

DOCUMENTATION FILE

TUBERATE Ltd. Serial Number:

5457 A

Customer:

HUNTSMAN CORPORATION

Customer P.O. #

970094 T

Customer Item #

E - 201


End User:

HUNTSMAN CORPORATION

Plant Location:

MARYSVILLE, MICHIGAN**CONTENTS:**

One (1) set of Certified Design Calculations
One (1) set of Certified "As-Built" Drawings
Inspection Hold Points Check List
Material Control & Identification Sheet - Bill of Material
Mill Test Reports for Materials
Quality Control Inspection Report - Acceptance Record
Weld Procedures and Welder Qualifications
Weld Map
Tube Expansion Data Sheet
Hardness Reports
Radiography Map and Results
Certified Hydrostatic Test Reports
Nameplate Stamping
Form U-1 Manufacturer's Data Report
Inspection Check Sheet
Signed Packing Slip

HUNTSMAN, MARYSVILLE	
W.O. #	/ CER. # ____ / ____
P.O. #	_____
DOC. #	<u>21-AG-EC201-D1</u> 

12-1-97

NOTE:

THIS DOCUMENTATION FILE REMAINS IN THE CUSTODY OF TUBERATE Ltd.
A COPY IS MADE AVAILABLE TO THE CUSTOMER UPON REQUEST.

TUBE RATE Ltd.

Sarnia, Ontario CANADA

SHEET 1 OF 37
 REVISION: 1
 JOB NUMBER: 5457
 BY: CZ
 DATE: 1/8/97

DESIGN CALCULATIONS

CLIENT:	HUNTSMAN, MARYSVILLE, MI
PROJECT:	POLYPROPYLENE DEBOTTLENECKING PROJECT
PURCHASE ORDER NO.:	961974
DESCRIPTION:	FIRST REACTOR OVHD CONDENSER
ITEM NUMBER:	E-201
TEMA TYPE: BEM	SIZE: 36-240 PASSES: SS 1/TS 1 TEMA CLASS: "C"

CODES AND STANDARDS: ASME SECTION VIII DIV. 1, 1995 EDITION AND DEC.31,1995 ADDENDA
 TEMA 7th. EDITION 1988

DESIGN DATA	SHELL SIDE	TUBE SIDE
DESIGN PRESSURE (Psig)	430	638
DESIGN TEMPERATURE (°F)	210	200
CORROSION ALLOWANCE (in.)	1/8	0
RADIOGRAPHY	SPOT	FULL
P.W.H.T.	NO	NO
HYDROSTATIC TEST PRESS. (Psig)	645	957
MDMT (°F @ Psig)	-20 @ 430	-20 @ 638
IMPACT TEST	EXEMPT PER UG-20(f)	EXEMPT PER UHA-51(d)

MATERIAL	SHELL SIDE	TUBE SIDE
CYLINDER	SA 516-70	SA 240-304
HEAD/COVER	-	SA 240-304
NOZZLE NECKS	SA 106 Gr.B	SA 312-TP304
NOZZLE FLANGES	SA 105	SA 182-F304
REINFORCING PADS	SA 516-70	SA 240-304
GIRTH FLANGES	-	SA 182-F304
TUBESHEET	SA 240-304	
TUBES	SA 249-TP304	
SADDLES	SA 516 70 / SA 36	
BOLTS/NUTS	SA 193 B7 / SA 194 2H	

NOTE: SHELL MEAN METAL TEMP. = 93.5 °F
 TUBE MEAN METAL TEMP. = 98.0 °F

STAMP

REV	BY	DATE	DESCRIPTION	APP'D
1	CZ	1/8/97	REVISED PAGE 4, CHANNEL CAL'S ADDED NOZZLE MARKS IN REPORT	HRK
0	GSV/CZ	12/10/96	FOR REVIEW AND APPROVAL	HRK

FORM U-1 MANUFACTURER'S DATA REPORT FOR PRESSURE VESSELS Owner Item#
As Required by the Provisions of the ASME Code Rules, Section VIII, Division 1 E-201

Manufactured and certified by TUBERATE Ltd. 763 Chester St. Sarnia, Ont. N7S-5N2 Canada
(Name and address of Manufacturer)

Manufactured for Huntsman Corporation 2701 S. Range Road Marysville, Michigan 48040 U.S.A.
(Name and address of Purchaser)

Location of installation Huntsman Corporation 2701 S. Range Road Marysville, Michigan 48040 U.S.A.
(Name and address)

Type: Horizontal Reactor Overhead Condenser 5457A --- 5457-D1 R3 25 1997
(Horiz., vert., or sphere) (Tank, separator, jkt. vessel, heat exh., etc.) (Mfg's serial No.) (CRN) (Drawing No.) (Nat'l. Bd. No.) (Year built)

ASME Code, Section VIII, Div. 1 95 Ed and 95 Add. --- ---
Edition and Addenda (date) Code Case No. Special Service per UG-120(d)

Items 6 - 11 incl. to be completed for single wall vessels, jackets of jacketed vessels, shell of heat exchangers, or chamber of multi-chamber vessels.

3. Shell (a) No. of course(s): 2 (b) Overall length (ft & in.): 20'-0" TSA to TSB

Course(s)			Material		Thickness		Long Joint (Cat. A)			Circum. Joint (Cat. A, B & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type		Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	3'-0" ID	10'-0"	SA 516-70		3/4"	1/8"	1	Spot	0.85	1	Spot	0.85	None	
2	3'-0"	9'-4 5/8"	SA 516-70		3/4"	1/8"	1	Spot	0.85	1	Spot	0.85	None	

7. Heads: (a) None (b) None
(Mat'l Spec. No., Grade or Type) H.T. - Time & Temp (Mat'l Spec. No., Grade or Type) H.T. - Time & Temp

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	N/A													
(b)														

If removable, bolts used (describe other fastening) None

8. Type of jacket None Jacket closure N/A
(Mat'l Spec. No., Grade, size, No.) (Describe as ogee & weld, bar, etc.)

9. MAWP 430 psi at max. temp. 210 °F Min. design metal temp. -20 °F at 430 psi.
(internal) (external) (internal) (external)

10. Impact test Exempt per UG-20 (f)

11. Hydro., pneu., or comb. test press. Hydro-645 psig Proof test N/A
(Indicate yes or no and the component(s) impact tested)

Items 12 and 13 to be completed for tube sections.

12. Tubesheet: SA-240-TP304 44 1/2" OD 3 3/4" 0 Welded
Stationary (Mat'l Spec. No.) Dia., in. (subject to press.) Nom. thk., in. Corr. Allow., in. Attachment (welded or bolted)

SA 240-TP-304 44 1/2" 3 3/4" 0 Welded
Floating (Mat'l Spec. No.) Dia., in. Nom. thk., in. Corr. Allow., in. Attachment

13. Tubes: SA-249 TP-304 3/4" 14 BWG 1056 Straight
Mat'l Spec. No., Grade or Type O.D., in. Nom. thk., in. or gauge Number Type (Straight or U)

Items 14 - 18 incl. to be completed for inner chambers of jacketed vessels or channels of heat exchangers.

14. Shell (a) No. of course(s): 2 (b) Overall length (ft & in.): TSA End- 3'-9 5/8" TSB End- 3'-5 1/2"

Course(s)			Material		Thickness		Long Joint (Cat. A)			Circum. Joint (Cat. A, B & C)			Heat Treatment	
No.	Diameter, in.	Length (ft & in.)	Spec./Grade or Type		Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	36" ID	1'-11 3/8"	SA-240 TP304		3/4"	0	1	Full	1.0	1	Full	1.0	None	
2	36" ID	2'-3 1/2"	SA-240 TP304		3/4"	0	1	Full	1.0	1	Full	1.0	None	

15. Heads: (a) SA-240 TP-304 (b) SA-240 TP-304
(Mat'l Spec. No., Grade or Type) H.T. - Time & Temp (Mat'l Spec. No., Grade or Type) H.T. - Time & Temp

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	TSA-End	0.656"	0			2:1					X	1	Full	1.0
(b)	TSB-End	0.656"	0			2:1					X	1	Full	1.0

If removable, bolts used (describe other fastening) SA-193 Gr.B7 1 3/8" Ø 36 per Channel - 72 Total

(Mat'l Spec. No., Grade, Size, No.) NB-26

1. MAWP 638 psi at max. temp. 200 °F Min. design metal temp. -20 °F at 638 psi.
(internal) (external) (internal) (external)

2. Impact test Exempt per UHA-51 (d)

(Indicate yes or no and the component(s) impact tested)

3. Hydro., pneu., or comb. test press. Hydro-957psig

Proof test N/A

4. Nozzles, inspection, and safety valve openings:

Purpose (inlet, Outlet, Drain, etc.)	No.	Diameter or Size	Flange Type	Material		Nozzle Thickness		Reinforcement Material	How Attached		Location (Insp. Open.)
				Nozzle	Flange	Nom.	Corr.		Nozzle	Flange	
S Inlet/Outlet 2	1	6"	600#WN	*	**	Sch.80	0	SA-240 TP304	UW-16.1	***	
S Outlet	1	6"	600#WN	*	**	Sch.80	0	SA-240 TP304	UW-16.1	***	
S Inlet/Outlet 2	1	8"	300#WN	SA-106B	SA-105	Sch.80	1/8"	SA-516-70	UW-16.1	***	
hem. Clean	2	4"	300#WN	SA-106B	SA-105	Sch.120	1/8"	SA-516-70	UW-16.1	***	
S/SS Drain/Vent 4	1	1"	Note 1	Note 2	Note 3	Note 4	Note 5	None	UW-16.1	***	
S/SS Temp Conn 2	1	3/4"	6000#CPLG	---	Note 7	---	Note 5	None	UW-16.1	---	
S/SS Press Conn 2	1	3/4"	6000#CPLG	---	Note 3	---	Note 5	None	UW-16.1	---	

5. Supports: Skirt No Lugs None Legs None Others 2-Saddles Attached Welded to Shell
(Yes or No) (No.) (No.) (Describe) (Where and How)

6. Manufacturer's Partial Data Reports properly identified and signed by Commissioned Inspectors have been furnished for the following items of the report:
(List the name of part, item number, mfg's. name and identifying number)

7. Remarks:

Note 1: SS-300#WN TS-600#WN Note 2: SS-SA-106B TS-SA-312 TP304

Note 3: SS-SA-105 TS-SA-182 F304 Note 4: SS-XXS TS-Sch.160

Note 5: SS-0.125" TS-0 * - SA-312 TP 304

** - SA-182 F304 *** - UW-16.1 butt weld NOTE: No Inspection Openings Refer to UC-46 (a)

CERTIFICATE OF SHOP COMPLIANCE

We certify that the statements made in this report are correct and that all details of design, material, construction, and workmanship of this vessel conform to the ASME Code for Pressure Vessels, Section VIII, Division 1,

U Certificate of Authorization No. 28,487 Expires July 28, 1998

Date 97-04-24 Name TUBERATE Ltd. Signed [Signature]
(Manufacturer) (Representative)

CERTIFICATE OF SHOP INSPECTION

I, the undersigned, holding a valid commission issued by The National Board of Boiler and Pressure Vessel Inspectors and the State or Province of ONT and employed by M.C.C.R. of Ontario have inspected the pressure vessel described in this Manufacturer's Data Report on April 24, 1997, and state that, to the best of my knowledge and belief, the Manufacturer has constructed this pressure vessel in accordance with ASME Code, Section VIII, Division 1. By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date 97-04-24 Signed [Signature] Commissions 11396 BN ONTARIO #46
(Authorized Inspector) (Nat'l Board incl. endorsement, State, Province and No.)

CERTIFICATE OF FIELD ASSEMBLY COMPLIANCE

We certify that the statements on this report are correct and that the field assembly construction of all parts of this vessel conforms with the requirements of ASME Code, Section VIII, Division 1,

U Certificate of Authorization No. _____ Expires _____, 19 _____

Date _____ Name _____ Signed _____
(Assembler) (Representative)

CERTIFICATE OF FIELD ASSEMBLY INSPECTION

I, the undersigned, holding a valid commission issued by The National Board of Boiler and Pressure Vessel Inspectors and the State or Province of _____ and employed by _____ of _____ have compared the statements in this Manufacturer's Data Report with the described pressure vessel and state that parts referred to as data items _____, not included in the certificate of shop inspection, have been inspected by me and to the best of my knowledge and belief, the Manufacturer has constructed and assembled this pressure vessel in accordance with ASME Code, Section VIII, Division 1. The described vessel was inspected and subjected to a hydrostatic test of _____ psi. By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel described in this Manufacturer's Data Report. Furthermore, neither the Inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date _____ Signed _____ Commissions _____
(Authorized Inspector) (Nat'l Board incl. endorsement, State, Province and No.)

FORM U-2 MANUFACTURER'S PARTIAL DATA REPORT
A Part of a Pressure Vessel Fabricated by One Manufacturer for Another Manufacturer
As Required by the Provisions of the ASME Code Rules, Section VIII, Division 1

1. Manufactured and certified by TUBERATE Ltd. 763 Chester Street Sarnia, Ontario N7S 5N2 Canada
 (Name and address of Manufacturer)

2. Manufactured for Huntsman Corporation 2701 S. Range Road Marysville, Michigan 48040 U.S.A.
 (Name and address of Purchaser)

3. Location of installation Huntsman Corporation 2701 S. Range Road Marysville, Michigan 48040 U.S.A.
 (Name and address)

4. Type: Horizontal Overhead Condenser 5457B
 (Description of vessel part (shell, two-piece head, tube bundle)) (Mfg's serial No.)
26 5457- D1 R.3 TUBERATE Ltd. (CRN) 1997
 (Nat'l. Bd. No.) (Drawing No.) (Drawing prepared by) (Year built)

5. ASME Code, Section VIII, Div. 1 95 ED. & 95 ADD.
 Edition and Addenda (date) Code Case No. Special Service per UG-120(d)

Items 6 - 11 incl. to be completed for single wall vessels, jackets of jacketed vessels, shell of heat exchangers, or chamber of multi-chamber vessels.

6. Shell (a) No. of course(s): 2 (b) Overall length ft & in.: 20' - 0" TSA to TSB

Course(s)			Material	Thickness		Long Joint (Cat. A)			Circum. Joint (Cat. A, B & C)			Heat Treatment	
No.	Diameter, in.	Length ft & in.	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time
1	3'-0" ID	10'-0"	SA-516-70	3/4"	1/8"	1	Spot	0.85	1	Spot	0.85	None	
2	3'-0" ID	9'-4 5/8"	SA-516-70	3/4"	1/8"	1	Spot	0.85	1	Spot	0.85	None	

7. Heads: (a) <u>None</u> (b) <u>None</u>													
(Mat'l Spec. No., Grade or Type) H.T. - Time & Temp							(Mat'l Spec. No., Grade or Type) H.T. - Time & Temp						
Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
	Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a) <u>N/A</u>													

If removable, bolts used (describe other fastening) None

8. Type of jacket N/A (Mat'l Spec. No., Grade, Size, No.) N/A
 Jacket closure N/A
 If bar, give dimensions: N/A (Describe as ogee & weld, bar, etc.)
 9. MAWP 430 psi at max. temp. 210 °F Min. design metal temp. -20 °F at 430 ps
 (internal) (external) (internal) (external)

10. Impact test Exempt per UG - 20 (f)
 (Indicate yes or no and the component(s) impact tested)

11. Hydro., pneu., or comb. test press. Hydro-645 psiG Proof test N/A

Items 12 and 13 to be completed for tube sections.

12. Tubesheet: SA-240-TP 304 44 1/2" OD 3 3/4" 0 Welded
 Stationary (Mat'l Spec. No.) Dia., in. (subject to press.) Nom. thk., in. Corr. Allow., in. Attachment (welded or bolted)
SA-240- TP 304 44 1/2" OD 3 3/4" 0 Welded
 Floating (Mat'l Spec. No.) Dia., in. Nom. thk., in. Corr. Allow., in. Attachment
 13. Tubes: SA-249- TP 304 3/4" 14 Bwg. 1056 Straight
 Mat'l Spec. No., Grade or Type O.D., in. Nom. thk., in. or gauge Number Type (Straight or U)

Items 14 - 18 incl. to be completed for inner chambers of jacketed vessels or channels of heat exchangers.

14. Shell (a) No. of course(s): None (b) Overall length (ft & in.): None

Course(s)			Material		Thickness		Long Joint (Cat. A)			Circum. Joint (Cat. A, B & C)			Heat Treatment	
No.	Diameter, in.	Length ft & in.	Spec./Grade or Type	Nom.	Corr.	Type	Full, Spot, None	Eff.	Type	Full, Spot, None	Eff.	Temp.	Time	
	N/A													

Form U-2 (Back)

15. Heads: (a)

None

(b)

None

(Mat'l Spec. No., Grade or Type) H.T. - Time & Temp

(Mat'l Spec. No., Grade or Type) H.T. - Time & Temp

	Location (Top, Bottom, Ends)	Thickness		Radius		Elliptical Ratio	Conical Apex Angle	Hemispherical Radius	Flat Diameter	Side to Pressure		Category A		
		Min.	Corr.	Crown	Knuckle					Convex	Concave	Type	Full, Spot, None	Eff.
(a)	N/A													
(b)														

If removable, bolts used (describe other fastening)

None

16. MAWP

638

(internal)

psi at max. temp.

200

(internal)

(Mat'l Spec. No., Grade, Size, No.)

°F Min. design metal temp.

-20

°F at

638

p.

17. Impact test

Exempt per UHA - 51 (d)

(Indicate yes or no and the component(s) impact tested)

18. Hydro., pneu., or comb. test press.

Hydro - 957 psi

Proof test

N/A

19. Nozzles, inspection, and safety valve openings:

Purpose (inlet, Outlet, Drain, etc.)	No.	Diameter or Size	Flange Type	Material		Nozzle Thickness		Reinforcement Material	How Attached		Location (Insp. Open.)
				Nozzle	Flange	Nom.	Corr.		Nozzle	Flange	
SS Inlet	1	8"	300#WN	SA-106B	SA-105	SCH.80	1/8"	SA-516-70	Welded	Welded	
SS Outlet	1	8"	300#WN	SA-106B	SA-105	SCH.80	1/8"	SA-516-70	Welded	Welded	
Chem Clean	2	4"	300#WN	SA-106B	SA-105	SCH.120	1/8"	SA-516-70	Welded	Welded	
SS Drain	1	1"	300#WN	SA-106B	SA-105	XXS	1/8"	None	Welded	Welded	
SS Vent	1	1"	300#WN	SA-106B	SA-105	XXS	1/8"	None	Welded	Welded	
Temp. Conn.	1	3/4"	6000#CPLG	---	SA-105	---	1/8"	None	---	Welded	
Press. Conn.	1	3/4"	6000#CPLG	---	SA-105	---	1/8"	None	---	Welded	

Supports: Skirt

No

(Yes or No)

Lugs

None

(No.)

Legs

None

(No.)

Others

2-Saddles

(Describe)

Attached

Welded to st

1

(Where and How)

21. Remarks:

This is a spare unit for Item #E-201 "A" Nat'l Board Serial #25

Note: No inspection openings refer to UG - 46 (a)

CERTIFICATE OF SHOP/FIELD COMPLIANCE

We certify that the statements made in this report are correct and that all details of material, construction, and workmanship of this pressure vessel part conform to the ASME Code for Pressure Vessels, Section VIII, Division 1.

U Certificate of Authorization No.

28,487

Expires

July 28

, 1998

Date 97-05-08

Name

TUBERATE Ltd.

Signed

(Manufacturer)

(Representative)

CERTIFICATE OF SHOP/FIELD INSPECTION

I, the undersigned, holding a valid commission issued by The National Board of Boiler and Pressure Vessel Inspectors and the State or Province of Ont. and employed by M.C.E.R. T.S.S.A. of Ontario have inspected the pressure vessel part described in this Manufacturer's Data Report on May 08, 1997, and state that, to the best of my knowledge and belief, the Manufacturer has constructed this pressure vessel part in accordance with ASME Code, Section VIII, Division 1. By signing this certificate neither the Inspector nor his employer makes any warranty, expressed or implied, concerning the pressure vessel part described in this Manufacturer's Data Report. Furthermore, neither the inspector nor his employer shall be liable in any manner for any personal injury or property damage or a loss of any kind arising from or connected with this inspection.

Date 97-05-08 Signed

(Authorized Inspector)

Commissions

11396 B.N. - ONT #46.

(Nat'l Board incl. endorsement, State, Province and No.)

ITEM: E-201
Channel Flanges CFA/CFB

DATE: 12/04/96
REF : 176A1HFL

*** TAPERED HUB TYPE FLANGE, ASME SECTION V111, DIV 1 ***

DESIGN PRESSURE (psi)
DESIGN TEMPERATURE (degr F)

FLANGE : Forging SA182 F304
BOLTING: Studbolts SA193 B7
GASKET : Flat Metal Jckt Asb Filled Stnls St
FACING : Confined & 1/8" wide single nubbin
BARREL : Plate SA240 TP304

CORR ALL: 0.0000

ALLOWABLE STRESS psi

FLANGE (DES) Sfo = 15700
FLANGE (ATM) Sfa = 18800
BOLT (DES) Sb = 25000
BOLT (ATM) Sa = 25000

GASKET DETAILS:

O.D. = 39.875 " N = 0.500
bo = 0.2031 " Y = 9000
b = 0.2031 " m = 3.75
G = 39.469 "

BOLTING CALCULATIONS lbs

Wm2=b*Pi*G*y = 226678
Hp =2*Pi*b*G*p*m= 120517
H =Pi/4*G*G*p = 780580
Wm1=Hp+H = 901097

Am = Wm2/Sa or Wm1/Sb = 36.044
Ab = Bolts * Area = 41.580
W = 0.5*(Am+Ab)*Sa = 970298
Nmin = Ab*Sa/(2*Pi*y*G) = 0.466

LOADS lbf

OPERATING CONDITION

HD = Pi/4*B*B*p = 649405
HG = Wm1-H = 120517
HT = H-HD = 131175

GASKET SEATING

HG = W = 970298

LEVER ARMS in

OPERATING CONDITION

hd = R+0.5*gl = 2.3750
hg = 0.5*(C-G) = 1.1406
ht = 0.5*(R+gl+hg) = 2.0078

GASKET SEATING

hg = 0.5*(C-G) = 1.1406

FLANGE MOMENTS lbs*in

MD = HD*hd = 1542337
MG = HG*hg = 137465
MT = HT*ht = 263375
Mop= MD+MG+MT = 1943176

GASKET SEATING CONDITION lbf*in

Matm = W*hg = 1106747
Mo=max("Matm*Sfo/Sfa"or "Mop") = 1943176
Bolt Spacing Correction: CF = 1.000
M = Mo * CF / B = 53977

FLANGE DIMENSIONS in

t = 5.1250 A = 44.500
h = 1.25 B = 36.000
E = 1.375 C = 41.750
R = 1.875 G = 39.469
BOLTS: 36 gl = 1.000
SIZE: 1.375 go = 0.750
AREA: 41.58 Rise: 0.250

SHAPE CONSTANTS

K=A/B = 1.2361
T = 1.8242
Z = 4.7881
Y = 9.2818
U = 10.1998
gl/go = 1.3333
d = 69.507
ho = 5.1962
h/ho = 0.2406

F = 0.8923
V = 0.4289
f = 1.0541
e = F/ho = 0.1717
ALPHA = 1.8800
BETA = 2.1734
GAMMA = 1.0306
DELTA = 1.9366
LAMBDA = 2.9673

ALLOWABLE STR:

1.58Sfo 23550
Sfo 15700
Sfo 15700
Sfo 15700

ACTUAL STR:

LONG HUB
RADIAL
TANGENTIAL
GREATER

SH = f x M / (LAMBDA x gl^2) = 19176
SR = BETA x M / (LAMBDA x t^2) = 1505
ST = (M x Y / t^2) - Z x SR = 11867
(SH + SR) / 2 or (SH + ST) / 2 = 15522

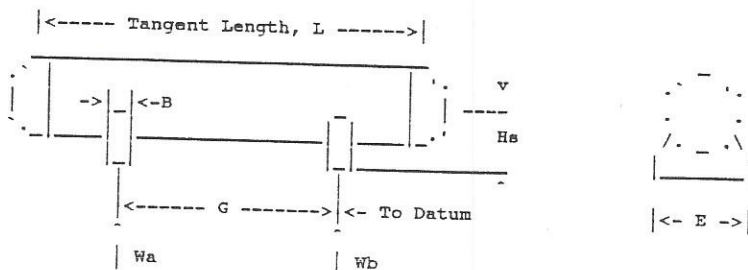
Gasket Rib Length: 0.000 "

Wm1 copied from a mating flange ? "NO"

Wm2 copied from a mating flange ? "NO"

SADDLE

Saddle material: SA 36 PLATE / SA 516-70
 Saddle allowable stress: 20000 psi
 Saddle yield stress: 36000 psi
 Saddle distance to datum: 61.5 in
 Tangent length: $L = 306.251$ in
 Vessel radius: $R = 18.75$ in
 Saddle tangent distance, left: $A = 69.25098$ in
 Saddle tangent distance, right: $A = 63$ in
 Distance between saddles: $G = 174$ in
 Saddle width: $B = 6$ in
 Saddle height: $H_s = 24$ in
 Saddle contact angle: $\theta = 120$ degrees
 Wear plate thickness: $t_p = 0.5$ in
 Wear plate width: $W_p = 13$ in
 Wear plate contact angle: 126 degrees
 Saddle construction is: Web at edge of rib
 Web plate thickness: $t_w = 0.5$ in
 Base plate length: $E = 34.5$ in
 Base plate width: $F = 7$ in
 Base plate thickness: $t_b = 0.5$ in
 Number of stiffener ribs: 2
 Stiffener rib spacing: 32.5 in
 Stiffener rib thickness: $t_r = 0.5$ in
 Anchor bolt material: SA 193 B7
 Anchor bolt allowable shear stress: 18000 psi
 Anchor bolt size & type: 1 inch, series 8 threads
 Number of anchor bolts per saddle: 2
 Saddle/foundation coeff of friction: 0.45



Weight on left saddle, op = 25568, test = 26042 lb
 Weight on right saddle, op = 24222, test = 24610 lb

Longitudinal bending stress between saddles

Circumferential seam joint efficiency: $E_c = 0.85$
 Allowable tensile stress: $S_t = 17500$ psi
 Allowable compressive stress: $S_c = 17036.4$ psi

Operating condition governs bending stress between saddles

$$\begin{aligned}
 S_1 &= \pm 3 \cdot K_1 \cdot Q \cdot (L/12) / (\pi \cdot R^2 \cdot t) \\
 &= 3 \cdot 0.1433 \cdot 24222 \cdot (306.251/12) / (\pi \cdot 18.4375^2 \cdot 0.625) \\
 &= 398.1435 \text{ psi}
 \end{aligned}$$

Longitudinal pressure stress

$$\begin{aligned}
 S_p &= P \cdot R / (2 \cdot t) \\
 &= 431.701 \cdot 18.4375 / (2 \cdot 0.625) \\
 &= 6367.589 \text{ psi}
 \end{aligned}$$

Max tensile stress $S_{1t} = S_1 + S_p = 6765.733$ psi
 Max compressive stress $S_{1c} = S_1 = 398.1435$ psi

Tensile stress is acceptable (≤ 14875 psi)

SADDLE

Compressive stress is acceptable (≤ 17036.4 psi)

Longitudinal bending stress at the saddles

Depth of left head: $H_l = 9.375$ in
 Depth of right head: $H_r = 9.375$ in

$$\begin{aligned} L_e &= 2 \cdot H_l / 3 + L + 2 \cdot H_r / 3 \\ &= 2 \cdot 9.375 / 3 + 306.251 + 2 \cdot 9.375 / 3 \\ &= 318.751 \text{ in} \end{aligned}$$

Operating condition governs, vessel weight $W_t = 49790$ lb

$$w = W_t / L_e = 156.2034 \text{ lb/in}$$

Bending moment at the left saddle (governing):

$$\begin{aligned} M_q &= w \cdot (2 \cdot H \cdot a / 3 + a^2 / 2 \cdot (R^2 - H^2) / 4) \\ &= 156.2034 \cdot (2 \cdot 9.375 \cdot 69.25098 / 3 + 69.25098^2 / 2 \cdot (18.75^2 - 9.375^2) / 4) \\ &= 431863.4 \text{ lb-in} \end{aligned}$$

$$\begin{aligned} S_1 &= \pm M_q \cdot K_1 / (\pi \cdot R^2 \cdot t) \\ &= 431863.4 \cdot 9.3799 / (\pi \cdot 18.4375^2 \cdot 0.625) \\ &= 6068.909 \text{ psi} \end{aligned}$$

Max tensile stress $S_{1t} = S_1 + S_p = 12436.5$ psi
 Max compressive stress $S_{1c} = S_1 = 6068.909$ psi

Tensile stress is acceptable (≤ 17500 psi)
 Compressive stress is acceptable (≤ 17036.4 psi)

Tangential shear stress in the shell

Operating condition governs.

$$\begin{aligned} Q_{\text{shear}} &= Q - w \cdot (a + 2 \cdot H / 3) \\ &= 25568 - 156.2034 \cdot (69.25098 + 2 \cdot 9.375 / 3) \\ &= 13774.49 \text{ lb} \end{aligned}$$

$$\begin{aligned} S_2 &= K_2 \cdot 2 \cdot Q_{\text{shear}} / (R \cdot t) \\ &= 1.1707 \cdot 13774.49 / (18.4375 \cdot 0.625) \\ &= 1399.391 \text{ psi} \end{aligned}$$

Tangential shear stress is acceptable ($\leq .8 \cdot 17500$ psi)

Circumferential stress at saddle horns

Operating condition governs.

$$\begin{aligned} S_3 &= -Q / (4 \cdot (t + t_p) \cdot (b + 1.56 \cdot \text{Sqr}(R_o \cdot t))) - 3 \cdot K_3 \cdot Q / (2 \cdot (t^2 + t_p^2)) \\ &= -25568 / (4 \cdot (0.625 + 0) \cdot (6 + 1.56 \cdot \text{Sqr}(18.75 \cdot 0.625))) - \\ &\quad 3 \cdot 0.0529 \cdot 25568 / (2 \cdot (0.625^2 + 0^2)) \\ &= -6095.627 \text{ psi} \end{aligned}$$

Circ stress at saddle horns is acceptable ($\leq 1.5 \cdot 17500$ psi)

Ring compression in shell over saddle

$$\begin{aligned} S_5 &= K_5 \cdot Q / ((t + t_p) \cdot (t_w + 1.56 \cdot \text{Sqr}(R_o \cdot t_c))) \\ &= 0.7603 \cdot 25568 / ((0.625 + 0.5) \cdot (0.5 + 1.56 \cdot \text{Sqr}(18.75 \cdot 1.125))) \\ &= 2254.399 \text{ psi} \end{aligned}$$

Ring compression in shell is acceptable ($\leq .5 \cdot 36000$ psi)

Saddle splitting load

Area resisting splitting force = Web area + wear plate area

$$A_e = H_{eff} \cdot t_w + t_p \cdot W_p$$

SADDLE

$$= 4.25 * 0.5 + 0.5 * 13$$

$$= 8.625 \text{ in}^2$$

$$S8 = K8 * Q / Ae$$

$$= 0.2035 * 26042 / 8.625$$

$$= 614.4402 \text{ psi}$$

Stress in saddle is acceptable ($\leq .666 * 20000 \text{ psi}$)

Shear Stress in anchor bolting, one end slotted

Maximum seismic base shear: 0 lb

Thermal expansion base shear: $W * U = 26042 * 0.45 = 11718.9 \text{ lb}$

Bolt shear stress $= 11718.9 / (.551 * 2) = 10634.21 \text{ psi}$

Anchor bolt stress is acceptable ($\leq 18000 \text{ psi}$)

Saddle plate buckling check (Escoe pg 251)

Allowable compressive stress Sc is the lesser of 20000 or:

$$Sc = 1.28 * \pi^2 * 29000000 / (12 * (1 - 0.3^2) * (32.5 / 0.5)^2)$$

$$= 7940.693 \text{ psi}$$

Allowable compressive load on the saddle

$$be = di * ts / (di * ts + 2 * tw * (b - 1))$$

$$= 32.5 * 0.5 / (32.5 * 0.5 + 2 * 0.5 * (6 - 1))$$

$$= .7647059$$

$$Fb = n * (As + 2 * be * ts) * Sc$$

$$= 2 * (5.5 * 0.5 + 2 * 0.7647059 * 0.5) * 7940.693$$

$$= 55818.4 \text{ lb}$$

Saddle loading of 26042 is $\leq Fb$; satisfactory.

Saddle base plate check (Bednar pg 182)

$$tb = \text{Sqr}(Q_{\max} * F / (Sa * 1.2 * E))$$

$$= \text{Sqr}(26042 * 7 / (20000 * 1.2 * 34.5))$$

$$= .4692141 \text{ in}$$

Base plate thickness of 0.5 in is adequate.

Foundation bearing check

$$Sf = Q_{\max} / (F * E)$$

$$= 26042 / (7 * 34.5)$$

$$= 107.8344 \text{ psi}$$

Concrete bearing stress $< 750 \text{ psi}$; satisfactory.