

# FABSCO SHELL & TUBE, LLC

## MANUFACTURING RECORD BOOK INDEX SHEET

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<b>KBR</b>	
<b>ACCEPTANCE FOR ENGINEERING USE</b>	
THIS DOCUMENT IS:	
ACCEPTED	(Code 1) _____
ACCEPTED WITH COMMENTS	(Code 2) _____
NOT ACCEPTED	(Code 3) _____
NOT REVIEWED	(Code 4) <u>  X  </u>
ACCEPTANCE DOES NOT RELIEVE SUPPLIER FROM FURNISHING MATERIAL IN CONFORMANCE WITH ORDER. REFER TO SDR-1 FOR FULL DEFINITION OF ACCEPTANCE CONDITIONS.	
DISCIPLINE _____	EXCH _____
DATE _____	BY _____ MW _____

29NOV12

HX2065 AGR FLASH GAS COOLER

Southern Company Generation    Kemper County  
MM233798    0    Unit 1

FABSCO SHELL & TUBE    PO: MPC17901-0001  
S11-10279-8 HX2065 DATA BOOK    Rev: 0  
IGCC - GASIFIER - MULTIPAGE - SELEXOL AREA MISC HEAT EXCHANGER -



**FORM U-1 MANUFACTURER'S DATA REPORT FOR PRESSURE VESSELS**  
**As Required by the Provisions of the ASME Code Rules, Section VIII, Division 1**

1. Manufactured and certified by FABSCO SHELL & TUBE, LLC., 2410 INDUSTRIAL ROAD, SAPULPA, OK 74066  
(Name and address of Manufacturer)

2. Manufactured for SOUTHERN COMPANY GENERATION  
(Name and address of Purchaser)

3. Location of installation KEMPER COUNTY, MS  
(Name and address)

4. Type: HORIZ. HEAT EXCHANGER S11-10279-8  
(Horiz., vert., or sphere) (Tank, separator, jkt. vessel, heat exch., etc.) (Mfg's serial No.)

N/A S11-10279-8-1 7539 2012  
(CRN) (Drawing No.) (Nat'l Bd. No.) (Year built)

5. ASME Code, Section VIII, Div. 1 2010 ED. N/A N/A  
(Edition and Addenda (date)) (Code Case No.) (Special Service per UG-120(d))

Items 6-11 incl. to be completed for single wall vessels, jackets of jacketed vessels, shell of heat exchangers, or chamber of multichamber vessels.

6. Shell (a) No. of course(s): 1/1 (b) Overall length (ft & in.): 16'-5 3/8" / 1'-4 1/4"

Course(s)			Material	Thickness		Long. Joint (Cat. A)			Circum. Joint (Cat. A, B, & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	18" O.D.	16'-5 3/8"	SA-312TP304L	1/4"	0"	S	NONE	0.85	1	FULL(RT-4)	1.0	N/A	N/A
1*	24" O.D.	1'-4 1/4"	SA-312TP304L	1/4"	0"	S	NONE	0.85	1	FULL(RT-4)	1.0	N/A	N/A
* ANNULAR DISTRIBUTOR													

7. Heads: (a) SA-240-304L (b) NA  
(Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.) (Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	END	3/16	0"	NA	NA	2:1	NA	NA	NA	NA	X	S	NONE	1.00
(b)														

If removable, bolts used (describe other fastening) NA  
(Mat'l Spec. No., Grade, size, No.)

8. Type of jacket NA Jacket closure NA  
(Describe as ogee & weld, bar, etc.)

If bar, give dimensions NA If bolted, describe or sketch.

9. MAWP 180 NA psi at max. temp. 290 NA °F Min. design metal temp. 10 °F at 180 psi.  
(internal) (external) (internal) (external)

10. Impact test NO, Exempt Per ug-20(f), ucs-66(a)(c), Fig ucs-66 Note(c) & ucs-67 at test temperature of NA °F.  
(Indicate yes or no and the component(s) impact tested)

11. Hydro. Test press. 234 Proof test NA

Items 12 and 13 to be completed for tube sections.

12. Tubesheet: SA-240-304L 19 1/8" 1 3/8" 0.1875" Bolted  
(Stationary (Mat'l Spec. No.)) (Dia., in. (subject to press.)) (Nom. thk., in.) (Corr. Allow., in.) (Attachment (welded or bolted))

NA NA NA NA NA  
(Floating (Mat'l Spec. No.)) (Dia., in.) (Nom. thk., in.) (Corr. Allow., in.) (Attachment)

13. Tubes: SA-213TP304L 1" 0.0910 THK avg 51 "U"  
(Mat'l Spec. No., Grade or Type) (O.D., in.) (Nom. thk., in. or gauge) (Number) [Type (Strait or U)]

Items 14-18 incl. to be completed for inner chambers of jacketed vessels or channels of heat exchangers.

14. Shell (a) No. of course(s): 1 (b) Overall length (ft & in.): 1'-10"

Course(s)			Material	Thickness		Long. Joint (Cat. A)			Circum. Joint (Cat. A, B, & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	18" O.D.	1'-10"	SA-312TP304L	1/4"	0"	S	NONE	0.85	1	FULL	1.00	N/A	N/A

15. Heads: (a) SA-240-304L (b) NA  
(Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.) (Mat'l Spec. No., Grade or Type) (H.T. - Time & Temp.)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	END	1 3/4"	0"	NA	NA	NA	NA	NA	22 5/8"	NA	NA	S	NONE	1.00
(b)														

If removable, bolts used (describe other fastening) (20)/7/8"Dia.&(20)/7/8"Dia.SA-193B7 Studs W/SA-194-2H Heavy Hex Nuts  
(Mat'l Spec. No., Grade, size, No.)

## FORM U-1 (Back)

16. MAWP 150 NA psi at max. temp. 200 NA °F Min. design metal temp. 10 °F at 150 psi.  
(internal) (external) (internal) (external)
17. Impact test NO, Exempt Per ucs-66, ucs-67 & ucs-68 NA °F.  
[Indicate yes or no and the component(s) impact tested]
18. Hydro., pneu., or comb. Test pres: 195 Proof test: NA
19. Nozzles, inspection, and safety valve openings:

Purpose (Inlet, Outlet, Drain, etc.)	No.	Diameter or Size	Flange Type	Material		Nozzle Thickness		Reinforcement Material	How Attached		Location (Insp. Open.)
				Nozzle	Flange	Nom.	Corr.		Nozzle	Flange	
INLET	1	6"-300#	RF-WN	SA-312TP304L	SA-182F304L	0.4320"	0"	NA	UW16.1	NONE, 0.7	NA
OUTLET	1	8"-300#	RF-WN	SA-312TP304L	SA-182F304L	0.5000"	0"	NA	UW16.1	NONE, 0.7	NA
INLET/OU	1	4"-150#	RF-WN	SA-312TP304L	SA-182F304L	0.2370"	0"	NA	UW16.1	NONE, 0.7	NA
INLET	1	4"-150#	RF-WN	SA-312TP304L	SA-182F304L	0.2370"	0"	NA	UW16.1	NONE, 0.7	NA
VENT	1	2"-150#	RF-WN	SA-312TP304L	SA-182F304L	0.2180"	0"	NA	UW16.1	NONE, 0.7	NA
DRAIN	1	2"-150#	RF-WN	SA-312TP304L	SA-182F304L	0.2180"	0"	NA	UW16.1	NONE, 0.7	NA
VENT	1	3/4"-6000#	CPLG	SA-182F304L	NA	6000#	0"	NA	UW16.1	NA	NA
DRAIN	1	3/4"-6000#	CPLG	SA-182F304L	NA	6000#	0"	NA	UW16.1	NA	NA

20. Supports: Skirt NO Lugs NONE Legs NONE Others (2) Saddles Attached Welded to Shell  
(Yes or no) (No.) (No.) (Describe) (Where and how)
21. Manufacturer's Partial Data Reports properly identified and signed by Commissioned Inspectors have been furnished for the following items of the report:  
 (List the name of part, item number, mfg's. name and identifying number)  
SHELL, CHANNEL & NOZZLES, ITEM #6, #7, #14, #15, #19, J & M WELDING, LLC., SN: 93057.3, 93057.4
22. Remarks: SERVICE: AGR FLASH GAS COOLER ITEM NO.: HX2065

STRAIGHT TUBE LENGTH = 16'-0"

## CERTIFICATE OF SHOP COMPLIANCE

We certify that the statements in this report are correct and that all details of design, material, construction, and workmanship of this vessel conform to the ASME Code for Pressure Cessels, Section VIII, Division 1.

U Certificate Authorization No 30112 Expires December 30, 21 12

Date 9/27/12 Name FABSCO SHELL & TUBE, LLC. Signed Anthony Cardwell  
(Manufacturer) (Representative)

## CERTIFICATE OF SHOP INSPECTION

I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of Oklahoma and employed by ONEBEACON AMERICA INSURANCE of LYNN, MA have inspected the pressure vessel described in this Manufacturer's Data Report on 9/19/12, and

state that, to the best of my knowledge and belief, the Manufacturer has constructed this pressure vessel in accordance with ASME Code, Section VIII, Division 1. By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date 9/27/12 Signed Mark Spang Commission NB 7935 A OK657  
(Authorized Inspector) (Nat'l Board incl. endorsements, State, Province, and No.)

## CERTIFICATE OF FIELD ASSEMBLY COMPLIANCE

We certify that the statements on this report are correct and that field assembly construction of all parts of this vessel conforms with the requirements of ASME Code, Section VIII, Division 1. U Certificate of Authorization No.

U Certificate of Authorization No \_\_\_\_\_ Expires \_\_\_\_\_, 20 \_\_\_\_\_

Date \_\_\_\_\_ Name \_\_\_\_\_ Signed \_\_\_\_\_  
(Assembler) (Representative)

## CERTIFICATE OF FIELD ASSEMBLY INSPECTION

I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of \_\_\_\_\_ and employed by \_\_\_\_\_

of \_\_\_\_\_, have compared the statements in this Manufacturer's Data Report with the described pressure vessel and state that parts referred to as data item \_\_\_\_\_ included in the certificate of shop inspection, have been inspected by me and to the best of my knowledge and belief, the Manufacturer has constructed and assembled this pressure vessel in accordance with ASME Code, Section VIII, Division 1. The described vessel was inspected and subjected to hydrostatic test of \_\_\_\_\_ psi.

By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date \_\_\_\_\_ Signed \_\_\_\_\_ Commission \_\_\_\_\_  
(Authorized Inspector) (Nat'l Board incl. endorsements, State, Province, and No.)

**A Part of a Pressure Vessel Fabricated by One Manufacturer for Another Manufacturer  
As Required by the Provisions of the ASME Boiler and Pressure Vessel Code Rules, Section VIII, Division 1**

[illegible]

# Form U-2 (Back)

15. Heads: (a) SA965-F304L (b) \_\_\_\_\_  
 (Material spec. number, grade or type) (H.T. - time & temp) (Material spec. number, grade or type) (H.T. - time & temp)

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff
(a)	(2) ENDS	3.9375"	0"						22.625"			S	NONE	100

If removable, bolts used (describe other fastenings) N/A  
 (Material spec. number, grade, size, number)

16. MAWP 150 psi at max temp. 200°F Min. design metal temp. 10°F at 150 psi  
 (Internal) (External) (Internal) (External)

17. Impact test NONE at test temperature of \_\_\_\_\_  
 [Indicate yes or no and the component(s) impact tested]

18. Hydro., pneu., or comb. test press. \_\_\_\_\_ Proof Test \_\_\_\_\_

19. Nozzles, inspection, and safety valve openings:

Purpose (Inlet, Outlet, Drain, etc)	No.	Diameter or Size	Flange Type	Material		Nozzle Thickness		Reinforcement Material	How Attached		Location (Insp. Open.)
				Nozzle	Flange	Nom.	Corr.		Nozzle	Flange	
INLET	1	6"-300#	RFWN	SA-312TP304L	SA182F304L	.4320"	0"		UW16.1(c)	TYPE 1	SHELL
OUTLET	1	8"-300#	RFWN	SA-312TP304L	SA182F304L	.5"	0"		UW16.1(c)	TYPE 1	SHELL
INLET	1	4"-150#	RFWN	SA-312TP304L	SA182F304L	.237"	0"		UW16.1(c)	TYPE 1	CHNL
OUTLET	1	4"-150#	RFWN	SA-312TP304L	SA182F304L	.237"	0"		UW16.1(c)	TYPE 1	CHNL
Vent/drain	2	2"-300#	RFLWN		SA182F304L		0"		UW16.1(c)	FLT	CHNL

20. Identification of Parts:

Name of Part	Quantity	Line No.	Mfr's Identification No.	Mfr's Drawing No.	CRN	National Board No.	Year Built
N/A							

21. Supports: Skirt NO Lugs - Legs - Others - Attached -  
 (Yes or No) (No.) (No.) (Describe) (Where and how)

22. Remarks AGR FLASH COOLER. PO# MPC17901. ITEM # HX1065/2065.  
SAFETY VALVE(S) SUPPLIED BY OTHERS PER UG-125(a).  
NO DESIGN BY J&M WELDING, LLC  
WPS QUALIFIED WITH AND WITHOUT PWHT.

CERTIFICATE OF SHOP/FIELD COMPLIANCE	
We certify that the statements made in this report are correct and that all details of material, construction and workmanship of this pressure vessel part conform to the ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1.	
U Certificate of Authorization Number <u>41,929</u>	Expires <u>07/18/2014</u>
Date <u>8/30/12</u> Name <u>J&amp;M Welding, LLC</u> Signed _____ (Manufacturer) (Representative)	
CERTIFICATE OF SHOP/FIELD INSPECTION	
I, the undersigned, holding a valid commission issued by the National Board of Boiler and Pressure Vessel Inspectors and/or the State or Province of <u>OKLAHOMA</u> and employed by <u>OneBeacon America Insurance Company</u> of <u>Lynn, Ma.</u>	
have inspected the pressure vessel part described in this Manufacturer's Data Report on <u>8/30/12</u>	
and state that, to the best of my knowledge and belief, the Manufacturer has constructed this pressure vessel part in accordance with ASME BOILER AND PRESSURE VESSEL CODE, Section VIII, Division 1. By signing this certificate neither the Inspector nor his/her employer makes any warranty, expressed or implied, concerning the pressure vessel part described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his/her employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.	
Date <u>8/30/12</u> Signed <u>Michael R. Pope</u> Commissions <u>NB9265-A</u> (Authorized Inspector) [National Board (incl endorsements) State, Province and number]	



[illegible]





6870 HIGHWAY 42 EAST

Certificate: 660875 9

Customer: 002830 021

Mail To:  
ROLLED ALLOYS  
CUSTOMER PICKUP  
289 MIFFLIN DRIVE  
WRIGHTSVILLE, PA 17368

Your Order: S05692

## METALLURGICAL TEST REPORT

NORTH AMERICAN STAINLESS  
6870 HIGHWAY 42 EAST  
GHENT, KY 41045

Ship To:  
ROLLED ALLOYS  
CUSTOMER PICKUP  
289 MIFFLIN DRIVE  
WRIGHTSVILLE, PA 17368

Date: 8/04/2011 Page: 1

Steel: 304/304L

Finish: HRAP

NAS Order: IN 0126036 03

Corrosion: ASTM A262/02aE;180Bend-OK

### PRODUCT DESCRIPTION:

STAINLESS STEEL PLATE, HRAP; UNS 30400/30403  
ASTM A240/10,A480/10,A666/10; ASME SA240/10,SA480/10,SA666/10  
CHEM ONLY ON FOLLOWING ASTM: A276/10,A479/10a,A484/10,A312/09  
CHEM ONLY ON FOLLOWING ASME: SA312/10,SA479/10  
AMS5511H/5513J XMRK; MIL-S-5059D AMD3(X CRN MEAS); MIL-S-4043B  
NACE MR0175/01, MR0103/07; QQS766D-A X MAG PERM  
MIN. SOLUTION ANNEAL TEMP 1900F, WATER QUENCHED  
SAE AMS QQ-S-763

### REMARKS:

Mat'l is Free of Mercury Contamination. No weld repairs.  
EN 10204:2004 3.1; QQS763F Cond A; RoHS Compliant  
Material is Free of Radioactive Contamination  
NAS Steel Making Process: EAF, AOD, & Cont. Casting  
Product Mfg.by a Quality Mgt.Sys. in Conf. w/ISO 9001  
\*Melted & Manufactured in the USA; Mat'l is DFARS Compliant

Product Id	Plate#	Skid #	Thickness	Width	Weight	-----Length-----	Mark	Pieces	Commodity Code
028PK1 BB	028PK1 BB		2.0213	60.0000	8,550	PLATE	240.00	13	1

### CHEMICAL ANALYSIS CM(Country of Melt) ES(Spain) US(United States) ZA(South Africa) JP(Japan)

HEAT	CM	C	CR	CU	MN	MO	N	NI	P	S
8PK1	US	.0225	18.2410	.3670	1.8445	.2800	.0772	8.0535	.0325	.0021
SI										
.2380										


Tony Cardwell

### MECHANICAL PROPERTIES

JAN 03 2012


Product Id#	Plate#	1 d o i c r	UTS KSI	.2% YS KSI	ELONG %-2"	Hard RB	A 262 Pr A	R of A %
028PK1 BB	028PK1 BB	F T	90.55	40.65	67.88	78.00	1.00	71.66

HEAT# 8PK1



TRAILER# 242100

**ROLLED ALLOYS QUALITY ASSURANCE**

APPROVED 

DATE 8-15-11

NAS hereby certifies that the analysis on this certification is correct and the material meets the specifications stated.

QC ENGINEER

ERIC HESS

8/04/2011

Kemper County MM233798



# METALLURGICAL TEST REPORT

6870 Highway 42 East  
Ghent, KY 41045-9615  
(502) 347-6000

Certificate: 667331 01  
Customer: 2830 023

Mail To:  
ROLLED ALLOYS  
6555 SOUTH 57TH WEST AVENUE  
TULSA, OK 74131

Ship To:  
ROLLED ALLOYS  
6555 SOUTH 57TH WEST AVENUE  
TULSA, OK 74131

Date: 8/26/2011 Page: 1

Steel: 304/304L

Finish: HRAP

Your Order: S05523

NAS Order: AN 0485050 02

Corrosion: ASTM A262/02aE; 180Bend-OK

## PRODUCT DESCRIPTION:

STAINLESS STEEL PLATE, HRAP; UNS 30400/30403  
ASTM A240/11, A480/11, A666/10; ASME SA240/10, SA480/10, SA666/10  
CHEM ONLY ON FOLLOWING ASTM: A276/10, A479/10a, A484/10, A312/09  
CHEM ONLY ON FOLLOWING ASME: SA312/10, SA479/10  
AMS5511H/5513J XMRK; MIL-S-5059D AMD3(X CRN MEAS); MIL-S-4043B  
NACE MR0175/01, MR0103/07; QQS766D-A X MAG PERM  
MIN. SOLUTION ANNEAL TEMP 1900F, WATER QUENCHED

## REMARKS:

Mat'l is Free of Mercury Contamination. No weld repairs.  
EN 10204:2004 3.1; QQS763F Cond A; RoHS Compliant  
Material is Free of Radioactive Contamination  
NAS Steel Making Process: EAF, AOD, & Cont. Casting  
Product Mfg. by a Quality Mgt. Sys. in Conf. w/ISO 9001  
\*Melted & Manufactured in the USA; Mat'l is DFARS Compliant

Product ID #	Coil #	Thickness	Width	Weight	-----Length-----	Mark	Pieces
039WT6 CA	* 039WT6 CA	2.0000	60.0000	8,225	PLATE	240.00	3

## CHEMICAL ANALYSIS CM(Country of Melt) ES(Spain) US(United States) ZA(South Africa) JP(Japan)

HEAT	CM	C	CO	CR	CU	MN	MO	N	NI	P
9WT6	US	.0215	.1455	18.0945	.4545	1.7385	.2865	.0784	8.0155	.0305
		S	SI							
		.0013	.2895							

Tony Cardwell

## MECHANICAL PROPERTIES

Product ID #	Coil #	UTS KSI	.2% YS KSI	ELONG %-2"	Hard RB	A 262 Pr A	R of A %
039WT6 CA	039WT6 CA	86.62	40.53	66.45	78.00	OK	71.08

HEAT# 9WT6  
  
TRACER# 243385

ROLLED ALLOYS QUALITY ASSURANCE  
APPROVED   
DATE 8/31/11

NAS hereby certifies that the analysis on this certification is correct and the material meets the specifications stated.

QC ENGINEER

ERIC HESS

8/26/2011



# OKLAHOMA FORGE, INC.

## Certification of Analysis and Tests

INVOICE NUMBER  
111-69864

Date: 12/13/11

Toll Free (866)909-5553  
Local (918)446-5553 FAX (918)446-4150

e-Mail ~ sales@oklahomaforge.com  
Web Site ~ www.oklahomaforge.com

P.O. Box 701500  
Tulsa, Oklahoma 74170-1500

Sold To	Your Purchase Order	Sales Order
FABSCO INC P. O. BOX 988 SAPULPA, OK 74066 USA	Number T11-10251-FG By TRACI STAMPS Date 10/24/2011	Number 111-53769 Shipped Via DELIVERY Ship Date 12/13/2011

Item #	PART DESCRIPTION				Quantity
	Your Part Number	Shop Order Number	Shape	Dimensions	
	Supplier	Heat Number	Material Specifications		
	Chemical Analysis				
	Physical Properties				
	Charpy Impact (ft. Lbs.)				

28	10279-1&2-27				111-53769-28			RING			44.937"OD x 34.500"ID x 5.125"TH										2
				ELECTRALLOY			50531			SA965-11 F304/304L SOL. ANNEAL											
C		Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other						
.016		1.79	.025	.002	.51	8.34	18.41		.22						N-.074						
Yield Lb/SqIn		Tensile Strength Lb/SqIn			Elong%in 2In		Red of Area%	Brinell	Rckwl	C Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3				
0		0			0		0														

29	10279-1&2-28				111-53769-29			RING		44.937"OD x 34.500"ID x 5.250"TH					2
					ELECTRALLOY			50531		SA965-11 F304/304L SOL. ANNEAL					
	C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other
	.016	1.79	.025	.002	.51	8.34	18.41		.22						N-.074
	Yield Lb/SqIn	Tensile Strength Lb/SqIn			Elong%in 2In		Red of Area%	Brinell	Rckwl C	Bend Test	Grain Size	Type	Temp	Average	#1 #2 #3
	0	0			0		0								

31	10279-7&8-2,3,7				111-53769-31			RING		22.625"OD x 17.500"ID x 3.937"TH							6
					ELECTRALLOY			50079		SA965-11 F304/304L SOL. ANNEAL							
	C	Mn	P	S	Si	Ni	Cr	Cu	Mo	V	Cb	Al	Pb	Ce	Other		
	.015	1.78	.028	.022	.54	8.21	18.36		.36						N-.076		
	Yield Lb/SqIn		Tensile Strength Lb/SqIn		Elong%in 2In		Red of Area%	Brinell	Rckwl	C Bend Test	Grain Size	Type	Temp	Average	#1	#2	#3
	41-100		86,000		60		77	152									

Tony Cardwell  
DEC 21 2011

Notes: UT REPORT ATTACHED

Page 1

I CERTIFY THAT THERE IS A TRUE COPY OF THE ORIGINAL TEST SHEET NOW ON FILE AT THE OFFICE OF OKLAHOMA FORGE, INC. AND THAT THIS MATERIAL WAS MELTED AND MANUFACTURED IN THE USA.

Anneal by heating to 1450-1550 degrees F for one hour/inch of thinnest section. Slow cool furnace to 700 degrees F then air cool by opening door.  
Normalize by heating to 1650 degrees F, Hold at 1650 degrees F for one hour/inch of thinnest section and air cool.  
Solution Anneal by heating to 1900-2050 degrees F and Water Quenching.

F040-9/2011

# OUTOKUMPU

\*\*\*\*\*  
\* CERTIFICATION \*  
\*\*\*\*\*

08/15/11  
HEAT 401095

ORDER 0025894/003

B00020711

SHIP TO:  
SHAW ALLOY PIPING PRODUCTS, INC.  
626 W. 41th Street

TULSA

OK 74107

----- YOUR ORDER & DATE -----  
46732 8/15/11 CUST# ALLPIP01

----- ITEM DESCRIPTION -----  
18" SCH10 TP304/TP304L A312

Country of Melt : SWE  
Country of Mfg. : USA

----- SPECIFICATIONS -----  
ASTM A312-11/ASME SA312-10  
WELDED  
MECHANICAL TESTING: ASTM A370  
YIELD STRENGTH: 0.2% OFFSET  
COLD FINISHED  
ELECTRIC FUSION WELDED (EFW)  
IM EINVERNEHMEN MIT DEM TUV  
SUD INDUSTRIE SERVICE GMBH  
WIRD AUF EINE GEGENZEICHNUNG  
LAUT SCHREIBEN VOM 05.AUGUST  
2008 DURCH TUV SUD INDUSTRIE  
SERVICE GMBH VERZICHTET  
Certificate in conformance with EN10204-95 3.1.B / EN10204-2004 3.1

2011 ADDENDA  
QUALITY SYSTEM CERTIFIED TO:  
PED 97/23/EC ANNEX 1, § 4.3  
BY TUV SUD INDUSTRIE SERVICE  
GMBH, NOTIFIED BODY 0036  
TOLERANCE: ASTM A999  
LONGITUDINAL WELDED  
RANDOM LENGTHS  
PLAIN ENDS  
DESCALED, PICKLED, AND  
PASSIVATED: ASTM A380

----- MECHANICAL & OTHER TESTS -----  
Tensile strength, KSI (MPa) 83.7 ( 577)  
Yield strength, KSI (MPa) 41.1 ( 283)  
Elongation % in 2 50.0

Guided Bend Face

OK

NACE MR0175/ISO 15156:09  
HRB-87

Bend Test  
Hydrotest PSI (MPa) 500 ( 3.45)  
NACE MR0103-10

Tony Cardwell

FEB 08 2012

----- CHEMICAL COMPOSITION -----  
Carbon (C) .024  
Phosphorus (P) .027  
Silicon (Si) .380  
Nickel (Ni) 8.110  
Manganese (Mn) 1.560  
Sulphur (S) .001  
Chromium (Cr) 18.230  
Nitrogen (N) .070

CAUTION: Processing that produces fumes and dust may cause respiratory disease: Especially alloys containing Chromium and Nickel.

CERTIFICATION: We certify that the analysis figures are correct as contained in the records of the company and that the material is free of known mercury, asbestos, and low melting alloy contamination. No weld repair unless otherwise noted.

HEAT TREATMENT: Solution annealed at a minimum of 1900 F and water quenched to below 800F within 3 minutes.

Knowingly & willfully falsifying or concealing a material act on this form, or making false, fictitious or fraudulent statements or representations herein could constitute a felony punishable under federal statutes.

We hereby certify that the chemical analysis and/or test results shown in this report are correct as contained in the records of the company.

Quality Representative  
*Martin James*

Outokumpu Stainless Pipe, Inc.

ADDRESS  
1101 North Main Street  
Wildwood, Florida 34785-9801

TELEPHONE  
(352) 748-1313

TELEFAX  
Administration: (352) 748-2751  
Purchasing: (352) 748-6576  
Production Control: (352) 748-0523  
Quality Assurance: (352) 748-0533

**MILL TEST REPORT**

9A

RM ID NUMBER

101185

SALES ORDER / RLS

002076 / 3

CERT ID / REV

00011284 / 01

SOLD TO

INDUSTRIAL PIPING SPECIALISTS  
606 N. 145TH EAST AVE  
TULSA, OK 74116  
USA

CUSTOMER P.O.  
TP180003

CUSTOMER PART

HEAT NO.  
742096

DESCRIPTION: 324001003120200

24" WELDED PIPE SCHED 10S TP304/TP304L (UNS# S30400/S30403) A312 DOUBLE RANDOM LENGTHS

CERTIFICATION REQUIREMENTSENGINEERING

ASTM A312-07 ASME SA312-07, No ADD.

HYDRO PRESSURE

300 PSI

HEAT TREAT

Annealed at 1900 Deg F. and water quenched to below 800 Deg. F. in less than 3 minutes.

Chemical

C	Cr	Mn	Ni	N	P	Si	S
.024	18.12	1.49	8.10	.070	.025	.39	.005

MechanicalTEST

Tensile PSI

UNITS

PSI

RESULTS

91000

Yield PSI

PSI

46000

Elong %

60

Hardness

RB83

TEST

TG Bend

RESULT

Pass

This test report represents the actual attributes of the items furnished and all items were manufactured, sampled, inspected, and tested in full compliance with applicable specifications and your purchase order.  
Certification is in accordance with EN10204:2004 type 3.1.  
Chemical content is % by weight. Mechanical test results are in English units (inches and pounds).  
No weld repairs have been performed on the base material.  
Hardness in accordance with NACE MR0175 and MR0103 and material is free of cold work to enhance mechanical properties.  
Pipe is Pickled and Passivated in accordance with ASTM A380.  
Bristol Metals has a Quality Management System in place that is in compliance with ISO 9001:2000.  
Bristol Metals does not add mercury during any manufacturing process.  
NAFTA country of origin: USA


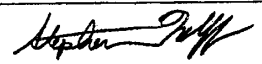
Tony Cardwell

FEB 08 2012

Rick Duncan - Quality Assurance Mgr



93

	<h1 style="margin: 0;">Certificate of Test</h1>		 <small>Stephen Wolff - Director, Corporate Quality Assurance</small>
	<b>Mill Information</b>		
<b>500 Green Street</b> <b>Washington, PA 15301</b>	<b>Cert Number</b> 0103733-00 <b>Sales Order</b> 50-032-356 <b>Cert Date</b> Apr-24-2012	<b>Name</b> ROLLED ALLOYS INC <b>PO</b> S07762 <b>PO Date</b> Mar-19-2012	

### Chemistry Testing

Allegheny Ludlum performs chemical analysis by the following techniques:  
 C, S by combustion/infrared; N, O, H by inert fusion/thermal conductivity;  
 Mn, P, Si, Cr, Ni, Mo, Cu, Cb, Co, V, by WDXRF; Pb, Bi, Ag by GFAA;  
 B by OES; Al and Ti (>=0.10%) by WDXRF, otherwise by OES.

837131 - Material was produced by EF melting with AOD refining.

### Mechanical Testing

		LOT 410386	
Condition:		ANNEALED	
Direction:		TRANSVERSE	
Temperature:		ROOM TEMP	
Spec:			
Test Limit	Units	Result	Loc
YIELD 0.2%	psi	45400.	TC
TENSILE	psi	92000.	TC
ELONGATION	%	58.	TC
RED OF AREA	%	79.	TC
HARDNESS	---	179. HBW	TC

### Mechanical Property Requirements


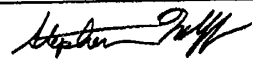
Condition:		ANNEALED	
Direction:		TRANSVERSE	
Temperature:		ROOM TEMP	
Spec:			
Test Limit	Units	Min	Max
YIELD 0.2%	psi	30000.	---
TENSILE	psi	75000.	100000.
ELONGATION	%	40.	---
RED OF AREA	%	---	---
HARDNESS	---	---	201. HBW

**Tony Cardwell**

JUL 27 2012



9B

 <b>ATI Allegheny Ludlum</b>  500 Green Street Washington, PA 15301	<b>Certificate of Test</b>		 <small>Stephen Wolff - Director, Corporate Quality Assurance</small>
	<b>Mill Information</b>		<b>Customer Information</b>
	Cert Number 0103733-00 Sales Order 50-032-356 Cert Date Apr-24-2012	Name ROLLED ALLOYS INC PO S07762 PO Date Mar-19-2012	

**Corrosion Testing**

Test ID	Result Name	Test Result	Wt Loss	Units	Visual Exam	Pit Depth	Units	Bend	Loc	Requirements
LOT 410386	ASTM-A262 PR E	STEP	---	---	---	---	---	---	TC	---

**Certification Statements**

Material does not have a continuous carbide network.

Material was solution annealed at 1900F (1038C) minimum for a time commensurate with thickness and rapidly cooled with air or water.

Allegheny Ludlum does not use mercury in the testing or production of its products.

Material is of USA melt and manufacture.

No welds/weld repairs performed.

Knowingly and willfully recording any false, fictitious or fraudulent statement or entry on this document may be punished as a felony under Federal statutes, including Federal Law, Title 18, Chapter 47.

DIN EN 10204:2005 3.1 Certificate

**General Statements**

TESTING WAS PERFORMED AT THE FOLLOWING LOCATIONS

BN = ATI-ALLEGHENY LUDLUM; 100 River Road; Brackenridge, PA 15014

TC = ATI-ALLEGHENY LUDLUM; 1300 Pacific Avenue; Natrona Heights, PA 15065

WARNING: Processing that makes fumes, dust, or solutions may cause lung disease. please see MSDS for further information which has been supplied to your Purchasing Department. For an additional copy, please refer to our web site at [www.alleghenyludlum.com/pages/assistance/MSDS.asp](http://www.alleghenyludlum.com/pages/assistance/MSDS.asp).

For access to online Certifications of Test, please register at [www.alcextra.com](http://www.alcextra.com).

The above is a true copy of the data on file. The material and test results conform to the sales contract and specification(s) as set forth in ATI Allegheny Ludlum's order acknowledgement. This certificate of Test may not be reproduced except in full without the written authorization of the company.

ATI Allegheny Ludlum's web site contains a listing of current quality and company accreditations, materials produced, general technical and contact information. Please visit us at [www.alleghenyludlum.com](http://www.alleghenyludlum.com).

TRACER # 262240



Tank Head Manufacturing Complex  
10703 Sheldon Road  
Houston, TX 77044

**UNI-FORM  
COMPONENTS CO.**

(281) 456-9310  
(800) 231-3272 toll-free  
(281) 456-0245 fax

18

**MATERIAL CERTIFICATION**

Packlist No. <b>PL-55850</b>		Customer ID <b>FASHTU</b>
Customer PO No. <b>T11-10251-HD</b>	UCC Job No. <b>100421</b>	Date Cert. Originated <b>12/16/2011</b>

**Sold To:**  
FABSCO SHELL & TUBE INC  
2410 INDUSTRIAL RD

**Ship To:**  
FABSCO SHELL & TUBE INC  
2410 INDUSTRIAL RD

SAPULPA OK 74066

SAPULPA OK 74066

Ln	Order Qty	Ship Qty	B/O Qty	Part Description	
1	2	2	0	HEAD 2:1 ELLIP, 18 OD X 3/8 SA240-304L Unit of Measure: EA	3/16" MIN

Code: 2C890 HEAT/SLAB: 70756/31A

GRADE: SA240-304L

Size: 3/8 X 25

MILL: ATI

**Tony Cardwell**  
DEC 20 2011

**MILL TEST REPORTS ATTACHED**

The chemical and physical properties as indicated on the attached report are the results of the Mill Tests of the raw material used in the manufacture of these products and are certified to meet only the minimum requirements of the ASME and/or ASTM specifications for the material.

1. We hereby certify that these heads were hot formed at 1950 degrees F. and air cooled in strict accordance with all applicable specifications.
2. We hereby certify that these heads comply with tolerances of UG-81 of ASME Section VIII, Div. 1.

WE HEREBY CERTIFY THAT THIS REPORT COVERING THE ABOVE AND ATTACHED INFORMATION IS TRUE AND  
CORRECT AS SHOWN AND CONTAINED IN OUR RECORDS.



Quality Control



Allegheny Technologies  
Ship SAMUEL SPECIALTY METALS  
5022 ASHLEY COURT  
HOUSTON TX

500 Green Street  
Washington, PA 15301

CERTIFIED MATERIAL  
TEST REPORT

CERTIFICATE OF CONFORMANCE  
SAMUEL SON & CO INC  
5022 ASHLEY COURT  
HOUSTON TX

77041

77041

OUR ORDER NO.  
YOUR ORDER NO.  
MEMO NO.  
DATE  
SALESMAN NO.

151417831  
4973855  
304260-00 DUAL CERT  
11/05/2011  
375

Tracy McFarland  
Tracy McFarland - Director, Corporate Quality Assurance

AL 304/304L HRAP  
ASTM A240 10B ASME SA240 10  
UNS S30400;  
UNS S30403

Matl ID	Slip	Sid	Lot No	Size(Inches)	Pcs	Weight(lb)
05B 70483 AAB			241669	.3750 x 96.0000 x 240.0000	1	2620
From Slip 03029 AAB 5315032			242231	.3750 x 96.0000 x 240.0000	1	2620
01A 70752 ABB						
From Slip 05051 ABB 5315032			242748	.3750 x 96.0000 x 240.0000	1	2620
01B 71008 ABA						
From Slip 07086 ABA 5315032			242748	.3750 x 96.0000 x 240.0000	1	2620
01B 71009 ABB			242344	.3750 x 96.0000 x 240.0000	1	2620
31A 70754 ABB						
From Slip 05112 ABB 5315032			242344	.3750 x 96.0000 x 240.0000	1	2620
31A 70755 ABA			242344	.3750 x 96.0000 x 240.0000	1	2620
31A 70756 AAB			242344	.3750 x 96.0000 x 240.0000	1	2620
31A 70757 AAA			243194	.2500 x 96.0000 x 240.0000	1	1773
04 71122 AD						
From Slip 09275 AD 5315017			243194	.2500 x 96.0000 x 240.0000	1	1773
04 71123 AB			243194	.2500 x 96.0000 x 240.0000	1	1773
04 71124 AA			243194	.2500 x 96.0000 x 240.0000	1	1773
04 71125 AC			243442	.2500 x 96.0000 x 240.0000	1	1773
06 71279 AA						
From Slip 10127 AA 5315017			243342	.2500 x 96.0000 x 240.0000	1	1773
08 71332 AB						
From Slip 10135 AB 5315017						

PAGE 1 CONTINUE ON PAGE 2

THIS CERTIFICATE OF TEST SHALL NOT BE REPRODUCED IN FULL WITHOUT THE WRITTEN APPROVAL OF THE COMPANY. THE RECORDING OF FALSE, FICTITIOUS, OR FRAUDULENT STATEMENTS OR ENTRIES ON THE CERTIFICATE MAY BE PUNISHED AS A FELONY UNDER FEDERAL LAW. TESTING WAS PERFORMED AT ALC NADCAP AND ISO/IEC 17025 APPROVED LABORATORIES LOCATED AT NATRONA HEIGHTS, BRACKENRIDGE, LATROBE, MIDLAND, AND LEECHBURG, PA FACILITIES OR A NADCAP AN ISO/IEC 17025 ACCREDITED LABORATORY. EN 10204 - 3.1 ALLEGHENY LUDLUM IS APPROVED AS MANUFACTURER ACCORDING TO AD-MERKBLATT W0170 100 AND THE PRESSURE EQUIPMENT DIRECTIVE PED 97/23/EC.

ALLEGHENY LUDLUM PERFORMS CHEMICAL ANALYSIS BY THE FOLLOWING TECHNIQUES: C, S BY COMBUSTION/INFRARED; N, O, H BY INERT FUSION/THERMAL CONDUCTIVITY; MN, P, SI, CR, NI, MO, CU, CB, CO, V, BY WDXRF; B BY OES; AL AND TI (>0.1%) BY WDXRF, OTHERWISE BY OES; PB, BI, AG BY GFAA  
CERTIFICATE OF TEST STATEMENT & CHEMISTRY STATEMENT EXCEPT AS OTHERWISE NOTED, THIS MATERIAL HAS BEEN MANUFACTURED AND TESTED IN ACCORDANCE WITH THE LISTED SPECIFICATIONS AND RESULTS CONFORM TO THE SPECIFICATION AND ORDER REQUIREMENT

Tony Cardwell

DEC 20 2011

253525  
253526  
253527  
253528  
253529  
253530  
253531-  
253532  
253539  
253520  
253521  
253522  
253523  
253524

Kemper County MM233798



NORTH AMERICAN  
STAINLESS

# METALLURGICAL TEST REPORT

6870 Highway 42 East  
Ghent, KY 41045-9615  
(502) 347-6000

Certificate: 667331 01  
Customer: 2830 023

Mail To:  
ROLLED ALLOYS  
6555 SOUTH 57TH WEST AVENUE  
TULSA, OK 74131

Ship To:  
ROLLED ALLOYS  
6555 SOUTH 57TH WEST AVENUE  
TULSA, OK 74131

Date: 8/26/2011 Page: 1

Steel: 304/304L

Finish: HRAP

Your Order: S05523

NAS Order: AN 0485050 02

Corrosion: ASTM A262/02aZ; 180Bend-OK

## PRODUCT DESCRIPTION:

STAINLESS STEEL PLATE, HRAP; UNS 30400/30403  
ASTM A240/11, A480/11, A666/10; ASME SA240/10, SA480/10, SA666/10  
CHEM ONLY ON FOLLOWING ASTM: A276/10, A479/10a, A484/10, A312/09  
CHEM ONLY ON FOLLOWING ASME: SA312/10, SA479/10  
AMS5511H/5513J XMRK; MIL-S-5059D AMD3(X CRN MEAS); MIL-S-4043B  
NACE MR0175/01, MR0103/07; QQS766D-A X MAG PERM  
MIN. SOLUTION ANNEAL TEMP 1900F, WATER QUENCHED

## REMARKS:

Mat'l is Free of Mercury Contamination. No weld repairs.  
EN 10204:2004 3.1; QQS763F Cond A; RoHS Compliant  
Material is Free of Radioactive Contamination  
NAS Steel Making Process: EAF, AOD, & Cont. Casting  
Product Mfg. by a Quality Mgt. Sys. in Conf. w/ISO 9001  
\*Melted & Manufactured in the USA; Mat'l is DFARS Compliant

TAG - 10279-7+8 #19

Product ID #	Coil #	Thickness	Width	Weight	-----Length-----	Mark	Pieces
039WT6 CA	* 039WT6 CA	2.0000	60.0000	8,225	PLATE	240.00	3

## CHEMICAL ANALYSIS CM(Country of Melt) ES(Spain) US(United States) ZA(South Africa) JP(Japan)

HEAT	CM	C	CO	CR	CU	MN	MO	N	NI	P
9WT6	US	.0215	.1455	18.0945	.4545	1.7385	.2865	.0784	8.0155	.0305
		S	SI							
		.0013	.2895							

Tony Cardwell

JAN 12 2012

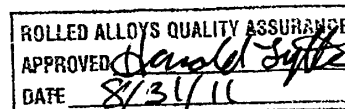
## MECHANICAL PROPERTIES

Product ID #	Coil #	UTS KSI	.2% YS KSI	ELONG %-2"	Hard RB	A 262 Pr A	R of A %
039WT6 CA	039WT6 CA	F T	86.62	40.53	66.45	78.00	OK

HEAT# SWTS



TRACER# 243385



NAS hereby certifies that the analysis on this certification is correct and the material meets the specifications stated.

QC ENGINEER

ERIC HESS

8/26/2011

NAS

## METALLURGICAL TEST REPORT

NORTH AMERICAN STAINLESS  
6870 HIGHWAY 42 EAST  
GHENT, KY 41045

6870 HIGHWAY 42 EAST

Certificate: 660875 9  
Customer: 002830 021  
Mail To:  
ROLLED ALLOYS  
CUSTOMER PICKUP  
289 MIFFLIN DRIVE  
WRIGHTSVILLE, PA 17368Ship To:  
ROLLED ALLOYS  
CUSTOMER PICKUP  
289 MIFFLIN DRIVE  
WRIGHTSVILLE, PA 17368

Date: 8/04/2011 Page: 1

Steel: 304/304L

Finish: HRAP

Your Order: S05692

NAS Order: IN 0126036 03

Corrosion: ASTM A262/02aE; 180Bend-OK

## PRODUCT DESCRIPTION:

STAINLESS STEEL PLATE, HRAP; UNS 30400/30403  
ASTM A240/10, A480/10, A666/10; ASME SA240/10, SA480/10, SA666/10  
CHEM ONLY ON FOLLOWING ASTM: A276/10, A479/10a, A484/10, A312/09  
CHEM ONLY ON FOLLOWING ASME: SA312/10, SA479/10  
AMS5511H/5513J XMRK; MIL-S-5059D AMD3(X CRN MEAS); MIL-S-4043B  
NACE MR0175/01, MR0103/07; QQS766D-A X MAG PERM  
MIN. SOLUTION ANNEAL TEMP 1900F, WATER QUENCHED  
SAE AMS QQ-S-763

## REMARKS:

Mat'l is Free of Mercury Contamination. No weld repairs.  
EN 10204:2004 3.1; QQS763F Cond A; RoHS Compliant  
Material is Free of Radioactive Contamination  
NAS Steel Making Process: EAF, AOD, & Cont. Casting  
Product Mfg. by a Quality Mgt. Sys. in Conf. w/ISO 9001  
\*Melted & Manufactured in the USA; Mat'l is DFARS Compliant

TAG-10279-7+8 #1

Product Id	Plate#	Skid #	Thickness	Width	Weight	-----Length-----	Mark	Pieces	Commodity Code
028PK1 BB	028PK1 BB		2.0213	60.0000	8,550	PLATE	240.00	13	1

## CHEMICAL ANALYSIS CM(Country of Melt) ES(Spain) US(United States) ZA(South Africa) JP(Japan)

HEAT	CM	C	CR	CU	MN	MO	N	NI	P	S	
8PK1	US	.0225	18.2410	.3670	1.8445	.2800	.0772	8.0535	.0325	.0021	Tony Cardwell
	SI										JAN 12 2012
		.2380									

## MECHANICAL PROPERTIES

Product Id#	Plate#	1 d o i c r	UTS KSI	.2% YS KSI	ELONG %-2"	Hard RB	A 262 Pr A	R of A %
028PK1 BB	028PK1 BB	F T	90.55	40.65	67.88	78.00	1.00	71.66
HEAT# 8PK1								
TRACER# 242100								

ROLLED ALLOYS QUALITY ASSURANCE  
APPROVED *[Signature]*  
DATE 8-15-11

NAS hereby certifies that the analysis on this certification is correct and meets the specifications stated.

QC ENGINEER

ERIC HESS

8/0 11

Kemper County MM233798



21

## MANUFACTURER'S CERTIFICATE OF COMPLIANCE

CUSTOMER: Fasbco

CUSTOMER ORDER NO: T11-10279-T

CUSTOMER JOB NO: 10279-7-21

7796-10279-7+8 #21

OUR ORDER NO: 2296B

PRODUCT DESCRIPTION: 1" OD X .091" AW X SA213/304

DATE: 1/25/2012

ALL U-BENDS WERE SOLUTION ANNELED IN ACCORDANCE WITH SA-688.

**Tony Cardwell**

FEB 08 2012

WE CERTIFY THAT THIS MATERIAL HAS BEEN MANUFACTURED, INSPECTED, AND TESTED IN ACCORDANCE WITH THE SPECIFICATIONS TO WHICH IT WAS ORDERED, AND THAT THE ABOVE INFORMATION IS CORRECT AS CONTAINED IN THE RECORDS OF THIS CORPORATION.

---

12225 FM 529 Houston, Texas 77041 Tel: 281-531-8088 Fax: 281-531-8099 [www.httproducts.com](http://www.httproducts.com)



No/Nr/N° (A03)

240447-1&2

Salzgitter Mannesmann Stainless Tubes USA, Inc

(A01)

12050 West Little York - Houston, TX 77041 - USA

www.smst-tubes.com

INSPECTION CERTIFICATE  
Abnahmeprüfzeugnis  
Certificat de réception

EN 10204: 2004 TYPE 3.1

(A02)

Page/Seite

1/6

Salzgitter Mannesmann Stainless Tubes USA, Inc - 12050 West Little York - Houston, TX 77041 - USA

HEAT TRANSFER TUBULAR PRODUCTS

12225 FM 529

77041 HOUSTON

USA

Purchaser/Besteller/Acheteur

Customer order no./Kunde Auftragsnr./N° Commande client (A07)

2296B

SMST-Tubes order no./Auftragsnr./N° Commande (A08)

0000240447

SMST-Tubes item Part number/Teilenummer/N° d'article (A09)

0000240447-000001 58 PCS X 33'-0" Row A, B, C, D

0000240447-000002 48 PCS X 34'-0" Row E, F, G, H & Spares

### Product Description/Produkt Beschreibung/Description du produit (B01) (B02) (B04)

Seamless Stainless Steel Cold Finished Tubes pickled Plain Ends Square Cut Deburred  
Kaltgefertigte, nahtlose Edelstahlrohre pickled Enden Glatt Abgeschnitten  
Tubes en Acier Inox Sans Soudure Finis à Froid pickled Coupés d'angle, lisses, ébavuré

Specifications/Spezifikationen/Spécifications ASME SA 213 08 a / NACE MR0103 2007 / NACE MR0175/ISO 15156-3 2009

Grade/Werkstoff/Nuance TP 304 / TP 304L

Tolerances/Toleranzen/Tolérances SA 213 / A 1016 AW

Marking of the product/Kennzeichnung des Produktes/Marquage du produit (B06-D01)

DMV 1.000 X .091AW TP 304/ TP 304L SA-213 HT (XXX) SML CF 240447-(X) ET PHU(XXX) QL(XXX) USA

Quantity/Menge/Quantité						Dimensions		Abmessungen	
Heat no Schmelze Nr. N° de Coulée	Quality lot Qualitätslos Lot qualité	SMST item	Pieces Stück Pièces (B08)	Total weight Gesamtgewicht Masse totale (B13)	Total length Gesamtlänge Longueur totale	OD (B09)	WT (B10)	Tube length Rohrlänge Longueur tube min (B11) max	
QL345	QL50000823	000001	58	1741 Lbs	1,914.00 Ft	1.000 "	0.091 "	33 Ft	33 Ft
QL347	QL50000829	000002	48	1491 Lbs	1,632.00 Ft	1.000 "	0.091 "	34 Ft	34 Ft
Total			106	3232 Lbs	3,546.00 Ft				

### Chemical Analysis / Chemische Zusammensetzung / Analyse chimique (C71 - C92)

Heat no/Schmelzen Nr./N° de coulée QL345

Melting Process/Erschmelzungsart/Élaboration (C70) E+AOD or VOD

Country of origin / Herstellungsland/Pays d'origine Germany

### Heat Analysis / Analyse de coulée

	C	Si	Mn	P	S	Cr	Ni
Min	0.00	0.00	0.00	0.00	0.00	18.00	8.00
Max	0.035	1.00	2.00	0.045	0.0300	20.00	11.00
	0.017	0.35	1.91	0.023	0.008	18.13	10.10

Tony Cardwell

FEB 08 2012

Quality Lot : QL50000823

Product Analysis / Analyse du produit

ASME SA 213

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Salzgitter Mannesmann Stainless Tubes USA, Inc





No/Nr/N° (A03)

240447-1&2

Salzgitter Mannesmann Stainless Tubes USA, Inc

12050 West Little York - Houston, TX 77041 - USA

[www.smsst-tubes.com](http://www.smsst-tubes.com)

INSPECTION CERTIFICATE  
Abnahmeprüfzeugnis  
Certificat de réception

EN 10204: 2004 TYPE 3.1

(A01)

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	C	Si	Mn	P	S	Cr	Ni
Min	0.000	0.00	0.00	0.000	0.0000	18.00	8.00
Max	0.035	1.00	2.00	0.045	0.0300	20.00	11.00
50003258	0.020	0,36	1,96	0,029	0,0079	18,12	10,28

Chemical Analysis / Chemische Zusammensetzung / Analyse chimique (C71 - C92)

Heat no/Schmelzen Nr./N° de coulée QL347

Melting Process/Erschmelzungsart/Élaboration (C70) E+AOD or VOD

Country of origin / Herstellungsländ/Pays d'origine Germany

Heat Analysis / Analyse de coulée

	C	Si	Mn	P	S	Cr	Ni
Min	0.00	0.00	0.00	0.00	0.00	18.00	8.00
Max	0.035	1.00	2.00	0.045	0.0300	20.00	11.00
	0,019	0,25	1,91	0,023	0,010	18,28	10,15

Quality Lot : QL50000829

Product Analysis / Analyse du produit

	C	Si	Mn	P	S	Cr	Ni
Min	0.000	0.00	0.00	0.000	0.0000	18.00	8.00
Max	0.035	1.00	2.00	0.045	0.0300	20.00	11.00
50003280	0.023	0,27	1,95	0,028	0,0090	18,25	10,29

ASME SA 213

Tony Cardwell

FEB 08 2012

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Salzgitter Mannesmann Stainless Tubes USA, Inc



No/Nr/N° (A03)

**240447-1&2**

**INSPECTION CERTIFICATE  
Abnahmeprüfzeugnis  
Certificat de réception**

EN 10204: 2004 TYPE 3.1

(A02)

**Mechanical testing**

**Quality Lot : QL50000823**

**Tensile test at room temperature/Zugversuch bei Raumtemperatur/Essai de traction à température ambiante (C10)**

**ASTM A 370**

Test no Proben Nr. N° d'échantillon (C00)	Direction Probenricht- ung Direction (C02)	Yield strength/Dehngrenze/Limite d'élasticité (C11)			Tensile strength / Zugfestigkeit / Résistance à la traction (C12)	Elongation/Bruchdehnung/Allongement (C13)					Reduction of area Z%
		0.2 %	0.5 %	1 %		2 "	50 mm	5D	5,65 √So	4D	
		ksi	/	/	ksi	%	/	/	/	/	/
	Min	30	/	/	75	35	/	/	/	/	/
	Max	/	/	/	/	/	/	/	/	/	/
50003258	LONGITUDINAL	38,90	/	/	90,00	53,90	/	/	/	/	/
50003287	LONGITUDINAL	36,50	/	/	91,10	55,91	/	/	/	/	/

**Hardness test/Härteprüfung/Essai de dureté (C30)**

**ASTM A 370 NACE MR0175/ISO 15156-3 NACE MR0103**

Test no Proben Nr. N° d'échantillon	HB			HRC			HV			HRB			HR 15-T			HR 30-T		
	min	max	avg	min	max	avg	min	max	avg	min	max	avg	min	max	avg	min	max	avg
Min	/	/	/	/	/	/	/	/	/	/	/	/	/	/	/	/	/	/
Max	/	/	/	22	22	22	/	/	/	90	90	90	90	/	/	90	/	/
	min	max	avg	min	max	avg	min	max	avg	min	max	avg	min	max	avg	min	max	avg
50003257	0,0	0,0	0	0,0	0,0	0,00	0,0	0,0	0,00	65,0	68,0	66,50	/	/	/	/	/	/
50003258	0,0	0,0	0	0,0	0,0	0,00	0,0	0,0	0,00	68,0	70,0	69,00	/	/	/	/	/	/

**Flattening test/Faltversuch/Essai d'aplatissement (C50)**

**ASTM A 1016**

Test Number	Result
50003258	OK
50003258	OK

**Tony Cardwell**

**FEB 08 2012**

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**Salzgitter Mannesmann Stainless Tubes USA, Inc**







No/Nr/N° (A03)

240447-1&2

Salzgitter Mannesmann Stainless Tubes USA, Inc

(A01)

12050 West Little York - Houston, TX 77041 - USA

[www.srst-tubes.com](http://www.srst-tubes.com)

INSPECTION CERTIFICATE  
Abnahmeprüfzeugnis  
Certificat de réception

EN 10204: 2004 TYPE 3.1

(A02)

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Flaring test/Aufweiterversuch/Essai d'évasement (C51)

ASTM A 1016

Test Number	Result
50003258	OK
50003258	OK

Tony Cardwell

FEB 08 2012

Quality Lot : QL50000829

Tensile test at room temperature/Zugversuch bei Raumtemperatur/Essai de traction à température ambiante (C10)

ASTM A 370

Test no Proben Nr. N° d'échantillon (C00)	Direction Probenricht- ung Direction (C02)	Yield strength/Dehngrenze/Limite d'élasticité (C11)			Tensile strength / Zugfestigkeit / Résistance à la traction (C12)	Elongation/Bruchdehnung/Allongement (C13)					Reduction of area Z%
		0.2 %	0.5 %	1 %		2 "	50 mm	5D	5,65 √So	4D	
		ksi	/	/		%	/	/	/	/	
	Min	30	/	/	75	35	/	/	/	/	/
	Max	/	/	/	/	/	/	/	/	/	/
50003280	LONGITUDINAL	40,90	/	/	94,10	51,30	/	/	/	/	/
50003288	LONGITUDINAL	45,80	/	/	95,80	49,90	/	/	/	/	/

Hardness test/Härteprüfung/Essai de dureté (C30)

ASTM A 370 NACE MR0175/ISO 15156-3 NACE MR0103

Test no Proben Nr. N° d'échantillon	HB			HRC			HV			HRB			HR 15-T			HR 30-T		
	Load			Load			Load			Load			Load			Load		
	min	max	avg	min	max	avg	min	max	avg	min	max	avg	min	max	avg	min	max	avg
50003279	0,0	0,0	0	0,0	0,0	0,00	0,0	0,0	0,00	67,0	68,0	67,50	/	/	/	/	/	/
50003280	0,0	0,0	0	0,0	0,0	0,00	0,0	0,0	0,00	69,0	71,0	70,00	/	/	/	/	/	/

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Salzgitter Mannesmann Stainless Tubes USA, Inc





No/Nr/N° (A03)

240447-1&2

Salzgitter Mannesmann Stainless Tubes USA, Inc (A01)

12050 West Little York - Houston, TX 77041 - USA

[www.sms-tubes.com](http://www.sms-tubes.com)

INSPECTION CERTIFICATE  
Abnahmeprüfzeugnis  
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EN 10204: 2004 TYPE 3.1

(A02)

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Page/Seite

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Flattening test/Faltversuch/Essai d'aplatissement (C50)

ASTM A 1016

Test Number	Result
50003280	OK
50003280	OK

Flaring test/Aufweiteversuch/Essai d'évasement (C51)

ASTM A 1016

Test Number	Result
50003280	OK
50003280	OK

Tony Cardwell

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Salzgitter Mannesmann Stainless Tubes USA, Inc



**Other Tests and Declarations / Andere Prüfungen und Prüffeststellungen / Autres tests et déclarations**

**QL50000823**

**QL50000829**

Heat treatment / Wärmebehandlung/Traitement thermique, 1950°F and water quenched

Eddy Current tested in accordance with ASTM E-426/ 12-1/2% Notch /Result - Satisfactory

No Weld repair / Keine Reparaturschweißung / Aucune réparation par soudure

The material is conforming to directive 2000/53/EC, 2002/95/EC and CD 2005/618/EC. / Das Material entspricht den Anforderungen der Richtlinien 2000/53/EC, 2002/95/EC und CD 2005/618/EC. / Le matériau est conforme aux directives 2000/53/EC, 2002/95/EC et CD 2005/618/EC.

Tubes are free from mercury contamination and from radioactive contamination / Die Rohre sind frei von Quecksilberverunreinigungen und frei vom radioaktiver Verunreinigung / Les tubes sont exempts de contamination par le mercure et de contamination radioactive

Confirmation with reference to Pressure Equipment Directive 97/23/EC:  
The works operates a quality management system that has undergone a specific assessment for materials for pressure equipment and is certified by a competent body (ABS QE Cert. No. 30788)  
Bestätigung in Bezug auf Druckgeräterichtlinie 97/23/EC:  
Das Werk wendet ein Qualitätsmanagementsystem an, das in Bezug auf Werkstoffe für Druckgeräte einer spezifischen Bewertung unterzogen wurde und von einer zuständigen Stelle (ABS QE Cert. No. 30788) zertifiziert ist.  
Confirmation concernant la Directive Equipements sous Pression 97/23/EC : L'usine applique un système de management de la qualité qui a fait l'objet d'une évaluation spécifique pour les matériaux pour équipements sous pression et qui est certifié par un organisme compétent (ABS QE Cert. No. 30788)

SOLUTION ANNEALED, PICKLED AND PASSIVATED.

SMST certify that the delivered products comply with the requirements stipulated in the order. / Die Erzeugnisse wurden bestellungsgemäß geprüft und für in Ordnung befunden. / SMST-Tubes atteste que les produits livrés sont conformes aux stipulations de la commande.

Validation by manufacturer's representative / Validierung durch Vertreter des Herstellers

Mill's Inspector  
Werksachverständiger  
Le contrôleur usine

Raynee Rangel - Quality Technical Analyst

Date of edition  
Ausgabedatum 06/07/2011  
Date d'édition

*Raynee Rangel*

**Tony Cardwell**

**FEB 08 2012**

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# BEBITZ



EN ISO 9001 : 2008

Zertifikat-Registrier-Nr.  
44 100 071888Flanschenwerk Bebitz GmbH - Lebendörfer Str. 1 - 06420 Bebitz / GERMANY (A01)  
Tel. +49 34691 40 0 - Fax +49 3469140 329 - Email: flanges@bebitz.de

Bereich Flansche

Abnahmeprüfzeugnis 3.1 / Inspection certificate 3.1 / Certificat de reception 3.1

nach (A02) / acc. DIN EN 10204 : 2005 Nr. (A03) / No.11-02624 Datum (Z02) / Date 31.01.2011

Zeichen des Herstellers (A04)  
Manufacturer's brandStempel des Abnehmers (Z03)  
Stamp of the testing engineer

Überprüft als Hersteller nach AD-Merkblatt W0 / TRD100 und VdTÜV Werkstoffblätter 350/3, 354, 399. / Approved as manufacturer acc. AD-Merkblatt W0 / TRD100 and VdTÜV Material Sheets 350/3, 354, 399.

Zertifiziert nach DGR 97/23/EG durch TÜV CERT Zertifizierungsst. für Druckgeräte der TÜV NORD GmbH Co KG, Benannte Stelle Nr. 0045. / Certified acc. to PED 97/23/EC, certifying body for pressure equipment TÜV NORD GmbH Co KG, registration no. 0045.

Besteller Industrial Piping Specialists, Inc.  
(A06) P.O. Box 581270  
Customer TULSA, OK 74158-1270Bestell-Nr (A07) / Order-No. TP276692  
Auftrag (A08) / Order 3021809  
Lieferschein / Delivery No. 3130566  
Position (B07.1) / Item 055RBStück (B08) Bezeichnung (B01; B09 - B11)  
Quantity Product

6 6 inch Welding Neck Flange 300 lbs S80s raised face

Schmelzprüfbes (B07)  
Heat no./Test no.  
82564/1Material F 304/F 304L - ASTM A 182M-09a / ASME SA-182M  
(B02; B05) austenitis. > 1040 °C and quenched in water

Anford. ASME B 16.5-2009

(B03) NACE MR0175-2002 / MR0103-2007 - DIN EN ISO 15156-3 : 2005

Requirements 2010 ASME Boiler & Pressure Vessel Code - Section II Part A  
PMI-tested 100%, IC-test acc. to DIN EN ISO 3651-2 Test A / ASTM A 262 practice E

Schmelzenanalyse (C71 - C92) / Ladle analysis

Erschmelzung (C70) / Melting process E

C	Si	Mn	P	S	Cr	Ni	Ti	Mo	Nb	V	N	Cu	Al	PREN
0,023	0,47	1,75	0,036	0,005	18,34	8,04					0,079			19,60

$$CEV = C + \frac{Mn}{6} + \frac{V + Mo + Cr}{5} + \frac{Ni + Cu}{15} \quad PREN = Cr + 3,3 \times Mo + 16 \times N$$

Mechanische Prüfungen / Mechanical tests

Probenlage (C02): q/t / Position of Specimen: q/t Zugversuch / Tensile Test - ASTM A 370						KBV / Charpy Impact Test DIN EN 10045-1			Härte (C32) Hardness ISO 6506 HBW 2,5/187,5	
Temp (C03) °C	Streckgr. (C11) / Yield strength 0,2% MPa (N/mm²)		Zugfestigk. (C12) Tensile strength MPa (N/mm²)		Dehng. (C13) Elongation %	Einschnürr. Red. of area %	Temp. (C03) °C	Einzelwerte (C42) Single values J	MW (C43) Average J	
20	min 205 max		515		30,0	50	20			
20	297		324		577	63,3	76	138 150 144	144,0	168

Tony Cardwell

TAG-10279-748 #71

FEB 08 2012

Besichtigung und Ausmessung ohne Beanstandungen. (D01) / Results of inspection and dimension without objections.  
Die gestellten Anforderungen sind erfüllt. (Z01) / The product meets the requirements.

R. Sparing,

Abnahmebeauftragter des Herstellers (Z02)  
Inspection Representative of the ManufacturerAPZ wurde mit EDV erstellt und ist ohne Unterschrift gültig. / This  
MTR was electronically generated and is valid without a signature



SSU680S304/LSL, NISHINO-CHO, HIGASHI-MUKOJIMA, AMAGASAKI, JAPAN

71A

# INSPECTION CERTIFICATE

CERTIFICATE NO. :0YU5273

PAGE : 1/1 DATE : 2011-08-08

CUSTOMER :MARUBENI-ITOCHU TUBULARS AMERICA INC.  
 ORDER NO. :HRO30934  
 SHIPPER :K AND I TUBULAR CORP. 194 JH2 LZD66108 1P13J001601  
 COMMODITY :HOT FINISHED STAINLESS PIPE  
 STANDARD :ASTM A312-09 / ASME 2010 SA-312 TP304 ASTM A312-09 / ASME 2010 SA-312 TP304L  
 ASTM A376-06 / ASME 2010 SA-376 TP304  
 SPECIFICATION :  
 MILL WORK NO. :0YU5273 O.D.:NPS6 W.T.:SCH80S LENGTH:MIN.20feet MAX.24feet QUANTITY:14pcs.  
 TOTAL LENGTH:316.06feet MASS:4099kg

HEAT NO. :F123012 F124108 F126013  
 PRODUCTS PCS. :1 4 9

HEAT TREATMENT :SOLUTION TREATED (1940°F X 2min. W.Q.)

## CHEMICAL COMPOSITION (%)

		#1	C	Si	Mn	P	S	Cr	Ni
			*3			*3	*3	*2	*2
SPEC. MIN.	L	-	-	-	-	-	-	180	80
MAX.	L	35	75	200	45	30	200	110	
HEAT NO.									
F123012	L	20	30	176	34	0	183	82	
F124108	L	25	29	182	36	0	185	84	
F126013	L	23	33	177	34	0	183	81	
*1 L:LADLE ANALYSIS *2:X10 *3:X1000 OTHER:X100									

## TENSILE TEST

		#1 #2	YS	TS	EL
			*3	*3	%
SPEC. MIN.	L B	P	30.0	P	75.0
MAX.	L B	P	-	P	-
HEAT NO.					
F123012	L B	P	40.3	P	88.8
F124108	L B	P	41.7	P	89.1
F126013	L B	P	40.8	P	89.4
TYPE OF SPECIMEN:STRIP 1in. WIDTH *1 SAMPLING DIRECTION L:LONGITUDINAL *2 SAMPLING POSITION B:BASE METAL					
*3 UNIT P:ksi GAUGE LENGTH:2.0in. KIND OF YS:0.2% OFFSET					

CORROSION TEST (MIL-P-1144D) : ACCEPTABLE  
 CORROSION TEST (MIL-P-24691/3) : GUARANTEED  
 FLATTENING TEST: ACCEPTABLE  
 VISUAL & DIMENSIONS: ACCEPTABLE  
 ULTRASONIC EXAMINATION (ASME SA-999/SE-213 U-SHAPED NOTCH) : ACCEPTABLE  
 NACE MRO175 HARDNESS: GUARANTEED  
 NO WELD REPAIR  
 MERCURY FREE  
 CERTIFIED ACCORDING TO PED97/23/EC, ANNEX I, PAR. 4.3 BY TUEV RHEINLAND INDUSTRIE SERVICE GmbH  
 (NOTIFIED BODY, ID-No. 0035/CERTIFICATE NO. 01 202 J/Q-02 008)  
 NACE MRO103 HARDNESS: GUARANTEED  
 EN 10204 3.1

Tony Cardwell

FEB 08 2012

WE HEREBY CERTIFY THAT THE MATERIAL HEREIN DESCRIBED HAS BEEN MANUFACTURED, SAMPLED, TESTED, AND INSPECTED IN ACCORDANCE WITH ABOVE STANDARD AND SPECIFICATION AND SATISFIES THE REQUIREMENTS.

MANAGER, QUALITY ASSURANCE SECTION

<An Important Message To Our Customers>  
 This certification is intended only for products listed. Modification to or unauthorized use of this certification is strictly prohibited. Offences may be regarded as forgery of documents and be subject to criminal prosecution.  
 If you have any questions on this certification, you can contact us by facsimile or e-mail as shown below;  
 Fax. No. : +81-3-4416-6789 E-mail: pipe-ipp@sumitomometals.co.jp

NO. 025594A11



**MILL TEST REPORT**

RM ID NUMBER

**113248**

SALES ORDER / RLS

**005032 / 3**

CERT ID / REV

**00043755 / 01**

SOLD TO

**INDUSTRIAL PIPING SPECIALISTS**  
**PO BOX 581270**  
**TULSA, OK 74158-1270**  
**USA**

CUSTOMER P.O. <b>TP262782</b>	CUSTOMER PART	HEAT NO. <b>402507</b>								
DESCRIPTION: <b>308008003121200</b> <b>8" WELDED PIPE SCHED 80S TP304/TP304L (UNS#S30400/S30403) A312 XRAY DOUBLE RANDOM LENGTH</b>										
CERTIFICATION REQUIREMENTS										
ENGINEERING <b>ASTM A312-09 ASME SA312-10, No ADD.</b>										
HYDRO PRESSURE <b>1,700 PSI</b>										
HEAT TREAT <b>Annealed at 1900 Deg F. and water quenched to below 800 Deg. F. in less than 3 minutes.</b>										
<b>Physical</b>										
Country of Origin <b>Sweden</b>										
<b>Chemical</b>										
C	Cr	Co	Cu	Mn	Mo	Ni	N	P	Si	S
.019	18.26	.140	.34	1.54	.004	8.11	.067	.029	.32	.001
<b>Mechanical</b>										
TEST	UNITS	RESULTS								
Tensile PSI	PSI	89000								
Yield PSI	PSI	47000								
Elong %		58								
Hardness										
HB174										
TEST	RESULT									
TG Bend	Pass									
NDE										
TEST	RESULT									
100% Radiography	Pass									

This report shall not be altered or reproduced, except in full, without the prior written approval of Bristol Metals LLC.  
This test report represents the actual attributes of the items furnished and all items were manufactured, sampled, inspected, and tested in full compliance with applicable specifications and your purchase order.  
Certification is in accordance with EN10204:2004 type 3.1.  
Chemical content is % by weight. Mechanical test results are in English units (inches and pounds).  
No weld repairs have been performed on the base material.  
Hardness in accordance with NACE MR0175 and MR0103 and material is free of cold work to enhance mechanical properties.  
Pipe is Pickled and Passivated in accordance with ASTM A380.  
Bristol Metals has a Quality Management System in place that is in compliance with ISO 9001:2008.  
Bristol Metals does not add mercury during any manufacturing process.  
NAFTA country of origin: USA

**Tony Cardwell**  
**FEB 08 2012**

*Rick Duncan*  
Rick Duncan - Quality Assurance Mgr



# MILL TEST REPORT

RM ID NUMBER

113248

SALES ORDER / RLS

005032 / 3

CERT ID / REV

00043755 / 01

SOLD TO

INDUSTRIAL PIPING SPECIALISTS  
PO BOX 581270  
TULSA, OK 74158-1270  
USA

CUSTOMER P.O.	CUSTOMER PART	HEAT NO.
TP262782		402507
<p>Raw Material Melt Source Sweden FAR BAA-Cannot certify compliance, DFARS BAA-Complies, FAA TAA-Complies</p>		
<p><b>Tony Cardwell</b> FEB 08 2012</p>		

*Rick Duncan*  
Rick Duncan - Quality Assurance Mgr





# BEBITZ

Bereich Flansche



EN ISO 9001 : 2008

Zertifikat-Registrier-Nr. 44 100 071888 Flanschenwerk Bebitz GmbH - Lebendorfer Str. 1 - 06420 Bebitz / GERMANY (A01)  
Tel. +49 34891 40 0 - Fax +49 3489140 329 - Email: flanges@bebitz.de

Abnahmeprüfzeugnis 3.1 / Inspection certificate 3.1 / Certificat de reception 3.1  
nach (A02) / acc. DIN EN 10204 : 2005 Nr. (A03) / No. 11-29255 Datum (Z02) / Date 22.11.2011

Zeichen des Herstellers (A04)  
Manufacturer's brand



Stempel des Abnehmers (Z03)  
Stamp of the testing engineer



Überprüft als Hersteller nach AD-Merkblatt W0 / TRD100 und VdTÜV Werkstoffblätter 350/3, 354, 399. / Approved as manufacturer acc. AD-Merkblatt W0 / TRD100 and VdTÜV Material Sheets 350/3, 354, 399.  
Zertifiziert nach DGR 97/23/EG durch TÜV CERT Zertifizierungsst. für Druckgeräte der TÜV NORD GmbH Co KG, Benannte Stelle Nr. 0045. / Certified acc. to PED 97/23/EC, certifying body for pressure equipment TÜV NORD GmbH Co KG, registration no. 0045.

Besteller Industrial Piping Specialists, Inc.  
(A06) P.O. Box 581270  
Customer TULSA, OK 74158-1270

Bestell-Nr (A07) / Order-No. TP309026  
Auftrag (A08) / Order 3121678  
Lieferschein / Delivery No. 3136208  
Position (B07.1) / Item 020RB

Stck (B08) Bezeichnung (B01; B09 - B11)  
Quantity Product

Schmelze/Prüflös (B07)  
Heat no./Test no.

35 4 inch Welding Neck Flange 150 lbs S40s raised face

87396/1

Material F 304/F 304L - ASTM A 182M-10 / ASME SA-182M  
(B02; B05) austenitis. > 1040 °C and quenched in water

Anford. ASME B 16.5-2009  
(B03) NACE MR0175-2003 / MR0103-2007 - DIN EN ISO 15156-3 : 2005  
Require- 2010 ASME Boiler & Pressure Vessel Code - Section II Part A  
ments PMI-tested 100%, IC-test acc. to DIN EN ISO 3651-2 Test A / ASTM A 262 practice E

Schmelzenanalyse (C71 - C92) / Ladle analysis

Erschmelzung (C70) / Melting process E

C	Si	Mn	P	S	Cr	Ni	Ti	Mo	Nb	V	N	Cu	Al	PREN
0,017	0,46	1,74	0,035	0,013	18,21	8,13					0,070			19,33

Mechanische Prüfungen / Mechanical tests

$$CEV = C + \frac{Mn}{6} + \frac{V + Mo + Cr}{5} + \frac{Ni + Cu}{15} \quad PREN = Cr + 3,3 \times Mo + 16 \times N$$

Probenlage (C02): q/t / Position of Specimen: q/t Zugversuch / Tensile Test - ASTM A 370						KBV / Charpy Impact Test DIN EN ISO 148-1			Härte (C32) Hardness ISO 6506-1	
Temp (C03) °C	Streckgr. (C11) / Yield strength 0,2% MPa (N/mm²)		Zugfestigk. (C12) Tensile strength MPa (N/mm²)		Dehng. (C13) Elongation %	Einschnürrg. Red. of area %	Temp. (C03) °C	Einzelwerte (C42) Single values J	MW (C43) Average J	HBW 2,5/187,5
20	min 205 max		515		30,0	50	20			
20	282	317	575		59,3	72	20	166 160 172	166,0	159

Besichtigung und Ausmessung ohne Beanstandungen. (D01) / Results of inspection and dimension without objections.  
Die gestellten Anforderungen sind erfüllt. (Z01) / The product meets the requirements.

R. Sparing,  
Abnahmebeauftragter des Herstellers (Z02)  
Inspection Representative of the Manufacturer

APZ wurde mit EDV erstellt und ist ohne Unterschrift gültig. / This  
MTR was electronically generated and is valid without a signature.

**Tony Cardwell**

FEB 08 2012

**MILL TEST REPORT**

RM ID NUMBER

110061

SALES ORDER / RLS

004732 / 5

CERT ID / REV

00033611 / 01

SOLD TO

INDUSTRIAL PIPING SPECIALISTS  
606 N. 145TH EAST AVE  
TULSA, OK 74116  
USA

BR440S04/L

81A  
22A

CUSTOMER P.O. TP255793	CUSTOMER PART	HEAT NO. 400023
DESCRIPTION: 304004003120200 4" WELDED PIPE SCHED 40S TP304/TP304L (UNS# S30400/S30403) A312 DOUBLE RANDOM LENGTHS		
CERTIFICATION REQUIREMENTS		
ENGINEERING ASTM A312-09 ASME SA312-07, 09 ADD.		
HYDRO PRESSURE 1,600 PSI		
HEAT TREAT Annealed at 1900 Deg F. and water quenched to below 800 Deg. F. in less than 3 minutes.		
Chemical		
C .021	Cr 18.12	Mn 1.366
Ni 8.10	N .0440	P .029
Si .458	S .0120	
Mechanical		
TEST Tensile PSI	UNITS PSI	RESULTS 87300
Yield PSI	PSI	39600
Elong %		57.6
Hardness		
RB81		
TEST TG Bend	RESULT Pass	
<p>This test report represents the actual attributes of the items furnished and all items were manufactured, sampled, inspected, and tested in full compliance with applicable specifications and your purchase order. Certification is in accordance with EN10204:2004 type 3.1. Chemical content is % by weight. Mechanical test results are in English units (Inches and pounds). No weld repairs have been performed on the base material. Hardness in accordance with NACE MR0175 and MR0103 and material is free of cold work to enhance mechanical properties. Pipe is Pickled and Passivated in accordance with ASTM A380. Bristol Metals has a Quality Management System in place that is in compliance with ISO 9001:2000. Bristol Metals does not add mercury during any manufacturing process. NAFTA country of origin: USA Raw Material Melt Source USA FAR BAA - Complies, DFARS BAA - Complies, FAR TAA - Complies</p>		
		<p><b>Tony Cardwell</b> FEB 08 2012</p>

*Rick Duncan*  
Rick Duncan - Quality Assurance Mgr

**FCI FORGED COMPONENTS INC.**

14527 Smith Rd.  
Humble, Texas 77399  
TEL: (281) 441-4088  
FAX: (281) 441-8899

Page 1 of 1  
**REPORT NUMBER 58984**  
**DATE 2/10/2012**  
**FCI ORDER NUMBER 55006**  
Processed I.A.W. EN10204 3.1

**CUSTOMER ORDER NUMBER** TP330675  
**SOLD/SHIPPED TO** IPS (TULSA)  
606 NORTH 145 EAST AVE

TULSA OK 74116 USA

**Item Quantity Description**

1	2	2" X 600 X 9" LG RF LWN
2	2	2" X 600 X 9" LG RF LWN
4	2	2" X 300 X 9" LG RF LWN
5	2	2" X 300 X 9" LG RF LWN

10279-728

**HEAT NUMBER:**  
M359

**MATERIAL TYPE:**  
SA182F304/L

I.A.W. ASME 2010 ED.

**CHEMICAL ANALYSIS**

**PHYSICAL PROPERTIES**

C .013  
Mn 1.770  
P .0290  
S .0300  
Si .580  
Cr 18.240  
Ac .500

**Yield PSI** 59,700  
**Tensile PSI** 91,800  
**Elongation** 52  
**Reduction of Area** 73.4  
**Hardness** 201 HBW

Al 8.250  
I .0900

**Heat Treatment** SOLUTION ANNEALED  
**Temperature** 1900 °F  
**Time at Temperature**  
**Cooling Media** WATER

We hereby certify that all test results and process information contained herein are correct and true as contained in the records of the company.

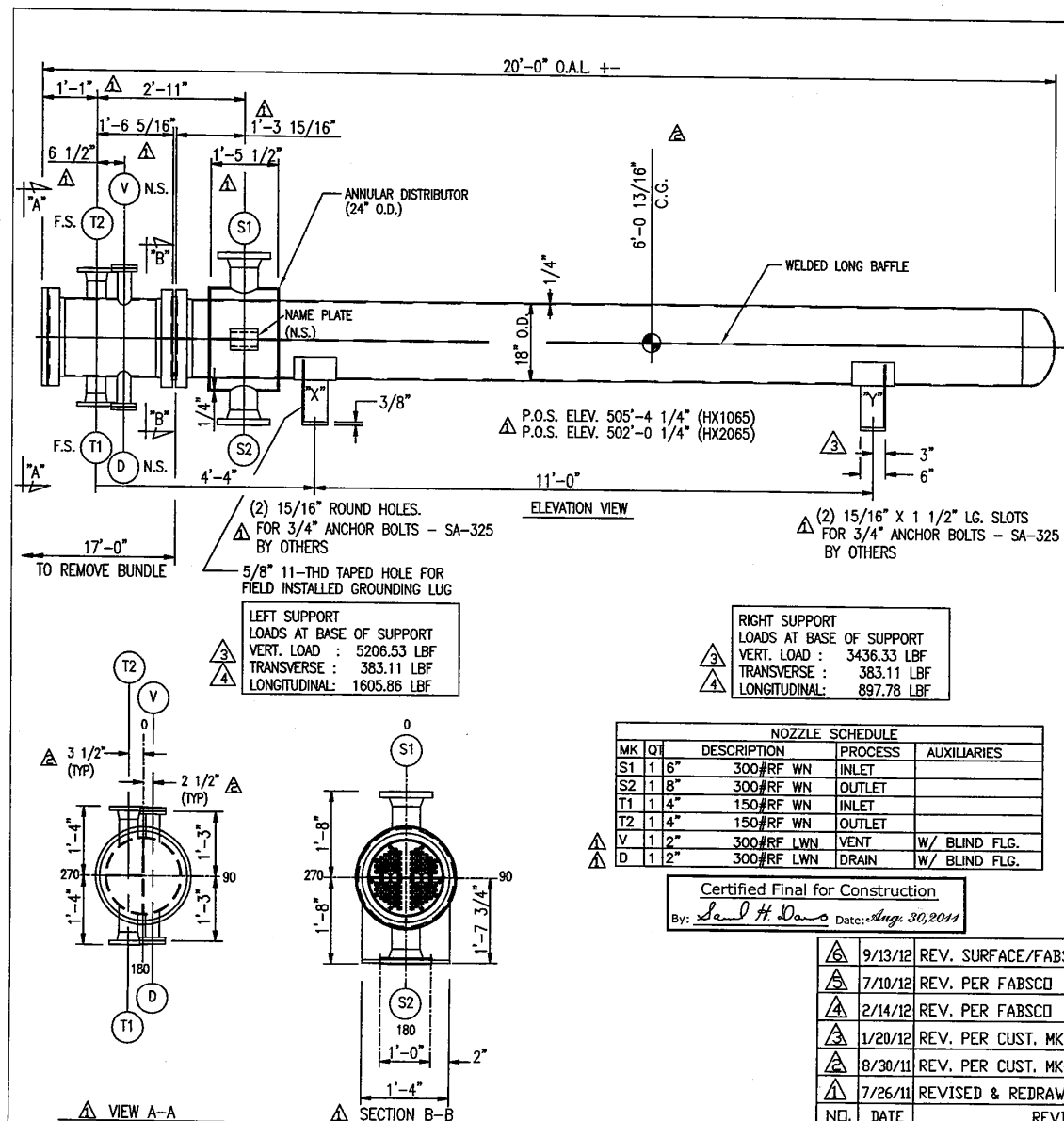
Prepared by

*Helen Fielder*

Name: Helen Fielder

Title: QA Representative





DESIGN DATA	SHELL	TUBE
DESIGN PRESSURE P.S.I.G.	180	150
VACUUM PRESSURE P.S.I.G.	N/A	N/A
TEST PRESSURE P.S.I.G.	234	195
DESIGN TEMP. Deg.F.	290	200
MDMT Deg. F.	10	10
CORROSION ALLOWANCE	0"	0"
NUMBER OF PASSES	2	4
RADIOGRAPHY	100% (RT-4)	100%
HEAT TREAT REQUIRED	NO	NO

ESTIMATED WEIGHTS, LBS.		
DRY: 4100	BUNDLE: 2200	WET: 5950
OPERATING: 4750		

**SPECIFICATIONS**  
ASME Code Section VIII, Div 1, 2010 Ed. (Stamp: Yes).  
TEMA Class R; National Board Registration is required.  
Design Reviewed For Compliance With The Requirements of UG-120(d).  
Exempt From Impact Testing Per UG-20(f), UCS-66(a)(c), Fig. UCS-66 Note (c), and UCS-67.

MATERIAL	
CHANNEL:	SA-312TP304L
SHELL:	SA-312TP304L
TUBESHEETS:	SA-240-304L
BAFFLES:	SA-240-304L
TUBES:	SA-213TP304L
( 51 U's )	1" O.D.X 0.091" (AVG) X 16'-0" LG.
TUBE PITCH	1 1/4" SURFACE 437 SQ. FT

**GENERAL NOTES**

ALL BOLT HOLES TO STRADDLE NATURAL CENTER LINES.

GASKETS: 1/8" THK 304L STN STL DBL JCKT'D  
GRAPHITE FILLED & GRAPHITE COATED  
(2-SETS SPARES REQ'D.)

INSULATION: SHELL SIDE 3" METAL GUARD (BY OTHERS)  
STRENGTH WELD TUBES TO TUBESHEET PER UW-20 (c or d)  
SHELL SIDE DESIGNED FOR WET H2S PER UOP 4.11 PARG 4.7

PAINT: SADDLE SUPPORT SURFACES TO BE SOLVENT CLEANED PER SSPC-SP1.  
SANDBLAST SADDLE SUPPORTS SURFACES PER SSPC-SP10 (PROFILE 1-2 MILS)  
PRIMER: ONE (1) COAT CARBOLINE CARBOZINC 859 (2-3 MILS DFT)  
INTERM.: ONE (1) COAT CARBOLINE CARBOGUARD 890 (4-6 MILS DFT)  
FINISH: ONE (1) COAT CARBOLINE CARBOHANE 134HG (2-3 MILS DFT)  
(8-12 MILS TOTAL D.F.T.)(LIGHT GREY (RAL 7035))

CUSTOMER: SOUTHERN COMPANY SERVICES

REQ. NO.: MC213

P.O. NO.: MPC17901

ITEM NO.: HX1065/2065

SERVICE: AGR FLASH GAS COOLER

SHIP TO: KEMPER COUNTY, MS

ASSEMBLY AND SPECIFICATIONS FOR			
TWO 17-192 A F U			
DWN. BY	DS	<b>Fabsco</b> SHELL & TUBE, LLC	
DATE	3-23-11		
CKD. BY	SR	DWG. NO.	S11-10279-7/8-1
NO.	DATE	REVISIONS	REV. 6

9/13/12	REV. SURFACE/FABSCO DD/
7/10/12	REV. PER FABSCO DD/SF
2/14/12	REV. PER FABSCO DD/SF
1/20/12	REV. PER CUST. MK-UP DD/SF
8/30/11	REV. PER CUST. MK-UP KS/DD
7/26/11	REVISED & REDRAWN AC/DGD

# CUSTOMER REQUIREMENTS

- 1) NOTIFY CUSTOMER AT LEAST 10 DAYS IN ADVANCE OF ANY SCHEDULED TEST, WITH VERBAL CONFIRMATION BEFORE THE HOLD POINT IS SCHEDULED TO OCCUR.
- 2) SUPPLIER IS REQUIRED TO PROVIDE A MINIMUM FIVE (5) DAY ADVANCE NOTICE PRIOR TO WHEN THE WITNESS POINT IS SCHEDULED TO OCCUR.
- 3) THE FLATNESS TOLERANCE (MAXIMUM DEVIATION FROM A PLANE) ON PERIPHERAL GASKET CONTACT SURFACES SHALL BE  $\pm 1/32"$ . USE OF A STRAIGHT EDGE TO DETERMINE FLATNESS TOLERANCE IS ACCEPTABLE.
- 4) GIRTH FLANGE FLATNESS TOLERANCE SHALL BE MEASURED AFTER THE FLANGE FINAL WELDED. FLATNESS TOLERANCE ON TUBESHEET GASKET SURFACES SHALL BE MEASURED AFTER TUBE-TO-TUBESHEET JOINTS ARE ROLLED OR WELDED.
- 5) THE INTERPASS TEMPERATURE FOR AUST. STN. STL. SHALL NOT EXCEED 350°F.
- 6) TEMPORARY ATTACHMENTS AND ARC STRIKES SHALL BE REMOVED, AND THE AREA CONDITIONED, NONDESTRUCTIVELY EXAMINED MAGNETIC PARTICLE EXAMINATION (MT) OR DYE PENETRANT EXAMINATION (PT), IN ADDITION TO VISUAL EXAMINATION, TO ENSURE NO CRACKS HAVE BEEN GENERATED.
- 7) PLUG SHELL INSIDE DIAMETER WITH A METAL TEMPLATE CONSISTING OF TWO DISCS (EACH HAVING A DIAMETER EQUAL TO THE DIAMETER OF THE BAFFLES) RIGIDLY MOUNTED PERPENDICULAR TO A SHAFT, AND SPACED 18" APART.
- 8) WELD JOINT SURFACES AND A MINIMUM OF 1" ON THE INTERNAL AND EXTERNAL SURFACES OF THE ADJACENT BASE METAL SHALL BE CLEAN AND FREE OF PAINT, OIL, DIRT, SCALE, OXIDES, AND OTHER FOREIGN MATTER.
- 9) ALL LIFTING LUG WELDS ARE TO BE M.T. OR P.T. OVER THE ENTIRE WELD SURFACE.
- 10) STAMPING TO BE DONE WITH "LOW STRESS" STAMPS.
- 11) SUPPORT PADS ARE TO BE AIR AND SOAP TESTED WITH 15 P.S.I.G. AIR PRESSURE. PLUG TEST HOLES WITH STIFF GREASE AFTER HYDROTEST.
- 12) THE P.O. No., ITEM No., SHIPPING WT., TO BE STENCILED ON THE SIDE OF THE SHELL IN 3" HIGH LETTERS.
- 13) SEAL WELD ALL ATTACHMENTS.
- 14) WHEN PREHEAT IS REQUIRED FOR WELDING, IT SHALL APPLY ALSO TO ELECTRIC ARC CUTTING, FLAME CUTTING, TACK WELDING, AND BACK GOUGING.

- 15) FOR CLEANING STN. STL., USE ALUMINUM OXIDE OR SILICON CARBIDE GRINDING WHEELS AND/OR STN. STL. BRUSHES NOT PREVIOUSLY USED ON CARBON OR LOW ALLOY STEEL.
- 16) PRECAUTION SHALL BE TAKEN TO AVOID CONTAMINATION ON STAINLESS STEEL DURING FABRICATION.
- 17) NOZZLE WELDS SHALL BE SPOT X-RAYED OR U.T. WHEN POSSIBLE. NOZZLE WELD SHALL BE M.T. OR P.T. AS A MIN. REQUIREMENT.
- 18) CHEMICAL ANALYSIS OF ALL WELDS ON SHELL & CHANNEL SHALL BE TAKEN AS FOLLOWS TO REPRESENT BOTH MANUAL AND AUTOMATIC WELD:
  - EACH COMPLETED CYLINDER, HEAD, NOZZLE WELD SHALL BE ANALYZED.
  - TESTING TO BE DONE ON INSIDE SURFACE
  - ANALYSIS SHALL INCLUDE EACH SOURCE OF DEPOSITED WELD MATERIAL (WIRE REEL, BOX OF WELD RODS)
  - CHECK FOR Cr, Ni, C, Nb (Cb), Mo, V, WHERE APPLICABLE
- 19) FERRITE DETERMINATION OF STAINLESS STEEL PRESSURE PARTS ARE REQ'D AS FOLLOWS:
  - FERRITED NUMBER TO BE BETWEEN 3.0 TO 8.
  - READINGS SHALL BE TAKEN FROM AT LEAST TWO (2) LOCATIONS FROM EACH NOZZLE GIRTH WELD, VESSEL WELD SEAM & OTHER STRENGTH WELDS. AT LEAST SIX (6) READINGS SHALL BE TAKEN AT EACH LOCATION.
- 20) P.M.I. IS TO BE IN ACCORDANCE WITH KBR SPECIFICATION 2-1TS-US-BO-M&U.
- 21) FLANGE BOLTING SHALL BE TIGHTENED WITH A TORQUE WRENCH TO A VALUE OF 50,000 P.S.I. STRESS IN STUDS.
- 22) BACK CHIPPED WELDS, WELDING GROOVE AND PLATE EDGES SHALL BE M.T. OR P.T. PRIOR TO WELDING ON THE REVERSE SIDE.
- 23) LIQUID PENETRANT MATERIALS SHALL BE FREE OF HALOGENS (CHLORIDES) ZINC, SULFUR AND OTHER MATERIALS DETREMENTAL TO AUSTENETIC STAINLESS STEEL.
- 24) MAXIMUM HORIZONTAL OR VERTICAL DEFLECTION OF MACHINED FACES OF NOZZLES FROM THE DESIGN PLANE SHALL BE 1/2 DEGREE OR 1/32", WHICHEVER IS GREATER.
- 25) EXPOSED THREADS OF BOLTS SHALL BE COATED WITH RUST VETO 342 RUST PREVENTATIVE, TO PREVENT CORROSION DURING TESTING, SHIPPING, AND STORAGE.

# NOT REQUIREMENTS

- SHELL: 100% (RT-4) RADIOGRAPH.  
FRONT CHANNEL: 100% RADIOGRAPH.
- AFTER WELDING LONG BAFFLE TO SHELL, M.T. FULL LENGTH OF WELD. RADIOGRAPH FILM SHALL BE FINE GRAIN, HIGH DEFINITION, HIGH CONTRAST FILM (KODAK TYPE AA, EQUIVALENT OR BETTER). FILM DENSITY SHALL BE WITHIN A RANGE OF 2.0 TO 3.5 AS DETERMINED BY FILM DENSITY SPECIMENS OR BY CALIBRATED DENSITOMETER.

# STANDARD SHOP PRACTICES

- 1) ALL BOLT HOLES TO STRADDLE NATURAL CENTER LINES.
- 2) GRIND INSIDE WELDS FLUSH IN BTM. 1/3 OF SHELL.
- 3) GRIND INSIDE WELDS SMOOTH WITH CHANNELS.
- 4) MATCH MARK ALL COMPONENTS FOR PROPER FIT UP.
- 5) INSTALL SHIPPING COVERS ON ALL NOZZLES FOR SHIPMENT.
- 6) VESSEL SHALL BE CLEANED INSIDE AND OUTSIDE OF WELD SPLATTER DIRT, WELD ROD, AND OTHER FOREIGN DEBRIS.
- 7) MAIN DIMENSIONS ARE TO BE HELD WITHIN TEMA TOLERANCE.

# STANDARD HYDROTEST REQUIREMENTS

- 1) HYDROTEST CHARTS REQUIRED.
- 2) HYDROTEST TO BE HELD FOR ONE (1) HOUR (MIN).
- 3) HYDROTEST WATER TEMPERATURE TO BE AT LEAST 50°F. (MIN)
- 4) HYDROTEST WATER TO HAVE A CHLORIDE CONTENT OF 50ppm (MAX.), AND THE pH CONTROLLED BETWEEN 6 AND 9.
- 5) AFTER HYDROTEST, THE EXCHANGER TO BE DRAINED AND DRIED ANY REMAINING WATER IS TO BE REMOVED BY BLOWING WARM AIR AT 120° F (MAX.)

# SPECIAL PAINT REQUIREMENTS

- 1) ALL STAINLESS STEEL MATERIAL SHALL BE PROTECTED FROM BLASTING, OVERSPRAY, AND COATING.
- 2) ITEMS SUCH AS CHAINS, HOOKS, TONGS, METAL BARS, NARROW STRAPS, SHALL NOT BE ALLOWED TO COME IN CONTACT W/ PAINTED SURFACES.

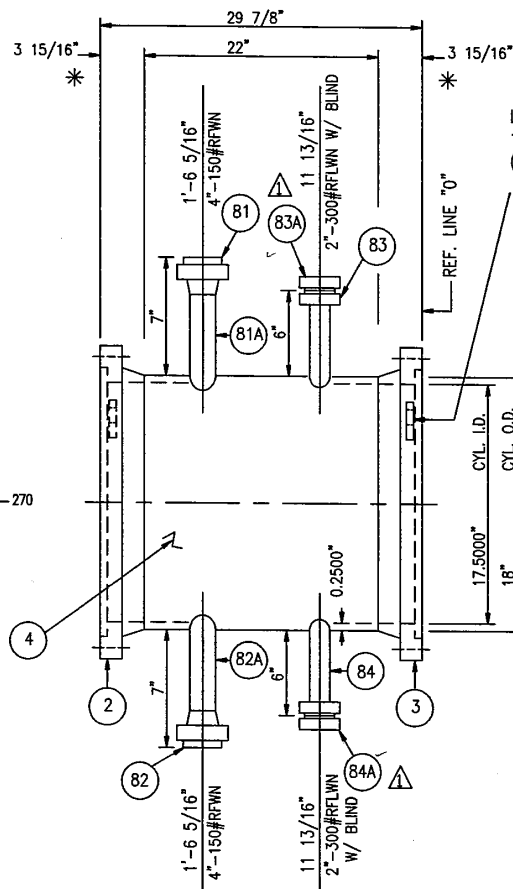
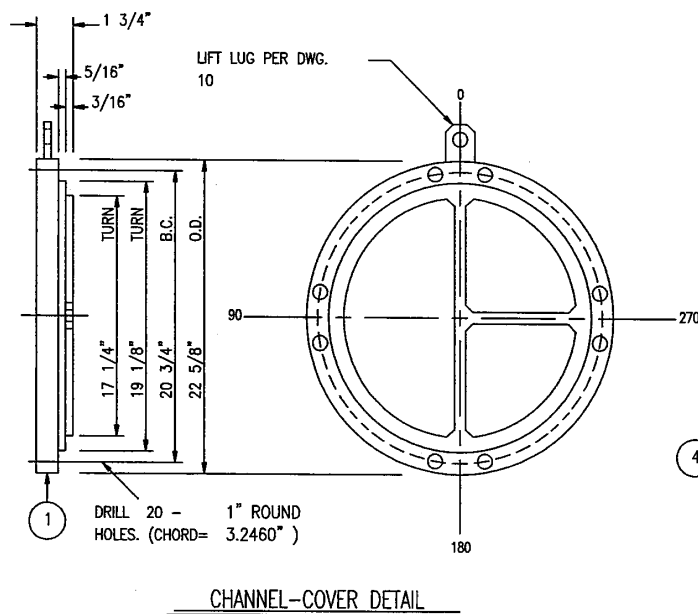
P.O. NO.: MPC17901  
ITEM NO.: HX1065/2065  
SERVICE: AGR FLASH GAS COOLER  
REQ'N. NO.: MC213

				JOB NOTES			
				DWN. BY AC		<b>Fabsco</b> <b>SHELL &amp; TUBE, LLC</b>	
				DATE 4-7-11			
				CKD. BY DGD		DWG. NO. S11-10279-7/8-1A	
				REV. 2			

7/10/12	REV. PER FABSCO	DGD	
8/30/11	REV. PER CUST. MK-UP	KS/DD	
REV.	DATE	REVISIONS	

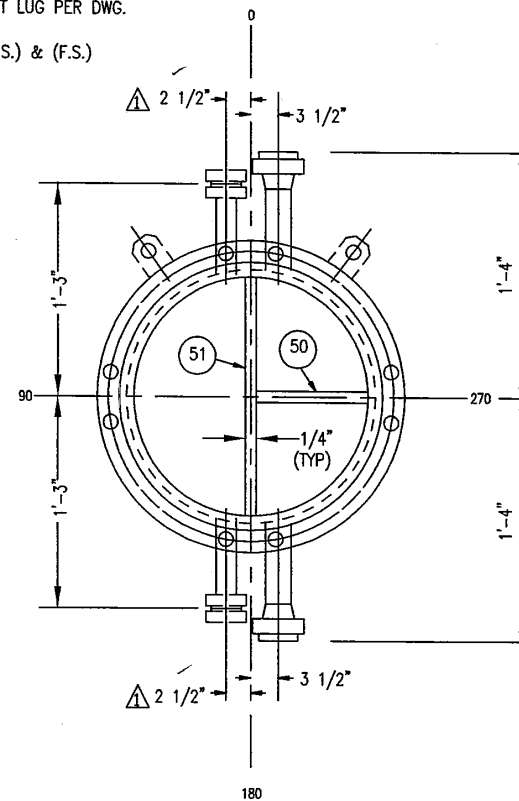
COVER NOTES:

- 1) MACHINE ALL COVER SURFACES TO 250 RMS EXCEPT GASKET SURFACES ARE 125 - 250u IN Ra.
- 2) ALL COVER PASS PLATE GROOVE WIDTHS TO BE 3/8" WIDE.
- 3) ALL COVER PASS PLATE GROOVE JUNCTIONS TO BE NOTCHED 3/8" AT 45 Deg.



\* MACHINE GASKET FACING AFTER ALL WELDING.  
FOR FLANGE DETAILS SEE DWG. 12

LIFT LUG PER DWG. 10 (N.S.) & (F.S.)

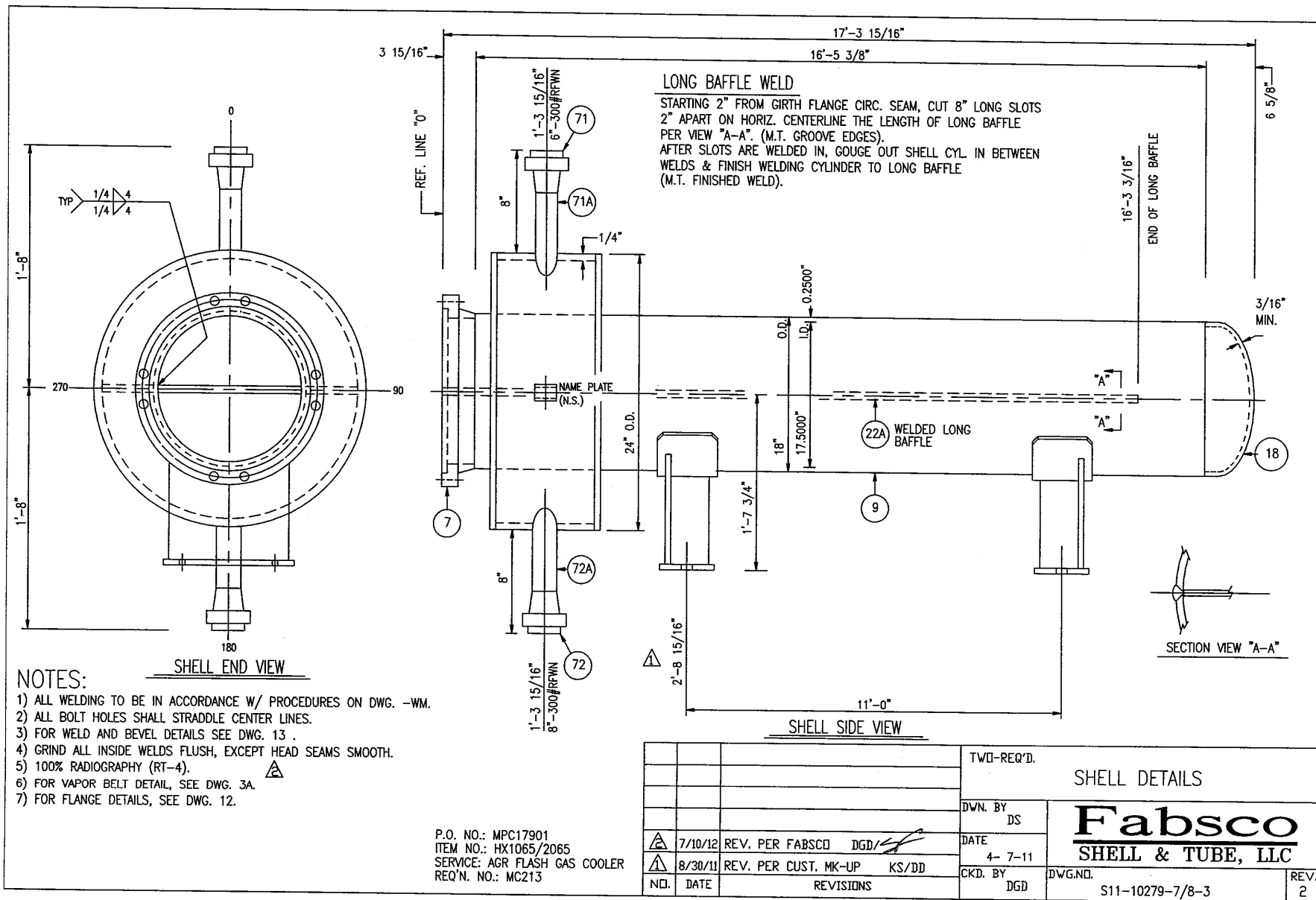


NOTES:

- 1) ALL WELDING TO BE IN ACCORDANCE W/ PROCEDURES ON DWG. -WM.
- 2) ALL BOLT HOLES TO STRADDLE CENTER LINES.
- 3) FOR WELD AND BEVEL DETAILS SEE DWG. 13
- 4) GRIND ALL INSIDE WELDS SMOOTH.
- 5) 100% RADIOGRAPHY.

P.O. NO.: MPC17901  
ITEM NO.: HX1065/2065  
SERVICE: AGR FLASH GAS COOLER  
REQ'N. NO.: MC213

			TWO-REQ'D.			
			FRONT CHANNEL DETAILS			
			DWN. BY DS		<b>Fabsco</b> <b>SHELL &amp; TUBE, LLC</b>	
			DATE 4- 7-11			
			CKD. BY DGD		DWG.NO.	REV.
					S11-10279-7/8-2	1

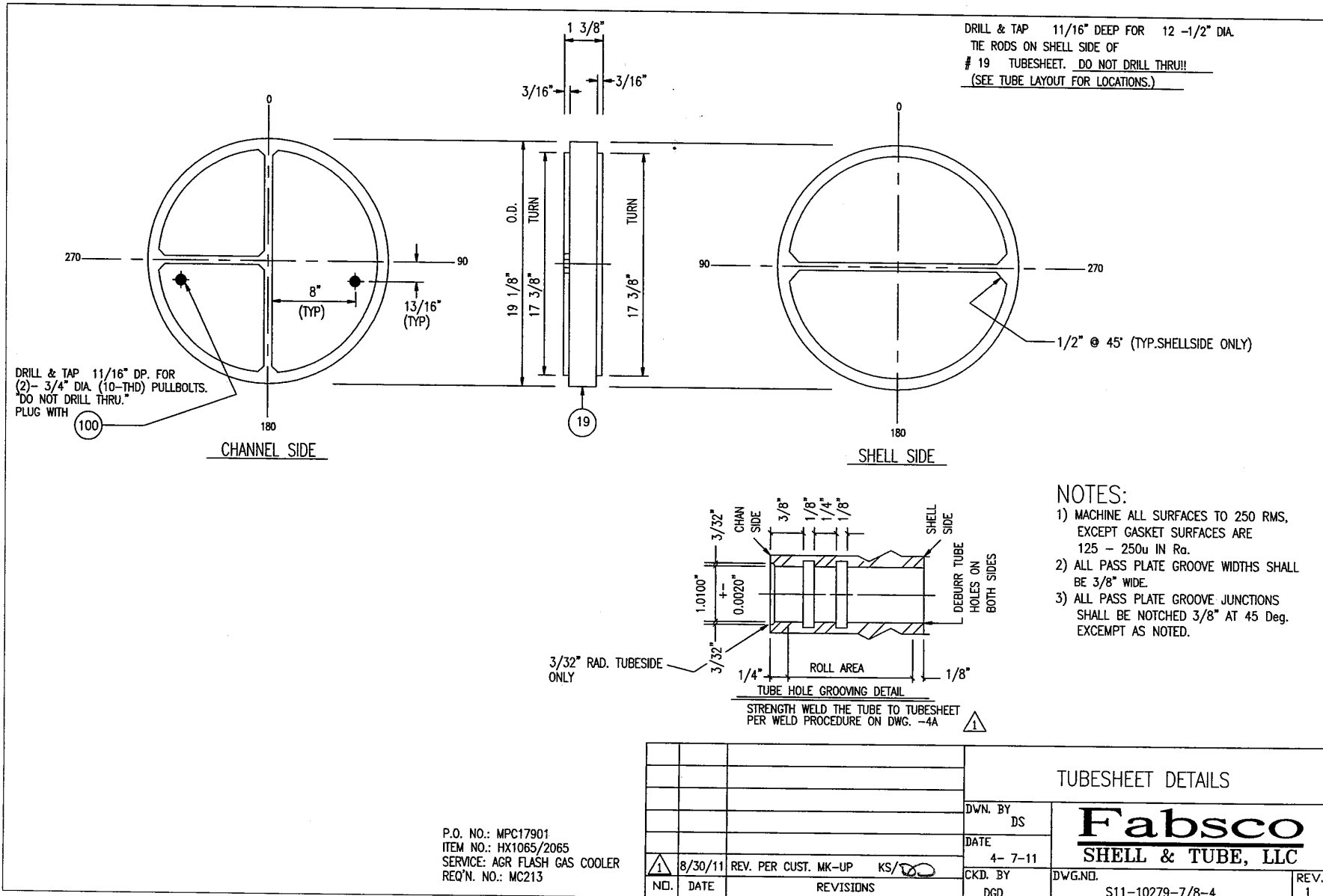


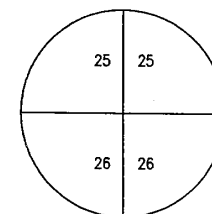
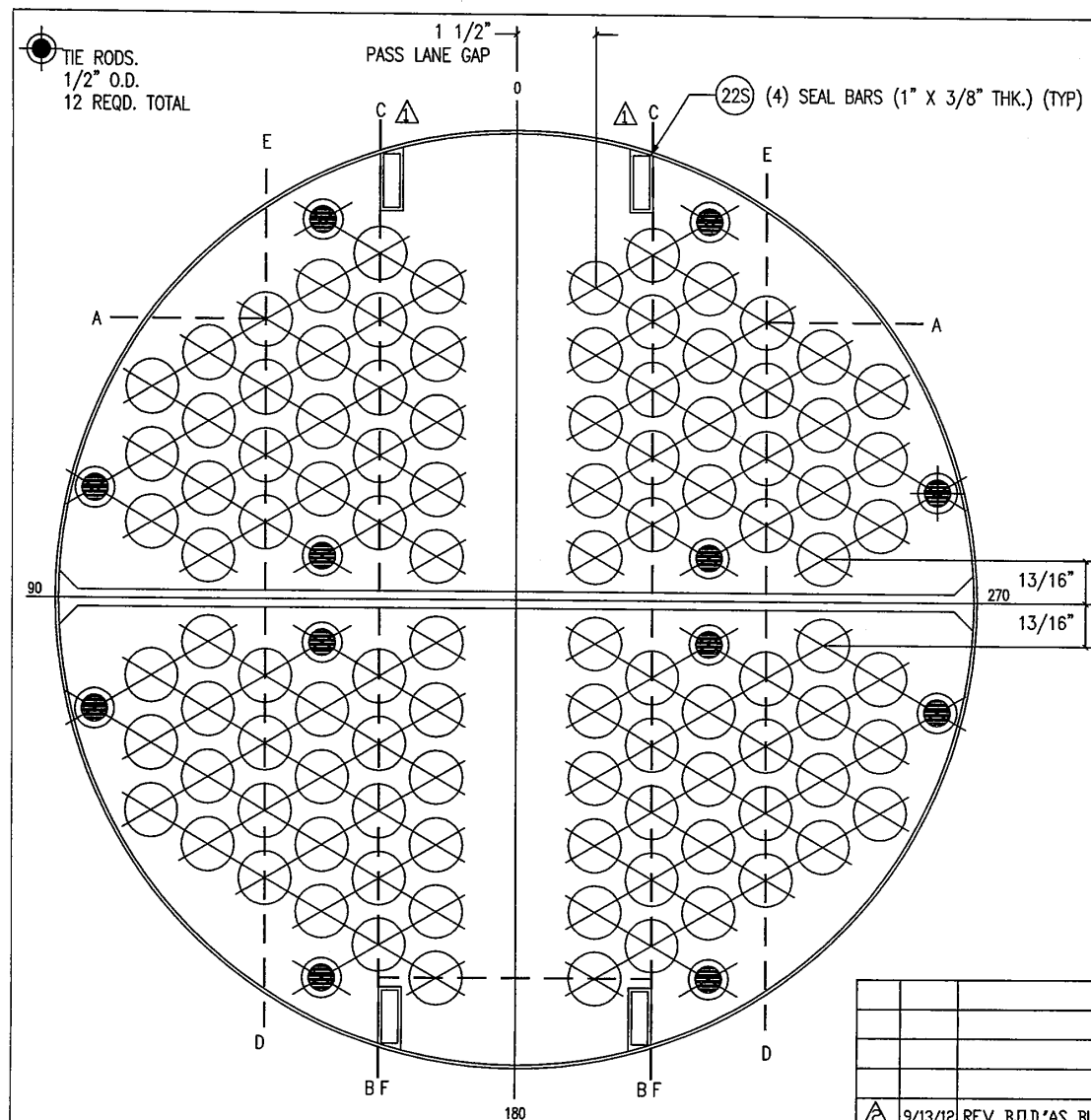
				TWO-REQ'D.	
				SHELL DETAILS	
				DWN. BY	DS
				DATE	4-7-11
				CKD. BY	DGD
				DWG. NO.	S11-10279-7/8-3
				REV.	2

**Fabsco**  
 SHELL & TUBE, LLC









TUBE DISTRIBUTION

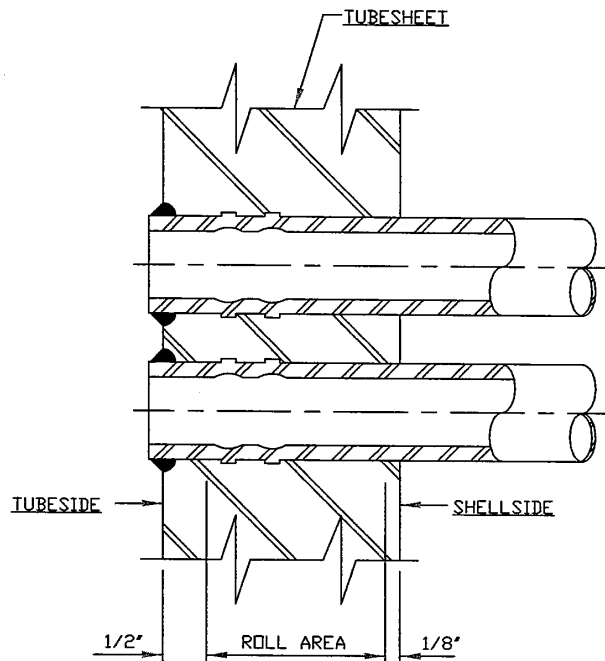
NOTES:

- 1) DRILL AND REAM TUBE HOLES IN TUBESHEETS TO 1.0100"  $\pm$  0.0020" DIA. FOR 51 - 1" O.D. U-TUBES. TOTAL HOLES = 102
- 2) TUBE PITCH = 1 1/4" ROT. TRIANGLE. (30°)
- 3) NUMBER OF PASSES = 4
- 4) SEE DRAWING 4 FOR TUBE GROOVING DETAILS.
- 5) BAFFLE O.D. = 17 3/16"  $\Delta$
- 6) O.T.L. = 16 15/16"

P.O. NO.: MOC17901  
 ITEM NO.: HX1065/2065  
 SERVICE: AGR FLASH GAS COOLER  
 REQ'N. NO.: MC213

VIEW LOOKING AT SHELL SIDE OF NO. 19 TUBESHEET

			TUBE LAYOUT			
			DWN. BY DS		<b>Fabsco</b> <b>SHELL &amp; TUBE, LLC</b>	
			DATE 4-7-11			
			CKD. BY DGD		DWG. NO.	REV. 2
			S11-10279-7/8-5			



NOTE: TUBES TO PROJECT  $1/8" \pm 1/16"$  PRIOR TO WELDING.

#### PROCEDURE FOR STRENGTH WELDING TUBES

1. CHAMFER ALL TUBE HOLES  $3/32"$  RADIUS ON TUBESIDE.
2. CLEAN ALL TUBES AND TUBE HOLES WITH NAPHTHA.
3. LIGHTLY EXPAND TUBES TO METAL CONTACT.
4. WELD TUBES TO TUBESHEET WITH A TWO (2) PASS WELD USING: T-189  
DO NOT BURN END OF TUBES.
5. AFTER ALL TUBES HAVE BEEN WELDED AND PRIOR TO ROLLING, CHECK EACH TUBE WELD WITH DYE PENETRANT TEST.
7. AFTER STRENGTH WELD TEST, EXPAND TUBES TO PROPER AMOUNT STARTING  $1/2"$  BEHIND STRENGTH WELD.
8. AFTER ALL TUBES HAVE BEEN ROLLED, CHECK EACH TUBE WELD WITH DYE PENETRANT TEST.
9. AIR AND SOAP TEST TUBE WELDS WITH 15 PSIG AIR PRESSURE ON SHELLSIDE.

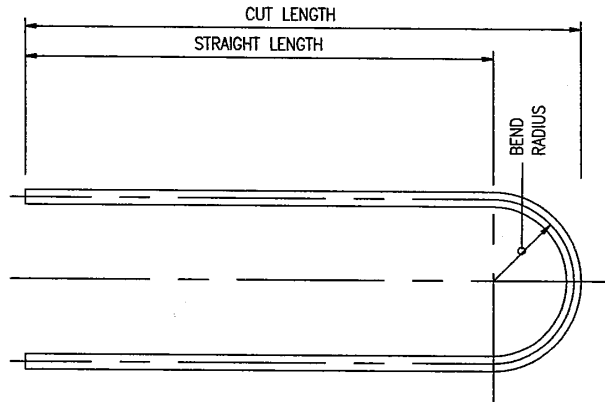


P.O. NO. : MPC17901  
ITEM NO. : HX 1065/2065  
SERVICE : AGR FLASH GAS COOLER  
REQ'N NO. : MC213

			STRENGTH WELD DETAIL	
			DWN. BY	KS
			DATE	8/31/2011
			CKD. BY	<i>D. L. L.</i>
			DWG. NO.	S11-10279-7/8-4A
			REV.	1
NO.	DATE	REVISIONS		

**Fabsco**  
SHELL & TUBE, LLC

ROW	NO.OF TUBES	BEND RADIUS	BEND DIAMETER	BEND LENGTH	STRAIGHT LENGTH	CUT LENGTH	DEVELOPED LENGTH
A	11	1 1/2"	3"	0'-4 3/4"	16'-0"	16'-2"	32'-4 3/4"
B	10	2 5/8"	5 1/4"	0'-8 1/4"	16'-0"	16'-3 1/8"	32'-8 1/4"
C	8	3 11/16"	7 3/8"	0'-11 5/8"	16'-0"	16'-4 3/16"	32'-11 5/8"
D	8	4 3/4"	9 1/2"	1'-2 15/16"	16'-0"	16'-5 1/4"	33'-2 15/16"
E	8	5 7/8"	11 3/4"	1'-6 1/2"	16'-0"	16'-6 3/8"	33'-6 1/2"
F	6	6 15/16"	13 7/8"	1'-9 13/16"	16'-0"	16'-7 7/16"	33'-9 13/16"



TOTAL NO. REQUIRED: 51  
 SIZE: 1" O.D. X .091" AVG  
 MATERIAL: SA-213TP304L

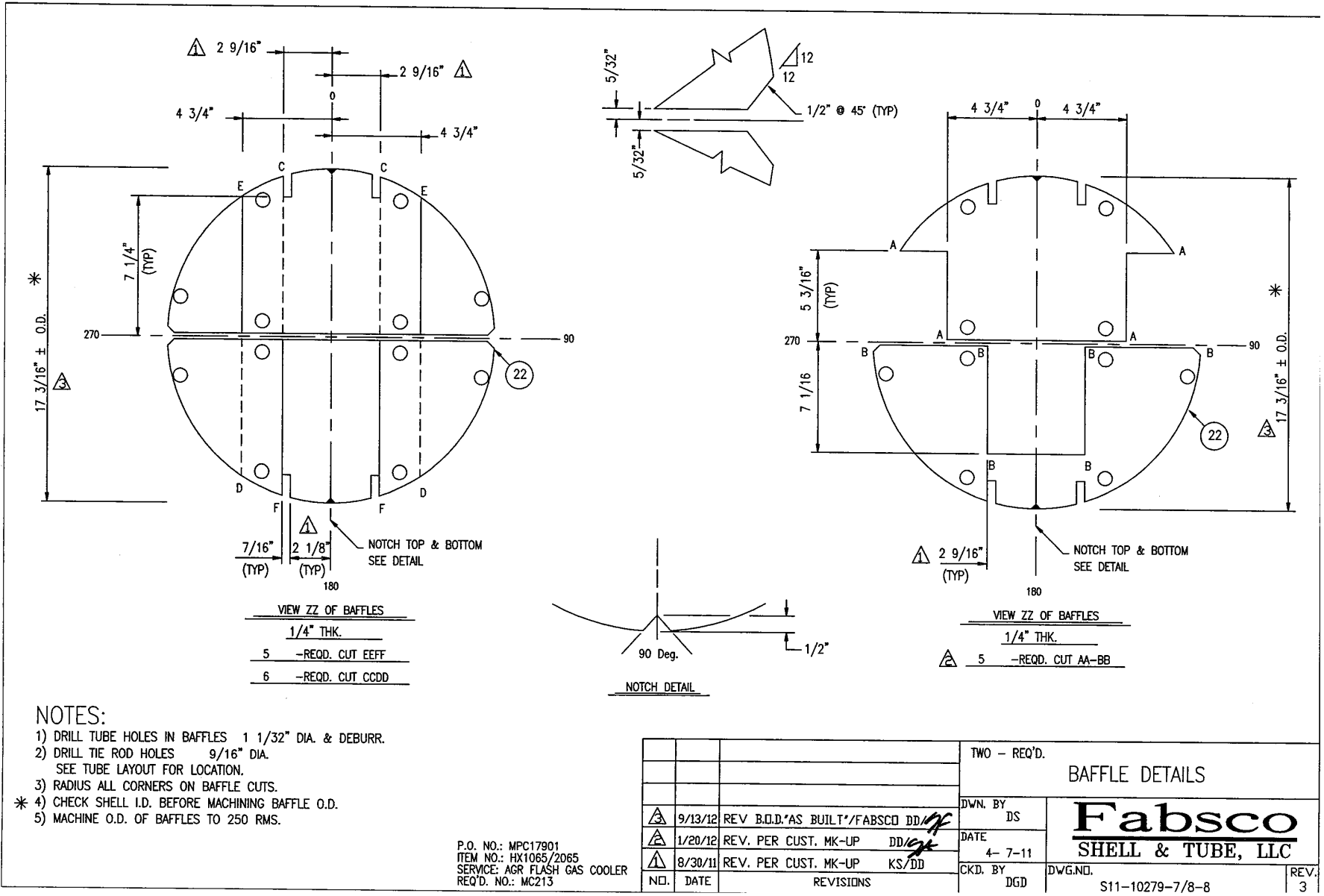
MAXIMUM OUT-OF-ROUNDNESS OF TUBES AT  
 U-BENDS NOT TO EXCEED 10%.

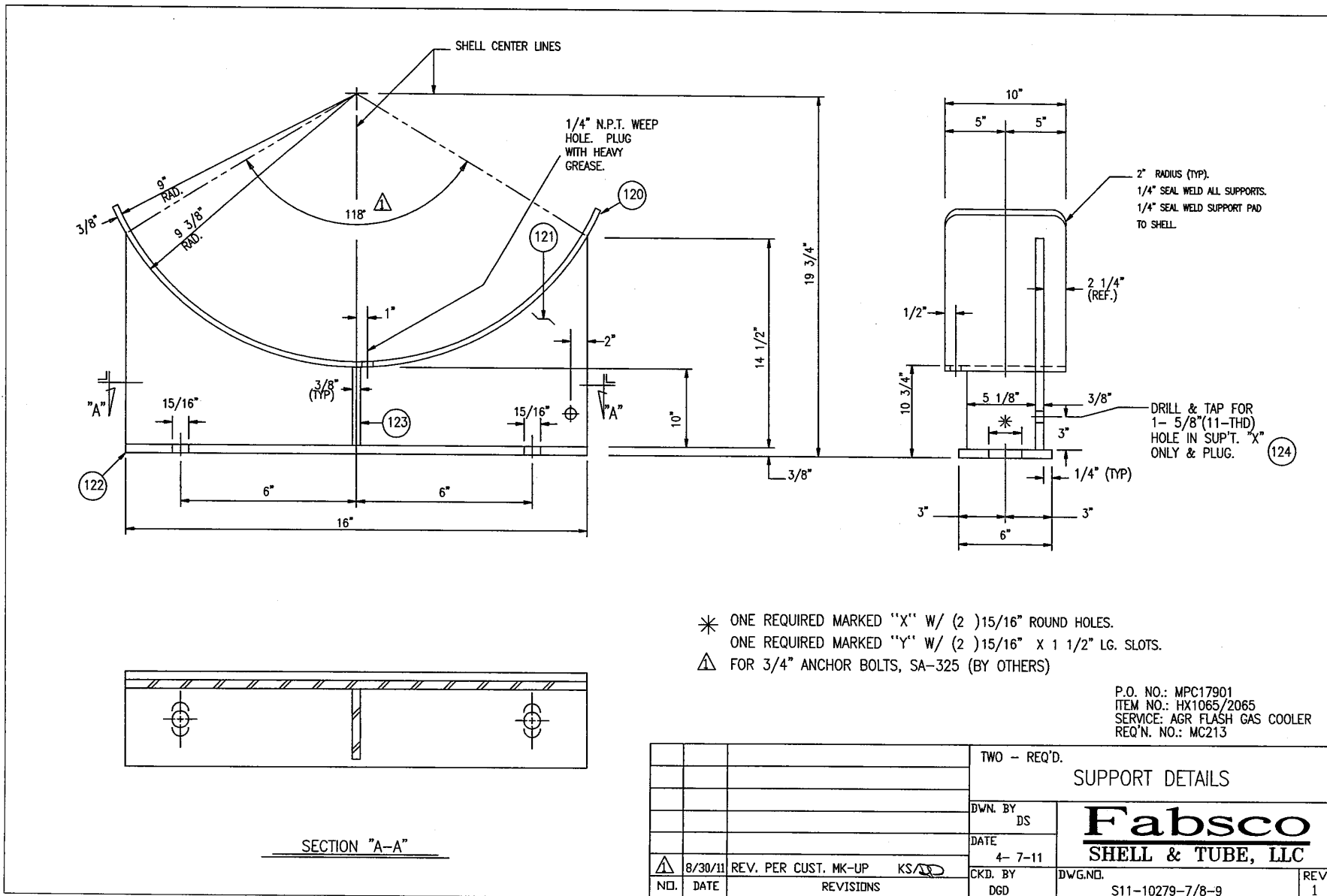
QUANTITIES SHOWN ARE FOR  
 ONE SET OF U-TUBES,  
 2 -SET REQ'D. TOTAL.

P.O. NO.: MPC17901  
 ITEM NO.: HX1065/2065  
 SERVICE: AGR FLASH GAS COOLER  
 REQ'N. NO.: MC213

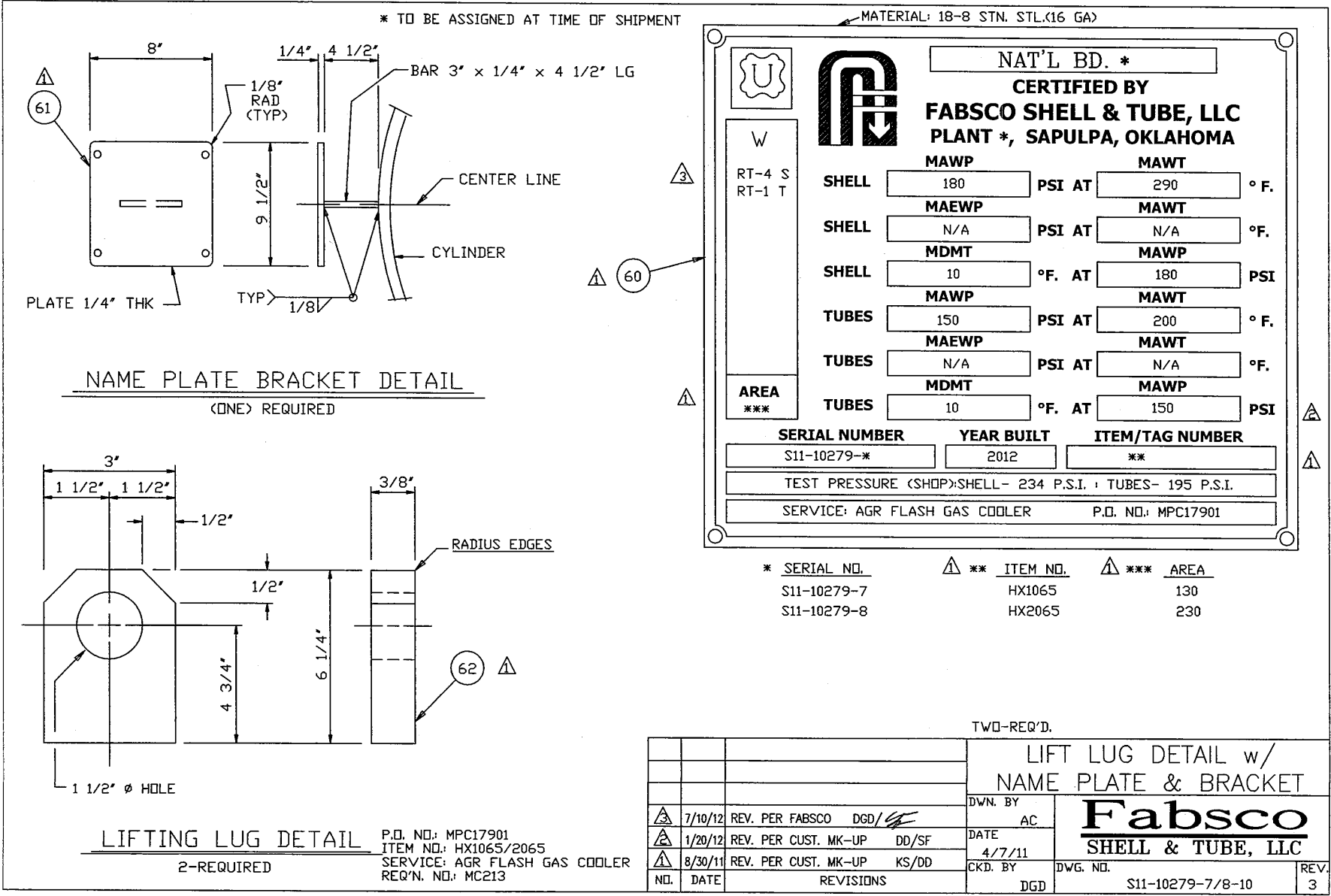
			TUBE BENDING SCHEDULE	
			DWN. BY DS	<b>Fabsco</b> <b>SHELL &amp; TUBE, LLC</b>
			DATE 4- 7-11	
			CKD. BY Dop	DWG.NO. S11-10279-7/8-6
NO.	DATE	REVISIONS		REV. 0

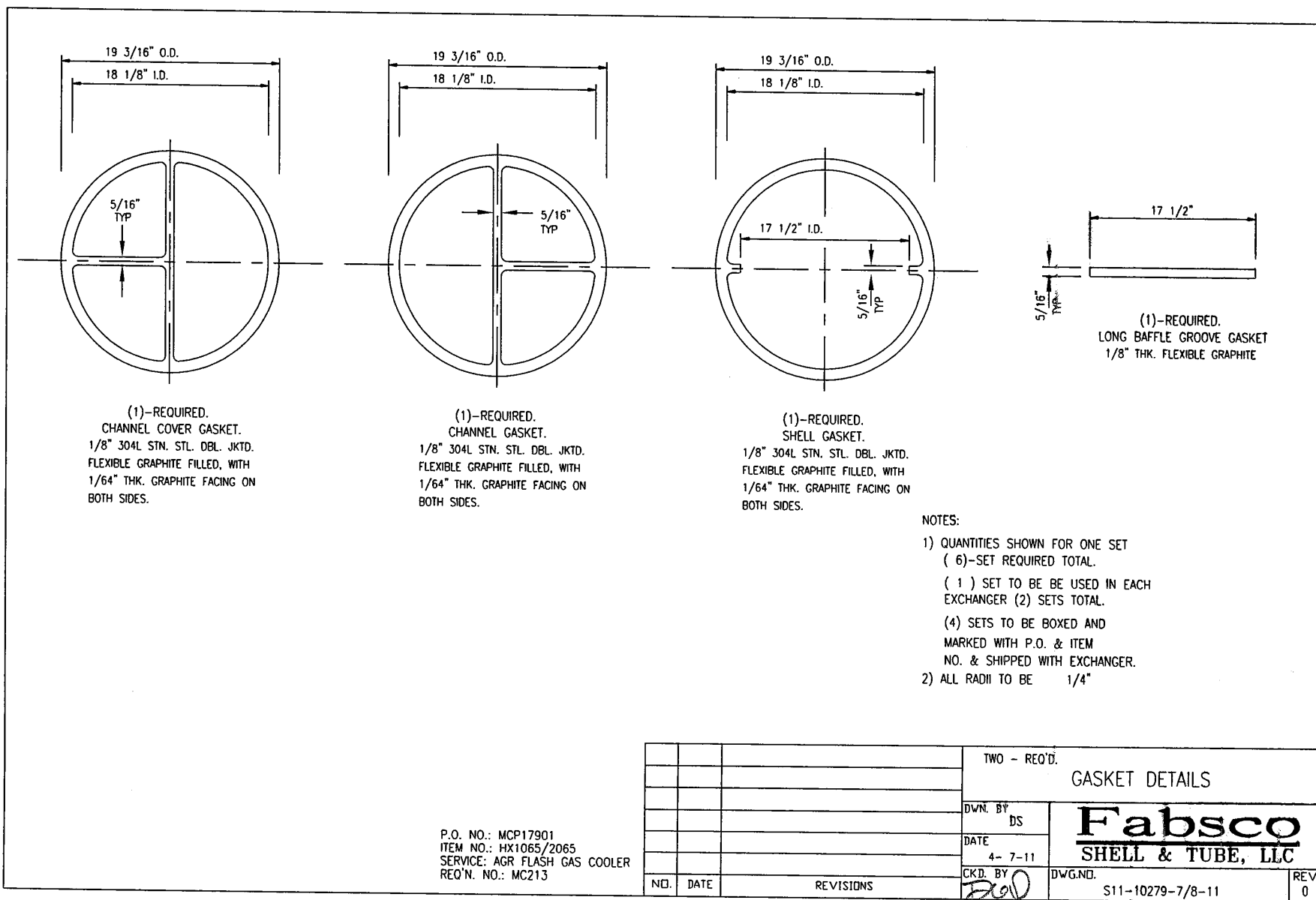








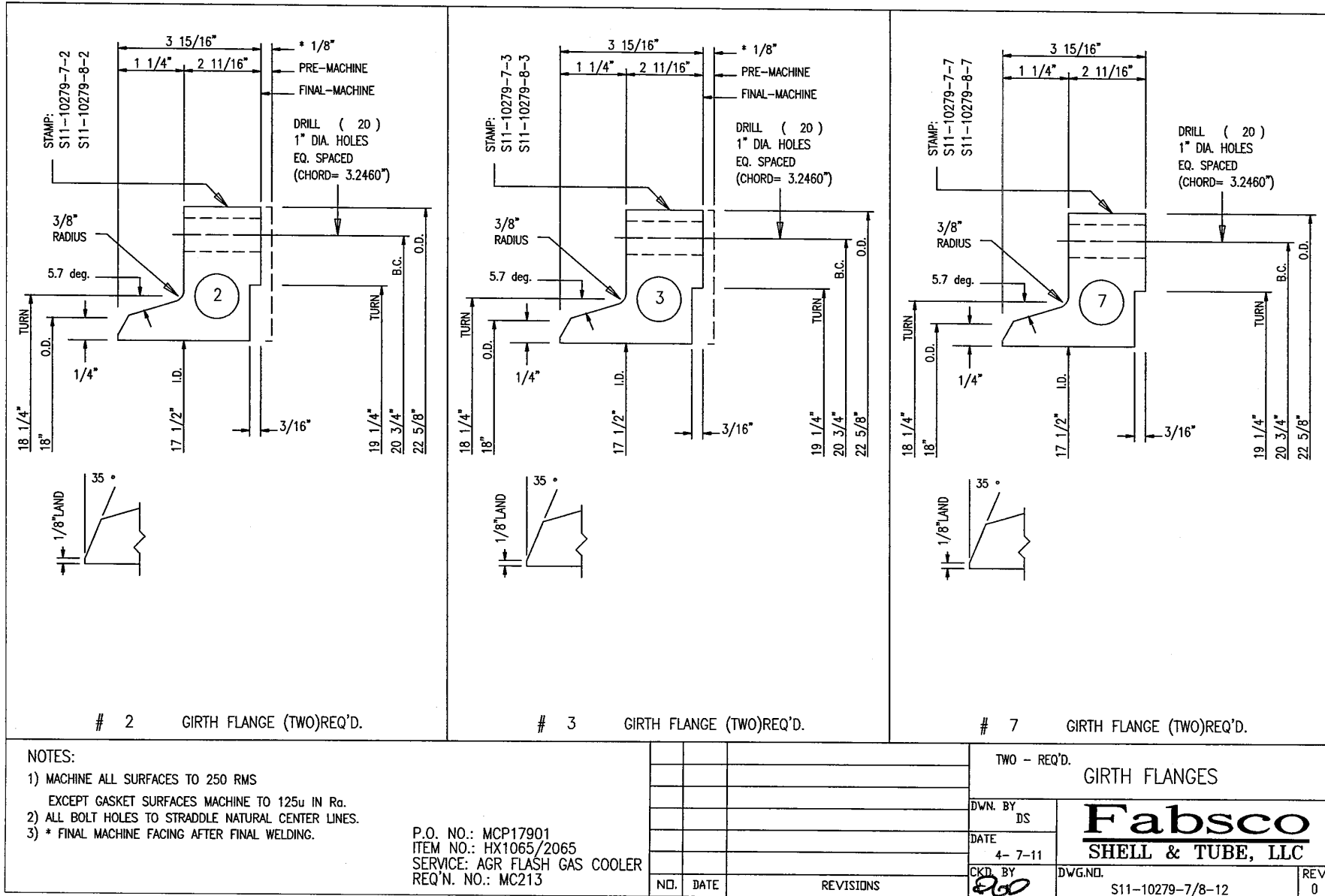


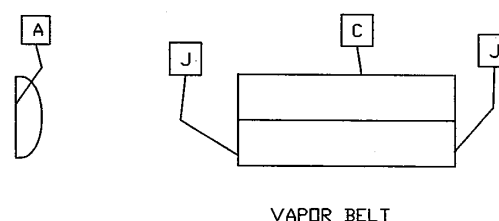
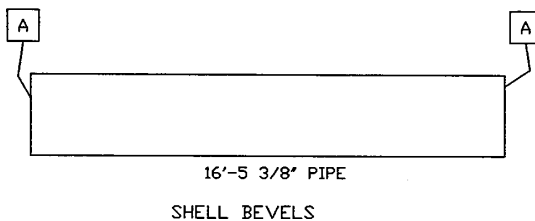
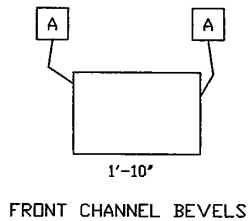
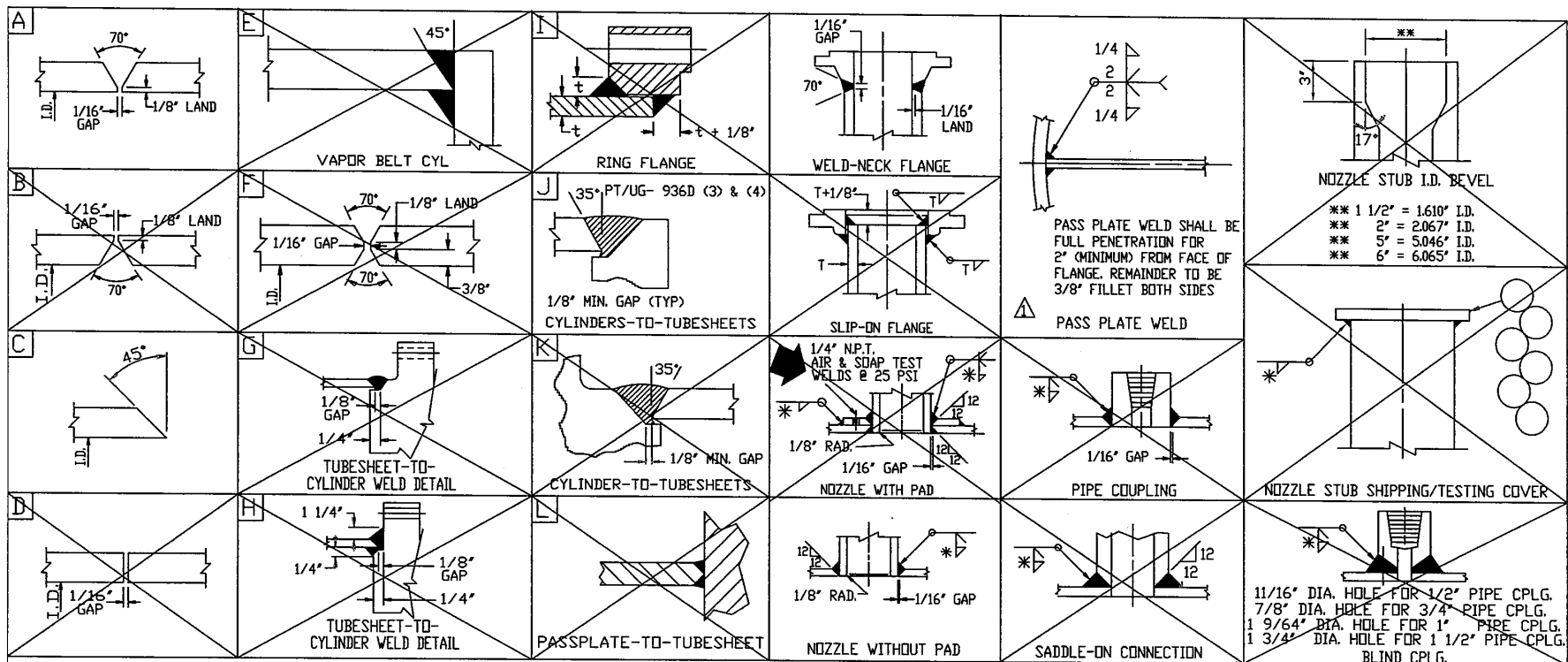


- NOTES:
- 1) QUANTITIES SHOWN FOR ONE SET  
( 6 )-SET REQUIRED TOTAL.  
( 1 ) SET TO BE USED IN EACH EXCHANGER (2) SETS TOTAL.
  - (4) SETS TO BE BOXED AND MARKED WITH P.O. & ITEM NO. & SHIPPED WITH EXCHANGER.
  - 2) ALL RADII TO BE 1/4"

P.O. NO.: MCP17901  
 ITEM NO.: HX1065/2065  
 SERVICE: AGR FLASH GAS COOLER  
 REQ'N. NO.: MC213

			TWO - REQ'D.	
			GASKET DETAILS	
			DWN. BY DS	<div>Fabsco</div> <div>SHELL &amp; TUBE, LLC</div>
			DATE 4-7-11	
			CKD. BY [Signature]	
NO.	DATE	REVISIONS	DWG. NO. S11-10279-7/8-11	REV. 0





P.O. NO: MCP17901  
ITEM NO: HX1065/2065  
SERVICE: AGR FLASH GAS COOLER  
REQ'N: MC213

#### GENERAL NOTES:

- 1) ALL WELDING SHALL BE IN ACCORDANCE WITH ASME CODE (SECTION IX).
- 2) ALL FILLET WELDS SHALL BE 3/8", UNLESS NOTED OTHERWISE.
- 3) LIFTING LUGS SHALL HAVE A 3/8" CONTINUOUS FILLET WELD, UNLESS NOTED OTHERWISE.
- 4) FOR LONG BAFFLE WELD DETAIL, SEE DWG. -3

#### WELD & BEVEL DETAILS

**Fabsco**  
SHELL & TUBE, LLC

NO.	DATE	REV.	PER CUST. MK-UP	KS/	DWN. BY	AC	DATE	CKD. BY	DGD	DWG. NO.	S11-10279-7/8-13	REV.	1

MK.NO.	QUAN.	DESCRIPTION	MATERIAL	SUPPLIER	P.O. NO.	DUE
21	102	TUBES 1" O.D.X 0.0910 THK avg PER BEND SCHEDULE	SA-213TP304L			
		FORGED RING MATERIAL				
2	2	FLG 22 5/8" O.D.X 17 1/2" I.D.X 3 15/16" THK (f)	SA-965-F304L			
3	2	FLG 22 5/8" O.D.X 17 1/2" I.D.X 3 15/16" THK (f)	SA-965-F304L			
7	2	FLG 22 5/8" O.D.X 17 1/2" I.D.X 3 15/16" THK (f)	SA-965-F304L			
		TUBESHEETS & COVERS				
19	2	TUBESHEET 19 1/8" O.D.X 1 3/8" THK (f)	SA-240-304L			
1	2	COVER 22 5/8" O.D.X 1 3/4" THK (f)	SA-240-304L			
		BAFFLES				
22	24	BAFFLES PER DWG. -8	SA-240-304L			
22A	2	PL 17 3/8" X 1/4" X 16'-3" LG LONG BAF.	SA-240-304L			
		HEADS				
18	2	ELLIP HD. 18" O.D.X 3/16" THK(min) W/2" SF	SA-240-304L			
9B	4	VAPOR BELT END PLATE: 24"O.D. X 18 1/8" I.D. X 3/4" THK.	SA-240-304L			
		CYLINDERS				
9	2	SMLS.PIPE 18" X(0.2500" Sch10 )X 16'-5 3/8" LG.	SA-312TP304L			
9A	2	SMLS.PIPE 24" X(0.2500" Sch10 )X 1'-4 1/4" LG.	SA-312TP304L			
4	2	SMLS.PIPE 18" X(0.2500" Sch10 )X 1'-10" LG.	SA-312TP304L			
		NOTES:				
		1) ALL WELD BEVELS PER WELD & BEVEL DETAIL DWG ATTACHED.				
		2) ALL FORMED HEADS TO BE IN ACCORDANCE WITH UCS-79.				
		3) ALL COLD FORMED PRESSURE PARTS, INCLUDING BUT NOT LIMITED TO CYLINDERS & HEADS, TO BE CERTIFIED AS REQ'D. BY ASME SECTION VIII, DIV. 1, para. UG-79(c).				
		4) 125-250u IN RaFINISH ON GASKET SURFACES.				
		5) QUANTITIES SHOWN ARE FOR TWO EXCHANGERS.				
CUSTOMER: SOUTHERN COMPANY GENERATION						
P.O. NO.: MPC17901			REQ'N. NO.: MC213			
ITEM NO.: HX1065/2065			SERVICES: AGR FLASH GAS COOLER			
			BILL OF MATERIAL			
			DWN. BY DS			
			DATE 4/12/11			
			<div style="text-align: center;"> <b>Fabsco</b>  <b>SHELL &amp; TUBE, LLC</b> </div>			
8/30/11	REV. PER CUST. MK-UP	KS/DA	CKD. BY DGD	DWG. NO. S11-10279-7/8-14	REV. 1	

Kemper County MM233798

MK.NO.	QUAN.	DESCRIPTION	MATERIAL	SUPPLIER	P.O. NO.	DUE
		BOLTING & GASKETS				
40	2	LOT OF GASKETS PER DWG. -11				
41	40	7/8" ROUND X 9-THREAD STUDS X 7" LG. 1&2	SA-193B7			
42	40	7/8" ROUND X 9-THREAD STUDS X 9" LG. 3&7	SA-193B7			
43	160	7/8" HEAVY HEX NUTS 9-THREAD	SA-194-2H			
		MISC. MATERIAL				
60	2	STD. NAME PLATE	STN.STL.			
61	2	STD. NAME PL BRKT . PER DWG. -10	SA-240-304L			
62	6	STD. LIFTING LUGS PER DWG. -10	SA-240-304L			
23	377	FT. 1/2" O.D. TIE RODS	304L STN. STL.			
23B	48	1/2" HEAVY HEX TIE ROD NUTS	304L STN. STL.			
23A	377	FT. 3/4" O.D. SPACER TUBING	304L STN. STL.			
22S	8	SEAL BARS 1" X 3/8" THK. X 14'-5 1/2"	304L STN. STL.			
100	4	PULL BOLT PLUGS: 3/4" Ø (10-THD.) X 3" LG.	304L STN. STL.			
9C	4	DIVIDER PLATES: 2 3/4" X 1/4" THK. X 1'-4" LG.	SA-240-304L			
50	2	PASS PLATES 8 1/2" X 1/4" THK X 29 7/8" LG.	SA-240-304L			
51	2	PASS PLATES 17 3/8" X 1/4" THK X 29 7/8" LG.	SA-240-304L			
		SUPPORT MATERIAL				
120	4	PLATE 10" X 3/8" THK X 23 1/4" LG.	SA-240-304L			
121	4	PLATE 16" X 3/8" THK X 14 1/2" LG.	SA-516-70			
122	4	PLATE 6" X 3/8" THK X 16" LG.	SA-516-70			
123	4	PLATE 5 1/8" X 3/8" THK X 10" LG.	SA-516-70			
124	2	5/8" Ø (11 - THD) MACH. BOLT X 2 1/2" LG.	C'ST.			
		W/ (2) WASHERS & (1) HEX. NUT EACH.				
		NOZZLE COVER MATERIAL				
130	2	PL. 15" O.D. X 3/8" THK. W/(12) 9/16" DIA. HOLES	C.-STL.			
		EQUALLY SPACED ON 13" B.C. (8"-300#)				
131	2	GASKET 12 1/2" O.D. X 1/2" THK. (8"-300#)	SOLID RUBBER			
132	12	1/2" HEX HD. ALL-THD. BOLTS X 3 1/2" LG. W/(1) HEX NUT	C.-STL.			
		AND (2) FLAT WASHERS EACH (8"-300#)				
133	2	PL. 12 1/2" O.D. X 3/8" THK. W/ (12) 9/16" DIA. HOLES	C.STL.			
		EQUALLY SPACED ON 10 5/8" B.C. (6"-300#)				
134	2	GASKET 10 1/8" O.D. X 1/2" THK. (6"-300#)	SOLID RUBBER			
135	12	1/2" HEX HD. ALL-THD. BOLT X 3 1/2" LG. W/(1) HEX NUT	C.-STL.			
		AND (2) FLAT WASHERS EACH (6"-300#)				
136	4	ASME 4" - 150 # RF BLIND FLANGE	SA-105			
137	4	GASKET 7" O.D. X 1/2" THK. (4"-150#)	SOLID RUBBER			
138	16	1/2" HEX HD. ALL-THD. BOLTS X 4" LG. W/(1) HEX NUT	C.-STL.			
		AND (2) FLAT WASHERS EACH (4"-150#)				
		PAINT PER DWG.-1A				
			CUSTOMER: SOUTHERN COMPANY GENERATION P.O. NO.: MPC17901 REQ'N. NO.: MC213 ITEM NO.: HX1065/2065 SERVICE: AGR FLASH GAS COOLER			
			BILL OF MATERIAL			
			DWN. BY DS		Fabsco SHELL & TUBE, LLC	
			DATE 5/ 2/11			
			CKD. BY DGD			
			DWG.NO. S11-10279-7/8-16		REV. 3	
			REV. PER FABSCO DD/			
			REV. PER FABSCO KS/DD			
			REV. PER FABSCO DGD/SF			
			REVISIONS			







# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

**Certified Correct for Fabrication**  
**by Fabsco Shell & Tube, LLC**

By: Samuel H. Davis Date: May 2, 2011

R1	Customer: SOUTHERN COMPANY SERVICES	HORIZONTAL
	Service : AGR FLASH GAS COOLER	U-TUBE
	No. Shells: 1 Par. 1 Ser. 1 Surface: 443 Each 443 TotalSq.ft.	
R1	Size: 17 x 192 " Type: A F U	
R1	(51) "U"-Tubes x 1.000 OD x 0.091 Thk BWG x 192 " Lg 1.2500 ROT TRI Pitch	

-Jurisdictional code requirements: Design, Material, Fabrication, Inspection, Testing and Documentation in accordance with the ASME Code Section VIII, Division 1, 2010 Edition, and so stamped.

-Impact test requirements: TUBESIDE Exempt from Impact Testing per UG-20(f), UCS-66(a), UHA-51

(c), Fig. UCS-66, general note (c), and UCS-67.SHELLSIDE Exempt from impact testing per fig ucs-66.1

-Loadings per ASME code para. UG-22 that have been defined by the user, or his designated agent, have been considered in the design process.

-This design has been reviewed for compliance with the requirements of UG-120(d).

	Shell	Tubes
Design Press/Ext	180.0 / 0.0 psig	150.0 / 0.0 psig
Test Pressure	234.0 psig	195.0 psig
Design Temp/MDMT	290 / 10 deg.F	200 / 10 deg.F
Corrosion Allowance	0.0000"	0.0000"
No. of Passes	2	4
Radiograph	RT4-S	100%
Stress Relieve	0 ** deg.F	0 deg.F

\*\* - Exchanger is fabricated using sold 304L Stn. Stl., Requirements Of UOP 4'-11-6, Para. 4.7, Post Weld Heat Treatment does not apply.

R1

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12
3	Corrected cylinder joint efficiency	DS	6-20-12	SHD	6-20-12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

## SHELL SIDE

```
=====
NOM.
SIZE      PRESS.      TEMP.      C.A.      RT      H.T.      TEMA
18.000"   180.0psig      290deg.F   0.0000"   SEE BELOW  0deg.F    R
=====
```

ASME CODE SECTION VIII, DIVISION 1, UG-27 [ CYLINDER ]

=====DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON O.D. FORMULA=====

### SHELL CYL

```
MATERIAL: SA-312TP304L      PRESS = 180.0psig      TEMP = 290deg.F
O.D.      NOM. THK.      RW=0/PIPE=1      CORR.ALLOW      EFFICIENCY
18.0000"   0.2500"      1      0.0000"      0.65
ATM.STRS      DES.STRS      STRS/STRS      MAWP N-C      MAWP CORR.
14200.0psi    14200.0psi    1.000      226.543psig    226.543psig
```

```
P*(RO)
----- = 0.17416" + CORR. ALLOW. = 0.17416" + MTOL. = 0.19904"
S*E+ (.4*P)
*****
```

ASME CODE SECTION VIII, DIVISION 1, UG-32 [ ELLIPTICAL HEAD ]

=====DESIGN CONDITIONS=====ELLIPTICAL HEAD CALCULATIONS ON O.D. FORMULA=====

### SHELL HEAD

```
MATERIAL: SA-240-304L      PRESS = 180.0psig      TEMP = 290deg.F
O.D.      (min) THK.      CORR.ALLOW      EFFICIENCY
18.0000"   0.1875"      0.0000"      1.00
ATM.STRS      DES.STRS      STRS/STRS      MAWP N-C      MAWP CORR.
16700.0psi    16700.0psi    1.000      354.565psig    354.565psig
```

```
P*(RO)      0.0000
----- = 0.09607" + CORR. ALLOW. = 0.09607" + MTOL. = 0.09607"
S*E+ (.9*P)
*****
```

ASME CODE SEC.VIII DIV.I UG-27 [ ANNULAR DISTRIBUTOR CYLINDER ]

=====DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON I.D. FORMULA=====

```
MATERIAL: SA-240-304L      PRESS = 180.0psig      TEMP = 290deg.F
I.D.      NOM. THK.      RW=0/PIPE=1      CORR.ALLOW      EFFICIENCY
23.5000"   0.2500"      0      0.0000"      1.00
ATM.STRS      DES.STRS      STRS/STRS      MAWP N-C      MAWP CORR.
16700.0psi    12950.0psi    1.290      263.962psig    204.689psig
```

```
P*(R+CA)
----- = 0.16469" + CORR. ALLOW. = 0.16469" + MTOL. = 0.16469"
S*E- (.6*P)
```

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12
3	Corrected cylinder joint efficiency	DS	6-20-12	SHD	6-20-12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

## FRONT CHAN.

```
=====
NOM.
SIZE          PRESS.      TEMP.      C.A.      RT      H.T.      TEMA
18.000"       150.0psig      200deg.F  0.0000"   100%      0deg.F    R
=====
ASME CODE SECTION VIII, DIVISION 1, UG-27          [ CYLINDER ]
=====DESIGN CONDITIONS=====CYLINDER CALCULATIONS ON O.D. FORMULA=====
CHAN. CYL
MATERIAL: SA-312TP304L      PRESS = 150.0psig  TEMP = 200deg.F
O.D.          NOM. THK.      RW=0/PIPE=1  CORR.ALLOW  EFFICIENCY
18.0000"       0.2500"          1          0.0000"     1.00
ATM. STRS      DES. STRS      STRS/STRS      MAWP N-C      MAWP CORR.
14200.0psi     14200.0psi     1.000        348.527psig   348.527psig

P*(RO)
----- = 0.09467" + CORR. ALLOW. = 0.09467" + MTOL. = 0.10819"
S*E+ (.4*P)
*****
```

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

NOZZLE REINFORCEMENT CALCULATIONS/ ASME SECTION VIII, DIV.1, UG-37 & UG-16

NOZ NO= 1 6.00" 300 LB RF WN  
[INLET ] SHELL SIDE  
PRESSURE = 180.0psig TEMP = 290.0deg.F

-----CYLINDER-----		-----NOZZLE-----	
SA-312TP304L		SA-312TP304L	
OPER STRS=	14200.0 psi	OPER STRS=	11020.0 psi
ATM STRS =	14200.0 psi	ATM STRS =	14200.0 psi
C.A. =	0.00000"	C.A. =	0.00000"
I.D. =	23.34375"	I.D. =	5.76100"
O.D. =	24.00000"	O.D. =	6.62500"
THK. =	0.32812"	THK. XH=	0.43200"
T =	0.32812"	TN =	0.43200"
TR =	0.15135"	TRN =	0.05375"
ET =	0.17678"	ETN =	0.37825"

MIN STD. WT. PIPE+C.A.= 0.24500" TR+C.A.= 0.15135"  
LIMITS= 11.52200"  
fr1= 0.7761 fr2= 0.7761 fr3= 0.0000 fr4= 0.0000 F= 1.00000  
Min weld sizes: NS= 0.3248" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD. A = 0.90118in^2  
EXCESS IN CYL. A1= 0.98422in^2  
EXCESS IN NOZ. (OUTSIDE) A2= 0.48159in^2 H = 0.82031"  
AREA OF FILLETS = 0.10913in^2  
AREA AVAILABLE = 1.57494in^2

NOZ. NECK WELD= 0.37500" AREA= 0.10913in^2

< AREA REQD-AREA AVAILABLE > = -0.67376in^2

MAWP N-C= 272.0psig LIMITED BY AR  
MAWP COR= 247.0psig LIMITED BY AR

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

NOZZLE REINFORCEMENT CALCULATIONS/ ASME SECTION VIII, DIV.1, UG-37 & UG-16

NOZ NO= 2 8.00" 300 LB RF WN

[OUTLET ] SHELL SIDE

PRESSURE = 180.0psig TEMP = 290.0deg.F

-----CYLINDER-----NOZZLE-----

SA-312TP304L

SA-312TP304L

OPER STRS= 14200.0 psi

OPER STRS= 11020.0 psi

ATM STRS = 14200.0 psi

ATM STRS = 14200.0 psi

C.A. = 0.00000"

C.A. = 0.00000"

I.D. = 23.34375"

I.D. = 7.62500"

O.D. = 24.00000"

O.D. = 8.62500"

THK. = 0.32812"

THK. XH= 0.50000"

T = 0.32812"

TN = 0.50000"

TR = 0.15135"

TRN = 0.06998"

ET = 0.17678"

ETN = 0.43002"

MIN STD. WT. PIPE+C.A.= 0.28175" TR+C.A.= 0.15135"

LIMITS= 15.25000"

fr1= 0.7761 fr2= 0.7761 fr3= 0.0000 fr4= 0.0000 F= 1.00000

Min weld sizes: NS= 0.3248" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD. A = 1.18790in<sup>2</sup>

EXCESS IN CYL. A1= 1.30836in<sup>2</sup>

EXCESS IN NOZ. (OUTSIDE) A2= 0.54751in<sup>2</sup> H = 0.82031"

AREA OF FILLETS = 0.10913in<sup>2</sup>

AREA AVAILABLE = 1.96499in<sup>2</sup>

NOZ. NECK WELD= 0.37500" AREA= 0.10913in<sup>2</sup>

< AREA REQD-AREA AVAILABLE > = -0.77709in<sup>2</sup>

MAWP N-C= 260.0psig LIMITED BY AR

MAWP COR= 238.0psig LIMITED BY AR

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

## NOZZLE REINFORCEMENT CALCULATIONS/ ASME SEC.VIII DIV.I UG-37 & UG-16

NOZ NO=11 4.00" 150 LB RF WN  
[INLET & OUTLET ] FRONT CHANNEL SIDE  
PRESSURE = 150.0psig TEMP = 200.0deg.F

-----CYLINDER-----NOZZLE-----

SA-312TP304L	SA-312TP304L
OPER STRS= 14200.0 psi	OPER STRS= 14200.0 psi
ATM STRS = 14200.0 psi	ATM STRS = 14200.0 psi
C.A. = 0.00000"	C.A. = 0.00000"
I.D. = 17.56250"	I.D. = 4.02600"
O.D. = 18.00000"	O.D. = 4.50000"
THK. = 0.21875"	THK. sc40= 0.23700"
T = 0.21875"	TN = 0.23700"
TR = 0.09467"	TRN = 0.02367"
ET = 0.12408"	ETN = 0.21333"

MIN STD. WT. PIPE+C.A.= 0.20738" TR+C.A.= 0.09467"  
LIMITS= 8.05200"  
fr1= 1.0000 fr2= 1.0000 fr3= 0.0000 fr4= 0.0000 F= 1.00000  
Min weld sizes: NS= 0.2166" NP= 0.0000" PS= 0.0000" IF= 0.0000"

AREA REQD. A = 0.38114in^2  
EXCESS IN CYL. A1= 0.49954in^2  
EXCESS IN NOZ. (OUTSIDE) A2= 0.23333in^2 H = 0.54688"  
AREA OF FILLETS = 0.14062in^2  
AREA AVAILABLE = 0.87350in^2

NOZ. NECK WELD= 0.37500" AREA= 0.14062in^2

< AREA REQD-AREA AVAILABLE > = -0.49236in^2

MAWP N-C= 244.0psig LIMITED BY AR  
MAWP COR= 244.0psig LIMITED BY AR

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SECTION VIII, DIVISION 1, APPENDIX 2 Front Channel to cover flange  
DESIGN CONDITIONS GASKET and BOLTING CALCULATIONS TABLE

Design Pres= 150.0 psig Eff.Gsk OD= 19.1250" No.Passes= 2  
Neg. Pres= None Eff.Gsk ID= 18.1250" N= 0.500"  
Design Temp= 200 deg.F THK= 0.1250" col.=2 b= 0.250  
Flg Matl=SA-182F304L Gsk Matl=304L DJ GRAPH F y= 9000psi  
Stud Matl=SA-193B7 Gsk Face=FLAT FACE w= 0.0000" m= 3.750  
Corr Allow= 0.0000" Wm2= 192824 # Am = 7.9515in^2  
Flange Desn Sfo 14300.0 psi Hp = 24103 # Ab = 8.3800in^2  
Atm. Sfa 16700.0 psi H = 40867 # W = 204143 #  
Bolting Desn Sb 25000.0 psi Wm1= 64970 # Wm1= 64970 #  
Atm. Sa 25000.0 psi Gasket Width Check Nmin = 0.1989"  
CONDITION LOAD x LEVER ARM = MOMENT  
HD = 36079 # hD= 1.43750" MD= 51863 in#  
Operating HG = 24103 # hG= 1.06250" MG= 25609 in#  
HT = 4787 # hT= 1.34375" MT= 6433 in#  
Mo= 83906 in#

Gasket  
Seating HG = 204143 # hG= 1.06250" mo= 216902 in#  
Allow.Stress-STRESS CALCULATIONS-Operating SHAPE CONSTANTS  
1.5 Sfo Long Hub,SH 7329.8 psi K = 1.2929 h/ho= 0.5976  
Sfo Radial Flg,SR 413.8 psi T = 1.8004 F = 0.8352  
Sfo Tang Flg,ST 4901.5 psi Z = 3.9785 V = 0.2948  
Sfo .5(SH+SR)or.5(SH+ST) 6115.7 psi Y = 7.7031 f = 1.0000  
J(APP.2-14)= 0.2571 U = 8.4649 e = 0.3993in^-1  
Allow.Stress-STRESS CALCULATIONS-Gsk.Seating gl/go= 1.5000 d = 3.753in^3  
1.5 Sfa Long Hub,SH 18947.9 psi ho = 2.0917"  
Sfa Radial Flg,SR 1069.7 psi  
Sfa Tang Flg,ST 12670.6 psi  
Sfa .5(SH+SR)or.5(SH+ST) 15809.3 psi  
J(APP.2-14)= 0.6459

OTHER STRESS FORMULA FACTORS  
t 2.3750"  
Alpha 1.9483  
Beta 2.2645  
Gamma 1.0822  
Delta 3.5694  
Lambda 4.6516  
M 4794 #  
m 12394 #

O.D. = 22.6250" THK. = 2.6875"  
I.D. = 17.5000" T-Adder = 0.1250"  
GO = 0.2500" G1 = 0.3750"  
HUB O.D. = 18.2500" HUB LEN = 1.2500"  
HUB ANG = 5.7106deg RIB LENGTH = 18.1250"  
G (MEAN) = 18.6250" G (CALC) = 18.6250"  
G (MIN.) = 18.0000" G (MAX.) = 18.7500"  
B.C. = 20.7500" B.S.C.F. = 1.0000  
No. STUDS = 20 STUD DIAMETER = 0.8750"  
R = 1.2500" E = 0.9375"  
BOLT SPAC = 3.2460" TORQUE = 208ft#  
MIN. SPAC = 2.0625" TEMA MAX. SPAC= 5.1029"  
FLG TURN = 0.1875" FACING = RECESS  
BPRIME = 0.0000" BG1 = 0.0000"  
FLG RWT = 183Lbs FLG FWT = 125Lbs

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12



## Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

Flange stress factor=0.9700

Bolting stress factor=0.9700

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	<b>CORRECTED LOADING CALC PG 27-37</b>	<b>RS</b>	<b>2/9/12</b>	<b>rds</b>	<b>2/9/12</b>

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

## ///// FRONT COVER CALCULATIONS /////

EXCHANGER TYPE....= A F U MATERIAL.....= SA-240-304L  
PRESSURE.....= 150.0 psig CORROSION ALLOWANCE= 0.0000"  
TURN LOW.....= 0.1875" TURN HIGH.....= 0.3125"  
S COVER STRESS...= 16700.0 psi E.....= 1.0  
Sb STRESS BOLTS..= 25000.0 psi Ab.....= 8.3800 in^2  
d.....= 18.6250" C.....= 0.3000  
W.....= 204143 # Wm1.....= 64970 #  
hG (ASME).....= 1.0625" hG (TEMA).....= 1.0625"  
G (TEMA).....= 18.6250" E (MOD.) x (10)-6..= 27.5 psi  
COVER THK SET HOLD= [ NO]

[ PER ASME SECT.VIII UG-34 USING Wm1 (OPERATING) ]

$$THK = d - \frac{\frac{C P}{S E} + \frac{1.9 W m1 hG}{S E d^3}}{1} = 1.1647"$$

[ PER ASME SECT.VIII UG-34 USING W (GASKET SEATING) ]

$$THK = d - \frac{\frac{1.9 W hG}{S E d^3}}{1} = 1.1511"$$

[ PER TEMA RCB-9.2 FLAT CHANNEL COVER ]

MAXIMUM DEFLECTION = 0.0300"

DEFLECTION at 1.5131" THICK = 0.0300"

$$Y = \frac{G}{E (1.562)^3} - \frac{0.0435 G^3 P + 0.5 S b A b hG}{E} = 0.0272"$$

[ COV THK Min ] TEMA Deflection = 1.5131" + 0.1875" = 1.7006"

[ COV THK Min ] CODE Operating = 1.1647" + 0.1875" = 1.3522"

[ COV THK Min ] CODE Gasket Seating = 1.1511" + 0.1875" = 1.3386"

ACTUAL COVER THICKNESS USED (SET HOLD[ NO])= 1.7500"

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Revisions to this Page					
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1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

## Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

MAIN FLG IS FLANGE MARK # 2

CODE 46 MAIN FLG. SET: GASKET WIDTH WILL NOT CONTROL MAWP.  
CODE 46 COMP FLG. SET: GASKET WIDTH WILL NOT CONTROL MAWP.

/// INDIVIDUAL COMPONENTS ///

MAWP CORR. MAIN FLG = 340.0 Limited by: Flange stress.  
MAWP CORR. COVER = 205.0

MAWP N-C MAIN FLG = 397.0 Limited by: Flange stress.  
MAWP N-C COVER = 222.0

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SECTION VIII, DIVISION 1, APPENDIX 2 Front chan to tubesheet flange  
DESIGN CONDITIONS GASKET and BOLTING CALCULATIONS TABLE

Design Pres= 150.0 psig Eff.Gsk OD= 19.1250" No.Passes= 2  
Neg. Pres= None Eff.Gsk ID= 18.1250" N= 0.500"  
Design Temp= 200 deg.F THK= 0.1250" col.=2 b= 0.250  
Flg Matl=SA-182F304L Gsk Matl=304L DJ GRAPH F y= 9000psi  
Stud Matl=SA-193B7 Gsk Face=FLAT FACE w= 0.0000" m= 3.750  
Corr Allow= 0.0000" Wm2= 192824 # Am = 7.9515in^2  
Flange Desn Sfo 14300.0 psi Hp = 24103 # Ab = 8.3800in^2  
Atm. Sfa 16700.0 psi H = 40867 # W = 204143 #  
Bolting Desn Sb 25000.0 psi Wm1= 77964 # Wm1= 77964 #  
Atm. Sa 25000.0 psi Gasket Width Check Nmin = 0.1989"  
CONDITION LOAD x LEVER ARM = MOMENT  
HD = 36079 # hD= 1.43750" MD= 51863 in#  
Operating HG = 37097 # hG= 1.06250" MG= 39415 in#  
HT = 4787 # hT= 1.34375" MT= 6433 in#  
Mo= 97713 in#

Gasket  
Seating HG = 204143 # hG= 1.06250" mo= 216902 in#  
Allow.Stress-STRESS CALCULATIONS-Operating SHAPE CONSTANTS  
1.5 Sfo Long Hub,SH 8535.9 psi K = 1.2929 h/ho= 0.5976  
Sfo Radial Flg,SR 481.9 psi T = 1.8004 F = 0.8352  
Sfo Tang Flg,ST 5708.0 psi Z = 3.9785 V = 0.2948  
Sfo .5(SH+SR)or.5(SH+ST) 7122.0 psi Y = 7.7031 f = 1.0000  
J(APP.2-14)= 0.2994 U = 8.4649 e = 0.3993in^-1  
Allow.Stress-STRESS CALCULATIONS-Gsk.Seating gl/go= 1.5000 d = 3.753in^3  
1.5 Sfa Long Hub,SH 18947.9 psi ho = 2.0917"  
Sfa Radial Flg,SR 1069.7 psi  
Sfa Tang Flg,ST 12670.6 psi  
Sfa .5(SH+SR)or.5(SH+ST) 15809.3 psi  
J(APP.2-14)= 0.6459

OTHER STRESS FORMULA FACTORS  
t 2.3750"  
Alpha 1.9483  
Beta 2.2645  
Gamma 1.0822  
Delta 3.5694  
Lambda 4.6516  
M 5583 #  
m 12394 #

O.D. = 22.6250" THK. = 2.6875"  
I.D. = 17.5000" T-Adder = 0.1250"  
GO = 0.2500" G1 = 0.3750"  
HUB O.D. = 18.2500" HUB LEN = 1.2500"  
HUB ANG = 5.7106deg RIB LENGTH = 18.1250"  
G(MEAN) = 18.6250" G(CALC) = 18.6250"  
G(MIN.) = 18.0000" G(MAX.) = 18.7500"  
B.C. = 20.7500" B.S.C.F. = 1.0000  
No. STUDS = 20 STUD DIAMETER = 0.8750"  
R = 1.2500" E = 0.9375"  
BOLT SPAC = 3.2460" TORQUE = 208ft#  
MIN. SPAC = 2.0625" TEMA MAX. SPAC= 5.1029"  
FLG TURN = 0.1875" FACING = RECESS  
BPRIME = 0.0000" BG1 = 0.0000"  
FLG RWT = 183Lbs FLG FWT = 125Lbs

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

Flange stress factor=0.9700

Bolting stress factor=0.9700

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SECTION VIII, DIVISION 1, APPENDIX 2 Shell to tubesheet flange  
DESIGN CONDITIONS GASKET and BOLTING CALCULATIONS TABLE

Design Pres= 180.0 psig Eff.Gsk OD= 19.1250" No.Passes= 2  
Neg. Pres= None Eff.Gsk ID= 18.1250" N= 0.500"  
Design Temp= 290 deg.F THK= 0.1250" col.=2 b= 0.250  
Flg Matl=SA-182F304L Gsk Matl=304L DJ GRAPH F y= 9000psi  
Stud Matl=SA-193B7 Gsk Face=FLAT FACE w= 0.0000" m= 3.750  
Corr Allow= 0.0000" Wm2= 192824 # Am = 7.9515in^2  
Flange Desn Sfo 12950.0 psi Hp = 28923 # Ab = 8.3800in^2  
Atm. Sfa 16700.0 psi H = 49040 # W = 204143 #  
Bolting Desn Sb 25000.0 psi Wm1= 77964 # Wm1= 77964 #  
Atm. Sa 25000.0 psi Gasket Width Check Nmin = 0.1989"  
CONDITION LOAD x LEVER ARM = MOMENT  
HD = 43295 # hD= 1.43750" MD= 62236 in#  
Operating HG = 28923 # hG= 1.06250" MG= 30731 in#  
HT = 5745 # hT= 1.34375" MT= 7720 in#  
Mo= 100688 in#

Gasket  
Seating HG = 204143 # hG= 1.06250" mo= 216902 in#  
Allow.Stress-STRESS CALCULATIONS-Operating SHAPE CONSTANTS  
1.5 Sfo Long Hub,SH 8795.8 psi K = 1.2929 h/ho= 0.5976  
Sfo Radial Flg,SR 496.6 psi T = 1.8004 F = 0.8352  
Sfo Tang Flg,ST 5881.8 psi Z = 3.9785 V = 0.2948  
Sfo .5(SH+SR)or.5(SH+ST) 7338.8 psi Y = 7.7031 f = 1.0000  
J(APP.2-14)= 0.3137 U = 8.4649 e = 0.3993in^-1  
Allow.Stress-STRESS CALCULATIONS-Gsk.Seating gl/go= 1.5000 d = 3.753in^3  
1.5 Sfa Long Hub,SH 18947.9 psi ho = 2.0917"  
Sfa Radial Flg,SR 1069.7 psi  
Sfa Tang Flg,ST 12670.6 psi  
Sfa .5(SH+SR)or.5(SH+ST) 15809.3 psi  
J(APP.2-14)= 0.6459

OTHER STRESS FORMULA FACTORS  
t 2.3750"  
Alpha 1.9483  
Beta 2.2645  
Gamma 1.0822  
Delta 3.5694  
Lambda 4.6516  
M 5753 #  
m 12394 #

O.D. = 22.6250" THK. = 2.6875"  
I.D. = 17.5000" T-Adder = 0.1250"  
GO = 0.2500" G1 = 0.3750"  
HUB O.D. = 18.2500" HUB LEN = 1.2500"  
HUB ANG = 5.7106deg RIB LENGTH = 18.1250"  
G (MEAN) = 18.6250" G (CALC) = 18.6250"  
G (MIN.) = 18.0000" G (MAX.) = 18.7500"  
B.C. = 20.7500" B.S.C.F. = 1.0000  
No. STUDS = 20 STUD DIAMETER = 0.8750"  
R = 1.2500" E = 0.9375"  
BOLT SPAC = 3.2460" TORQUE = 208ft#  
MIN. SPAC = 2.0625" TEMA MAX. SPAC= 5.1029"  
FLG TURN = 0.1875" FACING = RECESS  
BPRIME = 0.0000" BG1 = 0.0000"  
FLG RWT = 183Lbs FLG FWT = 125Lbs

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

Flange stress factor=0.9700

Bolting stress factor=0.9700

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Revisions to this Page					
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2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

—

MAIN FLG IS FLANGE MARK # 3  
COMP FLG IS FLANGE MARK # 7

CODE 46 MAIN FLG. SET: GASKET WIDTH WILL NOT CONTROL MAWP.  
CODE 46 COMP FLG. SET: GASKET WIDTH WILL NOT CONTROL MAWP.

## /// INDIVIDUAL COMPONENTS ///

MAWP CORR. MAIN FLG = 340.0 Limited by: Flange stress.  
MAWP CORR. COMP FLG = 308.0 Limited by: Flange stress.  
MAWP CORR. T.S. [STD DIA]= 164.0

MAWP N-C MAIN FLG = 397.0 Limited by: Flange stress.  
MAWP N-C COMP FLG = 397.0 Limited by: Flange stress.  
MAWP N-C T.S. [STD DIA]= 164.0

## /// AS MATING FLANGES (NO VACCUUM INCLUDED) ///

MAWP CORR. MAIN FLG = 308.0  
MAWP CORR. COMP FLG = 308.0 Limited by: Flange stress.

MAWP N-C MAIN FLG = 397.0 Limited by: Flange stress.  
MAWP N-C COMP FLG = 397.0 Limited by: Flange stress.

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Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12



# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

## \*\*\*\*\* TUBE CALCULATIONS \*\*\*\*\*

Tube size.....=	1.00000"	Straight or U-Tube.=	U-Tube
Internal Press...=	150.00000psig	External press....=	180.00000psig
Temperature used.=	290.00000deg.	Tube Matl MT#123...=	SA-213TP304L
Stress Oper.....=	12950.00000psi	Stress Atm.....=	16700.00000psi
Yield.....=	19400.00195psi	Inside Radius.....=	0.40900"
Joint Eff.....=	1.00000	Mill tol...= 0.00% =	0.00000"
Wall Thk. t.....=	0.09100"	Tube gauge.....=	
Mean Rad., R.....=	1.50000"	Tube Length.....=	192.00000"
Corrosion ID.....=	0.00000"	Corrosion OD.....=	0.00000"
Thinning factor..=	1.16667		
tr = PR/SE - .6P =	0.00477"	(tr*TF)+MT+CA.....=	0.00557"
MAWP Internal H/C=	2216.10352psig	MAWP Internal N/C..=	2857.83228psig
MAWP External H/C=	956.56287psig	MAWP External N/C..=	1309.67029psig

## \*\*\*\* ASME PRESSURE/TEMPERATURE RATING \*\*\*\*

----- SHELL NOZZLES -----DESIGN PRESS= 180.0 DESIGN TEMP= 290.0  
 NOZ# 1 6.0- 300# MT# FLG= 206 GROUP=2.3 @70deg.= 600.00 @TEMP= 460.50  
 NOZ# 2 8.0- 300# MT# FLG= 206 GROUP=2.3 @70deg.= 600.00 @TEMP= 460.50

----- FRONT CHANNEL NOZZLES -----DESIGN PRESS= 150.0 DESIGN TEMP= 200.0  
 NOZ#11 4.0- 300# MT# FLG= 206 GROUP=2.3 @70deg.= 600.00 @TEMP= 510.00  
 NOZ#12 4.0- 300# MT# FLG= 206 GROUP=2.3 @70deg.= 600.00 @TEMP= 510.00

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Revisions to this Page					
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1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

/// TEMA PASS PLATE CALCULATIONS ///

Per RCB-9.132

$$t = b \sqrt{\frac{q B}{1.5 S}}$$

TUBE SIDE PRESSURE DROP  $q = 10.00000$

FRONT PASS PLATE THK..... = 0.25000

MATERIAL..... = SA-240-304L

STRESS OPER..... = 16700.00

FRONT Chan ID..... = 17.50000

FRONT Chan OAL..... = 21.62500

a..... = 20.18750

b..... = 17.50000

TABLE RCB-9.132..... = Long sides fixed

a/b..... = 1.15357

B..... = 0.45229

t..... = 0.23515

Min fillet leg [both sides] =  $3/4t = .75 \times 0.23515 = 0.17636$

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1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

## ===== EXCHANGER DATA =====

```

//////// DRY WEIGHTS [Pounds] //////////
EXCHANGER.=      4086.55  ROUGH WT.=      4440.31  BUNDLE....=      2175.25
SHELL.....=      1107.23  FRONT CH.=      548.40  REAR CH...=          0.09
FLTG. HEAD=          0.00  BOLTS.....=      77.57
BOTT SUPTS=      143.03  TOP SUPTS.=          0.00
FC P-PLS..=       34.97  RC P-PLS..=          0.00  FH P-PLS..=          0.00

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//////// WET WEIGHTS [Pounds] //////////
EXCHANGER FLOODED WITH WATER TUBES AND SHELL SIDE.....=      5901.34
EXCHANGER FLOODED WITH WATER TUBE SIDE.....=      4645.43
EXCHANGER FLOODED WITH WATER SHELL SIDE.....=      5342.46
WTR.TOTAL.=      1814.79  WTR.TUBES.=      558.88  WTR.SHELL.=      1255.91

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//////// CENTER OF GRAVITIES [Inches from outline reference] //////////
EXCHANGER DRY.....=      72.81
BUNDLE DRY .....=      92.91
SHELL DRY.....=      81.19
FRONT CHANNEL DRY.....=     -13.51
REAR CHANNEL DRY.....=          0.00
EXCHANGER FLOODED WITH WATER TUBES AND SHELL SIDE.....=      78.74
EXCHANGER FLOODED WITH WATER TUBE SIDE.....=      71.99
EXCHANGER FLOODED WITH WATER SHELL SIDE.....=      80.07

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```

//////// WEIGHT ON SUPPORTS [Pounds] //////////
EXCH DRY.....[LEFT]=      2901.05 [RIGHT]=      1185.50
EXCH FLOODED W/ WTR TUBES & SHELL.[LEFT]=      3898.04 [RIGHT]=      2003.30
EXCH FLOODED W/ WTR TUBE SIDE....[LEFT]=      3329.69 [RIGHT]=      1315.74
EXCH FLOODED W/ WTR SHELL SIDE....[LEFT]=      3469.40 [RIGHT]=      1873.06

```

```

//////// SURFACE AREAS [Square Feet] //////////
TUBES OUTSIDE GROSS.....=      446.99
TUBES OUTSIDE EFFECTIVE.....=      443.38
EXCHANGER.....=      122.03
SHELL.....=      89.80
FRONT CHANNEL.....=      18.46
REAR CHANNEL.....=          0.00
FRONT TUBESHEET [Tubshts included to shell if fixed tubesheet]=      0.63
BOTTOM SUPPORTS.....=      13.15
TOP SUPPORTS.....=          0.00

```

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# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SEC.VIII DIV.1 UHX-12

(U-Tube) Configuration d

No or EPC=Elastic-Plastic	Calcs SSC=Simply Supported Calcs	=NO
MT Tubes	=SA-213TP304L MT Tubesheets	=SA-240-304L
MT Channel	=SA-312TP304L MT Shell	=SA-312TP304L
Diameter (Do)	= 17.1250"	Pass Lane Area (AL) = 74.328in^2
Perimeter of layout (CP)	= 0.0000"	Area enclosed by CP (AP) = 0.000in^2
Tubesheet thickness (h)	= 1.3750"	Tubesheet thickness HOLD = NO
Tube Pitch (p)	= 1.2500"	Pitch type = Rot Triangle (30 Deg)
Diameter of tube (dt)	= 1.0000"	Tube thickness (tt) = 0.0910"
Exp length of tube (ltx)	= 0.0000"	Exp depth ratio (Rho) = 0.9500
Groove channel side (hg)	= 0.1875"	Groove shell side = 0.0000"
Channel I.D. (Dc)	= 17.5000"	Channel thickness (Tc) = 0.2500"
Shell I.D. (Ds)	= 17.5000"	Shell thickness (Ts) = 0.2500"
TS outside diameter (A)	= 19.1250"	Bolt circle (C) = 20.7500"
G Channel side (Gc or G1)	= 18.6250"	G Shell side (Gs or G1) = 18.6250"
Bolt load Chan side (Wc)	=204143.89#	Bolt load Sh side (Ws) =204143.89#
1.8*Sqrt(Dc*Tc)	= 0.0000"	1.8*Sqrt(Ds*(Ts or Ts1)) = 0.0000"
Thickness at Top TS (tr)	= 1.0000"	Outer Tube Circle = 17.1250"
hr Ch Op= 0.0000" Ch Atm= 0.0000"		Sh Op= 0.0000" Sh Atm = 0.0000"
Tubes welded Backside TS =NO		Hole Size in Tubesheet = 0.0000"
Design Press Chan (Pt)	= 150.0psig	Design Press Shell (Ps) = 180.0psig
Design Temp Chan	= 200.0deg.	Design Temp Shell = 290.0deg.
Corr Chan	= 0.00000"	Corr Shell = 0.00000"
Corr Chan side TS (ct)	= 0.00000"	Corr Shell side TS (cs) = 0.00000"

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# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SEC.VIII DIV.1 UHX-12 Case 1

(U-Tube) Configuration d

Tubesheet Thickness (h) = 1.3750"  
 Exp length of tube (ltx) = 1.3062" Exp depth ratio (Rho) = 0.9500"  
 Channel I.D. (Dc) = 17.5000" Channel thickness (Tc) = 0.2500"  
 Shell I.D. (Ds) = 17.5000" Shell thickness (Ts) = 0.2500"

Press Chan (Pt) = 150.0psi Press Shell (Ps) = 0.0p  
 Corr Chan = 0.00000" Corr Shell = 0.00000"  
 Corr Chan side TS (ct) = 0.00000" Corr Shell side TS (cs) = 0.00000"

	Temp	Stress	Yield	Modulus x10 <sup>6</sup>	Coef x10 <sup>-6</sup>	Poiss Ratio
Temp TS	T = 290deg.F	S = 16700psi	Sy = 0psi	E = 27.0500		V = .000
Temp TBS	Tt = 290deg.F	St = 12950psi	Syt = 0psi	Et = 27.0500		Vt = .000
Temp CH	Tc = 200deg.F	Sc = 14200psi	Syc = 21400psi	Ec = 27.5000		Vc = .300
Temp SH	Ts = 290deg.F	Ss = 14200psi	Sys = 19400psi	Es = 27.0500		Vs = .300

Mu = 0.2000 d\* = 0.8659" P\* = 1.5189" Mu\* = 0.4299 hg' = 0.1875" hgs' =  
 Wmax = 204143.8906# Rho s = 1.087591290  
 Rho c = 1.087591290 Mts = -525.6760864In-Lb/In h/p = 1.10000  
 E\* = 13253880.00psi v\* = 0.288465917 Beta s =  
 Kappa s = 0.0# Lambda s = 0.0psi Delta s =  
 Delta c = 0.0In<sup>3</sup>/Lb Beta c = 0.0in<sup>-1</sup> Kappa c =  
 Lambda c = 0.0psi Omega s = 0.0in<sup>2</sup> Omega c =  
 K = 1.116788268 F = 0.160403192 M\* = -525.676  
 Mp = -262.9884338In-Lb/In Mo = -2523.286865In-Lb/In M = 2523.28  
 Sigma s,m = 0.0psi Sigma s,b = 0.0psi Sigma s =  
 Sigma c,m = 0.0psi Sigma c,b = 0.0psi Sigma c =  
 Sigma 24974 psi <= 2S 33400 psi [OK]  
 Tau 2335 psi <= .8S 13360 psi [OK]

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# Mechanical Design Calculations

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REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SEC.VIII DIV.1 UHX-12 Case 2

(U-Tube) Configuration d

Tubesheet Thickness (h) = 1.3750"  
 Exp length of tube (ltx) = 1.3062" Exp depth ratio (Rho) = 0.9500"  
 Channel I.D. (Dc) = 17.5000" Channel thickness (Tc) = 0.2500"  
 Shell I.D. (Ds) = 17.5000" Shell thickness (Ts) = 0.2500"

Press Chan (Pt) = 0.0psi Press Shell (Ps) = 180.0p  
 Corr Chan = 0.00000" Corr Shell = 0.00000"  
 Corr Chan side TS (ct) = 0.00000" Corr Shell side TS (cs) = 0.00000"

	Temp	Stress	Yield	Modulus x10 <sup>6</sup>	Coef x10 <sup>-6</sup>	Poiss Ratio
Temp TS	T = 290deg.F	S = 16700psi	Sy = 0psi	E = 27.0500		V = .000
Temp TBS	Tt = 290deg.F	St = 12950psi	Syt = 0psi	Et = 27.0500		Vt = .000
Temp CH	Tc = 200deg.F	Sc = 14200psi	Syc = 21400psi	Ec = 27.5000		Vc = .300
Temp SH	Ts = 290deg.F	Ss = 14200psi	Sys = 19400psi	Es = 27.0500		Vs = .300

Mu = 0.2000 d\* = 0.8659" P\* = 1.5189" Mu\* = 0.4299 hg' = 0.1875" hgs' =  
 Wmax = 204143.8906# Rho s = 1.087591290  
 Rho c = 1.087591290 Mts = 630.8112793In-Lb/In h/p = 1.10000  
 E\* = 13253880.00psi v\* = 0.288465917 Beta s =  
 Kappa s = 0.0# Lambda s = 0.0psi Delta s =  
 Delta c = 0.0In<sup>3</sup>/Lb Beta c = 0.0in<sup>-1</sup> Kappa c =  
 Lambda c = 0.0psi Omega s = 0.0in<sup>2</sup> Omega c =  
 K = 1.116788268 F = 0.160403192 M\* = 630.811  
 Mp = 315.5860901In-Lb/In Mo = 3027.944336In-Lb/In M = 3027.94  
 Sigma s,m = 0.0psi Sigma s,b = 0.0psi Sigma s =  
 Sigma c,m = 0.0psi Sigma c,b = 0.0psi Sigma c =  
 Sigma 29969 psi <= 2S 33400 psi [OK]  
 Tau 2802 psi <= .8S 13360 psi [OK]

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# Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

ASME SEC.VIII DIV.1 UHX-12 Case 3

(U-Tube) Configuration d

Tubesheet Thickness (h) = 1.3750"  
 Exp length of tube (ltx) = 1.3062" Exp depth ratio (Rho) = 0.9500"  
 Channel I.D. (Dc) = 17.5000" Channel thickness (Tc) = 0.2500"  
 Shell I.D. (Ds) = 17.5000" Shell thickness (Ts) = 0.2500"

Press Chan (Pt) = 150.0psi Press Shell (Ps) = 180.0p  
 Corr Chan = 0.00000" Corr Shell = 0.00000"  
 Corr Chan side TS (ct) = 0.00000" Corr Shell side TS (cs) = 0.00000"  
 Temp Stress Yield Modulus Coef Poiss  
 x10<sup>6</sup> x10<sup>-6</sup> Ratio

Temp TS T = 290deg.F S = 16700psi Sy = 0psi E = 27.0500 V = .000  
 Temp TBS Tt = 290deg.F St = 12950psi Syt = 0psi Et = 27.0500 Vt = .000  
 Temp CH Tc = 200deg.F Sc = 14200psi Syc = 21400psi Ec = 27.5000 Vc = .300  
 Temp SH Ts = 290deg.F Ss = 14200psi Sys = 19400psi Es = 27.0500 Vs = .300

Mu = 0.2000 d\* = 0.8659" P\* = 1.5189" Mu\* = 0.4299 hg' = 0.1875" hgs' =  
 Wmax = 204143.8906# Rho s = 1.087591290  
 Rho c = 1.087591290 Mts = 105.1352158In-Lb/In h/p = 1.10000  
 E\* = 13253880.00psi v\* = 0.288465917 Beta s =  
 Kappa s = 0.0# Lambda s = 0.0psi Delta s =  
 Delta c = 0.0In<sup>3</sup>/Lb Beta c = 0.0in<sup>-1</sup> Kappa c =  
 Lambda c = 0.0psi Omega s = 0.0in<sup>2</sup> Omega c =  
 K = 1.116788268 F = 0.160403192 M\* = 105.135  
 Mp = 52.59768295In-Lb/In Mo = 504.6573792In-Lb/In M = 504.657  
 Sigma s,m = 0.0psi Sigma s,b = 0.0psi Sigma s =  
 Sigma c,m = 0.0psi Sigma c,b = 0.0psi Sigma c =  
 Sigma 4994 psi <= 2S 33400 psi [OK]  
 Tau 467 psi <= .8S 13360 psi [OK]

////////// Recap of all Cases //////////

(Case Number 1)OK  
 (Case Number 2)OK  
 (Case Number 3)OK  
 (hr Calc StatTS)OK  
 (hr Calc StatTS)..

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## Mechanical Design Calculations

**Item/Tag No: HX1065/2065**  
**REQ. No: MC213**

**Doc. No: S11-10279-7/8-CP**

**P.O. No: MPC17901**

### Input Values:

Wind Design Code	ASCE-7 98/02/05/IBC-03/STS-1	
Basic Wind Speed	[V]	100.00 mile/hr
Surface Roughness Category	C: Open Terrain	
Importance Factor		1.15
Type of Surface	Moderately Smooth	
Base Elevation		396.00 in
Percent Wind for Hydrotest		33.0
Using User defined Wind Press. Vs Elev.		N
Height of Hill or Escarpment	H or Hh	0.0000 in
Distance Upwind of Crest	Lh	0.0000 in
Distance from Crest to the Vessel	x	0.0000 in
Type of Terrain ( Hill, Escarpment )		Flat
Damping Factor (Beta) for Wind (Ope)		0.0100
Damping Factor (Beta) for Wind (Empty)		0.0000
Damping Factor (Beta) for Wind (Filled)		0.0000

### Wind Analysis Results

#### Static Gust-Effect Factor, Operating Case [G]:

$= \min(0.85, 0.925((1 + 1.7 * gQ * Izbar * Q)/(1 + 1.7 * gV * Izbar)))$   
 $= \min(0.85, 0.925((1 + 1.7 * 3.400 * 0.228 * 0.959)/(1 + 1.7 * 3.400 * 0.228)))$   
 $= \min(0.85, 0.904)$   
 $= 0.850$

Natural Frequency of Vessel (Operating)	33.000 Hz
Natural Frequency of Vessel (Empty)	33.000 Hz
Natural Frequency of Vessel (Test)	33.000 Hz

Note: Per Section 1609 of IBC 2003/06/09 these results are also applicable for the determination of Wind Loads on structures (1609.1.1).

User Entered Importance Factor is	1.150
Force Coefficient	[Cf] 0.624
Structure Height to Diameter ratio	11.277

This is classified as a rigid structure. Static analysis performed.

### Sample Calculation for the First Element

The ASCE code performs all calculations in Imperial Units only. The wind pressure is therefore computed in these units.

Value of [Alpha] and [Zg]:

Exposure Category: C from Table C6-2  
Alpha = 9.500 : Zg = 10800.000 in

Effective Height [z]:

= Centroid Height + Vessel Base Elevation  
= 19.750 + 396.000 = 415.750 in  
= 34.646 ft. Imperial Units

Velocity Pressure coefficient evaluated at height z [Kz]:

Because z (34.646 ft.) > 15 ft.  
 $= 2.01 * (z / Zg)^{2 / Alpha}$   
 $= 2.01 * (34.646 / 900.000)^{2 / 9.500}$   
= 1.012

Type of Hill: No Hill

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## Mechanical Design Calculations

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**Doc. No: S11-10279-7/8-CP**

**P.O. No: MPC17901**

Wind Directionality Factor [Kd]:

= 0.95 per [6-6 ASCE-7 98] [6-4 ASCE-7 02/05]

As there is No Hill Present: [Kzt]:

K1 = 0, K2 = 0, K3 = 0

Topographical Factor [Kzt]:

= ( 1 + K1 \* K2 \* K3 )<sup>2</sup>

= ( 1 + 0.000 \* 0.000 \* 0.000 )<sup>2</sup>

= 1.0000

Velocity Pressure evaluated at height z, Imperial Units [qz]:

= 0.00256 \* Kz \* Kzt \* Kd \* I \* Vr(mph)<sup>2</sup>

= 0.00256 \* 1.012 \* 1.000 \* 0.950 \* 1.150 \* 100.000<sup>2</sup>

= 28.3 psf

Force on the first element [F]:

= qz \* G \* Cf \* WindArea

= 28.317 \* 0.850 \* 0.624 \* 0.454

= 6.8 lbf

Element	Hgt (z) in	K1	K2	K3	Kz	Kzt	qz psf
S11-10279-7/8	415.8	0.000	0.000	0.000	1.012	1.000	28.317
S11-10279-7/8	415.8	0.000	0.000	0.000	1.012	1.000	28.317
S11-10279-7/8	415.8	0.000	0.000	0.000	1.012	1.000	28.317

### Wind Loads on Masses/Equipment/Piping

ID	Wind Area in <sup>2</sup>	Elevation in	Pressure psf	Force lbf
WGHT:[1 OF 1]	1152.00	468.81	28.32	226.54

### Wind Load Calculation

From	To	Wind Height in	Wind Diameter in	Wind Area in <sup>2</sup>	Wind Pressure psf	Element Wind Load lbf
10	20	415.750	21.6000	65.4103	28.3171	6.81980
20	30	415.750	21.6000	4263.30	28.3171	671.036
30	40	415.750	21.6000	65.4103	28.3171	6.81980

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## Mechanical Design Calculations

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**REQ. No: MC213**

**Doc. No: S11-10279-7/8-CP**

**P.O. No: MPC17901**

### Input Values:

#### Earthquake Analysis Results per ASCE 7-2005

Short-period site coefficient 11.4-1	Fa: 1.200
Long -period site coefficient 11.4-2	Fv: 1.700
Maximum Mapped Acceleration Value for Short Periods	Ss: 0.200
Maximum Mapped Acceleration Value for 1 Sec. Period	S1: 1.000
Response Modification Factor	R: 3.000
Importance Factor	Ie: 1.250
Site Class	C

#### Seismic Analysis Results:

$Sms = Fa * Ss = 1.200 * 0.200 = 0.240$   
 $Sml = Fv * S1 = 1.700 * 1.000 = 1.700$   
 $Sds = 2/3 * Sms = 2/3 * 0.240 = 0.160$

$Sds = \text{Max}(0.8 * Sds, SdsUser)$   
 $= \text{Max}(0.128, 0.163)$   
 $= 0.163$

$Sd1 = 2/3 * Sml = 2/3 * 1.700 = 1.133$

$Sd1 = \text{Max}(0.8 * Sd1, Sd1User)$   
 $= \text{Max}(0.907, 0.099)$   
 $= 0.907$

#### Check Approximate Fundamental Period from 12.8-7 [Ta]:

$= Ct * hn^x$  where  $Ct = 0.020$ ,  $x = 0.75$  and  $hn$  = Structural Height (ft.)  
 $= 0.020 * (1.4583^{0.75})$   
 $= 0.027$  seconds

The Coefficient  $Cu$  from Table 12.8-1 is : 1.400

#### Fundamental Period (1/Frequency) [T]:

$= (1/\text{Natural Frequency}) = (1/33.000)$   
 $= 0.030$

#### Check the Value of T which is the smaller of $Cu * Ta$ and T:

= Minimum Value of  $(1.400 * 0.027, 0.030)$  per 12.8.2  
 $= 0.030$

As the time period is < 0.06 second, use section 15.4.2.

#### Compute the Base Shear per equation 15.4-5, [V]:

$= 0.3 * Sds * W * I$   
 $= 0.3 * 0.163 * 7181 * 1.25$   
 $= 438.974$  lbf

Note: Loads multiplied by the Scalar multiplier value of 0.7000

Final Base Shear,  $V = 307.28$  lbf

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# Mechanical Design Calculations

**Item/Tag No: HX1065/2065**  
**REQ. No: MC213**

**Doc. No: S11-10279-7/8-CP**

**P.O. No: MPC17901**

## Earthquake Load Calculation

From	To	Earthquake Height in	Earthquake Weight lbf	Element Ope Load lbf
10	20	8.75000	1436.32	61.4564
20	Sadl	8.75000	1436.32	61.4564
Sadl	30	8.75000	1436.32	61.4564
20	30	8.75000	1436.32	61.4564
30	40	8.75000	1436.32	61.4564

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# Mechanical Design Calculations

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**P.O. No: MPC17901**

## ASME Horizontal Vessel Analysis: Stresses for the Left Saddle

(per ASME Sec. VIII Div. 2 based on the Zick method.)

Horizontal Vessel Stress Calculations : Operating Case

### Input and Calculated Values:

Vessel Mean Radius	Rm	11.81	in
Stiffened Vessel Length per	L	182.06	in
Distance from Saddle to Vessel tangent	a	29.00	in
Saddle Width	b	5.50	in
Saddle Bearing Angle	theta	118.00	degrees
Wear Plate Width	b1	10.00	in
Wear Plate Bearing Angle	thetal	145.00	degrees
Wear Plate Thickness	tr	0.3750	in
Wear Plate Allowable Stress	Sr	17100.00	psi
Inside Depth of Head	h2	5.81	in
Shell Allowable Stress used in Calculation		17100.00	psi
Head Allowable Stress used in Calculation		20000.00	psi
Circumferential Efficiency in Plane of Saddle		1.00	
Circumferential Efficiency at Mid-Span		1.00	
Saddle Force Q, Operating Case		5135.57	lbf

Horizontal Vessel Analysis Results:	Actual	Allowable	
Long. Stress at Top of Midspan	4415.89	17100.00	psi
Long. Stress at Bottom of Midspan	5349.11	17100.00	psi
Long. Stress at Top of Saddles	6493.53	17100.00	psi
Long. Stress at Bottom of Saddles	3991.58	17100.00	psi
Tangential Shear in Shell	912.56	13680.00	psi
Circ. Stress at Horn of Saddle	941.96	21375.00	psi
Circ. Compressive Stress in Shell	119.74	17100.00	psi

### Intermediate Results: Saddle Reaction Q due to Wind or Seismic

Saddle Reaction Force due to Wind Ft [Fwt]:

$$= F_{tr} * (F_t / \text{Num of Saddles} + Z \text{ Force Load}) * B / E$$

$$= 3.00 * (766.2/2 + 0) * 19.7500/20.2506$$

$$= 1120.9 \text{ lbf}$$

Saddle Reaction Force due to Wind Fl or Friction [Fwl]:

$$= \text{Max}(F_l, \text{Friction Load, Sum of X Forces}) * B / L_s$$

$$= \text{Max}(106.75, 1605.86, 0) * 19.7500/134.0000$$

$$= 236.7 \text{ lbf}$$

Saddle Reaction Force due to Earthquake Fl or Friction [Fsl]:

$$= \text{Max}(F_l, \text{Friction Force, Sum of X Forces}) * B / L_s$$

$$= \text{Max}(273.88, 1605.86, 0) * 19.7500/134.0000$$

$$= 236.7 \text{ lbf}$$

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## Mechanical Design Calculations

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**REQ. No: MC213**

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**P.O. No: MPC17901**

### Saddle Reaction Force due to Earthquake Ft [Fst]:

$$\begin{aligned} &= F_{tr} * (F_t / \text{Num of Saddles} + Z \text{ Force Load}) * B / E \\ &= 3.00 * (273/2 + 0) * 19.7500/20.2506 \\ &= 400.7 \text{ lbf} \end{aligned}$$

### Load Combination Results for Q + Wind or Seismic [Q]:

$$\begin{aligned} &= \text{Saddle Load} + \text{Max}(F_{wl}, F_{wt}, F_{sl}, F_{st}) \\ &= 4014 + \text{Max}(236, 1120, 236, 400) \\ &= 5135.6 \text{ lbf} \end{aligned}$$

### Summary of Loads at the base of this Saddle:

Vertical Load (including saddle weight)	5206.53	lbf
Transverse Shear Load Saddle	383.11	lbf
Longitudinal Shear Load Saddle	1605.86	lbf

### Formulas and Substitutions for Horizontal Vessel Analysis:

Note: Wear Plate is Welded to the Shell,  $k = 0.1$

### The Computed K values from Table 4.15.1:

K1 = 0.1035	K2 = 1.2043	K3 = 0.9157	K4 = 0.4095
K5 = 0.7680	K6 = 0.0546	K7 = 0.0546	K8 = 0.3424
K9 = 0.2741	K10 = 0.0595	K1* = 0.1871	K6p = 0.0347
K7P = 0.0347			

The suffix 'p' denotes the values for a wear plate if it exists.

Note: Dimension a is greater than or equal to  $R_m / 2$ .

### Moment per Equation 4.15.3 [M1]:

$$\begin{aligned} &= -Q * a [1 - (1 - a/L + (R^2 - h^2) / (2a * L)) / (1 + (4h^2) / (3L))] \\ &= -5135 * 29.00 [1 - (1 - 29.00/182.06 + (11.812^2 - 5.812^2) / (2 * 29.00 * 182.06)) / (1 + (4 * 5.81) / (3 * 182.06))] \\ &= -27404.3 \text{ in-lb} \end{aligned}$$

### Moment per Equation 4.15.4 [M2]:

$$\begin{aligned} &= Q * L / 4 (1 + 2(R^2 - h^2) / (L^2)) / (1 + (4h^2) / (3L)) - 4a/L \\ &= 5135 * 182.1 / 4 (1 + 2(11.812^2 - 5.812^2) / (182.06^2)) / (1 + (4 * 5.812) / (3 * 182.06)) - 4 * 29.00 / 182.06 \\ &= 76704.5 \text{ in-lb} \end{aligned}$$

### Longitudinal Stress at Top of Shell (4.15.6) [Sigma1]:

$$\begin{aligned} &= P * R_m / (2t) - M_2 / (\pi * R_m^2 * t) \\ &= 310.00 * 11.812 / (2 * 0.375) - 76704.5 / (\pi * 11.8^2 * 0.375) \\ &= 4415.89 \text{ psi} \end{aligned}$$

### Longitudinal Stress at Bottom of Shell (4.15.7) [Sigma2]:

$$\begin{aligned} &= P * R_m / (2t) + M_2 / (\pi * R_m^2 * t) \\ &= 310.00 * 11.812 / (2 * 0.375) + 76704.5 / (\pi * 11.8^2 * 0.375) \\ &= 5349.11 \text{ psi} \end{aligned}$$

### Longitudinal Stress at Top of Shell at Support (4.15.10) [Sigma\*3]:

$$\begin{aligned} &= P * R_m / (2t) - M_1 / (K_1 * \pi * R_m^2 * t) \\ &= 310.00 * 11.812 / (2 * 0.375) - 27404.3 / (0.1035 * \pi * 11.8^2 * 0.375) \\ &= 6493.53 \text{ psi} \end{aligned}$$

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2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

## Mechanical Design Calculations

**Item/Tag No: HX1065/2065**  
**REQ. No: MC213**

**Doc. No: S11-10279-7/8-CP**

**P.O. No: MPC17901**

Longitudinal Stress at Bottom of Shell at Support (4.15.11) [Sigma\*4]:

$$\begin{aligned} &= P * Rm / (2t) + M1 / (K1 * \pi * Rm^2 * t) \\ &= 310.00 * 11.812 / (2 * 0.375) + -27404.3 / (0.1871 * \pi * 11.8^2 * 0.375) \\ &= 3991.58 \text{ psi} \end{aligned}$$

Maximum Shear Force in the Saddle (4.15.5) [T]:

$$\begin{aligned} &= Q(L-2a) / (L + (4 * h^2 / 3)) \\ &= 5135 * (182.06 - 2 * 29.00) / (182.06 + (4 * 5.81^2 / 3)) \\ &= 3356.6 \text{ lbf} \end{aligned}$$

Shear Stress in the shell no rings, not stiffened (4.15.14) [tau2]:

$$\begin{aligned} &= K2 * T / (\pi * Rm * t) \\ &= 1.2043 * 3356.64 / (\pi * 11.8125 * 0.3750) \\ &= 912.56 \text{ psi} \end{aligned}$$

Decay Length (4.15.22) [x1,x2]:

$$\begin{aligned} &= 0.78 * \sqrt{Rm * t} \\ &= 0.78 * \sqrt{11.812 * 0.375} \\ &= 1.642 \text{ in} \end{aligned}$$

Circumferential Stress in shell, no rings (4.15.23) [sigma6]:

$$\begin{aligned} &= -K5 * Q * k / (t * (b + X1 + X2)) \\ &= -0.7680 * 5135 * 0.1 / (0.375 * (5.50 + 1.64 + 1.64)) \\ &= -119.74 \text{ psi} \end{aligned}$$

Effective reinforcing plate width (4.15.1) [B1]:

$$\begin{aligned} &= \min(b + 1.56 * \sqrt{Rm * t}, 2a) \\ &= \min(5.50 + 1.56 * \sqrt{11.812 * 0.375}, 2 * 29.000) \\ &= 8.78 \text{ in} \end{aligned}$$

Wear Plate/Shell Stress ratio (4.15.29) [eta]:

$$\begin{aligned} &= \min(Sr/S, 1) \\ &= \min(17100.000/17100.000, 1) \\ &= 1.0000 \end{aligned}$$

Circumferential Stress at wear plate (4.15.26) [sigma6,r]:

$$\begin{aligned} &= -K5 * Q * k / (B1 * (t + eta * tr)) \\ &= -0.7680 * 5135 * 0.1 / (8.783 * (0.375 + 1.000 * 0.375)) \\ &= -59.87 \text{ psi} \end{aligned}$$

Circ. Comp. Stress at Horn of Saddle, L>=8Rm (4.15.27) [sigma7,r]:

$$\begin{aligned} &= -Q / (4(t + eta * tr) * b1) - 3 * K7 * Q / (2(t + eta * tr)^2) \\ &= -5135 / (4(0.375 + 1.000 * 0.375) * 8.783) - \\ &\quad 3 * 0.055 * 5135 / (2(0.375 + 1.000 * 0.375)^2) \\ &= -941.96 \text{ psi} \end{aligned}$$

Free Un-Restrained Thermal Expansion between the Saddles [Exp]:

$$\begin{aligned} &= \alpha * Ls * (\text{Design Temperature} - \text{Ambient Temperature}) \\ &= 0.671E-05 * 134.000 * (205.0 - 70.0) \\ &= 0.121 \text{ in} \end{aligned}$$

### Results for Vessel Ribs, Web and Base:

Baseplate Length	Bplen	16.0000	in
Baseplate Thickness	Bpthk	0.3750	in
Baseplate Width	Bpwid	6.0000	in
Number of Ribs ( inc. outside ribs )	Nribs	1	
Rib Thickness	Ribtk	0.3750	in
Web Thickness	Webtk	0.3750	in
Web Location	Webloc	Side	

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## Mechanical Design Calculations

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Moment of Inertia of Saddle - Lateral Direction

	Y	A	AY	Io
Shell	0.	5.	1.	0.
Wearplate	1.	4.	2.	1.
Web	5.	3.	19.	125.
BasePlate	10.	2.	23.	234.
Totals	16.	14.	45.	360.
Value	C1 = Sumof(Ay) / Sumof(A)			= 3. in
Value	I = Sumof(Io) - C1*Sumof(Ay)			= 222. in**4
Value	As = Sumof(A) - Ashell			= 9. in <sup>2</sup>

$$K1 = (1 + \cos(\beta) - .5 * \sin(\beta)^2) / (\pi - \beta + \sin(\beta) * \cos(\beta)) = 0.1999$$

$$Fh = K1 * Q = 0.1999 * 5135.569 = 1026.5900 \text{ lbf}$$

$$\begin{aligned} \text{Tension Stress, } St &= (Fh / As) = 108.4187 \text{ psi} \\ \text{Allowed Stress, } Sa &= 0.6 * \text{Yield Str} = 22800.0000 \text{ psi} \end{aligned}$$

$$\begin{aligned} d &= B - R * \sin(\theta) / \theta = 12.3233 \text{ in} \\ \text{Bending Moment, } M &= Fh * d = 12650.9326 \text{ in-lb} \end{aligned}$$

$$\begin{aligned} \text{Bending Stress, } Sb &= (M * C1 / I) = 175.9178 \text{ psi} \\ \text{Allowed Stress, } Sa &= 2/3 * \text{Yield Str} = 25333.3340 \text{ psi} \end{aligned}$$

### Minimum Thickness of Baseplate per Moss :

$$\begin{aligned} &= (3 * (Q + \text{Saddle\_Wt}) * \text{BasePlateWidth} / (2 * \text{BasePlateLength} * \\ &\quad \text{AllStress}))^{1/2} \\ &= (3 * (5135 + 70) * 6.00 / (2 * 16.000 * 25333.334))^{1/2} \\ &= 0.340 \text{ in} \end{aligned}$$

### Input Data for Base Plate Bolting Calculations:

Total Number of Bolts per BasePlate	Nbolts	2
Total Number of Bolts in Tension/Baseplate	Nbt	2
Bolt Material Specification	SA-325	
Bolt Allowable Stress	Stba	20200.00 psi
Bolt Corrosion Allowance	Bca	0.0000 in
Distance from Bolts to Edge	Edgedis	2.0000 in
Nominal Bolt Diameter	Bnd	0.7500 in
Thread Series	Series	TEMA
BasePlate Allowable Stress	S	20000.00 psi
Area Available in a Single Bolt	BlArea	0.3020 in <sup>2</sup>
Saddle Load QO (Weight)	QO	4085.6 lbf
Saddle Load QL (Wind/Seismic contribution)	QL	236.7 lbf
Maximum Transverse Force	Ft	383.1 lbf
Maximum Longitudinal Force	F1	1605.9 lbf
Saddle Bolted to Steel Foundation	No	

### Bolt Area Calculation per Dennis R. Moss

Bolt Area Requirement Due to Longitudinal Load [Bltarear1]:  
 = 0.0 (QO > QL --> No Uplift in Longitudinal direction)

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Bolt Area due to Shear Load [Bltarears]:  
=  $F1 / (Stba * Nbolts)$   
=  $1605.86 / (20200.00 * 2.00)$   
=  $0.0397 \text{ in}^2$

Bolt Area due to Transverse Load

Moment on Baseplate Due to Transverse Load [Rmom]:  
=  $B * Ft + \text{Sum of X Moments}$   
=  $19.75 * 383.11 + 0.00$   
=  $7566.36 \text{ in-lb}$

Eccentricity (e):  
=  $Rmom / QO$   
=  $7566.36 / 4085.62$   
=  $1.85 \text{ in} < Bplen/6$  --> No Uplift in Transverse direction

Bolt Area due to Transverse Load [Bltareart]:  
= 0 (No Uplift)

Required of a Single Bolt [Bltarear]  
=  $\max[Bltarearl, Bltarears, Bltareart]$   
=  $\max[0.0000, 0.0397, 0.0000]$   
=  $0.0397 \text{ in}^2$

### ASME Horizontal Vessel Analysis: Stresses for the Right Saddle (per ASME Sec. VIII Div. 2 based on the Zick method.)

#### Input and Calculated Values:

Vessel Mean Radius	Rm	11.81	in
Stiffened Vessel Length per 4.15.6	L	182.06	in
Distance from Saddle to Vessel tangent	a	43.00	in
Saddle Width	b	5.50	in
Saddle Bearing Angle	theta	118.00	degrees
Wear Plate Width	b1	10.00	in
Wear Plate Bearing Angle	thetal	145.00	degrees
Wear Plate Thickness	tr	0.3750	in
Wear Plate Allowable Stress	Sr	17100.00	psi
Inside Depth of Head	h2	5.81	in
Shell Allowable Stress used in Calculation		17100.00	psi
Head Allowable Stress used in Calculation		20000.00	psi
Circumferential Efficiency in Plane of Saddle		1.00	
Circumferential Efficiency at Mid-Span		1.00	
Saddle Force Q, Operating Case		3365.37	lbf

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## Mechanical Design Calculations

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Horizontal Vessel Analysis Results:		Actual	Allowable	
Long. Stress at Top of Midspan		4863.34	17100.00	psi
Long. Stress at Bottom of Midspan		4901.66	17100.00	psi
Long. Stress at Top of Saddles		7101.94	17100.00	psi
Long. Stress at Bottom of Saddles		3655.12	17100.00	psi
Tangential Shear in Shell		463.04	13680.00	psi
Circ. Stress at Horn of Saddle		617.27	21375.00	psi
Circ. Compressive Stress in Shell		78.47	17100.00	psi

### Intermediate Results: Saddle Reaction Q due to Wind or Seismic

#### Saddle Reaction Force due to Wind Ft [Fwt]:

$$\begin{aligned} &= F_{tr} * (F_t / \text{Num of Saddles} + Z \text{ Force Load}) * B / E \\ &= 3.00 * (766.2/2 + 0) * 19.7500/20.2506 \\ &= 1120.9 \text{ lbf} \end{aligned}$$

#### Saddle Reaction Force due to Wind Fl or Friction [Fwl]:

$$\begin{aligned} &= \text{Max}(F_l, \text{Friction Load, Sum of X Forces}) * B / L_s \\ &= \text{Max}(106.75, 897.78, 0) * 19.7500/134.0000 \\ &= 132.3 \text{ lbf} \end{aligned}$$

#### Saddle Reaction Force due to Earthquake Fl or Friction [Fsl]:

$$\begin{aligned} &= \text{Max}(F_l, \text{Friction Force, Sum of X Forces}) * B / L_s \\ &= \text{Max}(273.88, 897.78, 0) * 19.7500/134.0000 \\ &= 132.3 \text{ lbf} \end{aligned}$$

#### Saddle Reaction Force due to Earthquake Ft [Fst]:

$$\begin{aligned} &= F_{tr} * (F_t / \text{Num of Saddles} + Z \text{ Force Load}) * B / E \\ &= 3.00 * (273/2 + 0) * 19.7500/20.2506 \\ &= 400.7 \text{ lbf} \end{aligned}$$

#### Load Combination Results for Q + Wind or Seismic [Q]:

$$\begin{aligned} &= \text{Saddle Load} + \text{Max}(F_{wl}, F_{wt}, F_{sl}, F_{st}) \\ &= 2244 + \text{Max}(132, 1120, 132, 400) \\ &= 3365.4 \text{ lbf} \end{aligned}$$

#### Summary of Loads at the base of this Saddle:

Vertical Load (including saddle weight)	3436.33	lbf
Transverse Shear Load Saddle	383.11	lbf
Longitudinal Shear Load Saddle	897.78	lbf

#### Formulas and Substitutions for Horizontal Vessel Analysis:

Note: Wear Plate is Welded to the Shell,  $k = 0.1$

#### The Computed K values from Table 4.15.1:

K1 = 0.1035	K2 = 1.2043	K3 = 0.9157	K4 = 0.4095
K5 = 0.7680	K6 = 0.0546	K7 = 0.0546	K8 = 0.3424
K9 = 0.2741	K10 = 0.0595	K1* = 0.1871	K6p = 0.0347
K7P = 0.0347			

The suffix 'p' denotes the values for a wear plate if it exists.

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# Mechanical Design Calculations

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**Moment per Equation 4.15.3 [M1]:**

$$= -Q \cdot a \left[ 1 - \left( 1 - \frac{a}{L} + \frac{(R^2 - h^2)}{(2a \cdot L)} \right) / \left( 1 + \frac{(4h^2)}{(3L)} \right) \right]$$

$$= -3365 \cdot 43.00 \left[ 1 - \left( 1 - \frac{43.00}{182.06} + \frac{(11.812^2 - 5.812^2)}{(2 \cdot 43.00 \cdot 182.06)} \right) / \left( 1 + \frac{(4 \cdot 5.81^2)}{(3 \cdot 182.06)} \right) \right]$$

$$= -37753.7 \text{ in-lb}$$

**Moment per Equation 4.15.4 [M2]:**

$$= Q \cdot L / 4 \left( 1 + 2 \frac{(R^2 - h^2)}{(L^2)} / \left( 1 + \frac{(4h^2)}{(3L)} \right) - 4a/L \right)$$

$$= 3365 \cdot 182.1 / 4 \left( 1 + 2 \frac{(11.812^2 - 5.812^2)}{(182.06^2)} / \left( 1 + \frac{(4 \cdot 5.812^2)}{(3 \cdot 182.063)} \right) - 4 \cdot 43.00 / 182.06 \right)$$

$$= 3149.7 \text{ in-lb}$$

**Longitudinal Stress at Top of Shell (4.15.6) [Sigma1]:**

$$= P \cdot R_m / (2t) - M_2 / (\pi \cdot R_m^2 \cdot t)$$

$$= 310.00 \cdot 11.812 / (2 \cdot 0.375) - 3149.7 / (\pi \cdot 11.8^2 \cdot 0.375)$$

$$= 4863.34 \text{ psi}$$

**Longitudinal Stress at Bottom of Shell (4.15.7) [Sigma2]:**

$$= P \cdot R_m / (2t) + M_2 / (\pi \cdot R_m^2 \cdot t)$$

$$= 310.00 \cdot 11.812 / (2 \cdot 0.375) + 3149.7 / (\pi \cdot 11.8^2 \cdot 0.375)$$

$$= 4901.66 \text{ psi}$$

**Longitudinal Stress at Top of Shell at Support (4.15.10) [Sigma\*3]:**

$$= P \cdot R_m / (2t) - M_1 / (K_1 \cdot \pi \cdot R_m^2 \cdot t)$$

$$= 310.00 \cdot 11.812 / (2 \cdot 0.375) - 37753.7 / (0.1035 \cdot \pi \cdot 11.8^2 \cdot 0.375)$$

$$= 7101.94 \text{ psi}$$

**Longitudinal Stress at Bottom of Shell at Support (4.15.11) [Sigma\*4]:**

$$= P \cdot R_m / (2t) + M_1 / (K_1 \cdot \pi \cdot R_m^2 \cdot t)$$

$$= 310.00 \cdot 11.812 / (2 \cdot 0.375) + 37753.7 / (0.1871 \cdot \pi \cdot 11.8^2 \cdot 0.375)$$

$$= 3655.12 \text{ psi}$$

**Maximum Shear Force in the Saddle (4.15.5) [T]:**

$$= Q(L - 2a) / (L + (4 \cdot h^2 / 3))$$

$$= 3365 \cdot (182.06 - 2 \cdot 43.00) / (182.06 + (4 \cdot 5.81^2 / 3))$$

$$= 1703.2 \text{ lbf}$$

**Shear Stress in the shell no rings, not stiffened (4.15.14) [tau2]:**

$$= K_2 \cdot T / (\pi \cdot R_m \cdot t)$$

$$= 1.2043 \cdot 1703.19 / (\pi \cdot 11.8125 \cdot 0.375)$$

$$= 463.04 \text{ psi}$$

**Decay Length (4.15.22) [x1,x2]:**

$$= 0.78 \cdot \sqrt{R_m \cdot t}$$

$$= 0.78 \cdot \sqrt{11.812 \cdot 0.375}$$

$$= 1.642 \text{ in}$$

**Circumferential Stress in shell, no rings (4.15.23) [sigma6]:**

$$= -K_5 \cdot Q \cdot k / (t \cdot (b + X_1 + X_2))$$

$$= -0.7680 \cdot 3365 \cdot 0.1 / (0.375 \cdot (5.50 + 1.64 + 1.64))$$

$$= -78.47 \text{ psi}$$

**Effective reinforcing plate width (4.15.1) [B1]:**

$$= \min(b + 1.56 \cdot \sqrt{R_m \cdot t}, 2a)$$

$$= \min(5.50 + 1.56 \cdot \sqrt{11.812 \cdot 0.375}, 2 \cdot 43.000)$$

$$= 8.78 \text{ in}$$

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**Mechanical Design Calculations****Item/Tag No: HX1065/2065**  
**REQ. No: MC213****Doc. No: S11-10279-7/8-CP****P.O. No: MPC17901**

Wear Plate/Shell Stress ratio (4.15.29) [eta]:

$$= \min( S_r/S, 1 )$$

$$= \min( 17100.000/17100.000, 1 )$$

$$= 1.0000$$

Circumferential Stress at wear plate (4.15.26) [sigma6,r]:

$$= -K5 * Q * k / ( B1( t + eta * tr ) )$$

$$= -0.7680 * 3365 * 0.1 / ( 8.783 ( 0.375 + 1.000 * 0.375 ) )$$

$$= -39.23 \text{ psi}$$

Circ. Comp. Stress at Horn of Saddle, L&gt;=8Rm (4.15.27) [sigma7,r]:

$$= -Q / ( 4(t+eta*tr)b1 ) - 3*K7*Q / ( 2(t+eta*tr)^2 )$$

$$= -3365 / ( 4(0.375 + 1.000 * 0.375 ) 8.783 ) -$$

$$3 * 0.055 * 3365 / ( 2(0.375 + 1.000 * 0.375 )^2 )$$

$$= -617.27 \text{ psi}$$

**Results for Vessel Ribs, Web and Base**

Baseplate Length	Bplen	16.0000	in
Baseplate Thickness	Bpthk	0.3750	in
Baseplate Width	Bpwid	6.0000	in
Number of Ribs ( inc. outside ribs )	Nribs	1	
Rib Thickness	Ribtk	0.3750	in
Web Thickness	Webtk	0.3750	in
Web Location	Webloc	Side	

Moment of Inertia of Saddle - Lateral Direction

	Y	A	AY	Io
Shell	0.	5.	1.	0.
Wearplate	1.	4.	2.	1.
Web	5.	3.	19.	125.
BasePlate	10.	2.	23.	234.
Totals	16.	14.	45.	360.

$$\text{Value } C1 = \text{Sumof}(AY) / \text{Sumof}(A) = 3. \text{ in}$$

$$\text{Value } I = \text{Sumof}(Io) - C1 * \text{Sumof}(AY) = 222. \text{ in}^4$$

$$\text{Value } As = \text{Sumof}(A) - A_{shell} = 9. \text{ in}^2$$

$$K1 = (1 + \cos(\beta) - .5 * \sin(\beta)^2) / (\pi - \beta + \sin(\beta) * \cos(\beta)) = 0.1999$$

$$Fh = K1 * Q = 0.1999 * 3365.372 = 672.7310 \text{ lbf}$$

$$\text{Tension Stress, } St = ( Fh / As ) = 71.0475 \text{ psi}$$

$$\text{Allowed Stress, } Sa = 0.6 * \text{Yield Str} = 22800.0000 \text{ psi}$$

$$d = B - R * \sin(\theta) / \theta = 12.3233 \text{ in}$$

$$\text{Bending Moment, } M = Fh * d = 8290.2373 \text{ in-lb}$$

$$\text{Bending Stress, } Sb = ( M * C1 / I ) = 115.2801 \text{ psi}$$

$$\text{Allowed Stress, } Sa = 2/3 * \text{Yield Str} = 25333.3340 \text{ psi}$$

**Minimum Thickness of Baseplate per Moss :**

$$= ( 3 * ( Q + \text{Saddle\_Wt} ) * \text{BasePlateWidth} / ( 2 * \text{BasePlateLength} * \text{AllStress} ) )^{1/2}$$

$$= ( 3 * ( 3365 + 70 ) * 6.00 / ( 2 * 16.000 * 25333.334 ) )^{1/2}$$

$$= 0.276 \text{ in}$$

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# Mechanical Design Calculations

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## Input Data for Base Plate Bolting Calculations:

Total Number of Bolts per BasePlate	Nbolts	2	
Total Number of Bolts in Tension/Baseplate	Nbt	2	
Bolt Material Specification		SA-325	
Bolt Allowable Stress	Stba	20200.00	psi
Bolt Corrosion Allowance	Bca	0.0000	in
Distance from Bolts to Edge	Edgedis	2.0000	in
Nominal Bolt Diameter	Bnd	0.7500	in
Thread Series	Series	TEMA	
BasePlate Allowable Stress	S	20000.00	psi
Area Available in a Single Bolt	BltArea	0.3020	in <sup>2</sup>
Saddle Load QO (Weight)	QO	2315.4	lbf
Saddle Load QL (Wind/Seismic contribution)	QL	132.3	lbf
Maximum Transverse Force	Ft	383.1	lbf
Maximum Longitudinal Force	F1	1605.9	lbf
Saddle Bolted to Steel Foundation	No		

## Bolt Area Calculation per Dennis R. Moss

Bolt Area Requirement Due to Longitudinal Load [Bltarearl]:  
= 0.0 (QO > QL --> No Uplift in Longitudinal direction)

Bolt Area due to Shear Load [Bltarears]:  
= F1 / (Stba \* Nbolts)  
= 1605.86 / (20200.00 \* 2.00 )  
= 0.0397 in<sup>2</sup>

Bolt Area due to Transverse Load

Moment on Baseplate Due to Transverse Load [Rmom]:  
= B \* Ft + Sum of X Moments  
= 19.75 \* 383.11 + 0.00  
= 7566.36 in-lb

Eccentricity (e):  
= Rmom / QO  
= 7566.36 / 2315.42  
= 3.27 in > Bplen/6 --> Uplift in Transverse direction

f = Bplen / 2 - Edgedis  
= 16.00 / 2 - 2.00  
= 6.00 in

Modular Ratio Of Steel/Concrete (n1):  
= ES / EC  
= 29500000 / 3122018  
= 9.45

K1 = 3 (e - 0.5 \* Bplen)  
= 3 (3.27 - 0.5 \* 16.00 )  
= -14.20 in

K2 = 6 \* n1 \* At / Bpwid \* (f + e)  
= 6 \* 9.45 \* 0.60 / 6.00 \* (6.00 + 3.27 )  
= 52.89 in<sup>2</sup>

Page 35 of 37

Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

## Mechanical Design Calculations

**Item/Tag No: HX1065/2065**  
**REQ. No: MC213**

**Doc. No: S11-10279-7/8-CP**

**P.O. No: MPC17901**

$$\begin{aligned}K3 &= -K2 * (0.5 * Bplen + f) \\&= -52.89 * (0.5 * 16.00 + 6.00) \\&= -740.51 \text{ in}^3\end{aligned}$$

Iteratively Solving for the Effective Bearing Length:

$$\begin{aligned}Y^3 + K1 * Y^2 + K2 * Y + K3 &= 0 \\Y^3 + -14.20 * Y^2 + 52.89 * Y + -740.51 &= 0 \\Y &= 14.16 \text{ in}\end{aligned}$$

$$\begin{aligned}\text{Num} &= (Bplen / 2 - Y / 3 - e) \\&= (16.00/2 - 14.16/3 - 3.27) \\&= 0.01\end{aligned}$$

$$\begin{aligned}\text{Denom} &= (Bplen / 2 - Y / 3 + f) \\&= (16.00/2 - 14.16/3 + 6.00) \\&= 9.28\end{aligned}$$

Total Bolt Tension Force [Tforce]:

$$\begin{aligned}&= -QO * \text{Num} / \text{Denom} \\&= -2315.42 * 0.01/9.28 \\&= -3.41 \text{ lbf}\end{aligned}$$

Bolt Area Required due to Transverse Load [Bltareart]

$$\begin{aligned}&= \text{Tforce} / (\text{Stba} * \text{Nbt}) \\&= -3.41 / (20200.00 * 2.00) \\&= -0.0001 \text{ in}^2\end{aligned}$$

Required of a Single Bolt [Bltarear]

$$\begin{aligned}&= \max[\text{Bltarearl}, \text{Bltarears}, \text{Bltareart}] \\&= \max[0.0000, 0.0397, -0.0001] \\&= 0.0397 \text{ in}^2\end{aligned}$$

### Baseplate Thickness Calculation per D. Moss:

Bearing Pressure (fc)

$$\begin{aligned}&= 2 * (QO + \text{Tforce}) / (Y * Bplen) \\&= 2 * (2315.42 + -3.41) / (14.16 * 16.00) \\&= 20.42 \text{ psig}\end{aligned}$$

Distance from Baseplate Edge to the Web [ADIST]:

$$\begin{aligned}&= (Bplen - \text{Weblength}) / 2 \\&= (16.00 - 14.00) / 2 \\&= 1.0000 \text{ in}\end{aligned}$$

Overturning Moment due To Bolt Tension [Mt]:

$$\begin{aligned}&= \text{Tforce} * \text{Adist} \\&= -3.41 * 1.00 \\&= -3.41 \text{ in-lb}\end{aligned}$$

Equivalent Bearing Pressure (f1):

$$\begin{aligned}&= fc * (Y - \text{Adist}) / Y \\&= 20.42 * (14.16 - 1.00) / 14.16 \\&= 18.97 \text{ psig}\end{aligned}$$

Page 36 of 37

Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	CORRECTED LOADING CALC PG 27-37	RS	2/9/12	rds	2/9/12

## Mechanical Design Calculations

Item/Tag No: HX1065/2065  
REQ. No: MC213

Doc. No: S11-10279-7/8-CP

P.O. No: MPC17901

Overturning Moment due to Bearing Pressure [Mc]:

$$\begin{aligned} &= (Adist^2 * Bpwid / 6) * (f1 + 2 * fc) \\ &= (1.00^2 * 6.00/6) * (18.97 + 2 * 20.42) \\ &= 59.81 \text{ in-lb} \end{aligned}$$

Baseplate Required Thickness [Treq]:

$$\begin{aligned} &= (6 * \max(Mt, Mc) / (Bpwid * Sba))^{1/2} \\ &= (6 * \max(-3.41, 59.81 / (6.00 * 30000.00))^{1/2} \\ &= 0.0446 \text{ in} \end{aligned}$$

PV Elite is a trademark of Intergraph CADWorx & Analysis Solutions, Inc. 2012

Page 37 of 37

Revisions to this Page					
No.	Detailed Description of the Revision	By	Date	Apdd.	Date
1	Revised per Customer Comments	SR	1/12/12	GD	1/12/12
2	<b>CORRECTED LOADING CALC PG 27-37</b>	<b>RS</b>	<b>2/9/12</b>	<b>rds</b>	<b>2/9/12</b>



## QUALITY CONTROL INSPECTION PLAN / CHECKLIST

(\*) DENOTES A.I. HOLD POINTS

mk 9/1/11  
AUTHORIZED INSPECTOR DATE

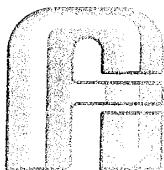
INSPECTIONS PERFORMED	FABSCO Q.C.	AUTHORIZED INSPECTOR	REMARKS
REVIEW DWG/CALC, WPS/PQR AND WELDER QUALIFICATION RECORDS	W	mk 9/1/11	
EXAM. PLATE EDGES/SURF.	W		
CHECK NOZZ. PIPE THKNS.	W		
CHECK HEAD/SHELL THKNS.	W		
VERIFY NOZZ. FLG. RATING	W		
CHECK MATERIAL IDENTIF.	W		
REVIEW MILL TEST REPORTS	W	mk 9/27/12	
CHK FITUP CHAN. LONG SEAM	W N/A		
CHK FITUP CHAN. GIRTH SEAMS	W 08.08.12 JB		
CHK FITUP CHAN NOZZ.	W 08.08.12 JB		
CHK. FITUP SHELL LONGSEAM	W		
CHK FITUP SHELL GIRTH SEAM	W		
CHK FITUP SHELL NOZZLES	W		
PERFORM PMI (IF REQUIRED)	W		
PERFORM/WITNESS NDE PT	W		
PERFORM/WITNESS NDE MT	W		
PERFORM/WITNESS NDE UT	W		
CHECK OUT OF ROUNDNESS	W 09.17.12 JB		
CHK ALL WELDS FOR APPEARANCE, SIZE & WELDERS STAMP	W 09.19.12 JB	Part Stamped	
REVIEW RADIOGRAPHS	W		
INTERNAL INSP. CHANNEL	W 09.18.12 JB		
	W		
INTERNAL INSP. SHELL	W 09.17.12 JB		
REVIEW HEAT CHARTS	W		
WITNESS HYDROSTATIC TEST SS	W 09.18.12 JB	mk 9/18/12	
TS	W 09.19.12 JB	mk 9/19/12	
CHK NAMEPLATE/STAMPING	W 09.19.12 JB	mk 9/27/12	
DATA REPORT CHECKED/SIGNED	W	mk 9/27/12	
CHK FINAL DIMENSIONAL	W 09.19.12 JB		
CHK FINAL APPEARANCE/PAINT	W		

NOTES: W= WITNESS

BLANK LINES ARE FOR CUSTOMER REQUIREMENTS

FABSCO SHELL & TUBE, L.L.C., 2410 INDUSTRIAL ROAD, SAPULPA, OK 74066 TELEPHONE (918) 224-7550 FAX: (918) 224-3564





**FABSCO**

**SHELL & TUBE, LLC**

2410 Industrial Road  
P.O. Box 988  
Sapulpa, OK 74066  
918/224-7550 Fax 918/224-3564

## CERTIFICATE OF HYDROSTATIC TESTING

FABSCO SHELL & TUBE, L.L.C. JOB NUMBER: <u>S11-10279.8</u>
CUSTOMER NAME: <u>SOUTHERN COMPANY SERVICES</u>
CUSTOMER PURCHASE ORDER NUMBER <u>MPC17901</u>
CUSTOMER ITEM NUMBER: <u>HX2065</u>

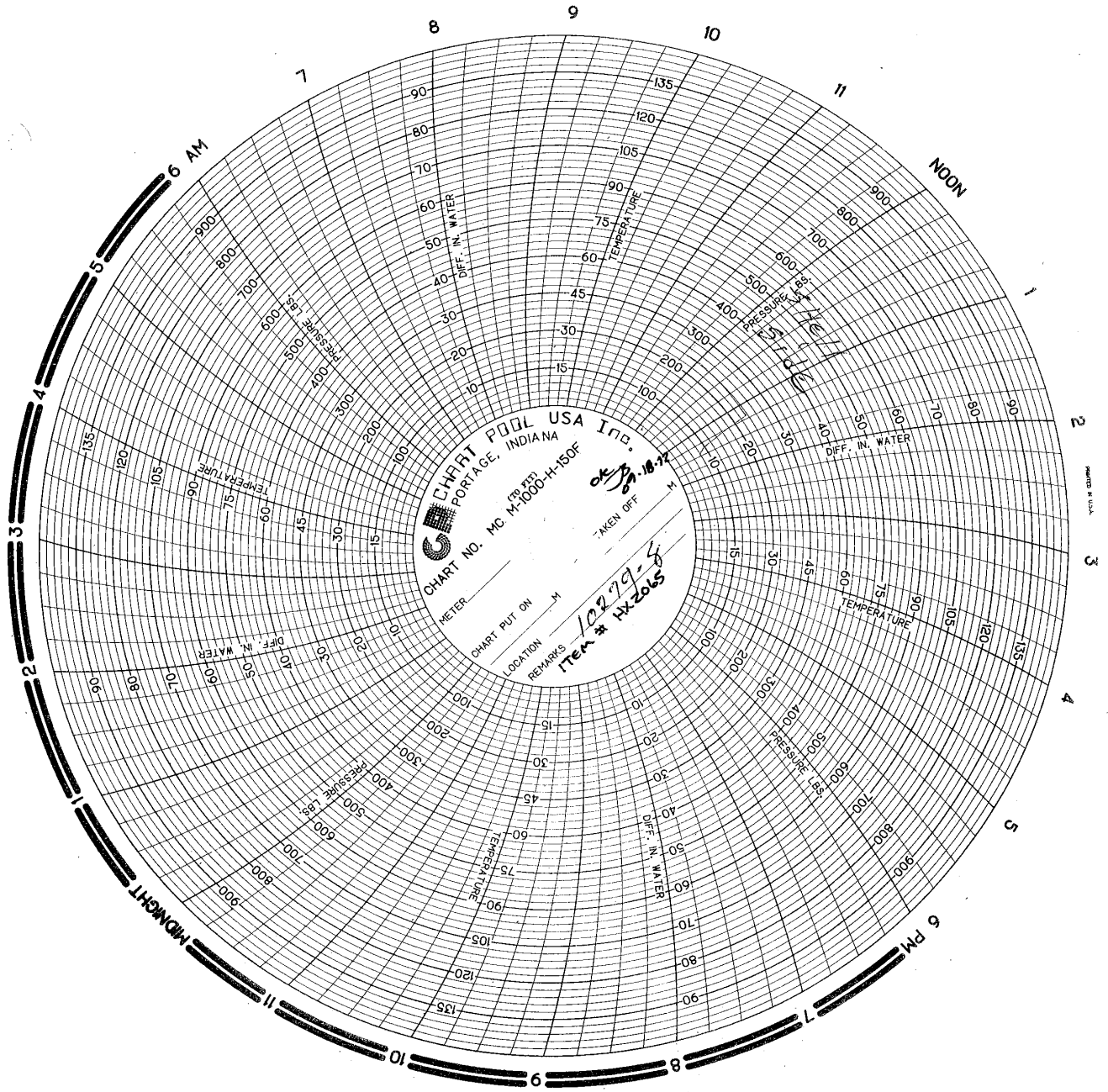
THIS IS TO CERTIFY THE ABOVE UNIT WAS HYDROSTATICALLY TESTED PER ASME CODE SECTION VIII DIVISION 1.

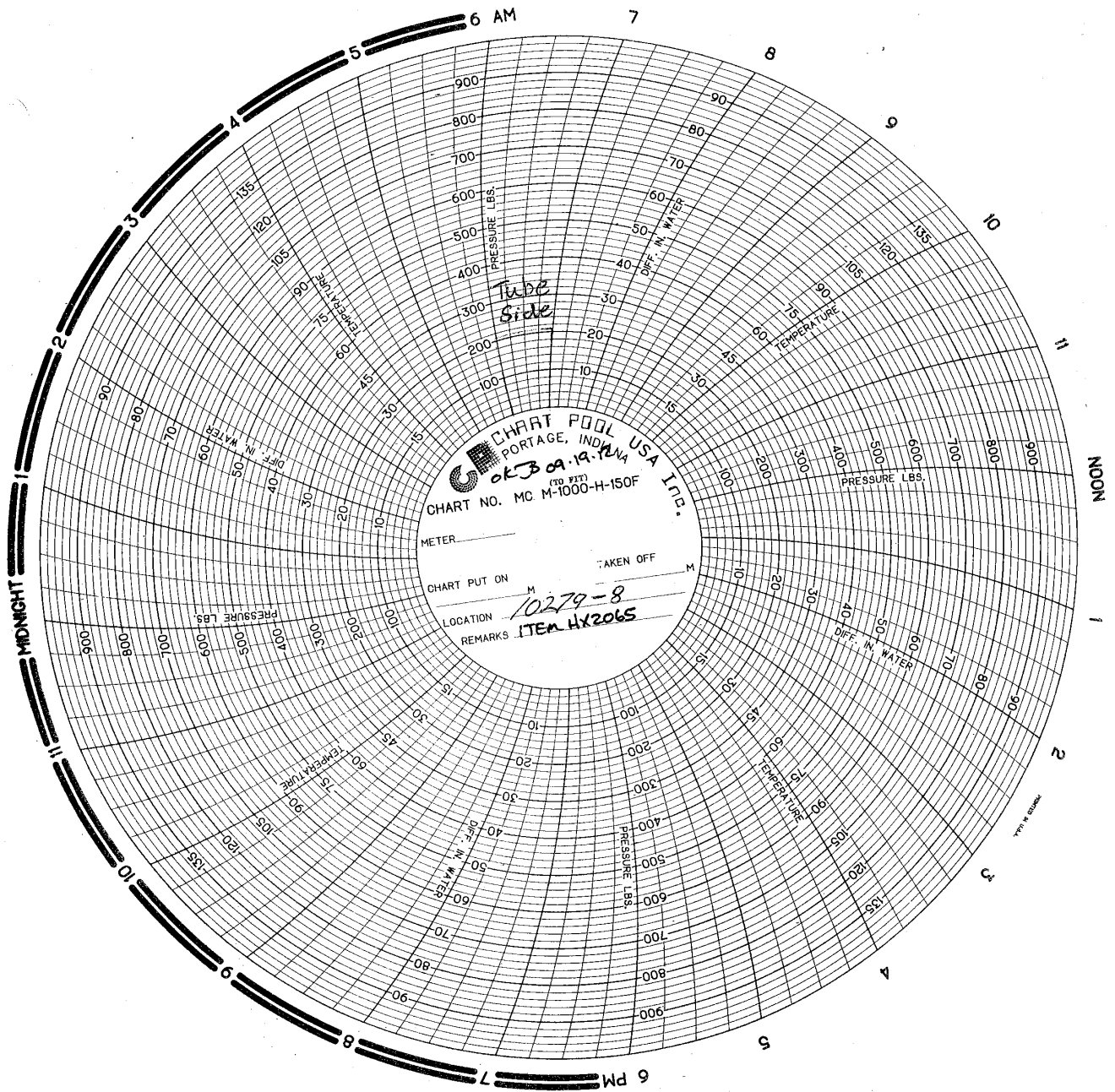
SHELLSIDE TEST PRESSURE PSI	TUBESIDE TEST PRESSURE PSI	DATE TESTING PERFORMED
234		09-18-12
	195	09-19-12

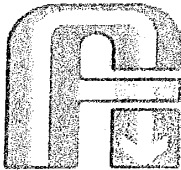
WITNESSED BY: ONEBEACON INSURANCE COMPANY AUTHORIZED INSPECTOR.

SIGNED FABSCO SHELL & TUBE, L.L.C. BY

John Davis  
QUALITY CONTROL DEPARTMENT





**FABSCO****SHELL & TUBE, LLC**

2410 Industrial Road

P.O. Box 988

Sapulpa, OK 74066

(918)224-7550 Fax(918)224-3564

**LIQUID PENETRANT EXAMINATION**

FABSCO SHELL & TUBE, L.L.C. JOB NUMBER: <b>SI110279.8</b>
CUSTOMER NAME: <b>Southern Co.</b>
CUSTOMER P.O. NO: <b>MPC 17901</b>
CUSTOMER ITEM NUMBER: <b>HX 1065</b>

PARTS EXAMINED / WELD JOINT EXAMINED: <b>SHELL</b>
<b>EDGES OF GROOVED AND BEVELED LONG BAFFLE SLOTS.</b>

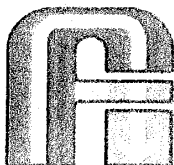
Liquid Penetrant Type: <b>II</b>	Method: <b>A</b>
Manufacturer I.D.: <b>Met-L-Chek</b>	Cleaner- <b>E-59A</b> Penetrant- <b>VP30</b>
Developer- <b>D70</b>	Lighting Equipment: <b>Maglite &gt;100 fc</b>
RESULTS: <b>Acceptable</b>	

REMARKS:
<b>No defects found.</b>

08.14.12

DATE

*John Burris L.T.***FABSCO(SHELL & TUBE, L.L.C.)  
QUALITY CONTROL DEPARTMENT  
NDE EXAMINER**Reference Fabco Shell Tube  
LLc. NDE Procedure #4Liquid Penetrant Examination  
11.15.2007  
Rev. 3

**FABSCO**

SHELL &amp; TUBE, LLC

2410 Industrial Road

P.O. Box 988

Sapulpa, OK 74066

(918)224-7550 Fax(918)224-3564

## LIQUID PENETRANT EXAMINATION

FABSCO SHELL & TUBE, L.L.C. JOB NUMBER: S11-10279-8
CUSTOMER NAME: SOUTHERN COMPANY SERVICES
CUSTOMER P.O. NO: MPC17901
CUSTOMER ITEM NUMBER: HX2065

PARTS EXAMINED / WELD JOINT EXAMINED: #19 TUBESHEET
TUBE TO TUBESHEET WELDS

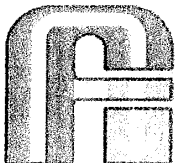
Liquid Penetrant Type: II	Method: A
Manufacturer I.D.: Met-L-Chek	Cleaner- E-59A Penetrant- VP30
Developer- D70	Lighting Equipment: Streamlite >100 fc
RESULTS: Acceptable	

REMARKS:
No defects found.

09.03.12

DATE

FABSCO SHELL & TUBE, L.L.C.  
QUALITY CONTROL DEPARTMENT  
NDE EXAMINERReference Fabco Shell Tube  
LLC. NDE Procedure #4Liquid Penetrant Examination  
11.15.2007  
Rev. 6



**FABSCO**  
SHELL & TUBE, LLC  
2410 Industrial Road  
P.O. Box 988  
Sapulpa, OK 74066  
(918)224-7550 Fax(918)224-3564

## LIQUID PENETRANT EXAMINATION

FABSCO SHELL & TUBE, L.L.C. JOB NUMBER: <b>S11-10279-8</b>
CUSTOMER NAME: <b>SOUTHERN COMPANY SERVICES</b>
CUSTOMER P.O. NO: <b>MPC17901</b>
CUSTOMER ITEM NUMBER: <b>HX2065</b>

PARTS EXAMINED / WELD JOINT EXAMINED: <b>FRONT CHANNEL &amp; SHELL</b>
<b>ALL PRESSURE RETAINING WELDS, TEMPORARY ATTACHMENTS, ARC STRIKES AND LIFTING LUG ATTACHMENT WELDS</b>

Liquid Penetrant Type: <b>II</b>	Method: <b>A</b>
Manufacturer I.D.: <b>Met-L-Chek</b>	Cleaner- <b>E-59A</b> Penetrant- <b>VP30</b>
Developer- <b>D70</b>	Lighting Equipment: <b>Streamlite &gt;100 fc</b>
RESULTS: <b>Acceptable</b>	

REMARKS:
<b>No defects found.</b>

9-13-12  
DATE

  
FABSCO SHELL & TUBE, L.L.C.  
QUALITY CONTROL DEPARTMENT  
NDE EXAMINER

Reference Fabco Shell Tube  
LLC. NDE Procedure #4

Liquid Penetrant Examination  
11.15.2007  
Rev. 6

**Globe X-Ray Services, Inc.**  
 8441 South Union Tulsa, Oklahoma 74132  
 (918) 446-1696

**OPTICAL EMISSION**

Cert. # \_\_\_\_\_

Customer FABSCO  
 Mail Reports to: QC/QA  
 Test Instrument ARC MET-8000

Report Number: 63815  
 Date: 8-15-12  
 Job Number: 10279-7

SN 515666

Acceptance Criteria/ Applicable Code		Element Analysis--%												Accept	Reject
Item/Part		Material Analysis	C	CR	NI	MO	NB	V	Pm1	AT					
Asme SECT II															
IT-22A	T-304L	0.020	18.38	8.26	0.37	0.015	0.05		1					✓	
IT-9	T-304L	0.020	18.29	8.04	0.32	0	0.04		2					✓	
IT-7	F-304L	0.021	18.03	8.35	0.34	0.002	0.05		3	50079				✓	
IT-120	T-304L	0.023	18.04	8.21	0.44	0	0.044		4	401468-4				✓	
IT-120	↓	0.019	18.24	8.07	0.42	0	0.041		5	↓				✓	
Shell-RS1	ER-308L	0.023	19.65	10.33	0.14	0	0.019		6	-				✓	
IT-9C	T-304L	0.020	18.40	8.06	0.31	0.002	0.056		7	979207				✓	
↓	↓	0.021	18.74	8.14	0.30	0.010	0.059		8	5V05				✓	
IT-9B	T-304L	0.017	18.41	8.05	0.33	0.006	0.058		9	837131				✓	
↓	↓	0.016	18.18	8.13	0.27	0	0.056		10	A7Y7				✓	
IT-18	T-304L	0.016	18.28	8.06	0.38	0.011	0.051		11	2C890				✓	
IT-2	F-304L	0.021	18.48	8.11	0.34	0.005	0.055		12	50079				✓	
IT-3	↓	0.020	18.40	8.18	0.34	0.009	0.058		13	50079				✓	
IT-4	TP-304L	0.023	18.19	8.10	0.32	0	0.048		14					✓	
chan-RS1	ER-308L	0.021	19.61	10.18	0.15	0.012	0.032		15	-				✓	
↓ RS-2	↓	0.022	19.57	10.01	0.16	0.011	0.032		16	-				✓	

Harvey LV II  
 Inspector

\_\_\_\_\_  
 Customer Representative

**Globe X-Ray Services, Inc.**  
 8441 South Union Tulsa, Oklahoma 74132  
 (918) 446-1696

**OPTICAL EMISSION**

Cert. # \_\_\_\_\_

Customer FABSCO  
 Mail Reports to: QC/CA  
 Test Instrument ARC MET-8000  
SN 515666

Report Number: 63819  
 Date: 8-16-12  
 Job Number: 10279-7

Acceptance Criteria/ Applicable Code		Element Analysis--%												Accept	Reject
<u>ASME SECT-II</u>															
Item/Part	Material Analysis	C	CR	NI	MO	NB	V		Pm1		HT				
IT-84	F-304L	0.023	18.10	8.28	0.22	0.018	0.079		17		M359			✓	
IT-83	↓	0.022	18.10	8.29	0.44	0.020	0.074		18		↓			✓	
IT-81	F-304L	0.022	18.03	8.38	0.36	0.023	0.088		19		87396/1			✓	
IT-82	F-304L	0.022	18.05	8.39	0.36	0.006	0.068		20		↓			✓	
IT-81 PIP	TP-304L	0.027	18.22	8.06	0.24	0.001	0.069		21					✓	
IT-82 PIP	↓	0.027	18.30	8.06	0.22	0.00	0.069		22					✓	
IT-81 RS	ER-308L	0.028	19.66	10.25	0.13	0	0		23		-			✓	
IT-82 RS	↓	0.029	19.60	10.28	0.15	0	0		24		-			✓	
IT-62	T-304L	0.018	18.33	8.11	0.29	0.019	0.077		25		COX3			✓	
↓	↓	0.018	18.30	8.07	0.29	0.020	0.076		26		↓			✓	
IT-72 FLNG	F-304L	0.018	18.27	8.12	0.48	0	0.053		27		83171/1			✓	
IT-72 PIP	TP-304L	0.026	18.04	8.01	0.50	0	0.075		28					✓	
IT-72 RS	ER-308L	0.022	18.87	10.07	0.13	0.041	0.056		29		-			✓	
IT-71 FLNG	F-304L	0.021	18.38	8.44	0.31	0.016	0.077		30		84245/1			✓	
IT-71 PIP	TP-304L	0.025	18.16	8.52	0.27	0.024	0.024		31		F126013			✓	
IT-71 RS	ER-308L	0.025	18.97	10.17	0.15	0.043	0.02		32		-			✓	

Inspector

Customer Representative



**Globe X-Ray Services, Inc.**  
 8441 South Union Tulsa, Oklahoma 74132  
 (918) 446-1696

**OPTICAL EMISSION**

Cert. # \_\_\_\_\_

Customer FABSCO  
 Mail Reports to: QC/QA  
 Test Instrument ARCMET-8000

Report Number: 63819  
 Date: 8-16-12 8/17-12  
 Job Number: 10279-7

SN 515666

Acceptance Criteria/ Applicable Code		Element Analysis--%												Accept	Reject
<u>SA-240</u> <u>ASME SECT II SA-312</u>															
Item/Part	Material Analysis	C	CR	NI	MO	NB	V		Pm1		HT				
IT-9A	TP-304L	0.020	18.12	8.04	0.14	0	0.08		33		743096			✓	
IT-62	T-304L	0.021	18.43	8.11	0.29	0.018	0.078		34		COX3			✓	
IT-61	T-304L	0.014	18.76	8.40	0.29	0.007	0.053		35		5V05			✓	
IT-61A	↓	0.019	18.67	8.21	0.37	0.015	0.050		36		979475			✓	
IT-83A	F-304L	0.026	18.29	8.66	0.23	0	0.055		37		85853/1			✓	
IT-84A	↓	0.022	18.41	8.60	0.23	0	0.059		38		↓			✓	
IT-50	T-304L	0.021	18.57	8.06	0.36	0.015	0.047		39		<del>8050</del> 979475			✓	
IT-51	↓	0.020	18.55	8.10	0.36	0.014	0.052		40		979475			✓	

Dan [Signature]  
 Inspector

\_\_\_\_\_  
 Customer Representative

8441 South Union Tulsa, Oklahoma 74132  
(918) 446-1696

Cert. # \_\_\_\_\_

Customer FABSCG  
Mail Reports to: QC/QA  
Test Instrument ARC MET 8000  
SN 515666

Report Number: 64764  
Date: 9-12-12  
Job Number: 10279-7

[illegible]

Inspector

Customer Representative



# FERRITE CONTENT TESTING REPORT

FABSCO SHELL & TUBE, L.L.C. JOB NUMBER: SII-10279-8  
CUSTOMER NAME: SOUTHERN Co.  
CUSTOMER P. O. NUMBER: MPC17901  
CUSTOMER ITEM NUMBER: HX2065

Instrument: **FISCHER FERRITESCOPE MP30, S/N:085-22010A**

Probe: EGAB1.3-FE

**Reference Standards: S/N: N-2298**

FN ✓

Reference Calibration: \_\_\_\_\_

3042  
SA240-2162

[illegible]

DATE: 09.17.12

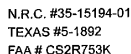
*John Burns*  
FABSCO SHELL & TUBE, L.L.C.  
QUALITY CONTROL DEPARTMENT  
NDE EXAMINER

REFERENCE FABSCO SHELL TUBE, L.L.C.  
NDE PROCEDURE # 34

FERRITE CONTENT TESTING REPORT  
12/08/2005  
REV. 0







8441 SOUTH UNION • TULSA, OKLAHOMA 74132 • (918) 446-1696

## RADIOGRAPHIC TECHNIQUE SHEET

PAGE OF

[illegible]

LIABILITY FOR LOSSES OF ANY KIND DUE TO INTERPRETATION



# GLOBE X-RAY SERVICES, INC.

8441 SOUTH UNION • TULSA, OKLAHOMA 74132 • (918) 446-1696

N.R.C. #35-15194-01  
TEXAS #5-1892  
FAA # CS2R753K

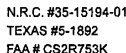
## RADIOGRAPHIC TECHNIQUE SHEET

PAGE \_\_\_\_\_ OF \_\_\_\_\_

MFG/CUST. <b>J+M</b>		JOB#/ID <b>93057.3 Shell (R)</b>		RT DATE <b>8-28-12</b>													
MAT'L TYPE <b>S</b>	MAT'L THK. <b>.432</b>	WELD REINF. <b>.125</b>	TOTAL THK. <b>.557</b>	RT TECHNIQUE <b>GXS-1B-03</b>													
ISOTOPE/ X-RAY VOLTAGE <b>Ir-192</b>	CI./MA. <b>89ci</b>	FOCAL SIZE <b>.159</b>	SOD <b>6 5/8</b>	OFD <b>.557</b>	EXPOSURE TIME <b>:17</b>												
EXPOSURE SINGLE WALL <input type="checkbox"/> DBL. WALL <input checked="" type="checkbox"/>		VIEWING SINGLE WALL <input checked="" type="checkbox"/> DBL. WALL <input type="checkbox"/>		FILM PROCESS MAN. <input checked="" type="checkbox"/> AUTO <input type="checkbox"/>													
FILM MFG/TYPE <b>AGFA D-5</b>	NO. EXPOSURES	RT QUALITY <b>2T</b>	APPLICABLE CODE SPEC. <b>ASME Sec. II</b>		ACCEPTANCE STD. <b>ASME VIII</b>												
RT TRCH. LEVEL II <b>A. Hall Lv. II</b>		<b>E. Hise</b>		RT PROCEDURE <b>GXS-1B</b>													
PART # SEAM # OR WELD #	LOCATION MARKER INTERVAL	W E L D E R	# H F O I L D M E /R	S S H I Z M E	I Q I & S I D E	D E N S I T Y	A C C E P T	R E J E C T	S L A G	P O R	C R A C K	I P	L F	U C	A R T I F I C I A L F I L M C T	COMMENTS	
<b>Noz 71</b>	<b>T-2</b>	<b>F</b>	<b>1</b>	<b>NA</b>	<b>IB</b>	<b>2-4</b>	<b>/</b>										
	<b>2-3</b>						<b>/</b>										
	<b>3-T</b>						<b>/</b>										
FILM INTERPRETATION BY: <b>[Signature]</b>																	
A.I./CUST. <b>[Signature]</b>															DATE: <b>8-27-12</b>		
DATE: <b>8-28-12</b>															DATE: <b>8-28-12</b>		

(GX-9)

WARRANTY FOR LOSSES OF ANY KIND DUE TO INTERPRETATION



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PAGE OF

[illegible]

GSX ASSUMES NO RESPONSIBILITY FOR LOSSES OF ANY KIND DUE TO INTERPRETATION

(GX-9)





# GLOBE X-RAY SERVICES, INC.

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N.R.C. #35-15194-01  
TEXAS #5-1892  
FAA # CS2R753K

## RADIOGRAPHIC TECHNIQUE SHEET

PAGE \_\_\_\_\_ OF \_\_\_\_\_

MFG/CUST. <b>J &amp; M</b>		JOB# / ID <b>93057.4</b>		RT DATE <b>9-6-12</b>	
MAT'L TYPE <b>SIS</b>		MAT'L THK. <b>1/4"</b>		WELD REINF. <b>1/25</b>	
TOTAL THK. <b>3/8"</b>		RT TECHNIQUE <b>GXS-I-B-02</b>			
ISOTOPE / X-RAY VOLTAGE <b>IR 192</b>		CI./MA. <b>81 ci.</b>		FOCAL SIZE <b>1154</b>	
SOD <b>83 1/4" / 17"</b>		OFD <b>38"</b>		EXPOSURE TIME <b>:16 / 1:00</b>	
EXPOSURE SINGLE WALL <input checked="" type="checkbox"/> DBL. WALL <input type="checkbox"/>		VIEWING SINGLE WALL <input checked="" type="checkbox"/> DBL. WALL <input type="checkbox"/>		FILM PROCESS MAN. <input checked="" type="checkbox"/> AUTO <input type="checkbox"/>	
FILM MFG/TYPE <b>Agfa D-5</b>		NO. EXPOSURES <b>2-2T</b>		RT QUALITY <b>2-2T</b>	
APPLICABLE CODE SPEC. <b>ASME Sec. V</b>		ACCEPTANCE STD. <b>ASME Sec. VIII</b>			
RT JRCH. LEVEL II <b>G. Zulpo Level II</b>		RT PROCEDURE <b>GXS-I-B</b>			

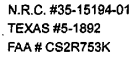
PART # SEAM # OR WELD #	LOCATION MARKER INTERVAL	W E L D E R	# H F O I L D L M E / R	SS H I I Z M E	IQI & S I D	D E N S I T Y	A C C E P T	R E J E C T	S L A G	P O R	C R A C K	I P	L F	U C	A R T I F I C I A L	COMMENTS
RS-1	T-2	T	1				/									
	2-3						/									
	3-4						/									
	4-T						/									
RS-2	T-2	T					/									
	2-3						/									
	3-4						/									
	4-T						/									
LS-1	1-2	A					/									
	2-3						/									

FILM INTERPRETATION BY: <b>Mary Zulpo Level II</b>		DATE: <b>9-6-12</b>	
A.I./CUST. <b>W Pope 9/7/12</b>		DATE: <b>9-7-12</b>	

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(GX-9)





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PAGE \_\_\_\_\_ OF \_\_\_\_\_

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Kemper County MM233798





# Protective Coating & Sandblasting Paint Report pg # 1

DATE 9-21-12 TIME 8 AM PM THIS SECTION COVERS SANDBLASTING  
CUSTOMER Fabryco JOB OR ITEM NO 10279-708

CONDITIONS CHECK ALL THAT APPLY CLEAR ☒ CLOUDY ☐ RAIN ☐ SNOW ☐

AMBIENT TEMP 71 R/H 66 DEW POINT 59 SURFACE TEMP 71

FIVE DEGREES ABOVE DEW POINT YES ☒ OR NO ☐ TYPE OF S/B SSPC-SP 10

CONDITION OF STEEL NEW ☒ MILL SCALE ☐ LT RUST ☐ PAINTED ☐

SURFACE PROFILE AFTER S/B 2.0 - 2.10 NOZZLE PRESSURE 110

THIS SECTION COVERS PAINT APPLICATION UNLESS NOTED ALL PAINTING IS DONE  
INDOORS IN CLIMATED CONTROLLED ENVIRONMENT

AMBIENT TEMP 80 R/H 45 DEW POINT 57 SURFACE TEMP 80

PRIMER

PRODUCT No 859 MFG Carbolinc BATCH No 12HD0739B SHELF LIFE 36/24/24

2<sup>ND</sup> COAT

PRODUCT No \_\_\_\_\_ MFG \_\_\_\_\_ BATCH No \_\_\_\_\_ SHELF LIFE \_\_\_\_\_

3<sup>RD</sup> COAT

PRODUCT No \_\_\_\_\_ MFG \_\_\_\_\_ BATCH No \_\_\_\_\_ SHELF LIFE \_\_\_\_\_

4<sup>TH</sup> COAT

PRODUCT No \_\_\_\_\_ MFG \_\_\_\_\_ BATCH No \_\_\_\_\_ SHELF LIFE \_\_\_\_\_

MILLAGE PER COAT 1<sup>ST</sup> 2 1/2 2<sup>ND</sup> \_\_\_\_\_ 3<sup>RD</sup> \_\_\_\_\_ 4<sup>TH</sup> \_\_\_\_\_ TOTAL \_\_\_\_\_

MILL GAUGE CALIBRATION IS DONE ON A DAILY BASES  
ATTACH PRESS-O-FILM TAPE BELOW



#7



#8

SIGNATURE Mark M. Brown DATE 9-21-12

Black Beauty was used for blasting

# Protective Coating & Sandblasting Paint Report pg # 2

DATE 9-22-12 TIME 9 AM/PM THIS SECTION COVERS SANDBLASTING

CUSTOMER Fabglo JOB OR ITEM NO 10279-748

CONDITIONS CHECK ALL THAT APPLY CLEAR ☒ CLOUDY ☐ RAIN ☐ SNOW ☐

AMBIENT TEMP 82 R/H 46 DEW POINT 60 SURFACE TEMP 82

FIVE DEGREES ABOVE DEPOINT YES ☒ OR NO ☐ TYPE OF S/B SSPC-SP       

CONDITION OF STEEL NEW ☐ MILL SCALE ☐ LT RUST ☐ PAINTED ☒

SURFACE PROFILE AFTER S/B        NOZZLE PRESSURE       

THIS SECTION COVERS PAINT APPLICATION UNLESS NOTED ALL PAINTING IS DONE  
INDOORS IN CLIMATED CONTROLLED ENVIRONMENT

AMBIENT TEMP 87 R/H 37 DEW POINT 58 SURFACE TEMP 87

PRIMER

PRODUCT No 859

MFG Carboline

BATCH No SH#1

SHELF LIFE 39/24/12.4

2<sup>ND</sup> COAT

PRODUCT No 890

MFG Carboline

BATCH No 12GD3400L

SHELF LIFE 36/15

3<sup>RD</sup> COAT

PRODUCT No       

MFG       

BATCH No       

SHELF LIFE       

4<sup>TH</sup> COAT

PRODUCT No       

MFG       

BATCH No       

SHELF LIFE       

MILLAGE PER COAT

1<sup>ST</sup> 2 1/2 - 3 2<sup>ND</sup> 4-6 3<sup>RD</sup>        4<sup>TH</sup>       

TOTAL       

MILL GAUGE CALIBRATION IS DONE ON A DAILY BASES  
ATTACH PRESS-O-FILM TAPE BELOW

SIGNATURE Mad Madson

DATE 9-22-12

# Protective Coating & Sandblasting Paint Report pg # 2

DATE 9-24-12 TIME 8 AM PM THIS SECTION COVERS SANDBLASTING

CUSTOMER Fabco JOB OR ITEM NO 10279-708

CONDITIONS CHECK ALL THAT APPLY CLEAR ☒ CLOUDY ☐ RAIN ☐ SNOW ☐

AMBIENT TEMP 84 R/H 35 DEW POINT 54 SURFACE TEMP 84

FIVE DEGREES ABOVE DEW POINT YES ☒ OR NO ☐ TYPE OF S/B SSPC-SP       

CONDITION OF STEEL NEW ☐ MILL SCALE ☐ LT RUST ☐ PAINTED ☒

SURFACE PROFILE AFTER S/B        NOZZLE PRESSURE       

THIS SECTION COVERS PAINT APPLICATION UNLESS NOTED ALL PAINTING IS DONE INDOORS IN CLIMATED CONTROLLED ENVIRONMENT

AMBIENT TEMP 90 R/H 36 DEW POINT 60 SURFACE TEMP 90

*ports*  
PRIMER Carboline  
PRODUCT No 859 MFG        BATCH No SH#1 SHELF LIFE 36/24/24  
2<sup>ND</sup> COAT Carboline  
PRODUCT No 890 MFG        BATCH No SH#2 SHELF LIFE 36/15  
3<sup>RD</sup> COAT Carboline  
PRODUCT No 134 MFG        BATCH No 12ER2676L SHELF LIFE 36/24  
4<sup>TH</sup> COAT         
PRODUCT No        MFG        BATCH No        SHELF LIFE       

MILLAGE PER COAT 1<sup>ST</sup> 2 1/2 - 3 2<sup>ND</sup> 4-6 3<sup>RD</sup> 2 1/2 - 3 1/2 4<sup>TH</sup>        TOTAL 9-12 1/2

MILL GAUGE CALIBRATION IS DONE ON A DAILY BASES  
ATTACH PRESS-O-FILM TAPE BELOW

SIGNATURE Mark M. Ben DATE 9-24-12







NB 7539

CERTIFIED BY  
FABSCO SHELL & TUBE, LLC  
SAPULPA, OKLAHOMAU  
W  
RT 4-S  
RT 1-TAREA  
230

	MAWP		MAWT	
SHELL	180	PSI AT	290	° F.
	MAEWP		MAWT	
SHELL	N/A	PSI AT	N/A	° F.
	MDMT		MAWP	
SHELL	10	° F. AT	180	PSI
	MAWP		MAWT	
TUBES	150	PSI AT	200	° F.
	MAEWP		MAWT	
TUBES	N/A	PSI AT	N/A	° F.
	MDMT		MAWP	
TUBES	10	° F. AT	150	PSI

SERIAL NUMBER

YEAR BUILT

ITEM/TAG NUMBER

S11.10279.8

2012

HX 2065

TEST PRESSURE (SHOP) SHELL = 234 PSI TUBES = 195 PSI

SERVICE: AGR FLASH GAS COOLER

P.O. NO.: MPC 17901





# *INSTALLATION, OPERATION & MAINTENANCE MANUAL*

*for*

## **SHELL AND TUBE HEAT EXCHANGERS**

INSTALLATION, OPERATION & MAINTENANCE MANUAL – Shell & Tube Heat Exchangers

**FABSCO**  
**SHELL & TUBE, LLC**

P.O. Box 988  
Sapulpa, Oklahoma 74066  
918/224-7550

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## **FORWARD**

Included in this manual are Tables and Figures which are reprinted in their entirety from the Standards of the Tubular Exchanger Manufacturers Association Seventh Edition (1988). Pages containing such material contain a citation to that publication.

Additionally, there are extensive extracts of text reprinted from the 7<sup>th</sup> Edition TEMA Standards. Pages containing such excerpts contain a footnote to that effect.

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# **GENERAL INSTRUCTIONS FOR INSTALLATION, OPERATION AND MAINTENANCE OF SHELL AND TUBE HEAT EXCHANGERS**

## **SAFETY CONSIDERATIONS**

### **LIFTING DEVICES**

When eyebolts, lugs, clips or other devices are installed by the manufacturer, it is intended that these lifting devices are to lift only the empty components to which they are attached and not the assembled heat exchanger, unless explicitly specified otherwise.

Proper rigging must be used in lifting assembled heat exchangers. Exchangers are usually much heavier than pressure vessels of comparable dimensions; therefore, weights should be carefully evaluated. Where available, determinations should include reference to manufacturers drawings, added components, bill of lading, etc. Special care must be exercised in handling exchangers with expansion joints.

### **ROUTING PRECAUTIONARY PROCEDURES**

A heat exchanger is a multi-chamber pressure vessel designed at specific limits of pressure, temperature and fluid flow conditions as shown on the exchanger specification sheet and heat exchanger nameplate(s). The process system, which includes the heat exchanger, must be safeguarded so that the heat exchanger design conditions and operating limits are not exceeded. All operating and maintenance personnel should be made aware of specific limitations including pressures and temperatures flow rates, start-up/shut-down procedures and cleaning procedures.

Do not remove channel covers, shell covers, floating head covers, bonnets or connecting piping until all pressure has been relieved and both shell and tube sides are completely drained. Exceptions are permissible when design permits independent pressure testing of shell or tube sides.

Plugged tubes and double tubesheets, unless vented, may remain pressurized after shell and tube side are depressurized. Caution may be exercised in loosening of tube plugs or opening of vents to avoid sudden release of pressure or harmful fluids.

Many heat exchangers circulate fluids, which are toxic, lethal or flammable and dangerous to the human system. These fluids could cause problems if bolted or threaded joints are not maintained in leak-tight condition under operating or no-flow ambient conditions. Proper precautions, such as effective draining and purging, must be taken in handling and decontamination when exchangers are opened for any reason. It is essential that the user advises maintenance or repair personnel, including outside contractors, when toxic, lethal, or flammable conditions exist.

When heat exchangers are cleaned, it is important that full characteristics of the circulating fluids, fouling material, and the cleaning agent be known and care exercised in handling them. Use eye protection, a respirator, or other appropriate protective devices.

Do not blow out heat exchangers with air when the process fluids or the cleaning fluids being handled are flammable or reactive.

# SECTION 1 HEAT EXCHANGER NOMENCLATURE

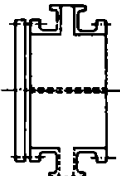
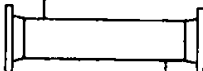

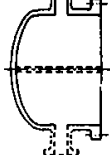
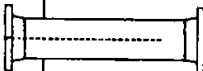

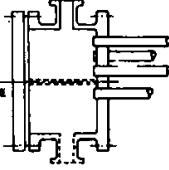
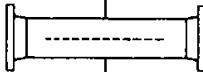

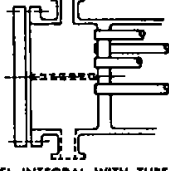
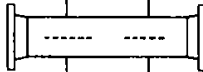

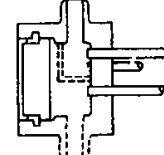
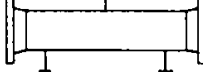

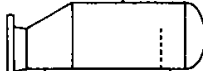

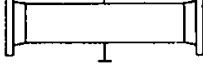
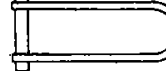

## N-1 TYPE DESIGNATION-RECOMMENDED PRACTICE

It is recommended that heat exchanger type be designated by letters as described below.

### N-1.1 TYPE

Type designation shall be by letters describing stationary head, shell (omitted for bundles only), and rear head, in that order, as indicated in Figure N-1.1.

FIGURE N-1.1

FRONT END STATIONARY HEAD TYPES		SHELL TYPES		REAR END HEAD TYPES	
<b>A</b>	 CHANNEL AND REMOVABLE COVER	<b>E</b>	 ONE PASS SHELL	<b>L</b>	 FIXED TUBESHEET LIKE "A" STATIONARY HEAD
<b>B</b>	 BONNET (INTEGRAL COVER)	<b>F</b>	 TWO PASS SHELL WITH LONGITUDINAL BAFFLE	<b>M</b>	 FIXED TUBESHEET LIKE "B" STATIONARY HEAD
<b>C</b>	 REMOVABLE TUBE BUNDLE ONLY CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER	<b>G</b>	 SPLIT FLOW	<b>N</b>	 FIXED TUBESHEET LIKE "N" STATIONARY HEAD
<b>N</b>	 CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER	<b>H</b>	 DOUBLE SPLIT FLOW	<b>P</b>	 OUTSIDE PACKED FLOATING HEAD
<b>D</b>	 SPECIAL HIGH PRESSURE CLOSURE	<b>J</b>	 DIVIDED FLOW	<b>S</b>	 FLOATING HEAD WITH BACKING DEVICE
		<b>K</b>	 KETTLE TYPE REBOILER	<b>T</b>	 PULL THROUGH FLOATING HEAD
		<b>X</b>	 CROSS FLOW	<b>U</b>	 U-TUBE BUNDLE
				<b>W</b>	 EXTERNALLY SEALED FLOATING TUBESHEET

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## SECTION 1 HEAT EXCHANGER NOMENCLATURE

### N-2 NOMENCLATURE OF HEAT EXCHANGER COMPONENTS

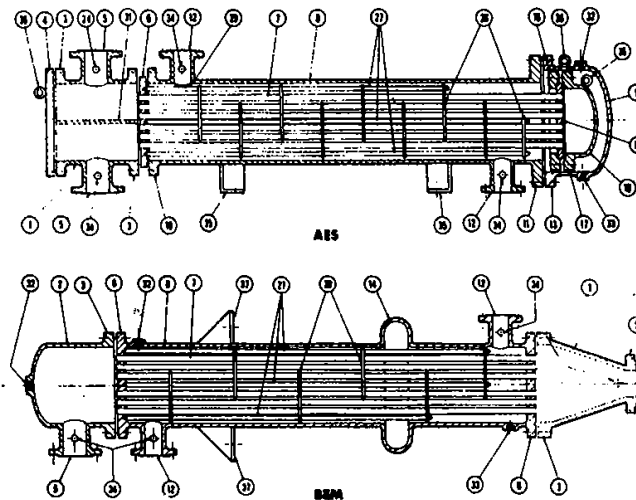
For the purpose of establishing standard terminology, Figure N-2 illustrates various types of heat exchangers. Typical parts and connections, for illustrative purposes only, are numbered for identification in Table N-2. (See Note).

**TABLE N-2**

1. Stationary Head – Channel	20. Slip-on Backing Flanges
2. Stationary Head – Bonnet	21. Floating Head Cover – External
3. Stationary Head Flange – Channel or Bonnet	22. Floating Tubesheet Skirt
4. Channel Cover	23. Packing Box
5. Stationary Head Nozzle	24. Packing
6. Stationary Tubesheet	25. Packing Gland
7. Tubes	26. Lantern Ring
8. Shell	27. Tie rods and Spacers
9. Shell Cover	28. Transverse Baffles or Support Plates
10. Shell Flange – Stationary Head End	29. Impingement Plate
11. Shell Flange – Rear Head End	30. Longitudinal Baffle
12. Shell Nozzle	31. Pass Partition
13. Shell Cover Flange	32. Vent Connection
14. Expansion Joint	33. Drain Connection
15. Floating Tubesheet	34. Instrument Connection
16. Floating Head Cover	35. Support Saddle
17. Floating Head Flange	36. Lifting Lug
18. Floating Head Backing Device	37. Support Bracket
19. Split Shear Ring	38. Weir
	39. Liquid Level Connection

NOTE: Manufacturers may identify corresponding parts  
Using numbers differing from the above.

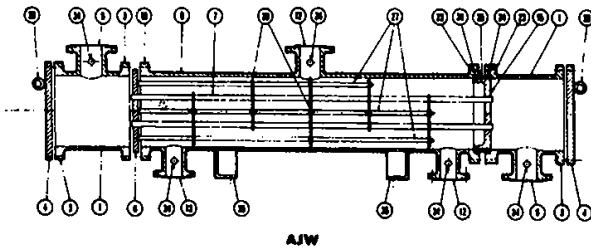
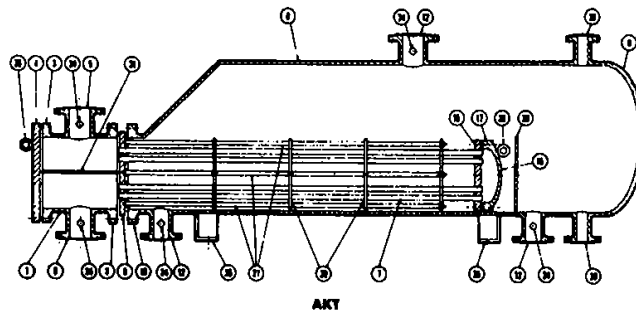
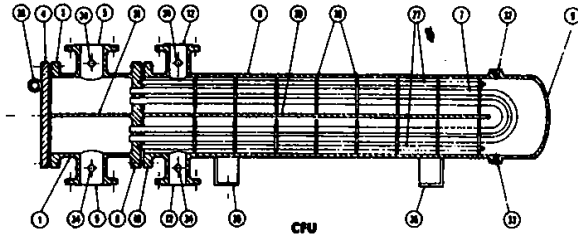
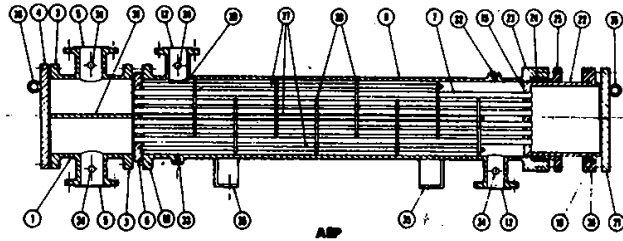
**Figure N-2**



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**SECTION 1**  
**HEAT EXCHANGER NOMENCLATURE**

Figure N-2 (Continued)



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## **SECTION 2**

### **HEAT EXCHANGER IDENTIFICATION AND DEFINITIONS**

---

#### **D-2.1 EXCHANGER IDENTIFICATION**

##### **D-2.11 COMPLETE EXCHANGER**

Manufacturers normally assigned on a serial number, which is unique and permanently identifies each exchanger. Inquiries to the manufacturer should reference this number when it is shown on the nameplate and manufacturer's drawings. Additional useful information normally shown includes size, type, and item number.

##### **D-2.12 EXCHANGER PARTS**

Exchanger parts should be identified by the exchanger serial number and by the name and part number corresponding to those shown in Section 1, Table N-2, or as shown on the manufacturer's drawings.

#### **D-2.2 DEFINITIONS**

##### **D-2.21 EXCHANGER UNIT**

One or more exchangers designed for a specific service.

##### **D-2.22 ITEM NUMBER**

The purchaser's identification number for an exchanger unit.

##### **D-2.23 NORMAL OPERATING CONDITIONS**

The thermal and hydraulic performance requirements generally specified for designing the heat exchanger.

##### **D-2.24 UPSET CONDITIONS**

A departure from specified operating conditions.

##### **D-2.25 PULSATING FLUID CONDITIONS**

Conditions of flow generally characterized by rapid fluctuations in pressure and flow rate resulting from sources outside of the heat exchanger such as: pumps, compressors, blowers, etc.

##### **D-2.26 START-UP CONDITIONS**

The condition of operation, which exist from the time that flow of either or both process streams, is initiated to the time that steady-state operating conditions are achieved.

##### **D-2.27 SHUTDOWN CONDITIONS**

The conditions of operation, which exist, from the time of steady-state operating conditions to the time that flow of both process streams has ceased.

##### **D-2.28 CODE**

All references to Code herein mean the ASME Boiler and Pressure Vessel Code, Section VIII, Division 1, Pressure Vessels.

## **SECTION 3**

### **INSTALLATION OF HEAT EXCHANGERS**

---

#### **E-3 INSTALLATION**

##### **E-3.1 INSTALLATION PLANNING**

###### **E-3.11 CLEARANCE FOR DISMANTLING**

For straight tube exchangers fitting with removable bundles, provide sufficient clearance at the stationary head end to permit removal of the bundle from the shell and provide adequate space beyond the rear head and to permit removal of the shell cover and/or floating head cover.

For fixed tubesheet exchangers, provide sufficient clearance at one end to permit withdrawal and replacement of the tubes and enough space beyond the head at the opposite end to permit removal of the bonnet or channel cover.

For U-tube heat exchangers, provide sufficient clearance at the stationary head end to permit withdrawal of the tube bundle, or at the opposite end to permit removal of the shell.

###### **E-3.12 FOUNDATIONS**

Foundations must be adequate so that exchangers will not settle and impose excessive strains on the exchanger. Foundation bolts should be set to allow for setting inaccuracies. In concrete footings, pipe sleeves at least one size larger than bolt diameter slipped over the bolt and cast in place are best for this purpose, as they allow the bolt center to be adjusted after the foundation has set.

###### **E-3.13 PIPING**

###### **E-3.131 THERMAL EXPANSION**

It is essential that adequate provisions be made in the connecting piping to allow for thermal expansion without imposing unspecified piping loads on the exchanger. This is particularly important for single pass internal floating head designs, outside packed floating heads, and fixed tubesheet types with shell expansion joints. Unless precautions are taken, leakage or damage may result.

###### **E-3.132 PULSATION AND VIBRATION**

In all installations, care must be taken to eliminate or minimize transmission of fluid pulsations and mechanical vibrations to the heat exchangers.

###### **E-3.14 CLEANING**

Convenient means should be provided for cleaning the unit as suggested under "Maintenance." (Section 5).

###### **E-3.15 LEVELING**

Exchangers must be installed so that pipe connections are made without forcing, disassembling or loosening of flanged assemblies.

###### **E-3.16 STACKED HEAT EXCHANGERS**

The manufacturer normally stacks exchangers prior to completion of fabrication to insure proper alignment of interconnecting nozzles. Shims separating the intermediate shell supports are normally used. Therefore, it is important that shims, if used, match the thicknesses and locations used by the manufacturer.

### **SECTION 3**

#### **INSTALLATION OF HEAT EXCHANGERS**

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##### **E-3.17 CONNECTION PROTECTORS**

Remove shipping covers and plugs immediately prior to installation. Before connecting to piping, inspect all openings in the heat exchanger or foreign material. The entire system should be clean and free of foreign objects before starting operation. Do not expose heat exchanger internals to moisture or harmful contaminants.

##### **E-3.18 PIPE CONNECTIONS**

Pipe connections must be in accordance with the manufacturer's dimensional outline drawing.

##### **E-3.19 FOUNDATION BOLTS**

Foundation bolts should be loosened at one end of the unit to allow free expansion of shells. Slotted holes in supports are provide for this purpose.

E-3.20 If the heat exchanger is equipped with a bellows type expansion joint, the expansion joint must be adequately protected during shipping and setting. Shipping supports must be removed in accordance with instructions.

##### **E-3.3 HEAT EXCHANGER STORAGE**

Heat exchangers normally are not protected for storage. If storage is necessary, a covered location at uniform temperature is preferable and provisions must be made to protect the heat exchanger interior from moisture, freezing, or harmful contaminants.

## SECTION 4

### OPERATION OF HEAT EXCHANGERS

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#### E-4 OPERATION

##### E-4.1 PERFORMANCE OF HEAT EXCHANGERS

Satisfactory performance and service life can be expected only from heat exchangers which are properly installed, operated in accordance with design conditions, and receive preventative maintenance and cleaning on an appropriate schedule. These requirements are the responsibility of the user.

##### E-4.11 PERFORMANCE FAILURES

The failure of heat exchanger equipment to perform satisfactorily may be caused by one or more factors, such as:

- (1) Excessive fouling.
- (2) Air or gas binding resulting from improper piping installation or lack of suitable vents.
- (3) Operating conditions differing from design conditions.
- (4) Mal-distribution of flow in the unit.
- (5) Excessive clearances between the baffles and shell and/or tubes, due to corrosion.
- (6) Improper thermal design.
- (7) Flooding resulting from inadequate drainage of condensate.

##### E-4.2 OPERATION OF HEAT EXCHANGERS

##### E-4.21 OPERATING PROCEDURES – NORMAL

Before placing any exchanger in operating, reference should be made to the exchanger drawings, specification sheet(s), and name plate(s) for any special instructions. Local safety and health regulations must be considered. Improper start-up or shutdown sequences, particularly of fixed tubesheet units, may cause leaking of tube-to-tubesheet and/or bolted flanged joints.

##### E-4.22 START-UP OPERATION

Most exchangers with removable tube bundles may be placed in service by first establishing circulation of the cold medium, followed by the gradual introduction of the hot medium. During start-up all vent valves should be opened and left open until all passages have been purged of air and are completely filled with fluid. For fixed tubesheet exchangers, fluids must be introduced in a manner to minimize differential expansion between the shell and tubes. (Refer E-4.24).

##### E-4.23 SHUT-DOWN OPERATION

For exchangers with removable bundles, the units may be shut down by first gradually stopping the flow of the hot medium and then stopping the flow of the cold medium. If it is necessary to stop the flow of cold medium, the circulation of hot medium through the exchanger should also be stopped. For fixed tubesheet exchangers the unit must be shut down in a manner to minimize differential expansion.

##### E-4.24 START-UP AND SHUT-DOWN PROCEDURES FOR FIXED TUBESHEET EXCHANGERS

When start-up and shutdown instructions are furnished, they must be followed. When specific instructions are not supplied, the recommended procedure is to start, or stop, the circulation of both shell and tube side fluids gradually and at the same time.

## SECTION 4

### OPERATION OF HEAT EXCHANGERS

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#### E-4.3 OPERATING LIMITATIONS

##### E-4.31 NORMAL OPERATING CONDITIONS

Equipment must not be operated at conditions more severe than those specified on the heat exchanger specification sheet, and/or nameplates. A change in service or deviation from the specified operating conditions may require that a mechanical design check be made in order to avoid possible damage to the heat exchanger.

##### E-4.32 SUPPLEMENTAL NAME PLATE DATA

The manufacturer may supply supplemental data where it is pertinent to the operation or testing of the exchanger. This may include information pertaining to differential design and test pressure conditions, restrictions on operating conditions for fixed tubesheet type exchangers, or other restrictive conditions applicable to the design and/or operation of the unit or its components. Such information may be noted on the standard nameplate or on a supplemental plate attached to the exchanger at the nameplate location.

##### E-4.33 FLOW RATE

Heat exchangers are not to be operated at flow rates greater than those shown on the heat exchanger specification sheet. Excessive flow rates can cause vibration and/or erosion and severely damage the heat exchanger.

##### E-4.34 HYDRAULIC HAMMER

Exchangers should not be subjected to pulsating, slugging, or unstable flow conditions, which may impair performance or result in destructive hydraulic hammer. Adequate condensate drainage is essential for steam heaters and similar apparatus.

##### E-4.35 TEMPERATURE SHOCKS

Exchangers normally should not be subjected to abrupt temperature fluctuations. Hot fluid must not be suddenly introduced when the unit is cold, nor cold fluid when the unit is hot.

##### E-4.36 FLUID TEMPERATURE LIMITS

E-4.361 Heat exchangers are not to be operated at fluid temperature more severe than those shown on the heat exchanger specification sheet.

E-4.362 For fixed tubesheet exchanger's fluid inlet temperatures may be severely limited when there is no fluid on the opposite side. Operation with fluid circulation on one side only should be avoided, unless specifically provided for in the design of the heat exchanger.

#### E-4.4 BOLTED JOINTS

##### E-4.41 RECOMMENDATIONS FOR RETIGHTENING

Heat exchangers are pressure tested before leaving the manufacturer's shop in accordance with ASME Code requirements. However, normal yielding of gasket joints may occur in the interval between testing in the manufacturer's shop and installation at the job site. Therefore, all external bolted joints may require retightening after installation and, if necessary, after the exchanger has reached operating temperature.

## SECTION 4

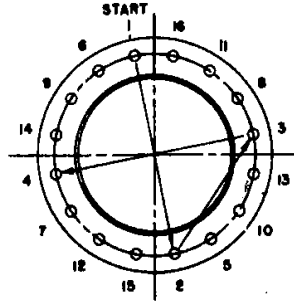
### OPERATION OF HEAT EXCHANGERS

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#### E-4.3 RECOMMENDED BOLT TIGHTENING PROCEDURE

It is important that all bolted joints be tightened uniformly and in a diametrically staggered pattern as illustrated in Figure E-4.2 except for special high pressure closures when the instructions of the manufacturer should be followed.

Figure E-4.2



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## SECTION 5 MAINTENANCE OF HEAT EXCHANGERS

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### E-5 MAINTENANCE

#### E-5.1 INSPECTION OF UNIT

At regular intervals and as frequently as experience indicates, an examination should be made of the interior and exterior condition of the unit. Neglect in keeping all tubes clean may result in complete stoppage of flow through some tubes, which could cause severe thermal strains, leaking tube joints, or structural damage to other components. Sacrificial anodes, when provided, should be inspected to determine whether they should be cleaned or replaced.

#### E-5.11 INDICATIONS OF FOULING

Exchangers subject to fouling or scaling should be cleaned periodically. A light sludge or scale coating on the tube greatly reduces its efficiency. A marked increase in pressure drop and/or reduction in performance usually indicate cleaning is necessary. The unit should first be checked for air or vapor binding to confirm that this is not the cause for the reduction in performance. Since the difficulty of cleaning increases rapidly as the scale thickness or deposit increases, the intervals between cleanings should not be excessive.

#### E-5.12 PREVENTATIVE MAINTENANCE

Additional important reasons for inspections include:

- (1) Corrosion and erosion may weaken various parts of an exchanger and may eventually cause mechanical failure, or impaired performance.
- (2) Determination of fouling and corrosion rates for prediction of service life scheduling downtime.
- (3) Implementation of preventive maintenance procedures, which reduce rates of fouling and general deterioration, or eliminate leaks.

#### E-5.13 ACCESS TO TUBES

Before disassembly, the user must assure himself that the unit has been depressurized, vented and drained, neutralized and/or purged of hazardous materials.

To inspect the inside of the tubes and also make them accessible for cleaning, the following procedures should be used:

- (1) Stationary Head End
  - (a) Type A, C, & D, removes cover only.
  - (b) Type B, remove bonnet.
- (2) Rear Head End
  - (a) Type L, N, & P, remove cover only.
  - (b) Type M, remove bonnet.
  - (c) Type S & T, remove shell cover and floating head cover.
  - (d) Type W, remove channel cover or bonnet.

## SECTION 5 MAINTENANCE OF HEAT EXCHANGERS

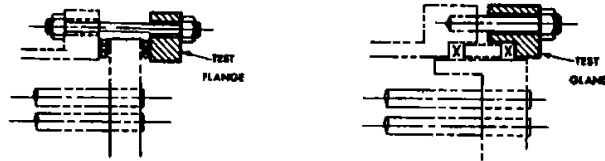
### E-5.14 LOCATING LEAKS IN TUBES

The following procedures may be used to locate perforated or split tubes and leaking joints between tubes and tubesheets. In most cases, the entire front face of each tubesheet will be accessible for inspection. The point where water escapes indicates a defective tube or tube-to-tubesheet joint.

- (1) Units with removable channel cover:  
Remove channel cover and apply hydraulic pressure in the shell.
- (2) Units with bonnet type head:
  - (a) On fixed tubesheet units where tubesheets are an integral part of the shell, remove bonnet and apply hydraulic pressure in the shell.
  - (b) On fixed tubesheet units where tubesheets are not an integral part of the shell and on units with removable bundles, remove bonnet, re-bolt tubesheet to shell or install test flange or gland, whichever is applicable, and apply hydraulic pressure in the shell.See Figure E-5.14-1 for typical test flange and test gland.

**CAUTION:** Bolting directly to the edges of the tubesheets may result in overstress unless it was considered in the design of the tubesheets.

**Figure E-5.14-1**

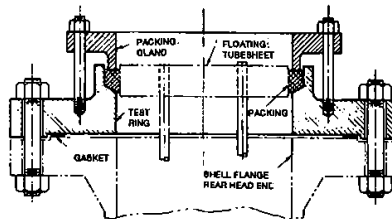


- (3) Units with type S or T floating head:  
Remove channel cover or bonnet, shell cover and floating head cover. Install test ring and bolt in place with gasket and packing. Apply hydraulic pressure in the shell. A typical test ring is shown in Figure E-5.14-2. When a test ring is not available it is possible to locate leaks in the floating head end by removing the shell cover and applying hydraulic pressure in the tubes. Leaking tube joints may then be located by sighting through the tube lanes.

**NOTE:** Care must be exercised when testing partially assembled exchangers to prevent over extension of expansion joints or overloading of tubes and/or tube-to-tube-to-tubesheet joints.

Hydrostatic test should be performed so that the temperature of the metal is at least 30°F above the minimum design metal temperature, MDMT.

**Figure E-5.14-2**



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## SECTION 5

### MAINTENANCE OF HEAT EXCHANGERS

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#### E-5.2 TUBE BUNDLE REMOVAL

Bundles must be withdrawn from the stationary end as noted on the drawing. All parts such as glands, retainer rings, split shear rings and closure packing must be removed before attempting to remove bundle. Threaded eyebolts may be attached to the stationary tubesheet when tapped holes have been provided for this purpose. When eyebolt tappings are not provided, rods may be passed through two or more tubes and attached to a steel bearing plate over the floating tubesheet. Provide a protective spacer between the bearing plate and tubesheet to protect the tube ends.

#### E-5.21 HANDLING TUBE BUNDLES

Tube bundles should not be handled with hooks or other devices, which might damage the tubes. Bundles should be supported on cradles or skids. Horizontal tube bundles should be lifted by means of suitable slings. Baffles can be bent and damaged by dragging a bundle over a rough surface. All gasket and packing contact surfaces should be protected from accidental damage since these areas are generally difficult to repair.

#### E-5.3 CLEANING TUBE BUNDLES

##### E-5.31 CLEANING METHODS

The heat transfer surfaces of heat exchangers should be kept reasonably clean to assure satisfactory performance. Convenient means for cleaning should be made available.

Heat exchangers may be cleaned by either chemical or mechanical methods. The method selected must be the choice of the operator of the plant and will depend on the type of deposit and the facilities available in the plant. Following are several cleaning procedures that may be considered:

- (1) Circulating hot wash oil or light distillate through tubes or shell at high velocity may effectively remove sludge or similar soft deposits.
- (2) Some salt deposits may be washed out by circulating hot fresh water.
- (3) Some commercial cleaning compounds are available for removing sludge or scale provided hot wash oil or water is not available or does not give satisfactory results.
- (4) High pressure water jet cleaning.
- (5) Scrapers, rotating wire brushes, and other mechanical means for removing hard scale, coke, or other deposits.
- (6) Employ services of a qualified organization that provides cleaning services. These organizations will check the nature of the deposits to be removed, furnish proper solvents and/or acid solutions containing inhibitors, and provide equipment and personnel for a complete cleaning job.

##### E-5.32 CLEANING PRECAUTIONS

- (1) Tubes should not be cleaned by blowing steam through individual tubes since this heats the tube and may result in severe expansion strain, deformation of the tube, or loosening of the tube-to-tubesheet joint.
- (2) When mechanically cleaning a tube bundle, care should be exercised to avoid damaging the tubes.
- (3) Cleaning compounds must be compatible with the metallurgy of the exchanger.

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## SECTION 5

### MAINTENANCE OF HEAT EXCHANGERS

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- (4) Cleaning procedures, especially those which circulate hot fluids through the tubes, can cause significant temperature differentials between the shell and tubes. Such cleaning procedures can cause damage to fixed tubesheet heat exchangers and should be avoided unless specifically provided of in the design of the heat exchanger.

#### E-5.4 GASKET REPLACEMENT

Gasket and gasket surfaces should be thoroughly cleaned and should be free of scratches and other defects. Gaskets should be properly positioned before attempting to retighten bolts. It is recommended that when a heat exchanger is dismantled for any cause, it's to be reassembled with new gaskets. This will tend to prevent future leaks and/or damage to the gasket seating surfaces of the heat exchanger. Composition gaskets become dried out and brittle so that they do not always provide an effective seal when reused. Metal or metal-jacketed gaskets, when compressed initially, flow to match their contact surfaces. In so doing they are work hardened and, if reused, may provide an imperfect seal or result in deformation and damage to the gasket contact surfaces of exchanger.

Bolted joints and flanges are designed for use with the particular type of gasket specified. Substitution of a gasket of different construction or improper dimensions may result in leakage and damage to gasket surfaces. Therefore, any gasket substitutions should be of compatible design.

Any leakage at a gasketed joint should be rectified and not permitted to persist as it may result in damage to the gasket surfaces.

When metal jacketed filled type gaskets are used with a tongue and groove joint without a nubbin, the gasket should be installed so that the tongue bears on the seamless side of the gasket jacket. When a nubbin is used, the nubbin should bear on the seamless side.

#### E-5.5 SPARE AND REPLACEMENT PARTS

The procurement of spare or replacement parts from the manufacturer will be facilitated if the correct name for the part, as shown in Section 1, Table N-2, of this manual is given, together with the serial number, type, size, and other information from the nameplate. Replacement parts should be purchased from the original manufacturer.

#### E-5.6 EXPANDED TUBE JOINTS

##### E-5.61 TUBE HOLE FINISH

Tube hole finish affects the mechanical strength and leak tightness of an expanded tube-to-tubesheet joint. In general:

- (1) A rough tube hole provides more mechanical strength than a smooth tube hole. This is influenced by a complex relationship of modulus of elasticity; yield strength, and hardness of the materials being used.
- (2) A smooth tube hole does not provide the mechanical strength that a rough tube hole does, but it can provide a pressure tight joint at a lower level of tube wall reduction.
- (3) Very light wall tubes require a smother tube hole finish than heavier wall tubes.
- (4) Significant longitudinal scratches can provide leak paths through an expanded tube-to-tubesheet joint and should therefore be removed.

## SECTION 5

### MAINTENANCE OF HEAT EXCHANGERS

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#### E-5.62 TUBE WALL REDUCTION

The optimum tube wall reduction for an expanded tube-to-tubesheet joint depends on a number of factors, some of which are:

- (1) Tube hole finish.
- (2) The presence or absence of the tube hole serrations (grooves).
- (3) The tube hole size and tolerance.
- (4) Tubesheet ligament width and its relation to tube diameter and thickness.
- (5) Tube wall thickness.
- (6) Tube hardness and change in hardness during cold working.
- (7) Tube O.D. tolerance.
- (8) Type of expander used.
- (9) Type of torque control or final tube thickness control.
- (10) Function of tube joint, i.e., strength in resistance to pulling out, minimum cold work for corrosion purposes, freedom from leaks, ease of replacement, etc.
- (11) Length of expanded joint.
- (12) Compatibility of tube and tubesheet materials.

#### E-5.63 TUBE EXPANDING

A suitable tube expander should be used to tighten a leaking tube joint. Care should be taken to insure that tubes are not over expanded. Do not re-expand tubes, which are not leaking. Do not expand tubes beyond 1/8" from the back face of the tubesheet.

#### E-5.64 TYPICAL COMPUTATION OF TUBE WALL REDUCTION

##### *EXAMPLE OF DETERMINING THE EXPANDED TUBE INSIDE DIAMETER FOR A SPECIFIED AMOUNT OF WALL REDUCTION*

**GIVEN:** TUBE DIAMETER = 0.750  
TUBE HOLE DIAMETER = 0.760  
TUBE WALL THICKNESS = 0.065  
SPECIFIED WALL REDUCTION = 5%

**SOLUTION:** WALL REDUCTION =  $0.065 \times 0.05 = 0.00325$   
FINAL WALL THICKNESS =  $0.065 - 0.00325 = 0.06175$   
FINAL EXPANDED TUBE I.D. =  $0.760 - 2(0.06175) = 0.6365$

**CAUTION:** *THE USE OF NOMINAL DIMENSIONS, WITHOUT ATTENTION TO PERMISSIBLE TOLERANCES CAN LEAD TO INCORRECT CONCLUSIONS.*

*EXPANDING PROCEDURES SHOULD BE BASED ON MEASURED DIMENSIONS, NOT NOMINAL DIMENSIONS.*

## SECTION 5 MAINTENANCE OF HEAT EXCHANGERS

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### E-5.65 PLUGGING TUBES IN TUBE BUNDLES

In U-tube heat exchangers, and other exchangers of special design, it may not be feasible to remove and replace defective tubes. Defective tubes may be plugged using commercially available tapered plugs with ferrules or tapered only plugs, which may or may not be seal welded. Excessive tube plugging may result in reduced thermal performance, higher-pressure drop, and/or mechanical damage.

### E-5.7 WELDED TUBE JOINTS

Welding procedure and testing techniques for either seal welded or strength welded tube joints normally are by agreement between manufacturer and purchaser and therefore should be considered prior to welded tube joints or tube replacements.

### E-5.8 END FLANGES AND BOLTING

The purpose of this section is to alert the users, and provide basic data for consideration in assembly and maintenance of bolted flanged joints. Controlling the torque applied in tightening threaded bolts is the most economical and commonly used method for the control of initial tension. There are several factors, which affect the relationship between torque and tension of threaded bolts. A few of the factors are the type of lubricant and/or plating, if any, used on the threads and the materials from which the bolt and nut are manufactured.

In assembling gasketed joints it is essential that the gasket first be seated uniformly by initial tightening of studs using a staggered quarterly pattern. After the gasket is seated, torquing may be completed stepwise to the final desired value, following a staggered or sequential pattern.

The following may be used as references and aids when assembling bolted flanged joints:

1. Recommended assembly and maintenance of flanged joints as shown in the Crane Company Catalog #60, Engineering Data Section.
2. Torque Control, Sixth Edition, Sturtevant Company.
3. ASME Code, Appendix S.

### E-5.9 SPECIAL HIGH PRESSURE CLOSURES

### E-5.10 REPAIRS, ALTERATION, RERATING, AND REPLACEMENT PARTS.

In order to maintain the Code integrity of the vessel, any repairs by welding, subsequent heat-treating, alterations, replacement pressure parts, or re-rating must be done in accordance with the National Board Inspection Code.

Potentially toxic or harmful contaminants must be neutralized prior to repairs.

Following repair by welding, post weld heat treatment may be a requirement for preservation of resistance to corrosion.