEAK SHALL BE DETERMINED, DOCUMENTED BY AN NCR AND REPAIRED TO THE SATISFACTION OF THE CUSTOMER.
6. IMMEDIATELY AFTER HYDRO, DRAIN AND DRY BY BLOWING WARM AIR NOT TO EXCEED 120°F.

1. PAINT ON THE SIDE OF THE EXCHANGER AND EACH CRATED ITEM NO., P.L. NO. AND GROSS WEIGHT. IDENTIFY SPARE PARTS AS "SPARES" AND ITEM NUMBER.
2. CRATS SHALL HAVE A MINIMUM OF 2 STEEL STRAPS ON APPROX. 24" CENTERS. A PACKING LIST SHALL BE PLACED INSIDE AND ATTACHED TO THE OUTSIDE.

3. ALL NOZZLE GASKET SURFACES SHALL BE COATED WITH RUST VETO 342 PRIOR TO INSTALLING THE GASKETS AND SHALL BE PROTECTED BY METAL COVERS, PROVIDE A 1/4" N.P.T. WITH PLUG IN BOTTOM NOZZLE COVERS FOR DRAINING PURPOSES.

4. EXPOSED THREADS OF BOLTS SHALL BE PROTECTED WITH RUST VETO 342 RUST PREVENTATIVE TO PREVENT CORROSION DURING TESTING, SHIPPING, AND STORAGE. TAPPED HOLES SHALL BE PLUGGED WITH GREASE.

5. ATTACH A LARGE WEATHERPROOF TAG "CAUTION THIS EQUIPMENT HAS BEEN TREATED WITH A RUST PREVENTATIVE AND NO PARTS SHOULD BE DISTURBED UNTIL READY TO BE PLACED IN SERVICE. IF PROTECTIVE FILM IS BROKEN, IT SHALL BE RESTORED FOR CONTINUED STORAGE."

5. MARK C-D-G (DRY) WITH A 3" CIRCLE ON HORIZ. & VERTICLE CENTERLINES.

7) MARK ON TWO SIDES OF EXCHANGER 180° APART IN 3" LETTERS "NO WELDING OR BURNING PERMITTED ON THIS EXCHANGER"

3) THE NORTH ORIENTATION SHALL BE CENTER PUNCHED AND MARKED IN A CONTRASTING COLOR ON THE OUTSIDE OF EXCHANGER, LOCATED APPROX. 12 INCHES ABOVE SUPPORT BASE PLATE. THE MARK SHALL BE A 6 INCH LONG LINE AND NOTED AS "N".

3. NITROGEN PURGE REQUIRED FOR SHIPPING AND STORAGE. PAINT ON BOTH SIDES OF EXCHANGER WITH 4" HIGH LETTERS, "WARNING: NITROGEN PURGED, DANGER OF SUFFICATION UPON OPENING."

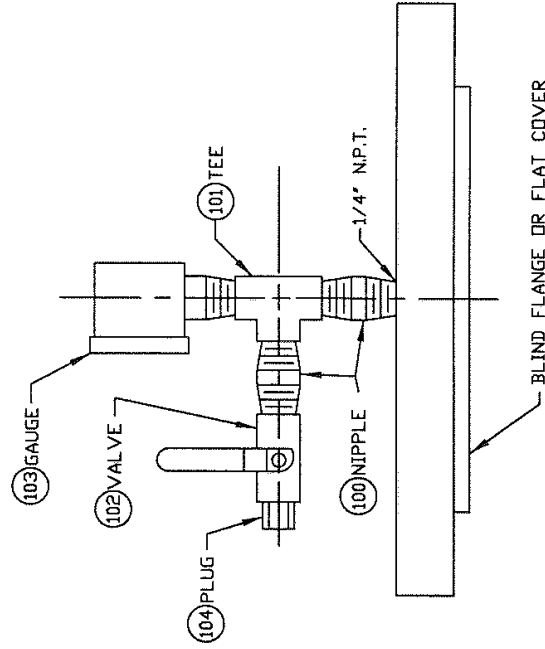
OWN. BY	D. MORRISON		Energy Exchanger Co. 304 So Everett Rd., Tulsa, Okla. 74105	REV	2
DATE	3/14/11				
CKD. BY		DWG. NO.	X-7929-R		
SM					

CUSTOMER: MISSISSIPPI POWER
/ SOUTHERN COMPANY SERVICES
P.D. NO: 17496-0001 REQ NO: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER

LOCATE LONG SEAM 15" ABOVE CL UNLESS OTHERWISE SPECIFIED. ALL SEAMS BACK GOUGED & BACK WELDED CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ LONG OR CIRC WELD BEVEL (CYL TO FLG, OR CYL) LOCATE LONG SEAM 15" ABOVE CL UNLESS OTHERWISE SPECIFIED. ALL SEAMS BACK GOUGED & BACK WELDED CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ LONG OR CIRC WELD BEVEL (CYL TO FLG, OR CYL) LOCATE LONG SEAM 15" ABOVE CL UNLESS OTHERWISE SPECIFIED. ALL SEAMS BACK GOUGED & BACK WELDED CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ LONG OR CIRC WELD BEVEL (CYL TO FLG, OR CYL)	BACK GOUGE AND BACK WELD CHANNEL WELD PROC: M-79-H, A-62-H SHELL WELD PROC: M-5 WELD-NECK FLANGE INSTALLATION BACK GOUGE AND BACK WELD CHANNEL WELD PROC: M-76-H SHELL WELD PROC: M-1 NOZZLE INSTALLATION WITHOUT PAD (1) 1/4" NPT IN EACH WELD PAD OR SEC. OF WELD PAD. PLUG W/ STIFF GREASE. UNLESS DETAILS SPECIFY OTHERWISE. BACK GOUGE AND BACK WELD CHANNEL WELD PROC: M-1 SHELL WELD PROC: M-1 NOZZLE INSTALLATION WITH PAD WELD THE FOLLOWING ITEMS WITH PROC'S: SUPPORT PAD TO SHELL, M-1 NAME PL. BRKT. TO SHELL, M-1 IMP. PL. TO SHELL, M-1	1/4" (MIN.) AT FRONT CHANNEL PASS-PLATE WELDS TO BE FULL PENETRATION FOR 2" (MIN) FROM FACE OF GASKET SURFACE. WELD PROCEDURE: _____ PASS PLATE WELDS 1 1/4" ROUND CORNERS 1/2" 62 12 12 2" Ø HOLE 3 1/2" 6 1/2" 3/4" 4 12 12 1/4" SEAL WELD WITH 1/4" FILLET WELD PROCEDURE: M-75-H LIFTING LUG DETAIL SHELLSIDE WELDS THAT MAY INTERFERE WITH BUNDLE INSTALLATION OR REMOVAL SHALL BE GROUND FLUSH. 1/4" MINIMUM FILLET WELDS FOR ALL NON PRESSURE ATTACHMENTS UNLESS NOTED OTHERWISE * ALL FILLETS WELDS TO BE 3/8" UNLESS NOTED OTHERWISE	BACK GOUGE AND BACK WELD CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ AUXILIARY NOZZLE INSTALLATION (TO PIPE, CYLINDER OR HEAD) CHANNEL WELD PROC: _____ SHELL WELD PROC: _____ 30° 30° 54° 65° 30° 1/8" FILLET WHERE POSSIBLE TUBESIDE CHANNEL WELD PROC: A-62-H SHELL WELD PROC: M-6 CIRCUMFERENTIAL SEAM (TUBESHEET TO CYL) CHANNEL WELD PROC: M-76-H SHELL WELD PROC: _____ SADDLE-ON NOZZLE INSTALLATION CERTIFIED AS-BUILT CUSTOMER: MISSISSIPPI POWER / SOUTHERN COMPANY SERVICES P.O. NO: 17496-0001 REQ NO: MC202 SERVICE: HYDROGEN SULFIDE STRIPPER REBUILDER ITEM NO: HX0042
WELD & BEVEL DETAILS DOWN BY DM DATE 3/14/11 REV BY REV 0 DWG. NO. X-7929-S Energy Exchanger Co. 1944 No. Garrett Rd. • Tulsa, Okla 74126			

NOTES:

- 1) BOTH SIDES SHALL BE THOROUGHLY DRAINED AND DRIED AFTER HYDROTEST. REMOVE MOISTURE BY BLOWING HOT AIR (150°F., MAXIMUM).
- 2) A) SEAL ALL EXTERNAL OPENINGS WITH A 1/8" THK. RUBBER-GASKETED BLIND FLANGE OR FLAT COVER.
B) INSTALL A CONNECTION WITH A VALVE AND PRESSURE GAUGE.
C) PRESSURE TO 15 PSIG AND DEPRESSURIZE TWICE; REPRESSURE TO 15 PSIG WITH DRY NITROGEN.
- 3) STENCIL BY EACH NOZZLE; CONTENTS UNDER N2 PRESSURE; DO NOT OPEN WITHOUT PURCHASER'S APPROVAL.
- 4) TACKWELD A PIECE OF PIPE TO EACH COVER EQUIPPED WITH THE FITTING AND GAUGE ASSEMBLY, FOR PROTECTION DURING SHIPMENT.



ONLY ONE BLIND FLANGE OR COVER SHALL BE TAPPED FOR CHANNEL SIDE, AND ONE FOR SHELL SIDE. PROVIDE 1/4" N.P.T. WITH PLUG ON BOTTOM COVERS FOR DRAINING PURPOSES.

CERTIFIED AS-BUILT

CUSTOMER: MISSISSIPPI POWDER
/ SOUTHERN COMPANY SERVICES
P.J. NO. 17496-0001 REQ NO. MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
ITEM NO. HX0042

NITROGEN PURGE DETAILS			
NO.	DATE	REVISIONS	REV
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DWN. BY: DMORRISON
 DATE: 8/31/12
 CKD. BY: MJD
 DWG. NO.: X-7929-X
 REV: 0

Energy Exchanger Co.
 2014 W. GARY ST. - TAMPA, FL 33605

BILL OF MATERIAL

PART NO	ONE REQ'D, DESCRIPTION OF ONE	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
21	TUBES 3/4" O.D.X 0.0830" (MW) THK. X 14'-0" LG.	SA-179	462	462				*
FORGED RING MATERIAL								
2	FLG 28 3/16" O.D.X 23" I.D.X 3 1/16" THK	SA-266-4N	1	1	1			*
2A	FLG 28 3/16" O.D.X 23" I.D.X 3 1/16" THK	SA-266-4N	1	1	1			*
TUBESHEETS & COVERS								
4	TUBESHEET 24" O.D.X 3 3/8" THK	SA-516-70N	1	1	1			*
9	TUBESHEET 24" O.D.X 3 3/8" THK	SA-516-70N	1	1	1			*
1	COVER 28 3/16" O.D.X 1 3/4" THK	SA-516-70N	1	1	1			*
1A	COVER 28 3/16" O.D.X 1 3/4" THK	SA-516-70N	1	1	1			*
BAFFLES								
22	BAFFLES 22 11/16" O.D.X 1/4" THK	SA-36	7	7				*
22A	FIRST & LAST BAFFLES 22 11/16" O.D.X 1/2" THK	SA-36	2	2				*
HEADS								
CYLINDERS								
12	SMLS.PIPE 24" ODX(0.5000" SchXH)X 1'-9" LG.	SA-333-6	1	1	1			*
15	SMLS.PIPE 24" ODX(0.5620" Sch30)X 13'-5 7/8" LG.	SA-106B	1	1				*
17	SMLS.PIPE 24" ODX(0.5000" SchXH)X 1'-7 1/8" LG.	SA-333-6	1	1	1			*

NOTES: MATERIAL SHALL NOT ORIGINATE IN EITHER CHINA OR NORTH KOREA.

1) MATERIAL HARDNESS SHALL NOT EXCEED 22 HRC.

2) MATERIAL HARDNESS SHALL NOT EXCEED 187 HB.

* DENOTES PRELIMINARY ORDER

CERTIFIED AS-BUILT

REQ No: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER

△ △ △ △ △ △		 Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla. 74116 CUST: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES P.O. 17496-0001 ITEM: HX0042						
NO.	DATE	DESCRIPTION	FB/M	DR: DM	CK: <i>SM</i>	DATE: 3-14-11	DWG. NO. X-7929-1BM	REV 0

BILL OF MATERIAL

PART NO	ONE REQ'D, DESCRIPTION OF ONE	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
ASME FLANGES								
71	ASME 6" - 300 # RF WN W/ (SchXH BORE)	SA-105	1	1				*
72	ASME 3" - 300 # RF WN W/ (SchXXH BORE)	SA-105	1	1				*
81	ASME 10" - 300 # RF WN W/ (SchXH BORE)	SA-105	1	1		2		*
87	ASME 6" - 300 # RF WN W/ (SchXH BORE)	SA-105	1	1		2		*
73	ASME 3/4" - 300 # RF LWN X 9' LG(0.750"BORE)	SA-105	1	1				*
74	ASME 3/4" - 300 # RF LWN X 9' LG(0.750"BORE)	SA-105	1	1				*
75	ASME 3/4" - 300 # RF LWN X 9' LG(0.750"BORE)	SA-105	1	1				*
82	ASME 2" - 300 # RF LWN X 9' LG(2.000"BORE)	SA-105	1	1		2		*
89	ASME 2" - 300 # RF LWN X 9' LG(2.000"BORE)	SA-105	1	1		2		*
71A	SMLS.PIPE 6 5/8" ODX(0.4320" SchXH)X 7 3/4" LG.	SA-106B	1	1				*
72A	SMLS.PIPE 3 1/2" ODX(0.6000" SchXXH)X 8 1/4" LG.	SA-106B	1	1				*
81A	SMLS.PIPE 10 3/4" ODX(0.5000" SchXH)X 7 3/4" LG.	SA-333-6	1	1				*
87A	SMLS.PIPE 6 5/8" ODX(0.4320" SchXH)X 7 3/4" LG.	SA-333-6	1	1				*
71B	WELD PAD 9 3/4" O.D. X 3/8" THK	SA-516-70	1	1				*
73D	ASME 3/4" - 300 # RF BLIND FLANGE	SA-105	1	1				*
74D	ASME 3/4" - 300 # RF BLIND FLANGE	SA-105	1	1				*
75D	ASME 3/4" - 300 # RF BLIND FLANGE	SA-105	1	1				*
82D	ASME 2" - 300 # RF BLIND FLANGE	SA-105	1	1		2		*
89D	ASME 2" - 300 # RF BLIND FLANGE	SA-105	1	1		2		*
COUPLINGS								

SEE DWG. X-7929-1BM FOR MATERIAL NOTES.

* DENOTES PRELIMINARY ORDER

CERTIFIED AS-BUILT

REQ No: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER

▲			Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla.74116	
▲			CUST: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES	
▲			P.O. 17496-0001 ITEM: HX0042	
▲			PB/M DR: MD CK: DN DATE: 1/19/11	DWG. NO. X-7929-2BM
▲			FB/M DR: DM CK: <i>my</i> DATE: 3/14/11	REV 0
NO.	DATE	DESCRIPTION		

BILL OF MATERIAL

PART NO	ONE REQ'D, DESCRIPTION OF ONE	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
BOLTING & GASKETS								
40	LOT OF GASKETS PER DWG X-7929-0		2	2				
41	3/4" ROUND X 10-THREAD STUDS X 5 1/4" LG. 1&2	SA-193B7	24	24				
42	3/4" ROUND X 10-THREAD STUDS X 5 1/4" LG. 1A&2A	SA-193B7	24	24				
43	3/4" HEAVY HEX NUTS 10-THREAD	SA-194-2H	96	96				
44	5/8" ROUND X 11 THREAD STUDS X 3 1/8" LG (3/4"-300# RF)	SA-193B7	12	12				
44A	5/8" HEAVY HEX NUTS (3/4"-300# RF)	SA-194-2H	24	24				
45	5/8" ROUND X 11 THREAD STUDS X 3 5/8" LG (2"-300# RF)	SA-193B7	16	16				
45A	5/8" HEAVY HEX NUTS (2"-300# RF)	SA-194-2H	32	32				
47	GASKETS 0.175" THK 321 STN STL SP. W/D. FLEXIBLE GRAPHITE FILLED							
	W/ 1/8" THK C'STL OUTER COMPRESSION RING FOR (3/4"-300# RF)		3	3				
	(2"-300# RF)		2	2				
MISC. MATERIAL								
60	STD. NAME PLATE	STN.STL.	2	2				
61	STD. NAME PLATE BRK'T.	SA-516-70	2	2				
62	LIFTING LUGS 3 1/2" X 1/2" X 6 1/2" LG	SA-516-70	4	4				
23	FT. 1/2" O.D. TIE RODS	C'STL	72	72				
23A	FT. 3/4" O.D. SPACER TUBING	SA-214	72	72				
23B	1/2" TIE ROD HEX NUTS	C'STL	12	12				
22C	6 1/8" X 1/4" THK X 12" IMP. PLATE	SA-516-70	1	1	1			
63	DELETE				2			
64	DELETE				2			
INSULATION SUPPORT MATERIAL								
65	PLATE 1 1/2" X 1/4" THK X 3' LG (CLIPS)	SA-36	8	8	2			
66	PLATE RING 27" O.D. X 25" I.D. X 1/4" THK	SA-36	2	2	2			
67	ANGLE 3" X 3" X 1/4" THK X 95" LG. (ROLLED LEG IN TO 24 1/8" I.D.)	SA-36	1	1	2			
70	NOZZLE SHIPPING COVERS PER SKETCH							
SUPPORT MATERIAL								
120	PLATE 13" X 3/8" THK X 10 1/8" LG	SA-516-70	2	2				
121	PLATE 6" X 3/4" THK X 11" LG	SA-516-70	2	2				
122	PLATE 8 1/4" X 1/2" THK X 11" LG	SA-516-70	4	4				
125	5/8" DIA X 11 THD X 2" LG HEX HD BOLT	C'STL	1	1				
NITROGEN PURGE MATERIAL								
100	1/4" PIPE NIPPLE X 2' LG. (THRD. OUTSIDE EACH END)	C. STL.	4	4	3			
101	1/4" STRAIGHT TEE (THRD. INSIDE EACH END)	C. STL.	2	2	3			
102	1/4" PIPE VALVE (INSIDE THRD.) (#5X713)	BRASS	2	2	3			
103	1/4" PRESSURE GAUGE (#5X939)(P570K-60LM)(0-60 PSIG)		2	2	3			
104	1/4" PIPE PLUG	C. STL.	2	2	3			
SEE DWG. X-7929-1BM FOR MATERIAL NOTES. * DENOTES PRELIMINARY ORDER								
CERTIFIED AS-BUILT								
REQ No: MC202 SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER								
			 Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla. 74116					
8-31-11	REV'D PER CUST.	DM/	CUST: MISSISSIPPI POWER / SOUTHERN COMPANY SERVICES					
8-10-11	REV'D PER CUST.	WE/MD	P.O. 17496-0001 ITEM: HX0042					
6-24-11	REV'D PER CUST.	WE/DN	PB/M	DR: ----	CK: ----	DATE: ----	DWG. NO.	REV
ND.	DATE	DESCRIPTION	FB/M	DR: DM	CK: SM	DATE: 3/15/11	X-7929-3BM	3

X-7929 PACKING LIST

LINE NO	SPARE PARTS LIST	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
SP1	SPARE GASKETS PER DWG. X-7929-0		SET(S)	SET(S)				
SP2								
SP3								
SP4								
SP5								
SP6								
SP7								
SP8								
SP9								
SP10								

SPARE PARTS TO BE BOXED & TAGGED WITH P.O. NO, ITEM NO, SHIPPING WT. & 'SPARE GASKETS AND PARTS'

LINE NO	STACKING MATERIAL	MATERIAL SPEC.	QTY ONE	TOT QTY	REV	NOTE	PO NO	PROMISED
S1								
S2								
S3								
S4								
S5								
S6								
S7								
S8								
S9								
S10								
S11								
S12								
S13								
S14								
S15								
S16								
S17								
S18								
S19								
S20								
S21								
S22								
S23								
S24								
S25								

* DENOTES PRELIMINARY ORDER

DO NOT ORDER
FROM PACKING LIST

CERTIFIED AS-BUILT

REQ No: MC202
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER

△			 Energy Exchanger Co. 1844 No. Garnett Rd. • Tulsa, Okla. 74116
△			
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△			
CUST: MISSISSIPPI POWER / SOUTHERN COMPANY SERVICES			
P.O.: 17496-0001			ITEM: HX0042
PB/M	DR: ---	CK: ---	DATE: ---
FB/M	DR: DM	CK: <i>[Signature]</i>	DATE: 3/14/11
NEL	DATE	DESCRIPTION	REV 0

Energy Exchanger Company



Miles Howell 8/15/12
CERTIFIED AS-BUILT

DATE: 1/19/2011
PAGE: 1 of 38

MECHANICAL CALCULATIONS

CUSTOMER: MISSISSIPPI POWER c/o SOUTHERN COMPANY SERVICES
DESTINATION: KEMPER COUNTY, MISSISSIPPI
P.O. No.: 17496-0001
ITEM No.: HX0042
E.E.C. SERIAL No.: X-7929
SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER
REQ No.: MC202

DESIGN DATA		
	SHELL SIDE	TUBE SIDE
DESIGN PRESSURE	455 PSIG * / FV	205 PSIG *
DESIGN TEMP.	515°F	390°F
M.D.M.T.	10 °F	10 °F
CORR. ALLOW.	1/8"	3/16"
NO. PASSES	ONE	ONE
M.A.W.P. (H&C)	455 PSIG	205 PSIG
LIMITED BY	FIXED TUBESHEET	FIXED TUBESHEET

NOTE: ALL LOADING CONDITIONS REFERENCED IN
PARA. UG-22 HAVE BEEN ADDRESSED

NOTE: SEE PG. 35 FOR ANCHOR BOLT CALCULATIONS.

- DESIGNED TO ASME SEC VIII, DIV 1, 2007 ED, '09 AD.
- MATERIAL ARE IMPACT TEST EXEMPT PER UG-20(f)

* 9 PSIG ADDED IN DESIGN TO ACCOUNT FOR STATIC HEAD WHEN ERECTED

REVISION LOG

DATE	PAGE	REV.	DESCRIPTION
3/8/2011	AS NOTED	1	REVISED CA TUBE SIDE & PER CUST. COMMENT
5/12/2011	AS NOTED	2	REVISED PER CUST. COMMENT
6/23/2011	AS NOTED	3	REV PER CUST. COMMENT & ADDED FV FOR STEAM OUT

W.O. No.: X-7929

SHELL SIDE

```

=====
NOM.
SIZE      PRESS.      TEMP.      C.A.      RT      H.T.      TEMA
  24.000"   464.0psig *    515deg.F  0.1250"   NONE      0deg.F    R
=====
ASME CODE SEC.VIII DIV.I UG-27 [ CYLINDER ]
====DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON O.D. FORMULA=====
SHELL CYLINDER
MATERIAL: SA-106B      PRESS = 464.0 * psig  TEMP = 515deg.F
O.D.      NOM. THK.      RW=0/PIPE=1  CORR.ALLOW  EFFICIENCY
  24.0000"   0.5620"           1      0.1250"     1.00
ATM.STRS  DES.STRS      STRS/STRS    MAWP N-C    MAWP CORR.
17100.0psi 17100.0psi    1.000      712.422psig 529.087psig

P*(RO)
----- = 0.32212" + CORR. ALLOW. = 0.44712" + MTOL. = 0.51099"
S*E+(.4*P)
*****

```

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

```

----- External Pressure Calculations -----
Ext Press Chart No.= 1
Ext Press Chart....= CS-2
OD.....= 24.00000
T.....= 0.36675
L.....= 162.00000
Temp.....= 515.00000
Stress from MT.....= 17100.00000
Yield from MT.....= 28244.99805
2 x Stress.....= 34200.00000
B1 from Chart@Right= 0.00000
Yield= 2 x B1 x 0.9= 0.00000
S.....= 0.00000
DO/T.....= 65.43967
L/DO.....= 6.75000
A.....= 0.00033
B.....= 4341.26172
PA1.....= 0.00000
PA2.....= 0.00000
PA.....= 88.45321

```

Item No.: HX0042
Job No.: X-7929

REV 1
By: MD/DN 3/9/2011

P.O. No.: 17496-0001
Page: 2

TOP & BOTTOM CHANNELS

```

=====
NOM.
SIZE          PRESS.      TEMP.      C.A.      RT      H.T.      TEMA
24.000"       214.0psig *    390deg.F  0.1875"  SPOT    1150deg.F  R
=====
ASME CODE SEC.VIII DIV.I UG-27 [ CYLINDER ]
===DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON O.D. FORMULA===
CHANNEL CYLINDERS
MATERIAL: SA-333-6      PRESS = 214.0psig * TEMP = 390deg.F
O.D.      NOM. THK.      RW=0/PIPE=1  CORR.ALLOW  EFFICIENCY
24.0000"   0.5000"        1          0.1875"    0.85
ATM.STRS   DES.STRS      STRS/STRS    MAWP N-C    MAWP CORR.
17100.0psi 17100.0psi  1.000      537.764psig 305.357psig

P*(RO)
----- = 0.17564" + CORR. ALLOW. = 0.36314" + MTOL. = 0.41502"
S*E+(.4*P)
*****
* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

```

----- External Pressure Calculations -----

```

Ext Press Chart....= 1 CS-2
Actual Chart Used..= 1 CS-2
Material.....= SA-516-70
OD.....= 24.00000
T.....= 0.25000
L.....= 25.00000
Temp.....= 250.00000
Stress from MT....= 20000.00000
Yield from MT....= 34200.00000
2 x Stress.....= 40000.00000
B1 from Chart@Right= 0.00000
Yield= 2 x B1 x 0.9= 0.00000
S.....= 0.00000
DO/T.....= 96.00000
L/DO.....= 1.04167
A.....= 0.00137
B.....= 13468.19434
PA1.....= 0.00000
PA2.....= 0.00000
PA.....= 187.05826

```



Item No.: HX0042
Job No.: X-7929

REV 3
By: MD/DN 6/23/2011

P.O. No.: 17496-0001
Page: 3

NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16
 NOZ NO= S1 6.00" 300 LB RF WN
 [INLET] SHELL SIDE
 PRESSURE = 464.0psig * TEMP = 515.0deg.F

```

-----CYLINDER-----NOZZLE-----
SA-106B                      SA-106B
OPER STRS= 17100.0 psi      OPER STRS= 17100.0 psi
ATM STRS = 17100.0 psi      ATM STRS = 17100.0 psi
C.A. = 0.12500"             C.A. = 0.12500"
I.D. = 23.01650" **  I.D. = 5.76100"
O.D. = 24.00000"             O.D. = 6.62500"
THK. = 0.49175" **          THK. XH= 0.43200"
T = 0.36675" **  TN = 0.30700"
TR = 0.32212" **           TRN = 0.08892"
ET = 0.04463" **           ETN = 0.21808"
  
```

MIN STD. WT. PIPE+C.A.= 0.37000" TR+C.A.= 0.44712"
 LIMITS= 12.02200"
 fr1= 1.0000 fr2= 1.0000 fr3= 1.0000 fr4= 1.0000 F= 1.00000
 Min weld sizes: NS= 0.0000 NP= 0.3039 PS= 0.2593 IF= 0.0000"

AREA REQD. A = 1.93625in²
 EXCESS IN CYL. A1= 0.26828in²
 EXCESS IN NOZ. (OUTSIDE) A2= 0.39991in² H = 0.91687"
 AREA OF FILLETS = 0.28125in²
 AREA AVAILABLE = 0.94944in²

NOZ. NECK WELD= 0.37500" AREA= 0.14062in²

PAD MATL.= SA-516-70 S= 19910.psi
 PAD O.D.= 9.62500" THK.= 0.37500" AREA= 1.12500in²
 PAD TO CYL. WELD= 0.37500" AREA= 0.14062in²
 ARC= 6.712"
 PAD RET. DIM.= 9.625" X 9.750"

< AREA REQD-AREA AVAILABLE > = -0.13819in²

MAWP N-C= 628.0psig LIMITED BY AR
 MAWP COR= 480.0psig LIMITED BY AR

All Weld Paths are stronger than the required strength /[UG-41 UG-45 UW-15]



* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD
 ** Please see UG-37, pipe thickness and ID based on nominal thickness less manufacturing undertolerances allowed by material specification.

EXTERNAL PRESSURE VERIFICATION:

```

NOZTYP[ 0.0] X-7929          6.00"- 300# RF WN PRESS= -15 TEMP= 515 INLET
<<<<<< CYLINDER >>>>>> <<<<<< NOZZLE >>>>>>
SA-106B                      SA-106B
OPER STRS=17100.0000 OPER STRS=17100.0000 Area reqd. A= 0.5447
ATM STRS =17100.0000 ATM STRS =17100.0000 Excess in cyl A1= 1.1150
C.A. = 0.12500 C.A. = 0.12500 Excess in noz A2= 0.5146 H= 0.9169
I.D. = 23.01650 I.D. = 5.76100 Excess in noz A3= 0.0000 H= 0.0000
O.D. = 24.00000 O.D. = 6.62500 Area fillets = 0.2812
THK. = 0.49175 THK. XH= 0.43200 Area avail. = 1.9108
T = 0.36675 TN = 0.30700 Noz neck weld = 0.3750 A= 0.1406
TR = 0.18125 TRN = 0.02640 Pad weld = 0.3750 A= 0.1406
ET = 0.18550 ETN = 0.28060 Internal weld = 0.0000 A= 0.0000
PIPE F=1.0000 EI=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)
MIN STD. WT. PIPE+C.A.=0.37000 TR[using abs(P)]+C.A.=0.13552 CHORD= 0.0000
LIMITS= 12.02200 fr1= 1.0000 fr2= 1.0000 fr3= 1.0000 fr4= 1.0000
Min weld sizes: NS= 0.0000 NP= 0.3039 PS= 0.2593 IF= 0.0000
PAD MATL.= SA-516-70 O.D.= 9.62500 THK.= 0.37500 AREA= 1.12500 S= 19910.
Area reqd-Area avail =<< -2.49109>>
  
```

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NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO= S2 3.00" 300 LB RF WN

[OUTLET] SHELL SIDE

PRESSURE = 464.0psig * TEMP = 515.0deg.F

```

-----CYLINDER-----NOZZLE-----
SA-106B                      SA-106B
OPER STRS= 17100.0 psi      OPER STRS= 17100.0 psi
ATM STRS = 17100.0 psi      ATM STRS = 17100.0 psi
C.A.      = 0.12500"        C.A.      = 0.12500"
I.D.      = 23.01650" **    I.D.      = 2.30000"
O.D.      = 24.00000"        O.D.      = 3.50000"
THK.      = 0.49175" **    THK. XXH= 0.60000"
T          = 0.36675" **    TN          = 0.47500"
TR         = 0.32212" **    TRN        = 0.04698"
ET         = 0.04463"        ETN         = 0.42802"

```

MIN STD. WT. PIPE+C.A.= 0.31400" TR+C.A.= 0.44712"

LIMITS= 5.10000"

fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.3536" NP= 0.0000" PS= 0.0000" IF= 0.0000"

```

AREA REQD.      A = 0.82140in^2
EXCESS IN CYL.  A1= 0.11381in^2
EXCESS IN NOZ. (OUTSIDE) A2= 0.78489in^2 H = 0.91687"
AREA OF FILLETS = 0.14062in^2
AREA AVAILABLE  = 1.03933in^2

```

NOZ. NECK WELD= 0.37500" AREA= 0.14062in^2

< AREA REQD-AREA AVAILABLE > = -0.21793in^2

MAWP N-C= 806.0psig LIMITED BY AR

MAWP COR= 523.0psig LIMITED BY AR

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

 ** Please see UG-37, pipe thickness and ID based on nominal thickness less manufacturing undertolerances allowed by material specification.

EXTERNAL PRESSURE VERIFICATION:

```

NOZTYP[ 0.0] X-7929          3.00"- 300# RF WN  PRESS= -15 TEMP= 515  OUTLET
<<<<<< CYLINDER >>>>>> <<<<<< NOZZLE >>>>>>
SA-106B                      SA-106B
OPER STRS=17100.0000 OPER STRS=17100.0000 Area reqd.  A= 0.2311
ATM STRS =17100.0000 ATM STRS =17100.0000 Excess in cyl A1= 0.4730
C.A.      = 0.12500 C.A.      = 0.12500 Excess in noz A2= 0.8377 H= 0.9169
I.D.      = 23.01650 I.D.      = 2.30000 Excess in noz A3= 0.0000 H= 0.0000
O.D.      = 24.00000 O.D.      = 3.50000 Area fillets  = 0.1406
THK.      = 0.49175 THK. XXH= 0.60000 Area avail.   = 1.4513
T          = 0.36675 TN          = 0.47500 Noz neck weld = 0.3750 A= 0.1406
TR         = 0.18125 TRN        = 0.01820 Pad weld    = 0.0000 A= 0.0000
ET         = 0.18550 ETN        = 0.45680 Internal weld = 0.0000 A= 0.0000
PIPE F=1.0000 E1=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)
MIN STD. WT. PIPE+C.A.=0.31400 TR[using abs(P)]+C.A.=0.13552 CHORD= 0.0000
LIMITS= 5.10000 fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000
Min weld sizes: NS= 0.3536 NP= 0.0000 PS= 0.0000 IF= 0.0000
Area reqd-Area avail =<< -1.22021>>

```

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NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO= T1 6.00" 300 LB RF WN

[INLET] REAR CHANNEL or COVER

PRESSURE = 214.0psig * TEMP = 390.0deg.F

-----CYLINDER-----NOZZLE-----

SA-333-6		SA-333-6	
OPER STRS=	17100.0 psi	OPER STRS=	17100.0 psi
ATM STRS =	17100.0 psi	ATM STRS =	17100.0 psi
C.A. =	0.18750"	C.A. =	0.18750"
I.D. =	23.12500" ** 	I.D. =	5.76100"
O.D. =	24.00000"	O.D. =	6.62500"
THK. =	0.43750" ** 	THK. XH=	0.43200"
T =	0.25000" **	TN =	0.24450"
TR =	0.14943" **	TRN =	0.04125"
ET =	0.10057"	ETN =	0.20325"

MIN STD. WT. PIPE+C.A.= 0.43250" TR+C.A.= 0.33693"

LIMITS= 12.27200"

fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.2420" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD.	A =	0.91689in ²
EXCESS IN CYL.	A1=	0.61711in ²
EXCESS IN NOZ. (OUTSIDE)	A2=	0.24848in ² H = 0.61125"
AREA OF FILLETS	=	0.14062in ²
AREA AVAILABLE	=	1.00621in ²

NOZ. NECK WELD= 0.37500" AREA= 0.14062in²< AREA REQD-AREA AVAILABLE > = -0.08933in²

MAWP N-C= 426.0psig LIMITED BY AR
 MAWP COR= 224.0psig LIMITED BY AR.

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

** Please see UG-37, pipe thickness and ID based on nominal thickness less manufacturing 
 undertolerances allowed by material specification.

EXTERNAL PRESSURE FOR STEAM OUT:

NOZTYP[11.0] X-7929 6.00"- 300# RF WN PRESS= -15 TEMP= 250 INLET
 <<<<< CYLINDER >>>>> <<<<< NOZZLE >>>>>
 SA-106B SA-106B
 OPER STRS=17100.0000 OPER STRS=17100.0000 Area reqd. A= 0.2194
 ATM STRS =17100.0000 ATM STRS =17100.0000 Excess in cyl A1= 1.0953
 C.A. = 0.18750 C.A. = 0.18750 Excess in noz A2= 0.2654 H= 0.6113
 I.D. = 23.12500 I.D. = 5.76100 Excess in noz A3= 0.0000 H= 0.0000
 O.D. = 24.00000 O.D. = 6.62500 Area fillets = 0.1406
 THK. = 0.43750 THK. XH= 0.43200 Area avail. = 1.5013
 T = 0.25000 TN = 0.24450 Noz neck weld = 0.3750 A= 0.1406
 TR = 0.07150 TRN = 0.02740 Pad weld = 0.0000 A= 0.0000
 ET = 0.17850 ETN = 0.21710 Internal weld = 0.0000 A= 0.0000
 PIPE F=1.0000 E1=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)
 MIN STD. WT. PIPE+C.A.=0.43250 TR[using abs(P)]+C.A.=0.19802 CHORD= 0.0000
 LIMITS= 12.27200 fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000
 Min weld sizes: NS= 0.2420 NP= 0.0000 PS= 0.0000 IF= 0.0000
 Area reqd-Area avail =<< -1.28194>>

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NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO= T2 10.00" 300 LB RF WN

[OUTLET] FRONT CHANNEL SIDE

PRESSURE = 214.0psig * TEMP = 390.0deg.F

-----CYLINDER-----NOZZLE-----

SA-333-6		SA-333-6	
OPER STRS=	17100.0 psi	OPER STRS=	17100.0 psi
ATM STRS =	17100.0 psi	ATM STRS =	17100.0 psi
C.A. =	0.18750" 	C.A. =	0.18750"
I.D. =	23.12500" **	I.D. =	9.75000"
O.D. =	24.00000"	O.D. =	10.75000"
THK. =	0.43750" **	THK. XH=	0.50000"
T =	0.25000" ** 	TN =	0.31250"
TR =	0.14943" **	TRN =	0.06693"
ET =	0.10057"	ETN =	0.24557"

MIN STD. WT. PIPE+C.A.= 0.50688" TR+C.A.= 0.33693"

LIMITS= 20.25000"

fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.2475" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD.	A =	1.51295in ²	
EXCESS IN CYL.	A1=	1.01830in ²	
EXCESS IN NOZ.(OUTSIDE)	A2=	0.30696in ²	H = 0.62500"
AREA OF FILLETS	=	0.25000in ²	
AREA AVAILABLE	=	1.57526in ²	

NOZ. NECK WELD= 0.50000" AREA= 0.25000in²< AREA REQD-AREA AVAILABLE > = -0.06231in²

MAWP N-C= 393.0psig LIMITED BY AR

MAWP COR= 218.0psig LIMITED BY AR

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

 ** Please see UG-37, pipe thickness and ID based on nominal thickness less manufacturing undertolerances allowed by material specification.

EXTERNAL PRESSURE FOR STEAM OUT:

NOZTYP[0.0] X-7929 10.00"- 300# RF WN PRESS= -15 TEMP= 250 OUTLET
 <<<<< CYLINDER >>>>> <<<<< NOZZLE >>>>>
 SA-106B SA-106B
 OPER STRS=17100.0000 OPER STRS=17100.0000 Area reqd. A= 0.3989
 ATM STRS =17100.0000 ATM STRS =17100.0000 Excess in cyl A1= 1.7334
 C.A. = 0.18750 C.A. = 0.18750 Excess in noz A2= 0.3451 H= 0.6250
 I.D. = 23.12500 I.D. = 9.75000 Excess in noz A3= 0.0000 H= 0.0000
 O.D. = 24.00000 O.D. = 10.75000 Area fillets = 0.2500
 THK. = 0.43750 THK. XH= 0.50000 Area avail. = 2.3285
 T = 0.25000 TN = 0.31250 Noz neck weld = 0.5000 A= 0.2500
 TR = 0.07880 TRN = 0.03640 Pad weld = 0.0000 A= 0.0000
 ET = 0.17120 ETN = 0.27610 Internal weld = 0.0000 A= 0.0000
 PIPE F=1.0000 E1=1.0000 SHELL TOL=0.8750 (Min USED) NOZ TOL=1.0000 (Nom USED)
 MIN STD. WT. PIPE+C.A.=0.50688 TR[using abs(P)]+C.A.=0.19802 CHORD= 0.0000
 LIMITS= 20.25000 fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000
 Min weld sizes: NS= 0.2475 NP= 0.0000 PS= 0.0000 IF= 0.0000
 Area reqd-Area avail. =<< -1.92960>>

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ASME SEC.VIII DIV.1 APPENDIX 2			Front Channel to cover flange		
DESIGN CONDITIONS			GASKET and BOLTING CALCULATIONS		
Design Pres=	214.0	psig	Eff.Gsk OD=	24.6875"	No.Passes= 1
Neg. Pres=	None		Eff.Gsk ID=	23.9375"	N= 0.375"
Design Temp=	390	deg.F	THK=	0.1250"	col.=2
Flg Matl=SA-266-4			Gsk Matl=S.S. DJNAF		y= 9000psi
Bolt Matl=SA-193B7			Gsk Face=FLAT FACE	w= 0.0000"	m= 3.750
Corr Allow=	0.1875"	Δ	Wm2=	128891 #	Am = 5.1556in ²
Flange Desn Sfo	20000.0	psi	Hp =	22985 #	Ab = 7.2480in ²
Atm. Sfa	20000.0	psi	H =	99348 #	W = 155045 #
Bolting Desn Sb	25000.0	psi	Wm1=	122334 #	Wm1= 122334 #
Atm. Sa	25000.0	psi	Gasket Width Check	Nmin =	0.1318"
CONDITION	LOAD	x	LEVER ARM	=	MOMENT
Operating	HD =	91834 #	hD=	1.35938"	MD= 124837 in#
	HG =	22985 #	hG=	1.12500"	MG= 25858 in#
	HT =	7514 #	hT=	1.35938"	MT= 10214 in#
Gasket					Mo= 160911 in#
Seating	HG =	155045 #	hG=	1.12500"	mo= 174426 in#
Allow.Stress-STRESS CALCULATIONS-Operating			SHAPE CONSTANTS		
1.5 Sfo&2.5 Sno Long Hub,SH	25820.0	psi	K	= 1.2059	h/ho= 0.4625
Sfo Radial Flg,SR	3619.6	psi	T	= 1.8366	F = 0.8541
Sfo Tang Flg,ST	9205.3	psi	Z	= 5.4038	V = 0.3214
Sfo .5(SH+SR)or.5(SH+ST)	17512.7	psi	Y	=10.4756	f = 1.0000
J(APP.2-14)=	0.9607		U	=11.5116	e = 0.3160in ⁻¹
Allow.Stress-STRESS CALCULATIONS-Gsk.Seating			g1/go= 1.5000 d = 9.452In ³		
1.5 Sfa&2.5 Sna Long Hub,SH	27988.6	psi	ho	= 2.7027"	
Sfa Radial Flg,SR	3923.6	psi			
Sfa Tang Flg,ST	9978.5	psi	OTHER STRESS FORMULA FACTORS		
Sfa .5(SH+SR)or.5(SH+ST)	18983.6	psi	t	1.6250"	
J(APP.2-14)=	0.9893		Alpha	1.5135	
			Beta	1.6847	
			Gamma	0.8241	
			Delta	0.4540	
			Lambda	1.2781	
			M	7250 #	
			m	7859 #	
O.D.	= 28.1875"		THK.	= 1.8125"	
I.D.	= 23.0000"		T-Adder	= 0.0000"	
GO	= 0.5000"		G1	= 0.6562"	
HUB O.D.	= 24.3125"		HUB LEN	= 1.2500"	
HUB ANG	= 7.1250deg		RIB LENGTH	= 0.0000"	
G(MEAN)	= 24.3125"		G(CALC)	= 24.3125"	
G(MIN.)	= 23.7500"		G(MAX.)	= 24.8125"	
B.C.	= 26.5625"		B.S.C.F.	= 1.0533	
No. BOLTS	= 24		BOLT DIAMETER	= 0.7500"	
R	= 1.1250"		E	= 0.8125"	
BOLT SPAC	= 3.4671"		TORQUE	= 96ft#	
MIN. SPAC	= 1.7500"		TEMA MAX. SPAC	= 3.7941"	
FLG TURN	= 0.1875"		FACING	= RECESS	
BPRIME	= 0.0000"		BG1	= 0.0000"	
FLG RWT	= 181Lbs		FLG FWT	= 116Lbs	

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

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///// FRONT COVER CALCULATIONS /////

```

EXCHANGER TYPE....= N E N      MATERIAL.....= SA-516-70N
PRESSURE.....=      214.0 psig  CORROSION ALLOWANCE=      0.1875"
TURN LOW.....=      0.1875"    TURN HIGH.....=      0.2500"
S COVER STRESS...=      20000.0 psi  E.....=      1.0
Sb STRESS BOLTS..=      25000.0 psi  Ab.....=      7.2480 in^2
d.....=      24.3125"             C.....=      0.3000
W.....=      155045 #             Wml.....=      122334 #
hG (ASME).....=      1.1250"       hG (TEMA).....=      0.0000"
G (TEMA).....=      24.3125"       E (MOD.) x (10)-6..=      27.7 psi
COVER THK SET HOLD=      [ NO]

```

[PER ASME SECT.VIII UG-34 USING Wml (OPERATING)]

$$THK = d \sqrt{\frac{C P}{S E} + \frac{1.9 Wml hG}{S E d^3}} = 1.5605"$$

[PER ASME SECT.VIII UG-34 USING W (GASKET SEATING)]

$$THK = d \sqrt{\frac{1.9 W hG}{S E d^3}} = 0.8256"$$

[COV THK Min] CODE Operating = 1.5605" + 0.1875" = 1.7480"

[COV THK Min] CODE Gasket Seating = 0.8256" + 0.1875" = 1.0131"

ACTUAL COVER THICKNESS USED (SET HOLD[NO])= 1.7500"

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

MAIN FLG IS FLANGE MARK # 2
 COMP FLG IS FLANGE MARK # 0

/// INDIVIDUAL COMPONENTS ///

MAWP CORR. MAIN FLG = 222.0 Limited by: J-Calc. Operating
 MAWP CORR. COVER = 214.0
 MAWP CORR. T.S.[STD DIA]= 0.0

MAWP N-C MAIN FLG = 316.0 Limited by: Minimum bolting area Am.
 MAWP N-C COVER = 269.0

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ASME SEC.VIII DIV.1 UHX-13

(FTS) Configuration a

No or EPC=Elastic-Plastic Calcs	SSC=Simply Supported Calcs	=SSC
MT Tubes	=SA-179	MT Tubesheets =SA-516-70N
MT Channel	=SA-333-3	MT Shell =SA-106B
Diameter (Do)	= 22.3736"	Pass Lane Area (AL) = 0.000in ²
Perimeter of layout (CP)	= 0.0000"	Area enclosed by CP (AP) = 0.000in ²
Tubesheet thickness (h)	= 3.3750"	Tubesheet thickness HOLD = NO
Tube Pitch (p)	= 0.9375"	Pitch type = Triangle (30 Deg) 
Diameter of tube (dt)	= 0.7500"	Tube thickness (tt) = 0.0830"
Exp length of tube (ltx)	= 0.0000"	Exp depth ratio (Rho) = 0.9500
Groove channel side (hg)	= 0.0000"	Groove shell side = 0.0000"
Channel I.D. (Dc)	= 23.0000"	Channel thickness (Tc) = 0.5000"
Shell I.D. (Ds)	= 22.8760"	Shell thickness (Ts) = 0.5620"
TS outside diameter (A)	= 24.0000"	Bolt circle (C) = 0.0000"
G Channel side (Gc or G1)	= 0.0000"	G Shell side (Gs or G1) = 0.0000"
Bolt load Chan side (Wc)	= 0.0#	Bolt load Sh side (Ws) = 0.0#
1.8*sqrt(Dc*Tc)	= 0.0000"	1.8*sqrt(Ds*(Ts or Ts1)) = 0.0000"
Thickness at Top TS (tr)	= 3.3750"	Outer Tube Circle = 22.3736"
hr Ch Op= 0.0000" Ch Atm= 0.0000"		Sh Op= 0.0000" Sh Atm= 0.0000"
Tubes welded Backside TS	=NO	Hole Size in Tubesheet = 0.0000"
Radial Differential Thermal Expansion Adjacent to Tubesheet		= NO
Design Press Chan (Pt)	= 214.0psig	* Design Press Shell (Ps) = 464.0psig *
Design Temp Chan	= 390.0deg.	Design Temp Shell = 515.0deg.
Corr Chan	= 0.18750"	Corr Shell = 0.12500"
Corr Chan side TS (ct)	= 0.18750" 	Corr Shell side TS (cs) = 0.12500"
Number Tubes (Nt)	= 462.00	F-F Tubesheets (Lt) = 167.8750"
Un-supported Tube Span (l)	= 31.25000"	Support Constant (k) = 1.00000
Expansion Joint	= None	Exp Joint I.D. = 0.00000"
Spring Rate Corr (Kj)	= 0.0#	Spring Rate Un-Corr (Kjn)= 0.0#
MT Thickened Shl	=	Thk Thickened Shl (Ts1) = 0.00000"
Len Thickened Shl (lll)	= 0.00000"	Len Thickened Shl (lll') = 0.00000"

Calculate/Conservative For Ps* and Pc* = Calculated values for Ps* and Pc*

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"	F-F Inside Tubesheets (L)	=	161.3750"
F-F Tubesheets (Lt)	=	167.5000"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.9094"	Channel thickness (Tc)	=	0.3125"
Channel I.D. (Dc)	=	23.3750"	Shell thickness (Ts)	=	0.4370"
Shell I.D. (Ds)	=	23.1260"			
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.18750"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps =	61430psi
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spssc =	60020psi
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spsss =	56489psi

Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"
W =	0.0#	wj =	0.0"	ao =	11.18677998"
Rho s =	1.033630729	Rho c =	1.044759989	Chi s =	0.480848223
Chi t =	0.685226977	Ks =	5411391.500Lb/In	Kt =	29093.82812Lb/In
Ks,t =	0.402592897	j =	1.000000000	Beta s =	0.566499650in ⁻¹
Kappa s =	233740.8281#	Lambda s =	4787501.000psi	Beta c =	0.668145478in ⁻¹
Kappa c =	103594.0859#	Lambda c =	2599820.750psi	h/p =	3.266666651
E* =	8053920.000psi	v* =	0.345361441	Xa =	2.644720793
Zd =	0.077879079	Zv =	0.126476079	Zm =	0.564917266
K =	1.072694778	F =	0.754432321	Phi =	1.014984131
Q1 =	-0.060214549	Qz1 =	1.718772292	Qz2 =	2.261739016
U =	4.523478031	Gamma =	0.0"	Omega s =	0.0in ²
Omega c =	0.0in ²	Omega s* =	-3.534133911in ²	Omega c* =	-2.125888348in ²
Delta s =	0.000009630In ³ /Lb	Delta c =	0.000013390In ³ /Lb	Gamma b =	0.0
Ps' =	0.0psi	Pt' =	718.6097412psi	P Gamma =	0.0psi
PW =	0.0psi	Prim =	-16.44434357psi	Es,w =	1.000000000
Pe =	-171.7923889psi	Q2 =	289.1478882#	Q3 =	-0.087113529
Fm =	0.094685681	Es,w1 =	0.0	lt =	31.25000000"
Rt =	0.237638906"	Ft =	131.5020294	Ct =	159.2741089
Fs =	2.000000000	Stb =	6166.893066psi	Ft,min =	0.234580144
Sigmat,1 =	-520.3059692psi	Sigmat,2 =	651.9843750psi	Ft,max =	1.629235864
Sigma s,m =	512.9665527psi	Sigma s,b =	-6095.613281psi	Sigma s =	6608.580078psi
Sigma c,m =	3949.005859psi	Sigma c,b =	17655.07422psi	Sigma c =	21604.08008psi

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas=0		Betac, Kappac, Lambdac, Deltac, Omegac=0			
F	= 0.153976202	Phi	= 0.207153648	Q1	= 0.006652270
Qz1	= 1.925647497	Qz2	= 3.185766220	U	= 6.371532440
Omega s*	= 0.071960613in^2	Omega c*	= 0.824546218in^2	Prim	= 8.983849525psi
Pe	=-157.1120758psi	Q2	=-157.9668732#	Q3	= 0.022720801

Sigma	-4159 psi	<=	1.5S	29865 psi	[OK Per Simply Sup]
Tau	-1568 psi	<=	.8S	15928 psi	[OK]
Sigmat,max	651 psi	<	St	13384 psi	[OK]
Sigmat,min	-520 psi	<=	Stb	6166 psi	[OK]
Sigma s	6608 psi	<=	Spss	56489 psi	[OK Per Simply Sup]
Sigma c	21604 psi	<=	Spssc	60020 psi	[OK Per Simply Sup]
Sigmatas,m	512 psi	<=	SsEs,w	17100 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	= 3.0625"	F-F Inside Tubesheets (L)	= 161.3750"
F-F Tubesheets (Lt)	= 167.5000"	Exp depth ratio (Rho)	= 0.9500"
Exp length of tube (ltx)	= 2.9094"	Channel thickness (Tc)	= 0.3125"
Channel I.D. (Dc)	= 23.3750"	Shell thickness (Ts)	= 0.4370"
Shell I.D. (Ds)	= 23.1260"		

Press Chan (Pt)	= 0.0psi	Press Shell (Ps)	= 464.0psi
Corr Chan	= 0.18750"	Corr Shell	= 0.12500"
Corr Chan side TS (ct)	= 0.18750"	Corr Shell side TS (cs)	= 0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spcc = 60020psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	

Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"
W = 0.0#	wj = 0.0"	ao = 11.18677998"			
Rho s = 1.033630729	Rho c = 1.044759989	Chi s = 0.480848223			
Chi t = 0.685226977	Ks = 5411391.500Lb/In	Kt = 29093.82812Lb/In			
Ks,t = 0.402592897	j = 1.000000000	Beta s = 0.566499650in ⁻¹			
Kappa s = 233740.8281#	Lambda s = 4787501.000psi	Beta c = 0.668145478in ⁻¹			
Kappa c = 103594.0859#	Lambda c = 2599820.750psi	h/p = 3.266666651			
E* = 8053920.000psi	v* = 0.345361441	Xa = 2.644720793			
Zd = 0.077879079	Zv = 0.126476079	Zm = 0.564917266			
K = 1.072694778	F = 0.754432321	Phi = 1.014984131			
Q1 = -0.060214549	Qz1 = 1.718772292	Qz2 = 2.261739016			
U = 4.523478031	Gamma = 0.0"	Omega s = 0.0in ²			
Omega c = 0.0in ²	Omega s* = -3.534133911in ²	Omega c* = -2.125888348in ²			
Delta s = 0.000009630In ³ /Lb	Delta c = 0.000013390In ³ /Lb	Gamma b = 0.0			
Ps' = 1027.633179psi	Pt' = 0.0psi	P Gamma = 0.0psi			
PW = 0.0psi	Prim = 59.27386856psi	Es,w = 1.000000000			
Pe = 254.0253296psi	Q2 = -1042.237671#	Q3 = -0.125785202			
Fm = 0.085965946	Es,w1 = 0.0	lt = 31.25000000"			
Rt = 0.237638906"	Ft = 131.5020294	Ct = 159.2741089			
Fs = 2.000000000	Stb = 6166.893066psi	Ft,min = 0.336023808			
Sigmat,1 = 674.0183105psi	Sigmat,2 = -786.8353882psi	Ft,max = 1.511368990			
Sigma s,m = 3472.952148psi	Sigma s,b = 22914.10742psi	Sigma s = 26387.05859psi			
Sigma c,m = 0.0psi	Sigma c,b = -5796.971680psi	Sigma c = 5796.971680psi			

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas = 0	Betac, Kappac, Lambdac, Deltac, Omegac = 0	
F = 0.153976202	Phi = 0.207153648	Q1 = 0.006652270
Qz1 = 1.925647497	Qz2 = 3.185766220	U = 6.371532440
Omega s* = 0.071960613in ²	Omega c* = 0.824546218in ²	Prim = -1.699990630psi
Pe = 227.1429138psi	Q2 = 29.89166260#	Q3 = 0.008755420

Sigma	5584 psi	<= 1.5S	29865 psi	[OK Per Simply Sup]
Tau	2319 psi	<= .8S	15928 psi	[OK]
Sigmat,max	786 psi	< St	13384 psi	[OK]
Sigmat,min	-786 psi	<= Stb	6166 psi	[OK]
Sigma s	26387 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	5796 psi	<= Spcc	60020 psi	[OK Per Simply Sup]
Sigmatas,m	3472 psi	<= SsEs,w	17100 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"		
F-F Tubesheets (Lt)	=	167.5000"	F-F Inside Tubesheets (L)	= 161.3750"
Exp length of tube (ltx)	=	2.9094"	Exp depth ratio (Rho)	= 0.9500"
Channel I.D. (Dc)	=	23.3750"	Channel thickness (Tc)	= 0.3125"
Shell I.D. (Ds)	=	23.1260"	Shell thickness (Ts)	= 0.4370"
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	= 464.0psi
Corr Chan	=	0.18750"	Corr Shell	= 0.12500"
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	= 0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps =	61430psi
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spac =	60020psi
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss =	56489psi

Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"
W =	0.0#	wj =	0.0"	ao =	11.18677998"
Rho s =	1.033630729	Rho c =	1.044759989	Chi s =	0.480848223
Chi t =	0.685226977	Ks =	5411391.500Lb/In	Kt =	29093.82812Lb/In
Ks, t =	0.402592897	j =	1.000000000	Beta s =	0.566499650in ⁻¹
Kappa s =	233740.8281#	Lambda s =	4787501.000psi	Beta c =	0.668145478in ⁻¹
Kappa c =	103594.0859#	Lambda c =	2599820.750psi	h/p =	3.266666651
E* =	8053920.000psi	v* =	0.345361441	Xa =	2.644720793
Zd =	0.077879079	Zv =	0.126476079	Zm =	0.564917266
K =	1.072694778	F =	0.754432321	Phi =	1.014984131
Q1 =	-0.060214549	Qz1 =	1.718772292	Qz2 =	2.261739016
U =	4.523478031	Gamma =	0.0"	Omega s =	0.0in ²
Omega c =	0.0in ²	Omega s* =	-3.534133911in ²	Omega c* =	-2.125888348in ²
Delta s =	0.000009630In ³ /Lb	Delta c =	0.000013390In ³ /Lb	Gamma b =	0.0
Ps' =	1027.633179psi	Pt' =	718.6097412psi	P Gamma =	0.0psi
PW =	0.0psi	Prim =	42.82952499psi	Es, w =	1.000000000
Pe =	82.23294067psi	Q2 =	-753.0898438#	Q3 =	-0.206573993
Fm =	0.101863697	Es, w1 =	0.0	lt =	31.25000000"
Rt =	0.237638906"	Ft =	131.5020294	Ct =	159.2741089
Fs =	2.000000000	Stb =	6166.893066psi	Ft, min =	0.547949255
Sigmat, 1 =	153.7122803psi	Sigmat, 2 =	-134.8510742psi	Ft, max =	1.265134096
Sigma s, m =	3985.918701psi	Sigma s, b =	16818.49219psi	Sigma s =	20804.41016psi
Sigma c, m =	3949.005859psi	Sigma c, b =	11858.10254psi	Sigma c =	15807.10840psi

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas=0		Betac, Kappac, Lambdac, Deltac, Omegac=0			
F	= 0.153976202	Phi	= 0.207153648	Q1	= 0.006652270
Qz1	= 1.925647497	Qz2	= 3.185766220	U	= 6.371532440
Omega s*	= 0.071960613in^2	Omega c*	= 0.824546218in^2	Prim	= 7.283859253psi
Pe	= 70.03083801psi	Q2	= -128.0752106#	Q3	= -0.022575490

Sigma	2141 psi	<= 1.5S	29865 psi	[OK Per Simply Sup]
Tau	750 psi	<= .8S	15928 psi	[OK]
Sigmat, max	153 psi	< St	13384 psi	[OK]
Sigmat, min	-134 psi	<= Stb	6166 psi	[OK]
Sigma s	20804 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	15807 psi	<= Spsc	60020 psi	[OK Per Simply Sup]
Sigmat, m	3985 psi	<= SsEs, w	17100 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"	F-F Inside Tubesheets (L)	=	161.3750"
F-F Tubesheets (Lt)	=	167.5000"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.9094"	Channel thickness (Tc)	=	0.3125"
Channel I.D. (Dc)	=	23.3750"	Shell thickness (Ts)	=	0.4370"
Shell I.D. (Ds)	=	23.1260"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.18750"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spssc = 60020psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	
Tt,m = 393deg.F			At,m = 7.0860In/In			
Ts,m = 423deg.F			As,m = 7.1460In/In			
Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"	
W = 0.0#		wj = 0.0"	ao = 11.18677998"			
Rho s = 1.033630729		Rho c = 1.044759989	Chi s = 0.480848223			
Chi t = 0.685226977		Ks = 5411391.500Lb/In	Kt = 29093.82812Lb/In			
Ks,t = 0.402592897		j = 1.000000000	Beta s = 0.566499650in ⁻¹			
Kappa s = 233740.8281#		Lambda s = 4787501.000psi	Beta c = 0.668145478in ⁻¹			
Kappa c = 103594.0859#		Lambda c = 2599820.750psi	h/p = 3.266666651			
E* = 8053920.000psi		v* = 0.345361441	Xa = 2.644720793			
Zd = 0.077879079		Zv = 0.126476079	Zm = 0.564917266			
K = 1.072694778		F = 0.754432321	Phi = 1.014984131			
Q1 = -0.060214549		Qz1 = 1.718772292	Qz2 = 2.261739016			
U = 4.523478031		Gamma = -0.037723009"	Omega s = 3.606094599in ²			
Omega c = 2.950434446in ²		Omega s* = -3.534133911in ²	Omega c* = -2.125888348in ²			
Delta s = 0.000009630In ³ /Lb		Delta c = 0.000013390In ³ /Lb	Gamma b = 0.0			
Ps' = 0.0psi		Pt' = 0.0psi	P Gamma = -1289.701172psi			
PW = 0.0psi		Prim = 0.0psi	Es,w = 1.000000000			
Pe = -301.4211426psi		Q2 = 0.0#	Q3 = -0.060214549			
Fm = 0.100750893		Es,w1 = 0.0	lt = 0.0"			
Rt = 0.0"		Ft = 0.0	Ct = 0.0			
Fs = 0.0		Stb = 0.0psi	Ft,min = 0.164018631			
Sigmat,1 = 241.8973694psi		Sigmat,2 = 2523.736572psi	Ft,max = 1.711220860			
Sigma s,m = -3663.295166psi		Sigma s,b = -10264.39941psi	Sigma s = 13927.69434psi			
Sigma c,m = 0.0psi		Sigma c,b = 11476.28613psi	Sigma c = 11476.28613psi			
Thermal Expansion Acting on this Case (Yes)						
Sigma	-7765 psi	<= Sps	61430 psi	[OK		
Tau	-2752 psi	<= .8S	15928 psi	[OK		
Sigmat,max	2523 psi	<= 2St	26769 psi	[OK		
Sigma s	13927 psi	<= Spss	56489 psi	[OK		
Sigma c	11476 psi	<= Spssc	60020 psi	[OK		
Sigmatas,m	-3663 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmatas,m	-3663 psi	<= Ss,b	12939 psi	[OK		Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"	F-F Inside Tubesheets (L)	=	161.3750"
F-F Tubesheets (Lt)	=	167.5000"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.9094"	Channel thickness (Tc)	=	0.3125"
Channel I.D. (Dc)	=	23.3750"	Shell thickness (Ts)	=	0.4370"
Shell I.D. (Ds)	=	23.1260"			
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.18750"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+ Temp TS T = 515deg.F S = 19910psi Sy = 30715psi E = 26.9950psi V = .300 Sps = 61430psi						
Temp TBS Tt= 515deg.F St = 13384psi Syt= 21005psi Et = 26.9950psi Vt=.300						
Temp CH Tc= 390deg.F Sc = 17100psi Syc= 30010psi Ec = 27.7400psi Vc=.300 Spsc= 60020psi						
Temp SH Ts= 515deg.F Ss = 17100psi Sys= 28244psi Es = 26.9950psi Vs=.300 Spss= 56489psi						
Tt,m = 393deg.F At,m= 7.0860In/In						
Ts,m = 423deg.F As,m= 7.1460In/In						
Mu= 0.2000 d*= 0.6440" P*= 0.9375" Mu*= 0.3131 hg'= 0.0000" hgs'= 0.0000"						
W = 0.0# wj = 0.0" ao = 11.18677998"						
Rho s = 1.033630729 Rho c = 1.044759989 Chi s = 0.480848223						
Chi t = 0.685226977 Ks = 5411391.500Lb/In Kt = 29093.82812Lb/In						
Ks,t = 0.402592897 j = 1.000000000 Beta s = 0.566499650in ⁻¹						
Kappa s = 233740.8281# Lambda s = 4787501.000psi Beta c = 0.668145478in ⁻¹						
Kappa c = 103594.0859# Lambda c = 2599820.750psi h/p = 3.266666651						
E* = 8053920.000psi v* = 0.345361441 Xa = 2.644720793						
Zd = 0.077879079 Zv = 0.126476079 Zm = 0.564917266						
K = 1.072694778 F = 0.754432321 Phi = 1.014984131						
Q1 = -0.060214549 Qz1 = 1.718772292 Qz2 = 2.261739016						
U = 4.523478031 Gamma = -0.037723009" Omega s = 3.606094599in ²						
Omega c = 2.950434446in ² Omega s* = -3.534133911in ² Omega c* = -2.125888348in ²						
Delta s = 0.000009630In ³ /Lb Delta c = 0.000013390In ³ /Lb Gamma b = 0.0						
Ps' = 0.0psi Pt' = 718.6097412psi P Gamma = -1289.701172psi						
PW = 0.0psi Prim = -16.44434357psi Es,w = 1.000000000						
Pe = -473.2135315psi Q2 = 289.1478882# Q3 = -0.069979779						
Fm = 0.098549023 Es,w1 = 0.0 lt = 31.25000000"						
Rt = 0.237638906" Ft = 131.5020294 Ct = 159.2741089						
Fs = 2.000000000 Stb = 6166.893066psi Ft,min = 0.189634830						
Sigmat,1 = -278.4085999psi Sigmat,2 = 3175.720947psi Ft,max = 1.681457520						
Sigma s,m = -3150.328613psi Sigma s,b = -16360.01270psi Sigma s = 19510.34180psi						
Sigma c,m = 3949.005859psi Sigma c,b = 29131.36133psi Sigma c = 33080.36719psi						
Thermal Expansion Acting on this Case (Yes)						
Sigma -11924 psi <= Sps 61430 psi [OK]						
Tau -4321 psi <= .8S 15928 psi [OK]						
Sigmat,max 3175 psi <= 2St 26769 psi [OK]						
Sigmat,min -278 psi <= Stb 6166 psi [OK]						
Sigma s 19510 psi <= Spss 56489 psi [OK]						
Sigma c 33080 psi <= Spsc 60020 psi [OK]						
Sigmas,m -3150 psi <= 2Ss 34200 psi [OK] Step 10						
Sigmas,m -3150 psi <= Ss,b 12939 psi [OK] Step 10						

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(FTS) Configuration a
 Tubesheet Thickness (h) = 3.0625"
 F-F Tubesheets (Lt) = 167.5000" F-F Inside Tubesheets (L) = 161.3750"
 Exp length of tube (ltx) = 2.9094" Exp depth ratio (Rho) = 0.9500"
 Channel I.D. (Dc) = 23.3750" Channel thickness (Tc) = 0.3125"
 Shell I.D. (Ds) = 23.1260" Shell thickness (Ts) = 0.4370"
 Press Chan (Pt) = 0.0psi Press Shell (Ps) = 464.0psi
 Corr Chan = 0.18750" Corr Shell = 0.12500"
 Corr Chan side TS (ct) = 0.18750" Corr Shell side TS (cs) = 0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spssc = 60020psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	
Tt,m = 393deg.F			At,m = 7.0860In/In			
Ts,m = 423deg.F			As,m = 7.1460In/In			
Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"	
W = 0.0#	wj = 0.0"		ao = 11.18677998"			
Rho s = 1.033630729	Rho c = 1.044759989		Chi s = 0.480848223			
Chi t = 0.685226977	Ks = 5411391.500Lb/In		Kt = 29093.82812Lb/In			
Ks,t = 0.402592897	j = 1.000000000		Beta s = 0.566499650in ⁻¹			
Kappa s = 233740.8281#	Lambda s = 4787501.000psi		Beta c = 0.668145478in ⁻¹			
Kappa c = 103594.0859#	Lambda c = 2599820.750psi		h/p = 3.266666651			
E* = 8053920.000psi	v* = 0.345361441		Xa = 2.644720793			
Zd = 0.077879079	Zv = 0.126476079		Zm = 0.564917266			
K = 1.072694778	F = 0.754432321		Phi = 1.014984131			
Q1 = -0.060214549	Qz1 = 1.718772292		Qz2 = 2.261739016			
U = 4.523478031	Gamma = -0.037723009"		Omega s = 3.606094599in ²			
Omega c = 2.950434446in ²	Omega s* = -3.534133911in ²		Omega c* = -2.125888348in ²			
Delta s = 0.000009630In ³ /Lb	Delta c = 0.000013390In ³ /Lb		Gamma b = 0.0			
Ps' = 1027.633179psi	Pt' = 0.0psi		P Gamma = -1289.701172psi			
PW = 0.0psi	Prim = 59.27386856psi		Es,w = 1.000000000			
Pe = -47.39581299psi	Q2 = -1042.237671#		Q3 = 0.291221738			
Fm = 0.213436186	Es,w1 = 0.0		lt = 0.0"			
Rt = 0.0"	Ft = 0.0		Ct = 0.0			
Fs = 0.0	Stb = 0.0psi		Ft,min = -0.757870197			
Sigmat,1 = 915.9156494psi	Sigmat,2 = 1736.901245psi		Ft,max = 2.782358408			
Sigma s,m = -190.3430939psi	Sigma s,b = 12649.70801psi		Sigma s = 12840.05078psi			
Sigma c,m = 0.0psi	Sigma c,b = 5679.314941psi		Sigma c = 5679.314941psi			
Thermal Expansion Acting on this Case (Yes)						
Sigma	-2586 psi	<= Sps	61430 psi	[OK		
Tau	-432 psi	<= .8S	15928 psi	[OK		
Sigmat,max	1736 psi	<= 2St	26769 psi	[OK		
Sigma s	12840 psi	<= Spss	56489 psi	[OK		
Sigma c	5679 psi	<= Spssc	60020 psi	[OK		
Sigmas,m	-190 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmas,m	-190 psi	<= Ss,b	12939 psi	[OK		Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"	F-F Inside Tubesheets (L)	=	161.3750"
F-F Tubesheets (Lt)	=	167.5000"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.9094"	Channel thickness (Tc)	=	0.3125"
Channel I.D. (Dc)	=	23.3750"	Shell thickness (Ts)	=	0.4370"
Shell I.D. (Ds)	=	23.1260"			
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	=	464.0psi
Corr Chan	=	0.18750"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	=	0.12500"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+						
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 26.9950psi	Vt=.300		
Temp CH Tc= 390deg.F	Sc = 17100psi	Syc= 30010psi	Ec = 27.7400psi	Vc=.300	Spssc= 60020psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	
Tt,m = 393deg.F			At,m= 7.0860In/In			
Ts,m = 423deg.F			As,m= 7.1460In/In			
Mu= 0.2000 d*= 0.6440"	P*= 0.9375"	Mu*= 0.3131 hg'= 0.0000"	hgs'= 0.0000"			
W = 0.0#	wj = 0.0"	ao = 11.18677998"				
Rho s = 1.033630729	Rho c = 1.044759989	Chi s = 0.480848223				
Chi t = 0.685226977	Ks = 5411391.500Lb/In	Kt = 29093.82812Lb/In				
Ks,t = 0.402592897	j = 1.000000000	Beta s = 0.566499650in ⁻¹				
Kappa s = 233740.8281#	Lambda s = 4787501.000psi	Beta c = 0.668145478in ⁻¹				
Kappa c = 103594.0859#	Lambda c = 2599820.750psi	h/p = 3.266666651				
E* = 8053920.000psi	v* = 0.345361441	Xa = 2.644720793				
Zd = 0.077879079	Zv = 0.126476079	Zm = 0.564917266				
K = 1.072694778	F = 0.754432321	Phi = 1.014984131				
Q1 = -0.060214549	Qz1 = 1.718772292	Qz2 = 2.261739016				
U = 4.523478031	Gamma = -0.037723009"	Omega s = 3.606094599in ²				
Omega c = 2.950434446in ²	Omega s* = -3.534133911in ²	Omega c* = -2.125888348in ²				
Delta s = 0.000009630In ³ /Lb	Delta c = 0.000013390In ³ /Lb	Gamma b = 0.0				
Ps' = 1027.633179psi	Pt' = 718.6097412psi	P Gamma = -1289.701172psi				
PW = 0.0psi	Prim = 42.82952499psi	Es,w = 1.000000000				
Pe = -219.1882019psi	Q2 = -753.0898438#	Q3 = -0.005304800				
Fm = 0.113132007	Es,w1 = 0.0	lt = 0.0"				
Rt = 0.0"	Ft = 0.0	Ct = 0.0				
Fs = 0.0	Stb = 0.0psi	Ft,min = 0.019979211				
Sigmat,1 = 395.6096497psi	Sigmat,2 = 2388.885498psi	Ft,max = 1.878579497				
Sigma s,m = 322.6234741psi	Sigma s,b = 6554.093750psi	Sigma s = 6876.717285psi				
Sigma c,m = 3949.005859psi	Sigma c,b = 23334.38867psi	Sigma c = 27283.39453psi				
Thermal Expansion Acting on this Case (Yes)						
Sigma	-6340 psi	<= Sps	61430 psi	[OK		
Tau	-2001 psi	<= .8S	15928 psi	[OK		
Sigmat,max	2388 psi	<= 2St	26769 psi	[OK		
Sigma s	6876 psi	<= Spss	56489 psi	[OK		
Sigma c	27283 psi	<= Spssc	60020 psi	[OK		
Sigmatas,m	322 psi	<= 2Ss	34200 psi	[OK		] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.3750"	F-F Inside Tubesheets (L)	=	161.1250"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	3.2062"	Channel thickness (Tc)	=	0.5000"
Channel I.D. (Dc)	=	23.0000"	Shell thickness (Ts)	=	0.5620"
Shell I.D. (Ds)	=	22.8760"			
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 26.9950psi	Vt=.300		
Temp CH Tc= 390deg.F	Sc = 17100psi	Syc= 30010psi	Ec = 27.7400psi	Vc=.300	Spssc= 60020psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	

Mu= 0.2000 d*= 0.6440" P*= 0.9375" Mu*= 0.3131 hg'= 0.0000" hgs'= 0.0000"
 W = 0.0# wj = 0.0" ao = 11.18677998"
 Rho s = 1.022456884 Rho c = 1.027999163 Chi s = 0.480848223
 Chi t = 0.685226977 Ks = 6933094.500Lb/In Kt = 29138.97070Lb/In
 Ks,t = 0.515004396 j = 1.000000000 Beta s = 0.500872552in⁻¹
 Kappa s = 439568.7500# Lambda s = 6464793.000psi Beta c = 0.530318439in⁻¹
 Kappa c = 336791.0938# Lambda c = 5309300.000psi h/p = 3.599999905
 E* = 8053920.000psi v* = 0.345361441 Xa = 2.459800720
 Zd = 0.095104538 Zv = 0.136877611 Zm = 0.596329451
 K = 1.072694778 F = 1.110997796 Phi = 1.494693637
 Q1 = -0.096299022 Qz1 = 1.499606967 Qz2 = 1.454362631
 U = 2.908725262 Gamma = 0.0" Omega s = 0.0in²
 Omega c = 0.0in² Omega s* = -4.407473087in² Omega c* = -3.754968882in²
 Delta s = 0.000007330in³/Lb Delta c = 0.000008100in³/Lb Gamma b = 0.0
 Ps' = 0.0psi Pt' = 602.5858765psi P Gamma = 0.0psi
 PW = 0.0psi Prim = -18.67723656psi Es,w = 1.000000000
 Pe = -178.8322906psi Q2 = 424.8668518# Q3 = -0.134267807
 Fm = 0.094128624 Es,w1 = 0.0 lt = 31.25000000"
 Rt = 0.237638906 Ft = 131.5020294 Ct = 159.2741089
 Fs = 2.000000000 Stb = 6166.893066psi Ft,min = 0.468373537
 Sigmat,1 = -307.6555481psi Sigmat,2 = 511.4367981psi Ft,max = 1.404474497
 Sigma s,m = 334.1159363psi Sigma s,b = -3917.888428psi Sigma s = 4252.004395psi
 Sigma c,m = 2408.638184psi Sigma c,b = 11152.63965psi Sigma c = 13561.27734psi
 Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas,Kappas,Lambdas,Deltas,OmeGas=0 Betac,Kappac,Lambdac,Deltac,OmeGac=0
 F = 0.153976202 Phi = 0.207153648 Q1 = -0.005249350
 Qz1 = 1.727735877 Qz2 = 2.448243618 U = 4.896487236
 Omega s* = 0.031910121in² Omega c* = 0.396531820in² Prim = 3.320215225psi
 Pe = -160.9005890psi Q2 = -75.52773285# Q3 = 0.002252500
 Sigma = | -4448 psi |

Sigma	-3544 psi	<= 1.5S	29865 psi	[OK Per Simply Sup]
Tau	-1481 psi	<= .8S	15928 psi	[OK]
Sigmat,max	511 psi	< St	13384 psi	[OK]
Sigmat,min	-307 psi	<= Stb	6166 psi	[OK]
Sigma s	4252 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	13561 psi	<= Spsc	60020 psi	[OK Per Simply Sup]
Sigmat,m	334 psi	<= SsEs,w	17100 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.3750"	F-F Inside Tubesheets (L)	=	161.1250"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	3.2062"	Channel thickness (Tc)	=	0.5000"
Channel I.D. (Dc)	=	23.0000"	Shell thickness (Ts)	=	0.5620"
Shell I.D. (Ds)	=	22.8760"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	464.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spac = 60020psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	

Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"
W = 0.0#	wj = 0.0"	ao = 11.18677998"			
Rho s = 1.022456884	Rho c = 1.027999163	Chi s = 0.480848223			
Chi t = 0.685226977	Ks = 6933094.500Lb/In	Kt = 29138.97070Lb/In			
Ks,t = 0.515004396	j = 1.000000000	Beta s = 0.500872552in ⁻¹			
Kappa s = 439568.7500#	Lambda s = 6464793.000psi	Beta c = 0.530318439in ⁻¹			
Kappa c = 336791.0938#	Lambda c = 5309300.000psi	h/p = 3.599999905			
E* = 8053920.000psi	v* = 0.345361441	Xa = 2.459800720			
Zd = 0.095104538	Zv = 0.136877611	Zm = 0.596329451			
K = 1.072694778	F = 1.110997796	Phi = 1.494693637			
Q1 = -0.096299022	Qz1 = 1.499606967	Qz2 = 1.454362631			
U = 2.908725262	Gamma = 0.0"	Omega s = 0.0in ²			
Omega c = 0.0in ²	Omega s* = -4.407473087in ²	Omega c* = -3.754968882in ²			
Delta s = 0.000007330In ³ /Lb	Delta c = 0.000008100In ³ /Lb	Gamma b = 0.0			
Ps' = 891.8553467psi	Pt' = 0.0psi	P Gamma = 0.0psi			
PW = 0.0psi	Prim = 47.53353882psi	Es,w = 1.000000000			
Pe = 270.4056702psi	Q2 = -1081.285522#	Q3 = -0.160205424			
Fm = 0.087062247	Es,w1 = 0.0	lt = 31.25000000"			
Rt = 0.237638906"	Ft = 131.5020294	Ct = 159.2741089			
Fs = 2.000000000	Stb = 6166.893066psi	Ft,min = 0.525762796			
Sigmat,1 = 396.0506287psi	Sigmat,2 = -680.5560303psi	Ft,max = 1.339486599			
Sigma s,m = 2769.246582psi	Sigma s,b = 16868.38086psi	Sigma s = 19637.62695psi			
Sigma c,m = 0.0psi	Sigma c,b = -4091.201172psi	Sigma c = 4091.201172psi			

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas=0		Betac, Kappac, Lambdac, Deltac, Omegac=0			
F	= 0.153976202	Phi	= 0.207153648	Q1	=-0.005249350
Qz1	= 1.727735877	Qz2	= 2.448243618	U	= 4.896487236
Omega s*	= 0.031910121in^2	Omega c*	= 0.396531820in^2	Prim	=-0.579323113psi
Pe	= 239.3042755psi	Q2	= 13.17835045#	Q3	=-0.004369250

Sigma	4956 psi	<= 1.5S	29865 psi	[OK Per Simply Sup]
Tau	2240 psi	<= .8S	15928 psi	[OK]
Sigmat,max	680 psi	< St	13384 psi	[OK]
Sigmat,min	-680 psi	<= Stb	6166 psi	[OK]
Sigma s	19637 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	4091 psi	<= Spsc	60020 psi	[OK Per Simply Sup]
Sigmat,s,m	2769 psi	<= SsEs,w	17100 psi	[OK] Step 10

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.3750"	F-F Inside Tubesheets (L)	=	161.1250"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	3.2062"	Channel thickness (Tc)	=	0.5000"
Channel I.D. (Dc)	=	23.0000"	Shell thickness (Ts)	=	0.5620"
Shell I.D. (Ds)	=	22.8760"			
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	=	464.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spssc = 60020psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	

Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"
W =	0.0#	wj =	0.0"	ao =	11.18677998"
Rho s =	1.022456884	Rho c =	1.027999163	Chi s =	0.480848223
Chi t =	0.685226977	Ks =	6933094.500Lb/In	Kt =	29138.97070Lb/In
Ks, t =	0.515004396	j =	1.000000000	Beta s =	0.500872552in ⁻¹
Kappa s =	439568.7500#	Lambda s =	6464793.000psi	Beta c =	0.530318439in ⁻¹
Kappa c =	336791.0938#	Lambda c =	5309300.000psi	h/p =	3.599999905
E* =	8053920.000psi	v* =	0.345361441	Xa =	2.459800720
Zd =	0.095104538	Zv =	0.136877611	Zm =	0.596329451
K =	1.072694778	F =	1.110997796	Phi =	1.494693637
Q1 =	-0.096299022	Qz1 =	1.499606967	Qz2 =	1.454362631
U =	2.908725262	Gamma =	0.0"	Omega s =	0.0in ²
Omega c =	0.0in ²	Omega s* =	-4.407473087in ²	Omega c* =	-3.754968882in ²
Delta s =	0.000007330in ³ /Lb	Delta c =	0.000008100in ³ /Lb	Gamma b =	0.0
Ps' =	891.8553467psi	Pt' =	602.5858765psi	P Gamma =	0.0psi
PW =	0.0psi	Prim =	28.85630417psi	Es, w =	1.000000000
Pe =	91.57337952psi	Q2 =	-656.4186401#	Q3 =	-0.210858613
Fm =	0.105429307	Es, w1 =	0.0	lt =	31.25000000"
Rt =	0.237638906"	Ft =	131.5020294	Ct =	159.2741089
Fs =	2.000000000	Stb =	6166.893066psi	Ft, min =	0.637837410
Sigmat, 1 =	88.39507294psi	Sigmat, 2 =	-169.1192780psi	Ft, max =	1.212572813
Sigma s, m =	3103.362305psi	Sigma s, b =	12950.49219psi	Sigma s =	16053.85449psi
Sigma c, m =	2408.638184psi	Sigma c, b =	7061.438477psi	Sigma c =	9470.076172psi

Thermal Expansion Acting on this Case (No)

/// Sigma per Simply Supported Calculations ///

Betas, Kappas, Lambdas, Deltas, Omegas=0		Betac, Kappac, Lambdac, Deltac, Omegac=0			
F	= 0.153976202	Phi	= 0.207153648	Q1	= -0.005249350
Qz1	= 1.727735877	Qz2	= 2.448243618	U	= 4.896487236
Omega s*	= 0.031910121in^2	Omega c*	= 0.396531820in^2	Prim	= 2.740892172psi
Pe	= 78.40368652psi	Q2	= -62.34938049#	Q3	= -0.017958470

Sigma	2032 psi	<= 1.5S	29865 psi	[OK Per Simply Sup]
Tau	758 psi	<= .8S	15928 psi	[OK]
Sigmat, max	169 psi	< St	13384 psi	[OK]
Sigmat, min	-169 psi	<= Stb	6166 psi	[OK]
Sigma s	16053 psi	<= Spss	56489 psi	[OK Per Simply Sup]
Sigma c	9470 psi	<= Spssc	60020 psi	[OK Per Simply Sup]
Sigmat, m	3103 psi	<= SsEs, w	17100 psi	[OK] Step 10

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(FTS) Configuration a

Tubeheet Thickness (h)	=	3.3750"	F-F Inside Tubesheets (L)	=	161.1250"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	3.2062"	Channel thickness (Tc)	=	0.5000"
Channel I.D. (Dc)	=	23.0000"	Shell thickness (Ts)	=	0.5620"
Shell I.D. (Ds)	=	22.8760"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	0.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+						
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 26.9950psi	Vt=.300		
Temp CH Tc= 390deg.F	Sc = 17100psi	Syc= 30010psi	Ec = 27.7400psi	Vc=.300	Spssc= 60020psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	
Tt,m = 393deg.F			At,m= 7.0860In/In			
Ts,m = 423deg.F			As,m= 7.1460In/In			
Mu= 0.2000 d*= 0.6440"	P*= 0.9375"	Mu*= 0.3131	hg'= 0.0000"	hgs'= 0.0000"		
W = 0.0#	wj = 0.0"	ao = 11.18677998"				
Rho s = 1.022456884	Rho c = 1.027999163	Chi s = 0.480848223				
Chi t = 0.685226977	Ks = 6933094.500Lb/In	Kt = 29138.97070Lb/In				
Ks,t = 0.515004396	j = 1.000000000	Beta s = 0.500872552in ⁻¹				
Kappa s = 439568.7500#	Lambda s = 6464793.000psi	Beta c = 0.530318439in ⁻¹				
Kappa c = 336791.0938#	Lambda c = 5309300.000psi	h/p = 3.599999905				
E* = 8053920.000psi	v* = 0.345361441	Xa = 2.459800720				
Zd = 0.095104538	Zv = 0.136877611	Zm = 0.596329451				
K = 1.072694778	F = 1.110997796	Phi = 1.494693637				
Q1 = -0.096299022	Qz1 = 1.499606967	Qz2 = 1.454362631				
U = 2.908725262	Gamma = -0.037664570"	Omega s = 4.439383030in ²				
Omega c = 4.151500702in ²	Omega s* = -4.407473087in ²	Omega c* = -3.754968882in ²				
Delta s = 0.000007330In ³ /Lb	Delta c = 0.000008100In ³ /Lb	Gamma b = 0.0				
Ps' = 0.0psi	Pt' = 0.0psi	P Gamma = -1289.701294psi				
PW = 0.0psi	Prim = 0.0psi	Es,w = 1.000000000				
Pe = -371.2440796psi	Q2 = 0.0#	Q3 = -0.096299022				
Fm = 0.104472697	Es,w1 = 0.0	lt = 0.0"				
Rt = 0.0"	Ft = 0.0	Ct = 0.0				
Fs = 0.0	Stb = 0.0psi	Ft,min = 0.384364277				
Sigmat,1 = 698.1790161psi	Sigmat,2 = 2723.963135psi	Ft,max = 1.499606967				
Sigma s,m = -3527.060303psi	Sigma s,b = -8983.994141psi	Sigma s = 12511.05469psi				
Sigma c,m = 0.0psi	Sigma c,b = 10794.48633psi	Sigma c = 10794.48633psi				
Thermal Expansion Acting on this Case (Yes)						
Sigma	-8166 psi	<= Sps	61430 psi	[OK		
Tau	-3076 psi	<= .8S	15928 psi	[OK		
Sigmat,max	2723 psi	<= 2St	26769 psi	[OK		
Sigma s	12511 psi	<= Spss	56489 psi	[OK		
Sigma c	10794 psi	<= Spssc	60020 psi	[OK		
Sigmas,m	-3527 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmas,m	-3527 psi	<= Ss,b	13400 psi	[OK		Step 10

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Job No.: X-7929

REV 1

By: MD/DN

3/9/2011

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.3750"	F-F Inside Tubesheets (L)=	161.1250"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	= 0.9500"
Exp length of tube (ltx)	=	3.2062"	Channel thickness (Tc)	= 0.5000"
Channel I.D. (Dc)	=	23.0000"	Shell thickness (Ts)	= 0.5620"
Shell I.D. (Ds)	=	22.8760"		
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	= 0.0psi
Corr Chan	=	0.00000"	Corr Shell	= 0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	= 0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+						
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 26.9950psi	Vt=.300		
Temp CH Tc= 390deg.F	Sc = 17100psi	Syc= 30010psi	Ec = 27.7400psi	Vc=.300	Spssc= 60020psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	
Tt,m = 393deg.F			At,m= 7.0860In/In			
Ts,m = 423deg.F			As,m= 7.1460In/In			
Mu= 0.2000 d*= 0.6440"	P*= 0.9375"	Mu*= 0.3131	hg'= 0.0000"	hgs'= 0.0000"		
W = 0.0#	wj = 0.0"	ao = 11.18677998"				
Rho s = 1.022456884	Rho c = 1.027999163	Chi s = 0.480848223				
Chi t = 0.685226977	Ks = 6933094.500Lb/In	Kt = 29138.97070Lb/In				
Ks,t = 0.515004396	j = 1.000000000	Beta s = 0.500872552in ⁻¹				
Kappa s = 439568.7500#	Lambda s = 6464793.000psi	Beta c = 0.530318439in ⁻¹				
Kappa c = 336791.0938#	Lambda c = 5309300.000psi	h/p = 3.599999905				
E* = 8053920.000psi	v* = 0.345361441	Xa = 2.459800720				
Zd = 0.095104538	Zv = 0.136877611	Zm = 0.596329451				
K = 1.072694778	F = 1.110997796	Phi = 1.494693637				
Q1 = -0.096299022	Qz1 = 1.499606967	Qz2 = 1.454362631				
U = 2.908725262	Gamma = -0.037664570"	Omega s = 4.439383030in ²				
Omega c = 4.151500702in ²	Omega s* = -4.407473087in ²	Omega c* = -3.754968882in ²				
Delta s = 0.000007330In ³ /Lb	Delta c = 0.000008100In ³ /Lb	Gamma b = 0.0				
Ps' = 0.0psi	Pt' = 602.5858765psi	P Gamma = -1289.701294psi				
PW = 0.0psi	Prim = -18.67723656psi	Es,w = 1.000000000				
Pe = -550.0763550psi	Q2 = 424.8668518#	Q3 = -0.108642846				
Fm = 0.101109803	Es,w1 = 0.0	lt = 0.0"				
Rt = 0.0"	Ft = 0.0	Ct = 0.0				
Fs = 0.0	Stb = 0.0psi	Ft,min = 0.411676079				
Sigmat,1 = 390.5234985psi	Sigmat,2 = 3235.399902psi	Ft,max = 1.468678951				
Sigma s,m = -3192.944336psi	Sigma s,b = -12901.88281psi	Sigma s = 16094.82715psi				
Sigma c,m = 2408.638184psi	Sigma c,b = 21947.12500psi	Sigma c = 24355.76367psi				
Thermal Expansion Acting on this Case (Yes)						
Sigma	-11710 psi	<= Sps	61430 psi	[OK		
Tau	-4558 psi	<= .8S	15928 psi	[OK		
Sigmat,max	3235 psi	<= 2St	26769 psi	[OK		
Sigma s	16094 psi	<= Spss	56489 psi	[OK		
Sigma c	24355 psi	<= Spssc	60020 psi	[OK		
Sigmas,m	-3192 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmas,m	-3192 psi	<= Ss,b	13400 psi	[OK		Step 10

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REV 1

By: MD/DN

3/9/2011

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.3750"	F-F Inside Tubesheets (L)	=	161.1250"
F-F Tubesheets (Lt)	=	167.8750"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	3.2062"	Channel thickness (Tc)	=	0.5000"
Channel I.D. (Dc)	=	23.0000"	Shell thickness (Ts)	=	0.5620"
Shell I.D. (Ds)	=	22.8760"			
Press Chan (Pt)	=	0.0psi	Press Shell (Ps)	=	464.0psi
Corr Chan	=	0.00000"	Corr Shell	=	0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	=	0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+						
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt= 515deg.F	St = 13384psi	Syt= 21005psi	Et = 26.9950psi	Vt=.300		
Temp CH Tc= 390deg.F	Sc = 17100psi	Syc= 30010psi	Ec = 27.7400psi	Vc=.300	Spssc= 60020psi	
Temp SH Ts= 515deg.F	Ss = 17100psi	Sys= 28244psi	Es = 26.9950psi	Vs=.300	Spss= 56489psi	
Tt,m = 393deg.F			At,m= 7.0860In/In			
Ts,m = 423deg.F			As,m= 7.1460In/In			
Mu= 0.2000 d*= 0.6440"	P*= 0.9375"	Mu*= 0.3131 hg'= 0.0000"	hgs'= 0.0000"			
W = 0.0#	wj = 0.0"	ao = 11.18677998"	Chi s = 0.480848223			
Rho s = 1.022456884	Rho c = 1.027999163	Kt = 29138.97070Lb/In	Beta s = 0.500872552in ⁻¹			
Chi t = 0.685226977	Ks = 6933094.500Lb/In	Beta c = 0.530318439in ⁻¹	h/p = 3.599999905			
Ks,t = 0.515004396	j = 1.000000000	Xa = 2.459800720	Zm = 0.596329451			
Kappa s = 439568.7500#	Lambda s = 6464793.000psi	Phi = 1.494693637	Qz2 = 1.454362631			
Kappa c = 336791.0938#	Lambda c = 5309300.000psi	Omega s = 4.439383030in ²	Omega c*=-3.754968882in ²			
E* = 8053920.000psi	v* = 0.345361441	Gamma b = 0.0	P Gamma = -1289.701294psi			
Zd = 0.095104538	Zv = 0.136877611	Es,w = 1.000000000	Q3 = 0.075070783			
K = 1.072694778	F = 1.110997796	lt = 0.0"	Ct = 0.0			
Q1 = -0.096299022	Qz1 = 1.499606967	Ft,min = 0.005193600	Ft,max = 1.928981662			
U = 2.908725262	Gamma = -0.037664570"	Sigma s = 8642.201172psi	Sigma c = 6703.284180psi			
Omega c = 4.151500702in ²	Omega s* = -4.407473087in ²					
Delta s = 0.000007330In ³ /Lb	Delta c = 0.000008100In ³ /Lb					
Ps' = 891.8553467psi	Pt' = 0.0psi					
PW = 0.0psi	Prim = 47.53353882psi					
Pe = -100.8383865psi	Q2 = -1081.285522#					
Fm = 0.151160136	Es,w1 = 0.0					
Rt = 0.0"	Ft = 0.0					
Fs = 0.0	Stb = 0.0psi					
Sigmat,1 = 1094.229614psi	Sigmat,2 = 2043.406982psi					
Sigma s,m=-757.8136597psi	Sigma s,b= 7884.387207psi					
Sigma c,m= 0.0psi	Sigma c,b= 6703.284180psi					
Thermal Expansion Acting on this Case (Yes)						
Sigma	-3209 psi	<= Sps	61430 psi	[OK		
Tau	-835 psi	<= .8S	15928 psi	[OK		
Sigmat,max	2043 psi	<= 2St	26769 psi	[OK		
Sigma s	8642 psi	<= Spss	56489 psi	[OK		
Sigma c	6703 psi	<= Spssc	60020 psi	[OK		
Sigmas,m	-757 psi	<= 2Ss	34200 psi	[OK		Step 10
Sigmas,m	-757 psi	<= Ss,b	13400 psi	[OK		Step 10

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REV 1

By: MD/DN

3/9/2011

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.3750"		
F-F Tubesheets (Lt)	=	167.8750"	F-F Inside Tubesheets (L)	= 161.1250"
Exp length of tube (ltx)	=	3.2062"	Exp depth ratio (Rho)	= 0.9500"
Channel I.D. (Dc)	=	23.0000"	Channel thickness (Tc)	= 0.5000"
Shell I.D. (Ds)	=	22.8760"	Shell thickness (Ts)	= 0.5620"
Press Chan (Pt)	=	214.0psi	Press Shell (Ps)	= 464.0psi
Corr Chan	=	0.00000"	Corr Shell	= 0.00000"
Corr Chan side TS (ct)	=	0.00000"	Corr Shell side TS (cs)	= 0.00000"

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 515deg.F	S = 19910psi	Sy = 30715psi	E = 26.9950psi	V = .300	Sps = 61430psi	
Temp TBS Tt = 515deg.F	St = 13384psi	Syt = 21005psi	Et = 26.9950psi	Vt = .300		
Temp CH Tc = 390deg.F	Sc = 17100psi	Syc = 30010psi	Ec = 27.7400psi	Vc = .300	Spssc = 60020psi	
Temp SH Ts = 515deg.F	Ss = 17100psi	Sys = 28244psi	Es = 26.9950psi	Vs = .300	Spss = 56489psi	
Tt, m = 393deg.F				At, m = 7.0860In/In		
Ts, m = 423deg.F				As, m = 7.1460In/In		
Mu = 0.2000	d* = 0.6440"	P* = 0.9375"	Mu* = 0.3131	hg' = 0.0000"	hgs' = 0.0000"	
W = 0.0#		wj = 0.0"	ao = 11.18677998"			
Rho s = 1.022456884		Rho c = 1.027999163	Chi s = 0.480848223			
Chi t = 0.685226977		Ks = 6933094.500Lb/In	Kt = 29138.97070Lb/In			
Ks, t = 0.515004396		j = 1.000000000	Beta s = 0.500872552in ⁻¹			
Kappa s = 439568.7500#		Lambda s = 6464793.000psi	Beta c = 0.530318439in ⁻¹			
Kappa c = 336791.0938#		Lambda c = 5309300.000psi	h/p = 3.599999905			
E* = 8053920.000psi		v* = 0.345361441	Xa = 2.459800720			
Zd = 0.095104538		Zv = 0.136877611	Zm = 0.596329451			
K = 1.072694778		F = 1.110997796	Phi = 1.494693637			
Q1 = -0.096299022		Qz1 = 1.499606967	Qz2 = 1.454362631			
U = 2.908725262		Gamma = -0.037664570"	Omega s = 4.439383030in ²			
Omega c = 4.151500702in ²		Omega s* = -4.407473087in ²	Omega c* = -3.754968882in ²			
Delta s = 0.000007330In ³ /Lb		Delta c = 0.000008100In ³ /Lb	Gamma b = 0.0			
Ps' = 891.8553467psi		Pt' = 602.5858765psi	P Gamma = -1289.701294psi			
PW = 0.0psi		Prim = 28.85630417psi	Es, w = 1.000000000			
Pe = -279.6706848psi		Q2 = -656.4186401#	Q3 = -0.058788441			
Fm = 0.114691973		Es, w1 = 0.0	lt = 0.0"			
Rt = 0.0"		Ft = 0.0	Ct = 0.0			
Fs = 0.0		Stb = 0.0psi	Ft, min = 0.301368862			
Sigmat, 1 = 786.5740967psi		Sigmat, 2 = 2554.843750psi	Ft, max = 1.593591332			
Sigma s, m = -423.6978149psi		Sigma s, b = 3966.498535psi	Sigma s = 4390.196289psi			
Sigma c, m = 2408.638184psi		Sigma c, b = 17855.92383psi	Sigma c = 20264.56250psi			
Thermal Expansion Acting on this Case (Yes)						
Sigma	-6753 psi	<= Sps	61430 psi	[OK		
Tau	-2317 psi	<= .8S	15928 psi	[OK		
Sigmat, max	2554 psi	<= 2St	26769 psi	[OK		
Sigma s	4390 psi	<= Spss	56489 psi	[OK		
Sigma c	20264 psi	<= Spssc	60020 psi	[OK		
Sigmas, m	-423 psi	<= 2Ss	34200 psi	[OK		] Step 10
Sigmas, m	-423 psi	<= Ss, b	13400 psi	[OK		] Step 10

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REV 1

By: MD/DN

3/9/2011

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"	F-F Inside Tubesheets (L)	=	161.3750"
F-F Tubesheets (Lt)	=	167.5000"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.9094"	Channel thickness (Tc)	=	0.3125"
Channel I.D. (Dc)	=	23.3750"	Shell thickness (Ts)	=	0.4370"
Shell I.D. (Ds)	=	23.1260"			

Press Chan (Pt)	=	-15.0psi	Press Shell (Ps)	=	15.0psi	3
Corr Chan	=	0.18750"	Corr Shell	=	0.12500"	
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	=	0.12500"	

Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
Temp TS T = 250deg.F	S = 20000psi	Sy = 34200psi	E = 28.3500psi	V = .300	Sps = 68400psi	
Temp TBS Tt = 250deg.F	St = 13400psi	Syt = 23400psi	Et = 28.3500psi	Vt = .300		
Temp CH Tc = 250deg.F	Sc = 17100psi	Syc = 31500psi	Ec = 28.3500psi	Vc = .300	Spss = 63000psi	
Temp SH Ts = 250deg.F	Ss = 17100psi	Sys = 31500psi	Es = 28.3500psi	Vs = .300	Spss = 63000psi	
Tt, m = 250deg.F				At, m = 6.8000In/In		
Ts, m = 250deg.F				As, m = 6.8000In/In		
Mu = 0.2000	d* = 0.6443"	P* = 0.9375"	Mu* = 0.3127	hg' = 0.0000"	hgs' = 0.0000"	
W = 0.0#	wj = 0.0"		ao = 11.18677998"			
Rho s = 1.033630729	Rho c = 1.044759989		Chi s = 0.480848223			
Chi t = 0.685226977	Ks = 5683013.500Lb/In		Kt = 30554.17773Lb/In			
Ks, t = 0.402592897	j = 1.000000000		Beta s = 0.566499650in ⁻¹			
Kappa s = 245473.3281#	Lambda s = 5027807.000psi		Beta c = 0.668145478in ⁻¹			
Kappa c = 105872.1016#	Lambda c = 2656990.250psi		h/p = 3.266666651			
E* = 8443892.000psi	v* = 0.345594764		Xa = 2.645718098			
Zd = 0.077794857	Zv = 0.126402840		Zm = 0.564671695			
K = 1.072694778	F = 0.749756873		Phi = 1.008868933			
Q1 = -0.059816770	Qz1 = 1.720642805		Qz2 = 2.269223213			
U = 4.538446426	Gamma = 0.0"		Omega s = 3.606094599in ²			
Omega c = 2.950434446in ²	Omega s* = -3.534133911in ²		Omega c* = -2.125888348in ²			
Delta s = 0.000009170In ³ /Lb	Delta c = 0.000013110In ³ /Lb		Gamma b = 0.0			
Ps' = 33.22089767psi	Pt' = -50.36984253psi		P Gamma = 0.0psi			
PW = 0.0psi	Prim = 3.078976870psi		Es, w = 1.000000000			
Pe = 20.24587822psi	Q2 = -54.08768082#		Q3 = -0.102512218			
Fm = 0.091163047	Es, w1 = 0.0		lt = 31.25000000"			
Rt = 0.237638906"	Ft = 131.5020294		Ct = 154.6441498			
Fs = 2.000000000	Stb = 6725.438477psi		Ft, min = 0.274353087			
Sigmat, l = 58.40435410psi	Sigmat, 2 = -71.21263885psi		Ft, max = 1.582814932			
Sigma s, m = 76.22334290psi	Sigma s, b = 1169.741821psi		Sigma s = 1245.965210psi			
Sigma c, m = -276.7994690psi	Sigma c, b = -1411.112305psi		Sigma c = 1687.911743psi			
Thermal Expansion Acting on this Case (Yes)						
Sigma	472 psi	<= Sps	68400 psi	[OK		
Tau	184 psi	<= .8S	16000 psi	[OK		
Sigmat, max	71 psi	<= 2St	26800 psi	[OK		
Sigmat, min	-71 psi	<= Stb	6725 psi	[OK		
Sigma s	1245 psi	<= Spss	63000 psi	[OK		
Sigma c	1687 psi	<= Spss	63000 psi	[OK		
Sigmat, m	76 psi	<= 2Ss	34200 psi	[OK		
] Step 10						

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REV 3
By: MD/DN 6/23/2011

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(FTS) Configuration a

Tubesheet Thickness (h)	=	3.0625"	F-F Inside Tubesheets (L)	=	161.3750"
F-F Tubesheets (Lt)	=	167.5000"	Exp depth ratio (Rho)	=	0.9500"
Exp length of tube (ltx)	=	2.9094"	Channel thickness (Tc)	=	0.3125"
Channel I.D. (Dc)	=	23.3750"	Shell thickness (Ts)	=	0.4370"
Shell I.D. (Ds)	=	23.1260"			

Press Chan (Pt)	=	15.0psi	Press Shell (Ps)	=	-15.0psi
Corr Chan	=	0.18750"	Corr Shell	=	0.12500"
Corr Chan side TS (ct)	=	0.18750"	Corr Shell side TS (cs)	=	0.12500"



Temp	Stress	Yield	Modulus x10 ⁶	Coef x10 ⁻⁶	Poisson Ratio	Prim Plus Stress
+ Temp TS T = 250deg.F S = 20000psi Sy = 34200psi E = 28.3500psi V = .300 Sps = 68400psi						
Temp TBS Tt= 250deg.F St = 13400psi Syt= 23400psi Et = 28.3500psi Vt=.300						
Temp CH Tc= 250deg.F Sc = 17100psi Syc= 31500psi Ec = 28.3500psi Vc=.300 Spss= 63000psi						
Temp SH Ts= 250deg.F Ss = 17100psi Sys= 31500psi Es = 28.3500psi Vs=.300 Spss= 63000psi						
Tt,m = 250deg.F At,m= 6.8000In/In						
Ts,m = 250deg.F As,m= 6.8000In/In						
Mu= 0.2000 d*= 0.6443" P*= 0.9375" Mu*= 0.3127 hg'= 0.0000" hgs'= 0.0000"						
W = 0.0# wj = 0.0" ao = 11.18677998"						
Rho s = 1.033630729 Rho c = 1.044759989 Chi s = 0.480848223						
Chi t = 0.685226977 Ks = 5683013.500Lb/In Kt = 30554.17773Lb/In						
Ks,t = 0.402592897 j = 1.000000000 Beta s = 0.566499650in^-1						
Kappa s = 245473.3281# Lambda s = 5027807.000psi Beta c = 0.668145478in^-1						
Kappa c = 105872.1016# Lambda c = 2656990.250psi h/p = 3.266666651						
E* = 8443892.000psi v* = 0.345594764 Xa = 2.645718098						
Zd = 0.077794857 Zv = 0.126402840 Zm = 0.564671695						
K = 1.072694778 F = 0.749756873 Phi = 1.008868933						
Q1 = -0.059816770 Qz1 = 1.720642805 Qz2 = 2.269223213						
U = 4.538446426 Gamma = 0.0" Omega s = 3.606094599in^2						
Omega c = 2.950434446in^2 Omega s* = -3.534133911in^2 Omega c* = -2.125888348in^2						
Delta s = 0.000009170In^3/Lb Delta c = 0.000013110In^3/Lb Gamma b = 0.0						
Ps' = -33.22089767psi Pt' = 50.36984253psi P Gamma = 0.0psi						
PW = 0.0psi Prim = -3.078976870psi Es,w = 1.000000000						
Pe = -20.24587822psi Q2 = 54.08768082# Q3 = -0.102512218						
Fm = 0.091163047 Es,w1 = 0.0 lt = 31.25000000"						
Rt = 0.237638906" Ft = 131.5020294 Ct = 154.6441498						
Fs = 2.000000000 Stb = 6725.438477psi Ft,min = 0.274353087						
Sigmat,1 = -58.40435410psi Sigmat,2 = 71.21263885psi Ft,max = 1.582814932						
Sigma s,m = -76.22334290psi Sigma s,b = -1169.741821psi Sigma s = 1245.965210psi						
Sigma c,m = 276.7994690psi Sigma c,b = 1411.112305psi Sigma c = 1687.911743psi						
Thermal Expansion Acting on this Case (Yes)						
Sigma -472 psi <= Sps 68400 psi [OK]						
Tau -184 psi <= .8S 16000 psi [OK]						
Sigmat,max 71 psi <= 2St 26800 psi [OK]						
Sigmat,min -58 psi <= Stb 6725 psi [OK]						
Sigma s 1245 psi <= Spss 63000 psi [OK]						
Sigma c 1687 psi <= Spss 63000 psi [OK]						
Sigmas,m -76 psi <= 2Ss 34200 psi [OK] Step 10						
Sigmas,m -76 psi <= Ss,b 17030 psi [OK] Step 10						

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***** TUBE CALCULATIONS *****

Tube size.....=	0.75000"	Straight or U-Tube.=	Straight Tube
Internal Press...=	214.00000psig*	External press.....=	479.00000psig*
Temperature used.=	515.00000deg.	Tube Matl MT# 34...=	SA-179
Stress Oper.....=	13384.99902psi	Stress Atm.....=	13400.00000psi
Yield.....=	21005.00195psi	Inside Radius.....=	0.29200"
Joint Eff.....=	1.00000	Mill tol...= 0.00% =	0.00000"
Wall Thk. t.....=	0.08300"	Tube gauge.....=	
Mean Rad., R.....=	0.00000"	Tube Length.....=	168.00000"
Corrosion ID.....=	0.00000"	Corrosion OD.....=	0.00000"
Thinning factor..=	1.00000		



tr = PR/SE - .6P = 0.00471" (tr*TF)+MT+CA.....= 0.00471"

MAWP Internal H/C=	3250.30688psig	MAWP Internal N/C..=	3253.94971psig
MAWP External H/C=	1928.78552psig	MAWP External N/C..=	2159.90234psig

* DESIGN PRESSURE HAS BEEN INCREASED BY 9 PSI DUE TO STATIC HEAD

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Customer :	SOUTHERN / MISS. POWER CO.	Item No.	HX 0042	Page : 20	W.O. : X-7929
Subject :	Seismic and Wind Loading Factors	Date	5/11/2011	By :	DN/MD
Wind Loading		Seismic Loading			
Given Data:	From Customer Spec	Given Data:	From Customer Spec		
Exposure Cat	C	Occupancy Cat:	III		
Occupancy Cat	III	Design Cat:	B		
Importance Fact	1.150	Reliability	1		
Kzt	1.000	Importance Factor	1.25		
V	100.000 mph	Site Class	C		
qz = 0.00256KzKztKdV^2I		Ss	0.204 g		
Height of vessel	25.000 ft or less 	S1	0.087 g		
Alpha	9.5 Table 6-2	R (Table 15.4.2)	3		
Zg	900 Table 6-2	Fa	1.2		
Kz	0.945 Table 6-3	Fv	1.7		
Kd	0.950 (Table 6-4)	Sms	0.2448 g		
qz =	26.437	Sm1	0.1479 g		
G	0.850	Sds	0.1632 g		
Cf =	0.800 (Fig 6-21)	Sd1	0.0986 g		
Wind Pressure	= qz*G*Cf	Cs = Sds/(R/I)	0.0680		
	17.977 psf 	Min Cs =	0.0100		
SNOW LOADS					
Pg	5 psf				
CAT	C				
Ct	1.2				
I	1.1				
Pf = 0.7*Ct*I	6.6 psf				
SNOW LOAD AREA	48 sf				
ADDITIONAL WEIGHT	316.8 lbs				

Input Echo, Leg & Lug Item 1, Description: X-7929 SUPPORTS

Design Internal Pressure		464.00	psig	← INCLUDES 9 PSI FOR STATIC HEAD
Design Temperature for Attachment	TEMP	515.00	F	
Vessel Outside Diameter	OD	24.000	in.	
Vessel Wall Thickness	Ts	0.4918	in.	
Vessel Corrosion Allowance	Cas	0.1250	in.	
Vessel Material		SA-106 B		
Vessel Material UNS Number		K03006		
Vessel Allowable Stress at Design	S	17100.00	psi	
Analysis Type:		Support Lug		
Tangent to Tangent Length of Vessel	TANTAN	13.469	ft.	
Height of Bottom Tangent Above Grade		0.370	ft.	

Empty Weight of Vessel	Wemp	9016.80	lb.	← WEIGHTS INCLUDE 316.8 LB
Operating Weight of Vessel (vertical load)	W	12016.80	lb.	FOR SNOW LOADING

Force Coefficient (Shape factor)	Cf	0.800	
Additional Area		0.00	in ²
User defined Wind Pressure		17.977	psf
Design Wind Code	ASCE 7-98/02/05/IBC 2003		
Type of Hill		None	

User Entered Seismic Zone Factor	Cs	0.068
----------------------------------	----	-------

Additional Horizontal Force on Vessel	FF	0.00	lb.
Location of Horizontal Force above Base Point	FH	0.00	ft.

Support Lug with Full Reinforcing Rings?		No
Number of Support Lugs	NLUG	2
Location of Support Lugs above Base Point	LLUG	11.4635 ft.
Distance from Vessel OD to Lug Contact Point	DLUG	8.6250 in.
Lug Support Force Bearing Width	WFB	4.0000 in.
Support Lug Material		SA-516 70
Support Lug Material UNS Number		K02700
Support Lug Yield Stress	Ylug	30715.00 psi
Radial Width of bottom Support Lug Plate	Wpl	10.6250 in.
Effective Force Bearing Length	Lpl	6.0000 in.
Thickness of bottom Support Lug Plate	Tpl	0.7500 in.
Distance between Gussets	Dgp	5.0000 in.
Mean Width of Gusset Plate	Wgp	6.3125 in.
Height of Gusset Plate	Hgp	8.2500 in.
Thickness of Gusset Plate	Tgp	0.5000 in.

Perform WRC-107 (Loal Stress) Analysis on Lug		Yes
Lug Pad Width (Circumferential direction)	wpad	10.000 in.
Lug Pad Length (Longitudinal direction)	lpad	13.000 in.
Lug Pad Thickness	tpad	0.375 in.

Occasional Load Factor (AISC A5.2)	Occfac	1.00
------------------------------------	--------	------

Results for support lugs, Description : X-7929 SUPPORTS

Vessel Wind Area, Af		: 6250.522 in ²
Vessel Wind load = P * Af * Cf		: 780.317 lb.
Location of Centroid above Base Point		: 7.104 ft.

Overturning Moment at Support Lug	Mlug	3562. ft.lb.
Weight Load at the top of one Lug		6008. lb.
Shear at top of one Lug		0. lb.

Distance from vessel center to Lug centroid [Rlug]:
 = (OD / 2.0 + DLUG) / 12.0
 = (24.0000 / 2.0 + 8.6250) / 12.0
 = 1.72 ft.

Force on one Lug (compression side) [Flug]:
 = (W/Nlug + Mlug/(Rlug * 2))
 = (12016 / 2 + 3562 / (1.72 * 2))
 = 7044.68 lb.

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Bearing Area [Ba]:

$$= L_{pl} * W_{fb}$$

$$= 6.000 * 4.000$$

$$= 24.00 \text{ in}^2$$

Bending Str. in bottom Plate (Unif. Load) Per Bednar p.156 [Spl2]:

$$= B_{etl} * F_{lug}/B_a * W_{fb}^2 / T_{pl}^2 \text{ per Roark \& Young 5th Ed.}$$

$$= 0.903 * 7044.7 / 24.000 * 4.000^2 / 0.750^2$$

$$= 7542.68 \text{ psi}$$

Allowable Stress for Support Plate [Spa]:

$$= \text{Min}(0.66 * Y_{lug} * O_{ccfac}, 0.75 * Y_{lug})$$

$$= \text{Min}(0.66 * 30715 * 1.00, 23036.2)$$

$$= 20271.90 \text{ psi}$$

Stress in Gusset Plate (Force / Gusset Plate Area) [Sgp]:

$$= (F_{lug}/2) / (W_{gp} * T_{gp})$$

$$= (7044 / 2) / (6.313 * 0.5000)$$

$$= 1115.99 \text{ psi}$$

Gusset Plate Allowable Stress [Sga]:

$$= (1 - (K_{lr})^2 / (2 * C_c^2)) * F_y /$$

$$(5/3 + 3 * (K_{lr}) / (8 * C_c) - (K_{lr})^3 / (8 * C_c^3))$$

$$= (1 - (66.00)^2 / (2 * 136.52^2)) * 30715 /$$

$$(5/3 + 3 * (66.00) / (8 * 136.52) - (66.00)^3 / (8 * 136.52^3))$$

$$= 14791.67 \text{ psi}$$

Maximum Compressive Stress in the Gussets per Bednar [SgpB]:

$$= F_{lug} * (3 * D_{lug} - W_{pl}) / (T_{gp} * W_{pl}^2 * (\sin(\alpha_g))^2)$$

$$= 7044 * (3 * 8.625 - 10.625) / (0.5000 * 10.625^2 * (\sin(43.73))^2)$$

$$= 3983.52 \text{ psi}$$

Gusset Plate Allowable Compressive Stress per Bednar [SgaB]:

$$= 18000 / (1 + (1/18000) * (H_{gp} / \sin(\alpha_g)) / (0.289 * T_{gp}))^2$$

$$= 18000 / (1 + (1/18000) * (8.250 / \sin(43.73)) / (0.289 * 0.5000))^2$$

$$= 13052.73 \text{ psi}$$

Summary Of Results

Stress (psi)	Actual	Allowable	P/F
Bottom Plt. Str. (Unif. Load-Bednar) :	7542.68	20271.90	Ok
Gusset Plt. Str. (Force/Gusset Area) :	1115.99	14791.67	Ok
Gusset Plt. Max Comp. Str. (Bednar) :	3983.52	13052.73	Ok

Result of WRC-107 analysis on the Vessel-Support Lug Junction:

(Worst load case + Int. Pressure, ref. Pressure Vessel Design Manual, D. Moss)

Sustained (Wt) Forces/Moments for WRC107 Analysis :

Distance from Lug bottom to Lug centroid, b :	3.463	in.
Force on lug in Sustained Condition, F_{lug_Sus} :	6008.40	lb.
Longitudinal Shear = F_{lug_Sus} :	6008.40	lb.
Long. Moment = $F_{lug_Sus} * D_{LUG} + FF / N_{lug} * b$:	4318.54	ft.lb.

Occasional (Wind/Seismic) Forces/Moments for WRC107 Analysis :

Horizontal Live load max(Wind, Seismic), V :	817.142	lb.
Force on lug in Occasional Condition, F_{lug_Occ} :	1036.28	lb.
Longitudinal Shear = F_{lug_Occ} :	1036.28	lb.
Long. Moment = $F_{lug_Occ} * D_{LUG} + V / N_{lug} * b$:	862.74	ft.lb.

Parameter C11	C11	3.00	in.
Parameter C22	C22	6.60	in.
Pad Parameter C11P	C11P	10.00	in.
Pad Parameter C22P	C22P	13.00	in.
Thickness of Reinforcing Pad	tpad	0.38	in.

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WRC 107 Stress Calculation for Sustained loads:

Radial Load	P	0.0	lb.
Circumferential Shear	VC	0.0	lb.
Longitudinal Shear	VL	6008.4	lb.
Circumferential Moment	MC	0.0	ft.lb.
Longitudinal Moment	ML	4318.5	ft.lb.
Torsional Moment	MT	0.0	ft.lb.

Dimensionless Parameters used : Gamma = 16.18

Dimensionless Loads for Cylindrical Shells at Attachment Junction:

Curves read for 1979	Beta	Figure	Value	Location
N(PHI) / (P/Rm)	0.250	4C	2.501	(A,B)
N(PHI) / (P/Rm)	0.250	3C	1.857	(C,D)
M(PHI) / (P)	0.169	2C1	0.078	(A,B)
M(PHI) / (P)	0.169	1C	0.108	(C,D)
N(PHI) / (MC/(Rm**2 * Beta))	0.163	3A	0.522	(A,B,C,D)
M(PHI) / (MC/(Rm * Beta))	0.184	1A	0.094	(A,B,C,D)
N(PHI) / (ML/(Rm**2 * Beta))	0.211	3B	1.811	(A,B,C,D)
M(PHI) / (ML/(Rm * Beta))	0.198	1B	0.040	(A,B,C,D)
N(x) / (P/Rm)	0.212	3C	2.074	(A,B)
N(x) / (P/Rm)	0.212	4C	2.617	(C,D)
M(x) / (P)	0.219	1C1	0.089	(A,B)
M(x) / (P)	0.219	2C	0.056	(C,D)
N(x) / (MC/(Rm**2 * Beta))	0.163	4A	0.766	(A,B,C,D)
M(x) / (MC/(Rm * Beta))	0.235	2A	0.047	(A,B,C,D)
N(x) / (ML/(Rm**2 * Beta))	0.211	4B	0.594	(A,B,C,D)
M(x) / (ML/(Rm * Beta))	0.239	2B	0.054	(A,B,C,D)

Stress Concentration Factors Kn = 1.00, Kb = 1.00

Stresses in the Vessel at the Attachment Junction

Type of	Stress Values at (psi)							
Stress Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P	0	0	0	0	0	0	0	0
Circ. Bend. P	0	0	0	0	0	0	0	0
Circ. Memb. MC	0	0	0	0	0	0	0	0
Circ. Bend. MC	0	0	0	0	0	0	0	0
Circ. Memb. ML	-3531	-3531	3531	3531	0	0	0	0
Circ. Bend. ML	-9592	9592	9592	-9592	0	0	0	0
Tot. Circ. Str.	-13123	6061	13123	-6061	0	0	0	0
Long. Memb. P	0	0	0	0	0	0	0	0
Long. Bend. P	0	0	0	0	0	0	0	0
Long. Memb. MC	0	0	0	0	0	0	0	0
Long. Bend. MC	0	0	0	0	0	0	0	0
Long. Memb. ML	-1755	-1755	1755	1755	0	0	0	0
Long. Bend. ML	-10666	10666	10666	-10666	0	0	0	0
Tot. Long. Str.	-12421	8911	12421	-8911	0	0	0	0
Shear VC	0	0	0	0	0	0	0	0
Shear VL	0	0	0	0	-613	-613	613	613
Shear MT	0	0	0	0	0	0	0	0
Tot. Shear	0	0	0	0	-613	-613	613	613
Str. Int.	13123	8911	13123	8911	1227	1227	1227	1227

Dimensionless Parameters used : Gamma = 32.22

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Dimensionless Loads for Cylindrical Shells at Pad edge:

Curves read for 1979	Beta	Figure	Value	Location
N(PHI) / (P/Rm)	0.554	4C	2.466	(A,B)
N(PHI) / (P/Rm)	0.554	3C	0.964	(C,D)
M(PHI) / (P)	0.465	2C1	0.008	(A,B)
M(PHI) / (P)	0.465	1C	0.061	(C,D)
N(PHI) / (MC/(Rm**2 * Beta))	0.462	3A	0.872	(A,B,C,D)
M(PHI) / (MC/(Rm * Beta))	0.509	1A	0.062	(A,B,C,D)
N(PHI) / (ML/(Rm**2 * Beta))	0.504	3B	1.308	(A,B,C,D)
M(PHI) / (ML/(Rm * Beta))	0.504	1B	0.006	(A,B,C,D)
N(x) / (P/Rm)	0.512	3C	0.964	(A,B)
N(x) / (P/Rm)	0.512	4C	2.466	(C,D)
M(x) / (P)	0.520	1C1	0.016	(A,B)
M(x) / (P)	0.520	2C	0.031	(C,D)
N(x) / (MC/(Rm**2 * Beta))	0.462	4A	3.177	(A,B,C,D)
M(x) / (MC/(Rm * Beta))	0.586	2A	0.026	(A,B,C,D)
N(x) / (ML/(Rm**2 * Beta))	0.504	4B	0.783	(A,B,C,D)
M(x) / (ML/(Rm * Beta))	0.542	2B	0.011	(A,B,C,D)

Stress Concentration Factors $K_n = 1.00$, $K_b = 1.00$ **Stresses in the Vessel at the Edge of Reinforcing Pad**

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P		0	0	0	0	0	0	0	0
Circ. Bend. P		0	0	0	0	0	0	0	0
Circ. Memb. MC		0	0	0	0	0	0	0	0
Circ. Bend. MC		0	0	0	0	0	0	0	0
Circ. Memb. ML		-2326	-2326	2326	2326	0	0	0	0
Circ. Bend. ML		-2426	2426	2426	-2426	0	0	0	0
Tot. Circ. Str.		-4753	100	4753	-100	0	0	0	0
Long. Memb. P		0	0	0	0	0	0	0	0
Long. Bend. P		0	0	0	0	0	0	0	0
Long. Memb. MC		0	0	0	0	0	0	0	0
Long. Bend. MC		0	0	0	0	0	0	0	0
Long. Memb. ML		-1612	-1612	1612	1612	0	0	0	0
Long. Bend. ML		-3978	3978	3978	-3978	0	0	0	0
Tot. Long. Str.		-5590	2366	5590	-2366	0	0	0	0
Shear VC		0	0	0	0	0	0	0	0
Shear VL		0	0	0	0	-630	-630	630	630
Shear MT		0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-630	-630	630	630
Str. Int.		5590	2366	5590	2366	1260	1260	1260	1260

WRC 107 Stress Calculation for OCCasional loads:

Radial Load	P	0.0	lb.
Circumferential Shear	VC	0.0	lb.
Longitudinal Shear	VL	1036.3	lb.
Circumferential Moment	MC	0.0	ft.lb.
Longitudinal Moment	ML	862.7	ft.lb.
Torsional Moment	MT	0.0	ft.lb.

Dimensionless Parameters used : $\Gamma = 16.18$ Stress Concentration Factors $K_n = 1.00$, $K_b = 1.00$

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Stresses in the Vessel at the Attachment Junction

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P		0	0	0	0	0	0	0	0
Circ. Bend. P		0	0	0	0	0	0	0	0
Circ. Memb. MC		0	0	0	0	0	0	0	0
Circ. Bend. MC		0	0	0	0	0	0	0	0
Circ. Memb. ML		-705	-705	705	705	0	0	0	0
Circ. Bend. ML		-1916	1916	1916	-1916	0	0	0	0
Tot. Circ. Str.		-2621	1210	2621	-1210	0	0	0	0
Long. Memb. P		0	0	0	0	0	0	0	0
Long. Bend. P		0	0	0	0	0	0	0	0
Long. Memb. MC		0	0	0	0	0	0	0	0
Long. Bend. MC		0	0	0	0	0	0	0	0
Long. Memb. ML		-350	-350	350	350	0	0	0	0
Long. Bend. ML		-2130	2130	2130	-2130	0	0	0	0
Tot. Long. Str.		-2481	1780	2481	-1780	0	0	0	0
Shear VC		0	0	0	0	0	0	0	0
Shear VL		0	0	0	0	-105	-105	105	105
Shear MT		0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-105	-105	105	105
Str. Int.		2621	1780	2621	1780	211	211	211	211

Dimensionless Parameters used : Gamma = 32.22
 Stress Concentration Factors Kn = 1.00, Kb = 1.00

Stresses in the Vessel at the Edge of Reinforcing Pad

Type of		Stress Values at (psi)							
Stress	Load	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Memb. P		0	0	0	0	0	0	0	0
Circ. Bend. P		0	0	0	0	0	0	0	0
Circ. Memb. MC		0	0	0	0	0	0	0	0
Circ. Bend. MC		0	0	0	0	0	0	0	0
Circ. Memb. ML		-464	-464	464	464	0	0	0	0
Circ. Bend. ML		-484	484	484	-484	0	0	0	0
Tot. Circ. Str.		-949	20	949	-20	0	0	0	0
Long. Memb. P		0	0	0	0	0	0	0	0
Long. Bend. P		0	0	0	0	0	0	0	0
Long. Memb. MC		0	0	0	0	0	0	0	0
Long. Bend. MC		0	0	0	0	0	0	0	0
Long. Memb. ML		-322	-322	322	322	0	0	0	0
Long. Bend. ML		-794	794	794	-794	0	0	0	0
Tot. Long. Str.		-1116	472	1116	-472	0	0	0	0
Shear VC		0	0	0	0	0	0	0	0
Shear VL		0	0	0	0	-108	-108	108	108
Shear MT		0	0	0	0	0	0	0	0
Tot. Shear		0	0	0	0	-108	-108	108	108
Str. Int.		1116	472	1116	472	217	217	217	217

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WRC 107 Stress Summations:**Vessel Stress Summation at Attachment Junction**

Type of Stress Int.	Stress Values at (psi)							
Location	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Pm (SUS)	7052	7516	7052	7516	7052	7516	7052	7516
Circ. Pm (OCC)	0	0	0	0	0	0	0	0
Circ. Pm (TOTAL)	7052	7516	7052	7516	7052	7516	7052	7516
Circ. Pl (SUS)	-3531	-3531	3531	3531	0	0	0	0
Circ. Pl (OCC)	-705	-705	705	705	0	0	0	0
Circ. Pl (TOTAL)	-4236	-4236	4236	4236	0	0	0	0
Circ. Q (SUS)	-9592	9592	9592	-9592	0	0	0	0
Circ. Q (OCC)	-1916	1916	1916	-1916	0	0	0	0
Circ. Q (TOTAL)	-11509	11509	11509	-11509	0	0	0	0
Long. Pm (SUS)	3526	3526	3526	3526	3526	3526	3526	3526
Long. Pm (OCC)	0	0	0	0	0	0	0	0
Long. Pm (TOTAL)	3526	3526	3526	3526	3526	3526	3526	3526
Long. Pl (SUS)	-1755	-1755	1755	1755	0	0	0	0
Long. Pl (OCC)	-350	-350	350	350	0	0	0	0
Long. Pl (TOTAL)	-2105	-2105	2105	2105	0	0	0	0
Long. Q (SUS)	-10666	10666	10666	-10666	0	0	0	0
Long. Q (OCC)	-2130	2130	2130	-2130	0	0	0	0
Long. Q (TOTAL)	-12797	12797	12797	-12797	0	0	0	0
Shear Pm (SUS)	0	0	0	0	0	0	0	0
Shear Pm (OCC)	0	0	0	0	0	0	0	0
Shear Pm (TOTAL)	0	0	0	0	0	0	0	0
Shear Pl (SUS)	0	0	0	0	-613	-613	613	613
Shear Pl (OCC)	0	0	0	0	-105	-105	105	105
Shear Pl (TOTAL)	0	0	0	0	-719	-719	719	719
Shear Q (SUS)	0	0	0	0	0	0	0	0
Shear Q (OCC)	0	0	0	0	0	0	0	0
Shear Q (TOTAL)	0	0	0	0	0	0	0	0
Pm (SUS)	7052	7516	7052	7516	7052	7516	7052	7516
Pm (SUS+OCC)	7052	7516	7052	7516	7052	7516	7052	7516
Pm+Pl (SUS)	3521	3985	10583	11047	7156	7608	7156	7608
Pm+Pl (SUS+OCC)	2815	3279	11288	11752	7193	7642	7193	7642
Pm+Pl+Q (Total)	11376	14788	22797	7408	7193	7642	7193	7642
Type of Stress Int.	Max. S.I.		S.I. Allowable		Result			
			psi					
Pm (SUS)	7516		17100		Passed			
Pm (SUS+OCC)	7516		20520		Passed			
Pm+Pl (SUS)	11047		25650		Passed			
Pm+Pl (SUS+OCC)	11752		30780		Passed			
Pm+Pl+Q (TOTAL)	22797		51300		Passed			

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Job No.: X-7929

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By: MD/DN 5/11/2011

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WRC 107 Stress Summations:**Vessel Stress Summation at Reinforcing Pad Edge**

Type of Stress Int.	Stress Values at (psi)							
Location	Au	Al	Bu	Bl	Cu	Cl	Du	Dl
Circ. Pm (SUS)	14489	14953	14489	14953	14489	14953	14489	14953
Circ. Pm (OCC)	0	0	0	0	0	0	0	0
Circ. Pm (TOTAL)	14489	14953	14489	14953	14489	14953	14489	14953
Circ. Pl (SUS)	-2326	-2326	2326	2326	0	0	0	0
Circ. Pl (OCC)	-464	-464	464	464	0	0	0	0
Circ. Pl (TOTAL)	-2791	-2791	2791	2791	0	0	0	0
Circ. Q (SUS)	-2426	2426	2426	-2426	0	0	0	0
Circ. Q (OCC)	-484	484	484	-484	0	0	0	0
Circ. Q (TOTAL)	-2911	2911	2911	-2911	0	0	0	0
Long. Pm (SUS)	7244	7244	7244	7244	7244	7244	7244	7244
Long. Pm (OCC)	0	0	0	0	0	0	0	0
Long. Pm (TOTAL)	7244	7244	7244	7244	7244	7244	7244	7244
Long. Pl (SUS)	-1612	-1612	1612	1612	0	0	0	0
Long. Pl (OCC)	-322	-322	322	322	0	0	0	0
Long. Pl (TOTAL)	-1934	-1934	1934	1934	0	0	0	0
Long. Q (SUS)	-3978	3978	3978	-3978	0	0	0	0
Long. Q (OCC)	-794	794	794	-794	0	0	0	0
Long. Q (TOTAL)	-4773	4773	4773	-4773	0	0	0	0
Shear Pm (SUS)	0	0	0	0	0	0	0	0
Shear Pm (OCC)	0	0	0	0	0	0	0	0
Shear Pm (TOTAL)	0	0	0	0	0	0	0	0
Shear Pl (SUS)	0	0	0	0	-630	-630	630	630
Shear Pl (OCC)	0	0	0	0	-108	-108	108	108
Shear Pl (TOTAL)	0	0	0	0	-738	-738	738	738
Shear Q (SUS)	0	0	0	0	0	0	0	0
Shear Q (OCC)	0	0	0	0	0	0	0	0
Shear Q (TOTAL)	0	0	0	0	0	0	0	0
Pm (SUS)	14489	14953	14489	14953	14489	14953	14489	14953
Pm (SUS+OCC)	14489	14953	14489	14953	14489	14953	14489	14953
Pm+Pl (SUS)	12163	12627	16815	17279	14544	15004	14544	15004
Pm+Pl (SUS+OCC)	11698	12162	17280	17744	14564	15023	14564	15023
Pm+Pl+Q (Total)	8786	15074	20192	14832	14564	15023	14564	15023
Type of Stress Int.	Max. S.I. psi				S.I. Allowable		Result	
Pm (SUS)	14953				17100		Passed	
Pm (SUS+OCC)	14953				20520		Passed	
Pm+Pl (SUS)	17279				25650		Passed	
Pm+Pl (SUS+OCC)	17744				30780		Passed	
Pm+Pl+Q (TOTAL)	20192				51300		Passed	

PV Elite is a trademark of Intergraph CADWorx & Analysis Solutions, Inc. 2011

ANCHOR BOLTING CALC: (SHEAR ONLY)

BOLT MATERIAL: A325

BOLT DIAM: 1" (AREA=0.551IN²) 4 BOLTS TOTAL (2 PER SUPPORT)

GREATEST COMBINED TRANSVERSE LOAD = 1077 LB (COMBINED WIND & SEISMIC)

STRESS IN BOLTS = 488.7 PSI < 11880 PSI ALLOW DESIGN IS ACCEPTABLE

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Input Echo, Leg & Lug Item 2, Description: X-7929 FLL

Design Internal Pressure		0.00	psig
Design Temperature for Attachment	TEMP	100.00	F
Vessel Outside Diameter	OD	24.000	in.
Vessel Wall Thickness	Ts	0.4918	in.
Vessel Corrosion Allowance	Cas	0.1250	in.
Vessel Material		SA-106 B	
Vessel Material UNS Number		K03006	
Vessel Allowable Stress at Design	S	17100.00	psi
Analysis Type:		Lifting Lug	
Empty Weight of Vessel	Wemp	8700.00	lb.
Operating Weight of Vessel (vertical load)	W	11700.00	lb.
Force Coefficient (Shape factor)	Cf	0.800	
Additional Area		0.00	in ²
User defined Wind Pressure		17.977	psf
Design Wind Code	ASCE	7-05	
Type of Hill		None	
User Entered Seismic Zone Factor	Cs	0.068	
Lifting Lug Material		SA-516 70	
Lifting Lug Material UNS Number		K02700	
Lifting Lug Yield Stress	YIELD	38000.00	psi
Lifting Lug Orientation to Vessel		Perpendicular	
Total Height of Lifting Lug	w	8.0000	in.
Thickness of Lifting Lug	t	0.5000	in.
Diameter of Hole in Lifting Lug	dh	2.0000	in.
Radius of Semi-Circular Arc of Lifting Lug	r	4.0000	in.
Height of Lug from bottom to Center of Hole	h	4.0000	in.
Offset from Vessel OD to Center of Hole	off	5.0000	in.
Minimum thickness of Fillet Weld around Lug	tw	0.3750	in.
Length of weld along side of Lifting Lug	wl	8.0000	in.
Length of Weld along Bottom of Lifting Lug	wb	0.5000	in.
Lift Orientation		Vertical	
Force Along Vessel Axis	Fax	8700.00	lb.
Force Normal to Vessel	Fn	8700.00	lb.
Force Tangential to Vessel	Ft	0.00	lb.
Impact Factor	Impfac	1.50	
Occasional Load Factor (AISC A5.2)	Occfac	1.12	

Results for lifting lugs, Description : X-7929 FLL

Vessel Wind Area, Af : 8208.000 in²
Vessel Wind load = P * Af * Cf : 920.208 lb.
Location of Centroid above Base Point : 9.448 ft.

Weld Group Inertia about the Circumferential Axis	ILC	35.190	in**4
Weld Group Centroid distance in the Long. Direction	YLL	4.248	in.
Dist. of Weld Group Centroid from Lug bottom	YLL_B	3.873	in.
Weld Group Inertia about the Longitudinal Axis	ILL	0.098	in**4
Weld Group Centroid Distance in the Circ. Direction	YLC	0.250	in.

Applying the Impact factor to the loads:

Fax = 8700.00 * 1.50 = 13050.00 lb.
Fn = 8700.00 * 1.50 = 13050.00 lb.

Primary Shear Stress in the Welds due to Shear Loads [Ssl]:

$$= \text{Sqrt}(Fax^2 + Ft^2 + Fn^2) / ((2 * wl + wb) * tw)$$

$$= \text{Sqrt}(13050^2 + 0^2 + 13050^2) / ((2 * 8.0 + 0.5) * 0.3750)$$

$$= 2982.71 \text{ psi}$$
Shear Stress in the Welds due to Bending Loads [Sblf]:

$$= (Fn * (h - YLL_B)) * YLL / ILC + (Fax * off * YLL / ILC) + (Ft * off * YLC / ILL)$$

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$$\begin{aligned}
 &= (13050 * (4.000 - 3.873)) * 4.248 / 35.190 + \\
 &\quad (13050 * 5.000 * 4.248 / 35.190) + \\
 &\quad (0 * 5.000 * 0.250 / 0.098) \\
 &= 8076.74 \text{ psi}
 \end{aligned}$$

Total Shear Stress for Combined Loads [St]:

$$\begin{aligned}
 &= S_{s11} + S_{b1f} \\
 &= 2982.705 + 8076.736 \\
 &= 11059.44 \text{ psi}
 \end{aligned}$$

Allowable Shear Stress for Combined Loads [Sta]:

$$\begin{aligned}
 &= 0.4 * \text{Yield} * \text{Occfac} \text{ (AISC Shear All.)} \\
 &= 0.4 * 38000 * 1.12 \\
 &= 17100.00 \text{ psi}
 \end{aligned}$$

Shear Stress in Lug above Hole [Shs]:

$$\begin{aligned}
 &= \text{Sqrt}(F_{ax}^2 + F_n^2 + F_t^2) / S_{ha} \\
 &= \text{Sqrt}(13050^2 + 13050^2 + 0^2) / 3.000 \\
 &= 6151.83 \text{ psi}
 \end{aligned}$$

Allowable Shear Stress in Lug above Hole [Sas]:

$$\begin{aligned}
 &= 0.4 * \text{Yield} * \text{Occfac} \\
 &= 0.4 * 38000 * 1.12 \\
 &= 17100.00 \text{ psi}
 \end{aligned}$$

Pin Hole Bearing Stress [Pbs]:

$$\begin{aligned}
 &= \text{Sqrt}(F_{ax}^2 + F_n^2) / (t * d_h) \\
 &= \text{Sqrt}(13050^2 + 13050^2) / (0.500 * 2.000) \\
 &= 18455.49 \text{ psi}
 \end{aligned}$$

Allowable Bearing Stress [Pba]:

$$\begin{aligned}
 &= \text{Min}(0.75 * \text{Yield} * \text{Occfac}, 0.9 * \text{Yield}) \text{ AISC Bearing All.} \\
 &= \text{Min}(0.75 * 38000 * 1.12, 34200.0) \\
 &= 32062.50 \text{ psi}
 \end{aligned}$$

Bending stress at the base of the lug [Fbs]:

$$\begin{aligned}
 &= F_t * \text{off} / (w * t^2 / 6) + F_{ax} * \text{off} / (w^2 * t / 6) \\
 &= 0 * 5.000 / (8.000 * 0.500^2 / 6) + \\
 &\quad 13050 * 5.000 / (8.000^2 * 0.500 / 6) \\
 &= 12234.38 \text{ psi}
 \end{aligned}$$

Tensile stress at the base of the lug [Fa]:

$$\begin{aligned}
 &= F_n / (w * t) = 13050 / (8.000 * 0.500) \\
 &= 3262.50 \text{ psi}
 \end{aligned}$$

Total Combined Stress at the base of the lug:

$$= F_{bs} + F_a = 15496.9 \text{ psi}$$

Lug Allowable Stress for Bending and Tension:

$$\begin{aligned}
 &= \text{Min}(0.66 * \text{Yield} * \text{Occfac}, 0.75 * \text{Yield}) \\
 &= \text{Min}(0.66 * 38000 * 1.12, 28500.0) = 28215.0 \text{ psi}
 \end{aligned}$$

Note: Check the Shell Stresses using method such as WRC-107.

Summary Of Results

Stress (psi)	Actual	Allowable	P/F
Primary Shear Stress of Weld :	11059.44	17100.00	Ok
Shear Stress above Hole :	6151.83	17100.00	Ok
Pin Hole Bearing Stress :	18455.49	32062.50	Ok
Total Combined Stress at the lug base :	15496.88	28215.00	Ok

PVElite is a registered trademark of COADE, Inc. [2010]

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Energy Exchanger Company

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Customer : MISSISSIPPI POWER/SOUTHERN COMPANY SERVICES

Item No HX0042

W.O. : X-7929

Subject : LIFTING LUG CALCULATION

Date 3/9/2011

By : MD/DN

Note : Lugs are not designed for side loading. Do not side load lugs.

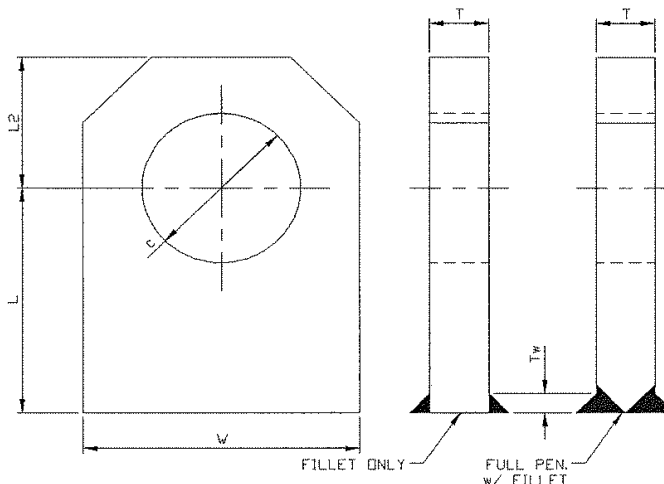
Material SA-516-70
Yield Strength 38000 psi
T = 1/2 in
Tw = 1/4 in
W = 3 1/2 in
L = 4 3/4 in
L2 = 1 3/4 in
OAL = 6 1/2 in
C = 2 in
Analysis Type Full Pen w/ Fillet 
Impact Factor 1.500
Occasional Load Factor 1.125
Heaviest Lug Load 305 #
(CHANNEL COVER)
Weld Group Area (Aw) 2.651 in²
Weld Group Inertia (Iw) 3.067 in⁴
Shear Area (As) 0.375 in²
Lug Inertia (I) 1.786 in⁴

Allowable Loading

Shear Stress 17100 psi
0.4 * Yield * Occ Factor

Bearing Stress 32062.5 psi
MIN(0.75*Yield*Occ Factor, 0.9*Yield)

Combined 28215 psi
MIN(.66*Yield*Occ Factor, 0.75*Yield)



Calculated Loads

Primary Shear in Welds 172.56 psi
Load*Impact/(Aw)
Bending Shear Stress in Welds 298.30 psi
Load*Impact*(W+2*Tw)/2/(Iw)
Total Combined Shear 470.86 psi
Shear in Lug Above Hole 813.33 psi
Load*Impact/(As)
Pin Hole Bearing Stress 457.50 psi
Load*Impact/(T*C)
Bending Stress @ Lug Base 2128.78 psi
Load*Impact*L/(W^2*T/6)
Tensile @ Lug Base 261.43 psi
Load*Impact/(W*T)
Total Combined Stress @ lug Base 2390.20 psi

SECTION 3

FORM U-1 MANUFACTURER'S DATA REPORT FOR PRESSURE VESSELS
As Required by the Provisions of the ASME Code Rules, Section VIII, Division 1

1. Manufactured and certified by ENERGY EXCHANGER COMPANY, 1844 NORTH GARNETT ROAD, TULSA, OK 74116
(Name and address of Manufacturer)

2. Manufactured for SOUTHERN COMPANY SERVICES, INC. 42 INVERNESS CENTER PARKWAY BIRMINGHAM, AL 35242
(Name and address of Purchaser)

3. Location of installation MISSISSIPPI POWER COMPANY, DEKALB, MS.
(Name and address)

4. Type: VERT. HEAT EXCHANGER X-7929
(Horiz., vert., or sphere) (Tank, separator, jkt. vessel, heat exch., etc.) (Mfg's serial No.)

N/A X-7929-A REV. 6 6066 2012
(CRN) (Drawing No.) (Nat'l Bd. No.) (Year built)

5. ASME Code, Section VIII, Div. 1 2007 EDITION, '09 ADD. N/A
[Edition and Addenda (date)] (Code Case No.) [Special Service per UG-120(d)]

Items 6-11 incl. to be completed for single wall vessels, jackets of jacketed vessels, shell of heat exchangers, or chamber of multichamber vessels.

6. Shell (a) No. of course(s): 1 (b) Overall length (ft & in.): 13'-5 5/8"

Course(s)			Material	Thickness		Long. Joint (Cat. A)			Circum. Joint (Cat. A, B, & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	24" O.D.	13'-5 5/8"	SA-106B	0.5620"	1/8"	S	NONE	1.0	7	NONE	1.0	N/A	N/A

7. Heads: (a) NA (b) NA
(Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.) (Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)														
(b)														

If removable, bolts used (describe other fastening)

NA
(Mat'l Spec. No., Grade, size, No.)

8. Type of jacket NA Jacket closure NA
(Describe as ogee & weld, bar, etc.)

If bar, give dimensions NA If bolted, describe or sketch.

9. MAWP 455 15 psi at max. te 515 515 °F Min. design metal temp. 10 °F at 455 psi.
(internal) (external) (internal) (external)

10. Impact test NO, EXEMPT PER UG-20(f) at test temperature of NA °F.
[Indicate yes or no and the component(s) impact tested]

11. Hydro., pneu., or comb. Test press. 605 P.S.I.G. (1) Proof test NA

Items 12 and 13 to be completed for tube sections.

12. Tubesheet: SA-516-70N 23 3/4" 3 3/8" 1/4" Welded
[Stationary (Mat'l Spec. No.)] [Dia., in. (subject to press.)] (Nom. thk., in.) (Corr. Allow., in.) [Attachment (welded or bolted)]

NA NA NA NA NA
[Floating (Mat'l Spec. No.)] (Dia., in.) (Nom. thk., in.) (Corr. Allow., in.) [Attachment]

13. Tubes: SA-179 3/4" 0.0830" (MW) THK. 462 Straight
(Mat'l Spec. No., Grade or type) (O.D., in.) (Nom. thk., in. or gauge) (Number) [Type (Strait or U)]

Items 14-18 incl. to be completed for inner chambers of jacketed vessels or channels of heat exchangers.

14. Shell (a) No. of course(s): 1/1 (b) Overall length (ft & in.): 1'-8 3/4" / 1'-6 7/8"

Course(s)			Material	Thickness		Long. Joint (Cat. A)			Circum. Joint (Cat. A, B, & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	24" O.D.	1'-8 3/4"	SA-333-6	1/2"	3/16"	S	NONE	0.85	1 (2)	SPOT	0.85	1150° F.	1-HR.
1	24" O.D.	1'-6 7/8"	SA-333-6	1/2"	3/16"	S	NONE	0.85	1 (2)	SPOT	0.85	1150° F.	1-HR.

15. Heads: (a) SA-516-70N H.T. @ 1150 deg F. for 1 HR. (b) SA-516-70N H.T. @ 1150 deg F. for 1 HR.
(Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.) (Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	END	1 3/4"	3/16						24 5/16"					
(b)	END	1 3/4"	3/16						24 5/16"					

If removable, bolts used (describe other fastening)

a) SA-193B7; 3/4" DIA; 24 b) SA-193B7; 3/4" DIA; 24
(Mat'l Spec. No., Grade, size, No.)

FORM U-1 (Back)

16. MAWP 205 NA psi at max. temp. 390 NA °F Min. design metal temp. 10 °F at 205 psi.
(internal) (external) (internal) (external)
17. Impact test NO, EXEMPT PER UG-20(f) at test temperature of NA °F.
(Indicate yes or no and the component(s) impact tested)
18. Hydro., pneu., or comb. Test press. 280 P.S.I.G. (f) Proof test NA
19. Nozzles, inspection, and safety valve openings:

Purpose (Inlet, Outlet, Drain, etc.)	No.	Diameter or Size	Flange Type	Material		Nozzle Thickness		Reinforcement Material	How Attached		Location (Insp. Open.)
				Nozzle	Flange	Nom.	Corr.		Nozzle	Flange	
INLET	1	6"-300#	RF-WN	SA-106B	SA-105	0.4320"	1/8"	SA-516-70	WELDED	WELDED	SHELL
OUTLET	1	3"-300#	RF-WN	SA-106B	SA-105	0.6000"	1/8"	WELD	WELDED	WELDED	SHELL
BALANCE LINE	1	3/4"-300#	RF-LWN	SA-105	—	0.5700"	1/8"	WELD	WELDED	—	SHELL
VENT	1	3/4"-300#	RF-LWN	SA-105	—	0.5700"	1/8"	WELD	WELDED	—	TUBESHEET
DRAIN	1	3/4"-300#	RF-LWN	SA-105	—	0.5700"	1/8"	WELD	WELDED	—	TUBESHEET
OUTLET	1	10"-300#	RF-WN	SA-333-6	SA-105	0.5000"	3/16"	WELD	WELDED	WELDED	TOP CHAN.
VENT	1	2"-300#	RF-LWN	SA-105	—	0.6600"	3/16"	WELD	WELDED	—	TOP CHAN CVR
INLET	1	6"-300#	RF-WN	SA-333-6	SA-105	0.4320"	3/16"	WELD	WELDED	WELDED	BTTM CHAN.
DRAIN	1	2"-300#	RF-LWN	SA-105	—	0.6600"	3/16"	WELD	WELDED	—	BTTM CHAN CVR

20. Supports: Skirt NO Lugs — Legs — Others 2-SADDLES Attached WELDED TO SHELL
(Yes or no) (No.) (No.) (Describe) (Where and how)
21. Manufacturer's Partial Data Reports properly identified and signed by Commissioned Inspectors have been furnished for the following items of the report:
 (List the name of part, item number, mfg's. name and identifying number)

22. Remarks: SERVICE: HYDROGEN SULFIDE STRIPPER REBOILER; P.O. NO: 17496-0001; ITEM NO.: HX0042; SIZE: 23-168; TYPE: NEN
 TUBES DESIGNED W/ 0" C.A., SAFETY RELIEF DEVICES BY OTHERS
 LINES 14-18 INCLUDES (2) 28 3/16" O.D. X 23" I.D. X 3 1/16" THK., SA-266-4N, RING
 (1) FIELD TEST: SHELL SIDE=595 PSIG; TUBE SIDE=270 PSIG (2) Cylinder to tubesheet circumferential seams are Type 7 with .85 Efficiency

CERTIFICATE OF SHOP COMPLIANCE

We certify that the statements in this report are correct and that all details of design, material, construction, and workmanship of this vessel conform to the ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1.

U Certificate Authorization No. 12370 Expires SEPTEMBER 27, 20 13

Date 8-24-12 Name ENERGY EXCHANGER COMPANY Signed Mark Wells
(Manufacturer) (Representative)

CERTIFICATE OF SHOP INSPECTION

I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of OKLAHOMA and employed by ONEBEACON AMERICA INSURANCE COMPANY of LYNN, MA

have inspected the pressure vessel described in this Manufacturer's Data Report on 8-24-2012, and state that, to the best of my knowledge and belief, the Manufacturer has constructed this pressure vessel in accordance with ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date 8-24-12 Signed [Signature] Commissions N.B.# 12064 A OKLA.# 7
(Authorized Inspector) (Nat'l Board incl. endorsements, State, Province, and No.)

CERTIFICATE OF FIELD ASSEMBLY COMPLIANCE

We certify that the statements on this report are correct and that field assembly construction of all parts of this vessel conforms with the requirements of ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. U Certificate of Authorization No.

U Certificate of Authorization No. _____ Expires _____, 20 _____

Date _____ Name _____ Signed _____
(Assembler) (Representative)

CERTIFICATE OF FIELD ASSEMBLY INSPECTION

I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of _____ and employed by _____

of _____, have compared the statements in this Manufacturer's Data Report with the described pressure vessel and state that parts referred to as data items _____, not included in the certificate of shop inspection, have been inspected by me and to the best of my knowledge and belief, the Manufacturer has constructed and assembled this pressure vessel in accordance with ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. The described vessel was inspected and : _____ psi.

By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date _____ Signed _____ Commissions _____
(Authorized Inspector) (Nat'l Board incl. endorsements, State, Province, and No.)

NR 6066



CERTIFIED BY
Energy Exchanger Company
TULSA, OKLAHOMA

 U W RT-4 HT-T	MAWP SHELL	455	P.S.I. @	515	OF	595	TEST P.S.I.G.
	MAWP	15	P.S.I. @	515	OF		
	MAWP TUBE	205	P.S.I. @	390	OF	270	TEST P.S.I.G.
	MAWP	- - -	P.S.I. @	- - -	OF		
MIN. DESIGN METAL TEMP.			SHELL	10	OF @	455	P.S.I.G.
			TUBE	10	OF @	205	P.S.I.G.
SERIAL NO.	X-7929			YEAR BUILT		2012	

P.O. NO. 17496-0001

SERVICE HYDROGEN SULFIDE STRIPPER REBOILER

ITEM HX0042 SIZE 23-168 TYPE NEN
DESIGN PRESSURE: SHELL: 464 PSI TUBE: 214 PSI
SHOP TEST: SHELLSIDE: 605 PSIG TUBESIDE: 280 PSIG

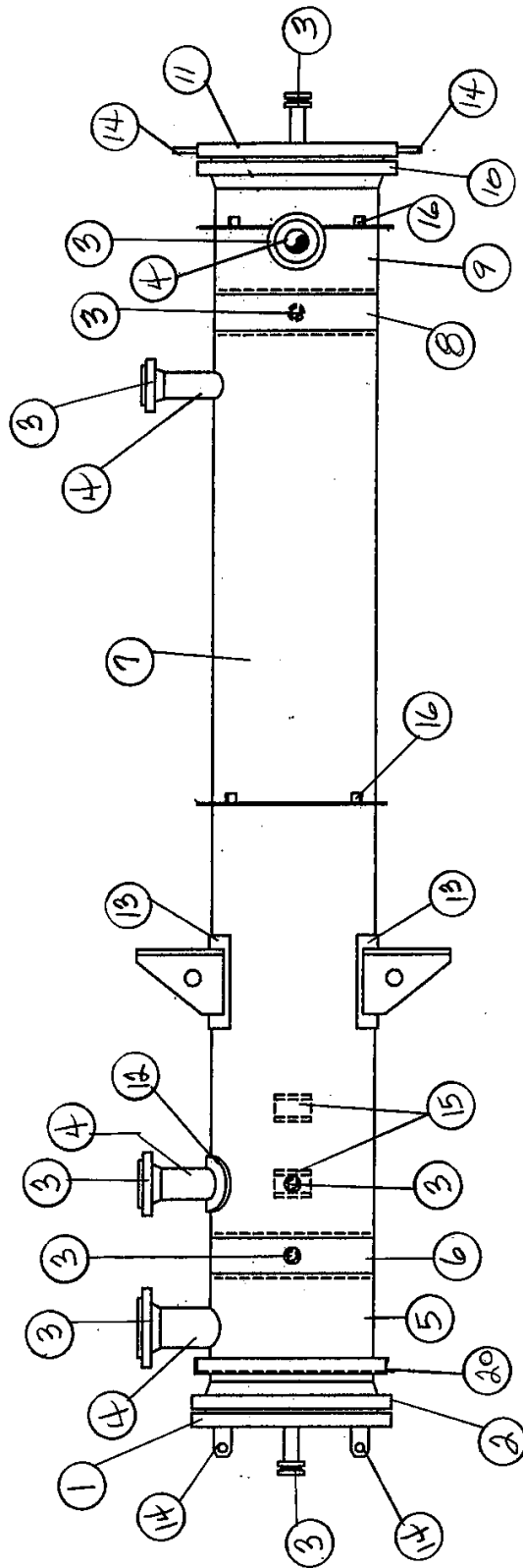
CAUTION

PRESSURES AND TEMPERATURES SHOWN ON THE NAMEPLATE OF THIS HEAT EXCHANGER RELATE ONLY TO PRESSURE RETAINING CAPABILITY, CERTAIN COMPONENTS HAVE BEEN DESIGNED FOR SPECIFIC OPERATING CONDITIONS. DEVIATION FROM THESE CONDITIONS OR A CHANGE IN SERVICE REQUIRE A MECHANICAL DESIGN CHECK BE PERFORMED TO AVOID POSSIBLE DAMAGE TO THE HEAT EXCHANGER OR DANGER TO PERSONNEL.

WARNING

FIXED TUBESHEET HEAT EXCHANGER. INTRODUCE SHELLSIDE FIRST BEFORE INTRODUCING TUBE SIDE. STEAMOUT TUBESIDE ALONE IS NOT ALLOWED FOR THIS UNIT. STEAMOUT SHELLSIDE FIRST, WHILE SHELLSIDE IS HOT STEAMOUT TUBE SIDE.

SECTION 4



17 TUBES
 18 STUDS
 19 NUTS

Reviewed / Accepted
 I.Q.S. #3
 William Brennaugh

WBS

X-7929

ENERGY EXCHANGER COMPANY MATERIAL HEAT NUMBER INDEX

Project; Kemper County 1 - IGCC

CUSTOMER: Southern Co. (for) Mississippi Power Company
DESTINATION: DeKalb, Mississippi
P.O. No: 17496-0001
Req. No: MC202

ITEM NO: HX0042
SERVICE: Hydrogen Sulfide Stripper Reboiler
DESCRIPTION: 1-23" x 168" NEN Exchanger

PAGE: 1 of 3
DATE: August 23, 2012
JOB NO: X-7929

PART NO.	HEAT NO.	SLAB NO.	STEEL PRODUCER	GRADE	DESCRIPTION	QCS	SIZE
1	822P34450	H032355	ArcelorMittal	SA-516-70N	(#1 Channel Cover)	1	28 3/16" OD x 1 3/4" thk.
2	110786		Nucor	SA-266-4N	(#2 Channel Flange)	1	28 3/16" OD x 23" ID x 3 1/16" thk.
3	W019		FCI	SA-105-N	(Nozzle Flange V-1)	1	2" 300# RF LWN x 9" lg.
3	3019785		Galperti	SA-105-N	(Nozzle Flange V-1)	1	2" 300# RF Blind
3	3000773		Galperti	SA-105-N	(Nozzle Flange T-2)	1	10" 300# RF WN Sch X-Hvy. Bore
3	A606AF		FVC	SA-105	(Nozzle Flange V-2)	1	3/4" 300# RF LWN x 9" lg.
3	79209		Galperti	SA-105-N	(Nozzle Flange V-2)	1	3/4" 300# RF Blind
3	102250N/02		Galperti	SA-105-N	(Nozzle Flange S-1)	1	6" 300# RF WN Sch X-Hvy. Bore
3	9721/02		Galperti	SA-105-N	(Nozzle Flange S-2)	1	3" 300# RF WN Sch XX-Hvy. Bore
3	A606AF		FVC	SA-105	(Nozzle Flange S-3)	1	3/4" 300# RF LWN x 9" lg.
3	79209		Galperti	SA-105-N	(Nozzle Flange S-3)	1	3/4" 300# RF Blind
3	A606AF		FVC	SA-105	(Nozzle Flange D-2)	1	3/4" 300# RF LWN x 9" lg.
3	79209		Galperti	SA-105-N	(Nozzle Flange D-2)	1	3/4" 300# RF Blind

I hereby certify that the above information is correct to the best of my knowledge and belief.

ENERGY EXCHANGER COMPANY

By:

Signature

Reviewed / Accepted
I.Q.S. # 3

William Brennaugh

ENERGY EXCHANGER COMPANY MATERIAL HEAT NUMBER INDEX

Project; Kemper County 1 - IGCC

CUSTOMER: Southern Co. (for) Mississippi Power Company
 DESTINATION: DeKalb, Mississippi
 P.O. No: 17496-0001
 Req. No: MC202
 ITEM NO: HX0042
 SERVICE: Hydrogen Sulfide Stripper Reboiler
 DESCRIPTION: 1-23" x 168" NEN Exchanger
 PAGE: 2 of 3
 DATE: August 23, 2012
 JOB NO: X-7929

PART NO.	HEAT NO.	SLAB NO.	STEEL PRODUCER	GRADE	DESCRIPTION	PCS.	SIZE
----------	----------	----------	----------------	-------	-------------	------	------

3	102250N/02		Galperti	SA-105-N	(Nozzle Flange T-1)	1	6" 300# RF WN Sch X-Hvy. Bore
3	W019		FCI	SA-105-N	(Nozzle Flange D-1)	1	2" 300# RF LWN x 9" lg.
3	3019785		Galperti	SA-105-N	(Nozzle Flange D-1)	1	2" 300# RF Blind
4	83776		V&M	SA-333-6	(Nozzle Pipe T-2)	1	10" Sch X-Hvy. X 7 3/4" lg.
4	X22424		U.S.S.	SA-106-B	(Nozzle Pipe S-1)	1	6" Sch X-Hvy. X 7 3/4" lg.
4	HA0470		U.S.S.	SA-106-B	(Nozzle Pipe S-2)	1	3" Sch XX-Hvy. X 8 1/4" lg.
4	ZX0008		V&M Star	SA-333-6	(Nozzle Pipe T-1)	1	6" Sch X-Hvy. X 7 3/4" lg.
5	MA2181		U.S.S.	SA-333-6	(Top Channel Cylinder)	1	24" OD x 1/2" thk. x 1'-8 3/4" lg.
6	D52809	68347	LeTourneau	SA-516-70N	(#4 Tubesheet)	1	24" OD x 3 3/8" thk.
7	MA1922		U.S.S.	SA-106-B	(Shell Cylinder)	1	24" OD x .562" thk. x 13'-5 5/8" lg.
8	D52809	68347	LeTourneau	SA-516-70N	(#9 Tubesheet)	1	24" OD x 3 3/8" thk.
9	MA2181		U.S.S.	SA-333-6	(Bottom Channel Cylinder)	1	24" OD x 1/2" thk. x 1'-6 7/8" lg.
10	110786		Nucor	SA-266-4N	(#2A Channel Flange)	1	28 3/16" OD x 23" ID x 3 1/16" thk.

I hereby certify that the above information is correct to the best of my knowledge and belief.

ENERGY EXCHANGER COMPANY

By: 

Reviewed / Accepted
 I.Q.S. # 3
 William Brennaugh 

ENERGY EXCHANGER COMPANY MATERIAL HEAT NUMBER INDEX

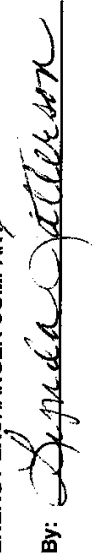
Project; Kemper County 1 - IGCC

CUSTOMER: Southern Co. (for) Mississippi Power Company
 DESTINATION: DeKalb, Mississippi
 P.O. No: 17496-0001
 Reg. No: MC202
 ITEM NO: HX0042
 SERVICE: Hydrogen Sulfide Stripper Reboiler
 DESCRIPTION: 1-23" x 168" NEN Exchanger
 PAGE: 3 of 3
 DATE: August 23, 2012
 JOB NO: X-7929

PART NO.	HEAT NO.	SLAB NO.	STEEL PRODUCER	GRADE	DESCRIPTION	PCS.	SIZE
11	822P34450	H032355	ArcelorMittal	SA-516-70N	(#1A Channel Cover)	1	28 3/16" OD x 1 3/4" thk.
12	1502825	09	Nucor	SA-516-70	(Shell Weld Pad)	1	9 3/4" OD x 3/8 thk.
13	A1S1804	02	Nucor	SA-516-70	(Support Pads)	2	13" x 3/8" thk. x 10 1/8" lg.
14	B1T7537	03	Nucor	SA-516-70	(Lifting Lugs)	4	3 1/2" x 1/2" thk. x 6 1/2" lg.
15	W0H543	0024	SSAB	SA-516-70	(Name Plate Bracket)	2	7 1/2" x 3/16" thk. x 22" lg.
16	3018892		CMC Steel	SA-36	(Insulation Clips)	8	1 1/2" x 1/4" Thk. x 3" lg.
17	181070		Benteler	SA-179	(Tubes)	390	3/4"OD x .083" (min) x 14'-0" lg.
17	548853		Benteler	SA-179	(Tube)	72	3/4"OD x .083" (min) x 14'-0" lg.
18	10037360		Vulcan Thrd. Prod.	SA-193-B7	(Studs)	48	3/4" (10 Thd) x 5 1/4" lg.
19	SS161		Unytite	SA-194-2H	(Nuts)	96	3/4" (10 Thd)
20	L81828		ArcelorMittal	SA-36	(Top Flashing)	1	3" x 3" x 1/4" thk. x 95" lg.

I hereby certify that the above information is correct to the best of my knowledge and belief.

ENERGY EXCHANGER COMPANY

By: 

Reviewed / Accepted
I.Q.S. # 3

William Brennaugh

ArcelorMittal Burns Harbor Plate

Part # 1 + 11

20010

SHIPMENT NO. 803-16933		DATE SHIPPED 12-16-10		CAR OR VEHICLE NO. CSS-CHGO-BNSF		LMIC 406516		PAGE 4	
STEEL SERVICE CO 24412 AMAH PKWY CLAREMORE OK 74017				STEEL SERVICE CO C/O TULOMA STEVEDORING INC TULSA PORT AUTHORITY OK X-7929-1 + 1A					
SOLID TO		SHIP TO							
NOT E		SIZE AND QUANTITY		YIELD POINT		TENSILE STRENGTH		ELONG	
SERIAL NUMBER	PAT NO.	HEAT NUMBER	NO. PCS.	THICKNESS	WIDTH OR DIA.	LENGTH	WEIGHT	YIELD POINT	TENSILE STRENGTH
				INCHES	INCHES	INCHES	POUNDS	PSI	PSI IN %

QUALITY STEEL MELTED & MANUFACTURED IN THE U. S. A. ✓
 PLATES - ASTM A516-06 GR 70 PVQ, ASME SA516 ✓
 GR 70 PVQ 2010 EDITION, CH-V A20S5
 PLT L 15/12 FTLBS AT -40F --- GAS
 CUT 4 SIDES PLT NORMALIZED & COOLED
 IN STILL AIR
 NO WELD REPAIR WAS PERFORMED ON BELOW PLATE(S)
 MFST - MFST MILL SERIAL# LIFT MAX 10 TON UNLDG
 OH-CHAIN
 CO# 31556 GH 816-3007J
 PLATES HEAT TREATED - TEST SPECIMENS ATTACHED & YIELD STRENGTH @ .5% EUL
 H032355 822P34450 1 2 96 480 26137 46300 / 74100 2 33 ✓
 N 1650 DEG F - 99 MIN

NAME: C.G.MARTIN
 PO#: 6237
 QUANTITY: 1
 SIZE: 2 X 96 X 480 A516-70 NORM
 HEAT #: 822P34450 H032355

CERTIFIED TRUE COPY TEST REPORT FROM
 C.G. MARTIN CO., INC

CUST ID	22	WO#	79689
CUST	ENERGY EXCHANGER COMPANY		
P.O. #	1171		
ITEM #	2		
DESC	(2) 28-3/16" OD(F) X 1-3/4" THK(F)		
DATE	05/18/11	BY:	<i>[Signature]</i>

Q: QUENCH TEMPERATURE T: TEMPER TEMPERATURE

SERIAL NUMBER	PAT NO.	HEAT NUMBER	HARD BHN	BEND	THICKNESS INCHES	TYPE	SIZE	DIR	TEST TEMP F	CHARPY IMPACT									LAT. EXP MILS	
										ENERGY FT LBS			SHEAR(%)							
										1	2	3	1	2	3	1	2	3		
H032355		822P34450			2.000	V	FULL	L	-40	100	111	110								

HEAT NUMBER	CHEMICAL ANALYSIS																LIQUID GRAIN SIZE
	C ✓	Mn ✓	P ✓	S ✓	Si ✓	Cu ✓	Ni ✓	Cr ✓	Mo ✓	V ✓	Ti ✓	Al ✓	B	Co ✓	N	Sb	
822P34450	.17	1.09	.015	.004	.335	.028	.02	.12	.077	.004	.002	.034	.0002	.002	.005	.005	

I certify that the above results are a true and correct copy of actual results contained in records maintained by ArcelorMittal Burns Harbor and are in full compliance with the requirements of the specification cited above. This test report cannot be altered and must be transmitted intact with any subsequent third party test reports, if required.

BHPLTRPT.TIF

SUPV. QUALITY ASSURANCE

D. W. ELWOOD PER WNK

VERIFIED BY

MAY 24 2011

EEC QC DEPT

[Signature: J. Jetterson]

Reviewed / Accepted
 I.Q.S. # 3

[Signature: William Brennaugh]

Part # 1+11

GLOBE X-RAY SERVICES, INC.

8441 South Union Tulsa, Ok. 74132

Report of Portable Hardness Test

Certification NO: 47524Customer C G MARTINJob or P.O.# WO# 079689Address: TULSA, OKProcedure: GXS VIII-B Rev.1-07Test Instrument: KBE MIL-1040 Ser.# 34148Probe: 10 kgf Ser.# 33766Calibration Verification Block: 194 BRINELL BARNOTES: ALL RECORDS ARE AN AVERAGE OF 5 READINGS IN
BRINELL

The material listed below have been tested and comply with the acceptance criteria of the above procedure. All hardness values stated below are measurements of the BRINELL scale.

Item# or Description	Weld	HAZ	HAZ	Base	Base
PL # 20010 822P34450 HO 32355				172 HBN	✓

See attachment for additional results

Examination By: WINTERED C#Date: 5/18/11

This is not a laboratory report and test does not meet any ASTM Specifications. This is an indirect method of testing and results may vary with depth of surface preparation and location of test on weld. Not responsible for losses due results of examination.

Reviewed / Accepted
I.Q.S. # 3William Brennaugh *WB*



OKLAHOMA FORGE, INC.

Certification of Analysis and Tests

Part # 2 of 10

INVOICE NUMBER
111-66965

Date: 6/20/11

Toll Free (866)909-5553
Local(918)446-5553 FAX (918)446-4150

e-Mail ~ sales@oklahomaforge.com
Web Site ~ www.oklahomaforge.com

P.O. Box 701500
Tulsa, Oklahoma 74170-1500

Sold To	Your Purchase Order	Sales Order
ENERGY EXCHANGER COMPANY 1844 N. GARNETT ROAD TULSA, OK 74116 USA	Number 1199 By JIM ELDER Date 5/17/2011	Number 111-51903 Shipped Via DELIVERY Ship Date 6/20/2011

PART DESCRIPTION			
Your Part Number	Shop Order Number	Shape	Dimensions
Supplier	Heat Number	Material Specifications	
Chemical Analysis			
Physical Properties			
Grain Size			
Charpy Impact Tests			

1	X-7926-4/9	111-51903-01	DISC	16.000"OD x 3.000"TH	2
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED	
	C ✓ Mn ✓ P ✓ S ✓	Si ✓ Ni ✓ Cr ✓ Cu ✓ Mo ✓	V ✓ Cb ✓ Al ✓ Pb	Ce	Other
	.18 1.16 .014 .008	.23 .10 .09 .27 .027	.017 <.001 .021	.42	SN-.011; B-.0003
	Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%/in 2In	Red of Area% Brinell Rckwl C	Bend Test Grain Size Type Temp Average #1 #2 #3
	49,600 ✓	75,500 ✓	34 ✓	68 ✓ 146	FINE CVN -50F 114 127 137 80

2	X-7926-1/1A	111-51903-02	DISC	21.125"OD x 1.625"TH	2
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED	
	C ✓ Mn ✓ P ✓ S ✓	Si ✓ Ni ✓ Cr ✓ Cu ✓ Mo ✓	V ✓ Cb ✓ Al ✓ Pb	Ce	Other
	.18 1.16 .014 .008	.23 .10 .09 .27 .027	.017 <.001 .021	.42	SN-.011; B-.0003
	Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%/in 2In	Red of Area% Brinell Rckwl C	Bend Test Grain Size Type Temp Average #1 #2 #3
	49,600 ✓	75,500 ✓	34 ✓	68 ✓ 146	FINE CVN -50F 114 127 137 80

3	X-7930-4	111-51903-03	DISC	22.000"OD x 3.000"TH	2
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED	
	C ✓ Mn ✓ P ✓ S ✓	Si ✓ Ni ✓ Cr ✓ Cu ✓ Mo ✓	V ✓ Cb ✓ Al ✓ Pb	Ce	Other
	.18 1.16 .014 .008	.23 .10 .09 .27 .027	.017 <.001 .021	.42	SN-.011; B-.0003
	Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%/in 2In	Red of Area% Brinell Rckwl C	Bend Test Grain Size Type Temp Average #1 #2 #3
	49,600 ✓	75,500 ✓	34 ✓	68 ✓ 146	FINE CVN -50F 114 127 137 80

4	X-7930-1	111-51903-04	DISC	27.875"OD x 3.250"TH	2
		ARCELOMITTAL	227N568	SA266-10 GR4 NORMALIZED	
	C ✓ Mn ✓ P ✓ S ✓	Si ✓ Ni ✓ Cr ✓ Cu ✓ Mo ✓	V ✓ Cb ✓ Al ✓ Pb	Ce	Other
	.18 1.16 .014 .008	.23 .10 .09 .27 .027	.017 <.001 .021	.42	SN-.011; B-.0003
	Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%/in 2In	Red of Area% Brinell Rckwl C	Bend Test Grain Size Type Temp Average #1 #2 #3
	49,600 ✓	75,500 ✓	34 ✓	68 ✓ 146	FINE CVN -50F 114 127 137 80

8	X-7930-2	111-51903-08	RING	27.875"OD x 20.500"ID x 4.188"TH	2
		NUCOR	110786	SA266-10 GR4 NORMALIZED	
	C ✓ Mn ✓ P ✓ S ✓	Si ✓ Ni ✓ Cr ✓ Cu ✓ Mo ✓	V ✓ Cb ✓ Al ✓ Pb	Ce	Other
	.17 1.17 .010 .002	.20 .10 .10 .19 .03	.016 .003 .024	.41	SN-.008; TI-.002
	Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%/in 2In	Red of Area% Brinell Rckwl C	Bend Test Grain Size Type Temp Average #1 #2 #3
	48,800 ✓	73,500 ✓	35 ✓	74 ✓ 143	FINE CVN -50F 85 85 104 67

9	X-7929-2/2A	111-51903-09	RING	28.188"OD x 23.000"ID x 3.063"TH	2
		NUCOR	110786	SA266-10 GR4 NORMALIZED	
	C ✓ Mn ✓ P ✓ S ✓	Si ✓ Ni ✓ Cr ✓ Cu ✓ Mo ✓	V ✓ Cb ✓ Al ✓ Pb	Ce	Other
	.17 1.17 .010 .002	.20 .10 .10 .19 .03	.016 .003 .024	.41	SN-.008; TI-.002
	Yield Lb/SqIn	Tensile Strength Lb/SqIn	Elong%/in 2In	Red of Area% Brinell Rckwl C	Bend Test Grain Size Type Temp Average #1 #2 #3
	48,800 ✓	73,500 ✓	35 ✓	74 ✓ 143	FINE CVN -50F 85 85 104 67

Notes:

VERIFIED BY

Page 1

JUN 27 2011

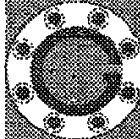
I CERTIFY THAT THERE IS A TRUE COPY OF THE ORIGINAL TEST SHEET NOW ON FILE AT THE OFFICE OF OKLAHOMA FORGE, INC. AND THAT THIS MATERIAL WAS MANUFACTURED IN THE UNITED STATES OF AMERICA.

Anneal by heating to 1450-1550 degrees F for one hour/inch of thinnest section. Slow cool furnace to 700 degrees F then air cool to room temperature.
Normalize by heating to 1650 degrees F, Hold at 1650 degrees F for one hour/inch of thinnest section and air cool.
Solution Anneal by heating to 2000-2050 degrees F and Water Quenching.

Reviewed / Accepted
I.Q.S. #3

L. Satterson

William Bronaugh
WB

	GALPER, INC. 180 Southbelt Industrial Drive Houston, Texas 77047 PH. 713-433-0700 - FAX 713-433-5580 Email: galpermail@galper-inc.com		PROGRESSIVE SUPPLY P. O. BOX 470748 TULSA OK 74145															
	TEST/CERTIFICATE EN 10204 INTERNAL JOB		091939															
	3.1B		091939															
M.T. CERTIFICATE N. DATE YOUR P.O. DATE		015325/B 08/06/2009 098352 X-7929-81 07/13/2009																
ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#												
013/013	10" 300 WN RF XS A105 B1 F	10	B	ASTM A/SA105 03c	3000773	A												
015/015	20" 300 WN RF XS A105 B1 F	2	B	ASTM A/SA105 03c	080474N/02	B												
016/016	10" 300 WN RF 80 A105 B1 F	10	B	ASTM A/SA105 03c	840491/02	C												
017/017	4" 300 WN RF 120 A105 B1 F	10	B	ASTM A/SA105 03c	3000972	D												
CHEMICAL ANALYSIS																		
#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NE %	TI %	N %	CB %	B %	FL %
A	0.180	0.910	0.190	0.008	0.018	0.130	0.120	0.037	0.260	0.0001			0.002			0.002		0.39
A C	0.170	0.930	0.210	0.011	0.015	0.120	0.130	0.047	0.270	0.0001			0.002			0.002		0.38
B	0.160	1.200	0.210	0.010	0.002	0.140	0.040	0.020	0.110	0.0200		0.020	0.0001	0.001		0.0001	0.0003	0.40
B C	0.150	1.220	0.230	0.013	0.002	0.130	0.030	0.030	0.120	0.0150		0.030	0.0001	0.001		0.0001	0.0003	0.39
C	0.180	1.180	0.240	0.015	0.002	0.110	0.010	0.0001	0.020	0.0030		0.034	0.0001	0.021	0.007	0.0001		0.40
C C	0.170	1.200	0.260	0.018	0.002	0.100	0.020	0.0001	0.030	0.0030		0.035	0.0001	0.021	0.007	0.0001		0.39
D	0.190	0.900	0.190	0.011	0.011	0.110	0.080	0.0230	0.220	0.0010			0.0020			0.0020		0.38
D C	0.180	0.920	0.170	0.008	0.008	0.100	0.090	0.0330	0.230	0.0010			0.0020			0.0020		0.38
NOTES: FL - Carbon Equivalent Long Formula C = CHECK ANALYSIS																		
MECHANICAL PROPERTIES																		
#	TYPE	Ø DIAMETER mm.	AREA mm²	LENGTH mm.	TEMP. F	TENSILE				CHARPY TEST				HB HARDNESS (AVERAGE)				
						TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES						
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187				
A	O	12.50	122.70	50.00	68	78,133	46,654	30.0	70.0					150				
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187				
B	O	12.50	122.70	50.00	68	77,785	47,188	31.0	67.3					159				
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187				
C	O	12.50	122.70	50.00	68	78,183	47,036	26.6	66.8					153				
D	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0					187				
D	O	12.50	122.70	50.00	68	78,172	47,015	27.6	69.3					158				
NOTES: O = OBTAINED S = STANDARD																		
LINE 013 COUNTRY OF ORIGIN USA ✓																		
#	HEAT TREATMENTS										GENERAL NOTES							
A	Normalized at 900 dgr.C for 2 h.-Cooling from 900 dgr.C in still air										ROUGHNESS F - STOCK FINISH 125-250 microin STANDARD BY - ASME B16.5 JUN 16 2011 EEC QC DEPT <i>L. Jefferson</i> GALPER, INC An ISO 9001/2000 certified company. Certification issued by: Lloyds register quality assurance Certificate No. UQA 0110391							
B	Normalized at 900 dgr.C for 3 h.-Cooling from 900 dgr.C in still air																	
B																		
C	Normalized at 900 dgr.C for 2 h.-Cooling from 900 dgr.C in still air																	
C																		
D	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air										VERIFIED William Brennaugh I.Q.S. # 3 Reviewed / Accepted							
NOTES: Material I.A.W. Nace MR-0175-2002																		
TRADE MARK / LOGO		INT. INSPECT DATE				INT. INSPECTOR				INSPECTION AUTHORITY <i>P. Ruzicka</i>								
Stamped on product 		08/06/2009				.RUZICKA												

*Part #3*CUSTOMER ORDER NUMBER 119490
SOLD/SHIPPED TO PROGRESSIVE SUPPLY
P.O. BOX 470748

This MTR applies to your

TULSA

OK 74147

USA

PO#

1221

Item Quantity Description

6 12 2" X 300 X 9" LG RF LWN

*X-7929-82489**Noz. V1, D1*HEAT NUMBER:
W019MATERIAL TYPE:
SA105NMaterial Origin:
USAManufactured in
USA

I.A.W. ASME 2010 ED.

CHEMICAL ANALYSISPHYSICAL PROPERTIES

C	.180 ✓
Mn	1.040 ✓
P	.0060 ✓
S	.0000 ✓
Si	.230 ✓
Cr	.140 ✓
Mo	.030 ✓
V	.014 ✓
Cu	.190 ✓
Ni	.070 ✓
Cb	.000 ✓
Sn	.013
As	.003
Sb	.001
Ti	.0010
Al	.026
B	.0000
CE	.40

Yield PSI	48,600 ✓
Tensile PSI	71,600 ✓
Elongation	33 ✓
Reduction of Area	74 ✓
Hardness	137 HBW ✓

VERIFIED BY

JUN 16 2011

EEC QC DEPT

*G. Satterson*Heat Treatment NORMALIZED ✓
Temperature 1650 °F
Time at Temperature 1/2 HR/IN THK MIN
Cooling Media AIR

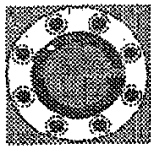
We hereby certify that all test results and process information contained herein are correct and true as contained in the records of the company.

Prepared by

Name: Helen Fielder

Title: QA Representative

Reviewed / Accepted
I.Q.S. #3*William Brennaugh**WZ*



GALPER, INC.
180 Southbelt Industrial Drive
Houston, Texas 77047
Ph: 713-433-0700 - FAX 713-433-6580
Email: galper@galper-inc.com

Noz. VI, DI

PROGRESSIVE SUPPLY

P. O. BOX 470748
TULSA OK 74145

This MTR applies to your
PO# 1221

TEST/CERTIFICATE

EN 10204

INTERNAL JOB

3.1B

111166

M.T. CERTIFICATE N.

DATE

YOUR P.O.

DATE

003083/B

04/07/2011

11-9523

03/31/2011

ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#
004/004	12" 600 WN RF 80 A105 B1 F	4	B	ASTM A/SA105 03c	102817N/02	A
005/005	8" 150 WN RF XS A105 B1 F	50	B	ASTM A/SA105 03c	100506N/02	B
006/006	2" 300 BL RF A105 B1 F	200	B	ASTM A/SA105 03c	3019785	C

CHEMICAL ANALYSIS

#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CB %	FL %
A	0.170	1.170	0.190	0.008	0.002	0.080	0.080	0.030	0.150	0.017		0.026	0.002			0.002	0.40
A C	0.160	1.190	0.210	0.005	0.002	0.090	0.070	0.040	0.160	0.012		0.037	0.002			0.002	0.40
B	0.170	1.150	0.260	0.011	0.001	0.110	0.070	0.040	0.200	0.016		0.028	0.003			0.003	0.41
B C	0.160	1.130	0.280	0.014	0.001	0.100	0.060	0.050	0.210	0.011		0.048	0.003			0.003	0.39
* C	0.190	1.050	0.230	0.006	0.017	0.050	0.090	0.020	0.190	0.003			0.001			0.001	0.39
C	0.180	1.070	0.250	0.009	0.020	0.060	0.100	0.010	0.180	0.003			0.001			0.001	0.39

NOTES
FL=Carbon Equivalent Long Formula

C = CHECK ANALYSIS

MECHANICAL PROPERTIES

#	TYPE	TENSILE								CHARPY TEST			
		Ø DIAMETER mm.	AREA mm²	LENGHT mm.	TEMP. F	TENSILE PSI	YIELD PSI	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES	HB HARDNESS (AVERAGE)
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	O	12.50	122.70	50.00	68	77,523	50,416	28.1	66.0				162
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	O	12.50	122.70	50.00	68	77,993	49,943	30.0	68.0				152
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
* C	O	12.50	122.70	50.00	68	77,474	49,259	27.0	70.0				155

NOTES

O = OBTAINED
S = STANDARD

HEAT TREATMENTS

A	Normalized at 900 dgr.C for 3 h.-Cooling from 900 dgr.C in still air
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air
B	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air
C	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air

GENERAL NOTES

ROUGHNESS
F - STOCK FINISH 125-250 microinch

VERIFIED BY

STANDARD
B1 JUN 16 2011

Material I.A.W. Nace MR-0175-2003 COUNTRY OF ORIGIN:USA ✓

EEC QC DEPT

TRADE MARK / LOGO

INT. INSPECT DATE

INT. INSPECTOR

Stamped on product

04/07/2011

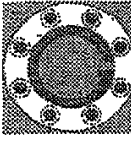
D.RUZICKA

INSPECTION AUTHORITY

GALPER, INC

An ISO 9001/2008 certified company.
Certification issued by:
Lloyds register quality assurance
Certificate No. UQA 0110391

Part #3

	GALPERCI, INC. 180 Southbelt Industrial Drive Houston, Texas 77047 PH. 713-433-0700 - FAX 713-433-6680 Email: galperci@galperci-em.com		PROGRESSIVE SUPPLY P. O. BOX 470748 TULSA OK 74145	
	TEST/CERTIFICATE EN 10204 INTERNAL JOB		This MTR applies to your PO# <u>1221</u>	
	3.1B		110099	

M.T. CERTIFICATE N.		DATE		YOUR P.O.		DATE	
000532/B		01/18/2011		11-9480		01/10/2011	

ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#
002/002	16" 150 WN RF XS A105 B1 F	3	B	ASTM A/SA105 03c	10411W/02	A
004/004	10" 300 WN RF XS A105 B1 F	10	B	ASTM A/SA105 03c	3000773	B
006/006	2" 300 WN RF XS A105 B1 F	15	B	ASTM A/SA105 03c	3016659	C
007/007	6" 300 WN RF XS A105 B1 F	50	B	ASTM A/SA105 03c	102250N/02	D

CHEMICAL ANALYSIS																			
#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CO %	B %	FL %	
A	0.170	1.200	0.250	0.009	0.007	0.090	0.110	0.040	0.160	0.022	0.010	0.027	0.0001	0.003	0.010	0.006	0.0003	0.41	
A C	0.160	1.220	0.270	0.012	0.010	0.100	0.120	0.030	0.170	0.017	0.010	0.036	0.0001	0.003	0.010	0.006	0.0003	0.41	
B	0.180	0.910	0.190	0.008	0.018	0.130	0.120	0.037	0.260	0.0001			0.0020					0.39	
B C	0.170	0.930	0.210	0.011	0.015	0.120	0.130	0.047	0.270	0.0001			0.0020					0.38	
C	0.190	1.040	0.220	0.008	0.019	0.090	0.070	0.020	0.230	0.0030		0.002	0.0010				0.0001	0.40	
C C	0.180	1.060	0.240	0.005	0.022	0.080	0.080	0.030	0.240	0.0030		0.010	0.0010				0.0001	0.40	
D	0.180	1.150	0.180	0.008	0.006	0.110	0.100	0.050	0.180	0.0170		0.025	0.0010					0.42	
D C	0.170	1.170	0.200	0.011	0.009	0.120	0.110	0.060	0.190	0.0120		0.028	0.0010					0.42	

NOTES
FL=Carbon Equivalent Long Formula

C = CHECK ANALYSIS

MECHANICAL PROPERTIES													
#	TYPE	TENSILE						CHARPY TEST					
		Ø DIAMETER mm	AREA mm²	LENGHT mm	TEMP. F	TENSILE PSI ✓	YIELD PSI ✓	ELONG %	REDUCT. %	TYPE	TEMP.	VALUES	HB HARNESS (AVERAGE)
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	O	12.50	122.70	50.00	68	78,868	47,920	29.6	66.5				154
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	O	12.50	122.70	50.00	68	78,566	48,153	27.0	67.1				156
C	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
C	O	12.50	122.70	50.00	68	78,362	47,533	27.2	66.8				162
D	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
D	O	12.50	122.70	50.00	68	78,457	46,889	29.9	69.6				151

NOTES
O = OBTAINED
S = STANDARD

#	HEAT TREATMENTS	GENERAL NOTES
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	ROUGHNESS F = STOCK FINISH 125-250 microinch VERIFIED BY STANDARD JUN 14 2011
B	Normalized at 900 dgr.C for 2 h.-Cooling from 900 dgr.C in still air	
C	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air	
D	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	

NOTES
Material I.A.W. Nace MR-0175-2003 COUNTRY OF ORIGIN: USA ✓

EEC QC DEPT
R. J. Jetterson

TRADE MARK / LOGO	INT. INSPECT DATE	INT. INSPECTOR	GALPERCI, INC An ISO 9001/2008 certified company. Certification issued by: Lloyds register quality assurance Certificate No. UQA 0110391
Stamped on product 	01/18/2011	D. RUZICKA	Reviewed / Accepted I.Q.S. #3 <i>William Brounagh</i>
INSPECTION AUTHORITY <i>Dion Ruzicka</i>			

04/08/2009 From: MATTSO SUPPLY CO.

To: ENERGY EXCHANGERS

Our# : 31620

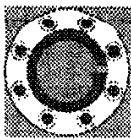
Line# : 1

QTY: : 6

To: : ENERGY EXCHANGERS

CPO# : 4326

X-7929 - 730, 740, 750 Part # 3

**GALPERT, INC.**160 Southbelt Industrial Drive
Houston, Texas 77047
PH: 713-433-0700 - FAX 713-433-6580
Email: galpmal@galperti-am.com

TEST/CERTIFICATE

EN 10204

INTERNAL JOB

083305

MATTSO SUPPLYP.O. BOX 2925
TULSA OK 74101

Noz. 53, 12, D2

M.T. CERTIFICATE N.

011219/B

DATE

10/21/2008

YOUR P.O.

946-S-09A

DATE

10/13/2008

ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#
024/024	3" 600 WN RF 160 A105 B1 F	15	A	ASTM A/SA105 03c	252597	A
024/024	3" 600 WN RF 160 A105 B1 F	5	B	ASTM A/SA105 03c	3832/02	B
026/026	6" 150 BL RF A105 B1 F	20	B	ASTM A/SA105 03c	30469/02	C
027/027	0.75" 300 BL RF A105 B1 F	25	B	ASTM A/SA105 03c	79209	D

#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CB %	FL %
A	0.200	0.910	0.200	0.010	0.023	0.140	0.090	0.028	0.320	0.0001			0.002			0.002	0.41
A C	0.190	0.930	0.180	0.013	0.020	0.150	0.100	0.018	0.330	0.0001			0.002			0.002	0.40
B	0.175	1.140	0.190	0.009	0.010	0.095	0.210	0.036	0.220	0.0010	0.014	0.020	0.001		0.009	0.001	0.42
B C	0.165	1.160	0.170	0.012	0.013	0.105	0.220	0.046	0.230	0.0010	0.014	0.040	0.001		0.009	0.001	0.41
C	0.180	1.190	0.250	0.011	0.009	0.100	0.060	0.010	0.180	0.0230	0.009	0.025	0.002	0.021		0.002	0.42
C C	0.170	1.210	0.230	0.014	0.012	0.110	0.070	0.020	0.190	0.0180	0.009	0.039	0.002	0.021		0.002	0.41
D	0.185	1.120	0.220	0.013	0.013	0.090	0.060	0.010	0.200	0.0240						0.001	0.41
D C	0.195	1.140	0.240	0.010	0.010	0.100	0.070	0.020	0.190	0.0340						0.001	0.43

NOTES
FL-Carbon Equivalent Long Formula

C = CHECK ANALYSIS

#		MECHANICAL PROPERTIES												
		TENSILE								CHARPY TEST				
		TYPE	Ø DIAMETER	AREA	LENGHT	TEMP.	TENSILE	YIELD	ELONG	REDUCT.	TYPE	TEMP.	VALUES	HB HARDNESS (AVER-POE)
A	S		12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	O		12.50	122.70	50.00	68	76,790	50,217	27.6	67.6				151
B	S		12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	O		12.50	122.70	50.00	68	77,987	50,277	31.0	68.5				157
C	S		12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
C	O		12.50	122.70	50.00	68	76,572	49,075	29.0	66.0				157
D	S		12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
D	O		12.50	122.70	50.00	68	77,220	50,301	29.0	69.8				164

NOTES

O = OBTAINED
S = STANDARD

#	HEAT TREATMENTS	GENERAL NOTES
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	ROUGHNESS
B	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	F = STOCK FINISH 125-250 microinch
C	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air	
D	Normalized at 900 dgr.C for 1 h.-Cooling from 900 dgr.C in still air	STANDARD B1 = ASME B16.5

NOTES
Material I.A.W. Nace MR-0175-2002

TRADE MARK	INT. INSPECT DATE	INT. INSPECTOR
	10/21/2008	D. RUZICKA
INSPECTION AUTHORITY		

CHECKED APR 15 2009

K. JattersonReviewed / Accepted
I.Q.S. # 3*William Brannagh*

04/08/2009 From: MATTSO SUPPLY CO.

To: ENERGY EXCHANGERS

Our# : 31620

Line# : 2

QTY: : 6

To: : ENERGY EXCHANGERS

CPO# : 4326

X-7929-73, 74, 75

Part # 3



A DIVISION OF AMERI-FORGE GROUP INC.

MATERIAL TEST REPORT

MTR # A183731

Forged Vessel Connections
2525 De Soto St. (77091)
P.O. Box 38421 (77238-8421)
Houston, Texas

http://www.fvc.cc

QA@fvc.cc

TEL: (713) 688-9705 FAX: (713) 688-7954

Noz. 53, V2, D2

S.O. No: 8-02434

Date: 10/21/2008

REV #

Purchaser: Mattsco Supply Co.

Distributor:

P.O. No.: 853-S-08

Distributor's Order No:

Page 1

ITEM NO	QTY	DESCRIPTION	SPEC	HEAT #	HEAT TREAT
6	1	LWN, SA105, 300#, .75 x 9", RF&D J55433-0 J55433-1	SA105 2007 Edition, 2008a Adden	A606AF	As-Forged

$$(C + (Mn / 6)) + ((Cr + Mo + V) / 5) + ((Ni + Cu) / 15) = .442$$

CHEMICAL ANALYSIS AND MECHANICAL PROPERTIES

Heat #: A606AF

11789240

Chemical values below represent percentage of weight

C (Carbon) .19 ✓ *CAS # 7349-96-5	Mn (Manganese) 1.08 ✓ *CAS # 7349-96-5	P (Phosphorus) .018 ✓ *CAS # 7723-14-0	S (Sulphur) .022 ✓	Si (Silicon) .26 ✓	Cr (Chromium) .19 ✓ *CAS # 7440-47-3	Ni (Nickel) .11 ✓ *CAS # 7440-02-0	Mo (Molybdenum) .05 ✓ *CAS # 1313-27-5
Cu (Copper) .32 ✓ *CAS # 7440-50-8	Al (Aluminum) .006 ✓	V (Vanadium) .002 ✓ *CAS # 7440-62-2	As (Arsenic)	Sb (Antimony)	Sn (Tin) .013	Co (Cobalt)	N (Nitrogen)
Ti (Titanium)	Ta (Tantalum)	Cb (Columbium) .013 ✓	O (Oxygen)	H (Hydrogen)			

TENSILE (PSI)	YIELD (PSI)	ELONG (IN 2")	R.A. (%)	CHARPYS (FT - LBS)	TEMP ° F
82988	58388	28.0	66.0		

SHEAR %	LATERAL EXPANSION (MILS)	BHN
		< 187

FVC STANDARD MANUFACTURING PRACTICES:

- 1) FLANGE DIMENSIONS & BOLTING COMPLY - ASME B16.5
- 2) SELF-REINFORCING NOZZLES COMPLY WITH SPECIFICATION OF SECT. VIII DIV. 1 OR DIV. 2 PER CUSTOMER DESIGN/FVC DESIGN.
- 3) MATERIAL SPECIFICATION COMPLIES WITH ASME SECT. II, PART A.
- 4) ALL FVC MATERIAL MANUFACTURED PER FINE GRAIN PRACTICE.
- 5) ALL FVC MATERIAL CERTIFIED PER EN 10204, 3.1.B UNLESS OTHERWISE SPECIFIED
- 6) CERTIFICATE OF INSURANCE AVAILABLE UPON REQUEST.

Test Bar: 187-187 BHN ✓

CHECKED APR 15 2009

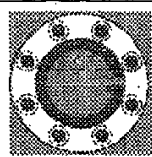
EPCRA Supplier Notification:

* This product may contain one or more toxic chemicals subject to the reporting requirements of Section 313 of the Emergency Planning and Community Right-to-Know Act (Title III of the Superfund Amendments and Reauthorization Act of 1986) and 40 C.F.R. Part 372. Potentially reportable chemicals are indicated with an asterisk (*) under the chemical's name and percent-by-weight value, followed by a Chemical Abstract Service (CAS) registry number for that chemical. It is your responsibility alone to determine whether your facility is required to submit a Toxic Release Inventory Report under EPCRA Section 313.

Reviewed / Accepted
I.Q.S. # 3

William Brunaugh

Quality Assurance



GALPER, INC.
180 Southbelt Industrial Drive
Houston, Texas 77047
PH: 713-433-0700 - FAX 713-433-5580
Email: galper@galper-inc.com

Noz 02

PROGRESSIVE SUPPLY

part #3

P. O. BOX 470748
TULSA OK 74145

This MTR applies to your

PO# *1221*

TEST/CERTIFICATE

EN 10204

INTERNAL JOB

091351

3.1B

M.T. CERTIFICATE N.

014577/B

DATE

07/06/2009

YOUR P.O.

098331

DATE

05/15/2009

17924-72

ITEM	DESCRIPTION	QUANTITY	B/P	MATERIAL	HEAT N°	#
016/016	3" 300 WN RF XXS A105 B1 F	10	B	ASTM A/SA105 03c	9721/02	A
018/018	3" 300 WN RF 160 A105 B1 F	25	B	ASTM A/SA105 03c	9721/02	B

#	C %	MN %	SI %	P %	S %	CR %	NI %	MO %	CU %	V %	SN %	AL %	NB %	TI %	N %	CB %	FL %
A	0.170	1.170	0.214	0.010	0.016	0.130	0.185	0.050	0.180	0.001		0.020	0.002	0.001	0.009	0.002	0.42
A C	0.160	1.150	0.234	0.013	0.019	0.120	0.175	0.060	0.190	0.001		0.030	0.002	0.001	0.009	0.002	0.41

NOTES
FL=Carbon Equivalent Long Formula

C = CHECK ANALYSIS

#	MECHANICAL PROPERTIES												
	TENSILE									CHARPY TEST			
	TYPE	Ø DIAMETER	AREA	LENGTH	TEMP.	TENSILE	YIELD	ELONG	REDUCT.	TYPE	TEMP.	VALUES	HB HARDNESS (AVERAGE)
		mm.	mm ² .	mm.	F	PSI	PSI	%	%				
A	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
A	Q	12.50	122.70	50.00	68	78,586	47,781	28.5	66.4				159
B	S	12.50	122.70	50.00	68	70,000	36,000	22.0	30.0				187
B	Q	12.50	122.70	50.00	68	78,487	47,214	29.0	68.0				154

NOTES

O = OBTAINED
S = STANDARD

LINE 016 COUNTRY OF ORIGIN ITALY ✓

#	HEAT TREATMENTS	GENERAL NOTES
A	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	ROUGHNESS F -STOCK FINISH 125-250 microinch
A		
B	Normalized at 900 dgr.C for 1.1/2 h.-Cooling from 900 dgr.C in still air	STANDARD B1 - ASME B16.5
B		

VERIFIED BY

JUN 16 2011

NOTES

Material I.A.W. Nace MR-0175-2002

EEC QC DEPT

TRADE MARK / LOGO	INT. INSPECT DATE	INT. INSPECTOR	GALPER, INC An ISO 9001/2000 certified company Certification issued by: Lloyds register quality assurance Certificate No. UQA 0110391
Stamped on product	07/06/2009	D.RUZICKA	
INSPECTION AUTHORITY <i>Dion Ruzicka</i>			Reviewed / Accepted I.Q.S. #3 <i>William Drenth</i>

1221



VALLOUREC & MANNESMANN TUBES

V & M do BRASIL S.A.

BARREIRO PLANT - Belo Horizonte - MG - Brazil

CEP: 30161-970 - P.O BOX: 1453


 ISO 9001 NUMBER 3872
 ISO 14001 NUMBER 4712
 ISO 9001 NUMBER 3872
 ISO 14001 NUMBER 4712

 INSPECTION CERTIFICATE N°.: She
 (According to EN 10204 / DIN 50049 3.1.B) 0050229660 1 / 2

 Work Order: 180400 / 000030
 Material Number: 376816

Country: Canada

Customer Material Number:

Customer: VALLOUREC TUBES CANADA INC

Customer Order: VM1678

Inspection: V & M do BRASIL S.A.

Vessel: DS FREEWAY

DIMENSION: 10.3/4" X 0.500" GRADE: GR 6 /GR 1 /

METALLURGICAL STANDARD: ASTM A 333 01.01.2005 # DIMENSIONAL STANDARD: ASTM A 333 01.01.2005

IN ACCORDANCE ALSO TO THE STANDARDS: ASME SA-333 / 2004 #

PRODUCT: SEAMLESS STEEL PIPE, HOT FINISHED, BEVELED ENDS 37.5 DEG. # NORMALIZED # SUPERFICIAL PROTECTION: LACQUER #

TOLERANCES: OUTSIDE DIAMETER: - 0.0315 " / + 0.0945 " WALL THICKNESS: - 0.0625 " / + 0.1125 "

CUT LENGTH: RANDOM 18.00 FT - 22.00 FT #

STANDARD MARKING: 175171/30 VMB ASTM A/ASME SA-333 6 1 S HF SCH 060 LT -50 F 10.3/4" X 0.500"

SHIPPING MARKING: MADE IN BRAZIL * VM1678 / P.O L-1891 * EDGEN * EDMONTON VIA HOUSTON

X-7929-81A Noz.Ta

Batch	Heat N.	Inspection Lot	Pieces
0006268359	83776	030001159262	30
TOTAL			30

THE PRODUCT IS SATISFACTORY IN THE FOLLOWING TESTS / INSPECTIONS: DIMENSIONAL # VISUAL # FLATTENING TEST # HYDROSTATIC TEST: 2.000.0 PSI 5 s #

CHEMICAL COMPOSITION(%):

 C $\checkmark$ Mn $\checkmark$ P $\checkmark$ S $\checkmark$ Si $\checkmark$ Ni $\checkmark$ Cr $\checkmark$ Mo $\checkmark$ Al $\checkmark$ Cu $\checkmark$ Sn $\checkmark$ V $\checkmark$ Nb $\checkmark$ Ti $\checkmark$ B $\checkmark$ Fe $\checkmark$

0.10

Required: Min 0.22 1.06 0.025 0.025

Heat N./ Batch:

83776/0006268359 0.11 0.92 0.010 0.003 0.17 0.02 0.05 0.01 0.010 0.000

Check1 0.11 0.92 0.011 0.002 0.17 0.03 0.05 0.01 0.004 0.000

Check2 0.11 0.91 0.010 0.002 0.17 0.03 0.05 0.01 0.004 0.000

MECHANICAL TESTS

Tensile

Required

Min.:

Max.:

Area (Sq.in)

Ys(TSD)

34835

35

S.Direction: Longitudinal

Ts(TSD)

60235

35

Temperature: Room Temperature

R.A.O

35

Gage Length: 10=2"

Ys/Ts

35

VERIFIED BY

JUL 11 2011

EEC QC DEPT

K. Jatterson

Reviewed / Accepted

I.Q.S. # 3

William Drenough

Part #4



V&M VALLOUREC & MANNESMANN TUBES



INSPECTION CERTIFICATE
(According to EN 10204 / DIN 50049 3.1.B) N°: 0050229660 Site 2 / 2

Heat N./ Batch 83776 / 0006268359	0.5	49930	66522	45	HY ✓
Hardness Required Min.: Max.: Heat N./ Batch 83776 / 0006268359					
Impact	134				
According: CHARPY 10X5X10 V NOTCH					
Absorbed Energy					
Heat N./ Batch 83776 / 0006268359	1	2	3	Average Min 13 Fl.b	Temperature: -50 °F ✓
	176	178	167	174	Lateral Expansion
					Required Min:
					1 2 3
					Average Min
					Average

Remarks:

MATERIAL IN ACCORDANCE TO ITEM A-2 NACE MR 0175/ISO 15156-2/2003

Part # 4
Reviewed / Accepted
I.Q.S. # 3
Branagh UK

We hereby certify that this product has been manufactured and examined in accordance with all requirements of the standards and specifications and all the results are found to be satisfactory. This testimonial and certificate respectively is recorded by a computer system and is valid without signature. Alteration or use for others products are regarded as falsification of documents and will be subject to criminal jurisdiction.

Eco Tubes: The tubes from V&M do BRASIL are manufactured with steel which uses charcoal as a source of energy in its production. This coal comes from more than 100,000 ha of forest planted by V&M Florestal. With the acquisition of 15,8 ton(s) of steel tubes from V&M do BRASIL, your company contributed to the reduction of the greenhouse effect, avoiding the accumulation of 28,4 ton(s) of Carbon D

QUALITY CONTROL DEPARTMENT
FAX: (55-31) 3328-2617
e-mail: jussilva@vmtubes.com.br
DR. JULES MARCO JULYANA S SILVA - CHEF/MS/20
TECHNICAL RESPONSIBLE
DATE 05.07.20

DATE: 02/04/11
TIME: 11:04:35
SERIAL NO: F0062922

VEHICLE ID	VEHICLE TYPE	VEHICLE MAKE	VEHICLE MODEL	VEHICLE YEAR	VEHICLE COLOR	VEHICLE VIN	VEHICLE LICENSE	VEHICLE REGISTRATION	VEHICLE INSURANCE	VEHICLE OWNER	VEHICLE ADDRESS	VEHICLE CITY	VEHICLE STATE	VEHICLE ZIP	VEHICLE PHONE	VEHICLE FAX	VEHICLE EMAIL	VEHICLE WEBSITE	VEHICLE NOTES
1	Car	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
2	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
3	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
4	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
5	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
6	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
7	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
8	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
9	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
10	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
11	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
12	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
13	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
14	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
15	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
16	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555	555-555-5555	john.doe@ford.com	www.ford.com	
17	Truck	Ford	Ford	2000	Black	1F3C10C100A100000	123456789	123456789	123456789	John Doe	1234 Main St	Anytown	CA	90210	555-555-5555				

VENDOR

USS TUBULAR PRODUCTS
USS FAIRFIELD WORKS
P.O. BOX 599
FAIRFIELD ALABAMA 3500

FAIRFIELD ALABAMA 35064

SPECIFICATION AND GRADE

STEEL PIPE API 5L-44TH ED DTD OCT 2007 AND ISO 3183:2007 MOD PSL-2 GRADE B AND GRADE C E ASTM A53-07 ASTM A106-08 GRADE B QUAD STENCIL ASME SA53-2010 EDITION GRADE B ASME SA106-2010 EDITION GRADE B/C BLK REG MILL COAT PE BEV 30 DEG MEETING ALL THE APPLICABLE REQUIREMENTS OF NACE STANDARD MR-01-75 *2003/COR.1:2005 AND MR0103-2007

(2 pcs.) Stock X-7929-71A

MATERIAL COND.		AS-ROLLED		O.D.		WALL		IN (mm)		WALL		IN (mm)	
				5.625 (168.275)		0.432 (10.972)							
PRODUCT IDENTIFICATION	TEST TYPE/ ORIENTATION	TEST COND.	GAUGE WIDTH	IN	YIELD	EXT %	TENSILE	PSI	VT	ELONG % (IN 2")	HARDNESS SCALE: HRB	MIN HYDRO PSI	RWALLS
X22424	*NOTE A	STRIP/L/B	AR	1.5	49,800	.50	75,500	0.66	38.3	85.4	4110	5	
** END OF DATA THIS SHEET **													

LEGEND:		L-LONGITUDINAL U-UPSET		T-TRANSVERSE RM-NORMALIZED		QT-QUENCH & TEMPERED SR-STRESS RELIEVED		AR-AS ROLLED TR-THERMOMECHANICAL ROLLED		B-BODY		W-WELD											
PRODUCT IDENTIFICATION	TYPE	C	G	MN	P	F	S	SI	CU	Ni	CR	MO	AL	N	V	S	TI	CB	CO	SN	ZNS	SB	CE-MAX
X22424	HEAT	19	11.1	005	007	22	03	01	02	01	032005	00	0001	001	000	00	00	003	-43				
X22424	PROD	21	106	006	005	21	03	02	03	02	032007	00	0003	002	005	007	00	00	00	00	00	00	00
X22424	PROD	20	114	007	008	23	03	01	03	01	030006	00	0000	001	000	003	00	00	003	00	00	00	00
** END OF DATA THIS SHEET **																							

C.E. = C + MN/5 + (CR+MO+ V)/5 + (CU+NI)/15 <= 0.43

*C.E. IS BASED ON THE FOLLOWING EQUATIONS:

VERIFIED BY

DECIMAL POSITIONS FOR ELEMENTS ARE INDICATED BY THE LEFT MARGIN, VERTICAL DOTTED LINE OR DECIMAL POINT.

APR 14 2011

FEC OG DEPT

Reviewed / Accepted
I.Q.S. # 3

William Brewster

Kemper County MM209937

Part #4

DATE: 02/04/11
TIME: 11:04:35
SERIAL NO: F0062922

**TUBULAR PRODUCTS
CERTIFIED TEST REPORT
(IN ACCORDANCE WITH ISO 10474/EN10204/DIN50048 "type 3.1")**

UNITED STATES STEEL



MILL ORIENTATION NO	SHIPPERS NO.	P.O. NUMBER	IN (mm)	WALL THICKNESS (mm)	IN (mm)
DRO0486 01		D1721000	6.625 (169.275)	0.432 (10.972)	
MATERIAL CONDITION: AS-ROLLED					
PRODUCT IDENTIFICATION	FLAT	BEND	GRAIN SIZE	MIN COLLAPSE	DIR
X22424	OK		** END OF DATA THIS SHEET **	T B + 32 3/4 AR	TEST COND. 1 2 3 AVG 1 2 3
LEGEND L - LONGITUDINAL T - TRANSVERSE B - BODY W - WELD HAZ - HEAT AFFECTED ZONE					
TEST / INSPECTION RESULTS / COMMENTS					
FULL LENGTH VISUAL	YES X				
FULL LENGTH EMI	X				
FULL LENGTH MPI		OD	OD/DI	X	L UT X 10.0/10.0% NOTCH
FULL LENGTH UT		ID	OD/DI		L UT
END AREA INSPECTION (PLAIN END)	X	MPI	X	UT	
SPECIAL END AREA (SEA) INSP		MPI		UT	
FULL LENGTH DRIFT		DRIFT MANDEL SIZE:			
ADDITIONAL NOTES/COMMENTS					
ALL MELTING AND MANUFACTURING TOOK PLACE IN THE USA. MANUFACTURED IN AN ISO 9001 CERTIFIED FACILITY - CERTIFICATE #30727. NO REPAIRS BY WELDING. NO MERCURY OR MERCURY COMPOUNDS ARE ADDED TO THE STEEL AND ALL MERCURY BEARING EQUIPMENT IS PROTECTED BY A DOUBLE BOUNDARY OF CONTAINMENT. **NOTE A SEE ATTACHED REPORT FOR TENSILE RESULTS. ** END OF DATA **					

THIS IS TO CERTIFY THAT THE PRODUCT DESCRIBED HEREIN WAS MANUFACTURED,
SAMPLED, TESTED AND/OR INSPECTED IN ACCORDANCE WITH THE SPECIFICATION
AND FULFILLS THE REQUIREMENTS IN SUCH RESPECTS.

PREPARED BY THE OFFICE OF:
J. MAJERZAK - MANAGER. O.A.

DATE 02/04/11

PAGE 2 OF 2

Reviewed / Accepted

I.Q.S. #3

William Brewster

Part #4

		UNITED STATES STEEL		TUBULAR PRODUCTS TENSILE TEST REPORT		DATE: 02/04/11 TIME: 11:04:51 PAGE: 1																																																					
BILL ORDER ITEM NO DR00486 01		SHIPPER'S NO.		P.O. NUMBER D1721000		SERIAL NUMBER: F0062922																																																					
SOLD TO ADDRESS PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145		MAIL TO ADDRESS		HEAT		VENDOR USS TUBULAR PRODUCTS USS FAIRFIELD WORKS P.O. BOX 599 FAIRFIELD ALABAMA 35054																																																					
<table border="1"><thead><tr><th colspan="2">PIPE/</th><th colspan="2">TYPE/</th><th colspan="2">TEST</th><th colspan="2">REDUC</th></tr><tr><th>HEAT</th><th>LOT</th><th>DIR/</th><th>LOC</th><th>COND</th><th>YIELD</th><th>EXT%</th><th>TENSILE</th><th>Y/T</th><th>ELONG</th><th>% AREA</th></tr></thead><tbody><tr><td>X22424</td><td>2013</td><td>STRIP/L/B</td><td>AR</td><td>49,800</td><td>0.50</td><td>75,500</td><td>0.66</td><td>38.3</td><td></td><td></td></tr><tr><td>X22424</td><td>2012</td><td>STRIP/L/B</td><td>AR</td><td>51,000</td><td>0.50</td><td>75,500</td><td>0.68</td><td>40.0</td><td></td><td></td></tr><tr><td>X22424</td><td>2011</td><td>STRIP/L/B</td><td>AR</td><td>51,500</td><td>0.50</td><td>76,000</td><td>0.68</td><td>43.0</td><td></td><td></td></tr></tbody></table>								PIPE/		TYPE/		TEST		REDUC		HEAT	LOT	DIR/	LOC	COND	YIELD	EXT%	TENSILE	Y/T	ELONG	% AREA	X22424	2013	STRIP/L/B	AR	49,800	0.50	75,500	0.66	38.3			X22424	2012	STRIP/L/B	AR	51,000	0.50	75,500	0.68	40.0			X22424	2011	STRIP/L/B	AR	51,500	0.50	76,000	0.68	43.0		
PIPE/		TYPE/		TEST		REDUC																																																					
HEAT	LOT	DIR/	LOC	COND	YIELD	EXT%	TENSILE	Y/T	ELONG	% AREA																																																	
X22424	2013	STRIP/L/B	AR	49,800	0.50	75,500	0.66	38.3																																																			
X22424	2012	STRIP/L/B	AR	51,000	0.50	75,500	0.68	40.0																																																			
X22424	2011	STRIP/L/B	AR	51,500	0.50	76,000	0.68	43.0																																																			

Reviewed / Accepted
I.Q.S. #3
William Brenough



UNITED STATES STEEL

DATE: 07/29/10
TIME: 04:06:33
SERIAL NO: L0032746TUBULAR PRODUCTS
CERTIFIED TEST REPORT
(IN ACCORDANCE WITH ISO 10474/EN10204/DIN5048 "Type 3.1")

MILL ORDER/ITEM NO DR48625 01	SUPPLIER NO. R82202	PO NUMBER D1092000	VENDOR L18256	MAIL TO ADDRESS	VENDOR USS TUBULAR PRODUCTS 2199 EAST 28TH ST. LORAIN, OH 44055
SOLD TO ADDRESS		MAIL TO ADDRESS			
PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145		This MTR applies to your PO# 1221			
SPECIFICATION AND GRADE					
PIPE CARBON SMLS STD PIPE API 5L-44TH ED DTD OCT 2007 AND ISO 3183:2007 MOD FSL-1 GRADE B AND GRADE X42 R N OR Q ASTM A53-07 ASTM A106-08 GRADE B QUAD STENCIL ASME SA53-2007 EDITION ASME SA106-2007 EDITION GRADE B BLK REG MILL COAT PE BEV 30 DEG MEETING ALL THE APPLICABLE REQUIREMENTS OF NACE STANDARD MR-01-75 *2003/COR.1:2005 AND MR0103-2007					
MATERIAL COND: AS ROLLED	OD: 3.500 (88.900)		ID: 0.600 (15.240)		IN (mm)
PRODUCT IDENTIFICATION	TENSILE TEST TYPE ORIENTATION	TEST COND.	GAGE WIDTH IN	YIELD MIN: 42100 MAX: 48300 TENSILE MIN: 70000 MAX: 79000 ELONG % MIN: .50 MAX: .59	HARDNESS SCALE HRB MIN: 27.0 MAX: 99.0 MIN HYDRO PSI 2970 DWELL (SEC) 5
HA0461 HA0470	STRIP/L/B STRIP/L/B	AR AR	0.750 0.750	0.61 0.59	2970 2970 2970
LEGEND: L - LONGITUDINAL U - UPSET					
PRODUCT IDENTIFICATION	T - TRANSVERSE NM - NORMALIZED	QT - QUENCH & TEMPERED SR - STRESS RELIEVED	AR - AS ROLLED TR - THERMO-MECHANICAL ROLLED	B - BODY N - N V - V B - B T - T CO - CO GE - GE	MAX
HA0461 HA0461 HA0461 HA0470 HA0470	HEAT PROD PROD HEAT PROD PROD	22 23 23 23 23	98 012 99 017 99 017 98 010 99 014	003 003 003 004 003	002 003 003 004 003
VERIFIED BY C.E. IS BASED ON THE FOLLOWING EQUATION(S): CE=C+(MN/6)+(CR+MO+V)/5+(NI+CU)/15					

X-79429-72H
Noz 22

Part #4

JUL 11 2011 DECIMAL POSITIONS FOR ELEMENTS ARE INDICATED BY THE LEFT MARGIN, VERTICAL DOTTED LINE OR DECIMAL POINT.

EEG QC DEPT

Reviewed / Accepted
I.Q.S. #3

William Brown

1 OF 2



UNITED STATES STEEL

TUBULAR PRODUCTS

CERTIFIED TEST REPORT

(IN ACCORDANCE WITH ISO 10474EN10246EN50049 "type 3.1")

DATE: 07/29/10

TIME: 04:06:33

SERIAL NO: L0032746

MILL ORDER ITEM NO DR48625 01		SHIPPER'S NO. R82202		P.O. NUMBER D1092000		O.D. 3.500 (88.900)		I.D. (mm) 0.600 (15.240)		WALL 0.017106							
MATERIAL COND: AS ROLLED		MIN COLLAPSE		DIR		TEST LOC.		TEMP		SIZE		CHARTER NOTCH IMPACT TESTING 1 2 3 1 2 3		K SHEAR 1 2 3		AVG	
PRODUCT IDENTIFICATION HA0461 HA0470		FLAT OK OK		GRAIN SIZE		DEG		END OF DATA THIS SHEET **									
LEGEND		L - LONGITUDINAL		T - TRANSVERSE		B - BODY		W - WELD		HAZ - HEAT AFFECTED ZONE							
TEST / INSPECTION		YES		RESULTS / COMMENTS													
FULL LENGTH VISUAL		X															
FULL LENGTH EMI																	
FULL LENGTH MPI																	
FULL LENGTH UT																	
END AREA INSPECTION (PLAIN END)																	
SPECIAL END AREA (SEA) INSP																	
FULL LENGTH DRIFT																	
ADDITIONAL NOTES/COMMENTS																	
MELTED AND MANUFACTURED IN THE USA. NO REPAIRS BY WELDING. NO MERCURY OR MERCURY COMPOUNDS ARE ADDED TO THE STEEL AND ALL MERCURY BEARING EQUIPMENT IS PROTECTED BY A DOUBLE BOUNDARY OF CONTAINMENT.																	
PRODUCT WAS HOT ROLLED AND HOT FINISHED																	
ALL MELTING AND MANUFACTURING TOOK PLACE IN THE USA																	
PIPE ALSO MEET THE REQUIREMENTS OF ASTM A106 GRADE C & ASME SA106 GRADE C																	

THIS IS TO CERTIFY THAT THE PRODUCT DESCRIBED HEREIN WAS MANUFACTURED, INSPECTED AND TESTED IN ACCORDANCE WITH THE SPECIFICATION AND FULFILLS THE REQUIREMENTS IN SUCH RESPECTS.

PREPARED BY THE OFFICE OF: R. HARRIS - MANAGER, Q.A.

DATE 07/29/10

Reviewed / Accepted
I.Q.S. # 3
William Brannough

PAGE 2 OF 2 **

Part #4

Part #549

UNITED STATES STEEL		TUBULAR PRODUCTS		DATE: 03/01/11	
		SUPPLEMENTAL STATEMENT SHEET		TIME: 11:09:40	
MAIL ORDER ITEM NO. DR00362 05	SHIPPER NO.	P.O. NUMBER 019928	HEAT MA2181	SERIAL NUMBER: D0035164	
			VENDOR USS TUBULAR PRODUCTS 2199 EAST 28TH ST. LORAIN, OH 44055		
PIPE ALSO MEETS THE CHEMICAL AND MECHANICAL PROPERTIES FOR ASTM/ASME A/SA333 GRADE 6 EXCEPT MARKINGS					

LONGVIEW, TEXAS, U.S.A.
ISO 9001:2008 Registered Quality System
Material Test Report

REPORT OF CHEMICAL AND PHYSICAL TESTS OF STEEL PLATE

SOLD TO: C.G. Martin Company, Inc.
P O Box 9345
Tulsa OK 74157

DATE: 03/10/11 PAGE: 1 of 1

SALES ORDER NO.: 191857
PURCHASE ORDER NO.: 6229

GRADE: ASTM A/ASME SA516 Grade 70, Normalized
HEAT NO.: D52809 ROLL NO.: 68347

SIZE: 3.75" x 94" x 205"

X 7929-449

C ✓ MN ✓ S ✓ P ✓ SI ✓ CR ✓ NI ✓ MO ✓ AL ✓ V ✓ CU ✓ NB ✓ B ✓ TI ✓
0.1900 1.0600 0.0070 0.0120 0.3000 0.1700 0.1500 0.0600 0.0390 0.0020 0.2500 0.0030 0.0000 0.0020

PHYSICAL ANALYSIS

HEAT	TREAT TEMP	TIME	YIELD POINT	TENSILE STRENGTH	2" GAGE	HARDNESS	CHARPY TEST INFO	FT/LBS	LAT EXP	% SHEAR
Normalized	(B) 1650F	2:45	47800 ✓	76300 ✓	31.0 ✓	59.0 ✓	143 - 149 BHN ✓			

ELECTRIC FURNACE MELTED.

ASTM 516-GRADE 70/60, 2010 EDITION. ASME SA516-GRADE 70/60, 2007 EDITION, 2009b ADDENDA.
MATERIAL IS MERCURY FREE, FULLY KILLED AND DEOXIDIZED.
MATERIAL TEST REPORT PREPARED IN ACCORDANCE WITH EN 10204 3.1. 2004 EDITION.

MATERIAL WAS MELTED AND MANUFACTURED IN THE U.S.A. ✓
IN ACCORDANCE WITH THE RECORDS OF LeTOURNEAU TECHNOLOGIES, INC., I CERTIFY THE ABOVE INFORMATION IS CORRECT.

VERIFIED BY

MAY 24 2011

ECC QC DEPT

K. Peterson

Reviewed / Accepted
I.Q.S. #3

William Brennaugh

TECHNICAL SERVICES DEPARTMENT

Jo Anne Allgood
JO ANNE ALLGOOD

DUPLICATE ORIGINAL TEST REPORT FROM C.G.

MARTH CO., INC.

CUST ID	22	WO#	79689
CUST	ENERGY EXCHANGER COMPANY		
P.O. #	1171		
ITEM #	1		
DESC	(2) 24" OD(t) X 3-3/8" THK(t)		
DATE	05/18/11	BY:	<i>[Signature]</i>

Part # 618



UNITED STATES STEEL

DATE: 11/13/10
TIME: 04:04:23
SERIAL NO: L0033912

TUBULAR PRODUCTS
CERTIFIED TEST REPORT

(IN ACCORDANCE WITH ISO 10474/EN10244/DIN5049 Type 3.1)

MILL ORDER ITEM NO DR45989 06	SHIPPER'S NO R82808	PO NUMBER 6047	VEINLEID L78124	VENDOR USS TUBULAR PRODUCTS 2199 EAST 28TH ST. LORAIN, OH 44055
SOLD TO ADDRESS PROGRESSIVE SUPPLY, INC. 7615 E. 42 ND. PL. TULSA, OK. 74145		MAIL TO ADDRESS This MTR applies to your PO# 1221		

SPECIFICATION AND GRADE
PIPE CARBON SMLS STD PIPE API 5L-44TH ED DTD OCT 2007 AND ISO 3183:2007 MOD PSL-1 GRADE B AND GRADE X42 R N OR Q ASTM A53-*07 ASTM A106-*08 GRADE B QUAD STENCIL SPECIAL OD TOLERANCE ASME SA53-*2010 EDITION ASME SA106-*2010 EDITION GRADE B BLK REG MILL COAT PE BEV 30 DEG MEETING ALL THE APPLICABLE REQUIREMENTS OF NACE STANDARD MR-01-75 *2003/COR.1:2005 AND MS0103-2007

X-7929-15

MATERIAL COND:	AS ROLLED	OD:	24.000 (609.600)	WALL:	0.562 (14.274)	IN (mm)
PRODUCT IDENTIFICATION	TENSILE	YIELD	EXT %	TENSILE	ELONG %	HARDNESS
TEST TYPE ORIENTATION	CAUSE WIDTH	MIN	PSI	MAX	MIN	SCALE: HRB
STRIP/L/B	IN	MAX	PSI	MAX	MIN	MIN HYDRO
MA1922	1.500	52000	.50	73000	30.0	MAX: 99.0
	** END OF DATA THIS SHEET **				47.0	B 75.6
						1780
						1780

LEGEND:	1. LONGITUDINAL	2. TRANSVERSE	3. QUENCH & TEMPERED	4. AS ROLLED	5. BODY	6. W. WELD
PRODUCT IDENTIFICATION	TYPE	MIN	MAX	MIN	MAX	MAX
MA1922	HEAT	20	106	008	005	12
MA1922	PROD	21	102	009	006	11
MA1922	PROD	18	102	009	006	11
	** END OF DATA THIS SHEET **					

VERIFIED BY

JUL 11 2011

EEC QC DEPT

D. Patterson

Reviewed / Accepted

I.Q.S. #3

William Brannough

DECIMAL POSITIONS FOR ELEMENTS ARE INDICATED BY THE LEFT MARGIN, VERTICAL DOTTED LINE OR DECIMAL POINT.

Part #17



UNITED STATES STEEL

TUBULAR PRODUCTS

CERTIFIED TEST REPORT

(IN ACCORDANCE WITH ISO 10474 EN 10244 DIN 50049 "type 3.1")

DATE: 11/13/10
TIME: 04:04:23
SERIAL NO: L0033912

MATERIAL CONDITION: DR49989 06		SUPPLIERS NO: RB2808		P.O. NUMBER: 6047		O.D.: 24.000 (609.600)		WALL: 0.562 (14.274)		ID (mm): 0018741																							
PRODUCT IDENTIFICATION: AS ROLLED		BEND		GRAN SIZE		MIN COLLAPSE		DIR		TEST LOC.		TEMP		SIZE		TEST COND.		1		2		3		AUG		1		2		3		AUG	
MA1922		OK		**		END OF DATA THIS SHEET **		DEG																									
LEGEND		L- LONGITUDINAL		T- TRANSVERSE		B- BODY		W- WELD		HAZ- HEAT AFFECTED ZONE																							
TEST / INSPECTION		YES																															
FULL LENGTH VISUAL		X																															
FULL LENGTH EMI		X																															
FULL LENGTH MPI																																	
FULL LENGTH UT																																	
END AREA INSPECTION (PLAIN END)																																	
SPECIAL END AREA (SEA) INSP																																	
FULL LENGTH DRIFT																																	
ADDITIONAL NOTES/COMMENTS																																	
MELTED AND MANUFACTURED IN THE USA. NO REPAIRS BY WELDING. NO MERCURY OR MERCURY COMPOUNDS ARE ADDED TO THE STEEL AND ALL MERCURY BEARING EQUIPMENT IS PROTECTED BY A DOUBLE BOUNDARY OF CONTAINMENT.																																	
PRODUCT WAS HOT ROLLED AND HOT FINISHED																																	
PIPE ALSO MEET THE REQUIREMENTS OF ASTM A106 GRADE C & ASME SA106 GRADE C																																	

THIS IS TO CERTIFY THAT THE PRODUCT DESCRIBED HEREIN WAS MANUFACTURED, SAMPLED, TESTED AND/OR INSPECTED IN ACCORDANCE WITH THE SPECIFICATION AND FULFILLS THE REQUIREMENTS IN SUCH RESPECTS.

PREPARED BY THE OFFICE OF: R. HARRIS - MANAGER, Q.A.

DATE: 11/13/10

Reviewed / Accepted
108. #3

PAGE 2 OF 2

Part # 7

NUCOR
PLATE MILL

P.O. Box 279
Wilton, NC 27986
(252) 356-3700

Mill Test Report

Page 3



Issuing Date : 05/01/2011 BIL No. : 292311
Vehicle No: TTPX 82223
Specification : 0.3750" x 96.000" x 480.000"
ASTM A516 70-10/ASME SA516 70 PVQ 2010 Normalized Test
Coupons at 1650 F

Load No. : 294273 Our Order No. : 90537/10
Sold To : METALS USA - SOUTH CENTRAL
2800 N 43RD STREET EAST
MUSKOGEE, OK 74403

Cust. Order No. : MUS-233263
Ship To : METALS USA - PLATES AND SHAPES
TRACK 747
MUSKOGEE, OK 74401

Marking :

X-7929.71B

Heat No	C	Mn	P	S	Si	Cu	Ni	Cr	Mo	Al	V	Nb	Ti	N	Ca	B	Sn	CEQ	PCM
1502825	0.19	1.02	0.013	0.002	0.19	0.21	0.07	0.10	0.01	0.025	0.005	0.002	0.002	0.002	0.0016	0.0002	0.009	0.40	0.26

Tensile Test					Charpy Impacts									
Plate Serial No	Pieces	Tons	Dir.	(psi) Yield / (psi) Tensile	Elongation % in 2"	Dir.	1 shear	2 shear	3 shear	(ft) shear Ave.	Size	Temp	Min Ave.	
1502825-08	5	12.25	T	53,500	79,600									
1502825-09	5	12.25	T	51,000	76,100									
1502825-10	5	12.25	T	51,800	78,300									
1502825-08	5	12.25	T	55,800	76,400									
1502825-09	5	12.25	T	60,600	72,800									
1502825-10	5	12.25	T	49,800	75,800									

VERIFIED BY

JUN 20 2011

EEC QC DEPT

G. Patterson

NACE MR0175 ANNEX 2.1.2 COMPLIANT

Test coupons only, normalized 60 minutes per inch of thickness at 1650 F ± 25 F. Hold 30 minutes minimum.

Manufactured to fully killed fine grain practice by Electric Arc Furnace. Welding or weld repair was not performed on this material. We hereby certify that the contents of this report are accurate and correct. All test results and operations performed by the material manufacturer are in compliance with the applicable specifications, including customer specifications.

Mercury has not been used in the direct manufacturing of this material. Produced as continuous cast discrete plate as-rolled, unless otherwise noted in Specification.

Yield by 0.5% UTS method unless otherwise specified. Ceq = C + (Mn/16) + ((Cr+Mo+V)/5) + ((Cu+Ni)/15)

Pcm = C + (Si/30) + (Mn/20) + (Cr/20) + (Ni/60) + (Cr/20) + (Mo/15) + (V/10) + 58

Melted and manufactured in the USA. (ISO 9001:2008 certified (#008063) by SRI Quality System Registrar (#0985-09), PED 9729/IEC 712 Annex 1, Para. 4.3 Compliant.

DIN 50049 3.1.8/EN 10204 3.1B(2004), DIN EN 10204 3.1(2005) compliant. For ABS grades only. Quality Assurance certificate 08-MMPQA-548

Reviewed / Accepted

I.Q.S. #3

William Brennaugh

T. A. Deprellis

T. A. Deprellis, Metallurgist

05/12/2011 12:52:48 PM

Part #12

NU 331

Load Number	Tally	Mill Order Number	P.O. Number	Part Number	Certificate Number	Date
387479	00000000412159	N-106041-003	MUS-233438		L328209-1	06/06/2011 17:48
Grade						
Order Description: A516 70, 0.3750 IN x 96.000 IN x 480.000 IN						
Quality Plan Description: A51670 NORM: ASTM A516-70-06/ASME SA516-70-06 09 add NORMTEST						
Customer: Sold TO: METALS USA MUSKOGEE OK Ship TO: Metals USA Muskogee OK						

Shipped Item	Heat/Slab Number	Certified By	C	Mn	P	S	Si	Cu	Ni	Cr	Mo	Cb	V	Al	Ti	N2	B	Ca	Sn	CEV
1E3293E	A1S1804-03 ***	A1S1804	0.23	1.08	0.009	0.006	0.19	0.18	0.08	0.09	0.023	0.001	0.002	0.038	0.001	0.007	0.0001	0.0031	0.011	0.45
1E3292B	A1S1804-02 ***	A1S1804	0.23	1.08	0.009	0.006	0.19	0.18	0.08	0.09	0.023	0.001	0.002	0.038	0.001	0.007	0.0001	0.0031	0.011	0.45

Shipped Item	Heat Number	Yield ksi	Tensile ksi	Y/T %	ELONGATION %		Bend OK?	Hard HB	Charpy Impacts (ft-lbf)				Shear %			Test Temp				
					1"	8"			Size mm	1	2	3	1	2	3					
1E3293E	A1S1804 ***	50.6	72.7	69.6	33.0															
1E3293E	A1S1804 ***	49.2	77.5	63.5	32.8															
1E3293E	A1S1804 ***	49.5	72.9	67.9	34.0															
1E3293E	A1S1804 ***	53.8	75.4	71.4	36.0															
1E3293E	A1S1804 ***	54.0	75.7	71.3	36.2															
1E3293E	A1S1804 ***	52.8	75.5	69.9	34.8															
1E3292B	A1S1804 ***	50.6	72.6	69.7	33.1															
1E3292B	A1S1804 ***	55.1	77.7	70.9	33.8															
1E3292B	A1S1804 ***	49.7	72.0	69.0	33.2															
1E3292B	A1S1804 ***	53.3	75.7	70.4	34.6															
1E3292B	A1S1804 ***	51.9	75.2	69.0	37.3															
1E3292B	A1S1804 ***	51.0	74.8	68.2	34.7															

VERIFIED BY

SEP 20 2011

REC QC DEPT

S. Patterson

Items: 2 PCS: 8 Weight: 39205 LBS

Mercury has not come in contact with this product during the manufacturing process nor has any mercury been used by the manufacturing process. Certified in accordance with EN 10204 3.1. No weld repair has been performed on this material. 1650 F FOR 60 MIN PER INCH OF THICKNESS NACE MRO175 ANNEX 2.1.2 COMPLIANT Manufactured to a fully killed fine grain practice. ** Produced from Coil **

ISO 9001:2008 Registered, PED Certified

We hereby certify that the product described above passed all of the tests required by the specifications.

April Davis
April Davis - QA Engineer

Reviewed / Accepted
I.Q.S. #3

William Brunaugh

*** Indicates Heats melted and Manufactured in the U.S.A.

STOCK LIFTING LUGS

Load Number	Tally	Mill Order Number	P.O. Number	Part Number	Certificate Number	Date
389976	0000000417856	N-106993-003	32981		L333877-1	07/23/2011 04:48

Grade: _____
Order Description: _____
AS16 70, 0.5000 IN x 96.000 IN x 480.000 IN
Quality Plan Description: _____
A/SAS16-70: ASTM A516-70-10/ASME SAS16-70 2009 ADDENDA

Sold TO:
STEEL SERVICE COMPANY CLARENORE OK
Ship TO:
STEEL SERVICE COMPANY CATOOSA OK

1/2 x 3 1/2" x 8 3/4" LG

Shipped Item	Heat/Slab Number	Certified By	C	Mn	P	S	Si	Cu	Ni	Cr	Mo	Co	V	Al	Ti	N2	B	Ca	Sn	CeV
1G1767D	B117537-03 ***	B117537	0.23	1.11	0.010	0.007	0.20	0.20	0.07	0.07	0.011	0.000	0.002	0.037	0.001	0.007	0.0001	0.0031	0.007	0.45
1G1767E	B117537-03 ***	B117537	0.23	1.11	0.010	0.007	0.20	0.20	0.07	0.07	0.011	0.000	0.002	0.037	0.001	0.007	0.0001	0.0031	0.007	0.45

Shipped Item	Certified By	Heat Number	Yield ksi	Tensile ksi	Y/T %	Bend OK?	Hard HB	Charpy Impacts (ft-lbf)			Shear %			Test Temp
								Size mm	1	2	3	Avg	1	
1G1767D	SLG1767BTT	B117537 ***	54.5	72.7	75.0	33.7								
1G1767D	SLG1767FTT	B117537 ***	55.5	76.3	72.7	34.2								
1G1767D	SLG1767MTT	B117537 ***	52.5	72.7	72.2	33.4								
1G1767E	SLG1767BTT	B117537 ***	54.5	72.7	75.0	33.7								
1G1767E	SLG1767FTT	B117537 ***	55.5	76.3	72.7	34.2								
1G1767E	SLG1767MTT	B117537 ***	52.5	72.7	72.2	33.4								

X-7929-62 Cut to 6 1/2" lg.

Name: PIONEER
PO# 767386
Count 1
Size 1/2 X 96 X 240 AS16-70
Heat# B117537-03

PIONEER STEEL INC.
DATE *22.12* QTY. *100*
PO# *1874*
PART# *stock*
BY _____

Items: 2 PCS: 6 Weight: 39205 LBS

Mercury has not come in contact with this product during the manufacturing process not has any mercury been used by the manufacturing process. Certified in accordance with EN 10204 3.1. No weld repair has been performed on this material. Manufactured to a fully killed fine grain practice. ** Produced from Coil ** ISO 9001:2008 Registered, PED Certified

We hereby certify that the product described above passed all of the tests required by the specifications.

VERIFIED BY

April Davis
April Davis - QA Engineer

23 2012

REC QC DEPT

J. J. Johnson

Reviewed / Accepted
I.Q.S. # 3

William Breenaugh

Part #14

SSAB

Test Certificate

Form TCI: Revision 1: Date 31 Oct 2000

13609 Industrial Road, Houston, TX 77015

Customer: METALS USA PLATES & SHAPES SOU P.O. BOX 328 141 EAST ILLINOIS EMHD OK 73702		Customer P.O. No.: MUS-231893 Product Description: ASTM A516-70(66)/ASME SA516-70(66A)		Mill Order No.: 41-277716-01 Shipping Manifest: HT044418						
Ship Date: 07 Sep 18 Cert Date: 07 Sep 18 Cert No: 031118772 (Page 1 of 1)		Stock NB BRACKETS Size: 0.188 X 96.00 X 480.0 (IN)								
Tested Pieces		Charpy Impact Tests								
Heat Id	Piece Id	Dimensions	YS (PSI)	UTS (PSI)	Average Hardness	Abs. Energy (FTLB)	% Shear	Tst Temp	Tst Dir	BDWTT
W0B789 W0H543	0051	0.187 X 96.00 (T.L.C.)	58000	75000	27					
	0024	0.187 X 96.00 (T.L.C.)	59000	75000	26					
			61000	74000	25					
			55000	70000	28					
			56000	71000	27					
			54000	70000	31					
Chemical Analysis										
Heat Id	C	Mn	P	S	Si	Sh	Ti	Al	Se	ORGN
W0B789 W0H543	.19	1.11	.014	.003	.20	.019	.017	.28	.14	USA
	.18	1.10	.009	.003	.21	.030	.029	.26	.14	USA
MERCURY IS NOT A METALLURGICAL COMPONENT OF THE STEEL AND NO MERCURY WAS INTENTIONALLY ADDED DURING THE MANUFACTURE OF THIS PRODUCT										
KILLED STEEL, PRODUCED TO A FINE GRAIN PRACTICE										
CEV (LIW) = C + Mn/6 + (Cr+Mo+V)/5 + (Ni+Cu)/15										
MATERIAL MARKED WITH AN ASTERISK IS PRODUCED FROM COIL										
MTR DIN EN10204:2004 TYPE 3.1 COMPLIANT										
100% MELTED AND MANUFACTURED IN THE USA.										
PRODUCTS SHIPPED: 0051 PCES: 6, WGT: 14741 * W0H543 0024 PCES: 12, WGT: 29482										
ENERGY EXCHANGER HT#W0H543 SL#0024										
PO#1462 ITEM#165PCS										
SO#203277										
VERIFIED BY										
SEP 19 2011										
EEC QC DEPT										
Jason Thomas										
RENDER REVALUATION										
Cust Part #:										

WE HEREBY CERTIFY THAT THIS MATERIAL WAS TESTED IN ACCORDANCE WITH, AND MEETS THE REQUIREMENTS OF, THE APPROPRIATE SPECIFICATION

Reviewed / Accepted
I.Q.S. # 3

William

Part # 15

We hereby certify that the test results presented here

We hereby certify that the test results presented here
are accurate and conform to the reported grade specification

Daniel J. Schacht
Daniel J. Schacht
Quality Assurance Manager

CERTIFIED MILL TEST REPORT
For additional copies call
830-372-8771

CMC STEEL TEXAS
1 STEEL MILL DRIVE
SEGUIN TX 78155-7510



STOCK X-7929-65

HEAT NO.: 3018892 SECTION: FLAT 1/4x1-1/2 20"0" A36/52950 GRADE: ASTM A36-08/A529-05 Gr 50 ROLL DATE: 09/03/2010 MELT DATE: 08/24/2010		Steel Service Co - OK S O L D T O 24412 Amah Pkwy Claremore OK US 74019-4304 9183416322 9183416390		Steel Service Co - OK S H I P T O 24412 Amah Pkwy Claremore OK US 74019-4304 9183416322 9183416390		Delivery#: 80441928 BOL#: 70155381 CUST PO#: 33419 CUST P/N: DLVRY LBS / HEAT: 9216,000 LB DLVRY PCS / HEAT: 362 EA	
Characteristic		Value		Characteristic		Value	
C		0.19%					
Mn		0.67%					
P		0.013%					
S		0.040%					
Si		0.17%					
Cu		0.30%					
Cr		0.12%					
Ni		0.09%					
Mo		0.023%					
V		0.009%					
Cb		0.001%					
Sn		0.013%					
Al		0.002%					
Carbon Eq A529		0.39%					
Yield Strength test 1		52.5ksi					
Tensile Strength test 1		72.5ksi					
Elongation test 1		26%					
Elongation Gage Lgth test 1		8IN					

THIS MATERIAL IS FULLY KILLED, 100% MELTED AND MANUFACTURED IN THE USA, WITH NO WELD REPAIR OR MERCURY CONTAMINATION IN THE PROCESS.

REMARKS :

MEETS THE REQUIREMENTS OF ASME SA36-2008A

Reviewed / Accepted 01/28/2011 14:51:32
I.Q.S. # 3 Page 1 OF 1
William Brennaugh

CHECKED FEB 24 2011

R. Jetterson

Benteler Stahlrohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13666

BENTELER

Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1

CERTIFICAT DE RECEPTION EN 10204-3.1

EN 10204:2005-01

Benteler Stahlrohr GmbH - Postfach 1340 - 33043 Paderborn - Deutschland
Webco Industries, Inc.
9101 West 21st Street
SAND SPRINGS, OK 74063
USA

Dokument-Nr.: 41-488044/001/P

Document No.:
No. du document:

Kunden-Bestell-Nr.:
Purchase Order No.:
No. de commande du client:

Benteler Auftrags-Nr.:
Benteler Order No.:
No. de commande Benteler:

Versandanzeigen-Nr.:
Dispatch Note No.:
No. d'avis d'expédition:

Produkt: NAHTLOSE STAHLROHRE
Product: SEAMLESS STEEL TUBES
Produit: TUBES D'ACIER SANS SOUDURE

Hersteller:
Manufacturer:
Producteur:

Herstellerzeichen:
Manufacturer's brand:
Marque du producteur:

Stempel des Abnahmebeauftragten:
Stamp of the inspection representative:
Timbre du contrôleur:

Stahlschmelzungsverfahren:
STEELMAKING PROCESS:
Procédé d'élaboration de l'acier:

ASTM-A 179-1990a, ASME SA-179, ASME Section II Part A Edition 2007

Lieferbedingungen:
Terms of delivery:
Conditions de livraison:

Maße - Toleranzen:
Dimensions-tolerances:
Dimensions-tolérances:

Stahlsorte:
Steel grade:
Nuance d'acier:

Normalized

Lieferzustand:
Delivery condition:
État de livraison:

Produktkennzeichnung:
Product marking:
Marquage du produit:

CHECKED FEB 22 2010

R. Johnson

Pos.	Stück	Maße	Länge	Gewicht	Schmelzen-Nr.	Prüfdruck	Rohr-Nr.-Gruppe	Viellängen
Item	Number	Dimensions	Length	Weight	Heat No.	Test pressure	Tube number group	Multiple lengths
Poste	Nombre	Dimensions	Longueur	Poids	No. de coulée	Pression d'épreuve	Série de no. des tubes	Longueurs multiples
			feet	lbs				
0001	218	0.750" O.D. * 0.083" MW	29 FT 1"	5851	181070	← (390 psi)		
0002	385	0.750" O.D. * 0.083" MW	30 FT 1"	7291	181070			
0003	344	0.750" O.D. * 0.083" MW	31 FT 3"	4815	181070			

Reviewed / Accepted
I.Q.S. # 3

William Brannagh

Part # 17

X. 7929
(390 psi)

Tel.: +49.5254.81-0 Fax: +49.5254.13666

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1

Dokument-Nr.: 41-488044/001/P

Document No.:
No. du document:

Prüf-Nr.:
Inspection No.:
No. du certificat:

Blatt: 2 / 6

Pos.	Stück	Maße	feet			Länge	Gewicht	Schmelzen-Nr.	Prüfdruck	Rohr-Nr. -Gruppe	Vielfachlängen
Item	Number	Dimensions				Length	Weight	Heat No.	Test pressure	Tube number group	Multiple lengths
Poste	Nombre	Dimensions				Longueur	Poids	No. de coulée	Pression d'épreuve	Série de no. des tubes	Longueurs multiples
0004	153	0.750" O.D. * 0.083"	MW	37	FT 1 "	5673,75	3583	181069			
0005	187	0.750" O.D. * 0.083"	MW	38	FT 1 "	7121,10	4497	181069			
0006	262	0.750" O.D. * 0.083"	MW	40	FT 2 "	10523,03	6706	181073			
0007	427	0.750" O.D. * 0.083"	MW	41	FT 1 "	17542,36	11158	181073			
0008	472	0.750" O.D. * 0.083"	MW	42	FT 1 "	19863,39	12606	181073			
0009	491	0.750" O.D. * 0.083"	MW	43	FT 1 "	21152,70	13395	548853			
0010	195	0.750" O.D. * 0.083"	MW	44	FT 1 "	8595,87	5423	548853			
0011	139	0.750" O.D. * 0.083"	MW	45	FT 1 "	6266,40	4001	548853			
0012	140	0.750" O.D. * 0.083"	MW	46	FT 1 "	6451,58	4123	548853			
0013	231	0.750" O.D. * 0.083"	MW	51	FT 6 "	11896,39	7571	549318			

Schmelzenanalyse [%] / Heat analysis [%] / Analyse sur coulée [%]

Pos.	Smelzen-Nr.	C	SI	MIN	P	S
Item	Heat No.					
Poste	No. de coulée					
0001	181070	0,100	0,160	0,48	0,008	0,003
0002	181070	0,100	0,160	0,48	0,008	0,003
0003	181070	0,100	0,160	0,48	0,008	0,003
0004	181069	0,100	0,160	0,48	0,008	0,003
0005	181069	0,100	0,160	0,48	0,008	0,003
0006	181073	0,100	0,170	0,47	0,010	0,003
0007	181073	0,100	0,170	0,47	0,010	0,003
0008	181073	0,100	0,170	0,47	0,010	0,003
0009	548853	0,100	0,160	0,47	0,007	0,002
0010	548853	0,100	0,160	0,47	0,007	0,002
0011	548853	0,100	0,160	0,47	0,007	0,002
0012	548853	0,100	0,160	0,47	0,007	0,002

~~Reviewed/Accepted~~

WED / ACC
LOS #3

William Brewster

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13666

BENTELER
Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1

Schmelzenanalyse [%] / Heat analysis [%] / Analyse sur coulée [%]

41-488044/001/P

Prüf-Nr.:
Inspection No.:
No. du certificat

Dokument-Nr.:
Document No.:
No. du document

Blatt: 3 / 6
Page:
Page:

Pos. Schmelzen-Nr.

Item Heat No. C SI MN P S

Poste No. de coulée

0013 549318 0,100 0,150 0,46 0,013 0,003

Prüfergebnisse / Test results / Résultats des essais

Die Röhre sind auf Dichtheit geprüft durch:

The tubes have been submitted to a leak tightness test by:

Les tubes ont passé un contrôle d'étanchéité par:

Hydrostatic test: eddy current testing instead of hydrostatic test

PASSED

Die Röhre wurden zerstörungsfrei geprüft:

The tubes are non destructive tested:

Les tubes ont passé un essai non destructif:

ET-test: acc. to ASME SA-450, ASTM-A 450; acc. to ASTM-E 309; imperfections

PASSED

Augensichtkontrolle:

PASSED

Aufweitversuch:

PASSED

Maßkontrolle:

PASSED

Visual inspection:

Drift expanding/flare test:

Dimensions examination:

Examen visuel:

Essai d'évasement:

Vérification des dimensions:

Ringfaltversuch:

PASSED

Flattening test:

Essai d'aplatissement:

Ergebnisse der mechanischen Prüfung / Results of mechanical testing / Résultats des essais mécaniques

Die Probenahme erfolgte an Vielfachlängen.

The sampling was carried out on multiple lengths.

L'échantillonnage était réalisé aux longueurs multiples.

Härteprüfung / Hardness test / Essai de dureté

Pos. Proben-Nr. Schmelzen-Nr. Härte

Item Specimen No. Heat No. Hardness

Poste No. de l'éprouv. No. de coulée Dureté

Anforderungen

Requirements

Exigences

HRC HB HV HRB HBW

MAX 72

0001 000001 181070

0001 000002 181070

064

065

Reviewed / Accepted

I.Q.S. # 3

William Braugh

Part #17

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13666

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1

CERTIFICAT DE RECEPTION EN 10204-3.1

Härteprüfung / Hardness test / Essai de dureté

Dokument-Nr.: 41-488044/001/P
Document No.:
No. du document:

Prüf-Nr.:
Inspection No.:
No. du certificat:

Blatt: 4 / 6

Page:

Page:

Pos.	Proben-Nr.	Schmelzen-Nr.	Härte
Item	Specimen No.	Heat No.	Hardness
Poste	No. de l'éprouv.	No. de coulée	Dureté

Anforderungen	HRC	HB	HV	HRB	HBW
Requirements					
Exigences					

MAX 72

0001	000003	181070			065
0001	000004	181070			063
0002	000001	181070			064
0002	000002	181070			065
0002	000003	181070			065
0002	000004	181070			063
0003	000001	181070			064
0003	000002	181070			065
0003	000003	181070			065
0003	000004	181070			063
0004	000001	181069			064
0004	000002	181069			064
0005	000001	181069			064
0005	000002	181069			065
0006	000001	181073			063
0006	000002	181073			064
0006	000003	181073			063
0006	000004	181073			063
0007	000001	181073			063
0007	000002	181073			063
0007	000003	181073			064
0007	000004	181073			063
0008	000001	181073			063

Reviewed / Accepted

I.Q.S. #3

William Brennan WB

Part #17

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Padernborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13686

BENTELER
Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1

INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1

Härteprüfung / Hardness test / Essai de dureté

Dokument-Nr.: 41-488044/001/P
Document No.:
No. du document:

Prüf-Nr.:
Inspection No.:
No. du certificat:

Blatt: 5 / 6
Page:
Page:

Pos. Item Poste	Proben-Nr. Specimen No. No. de l'éprouv.	Schmelzen-Nr. Heat No. No. de coulées	Härte Hardness Dureté	HRC	HB	HV	HRB	HBW
Anforderungen Requirements Exigences								
0008	000002	181073					MAX 72	
0008	000003	181073					064	
0008	000004	181073					065	
0008	000005	181073					063	
0008	000006	181073					063	
0009	000001	548853					064	
0009	000002	548853					065	
0009	000003	548853					065	
0009	000004	548853					064	
0010	000001	548853					064	
0010	000002	548853					065	
0011	000001	548853					064	
0011	000002	548853					065	
0012	000001	548853					064	
0012	000002	548853					065	
0013	000001	549318					065	
0013	000002	549318					064	

Part #17

Vermerk / Remark / Remarque

Certificate remarks: Steel and tubes manufactured by Benteler Stahl / Rohr ✓

Reviewed / Accepted
I.Q.S. #3
William Brennaugh

Benteler Stahl/Rohr GmbH
Postfach 13 40
33043 Paderborn
Deutschland
Tel.: +49.5254.81-0 Fax: +49.5254.13666

BENTELER 

Stahl/Rohr

ABNAHMEPRÜFZEUGNIS EN 10204-3.1
INSPECTION CERTIFICATE EN 10204-3.1
CERTIFICAT DE RECEPTION EN 10204-3.1
Beiblatt Produktkennzeichnung / Attachment product marking / Supplément marquage du produit

Pos.	Item	Poste	Produktkennzeichnung	Item	Poste	Produktkennzeichnung		
0001, 0002, 0003, 0004, 0005, 0006, 0007, 0008, 0011, 0012	FS:	BENTELER A179-SA179 HEAT-NO. DIMENSIONS TUBES-LENGTH FEET 2008 WA MADE IN GERMANY	0009, 0010	FS:	BENTELER A179-SA179 HEAT-NO. DIMENSIONS TUBES-LENGTH FEET 2108 WA MADE IN GERMANY	0013	FS:	BENTELER A179-SA179 HEAT-NO. DIMENSIONS TUBES-LENGTH FEET 2408 WA MADE IN GERMANY

Prüf-Nr.: 41-488044/001/P
Inspection No.:
No. du certificat:

Blatt: 6 / 6
Page:
Page:

AEZ = Äolienbeschriftung, Etching ink marking, Gravure à l'encre FX = Farbschablonierung, colour marking, marquage par couleur FS = Farbschablonierung, salin stencilling, marquage par peinture FSD = Farbschablonierung, Colour jet printer, Imprimante à jet d'encre de c
LS = Längsschablonierung, longitudinal stencilling, marquage longitudinal par peinture PKE = Etikettenkennzeichnung, tag marking, marquage sur étiquette PS = Prägestempel, die stamp, marquage par poinçonnage TS = Tintenstrahlkennzeichnung, ink jet spray marking.

Verkäufer(in) / Salesman/ woman in charge : Mr Buerger, Tel.: 05254/81-4266, Fax: 4299

Reviewed / Accepted
I.Q.S. # 3
William Brown

Part #17

Paderborn Schloß Neuhaus, 24.06.2008, TEL.: 05254-81-1767 FAX: 05254.81-1752

Abnahmebeauftragter
Inspection representative

Contrôleur
ALWIN PANKRATZ / TEM
Dieses Dokument wurde mittels EDV erstellt und ist ohne Unterschrift rechtsgültig.
We certify that the supplied products comply with the order specification. This document was prepared by means of electronic data processing and is valid without signature.
Nous attestons que les produits livrés sont conformes aux stipulations de la commande. Ce document a été établi par traitement électronique de l'information et est valide sans signature.

DATE: 01/12/12

W. M. HEITGRAS COMPANY
P.O. BOX 2044
1316 N. OSAGE DRIVE
TULSA, OKLAHOMA 74101
(918) 583-3131

Part #18

CERTIFIED MATERIAL TEST REPORT

ENERGY EXCHANGERS
1844 N. GARNETT RD.
TULSA OK 74116

LOT # 10037360
HEAT # 10037360
STOCK # B7S0750525
3/4-10 X 5-1/4 ASTM A-193 B-7
CUS POS# 1744
ORDER # 28168

SPECIFICATIONS:

ASTM A193 B7 ALL THREAD ROD

X-7929-41+42

VENDOR: VULCAN THREADED PRODUCTS, INC.

QTY SOLD

48

- CHEMICAL COMPOSITION -

C %	MN %	P %	S %	SI %	NI %	CR %
00.4000	00.8400	00.0090	00.0250	00.2500	00.0600	00.9200
MO %	CU %	CO %	V %	AL %	FE %	TA %
00.2100	00.1200		00.0030	00.0270		
TI %	ZN %	SE %	CB %	W %	N %	SN %
						00.0080

- CHARPY IMPACT -

FIRST: 0 SECOND: 0 THIRD: 0 AVERAGE: 0

- MECHANICAL PROPERTIES -

TENSILE STRENGTH: 132,200 Psi ✓
YIELD : 121,500 ✓
ELON : 22.00 ✓
RA : 65.00 ✓

HARDNESS : 28 HRC ✓
PROOF STRESS: 0 Psi

COMMENTS:

MELTED & MFG IN THE USA ✓
TEMPERING TEMP 1415 F
META S1/R1/C1

VERIFIED BY

1/12/12

W. M. HEITGRAS COMPANY

EEC QC DEPT

Reviewed / Accepted
I.Q.S. #3

William Branaugh

L. Jatterson

QUALITY CONTROL MANAGER

INSPECTION CERTIFICATE



UNYTE, INC.
One Unyte Drive
Peru, Illinois 61354
815-224-2221 — FAX# 815-224-3434

Customer	Specification	Size	Lot No.	Date
	ASTM A-194 GRADE 2H HEAVY HEX NUT	3/4-10 UNC.	SB151	May 09, '11

Mechanical properties tested in accordance to ASTM F606/F606M, ASTM A270, ASTM E10

Chemical Composition										(%)
STOCK										
Mill Mktg	Material	Heat No.	Spec.	C	Si	Mn	P	S	Cu	Ni
GERDAU AMER CARBON				MAX.	MAX.	MAX.	MAX.	MAX.		
				MIN	MAX	MAX	MAX	MAX		
				0.40	0.40	0.30	0.040	0.050		
ISTEEL (NO STEEL)				0.40	0.40	0.30	0.040	0.050		

Mechanical Property Inspection										Heat Treatment
After Heat Treatment										
Item	Proof Load	Cone stripping	Hardness	Hardness	Hardness	Hardness	Hardness	Absorbed Energy		
Spec.	58,450 lbf	1000 lbf	24-38 HRC	105.4	105.4	105.4	105.4	105.4	105.4	T: MIN 850 F
										Q: FORGING Q (W.Q.)
										T: 1202 F/45M (W.C.)
										Q: Quenching T: Tempering ST: Solution Treatment

OFFICIAL SEAL
JEAN MARCHEFIO
NOTARY PUBLIC - STATE OF ILLINOIS
MY COMMISSION EXPIRES 10/18/13
65-10-11

VERIFIED BY
JUN - 6 - 2011

EEC QC DEPT

L. J. Jettison

Part # 19

Production Quantity
46,735 pcs.

Reviewed / Accepted
I.Q.S. # 3
William Brennaugh

Chief of Quality Assurance Section

W. Jettison

Material used for the nut was melted and manufactured in the USA. The nut was manufactured in the USA to the above specification.

We hereby certify that the material described has been manufactured and inspected satisfactorily with the requirement of the above specification.

A

ArcelorMittal

MATERIAL CERTIFICATION REPORT

ArcelorMittal Laplace
138 HWY 3217
Laplace LOUISIANA 70068
Telephone (985) 652-4900

STEEL SERVICE COMPANY
CLAREMORE, OK
6725 AMAH PARKWAY
74019 Claremore

Tested in Accordance
With: ASTM A6

Invoice NO.
Product Equal Angl
Heat NO. L81828
Length 20'00"

Date 01/30/2012
Cust 40006655
Grade A3652950
Size 3"x3"x1/4"x4.900

PO: 35234

Ref. 80387026
Pieces 53

X-7929-67

CHEMICAL ANALYSIS		MECHANICAL PROPERTIES		TEST 1		TEST 2		TEST 3	
				IMPERIAL	METRIC	IMPERIAL	METRIC	IMPERIAL	METRIC
C	0.15	YIELD STRENGTH	57,500 PSI	396 MPa	55,500 PSI	383 MPa			
Mn	0.85	TENSILE STRENGTH	78,400 PSI	541 MPa	78,200 PSI	539 MPa			
P	0.019	ELONGATION	25 %	25 %	23 %	23 %			
S	0.031	GAUGE LENGTH	8 IN	203 mm	8 IN	203 mm			
Si	0.14	BEND TEST DIAMETER							
Cu	0.32	BEND TEST RESULTS							
Ni	0.13	SPECIMEN AREA							
Cr	0.17	REDUCTION OF AREA							
Mo	0.036	IMPACT STRENGTH							
Cb	0.015								
V	0.000								
B									
Al									
Sn	0.013								
N									
Ti									
C1	5.8								
CE	0.36								

A36-08 A52950-05, CSA50W, 44W, A70936-09a, AASHTO M270 Grade 36
ASME SA36-2008A, A57250-07, A70950-10.

VERIFIED BY

MAR 15 2012

EEC QC DEPT

R. Peterson

I hereby certify that the material test results presented here are from the reported heat and are correct. All tests were performed in accordance to the specification reported above. All steel is electric furnace melted (billets), manufactured, processed, and tested in the U.S.A with satisfactory results, and is free of Mercury contamination in the process. No weld repair was performed on this heat.

Notarized upon request:

Sworn to and subscribed before me in and for ST. John
Parish on this 30th day of January, 2012

Signed

Mark Edwards

MARK EDWARDS, QUALITY ASSURANCE SUPERVISOR

Direct any questions or necessary clarifications concerning
this report to the Sales Department 1-800-538-7682, USA

Reviewed/ Accepted
I.Q.S. # 3

William Brerough

Part # 20

SECTION 5

ENERGY EXCHANGER
JOB# 55364
P.O.# PX 9621

John Winkle
8/16/12

BODYCOTE THERMAL PROCESSING FCE# 2001

X-7929 Sep Channel



CALIBRATION DATE 8-10-12
CALIBRATION DUE DATE 11-10-12

Reviewed / Accepted
I.Q.S. # 3
William Brennaugh

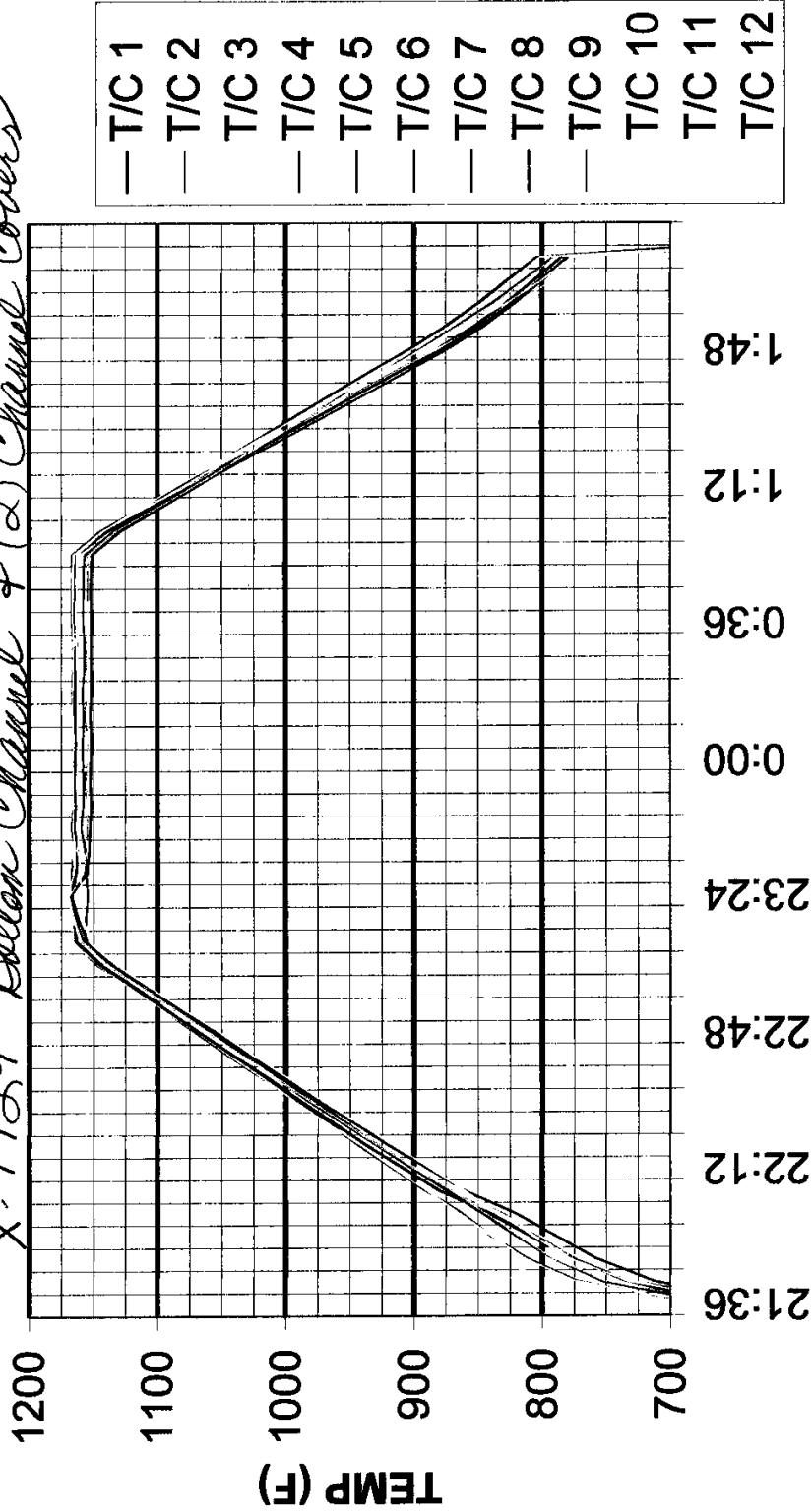
SIGNED BY JOHN WINKLE
John Winkle

ENERGY EXCHANGER
JOB# 55344
P.O.# PX 9620

blc BW 8/14/12

BODYCOTE THERMAL PROCESSING FCE# 2001

X-7929 Bottom Channel + (2) Channel Covers

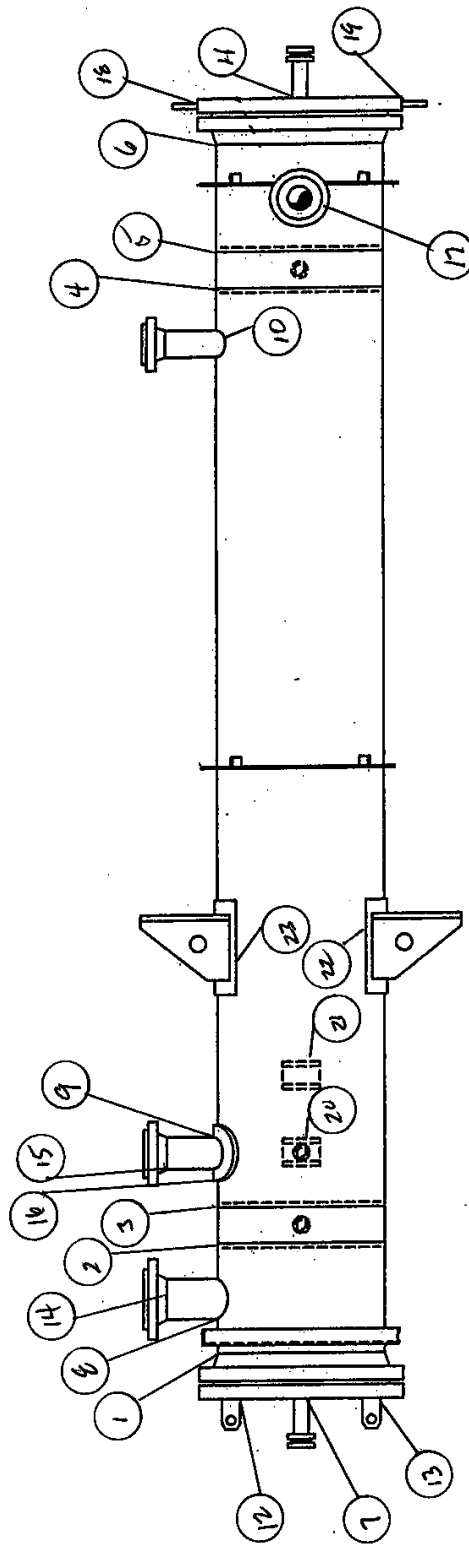


TIME: 6 MINUTE INCREMENTS

CALIBRATION DATE 8-10-12
CALIBRATION DUE DATE 11-10-12

Reviewed / Accepted
I.Q.S. # 3
William Brannough

SIGNED BY JOHN WINKLE
John Winkle



Reviewed / Accepted
I.Q.S. # 3
William Brumbaugh

X-7929 NDE/WELD MAP

ENERGY EXCHANGER COMPANY

P.O.#: 17496 - 0001

ENERGY JOB # X-7929

NDE/WELD INDEX

[illegible]

William Bronaugh

William Bronaugh

JOB # X-7929

17496-0001

CHANNEL B/S SPOT NW ST

SA-333-4

24 1/2

Q.C. WELDMENT

ASME SEC. VIII, DIV. 1, A-

RADIOGRAPHIC DATA

1. RT DATE 7-31-12

2. MATERIAL EXAMINED:

☒ WELDMENT
☐ CASTING
☐ OTHER

3. METHODS:

- ☐ SINGLE EXPOSURE
- ☐ MULTIPLE EXPOSURE
- ☐ DOUBLE WALL TECHNIQUE
- ☐ SINGLE WALL TECHNIQUE
- ☐ DOUBLE WALL VIEWING
- ☐ SINGLE WALL VIEWING
- ☐ SINGLE FILM LOADING
- ☐ DOUBLE FILM LOADING

4. IQI (PENETRAMETER) DATA:

DESIGNATION 18

SENSITIVITY LEVEL 0/6

LOCATION (FS) ((SS))

5. FILM DATA:

FILM TYPE 80

FILM DENSITY MAX. 3.5

FILM DENSITY MIN. 2.0

6. SOURCE DATA:

ISOTOPE 219C SIZE, 1/2

CURIES 60 E.T. 96

S.O.D. 12 S.S.O.F. 12

RADIOGRAPHER

INTERPRETER

CUSTOMER INSPECTOR

DATE REVIEWED

DATE REVIEWED

Reviewed / Accepted

L.Q.S. # 3

William Brennaugh

JOB # X-7929

17496. 0001

AFWHT

TOP / BOTTOM CH

SPOT

4W-51

SA - 333.4

MATERIAL THICKNESS

24 1/2

Q.C. WELDMENT

CODE/STANDARD:

ASME SEC. VIII, DIV. 1, A-

[illegible]

RADIOGRAPHIC DATA

1. RT DATE 8-17-12

2. MATERIAL EXAMINED:

4) WELDMENT

() **CASTING**

() OTHER

3. METHODS:

+) SINGLE EXPOSURE

() MULTIPLE EXPOSURE

() DOUBLE WALL TECHNIQUE

4) SINGLE WALL TECHNIQ

☐ DOUBLE WALL VIEWING

4) SINGLE WALL VIEWING

+) SINGLE FILM LOADING

() DOUBLE FILM LOADING

4. IQI (PENETRAMETER) DATA:

DESIGNATION_____

SENSITIVITY LEVEL 0.3

LOCATION (FS) (~~SS~~)

5. FILM DATA:

FILM TYPE 80

FILM DENSITY MAX. 3.5

FILM DENSITY MIN. 2.0

6. SOURCE DATA:

ISOTOPE ¹⁹² SIZE - 1x1.

CURIES 103 E.T. 24

S.O.D. 12 S.S.O.F. 12

RADIOGRAPHER Dean Lou

INTERPRETER Bruce Will 193332

DATE REVIEWED 8-18-12

CUSTOMER INSPECTOR

DATE REVIEWED
Reviewed / Accepted

I.Q.S. # 3

readersheet(A99).doc

William B. Brown

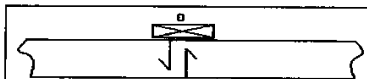
JOB # X-7929 Page 1 of 1
 PO # _____ Date 03/26/12

Client: Energy Exchanger Item Inspected: Blind Overlay IRISNDT# 136232
 Location: Tulsa, OK
 Code: ASME Sec. V ART. 4
SA578 Heat Treatment: _____

Procedure: 4.0 U.T. Surface Condition: As Welded Material: c/s
 UT Equipment: EPOCH XT IRISNDT# 101371412 Cal Date: 2/1/2012
 Calibration Block: IIV Probes: _____ Angle:(degrees) 0
 Test Piece: ASME 3/4" Basic Frequency:(mhz) 2.5
 Distance/Amp Cal: _____ Crystal Size:(inches) 1"
 Transfer Loss Cal: Direct Comparison Scan Gain: +6dB Couplant: UT-X Batch # IRIS
 Ref. Flaw Size: _____ Primary Reference Level: 80% FSH
 Scan Method: Contact TOF Lamination: X Thickness: _____ Shear Wave: _____

Scope: UT Inspection Performed On Blind Overlay

Results: X-7929 #1 - Accept
X-7929 #1A - Accept



UT Revealed No Rejectable Indications
 All Welds Are Acceptable Per Code

Unit: <u>455</u>	Miles: _____	Hrs.: _____	Consumables: _____	Technician: <u>Mark Rake</u> (print)
In: _____	Out: _____			Technician: <u>Mark Rake</u> (sign)
In: _____	Out: _____			SNT-TC -1A Level: <u>II</u> cert# <u>522</u>
				I am in full agreement with report contents:
				Client Representative: <u>[Signature]</u>

Reviewed / Accepted
 I.Q.S. # 3

William Branaugh
[Signature]

Tulsa, Oklahoma
 1115 West 41st Street

Phone: 918-446-8773

Fax: 918-446-8777

IRISNDT2529

DATE: 8-22-12

CERTIFICATION OF EXAMINATION
BY
MAGNETIC PARTICLE (DRY POWDER METHOD)
(Procedure MT-1 Rev. 6)

P.O. # 17496-0001

TAG # HX0042

ENERGY JOB # X-7929

MT OF THE FOLLOWING:
LIFTING LUG WELDS
BACK CUTS
ALL INT + EXT WELDS BEFORE
+ AFTER PWHT
ALL TEMP ATTACHMENTS AND
ARC STRIKES

This certification is made to document that all Magnetic Particle Examinations have been performed on X-7929 in accordance with the requirements of ASME Section VIII, Division I, Appendix VI, ASME Section V, Article 7, customer specifications and Energy Exchanger Company drawings.

TEST INSTRUMENT: ESC-Y400 / ESC MS-27D

PARTICLE TYPE & COLOR: DRY / GRAY Lot: 1774E

MATERIAL AND THICKNESS: SA-516-70 x .562

LIGHTING: 100 FT CANDLES MIN. @ SURFACE

ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager
ASNT NDT Level III, #193332
CWI # 10082731

Reviewed / Accepted
I.Q.S. # 3

William Bronaugh 

Energy Exchanger Company

BRINELL HARDNESS REPORT

JOB NO: X-7929

[illegible]

Reviewed / Accepted
I.Q.S. # 3

William Brennaugh

WP

SECTION 6

DATE: 8-21-12

CERTIFICATE OF WELD/SUPPORT PAD AIR TEST

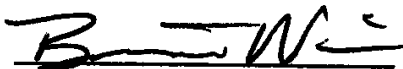
P.O. NO: 17496-0001

TAG NO: HK 0042

ENERGY JOB NO: X-7929

This is to certify all weld/support pads on the above referenced equipment were air tested @ 15 P.S.I. and found to be satisfactory.

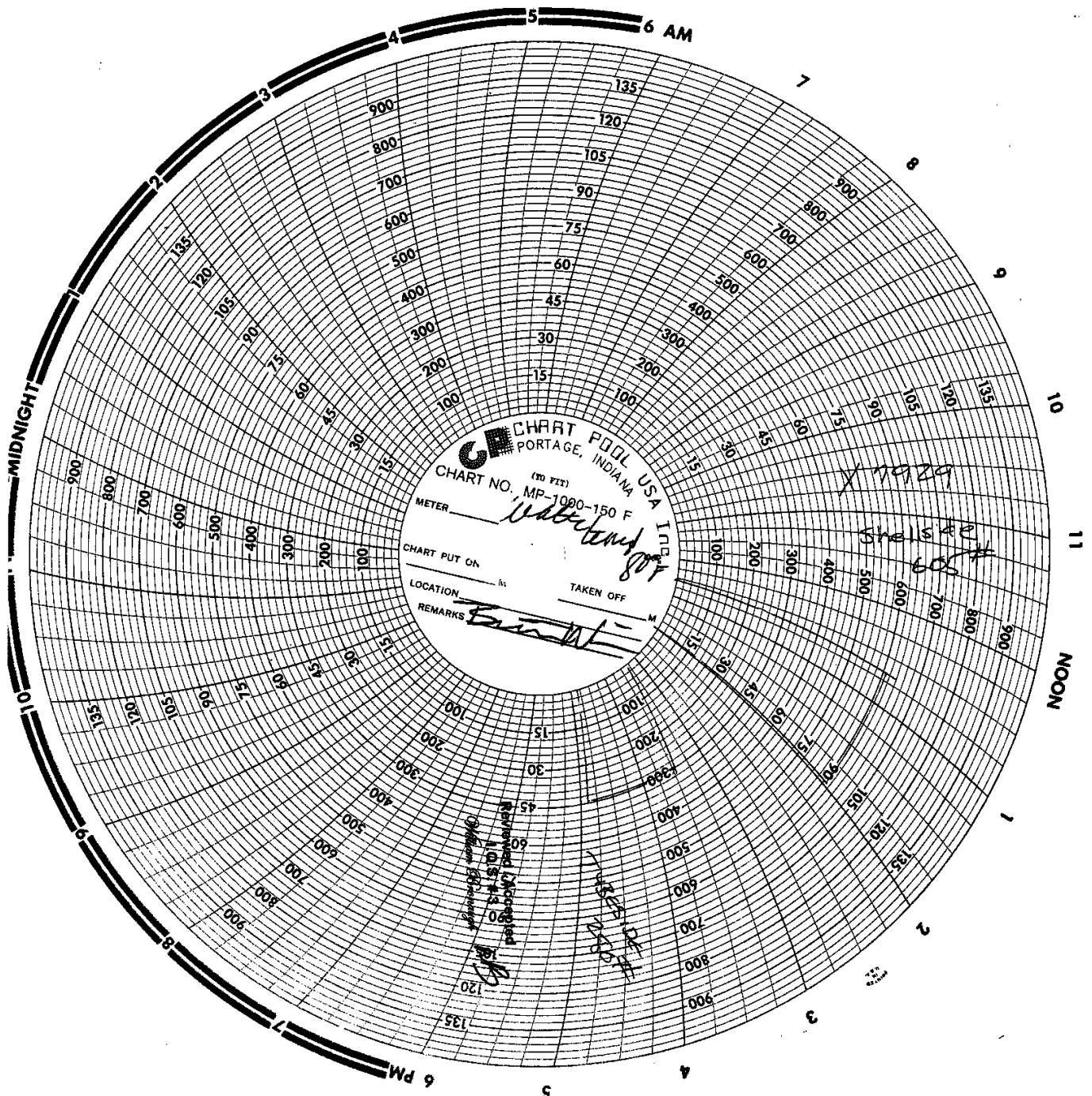
ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager

Reviewed / Accepted
I.Q.S. # 3

William Bronaugh 



DATE: 8-24-12

CERTIFICATE OF HYDROSTATIC TEST

P.O.# 17496-0001

TAG# HX0042

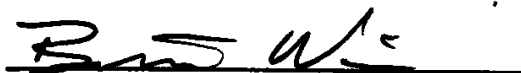
ENERGY JOB # X-7929

This is to certify the above referenced equipment was
hydro tested at the following pressures:

605 PSIG SHELL SIDE

280 PSIG TUBE SIDE

ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager

/lt

Reviewed / Accepted
I.Q.S. # 3

William Brennaugh



Date: 8-30-12

CERTIFICATE OF NITROGEN PURGE

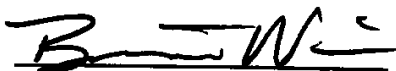
P.O. NO: 17496-0001

TAG NO: HK0042

ENERGY JOB NO: X-7929

This is to certify the above referenced equipment was purged
with 15 PSI dry nitrogen after hydro test.

ENERGY EXCHANGER COMPANY



Brent Wilson
Quality Control Manager

Reviewed / Accepted
I.Q.S. #3

William Brennaugh



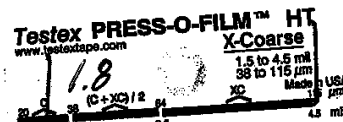
ENERGY EXCHANGER COMPANY

PAINT CHECK LIST

Contractor: EATON INDUSTRIAL COATINGS

Unit Serial No: X-7929 Customer Specification No: _____

Paint Required: 3-4 mils 2-9 Primer; _____ mils _____ intermediate;
1-2 mils 892 Finish



1. Paint Shelf Life

A. Materials in accordance with Project Specifications & Drawings

2. Notify Energy Exchanger Company

3. Weather Conditions:

83 temperature; 7/2 humidity; 80 substrate, 7 dew point,
 EEC verify _____

4. Surface Prep:

(Req'd) SSPC-SP- 10 1-2 mils; (Actual) SSPC-SP- 10 1.8 mils

5. Surface Contamination

6. Surface Imperfections

7. Paint Contractor Re-check above

8. Storage, Mixing, Application

9. Masked Surfaces

10. Check Compressed Air for Contamination

11. Wet Film Thickness:

(Req'd) _____ (Actual) _____

12. Dry Film Thickness:

SHELL (Req'd) 3-4 (Actual) 4
 SUPPLY " 4-6 " 6

13. Tie Down/Shipping Preparation

2nd COAT
 TEMP 80
 S. TEMP 80
 HUMIDITY 65%
 DRY POINT 68

Satisfactory	Date
<u>mc</u>	<u>8/27/12</u>

<u>mc</u>	<u>8/27/12</u>
-----------	----------------

<u>mc</u>	<u>8/27/12</u>

<u>mc</u>	<u>8/29/12</u>
-----------	----------------

<u>mc</u>	<u>8/29/12</u>
-----------	----------------

Reviewed / Accepted
 I.Q.S. #3

William Bronaugh

SECTION 7

ENERGY EXCHANGER COMPANY

SPARE PARTS LISTS

- * 1) PRICES ARE F.C.A. ENERGY SHOPS, TULSA, OK.
- * 2) SHIPPING TERMS ARE PREPAY AND CHARGE.
- * 3) PRICE ARE VALID FOR TWELVE (12) MONTHS.
- * 4) UNIT PRICES ARE PER SET OF SPARE GASKETS.

**** 1) ENERGY EXCHANGER COMPANY RECOMMENDS ONE SET OF SPARE GASKETS PER EXCHANGER PER YEAR OF OPERATION.**

Energy Exchanger Company



Installation, Operation and Maintenance Manual for **SHELL AND TUBE HEAT EXCHANGERS**

A Member of the
Tubular Exchanger
Manufacturers Association



Energy Industrial Park/1844 North Garnett/Tulsa, Oklahoma 74116-1698
918 437-3000 / FAX 918 437-7144

INSTALLATION, OPERATION & MAINTENANCE MANUAL — Shell & Tube Heat Exchangers

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FOREWORD

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GENERAL INSTRUCTIONS FOR INSTALLATION, OPERATION AND MAINTENANCE OF SHELL AND TUBE HEAT EXCHANGERS

SAFETY CONSIDERATIONS

LIFTING DEVICES

When eyebolts, lugs, clips or other devices are installed by the manufacturer, it is intended that these lifting devices are to lift only the empty components to which they are attached and not the assembled heat exchanger, unless explicitly specified otherwise.

Proper rigging must be used in lifting assembled heat exchangers. Exchangers are usually much heavier than pressure vessels of comparable dimensions; therefore, weights should be carefully evaluated. Where available, determinations should include reference to manufacturers drawings, added components, bill of lading, etc. Special care must be exercised in handling exchangers with expansion joints.

ROUTINE PRECAUTIONARY PROCEDURES

A heat exchanger is a multi-chamber pressure vessel designed at specific limits of pressure, temperature and fluid flow conditions as shown on the exchanger specification sheet and heat exchanger nameplate(s). The process system, which includes the heat exchanger, must be safeguarded so that the heat exchanger design conditions and operating limits are not exceeded. All operating and maintenance personnel should be made aware of specific limitations including pressures and temperatures, flow rates, start-up/shut-down procedures and cleaning procedures.

Do not remove channel covers, shell covers, floating head covers, bonnets or connecting piping until all pressure has been relieved and both shell and tube sides are completely drained. Exceptions are permissible when design permits independent pressure testing of shell or tube sides.

Plugged tubes and double tubesheets, unless vented, may remain pressurized after shell and tube sides are depressurized. Caution may be exercised in loosening of tube plugs or opening of vents to avoid sudden release of pressure or harmful fluids.

Many heat exchangers circulate fluids which are toxic, lethal or flammable and dangerous to the human system. These fluids could cause problems if bolted or threaded joints are not maintained in leak-tight condition under operating or no-flow ambient conditions. Proper precautions, such as effective draining and purging, must be taken in handling and decontamination when exchangers are opened for any reason. It is essential that the user advise maintenance or repair personnel, including outside contractors, when toxic, lethal, or flammable conditions exist.

When heat exchangers are cleaned, it is important that full characteristics of the circulating fluids, fouling material, and the cleaning agent be known and care exercised in handling them. Use eye protection, a respirator, or other appropriate protective devices.

Do not blow out heat exchangers with air when the process fluids or the cleaning fluids being handled are flammable or reactive.

SECTION 1 HEAT EXCHANGER NOMENCLATURE

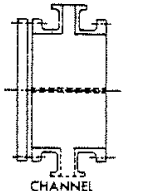
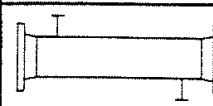
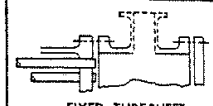
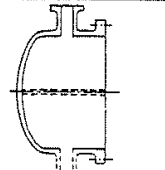
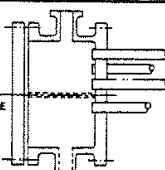
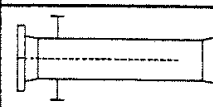
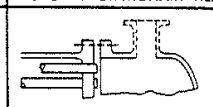
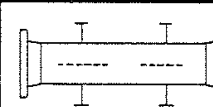
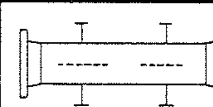
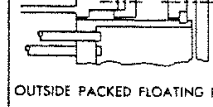
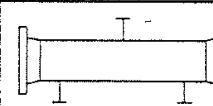
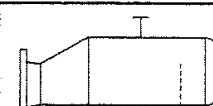
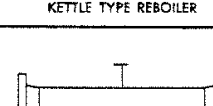
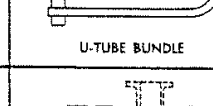
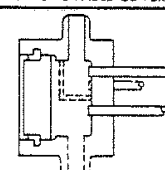
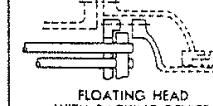
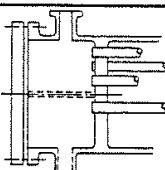
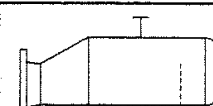
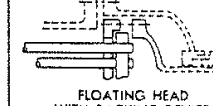
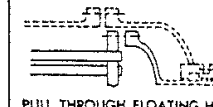
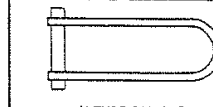
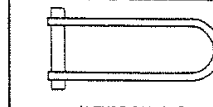
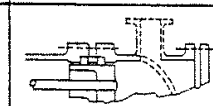
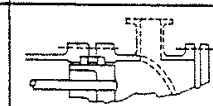
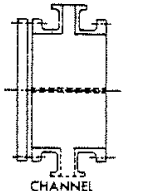
N-1 TYPE DESIGNATION-RECOMMENDED PRACTICE

It is recommended that heat exchanger type be designated by letters as described below.

N-1.1 TYPE

Type designation shall be by letters describing stationary head, shell (omitted for bundles only), and rear head, in that order, as indicated in Figure N-1.1.

Figure N-1.1

FRONT END STATIONARY HEAD TYPES		SHELL TYPES		REAR END HEAD TYPES	
A		E		L	
	CHANNEL AND REMOVABLE COVER				
					
	BONNET (INTEGRAL COVER)				
					
B	REMOVABLE TUBE BUNDLE ONLY	F		M	
	CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER				
					
	SPLIT FLOW				
					
C	DOUBLE SPLIT FLOW	H		N	
	DIVIDED FLOW				
					
	KETTLE TYPE REBOILER				
					
D	CROSS FLOW	X		P	
					
	SPECIAL HIGH PRESSURE CLOSURE				
					
	FLOATING HEAD WITH BACKING DEVICE				
N		K		S	
	CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER				
					
	PULL THROUGH FLOATING HEAD				
					
D	U-TUBE BUNDLE	U		W	
					
	EXTERNALLY SEALED FLOATING TUBESHEET				
					
	CHANNEL AND REMOVABLE COVER				

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SECTION 1 HEAT EXCHANGER NOMENCLATURE

N-2 NOMENCLATURE OF HEAT EXCHANGER COMPONENTS

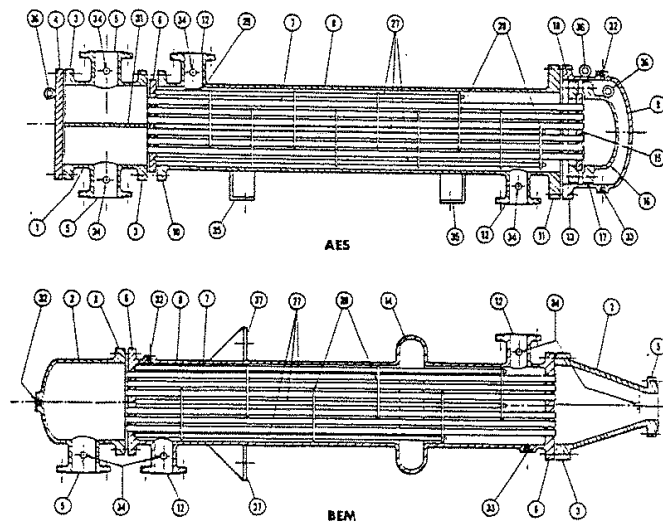
For the purpose of establishing standard terminology, Figure N-2 illustrates various types of heat exchangers. Typical parts and connections, for illustrative purposes only, are numbered for identification in Table N-2. (See Note)

Table N-2

1. Stationary Head — Channel	20. Slip-on Backing Flange
2. Stationary Head — Bonnet	21. Floating Head Cover — External
3. Stationary Head Flange — Channel or Bonnet	22. Floating Tubesheet Skirt
4. Channel Cover	23. Packing Box
5. Stationary Head Nozzle	24. Packing
6. Stationary Tubesheet	25. Packing Gland
7. Tubes	26. Lantern Ring
8. Shell	27. Tierods and Spacers
9. Shell Cover	28. Transverse Baffles or Support Plates
10. Shell Flange — Stationary Head End	29. Impingement Plate
11. Shell Flange — Rear Head End	30. Longitudinal Baffle
12. Shell Nozzle	31. Pass Partition
13. Shell Cover Flange	32. Vent Connection
14. Expansion Joint	33. Drain Connection
15. Floating Tubesheet	34. Instrument Connection
16. Floating Head Cover	35. Support Saddle
17. Floating Head Flange	36. Lifting Lug
18. Floating Head Backing Device	37. Support Bracket
19. Split Shear Ring	38. Weir
	39. Liquid Level Connection

NOTE: Manufacturers may identify corresponding parts using numbers differing from the above.

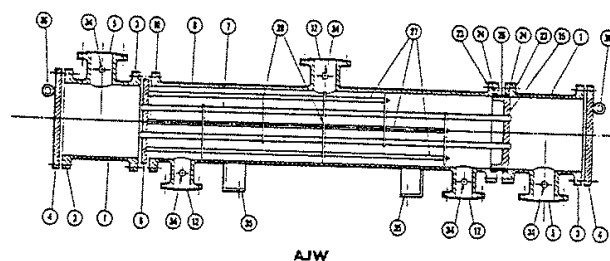
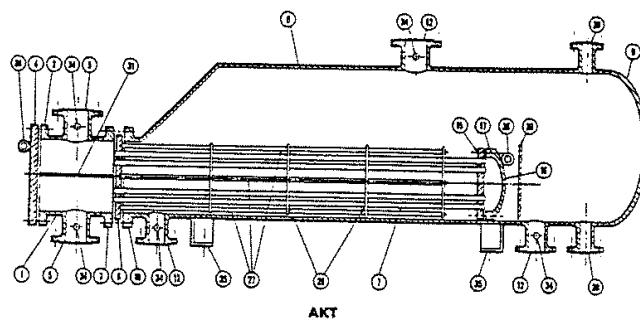
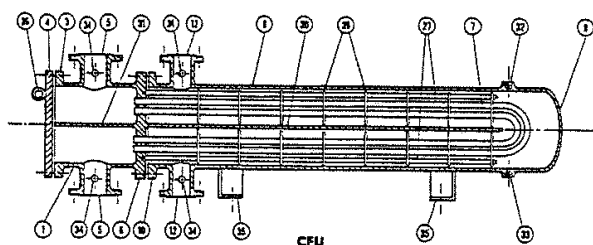
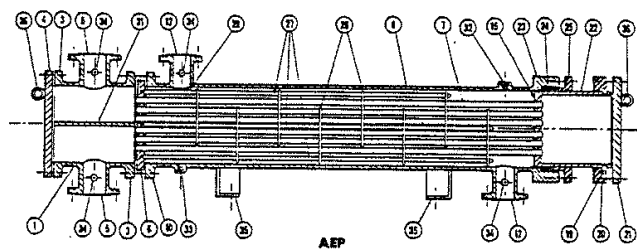
Figure N-2



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SECTION 1 HEAT EXCHANGER NOMENCLATURE

Figure N-2 (Continued)



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SECTION 2

HEAT EXCHANGER IDENTIFICATION AND DEFINITIONS

D-2.1 EXCHANGER IDENTIFICATION

D-2.11 COMPLETE EXCHANGER

Manufacturers normally assign a serial number which is unique and permanently identifies each exchanger. Inquiries to the manufacturer should reference this number when it is shown on the nameplate and manufacturer's drawings. Additional useful information normally shown includes size, type, and item number.

D-2.12 EXCHANGER PARTS

Exchanger parts should be identified by the exchanger serial number and by the name and part number corresponding to those shown in Section 1, Table N-2, or as shown on the manufacturer's drawings.

D-2.2 DEFINITIONS

D-2.21 EXCHANGER UNIT

One or more exchangers designed for a specific service.

D-2.22 ITEM NUMBER

The purchaser's identification number for an exchanger unit.

D-2.23 NORMAL OPERATING CONDITIONS

The thermal and hydraulic performance requirements generally specified for designing the heat exchanger.

D-2.24 UPSET CONDITIONS

A departure from specified operating conditions.

D-2.25 PULSATING FLUID CONDITIONS

Conditions of flow generally characterized by rapid fluctuations in pressure and flow rate resulting from sources outside of the heat exchanger such as: pumps, compressors, blowers, etc.

D-2.26 START-UP CONDITIONS

The conditions of operation which exist from the time that flow of either or both process streams is initiated to the time that steady-state operating conditions are achieved.

D-2.27 SHUT-DOWN CONDITIONS

The conditions of operation which exist from the time of steady-state operating conditions to the time that flow of both process streams has ceased.

D-2.28 CODE

All references to Code herein mean the ASME Boiler and Pressure Vessel Code, Section VIII, Division 1, Pressure Vessels.

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SECTION 3

INSTALLATION OF HEAT EXCHANGERS

E-3 INSTALLATION

E-3.1 INSTALLATION PLANNING

E-3.11 CLEARANCE FOR DISMANTLING

For straight tube exchangers fitted with removable bundles, provide sufficient clearance at the stationary head end to permit removal of the bundle from the shell and provide adequate space beyond the rear head end to permit removal of the shell cover and/or floating head cover.

For fixed tubesheet exchangers, provide sufficient clearance at one end to permit withdrawal and replacement of the tubes and enough space beyond the head at the opposite end to permit removal of the bonnet or channel cover.

For U-tube heat exchangers, provide sufficient clearance at the stationary head end to permit withdrawal of the tube bundle, or at the opposite end to permit removal of the shell.

E-3.12 FOUNDATIONS

Foundations must be adequate so that exchangers will not settle and impose excessive strains on the exchanger. Foundation bolts should be set to allow for setting inaccuracies. In concrete footings, pipe sleeves at least one size larger than bolt diameter slipped over the bolt and cast in place are best for this purpose, as they allow the bolt center to be adjusted after the foundation has set.

E-3.13 PIPING

E-3.131 THERMAL EXPANSION

It is essential that adequate provisions be made in the connecting piping to allow for thermal expansion without imposing unspecified piping loads on the exchanger. This is particularly important for single pass internal floating head designs, outside packed floating heads, and fixed tubesheet types with shell expansion joints. Unless precautions are taken, leakage or damage may result.

E-3.132 PULSATION AND VIBRATION

In all installations, care must be taken to eliminate or minimize transmission of fluid pulsations and mechanical vibrations to the heat exchangers.

E-3.14 CLEANING

Convenient means should be provided for cleaning the unit as suggested under "Maintenance." (Section 5)

E-3.15 LEVELING

Exchangers must be installed so that pipe connections are made without forcing, disassembling or loosening of flanged assemblies.

E-3.16 STACKED HEAT EXCHANGERS

The manufacturer normally stacks exchangers prior to completion of fabrication to insure proper alignment of interconnecting nozzles. Shims separating the intermediate shell supports are normally used. Therefore, it is important that shims, if used, match the thicknesses and locations used by the manufacturer.

E-3.17 CONNECTION PROTECTORS

Remove shipping covers and plugs immediately prior to installation. Before connecting to piping, inspect all openings in the heat exchanger for foreign material. The entire system should be clean and free of foreign objects before starting operation. Do not expose heat exchanger internals to moisture or harmful contaminants.

E-3.18 PIPE CONNECTIONS

Pipe connections must be in accordance with the manufacturer's dimensional outline drawing.

E-3.19 FOUNDATION BOLTS

Foundation bolts should be loosened at one end of unit to allow free expansion of shells. Slotted holes in supports are provided for this purpose.

E-3.20 If the heat exchanger is equipped with a bellows type expansion joint, the expansion joint must be adequately protected during shipping and setting. Shipping supports must be removed in accordance with instructions.

E-3.3 HEAT EXCHANGER STORAGE

Heat exchangers normally are not protected for storage. If storage is necessary, a covered location at uniform temperature is preferable and provisions must be made to protect the heat exchanger interior from moisture, freezing, or harmful contaminants.

SECTION 4

OPERATION OF HEAT EXCHANGERS

E-4. OPERATION

E-4.1 PERFORMANCE OF HEAT EXCHANGERS

Satisfactory performance and service life can be expected only from heat exchangers which are properly installed, operated in accordance with design conditions, and receive preventive maintenance and cleaning on an appropriate schedule. These requirements are the responsibility of the user.

E-4.11 PERFORMANCE FAILURES

The failure of heat exchanger equipment to perform satisfactorily may be caused by one or more factors, such as:

- (1) Excessive fouling.
- (2) Air or gas binding resulting from improper piping installation or lack of suitable vents.
- (3) Operating conditions differing from design conditions.
- (4) Maldistribution of flow in the unit.
- (5) Excessive clearances between the baffles and shell and/or tubes, due to corrosion.
- (6) Improper thermal design.
- (7) Flooding resulting from inadequate drainage of condensate.

E-4.2 OPERATION OF HEAT EXCHANGERS

E-4.21 OPERATING PROCEDURES — NORMAL

Before placing any exchanger in operation, reference should be made to the exchanger drawings, specification sheet(s), and name plate(s) for any special instructions. Local safety and health regulations must be considered. Improper start-up or shut-down sequences, particularly of fixed tubesheet units, may cause leaking of tube-to-tubesheet and/or bolted flanged joints.

E-4.22 START-UP OPERATION

Most exchangers with removable tube bundles may be placed in service by first establishing circulation of the cold medium, followed by the gradual introduction of the hot medium. During start-up all vent valves should be opened and left open until all passages have been purged of air and are completely filled with fluid. For fixed tubesheet exchangers, fluids must be introduced in a manner to minimize differential expansion between the shell and tubes. (Refer E-4.24)

E-4.23 SHUT-DOWN OPERATION

For exchangers with removable bundles, the units may be shut down by first gradually stopping the flow of the hot medium and then stopping the flow of the cold medium. If it is necessary to stop the flow of cold medium, the circulation of hot medium through the exchanger should also be stopped. For fixed tubesheet exchangers the unit must be shut down in a manner to minimize differential expansion between shell and tubes, (Refer E-4.24). When shutting down the system, all units should be drained completely when there is the possibility of freezing or corrosion damage. To guard against water hammer, condensate should be drained from steam heaters and similar apparatus during start-up or shut-down. To reduce water retention after drainage, the tube side of water cooled exchangers should be blown out with air.

E-4.24 START-UP AND SHUT-DOWN PROCEDURES FOR FIXED TUBESHEET EXCHANGERS

When start-up and shut-down instructions are furnished, they must be followed. When specific instructions are not supplied, the recommended procedure is to start, or stop, the circulation of both shell and tube side fluids gradually and at the same time.

E-4.3 OPERATING LIMITATIONS

E-4.31 NORMAL OPERATING CONDITIONS

Equipment must not be operated at conditions more severe than those specified on the heat exchanger specification sheet, and/or name plates. A change in service or deviation from the specified operating conditions may require that a mechanical design check be made in order to avoid possible damage to the heat exchanger.

E-4.32 SUPPLEMENTAL NAME PLATE DATA

The manufacturer may supply supplemental data where it is pertinent to the operation or testing of the exchanger. This may include information pertaining to differential design and test pressure conditions, restrictions on operating conditions for fixed tubesheet type exchangers, or other restrictive conditions applicable to the design and/or operation of the unit or its components. Such information may be noted on the standard name plate or on a supplemental plate attached to the exchanger at the name plate location.

E-4.33 FLOW RATE

Heat exchangers are not to be operated at flow rates greater than those shown on the heat exchanger specification sheet. Excessive flow rates can cause vibration and/or erosion and severely damage the heat exchanger.

E-4.34 HYDRAULIC HAMMER

Exchangers should not be subjected to pulsating, slugging, or unstable flow conditions which may impair performance or result in destructive hydraulic hammer. Adequate condensate drainage is essential for steam heaters and similar apparatus.

E-4.35 TEMPERATURE SHOCKS

Exchangers normally should not be subjected to abrupt temperature fluctuations. Hot fluid must not be suddenly introduced when the unit is cold, nor cold fluid when the unit is hot.

E-4.36 FLUID TEMPERATURE LIMITS

E-4.361 Heat exchangers are not to be operated at fluid temperatures more severe than those shown on the heat exchanger specification sheet.

E-4.362 For fixed tubesheet exchangers fluid inlet temperatures may be severely limited when there is no fluid on the opposite side. Operation with fluid circulation on one side only should be avoided, unless specifically provided for in the design of the heat exchanger.

E-4.4 BOLTED JOINTS

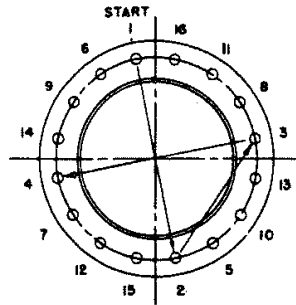
E-4.41 RECOMMENDATIONS FOR RETIGHTENING

Heat exchangers are pressure tested before leaving the manufacturer's shop in accordance with ASME Code requirements. However, normal yielding of gasket joints may occur in the interval between testing in the manufacturer's shop and installation at the job site. Therefore, all external bolted joints may require retightening after installation and, if necessary, after the exchanger has reached operating temperature.

E-4.42 RECOMMENDED BOLT TIGHTENING PROCEDURE

It is important that all bolted joints be tightened uniformly and in a diametrically staggered pattern as illustrated in Figure E-4.42 except for special high pressure closures when the instructions of the manufacturer should be followed.

Figure E-4.42



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SECTION 5 MAINTENANCE OF HEAT EXCHANGERS

E-5. MAINTENANCE

E-5.1 INSPECTION OF UNIT

At regular intervals and as frequently as experience indicates, an examination should be made of the interior and exterior condition of the unit. Neglect in keeping all tubes clean may result in complete stoppage of flow through some tubes which could cause severe thermal strains, leaking tube joints, or structural damage to other components. Sacrificial anodes, when provided, should be inspected to determine whether they should be cleaned or replaced.

E-5.11 INDICATIONS OF FOULING

Exchangers subject to fouling or scaling should be cleaned periodically. A light sludge or scale coating on the tube greatly reduces its efficiency. A marked increase in pressure drop and/or reduction in performance usually indicates cleaning is necessary. The unit should first be checked for air or vapor binding to confirm that this is not the cause for the reduction in performance. Since the difficulty of cleaning increases rapidly as the scale thickness or deposit increases, the intervals between cleanings should not be excessive.

E-5.12 PREVENTIVE MAINTENANCE

Additional important reasons for inspections include:

- (1) Corrosion and erosion may weaken various parts of an exchanger and may eventually cause mechanical failure, or impaired performance.
- (2) Determination of fouling and corrosion rates for prediction of service life and scheduling down-time.
- (3) Implementation of preventive maintenance procedures which reduce rates of fouling and general deterioration, or eliminate leaks.

E-5.13 ACCESS TO TUBES

Before disassembly, the user must assure himself that the unit has been depressurized, vented and drained, neutralized and/or purged of hazardous materials.

To inspect the inside of the tubes and also make them accessible for cleaning, the following procedures should be used:

- (1) Stationary Head End
 - (a) Type A, C, & D, remove cover only.
 - (b) Type B, remove bonnet.
- (2) Rear Head End
 - (a) Type L, N, & P, remove cover only.
 - (b) Type M, remove bonnet.
 - (c) Type S & T, remove shell cover and floating head cover.
 - (d) Type W, remove channel cover or bonnet.

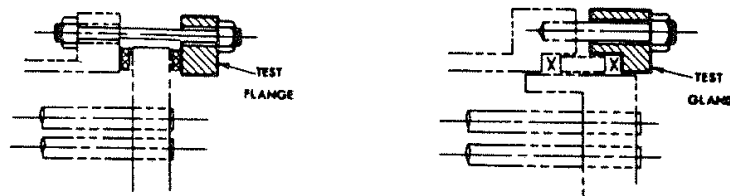
E-5.14 LOCATING LEAKS IN TUBES

The following procedures may be used to locate perforated or split tubes and leaking joints between tubes and tubesheets. In most cases, the entire front face of each tubesheet will be accessible for inspection. The point where water escapes indicates a defective tube or tube-to-tubesheet joint.

- (1) Units with removable channel cover:
Remove channel cover and apply hydraulic pressure in the shell.
- (2) Units with bonnet type head:
 - (a) On fixed tubesheet units where tubesheets are an integral part of the shell, remove bonnet and apply hydraulic pressure in the shell.
 - (b) On fixed tubesheet units where tubesheets are not an integral part of the shell and on units with removable bundles, remove bonnet, re-bolt tubesheet to shell or install test flange or gland, whichever is applicable, and apply hydraulic pressure in the shell. See Figure E-5.14-1 for typical test flange and test gland.

CAUTION: Bolting directly to the edges of the tubesheets may result in overstress unless it was considered in the design of the tubesheets.

Figure E-5.14-1

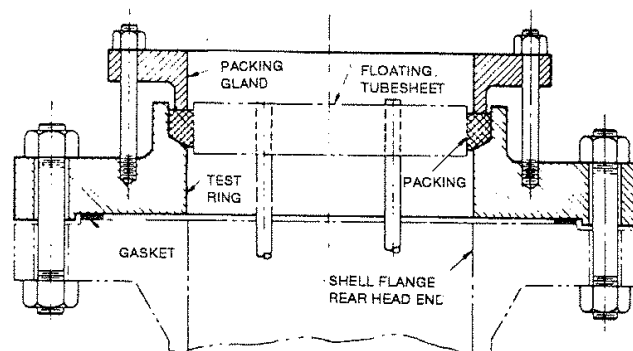


- (3) Units with type S or T floating head:
Remove channel cover or bonnet, shell cover and floating head cover. Install test ring and bolt in place with gasket and packing. Apply hydraulic pressure in the shell. A typical test ring is shown in Figure E-5.14-2. When a test ring is not available it is possible to locate leaks in the floating head end by removing the shell cover and applying hydraulic pressure in the tubes. Leaking tube joints may then be located by sighting through the tube lanes.

NOTE: Care must be exercised when testing partially assembled exchangers to prevent over extension of expansion joints or overloading of tubes and/or tube-to-tubesheet joints.

Hydrostatic test should be performed so that the temperature of the metal is at least 30°F above the minimum design metal temperature, MDMT.

Figure E-5.14-2



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E-5.2 TUBE BUNDLE REMOVAL

Bundles must be withdrawn from the stationary end as noted on the drawing. All parts such as glands, retainer rings, split shear rings and closure packing must be removed before attempting to remove bundle. Threaded eyebolts may be attached to the stationary tubesheet when tapped holes have been provided for this purpose. When eyebolt tappings are not provided, rods may be passed through two or more tubes and attached to a steel bearing plate over the floating tubesheet. Provide a protective spacer between the bearing plate and tubesheet to protect the tube ends.

E-5.21 HANDLING TUBE BUNDLES

Tube bundles should not be handled with hooks or other devices which might damage the tubes. Bundles should be supported on cradles or skids. Horizontal tube bundles should be lifted by means of suitable slings. Baffles can be bent and damaged by dragging a bundle over a rough surface. All gasket and packing contact surfaces should be protected from accidental damage since these areas are generally difficult to repair.

E-5.3 CLEANING TUBE BUNDLES

E-5.31 CLEANING METHODS

The heat transfer surfaces of heat exchangers should be kept reasonably clean to assure satisfactory performance. Convenient means for cleaning should be made available.

Heat exchangers may be cleaned by either chemical or mechanical methods. The method selected must be the choice of the operator of the plant and will depend on the type of deposit and the facilities available in the plant. Following are several cleaning procedures that may be considered:

- (1) Circulating hot wash oil or light distillate through tubes or shell at high velocity may effectively remove sludge or similar soft deposits.
- (2) Some salt deposits may be washed out by circulating hot fresh water.
- (3) Commercial cleaning compounds are available for removing sludge or scale provided hot wash oil or water is not available or does not give satisfactory results.
- (4) High pressure water jet cleaning.
- (5) Scrapers, rotating wire brushes, and other mechanical means for removing hard scale, coke, or other deposits.
- (6) Employ services of a qualified organization that provides cleaning services. These organizations will check the nature of the deposits to be removed, furnish proper solvents and/or acid solutions containing inhibitors, and provide equipment and personnel for a complete cleaning job.

E-5.32 CLEANING PRECAUTIONS

- (1) Tubes should not be cleaned by blowing steam through individual tubes since this heats the tube and may result in severe expansion strain, deformation of the tube, or loosening of the tube-to-tube sheet joint.
- (2) When mechanically cleaning a tube bundle, care should be exercised to avoid damaging the tubes.
- (3) Cleaning compounds must be compatible with the metallurgy of the exchanger.
- (4) Cleaning procedures, especially those which circulate hot fluids through the tubes, can cause significant temperature differentials between the shell and tubes. Such cleaning procedures can cause damage to fixed tubesheet heat exchangers and should be avoided unless specifically provided for in the design of the heat exchanger.

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E-5.4 GASKET REPLACEMENT

Gasket and gasket surfaces should be thoroughly cleaned and should be free of scratches and other defects. Gaskets should be properly positioned before attempting to retighten bolts. It is recommended that when a heat exchanger is dismantled for any cause, it be reassembled with new gaskets. This will tend to prevent future leaks and/or damage to the gasket seating surfaces of the heat exchanger. Composition gaskets become dried out and brittle so that they do not always provide an effective seal when reused. Metal or metal jacketed gaskets, when compressed initially, flow to match their contact surfaces. In so doing they are work hardened and, if reused, may provide an imperfect seal or result in deformation and damage to the gasket contact surfaces of the exchanger.

Bolted joints and flanges are designed for use with the particular type of gasket specified. Substitution of a gasket of different construction or improper dimensions may result in leakage and damage to gasket surfaces. Therefore, any gasket substitutions should be of compatible design.

Any leakage at a gasketed joint should be rectified and not permitted to persist as it may result in damage to the gasket surfaces.

When metal jacketed filled type gaskets are used with a tongue and groove joint without a nubbin, the gasket should be installed so that the tongue bears on the seamless side of the gasket jacket. When a nubbin is used, the nubbin should bear on the seamless side.

E-5.5 SPARE AND REPLACEMENT PARTS

The procurement of spare or replacement parts from the manufacturer will be facilitated if the correct name for the part, as shown in Section 1, Table N-2, of this manual is given, together with the serial number, type, size, and other information from the nameplate. Replacement parts should be purchased from the original manufacturer.

E-5.6 EXPANDED TUBE JOINTS

E-5.61 TUBE HOLE FINISH

Tube hole finish affects the mechanical strength and leak tightness of an expanded tube-to-tubesheet joint. In general:

- (1) A rough tube hole provides more mechanical strength than a smooth tube hole. This is influenced by a complex relationship of modulus of elasticity, yield strength, and hardness of the materials being used.
- (2) A smooth tube hole does not provide the mechanical strength that a rough tube hole does, but it can provide a pressure tight joint at a lower level of tube wall reduction.
- (3) Very light wall tubes require a smoother tube hole finish than heavier wall tubes.
- (4) Significant longitudinal scratches can provide leak paths through an expanded tube-to-tubesheet joint and should therefore be removed.

E-5.62 TUBE WALL REDUCTION

The optimum tube wall reduction for an expanded tube-to-tubesheet joint depends on a number of factors, some of which are:

- (1) Tube hole finish.
- (2) The presence or absence of the tube hole serrations (grooves).
- (3) The tube hole size and tolerance.
- (4) Tubesheet ligament width and its relation to tube diameter and thickness.

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- (5) Tube wall thickness.
- (6) Tube hardness and change in hardness during cold working.
- (7) Tube O.D. tolerance.
- (8) Type of expander used.
- (9) Type of torque control or final tube thickness control.
- (10) Function of tube joint, i.e., strength in resistance to pulling out, minimum cold work for corrosion purposes, freedom from leaks, ease of replacement, etc.
- (11) Length of expanded joint.
- (12) Compatibility of tube and tubesheet materials.

E-5.63 TUBE EXPANDING

A suitable tube expander should be used to tighten a leaking tube joint. Care should be taken to insure that tubes are not over expanded. Do not re-expand tubes which are not leaking. Do not expand tubes beyond 1/8" from the back face of the tubesheet.

E-5.64 TYPICAL COMPUTATION OF TUBE WALL REDUCTION

EXAMPLE OF DETERMINING THE EXPANDED TUBE INSIDE DIAMETER FOR A SPECIFIED AMOUNT OF WALL REDUCTION

GIVEN: TUBE DIAMETER = 0.750
 TUBE HOLE DIAMETER = 0.760
 TUBE WALL THICKNESS = 0.065
 SPECIFIED WALL REDUCTION = 5%

SOLUTION: WALL REDUCTION = $0.065 \times 0.05 = 0.00325$
 FINAL WALL THICKNESS = $0.065 - 0.00325 = 0.06175$
 FINAL EXPANDED TUBE I.D. = $0.760 - 2(0.06175) = 0.6365$

CAUTION: *THE USE OF NOMINAL DIMENSIONS, WITHOUT ATTENTION TO PERMISSIBLE TOLERANCES CAN LEAD TO INCORRECT CONCLUSIONS.*

EXPANDING PROCEDURES SHOULD BE BASED ON MEASURED DIMENSIONS, NOT NOMINAL DIMENSIONS.

E-5.65 PLUGGING TUBES IN TUBE BUNDLES

In U-tube heat exchangers, and other exchangers of special design, it may not be feasible to remove and replace defective tubes. Defective tubes may be plugged using commercially available tapered plugs with ferrules or tapered only plugs which may or may not be seal welded. Excessive tube plugging may result in reduced thermal performance, higher pressure drop, and/or mechanical damage.

E-5.7 WELDED TUBE JOINTS

Welding procedure and testing techniques for either seal welded or strength welded tube joints normally are by agreement between manufacturer and purchaser and therefore should be considered prior to repairs to welded tube joints or tube replacements.

E-5.8 END FLANGES AND BOLTING

The purpose of this section is to alert the users, and provide basic data for consideration in assembly and maintenance of bolted flanged joints. Controlling the torque applied in tightening threaded bolts is the most economical and commonly used method for the control of initial tension. There are several factors which affect the relationship between torque and tension of threaded bolts. A few of the factors are the type of lubricant and/or plating, if any, used on the threads and the materials from which the bolt and nut are manufactured.

In assembling gasketed joints it is essential that the gasket first be seated uniformly by initial tightening of studs using a staggered quarterly pattern. After the gasket is seated, torquing may be completed stepwise to the final desired value, following a staggered or sequential pattern.

The following may be used as references and aids when assembling bolted flanged joints:

1. Recommended assembly and maintenance of flanged joints as shown in the Crane Company Catalog #60, Engineering Data Section.
2. Torque Control, Sixth Edition, Sturtevant Company.
3. ASME Code, Appendix S.

E-5.9 SPECIAL HIGH PRESSURE CLOSURES

The instructions of the heat exchanger manufacturer should be followed when provided.

E-5.10 REPAIRS, ALTERATION, RERATING, AND REPLACEMENT PARTS.

In order to maintain the Code integrity of the vessel, any repairs by welding, subsequent heat treating, alterations, replacement pressure parts, or rerating must be done in accordance with the National Board Inspection Code.

Potentially toxic or harmful contaminants must be neutralized prior to repairs.

Following repair by welding, post weld heat treatment may be a requirement for preservation of resistance to corrosion.